

过程设备机械设计基础

----平面弯曲

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学习资料及论坛: www.chenjj.org

主要内容

1. 平面弯曲的概念及实例

2. 平面弯曲梁的内力分析

3. 平面弯曲时梁的正应力

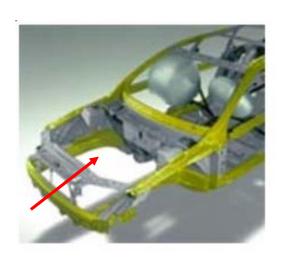
4. 平面弯曲时梁的变形

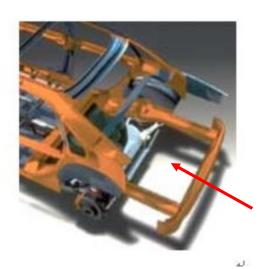
- □ 汽车工业的飞速发展使道路日益拥挤,发生碰撞意外 几率的增加。
- □一旦发生碰撞, 你认为车身的变形是大好, 还是小好?





为了在发生碰撞更好地保护车内乘客的安全,轿车车身的前后均应设计变形区,或者称为吸能区。以便保证在发生碰撞时,轿车车身的变形能够按照预先设计的方向逐渐变形直至停车,从而尽量减小传递到乘客舱和乘客身体的冲击,减小乘客舱的变形,保障车内乘客安全。





Volvo 某汽车车身前后吸能区设计

起重机大梁



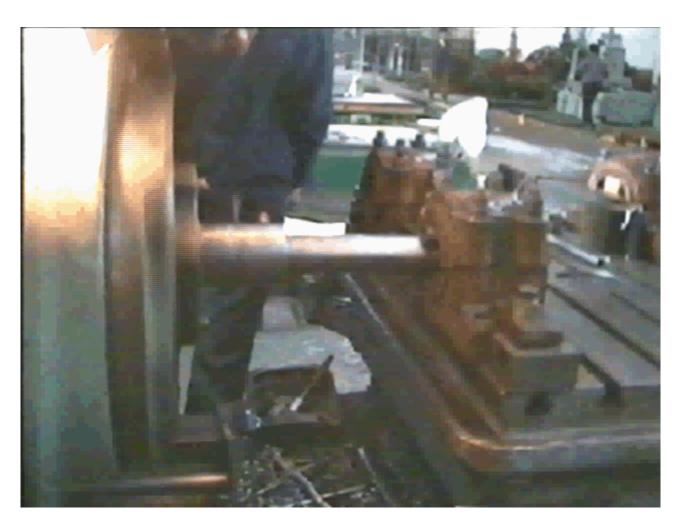
车削工件



火车轮轴

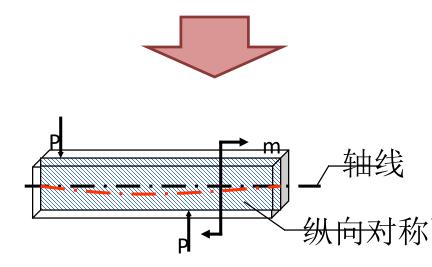


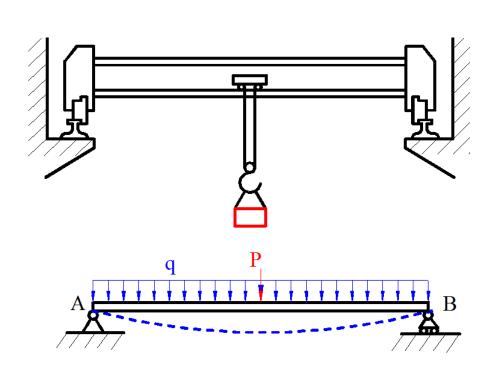
镗刀杆

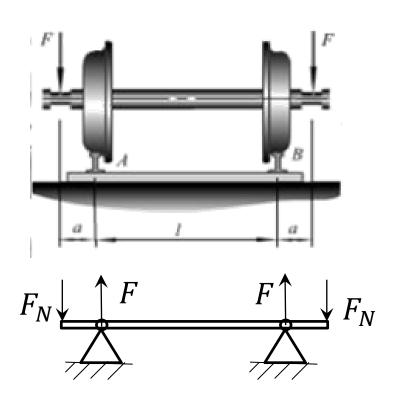


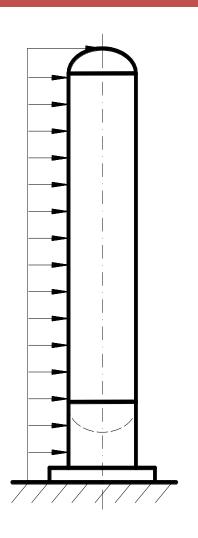
承受垂直于其轴线的外力,或在 其轴线平面内作用有外力偶矩。 受力后直的轴线变成了曲线,这 种变形称为**弯曲变形**。工程上把 以弯曲为主的构件称为梁。

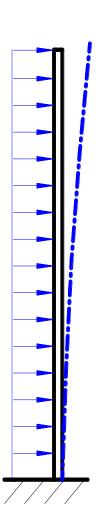
什么是弯曲变形?

