

## Exp. 6. Modulus of rigidity by statical method

### Object

To determine the modulus of rigidity for the material of a wire by statical method using Barton's apparatus.

### Apparatus

Barton's apparatus, metre scale, vernier callipers, screw gauge and slotted weights of half-kilogram each.

The apparatus consists of a wire  $AL$  clamped at  $A$  with its lower end fixed to a heavy cylinder as shown in fig. 6.1. Two parallel flexible threads leave opposite sides of the cylinder tangentially at two diametrically opposite points and passing over two identical frictionless pulleys carry pans of equal weights. Pointers screwed to the experimental wire can move over graduated circular scales  $B$ ,  $C$  and  $D$ .

### Theory

Suppose, by applying a couple (placing equal weights on the pans), the lower end of the wire is twisted through an angle  $\theta$  radian.

Let  $l$ ,  $r$ ,  $\eta$  be respectively the length, radius and co-efficient of rigidity of the material of the wire.

(a) The cylinder may be considered to consist of a large number of coaxial hollow cylindrical shells and in fig. 6.2, let us consider a cylindrical shell of radii  $x$  and  $x + dx$  respectively. Let  $AB$  be a line (fig. 6.3) parallel to the axis  $DC$  before the cylinder is twisted. On twisting,  $B$  shifts to the position  $B'$  and the hollow cylinder is sheared through an angle  $\phi$ . If this hollow cylinder is cut and flattened out, it will form a rectangle  $ABEF$  of sides  $l$  and  $2\pi x$  (fig. 6.4). After

