# EBU6503 Control Theory

Frequency Response and Root Locus

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## Stability

If G(s) is the forward path transfer function and H(s) is the feedback path transfer function, the open loop transfer function is G(s)H(s).

Closed-loop stability is achieved when (magnitude of GH is ≤ 1 when the phase of GH is -180°)

# Frequency Response

If the variable "s" in the open loop transfer function G(s)H(s) is replaced by "jw" we then have the system's open loop frequency response  $GH(j\omega)$ .

A plot of  $GH(j\omega)$  in magnitude and phase versus frequency can be analysed to discuss the system's stability and the system's transient response.

# Frequency Response

If GH(jω) is plotted on polar graph paper the result is called a Nyquist Diagram.

If it is plotted on a Nichol's Chart it is called a Nichol's Diagram, and useful information can be obtained more easily from this.

(Other plots such as Bode Plot can be used)

## Absolute and Relative Stability

- A system is Absolutely Stable if it is stable on closed loop.
- A measure of its Relative Stability are the values of Gain Margin and Phase Margin (both of the figures must be known).
- Good relative stability is a compromise between too much overshoot (lightly damped) and too slow (heavily damped)

## Absolute and Relative Stability

Measures of good relative stability are empirical figures (i.e. not mathematically calculated) and can be given approximately as:

- GM of between 10 and 12 dB
- and PM of between 45 and 50°

## System Response

- A system's response to an input signal (i.e. a change in its set-point value) comprises two parts:
- Transient Response. This is the output change with time until the output settles down.
- Steady-state Response. This is the output value (or function) after the transients have died away.

# Transient Response

A system's closed-loop transient response can be estimated from its open-loop frequency response plot.

The necessary information can be more easily obtained using a Nichol's diagram.

## Transient Response

From the Nichol's open-loop plot we can obtain two closed-loop values:

Maximum closed loop magnitude  $M_{max}$  and resonant frequency  $\omega_r$ 

These can be used to calculate  $\zeta$  and  $\omega_n$ 

If the system is second order then these two values are accurate, otherwise they are approximate.

## Transient Response

- $\zeta$  and  $\omega_n$  can be used to calculate the closed-loop system's:
- Peak overshoot
- Time to peak overshoot
- Settling time
- Rise time.
- If the system is not second order, then the transient response calculated is its second order approximation.

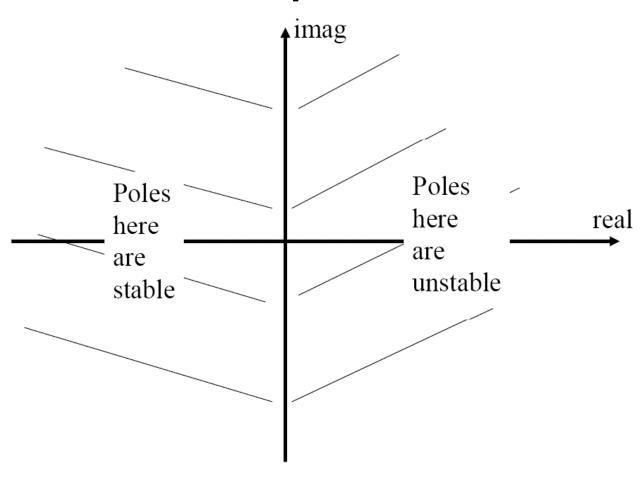
## Steady-State Response

The system's steady-state response can be found using the Laplace final value theorem:

Limit  $f(t)_{t\to\infty}$ =limit  $sF(s)_{s\to0}$ where F(s) is the system's closed loop

transfer function.

# S-Plane and Closed-Loop Response



## CL poles and pole dominance

#### If a closed loop system is stable:

- All its poles are in the LH half of the splane.
- The nearer a pole is to the origin, the longer its time-constant
- Poles nearer the origin dominate the response
- If a pair of poles is nearer the origin than all others, then the system can be approximated by a second order system

- The term "Root Locus" refers to an splane diagram of the locus of a system's closed loop poles as some parameter (usually a loop gain constant K) varies.
- The locus can then be used to analyse the system's CL behaviour for different values of K.
- The CL locus can be derived from the open loop poles

#### **Example:**

Consider the unity feedback system whose forward path transfer function is

$$G(s) = \frac{K}{s(s+4)}$$

## Example

The closed loop transfer function F(s):

$$F(s) = \frac{K}{s^2 + 4s + K}$$

## Example

The poles of F(s) are:

$$s = -2 \pm \sqrt{4 - K}$$

Plot the poles on an s-plane for:

- K=0
- Critical damping
- K→∞

This is a simple example to illustrate the technique.

The root locus can be used to calculate K that would give a particular closed loop response, e.g. a given value of ζ (and hence peak overshoot).

Relationship between closed loop pole position and  $\zeta$  and  $\omega$ :

There are rules to let us construct the root locus for any system:

- 1. Plot the open loop poles and zeroes
- 2. He number of loci is equal to the order of the characteristic equation
- 3. Each locus starts at an open looppole when K=0 and finishes either at an open loop zero or infinity when K=∞

- 4. Loci either move along the real axis or occur as complex conjugate pairs of loci
- 5. A point on the real axis is part of the locus if the number of poles and zeroes to the right of the point concerned is odd
- 6. When the locus is far enough from the open loop poles and zeroes, it becomes asymptotic to lines making angles to the real axis given by:

P=number of OL poles  $\frac{\pm n \pi}{P - Z}$  n=odd integer

7. The asymptotes intersect the real axis at a point x given by

$$x = \frac{\Sigma poles - \Sigma zeroes}{P - Z}$$

8. The break-away point between two poles or break-in point between two zeroes can be found by calculating the roots of the derivative of the characteristic equation

9. The angle of departure from a complex pole or zero is given by  $Angle = 180 - \phi_P + \phi_Z$ 

where  $\phi_P$  is the sum of all the angles subtended by the other poles and  $\phi_Z$  is the sum of the angles of any zeroes.

10. The limiting value of K for stability may be found using the Routh Array on the characteristic equation. This allows the value of the loci at the intersection of the imaginary axis to be determined.

## Routh Array

$$D(s) = a_n s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0$$

## Root Locus Example

#### **Example:**

A unity feedback system has forward path transfer function:

$$G(s) = \frac{K}{s(s+1)(s+2)}$$

Sketch the root locus plot and then determine the value of K to give a damping ratio of approximately 0.5.