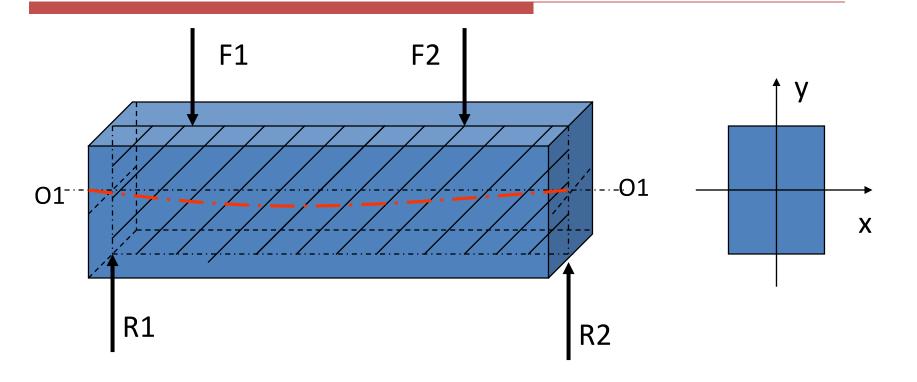










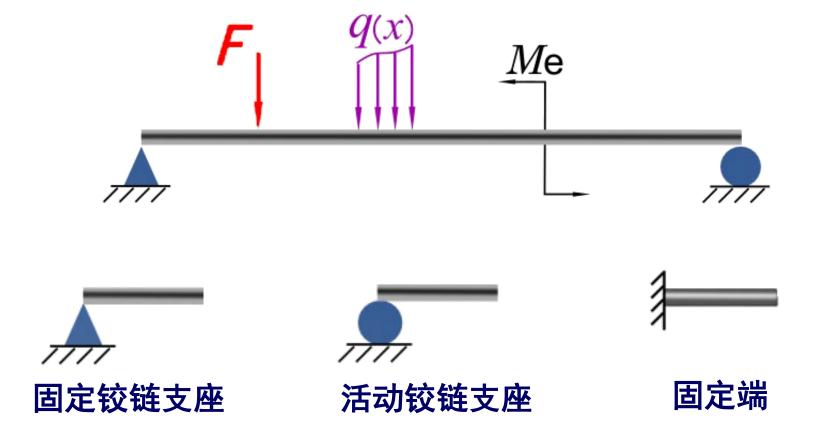


梁: 以弯曲变形(横向力)为主的杆件

纵向对称面: 对称轴与轴线组成的平面

平面弯曲: 梁轴线弯曲成此平面内的一条曲线

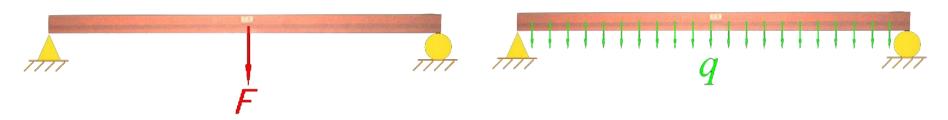
• 载荷的简化: 集中载荷、分布载荷、集中力偶



吊车大梁简化

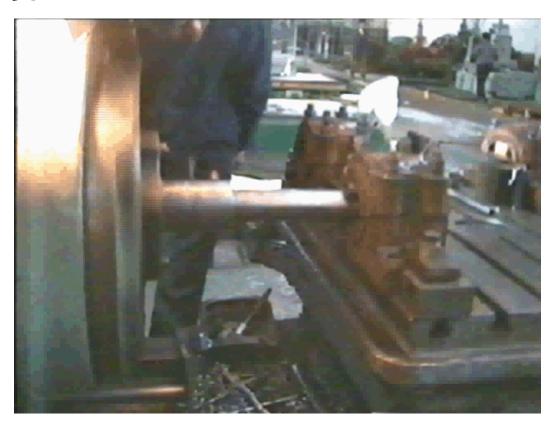






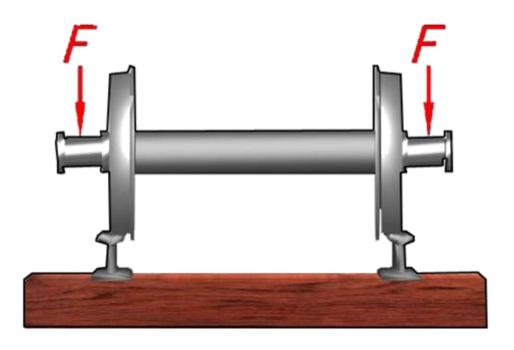
均匀分布载荷 简称均布载荷

镗刀杆的简化



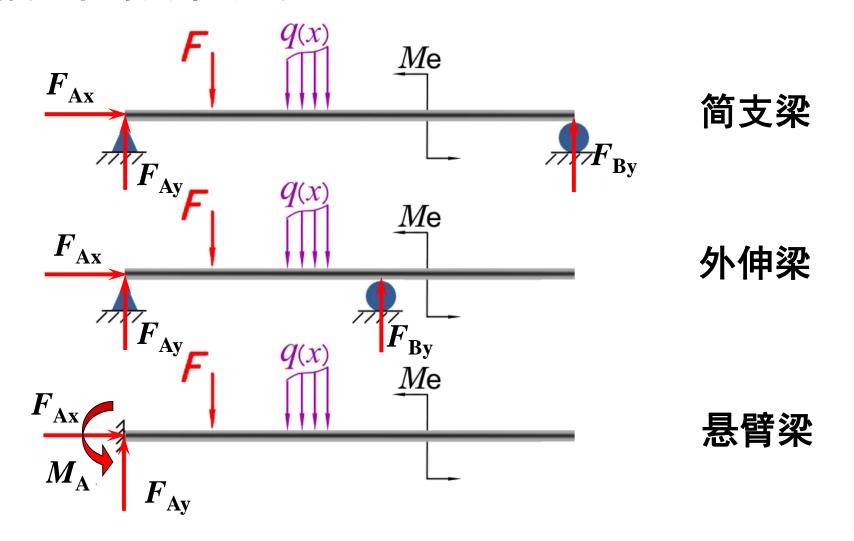


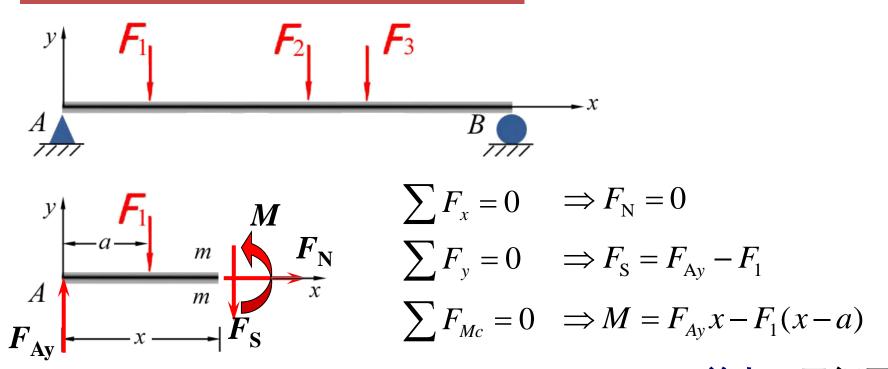
火车轮轴简化

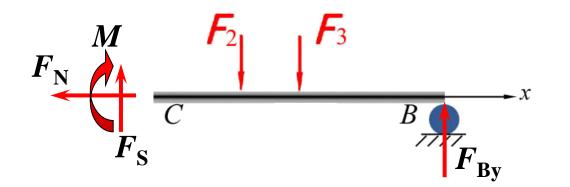




静定梁的基本形式







F_s剪力,平行于 横截面的内力合力

∦ 弯矩,垂直于 横截面的内力系的 合力偶矩

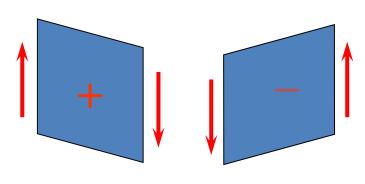
• 剪力方程和弯矩方程--剪力图和弯矩图

• 剪力和弯矩随横截面的位置而变化:

