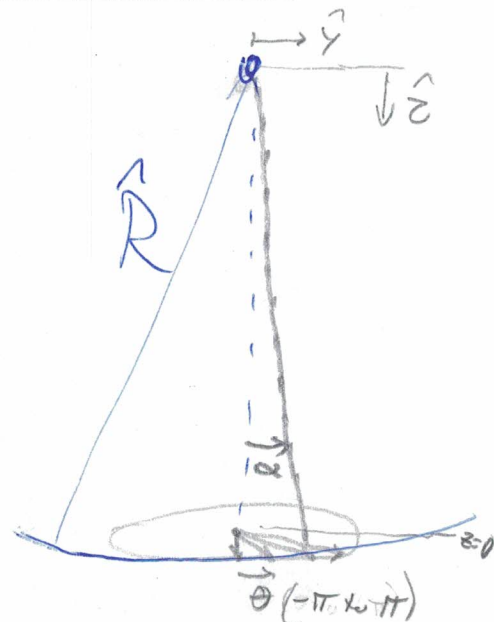
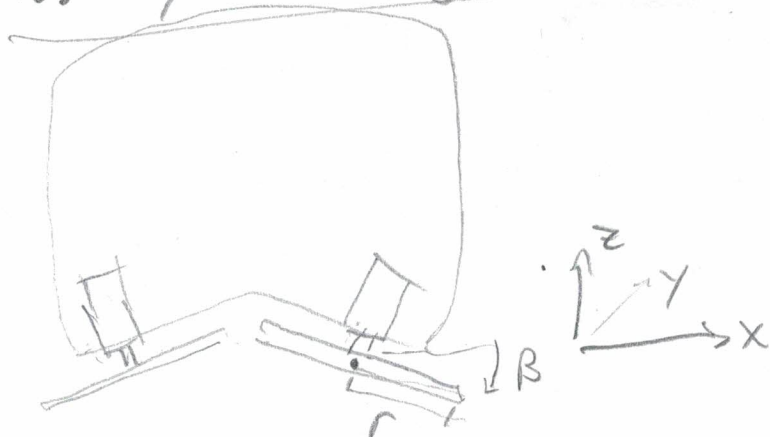


2019-03-21

# Passively Balancing robot



Point on wheel : side view :

$$z = -r \cos \theta \sin \beta$$

$$y = r \sin \theta \rightarrow \theta = \sin^{-1} \frac{y}{r}$$

$$\therefore z = r \cos(\sin^{-1}(\frac{y}{r})) \sin \beta \rightarrow \text{small angle } \theta : \theta \approx \frac{y}{r}$$

Effective Radius of Curvature :

$$\hat{R} = \frac{(1 + (\frac{dz}{dy})^2)^{3/2}}{\frac{d^2z}{dy^2}}$$

$$z \approx r \cos(\frac{y}{r}) \sin \beta$$

$$[z \approx r \sin \beta] \quad (y)$$

$$\frac{dz}{dy} = r \sin \beta (\sin(\frac{y}{r})) \cdot \frac{1}{r} = -\sin \frac{y}{r} \sin \beta$$

$$\text{small angle: } \frac{dz}{dy} \approx -\frac{y}{r} \sin \beta$$

$$\frac{d^2z}{dy^2} \approx -\frac{1}{r} \sin \beta \quad R = \frac{(1 + \frac{y^2}{r^2} \sin^2 \beta)^{3/2}}{-\frac{1}{r} \sin \beta}$$

or for any angle  $\theta$  : find equivalent angle  $\alpha$

$$y = r \sin \theta = R \sin \alpha = \hat{y}$$

$$z = -r \cos \theta \sin \beta = -r \sin \beta + R - R \cos \alpha$$

$$\text{or } \hat{z} = R \cos \alpha = R - r \sin \beta + r \cos \theta \sin \beta$$

small angles:  $y \approx R \alpha$

$$\therefore \frac{dy}{d\alpha} = R$$

$$\text{consider } \frac{\hat{y}}{\hat{z}} = \frac{R \sin \alpha}{R \cos \alpha} = \frac{r \sin \theta}{R + r \sin \beta (\cos \theta - 1)}$$