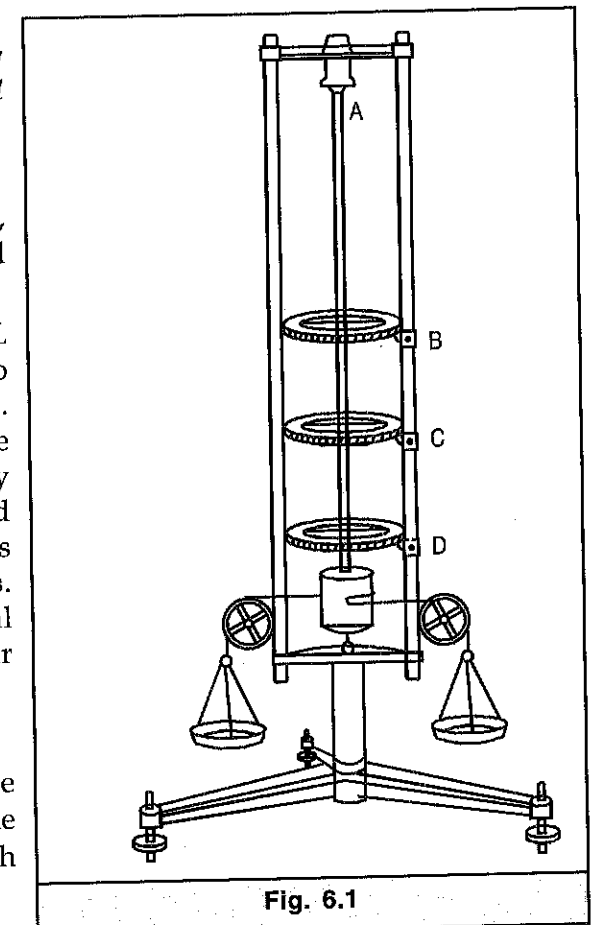


Fig. 6.1

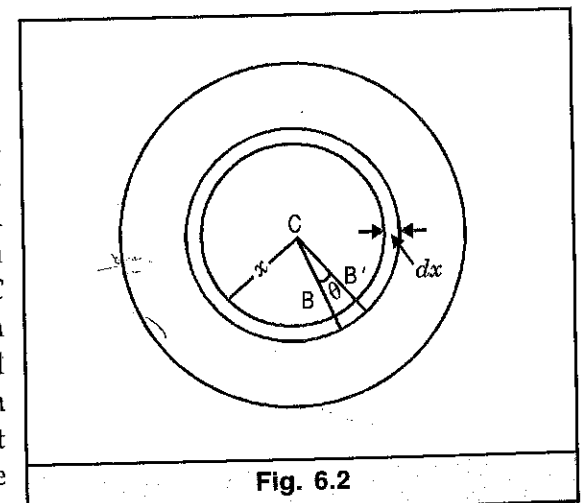


Fig. 6.2

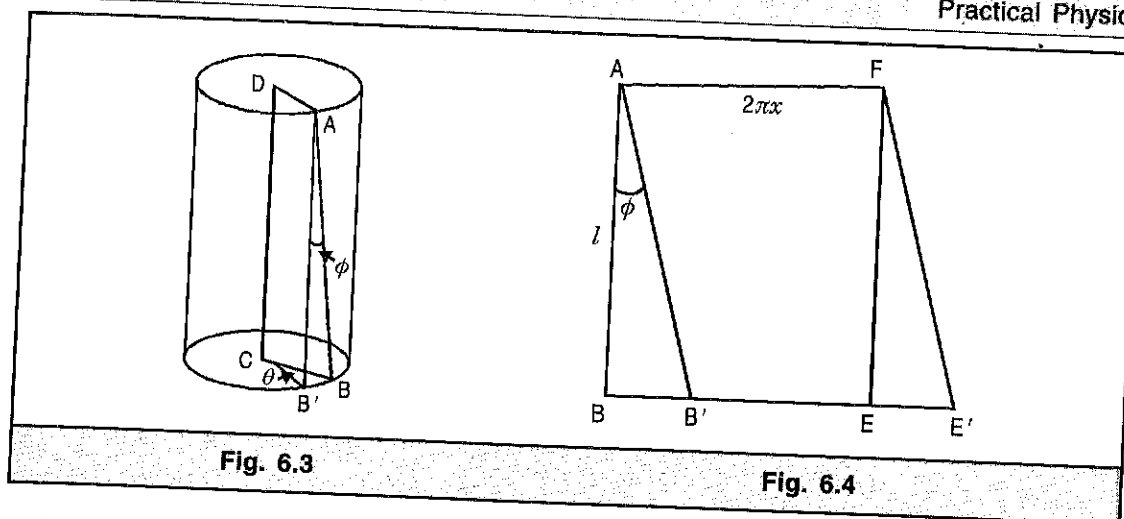


Fig. 6.3

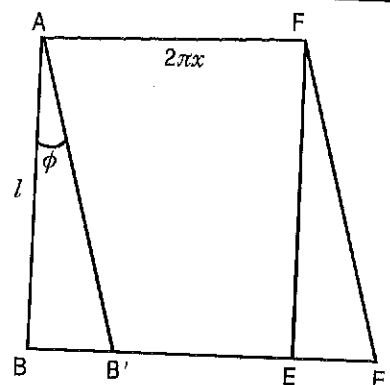


Fig. 6.4

twisting, a parallelogram  $AB'E'F$  is obtained. The angle through which this hollow cylinder is sheared is clearly  $\phi$ .

(b) Now,

$$\begin{aligned} \text{arc } BB' &= x\theta \\ &= l\phi. \end{aligned}$$

(fig. 6.3)

$\therefore$

$$\phi = \frac{x\theta}{l}.$$

Since,

$$\begin{aligned} \eta &= \frac{\text{shearing stress}}{\text{angle of shear}} \\ &= \frac{\text{shearing stress}}{\phi} \end{aligned}$$

$$\therefore \text{ we have, shearing stress} = \eta \times \phi = \eta_1 \frac{x\theta}{l}.$$

Now face area of this hollow cylinder  $= 2\pi x dx$ .

Therefore, total shearing force on this area

$$\begin{aligned} &= 2\pi x dx \times \frac{\eta x \theta}{l} \\ &= 2\pi \eta \frac{\theta}{l} x^2 dx \end{aligned}$$

Moment of the force about the axis  $CD$

$$= 2\pi \eta \frac{\theta}{l} x^2 dx \times x$$

The total twisting couple on the cylinder

$$\begin{aligned} &= \int_0^r 2\pi \eta \frac{\theta}{l} x^3 dx \\ &= \frac{\pi \eta \theta r^4}{2l} \end{aligned}$$

(c) In the Barton's apparatus, if  $d$  be the diameter of the cylinder,  $\theta$  radian be the angle of twist and  $m$  be the mass placed on each of the pans, the twisting couple  $= mgd$ .

$$\therefore mgd = \frac{\eta \theta \pi r^4}{2l}$$

and hence

$$\eta = \frac{2mgld}{\pi \theta r^4}$$

When the angle of twist  $\theta$  is measured in degrees, the expression for  $\eta$  is modified to

$$\eta = \frac{360mgld}{\pi^2 r^4 \theta} \quad \dots(1)$$

If  $\theta_1$  and  $\theta_2$  are the angles of twist at lengths  $l_1$  and  $l_2$  measured from the fixed end, then

$$\eta = \frac{360mgd(l_1 - l_2)}{\pi^2 r^4 (\theta_1 - \theta_2)} \quad \dots(2)$$

### Procedure

- After levelling the apparatus, the pointers are clamped at three suitable distances from the fixed end of the rod adjusting their positions near about zero mark on the circular scales. The initial readings of the pointers at their two ends are taken with no load on the pans.
- The pulleys are adjusted at the same height. Next, a load of 500 gm (say)\* is placed on each pan. Again the readings of the pointers on both the ends for different lengths are recorded.\*\*
- The load is gradually increased on each pan on steps of 500 gm up to 3500 gm taking down the readings of the pointers at each step. The procedure is repeated in the same stages with decreasing loads till all the loads are removed.
- The three lengths of the wire upto the pointers from its fixed end are noted.
- The diameter of the wire is measured with a screw gauge and that of the cylinder with a vernier callipers.
- Graphs are plotted between the load and the twist for each value of length of the wire. The formula is then applied to determine the modulus of rigidity of the given rod.

\*The exact magnitude of loads is so selected that the depression of the beam caused by it is appreciable and conveniently measurable.

\*\*The percentage error will be small if the values of  $(l_1 - l_2)$  and  $(\theta_1 - \theta_2)$  are large. This is possible if the entire length of the wire is considered and reading only on one of the circular scales is taken by keeping it at the bottom. In this case, formula (1) is to be used.