• 以x为横坐标、Q和M为纵坐标作图即得剪力图和弯矩图。

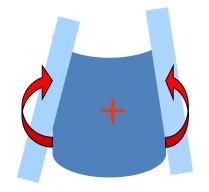
• 剪力和弯矩的符号规定

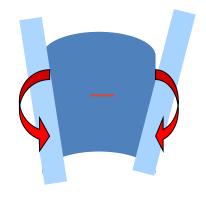
截面上的剪力对梁上任意 一点的矩为顺时针转向时, 剪力为正;反之为负。



左上右下为正; 反之为负

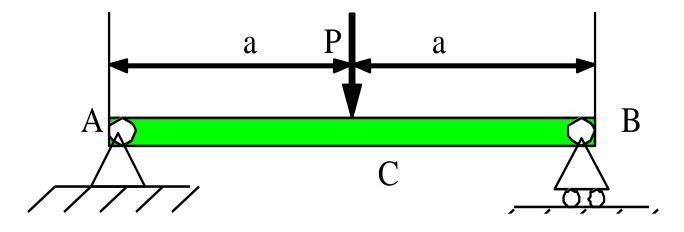
截面上的弯矩 使得梁呈凹形为正; 反之为负。





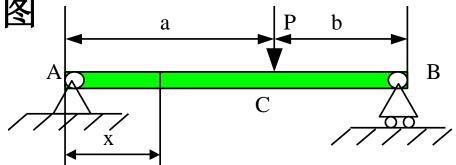
左顺右逆为正; 反之为负

例: 画简支梁的剪力图和弯矩图(Q-M图)



例:求简支梁的剪力和弯矩图

1、求反力:



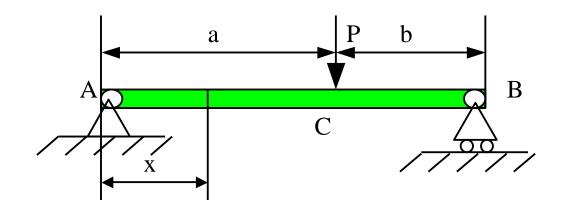
2、求剪力和弯矩方程

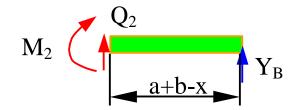
以左段作研究对象

AC段 (0<x<a)

$$\begin{cases} Q_1(x) = Y_A = \frac{Pb}{a+b} \\ M_1(x) = Y_A x = \frac{Pb}{a+b} x \end{cases}$$

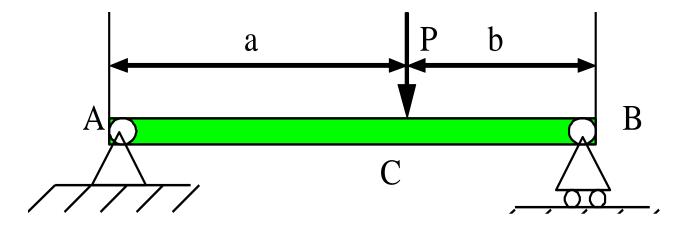
BC段(a<x<(a+b))



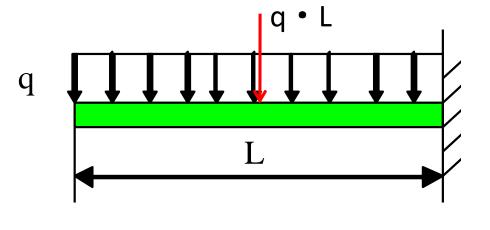


$$\begin{cases}
Q_{2}(x) = -Y_{B} = -\frac{Pa}{a+b} \\
M_{2}(x) = Y_{A}x - P(x-a) = Y_{B}(a+b-x) = Pa - \frac{Pa}{a+b}x
\end{cases}$$

3、画简支梁的剪力图和弯矩图(Q-M图)



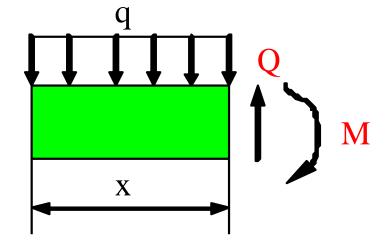
例求悬臂梁的剪力图和弯矩图





$$Q(x)=-qx$$

$$(0< x< L)$$

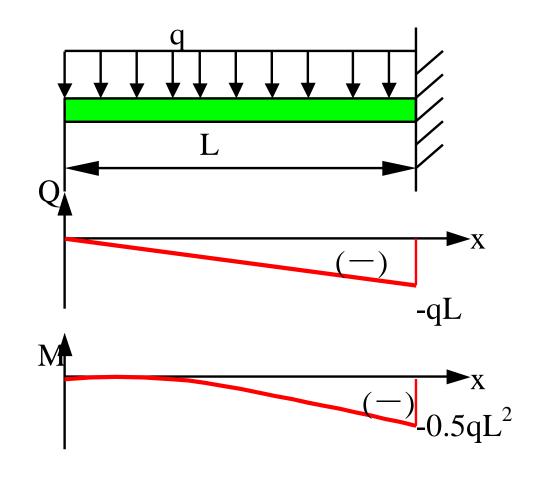


弯矩方程:

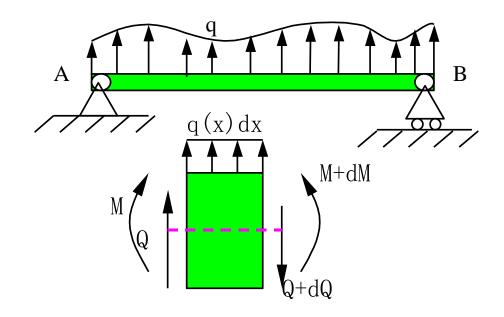
$$M(x) = -qx^2 / 2$$

(0

悬臂梁的剪力弯矩图(Q-M图)