Alt: Guess:

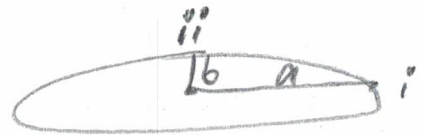
$$R = r \text{ when } \beta = \frac{\pi}{2}$$

$$R = \infty \text{ when } \beta = 0$$

$$\text{Guess } R = \frac{r}{\sin \beta}$$

From Wikipedia -- "Radius of Curvature"

For ellipse with major axis  $2a$   
and minor axis  $2b$ ,



curvature

$$\text{at } i: R = \frac{b^2}{a}$$

$$\text{ii: } R = \frac{a^2}{b}$$

$$\text{here } b = r \sin \beta$$

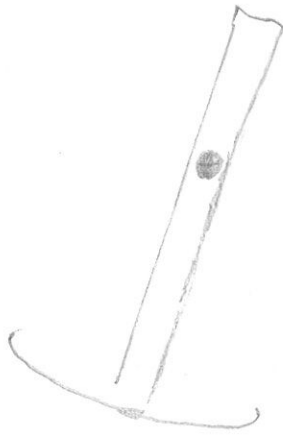
$$a = r$$

$$\therefore R = \frac{r^2}{r \sin \beta} = \frac{r}{\sin \beta}$$

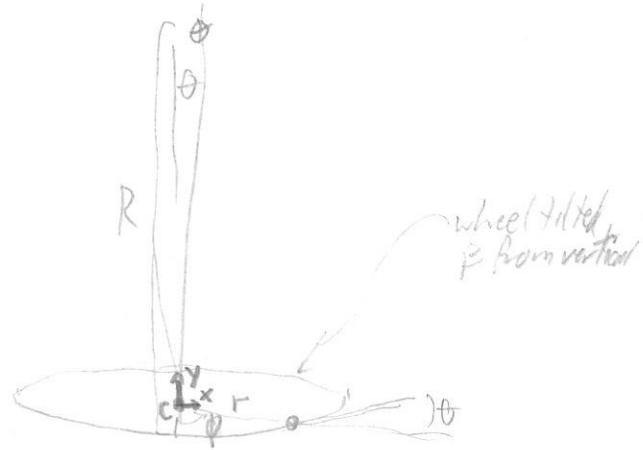
In practice  $r$  is fixed (eg. 30mm) and  $R$  is a design spec (eg 200mm)

Therefore  $\beta = \sin^{-1}\left(\frac{r}{R}\right)$  is solved for (eg.  $8.6^\circ$ )

2019-05-08



MAPPING  
of  $\theta$  to  $\phi$



point =  $x = r \sin \phi$   
rel to point C  $y = -r \cos \phi \cos \beta$

$$\tan \theta = \frac{dy}{dx} = \frac{\frac{\partial y}{\partial \phi} d\phi}{\frac{\partial x}{\partial \phi} d\phi}$$

$$= \frac{r \sin \phi \cos \beta d\phi}{r \cos \phi d\phi}$$

$$\tan \theta = \tan \phi \cos \beta$$

~~Ans~~

$\therefore \theta = \tan^{-1}(\tan \phi \cos \beta)$   
and  $\phi = \tan^{-1}(\tan \theta \sec \beta)$

$$\tan \theta = \frac{dy}{dx} = \frac{\frac{\partial y}{\partial \phi} d\phi}{\frac{\partial x}{\partial \phi} d\phi}$$

$$\tan \theta = \frac{dy}{dx} = \frac{r \cos \phi d\phi}{r \sin \phi \cos \beta d\phi}$$

