

## EXAMPLE Indian IS 456-2000 Wall-001

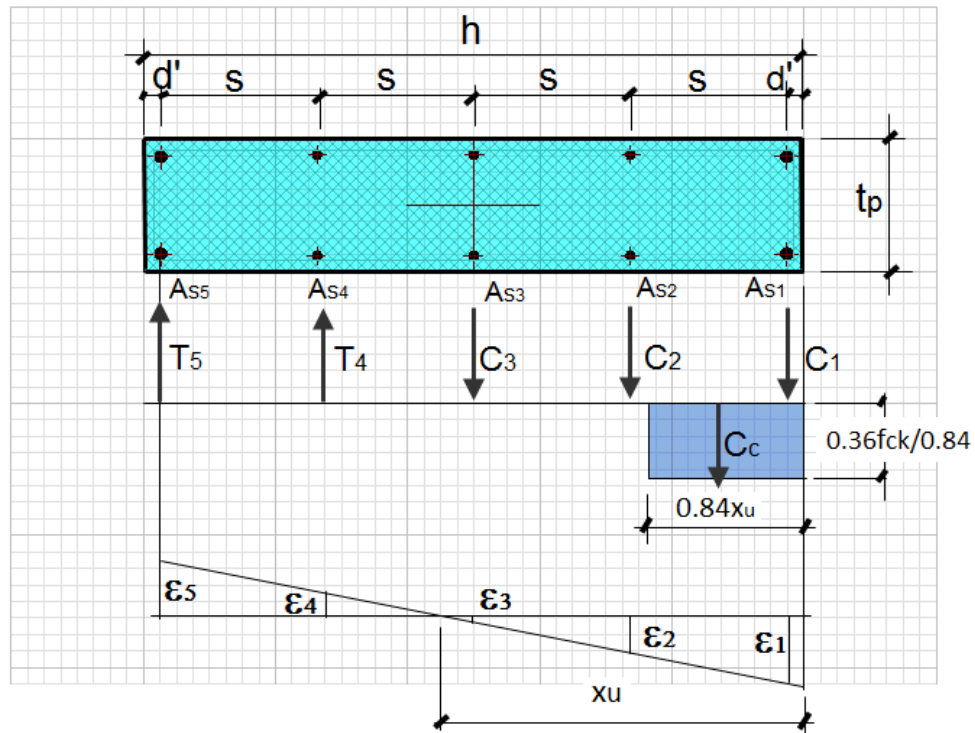
### P-M INTERACTION CHECK FOR A WALL

#### EXAMPLE DESCRIPTION

The Demand/Capacity ratio for a given axial load and moment are tested in this example.

A reinforced concrete wall is subjected factored axial load  $P_u = 3146$  kN and moments  $M_{uy} = 1875$  kN-m. This wall is reinforced with two 30M bars at each end and 15M bars at 350 mm on center of each face. The total area of reinforcement is  $4000 \text{ mm}^2$ . The design capacity ratio is checked by hand calculations and the results are compared with ETABS program results.

#### GEOMETRY, PROPERTIES AND LOADING



PROGRAM NAME: ETABS  
 REVISION NO.: 0

## Material Properties

E = 25000 MPa  
 v = 0.2

## Section Properties

tb = 300 mm  
 h = 1500 mm  
 d' = 50 mm  
 s = 350 mm  
 As1= As5 = 2-30M (1400 mm<sup>2</sup>)  
 As2, As3, As4 = 2-15M (400 mm<sup>2</sup>)

## Design Properties

$f'_c = 30$  MPa  
 $f_y = 460$  MPa

## TECHNICAL FEATURES OF ETABS TESTED

- Concrete wall flexural demand/capacity ratio

## RESULTS COMPARISON

Independent results are hand calculated and compared with ETABS design check.

Output Parameter	ETABS	Independent	Percent Difference
Wall Demand/Capacity Ratio	1.035	1.00	3.50%

## COMPUTER FILE: INDIAN IS 456-2000 WALL-001

## CONCLUSION

The ETABS results show an acceptable match with the independent results.