• 剪力、弯矩和载荷集度间的微分关系



$$\frac{\mathrm{d}Q(x)}{\mathrm{d}x} = q(x)$$

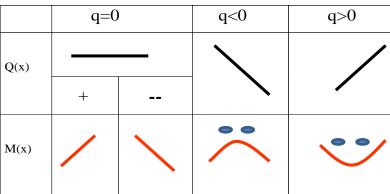
 $\frac{\mathrm{dM}(x)}{\mathrm{d}x} = Q(x)$

剪力图上任一点的切线斜率等于梁上相 应点处的载荷集度q

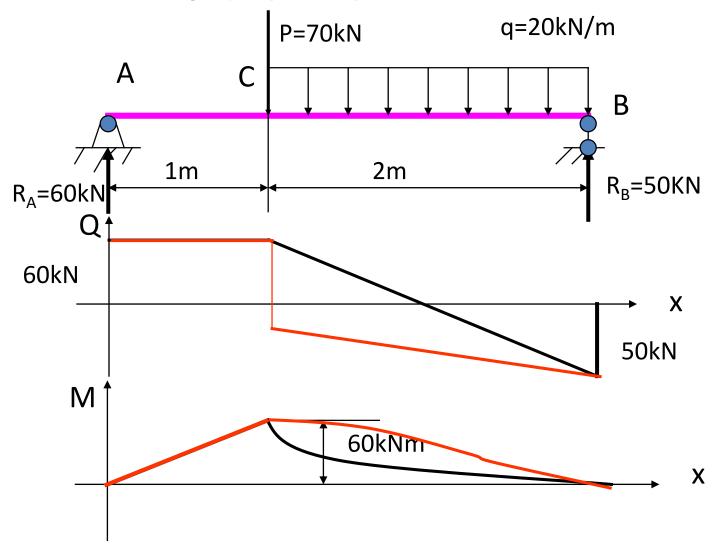
弯矩图上任一点的切线斜率等于梁相应 横截面上的剪力Q

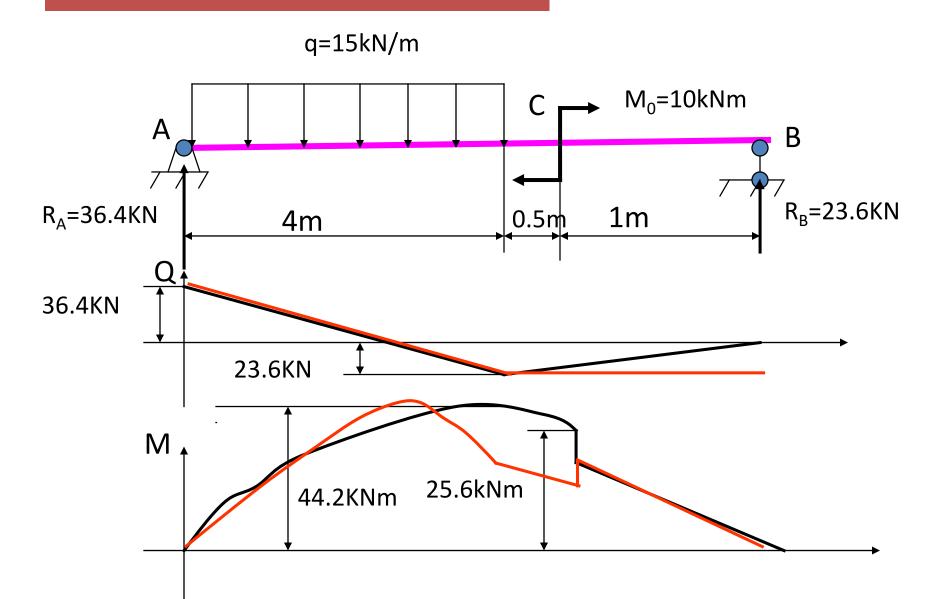
• Q-M图的规律

- 1. 梁上某段无分布力,Q为水平线,M为斜直线
- 有向下的分布力,Q图递减(↘),M为上凸(△) 有向上的分布力,Q图递增(↗),M为下凹(∪) 如分布力均匀,Q为斜直线,M为二次抛物线
- 3. 在集中力作用处,Q图有突变,M图有折角 在集中力偶处,弯矩图有突变
- 4. 某截面Q=0,则弯矩为极值。



例:判断Q、M图是否有错?



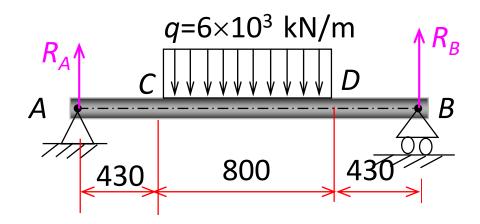


例:一受均匀载荷作用的简支梁(单位mm)。

$$R_A = R_B = 2400kN$$

$$Q_C = 2400kN$$

$$Q_D = -2400kN$$

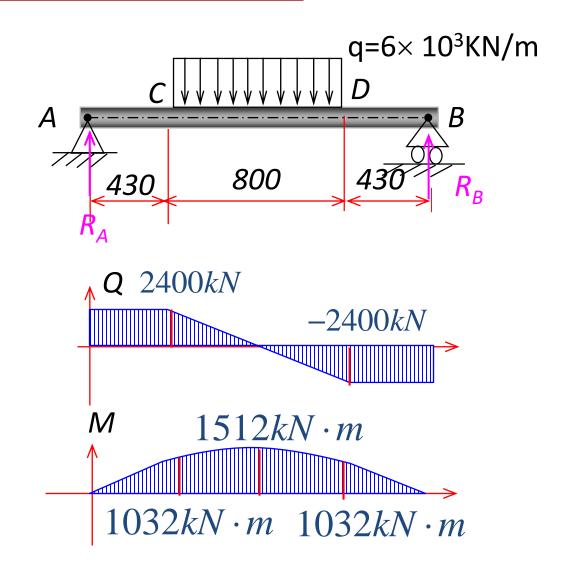


$$M_C = 2400 \times 430 = 1032kN \cdot m$$

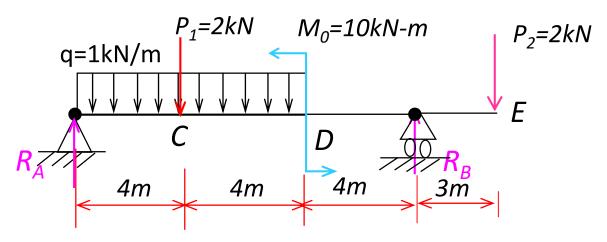
$$M_D = 2400 \times 430 = 1032kN \cdot m$$

$$M_{\text{max}} = 2400 \times (0.43 + 0.4) - 6 \times 10^3 \times 0.4 \times \frac{0.4}{2}$$