$$\tan \theta = \cot \phi \sec \beta$$

$$\therefore \theta = \tan^{-1}(\cot \phi \sec \beta)$$

$$\text{or } \phi =$$

$$\text{or } \cot \theta = \tan \phi \cos \beta$$

$$\tan^{-1}(\cot \theta \sec \beta) = \phi$$

RADIUS of CURVATURE

$$R = \frac{dx}{d\theta}$$

~~Not True~~

~~sin(tan^{-1}(tan~~ where  $x = r \sin \phi$

$$\frac{dx}{d\theta} = \frac{dx}{d\phi} \frac{d\phi}{d\theta}$$

$$\frac{dx}{d\theta} = r \cos \phi \frac{d}{d\theta} (\tan^{-1}(\tan \theta \sec \beta))$$

Consider  $\sin \phi$  given  $\tan \phi = \tan \theta \sec \beta$



$$1^2 = a^2 + a^2 \tan^2 \theta \sec^2 \beta$$

$$1^2 = a^2 (1 + \tan^2 \theta \sec^2 \beta)$$

$$a^2 = \frac{1}{1 + \tan^2 \theta \sec^2 \beta} = \frac{\cos^2 \beta}{\cos^2 \beta + \tan^2 \theta} \quad (\text{by mult } \frac{\cos^2 \beta}{\cos^2 \beta})$$

$$a = \sqrt{\frac{\cos^2 \beta}{\cos^2 \beta + \tan^2 \theta}} = \frac{\cos \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

$$\therefore \sin \phi = a \tan \theta \sec \beta = a \frac{\tan \theta}{\cos \beta} = \frac{\cos \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}} \frac{\tan \theta}{\cos \beta}$$

$$\therefore \sin \phi = \frac{\tan \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

? check  
and  
re-derive

$$\therefore x = r \sin \phi = \frac{r \tan \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

$$\text{Then } \cos \phi = a = \frac{\cos \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

$$\text{and } y = -r \cos \phi \cos \beta$$

$$y = -r \frac{\cos^2 \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

~~$R = \frac{dx}{d\theta}$~~   
~~not true~~

$$R = \sqrt{\left(\frac{dx}{d\theta}\right)^2 + \left(\frac{dy}{d\theta}\right)^2}$$

from Wolfram Alpha

$$\frac{dx}{d\theta} = \frac{r \cos^2 \beta \sec^2 \theta}{(\cos^2 \beta + \tan^2 \theta)^{3/2}}$$

$$\text{for } x = \frac{r \tan \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

$$\frac{dy}{d\theta} = \frac{r \cos^2 \beta \tan \theta \sec^2 \theta}{(\cos^2 \beta + \tan^2 \theta)^{3/2}}$$

$$\text{for } y = \frac{-r \cos^2 \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

$$\left(\frac{dx}{d\theta}\right)^2 = \frac{r^2 \cos^4 \beta \sec^4 \theta}{(\cos^2 \beta + \tan^2 \theta)^3}$$

$$\left(\frac{dy}{d\theta}\right)^2 = \frac{r^2 \cos^4 \beta \tan^2 \theta \sec^4 \theta}{(\cos^2 \beta + \tan^2 \theta)^3}$$

$$\begin{aligned} \therefore \left(\frac{dx}{d\theta}\right)^2 + \left(\frac{dy}{d\theta}\right)^2 &= \frac{r^2 \cos^4 \beta \sec^4 \theta}{(\cos^2 \beta + \tan^2 \theta)^3} (1 + \tan^2 \theta) \\ &= \frac{r^2 \cos^4 \beta \sec^6 \theta}{(\cos^2 \beta + \tan^2 \theta)^3} \end{aligned}$$