## HAND CALCULATION

### Wall Strength Determined as follows:

$$\begin{array}{ll} F'_c = 30\text{MPa} & f_y = 460\text{ MPa} \\ b = 300\text{mm} & h = 1500\text{ mm} \end{array}$$

- 1) A value of  $e = 596\text{ mm}$  was determined using  $e = M_u / P_u$  where  $M_u$  and  $P_u$  were taken from the ETABS test model PMM interaction diagram for the pier, P1. Values for  $M_u$  and  $P_u$  were taken near the balanced condition and large enough to produce a flexural D/C ratio very close to or equal to one. The depth to the neutral axis,  $c$ , was determined by iteration using an excel spreadsheet so that equations 1 and 2 below were equal.
- 2) From the equation of equilibrium:

$$P_n = C_c + C_s - T$$

where

$$C_c = \frac{0.36}{0.84} f_{ck} ab = 0.4286 \bullet 30 \bullet 300a = 3857a, \text{ where } a = 0.84x_u$$

$$C_s = \frac{A'_{s1}}{\gamma_s} (f_{s1} - 0.4286f'_c) + \frac{A'_{s2}}{\gamma_s} (f_{s2} - 0.4286f'_c) + \frac{A'_{s3}}{\gamma_s} (f_{s3} - 0.4286f'_c)$$

$$T = \frac{A_{s4}}{\gamma_s} f_{s4} + \frac{A_{s5}}{\gamma_s} f_{s5}$$

$$\begin{aligned} P_{n1} = 3857a + \frac{A'_{s1}}{\gamma_s} (f_{s1} - 0.4286f'_c) + \frac{A'_{s2}}{\gamma_s} (f_{s2} - 0.4286f'_c) + \\ \frac{A'_{s3}}{\gamma_s} (f_{s3} - 0.4286f'_c) - \frac{A_{s4}}{\gamma_s} f_{s4} + \frac{A_{s5}}{\gamma_s} f_{s5} \end{aligned} \quad (\text{Eqn. 1})$$

- 3) Taking moments about  $A_{s5}$ :

$$P_{n2} = \frac{1}{e'} \left[ C_c \left( d - \frac{a}{2} \right) + C_{s1} (d - d') + C_{s2} (d - d' - s) + C_{s3} (2s) - T_{s4} (s) \right] \quad (\text{Eqn. 2})$$

where  $C_{s1} = \frac{A_{s1}}{\gamma_s}(f_{s1} - 0.4286f'_c)$ ;  $C_{s2} = \frac{A_{s2}}{\gamma_s}(f_{s2} - 0.4286f'_c)$ ;  
 $C_{s3} = \frac{A_{s3}}{\gamma_s}(f_{s3} - 0.4286f'_c)$ ;  $T_{s4} = \frac{A_{s4}}{\gamma_s}(f_{s4})$  and the bar strains and stresses are determined below.

The plastic centroid is at the center of the section and  $d'' = 700$  mm  
 $e' = e + d'' = 596 + 700 = 1296$  mm.

- 4) Using  $c = 917.3$  mm (from iteration)

$$a = \beta_1 c = 0.84 \cdot 917.3 = 770.5 \text{ mm}$$

- 5) Assuming the extreme fiber strain equals 0.0035 and  $c = 917.3$  mm, the steel stresses and strains can be calculated. When the bar strain exceeds the yield strain, then

$$f_s = f_y :$$

$$\varepsilon_{s1} = \left( \frac{c - d'}{c} \right) 0.0035 = 0.00331; f_s = \varepsilon_s E \leq F_y ; f_{s1} = 460 \text{ MPa}$$

$$\varepsilon_{s2} = \left( \frac{c - s - d'}{c} \right) 0.0035 = 0.00197 \quad f_{s2} = 394.8 \text{ MPa}$$

$$\varepsilon_{s3} = \left( \frac{c - 2s - d'}{c} \right) 0.0035 = 0.00064 \quad f_{s3} = 127.7 \text{ MPa}$$

$$\varepsilon_{s4} = \left( \frac{d - c - s}{d - c} \right) \varepsilon_{s5} = 0.00070 \quad f_{s4} = 139.4 \text{ MPa}$$

$$\varepsilon_{s5} = \left( \frac{d - c}{c} \right) 0.0035 = 0.00203 \quad f_{s5} = 406.5 \text{ MPa}$$

Substituting in Eqn. 1 and 2 and iterating the value of the neutral axis depth until the two equations are equal give

$$P_{n1} = 3146 \text{ kN}$$

$$P_{n2} = 3146 \text{ kN}$$

$$M_n = P_n e = 3146(596) / 1000 = 1875 \text{ kN-m}$$