= 1512kN · m



例:一外伸梁如图,画其剪力和弯矩图。

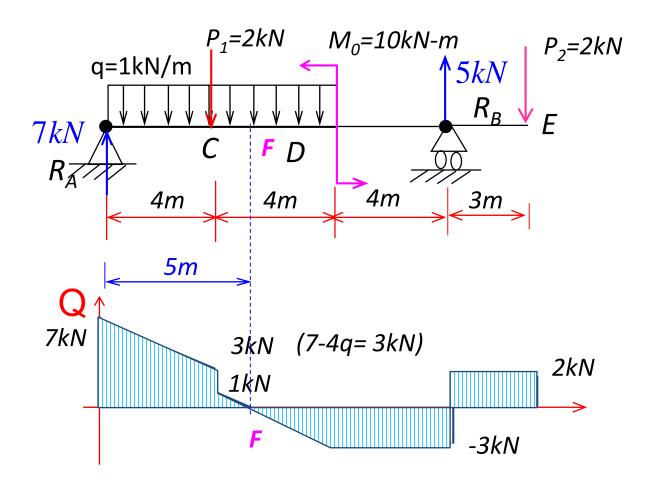


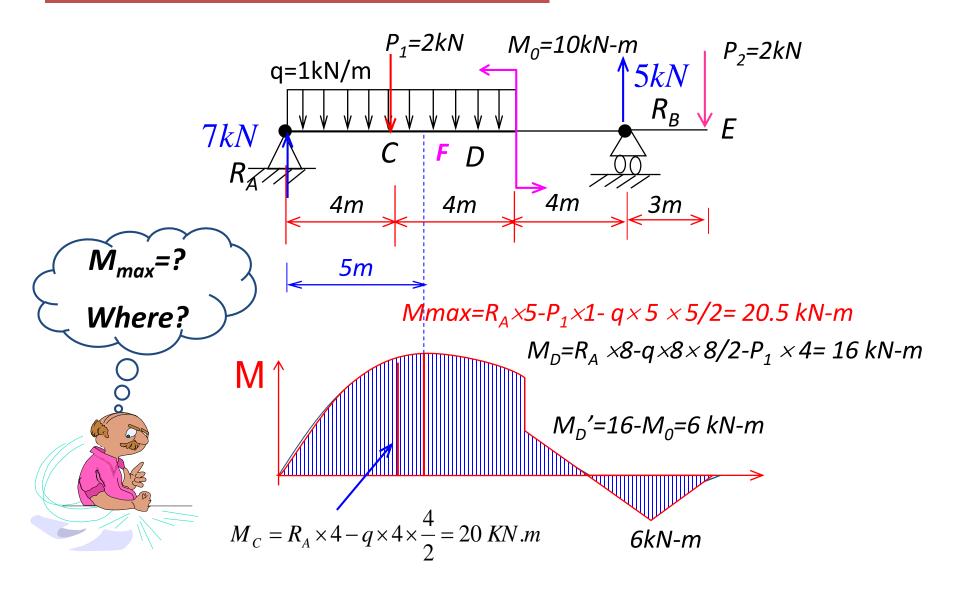
按静时倒獲到程勢表別成力为:

$$\sum_{A} M_{A} = \sum_{B} F_{Y} = 0$$

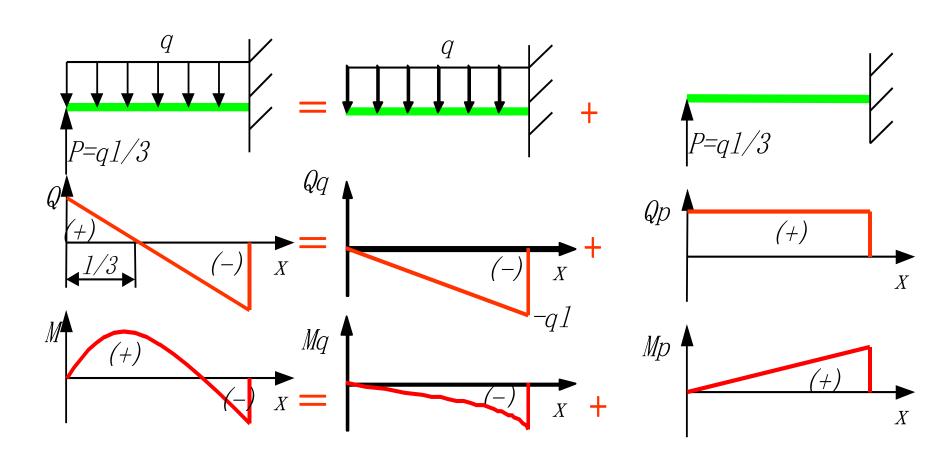
$$-P_{2} \times 15 R_{A} R_{B} \times 128 M_{P_{1}} P_{1} R_{A}^{4} = q_{P_{2}} 8 \times \frac{8}{2} = 0$$