$\underbrace{1 + \tan^2 \theta}_{= \sec^2 \theta}$

$$\therefore R = \sqrt{\left(\frac{dx}{d\theta}\right)^2 + \left(\frac{dy}{d\theta}\right)^2} = \sqrt{\frac{r^2 \cos^4 \beta \sec^6 \theta}{(\cos^2 \beta + \tan^2 \theta)^3}} = \frac{r \cos^2 \beta \sec^3 \theta}{(\cos^2 \beta + \tan^2 \theta)^{3/2}}$$

checks:

for  $\theta = 0$

$$R = \frac{r}{\cos \beta} \checkmark$$

for  $\theta = \frac{\pi}{2}$

$$R = \frac{\infty}{\cos \beta} \checkmark$$

but limit is  $r \cos \beta$

Then where is  $R = r$ ? (for what angle  $\theta$ )?

$$R = \frac{r \cos^2 \beta \sec^3 \theta}{(\cos^2 \beta + \tan^2 \theta)^{3/2}} = r$$

$$\cos^2 \beta \sec^3 \theta = (\cos^2 \beta + \tan^2 \theta)^{3/2} \quad \text{raise to } 2/3 \text{ power}$$

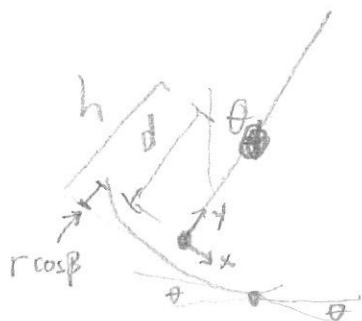
$$\cos^{4/3} \beta \sec^2 \theta = \cos^2 \beta + \tan^2 \theta \quad \text{mess}$$

= Big ugly mess!  
Solve numerically

$$\left. \begin{array}{l} \text{for } \beta = 81^\circ : \theta (R=r) \approx \pm 0.25 \text{ rad} \\ 80^\circ : \theta (R=r) = \pm 0.265 \end{array} \right\} \approx 15 \text{ deg}$$

Note! This is of no consequence.

Of consequence is the angle when the COM is farther forward than the contact point



distance of COM ahead of wheel center  $\overset{\text{Peprx}}{=} d \sin \theta$

position of contact point in Cartesian space:

$$\vec{p}_{cp} = \begin{bmatrix} \cos \theta & \sin \theta \\ -\sin \theta & \cos \theta \end{bmatrix} \begin{bmatrix} x(\theta) \\ y(\theta) \end{bmatrix}$$

$$\therefore p_{cpx} = \cos \theta x(\theta) + \sin \theta y(\theta)$$

$$= \cos \theta \left( \frac{r \tan \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}} \right) + \sin \theta \left( \frac{-r \cos^2 \beta}{\sqrt{\cos^2 \beta + \tan^2 \theta}} \right)$$

$$\therefore p_{cpx} = \frac{r \sin \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}} (1 - \cos^2 \beta) = \frac{r \sin^2 \beta \sin \theta}{\sqrt{\cos^2 \beta + \tan^2 \theta}}$$

Solve  $P_{cmx} = P_{cpX}$

to find  $\theta_{lim}$ : max statically stable lean angle

$$d \sin \theta_{lim} = \frac{r \sin^2 \beta \sin \theta_{lim}}{\sqrt{\cos^2 \beta + \tan^2 \theta_{lim}}}$$

$$\sqrt{\cos^2 \beta + \tan^2 \theta_{lim}} = \frac{r}{d} \sin^2 \beta$$

$$\cos^2 \beta + \tan^2 \theta_{lim} = \frac{r^2}{d^2} \sin^4 \beta$$

$$\tan^2 \theta_{lim} = \frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta \quad \leftarrow \textcircled{*}$$

$$\tan \theta_{lim} = \sqrt{\left(\frac{r}{d}\right)^2 \sin^4 \beta - \cos^2 \beta}$$

$$\theta_{lim} = \tan^{-1} \left( \sqrt{\frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta} \right)$$

at angle where ~~stability~~ COM is directly vertical from contact point.

This is  $\theta_{lim}$  for static stability.

Check for  $d=4r$  : For  $\beta = 81^\circ$  :  $\theta_{lim} = 0.185 \text{ rad} \approx 10.6^\circ$

$80^\circ$  :  $\theta_{lim} = 0.167 \text{ rad} \approx 9.6^\circ$

$90^\circ$  :  $\theta_{lim} = \tan^{-1} \left( \frac{r}{d} \right) \checkmark \Rightarrow 0.245 \text{ rad} \approx 14.0^\circ$

~~So, angle where static~~

Note: Speed limit for sudden stop is defined by energy conservation:

Moving Upright:

$$E = T + V$$

$$= \frac{1}{2}mv^2 + 0$$

After stops and rises to "edge" where height is max:

$$E = T + V$$

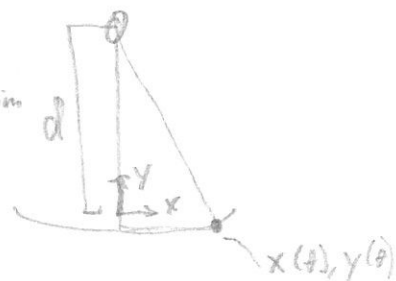
$$= 0 + mg \Delta y_{\text{com}} \quad \text{change in height of COM}$$

This max height is the same <sup>configuration</sup> as the static limit!

height when upright =  $h = d + r \cos \beta (= y_0)$

height at static limit =  $\| [x(\theta), d - y(\theta)] \|_{\theta_{\text{lim}}}$   
( $y_{\text{max}}$ )

$$\Delta y_{\text{com}} = y_{\text{max}} - y_0$$



$$y_{\text{max}} = \sqrt{x(\theta_{\text{lim}})^2 + (d - y(\theta_{\text{lim}}))^2}$$

$$= \sqrt{\frac{r^2 \tan^2 \theta_{\text{lim}}}{\cos^2 \beta + \tan^2 \theta_{\text{lim}}} + d^2 - 2d(-r \cos \beta) + \frac{r^2 \cos^2 \beta}{\cos^2 \beta + \tan^2 \theta_{\text{lim}}}}$$



all  $\tan^2 \theta_{lim}$  can be substituted with Eq. \*

$$\tan^2 \theta_{lim} = \frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta$$

$$\therefore \gamma_{max} = \sqrt{\frac{r^2 \left( \frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta \right) + d^2 + 2dr \cos^2 \beta}{\cos^2 \beta + \left( \frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta \right)} + \frac{r^2 \cos^4 \beta}{\cos^2 \beta + \frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta}}$$

$$\gamma_{max} = \sqrt{\frac{\frac{r^4}{d^2} \sin^4 \beta - \cos^2 \beta r^2}{\frac{r^2}{d^2} \sin^4 \beta} + d^2 + \frac{2dr \cos^2 \beta}{\sqrt{\frac{r^2}{d^2} \sin^4 \beta}} + \frac{r^2 \cos^4 \beta}{\frac{r^2}{d^2} \sin^4 \beta}}$$

$$\gamma_{max} = \sqrt{\frac{\frac{r^4}{d^2} \sin^4 \beta - r^2 \cos^2 \beta + r^2 \sin^4 \beta + r^2 \cos^4 \beta + 2r^2 \sin^2 \beta \cos^2 \beta}{\frac{r^2}{d^2} \sin^4 \beta}}$$

$$\gamma_{max} = \sqrt{\frac{r^2 \sin^4 \beta - d^2 \cos^2 \beta + d^2 \sin^4 \beta + d^2 \cos^4 \beta + 2d^2 \sin^2 \beta \cos^2 \beta}{\sin^4 \beta}}$$

$$\gamma_{max} = \frac{d}{\sin^2 \beta} \sqrt{\frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta + \sin^4 \beta + \cos^4 \beta + 2 \sin^2 \beta \cos^2 \beta}$$

$$Y_{\max} = \frac{d}{\sin^2 \beta} \sqrt{\frac{r^2}{d^2} \sin^4 \beta - \cos^2 \beta + \underbrace{(\sin^2 \beta + \cos^2 \beta)^2}_{=1^2}}$$