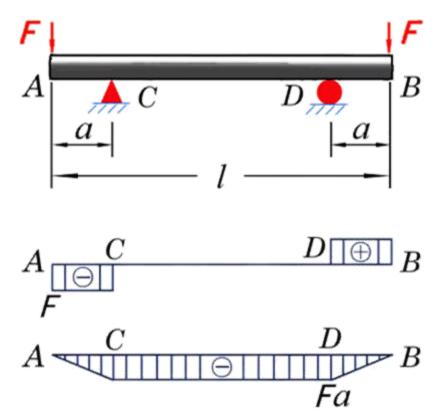
$$R_{B} = 5k R_{A} = 7kN$$





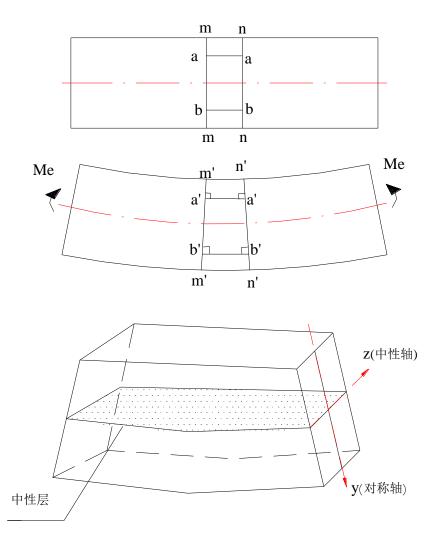
· 叠加法作Q - M图







梁段CD上,只有弯矩,没有剪力——<mark>纯弯曲</mark> 梁段AC和BD上,既有弯矩,又有剪力——横力弯曲



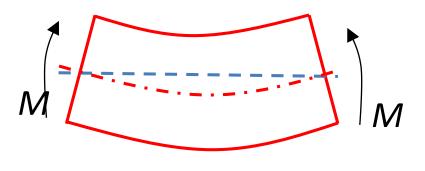
- 横向线仍为直线,只是转动一角 度;
- 纵向线变成弧线,仍垂直变形后横向线;
- 线段bb'伸长, aa'缩短。

平面假设: 横截面变形后仍为平面, 相邻横截面作相对转动,并仍与弯曲 后轴线正交。

中性层:一弯曲面,不伸长,不缩短。

中性轴:中性层和横截面交线。

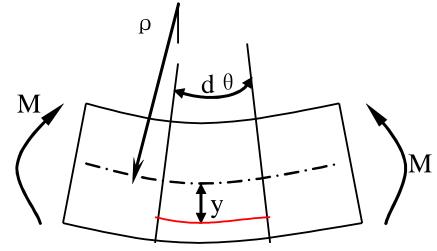
刚性平面假设:梁在纯弯曲变形后,其横截面仍然保持平面,并与 变形后的梁轴线垂直,只是绕截面内某一轴旋转了一个角度。





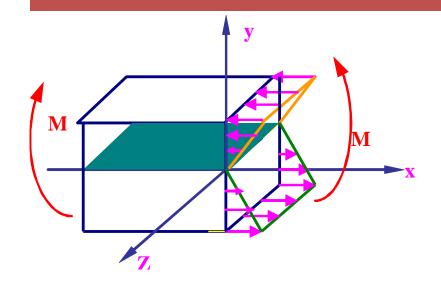
变形几何方程:

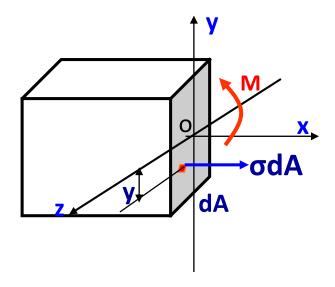
$$\varepsilon = \frac{(\rho + y)d\theta - \rho d\theta}{\rho d\theta} = \frac{y}{\rho}$$



应力应变方程:

$$\sigma = E\varepsilon = E\frac{y}{\rho}$$







$$\sum M_z = \int_A y \sigma dA = M$$

$$\sigma = E\varepsilon = E\frac{y}{\rho}$$

$$M = \frac{E}{\rho} \int_{A} y^{2} dA$$



$$I_z = \int y^2 dA$$

(横截面对中性轴z的 惯性矩)

$$\begin{cases}
\frac{1}{\rho} = \frac{M}{EI_Z} \\
\sigma = \frac{M}{I_Z}
\end{cases}$$

横截面上的最大正应力

$$\sigma_{\text{max}} = \frac{My_{\text{max}}}{I_z}$$

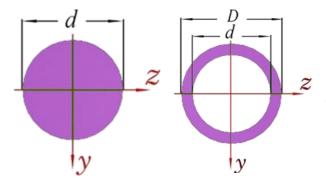
定义:
$$W_z = \frac{I_z}{y_{\text{max}}}$$

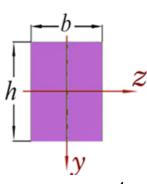
截面的抗弯截面模量

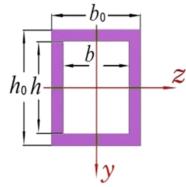
横截面上最大的正应力 $\sigma_{\text{max}} = \frac{M_{\text{max}}}{W}$

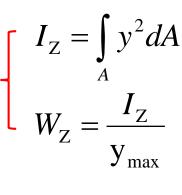
$$\sigma_{\max} = \frac{M_{\max}}{W_Z}$$

• 常见截面的 4和 11/2









圆截面

$$I_{\rm Z} = \frac{\pi d^4}{64}$$

$$W_{\rm Z} = \frac{\pi d^3}{32}$$

空心圆截面

$$I_{\rm Z} = \frac{\pi}{64} (D^4 - d^4)$$

$$I_{\rm Z} = \frac{\pi}{64} (D^4 - d^4)$$
 $W_{\rm Z} = \frac{\pi}{32D} (D^3 - d^4)$

矩形截面

$$I_{\rm Z} = \frac{bh^3}{12}$$

$$W_{\rm Z} = \frac{bh^2}{6}$$

空心矩形截面

$$I_{\rm Z} = \frac{b_0 h_0^3}{12} - \frac{bh^3}{12}$$

$$I_{\rm Z} = \frac{b_0 h_0^3}{12} - \frac{bh^3}{12}$$
 $W_{\rm Z} = (\frac{b_0 h_0^3}{12} - \frac{bh^3}{12})/(h_0/2)$

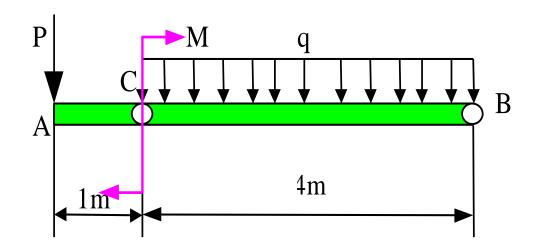
• 梁弯曲时正应力强度条件

$$\sigma_{\max} = \frac{M_{\max}}{W_Z} \leq [\sigma]$$

根据这一确定条件可进行三项工作:

- 1设计截面
- 2 强度校核
- 3 计算许可载荷

例: 某梁由工字钢制成,材料为Q235A.F, [σ]=160MPa, P=20KN, q=10KN/m, M=40KN·m, 试确定工字钢的型号。



STEP 1: 画受力图,求支座反力

$$\sum M_{c} = 0$$
 $20 \times 1 - 40 + Y_{B} \times 4 - 10 \times 4 \times 2 = 0$

$$Y_B = 25KN$$

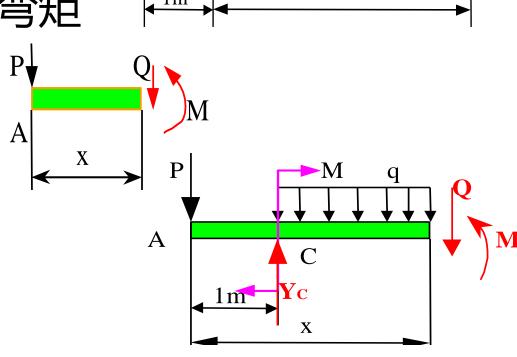
$$\sum M_B = 0$$
 $20 \times 5 - 40 - Y_c \times 4 + 10 \times 4 \times 2 = 0$

$$Y_C = 35KN$$

STEP2: 求出剪力和弯矩

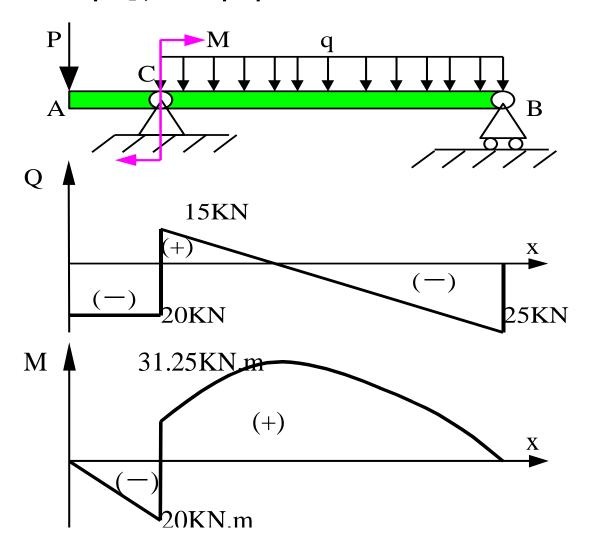
$$\begin{cases} Q_{AC} = -20 \text{ KN} \\ M_{AC} = -20x \text{ KN} \cdot \text{m} \end{cases}$$

CB段: (1<x<5)



$$\begin{cases} Q_{BC} = 35 - 20 - 10(x - 1) = 25 - 10x & \text{KN} \\ M_{BC} = 35(x - 1) + 40 - 20x - \frac{10(x - 1)^2}{2} & \text{KN} \cdot \text{m} \end{cases}$$

STEP 3: 画Q、M图



STEP 4: 求最大W_Z

$$W_Z \ge \frac{M_{\text{max}}}{[\sigma]} = \frac{31.25}{160} = 195 \times 10^3 \, \text{mm}^3$$

查表P293,应选用20a工字钢,W_z=237×10³ mm³

• 提高梁抗弯曲的主要措施

$$oldsymbol{\sigma}_{ ext{max}} = rac{oldsymbol{M}_{ ext{max}}}{oldsymbol{W}_{Z}}$$

• 选择合理截面,增加抗弯模量

• 合理布置支座和载荷作用位置(减少M_{max}

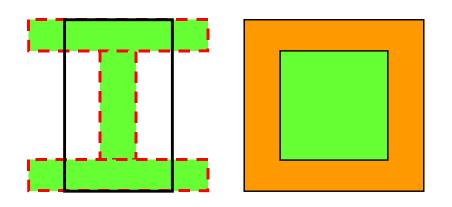
• 分散载荷

• 采用变截面梁

4

• 选择合理截面,增加抗弯模量

以比值 W_Z/A 来衡量截面的合理 程度,该值越大,截面越经济合理。

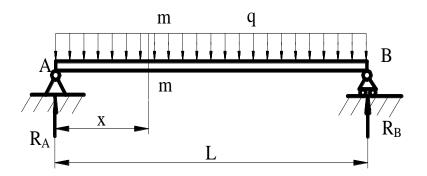


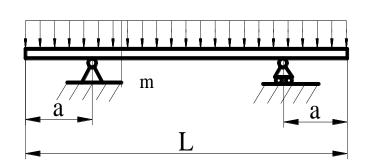
矩形截面:
$$\frac{W_z}{A} = \frac{bh^2/6}{bh} = \frac{h}{6} = 0.167h$$

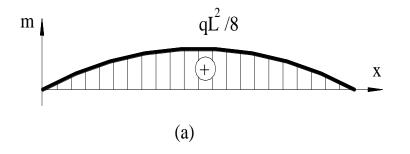
圆形截面:
$$\frac{W_z}{A} = \frac{\pi d^3/32}{\pi d^2/4} = \frac{d}{8} = 0.125d$$

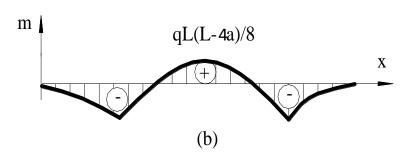
工字形截面:
$$\frac{W_z}{A} = (0.27 - 0.31)h$$

· 合理布置支座和载荷作用位置(减少M_{max})

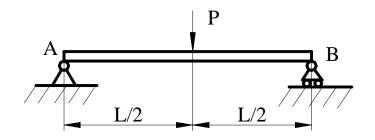


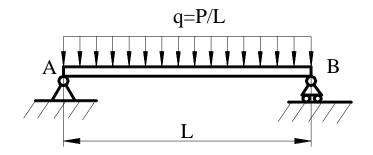


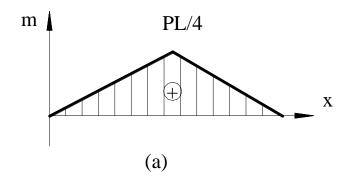


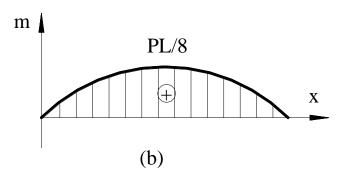


• 分散载荷



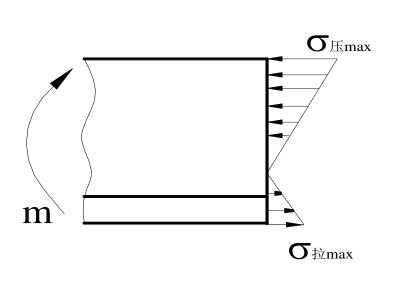


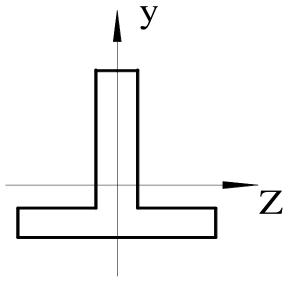




• 采用变截面梁

材料性能抗拉和抗压不同时,应采用不对称的中性轴,如铸铁:

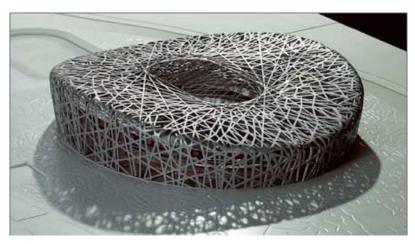




$$\frac{\sigma_{\underline{\mathbb{E}}_{max}}}{\sigma_{\underline{\mathbb{E}}_{max}}} = \frac{\frac{My_1}{I_z}}{\frac{My_2}{I_z}} = \frac{\left[\sigma_{\underline{\mathbb{E}}}\right]}{\left[\sigma_{\underline{\mathbb{E}}}\right]}$$

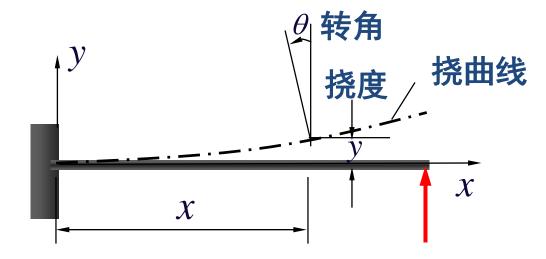
• 屋顶的设计必须考虑弯曲变形







弯曲变形的挠度和转角



挠曲线方程:

$$y = y(x)$$

挠度y: 截面形心 在y方向的位移

y 向上为正

转角 θ :截面绕中性轴转过的角度。 θ 逆向为正由于小变形,截面形心在x方向的位移忽略不计

挠度转角关系为: $\theta \approx \tan \theta = \frac{dy}{dx}$

挠曲线近似微分方程

• 挠曲线可用y=f(x)表示,则转角

$$tg\theta = f'(x) \approx \theta$$

• 挠曲线曲率: $\frac{1}{\rho(x)} = \frac{M(x)}{EI}$

• 数学定义曲率:
$$\frac{1}{\rho(x)} = \pm \frac{y''}{1+(y')^2} \approx \pm y''$$

• 综合两式, 得挠曲线近似微分方程:

$$\pm y'' = \pm \frac{d^2y}{d^2x} \approx \frac{M(x)}{EI}$$

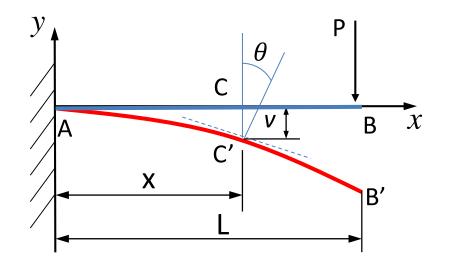
• 积分一次得转角方程:

$$EI\theta = \int M(x)dx + C$$

• 再积分得挠度方程:

$$Ely=\int M(x)dx.dx+Cx+D$$

• 积分法求梁的变形



1. 列出弯矩方程

$$M(x) = -P(L-x)$$

2. 建立挠曲线微分方程

$$EIy'' = -P(L-x)$$

两次积分:

$$EIy' = -PLx + \frac{1}{2}P\chi^{2} + C$$

$$EIy = -\frac{1}{2}PL\chi^{2} + \frac{1}{6}P\chi^{3} + Cx + D$$



当x=0时,y'=0,因此C=0 当x=0时,y=0,因此D=0

3. 确定转角和挠度方程

• 转角方程:

$$EIy' = -PLx + \frac{1}{2}P\chi^2$$

• 挠度方程:

$$EIy = -\frac{1}{2}PL\chi^2 + \frac{1}{6}P\chi^3$$

4. 确定最大转角和最大挠度

• 发生在x=L处

$$\theta_{\text{max}} = \frac{p L^2}{2EI}$$

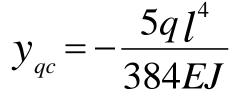
$$y_{\text{max}} = \frac{p L^3}{3EI}$$

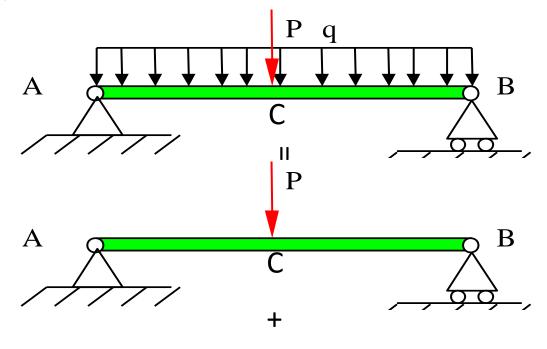
• 叠加法求梁的变形

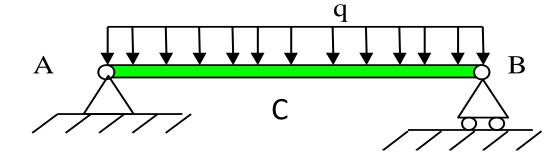
$$y_{max} = y_{pc} + y_{qc}$$

$$y_{pc} = -\frac{p \, l^3}{48EJ}$$









• 刚度条件:

$$\begin{cases} \mathbf{y}_{\text{max}} \leq [\mathbf{y}] \\ \mathbf{\theta}_{\text{max}} \leq [\mathbf{\theta}] \end{cases}$$

- 提高梁抗弯刚度的主要途径
 - 1减少垮度,或增加支座;
 - 2选用合理截面形状。

第四章的重点与要点

- (1)掌握Q、M方程的写法,Q、M图的画法, 注意内力符号
 - (2) 惯矩I_z和抗弯截面模量W_z

(3) 弯曲正应力强度条件

作业

- 4 -1(b,c,f)
- 4-2(a\b)
- 4-7
- 4-8
- 4-15