$$= \frac{d}{\sin^2 \beta} \sqrt{\frac{r^2}{d^2} \sin^4 \beta + \underbrace{-\cos^2 \beta + 1}_{=\sin^2 \beta}}$$

$$= \frac{d}{\sin^2 \beta} \sqrt{\frac{r^2}{d^2} \sin^4 \beta + \sin^2 \beta}$$

$$= \frac{d}{\sin \beta} \sqrt{\frac{r^2}{d^2} \sin^2 \beta + 1}$$

$\searrow = \frac{r^2}{d^2} \frac{d^2}{r^2}$

$$Y_{\max} = \frac{r}{\sin \beta} \sqrt{\sin^2 \beta + \frac{d^2}{r^2}}$$

$$Y_{\max} = \sqrt{r^2 + \frac{d^2}{\sin^2 \beta}} = \sqrt{r^2 + d^2 \csc^2 \beta}$$

Maximum value of  $R$  is  $\frac{r}{\cos \beta}$  (upright)

What <sup>min</sup> value of  $\beta$  is needed for  $R \geq h$ ?  
(i.e. static limit)

$$h = d + r \cos \beta$$

Find  $\beta$  s.t.  $R \geq h$

$$R \geq h$$

$$\frac{r}{\cos \beta} \geq d + r \cos \beta$$

$$r \geq d \cos \beta + r \cos^2 \beta$$

$$0 \geq r \cos^2 \beta + d \cos \beta - r$$

Quad Formula  $\therefore \cos \beta_{\text{lim}} = \frac{-d \pm \sqrt{d^2 + 4r^2}}{2r}$

MAX value  
of  $\cos \beta$

is MIN value (magnitude) of  $\beta$

consider for  $d=4r$  :  $\cos \beta_{\text{lim}} = \frac{-4r \pm \sqrt{16r^2 + 4r^2}}{2r} = -2 \pm \sqrt{5} = -4.2361$

$\therefore \beta_{\text{min}} = 1.3325 \text{ rad} = 76.35^\circ$

0.2361  
not feasible

## Next QUESTIONS

- Compute MAX speed that's ok for instantaneous stop.

↳ Finish calcs already underway

- Compute MAX <sup>allowable</sup> speed deviation as a function of speed.

- Consider the friction model:

- Friction force opposes slip direction

- Slip direction is complex:

depends on

- $\vec{V}_{com}$

- $\vec{\Omega}$  body

- $\vec{\omega}$  wheels (each wheel)

- $\theta$

\* LOWER FRICTION MAY BE BETTER FOR STABILITY

↳ but zero won't move

↳ So there may be an optimum. How to define it?

- Consider steering - including steering at speed.

- Design + Testing

Stability: Inv. pend: sanity check

2019-05-08



$$\sum M_o = I \ddot{\theta} = +mgd \sin \theta$$

$\uparrow$   
 $md^2$

$$md^2 \ddot{\theta} = mgd \sin \theta \approx mgd \theta$$

$$\ddot{\theta} = \frac{g}{d} \theta + \frac{\tau}{md^2}$$

State Space

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} = \underbrace{\begin{bmatrix} 0 & 1 \\ \frac{g}{d} & 0 \end{bmatrix}}_A \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{md^2} \end{bmatrix} \tau$$

or

$$\text{eig: } \det(\lambda I - A) = 0$$

$$\det \begin{pmatrix} \lambda & -1 \\ -\frac{g}{d} & \lambda \end{pmatrix} = 0$$

$$\lambda^2 - \frac{g}{d} = 0$$

$$\therefore \lambda = \pm \sqrt{\frac{g}{d}}$$

LaPlace:

$$s^2 \theta = \frac{g}{d} \theta + \frac{1}{md^2} \tau$$

$$\theta(s^2 - \frac{g}{d}) = \frac{1}{md^2} \tau$$

$$\therefore \frac{\theta}{\tau} = \frac{1}{md^2(s^2 - \frac{g}{d})}$$

poles:  $s^2 - \frac{g}{d} = 0$

$$s = \pm \sqrt{\frac{g}{d}}$$

existence of  
pole at  $+\sqrt{\frac{g}{d}}$   
implies unstable

(positive real part)

contrast with ordinary pendulum:

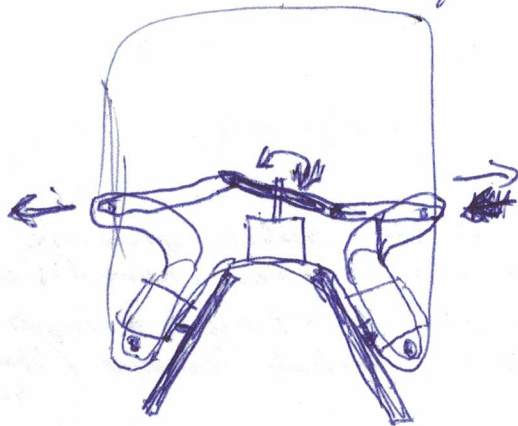


$$\ddot{\theta} = -\frac{g}{d} \theta + \frac{\tau}{md^2}$$

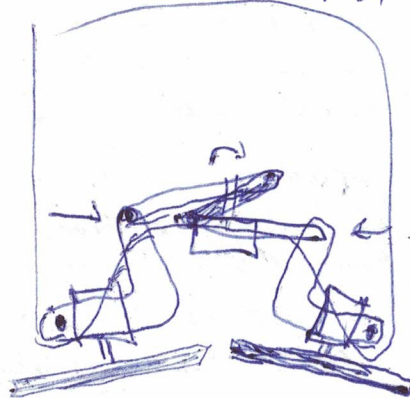
$$\therefore s = \pm \sqrt{-\frac{g}{d}} = \pm i \sqrt{\frac{g}{d}} \Rightarrow \text{zero real part.}$$

Not unstable.

## ① Wobblebot with dynamic wheel tilt control



B: BALANCER



W: WOBBLER

- Start in W mode when off
- Start moving slowly
- Activate dynamic balance if (?)
  - target speed is above/approaches max ~~hard stop~~ <sup>5-stop</sup> speed
  - detect angles approaching stable limits
  - detect "The Sitters"
  - Or simply "as speed increases"
  - or "as dynamic balance demands increase"
    - ↳ eg. observe  $(\text{avg accel} \cdot \ddot{\theta}) > 0$  (same sign)
    - ↳ but this could be confused by motor action

possibly  
just increase  
P, I gains?

- Deactivate dynamic balance as speed lowers... and other opposites of

• "Dynamic Balance" = B mode • "Deactivated" = W mode.

{ Cite Gordon McComb - Robot Banana }

## ~~Wobblebot~~ Motivation:

- Most <sup>Upright</sup> BALANCING Robots require are typically highly dynamic machines
- Balancing ~~high aspect~~ machines with high COM (higher than wheel axis) requires active ~~control and sensing and control~~ to drive wheels under the COM
- This ~~causes such machines~~ <sup>active control</sup> requires high power actuators capable of rapidly ~~continuous~~ righting the robot after any disturbance.
- ~~The risk~~ Such robots also require either manual launch from a standing position or ~~high-powered~~ stand-up maneuvers to raise themselves from a lying-down position.
- They also create a safety risk <sup>because</sup> ~~in that~~ system malfunctions such as power failure are highly likely to cause the robot to ~~fall~~ topple over, at great risk to <sup>nearby</sup> people and objects.
- This paper <sup>describes</sup> ~~proposes~~ an alternative ~~design for~~ <sup>concept for</sup> two-wheeled balancing robots with high <sup>COM</sup> centers of mass, which ~~includes a~~ includes a region of passive upright stability in which the robot will not fall even if it loses power. ~~We analyze the design~~ We analyze these ~~stability properties~~ origin and limits of <sup>leggy</sup> ~~stability~~ <sup>that</sup>