

传递过程

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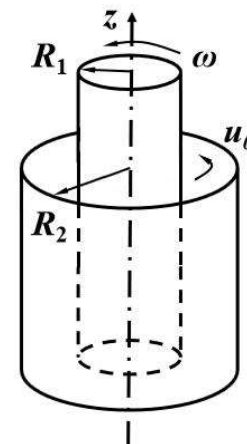
2.2.4 旋转柱面间的流动—旋转粘度计原理

垂直的同心套筒环隙间充满了流体。内筒外径为 R_1 ，外筒内径为 R_2 ，当内筒以角速度 ω 旋转时，环隙间的流体随之旋转。若圆筒足够长，端效应可以忽略。

$$\begin{aligned} \text{定常: } \frac{\partial u_\theta}{\partial t} &= 0 \\ \text{一维流动: } \begin{cases} u_r = 0 \\ u_\theta \neq 0 \\ u_z = 0 \end{cases} &\begin{cases} \frac{\partial u_\theta}{\partial r} \neq 0 \\ \frac{\partial u_\theta}{\partial \theta} = 0 \\ \frac{\partial u_\theta}{\partial z} = 0 \end{cases} \begin{cases} \frac{\partial^2 u_\theta}{\partial r^2} = 0 \\ \frac{\partial^2 u_\theta}{\partial \theta^2} = 0 \\ \frac{\partial^2 u_\theta}{\partial z^2} = 0 \end{cases} \end{aligned}$$

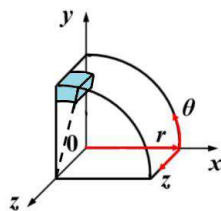
θ 方向无重力: $X_\theta = 0$

θ 方向无压差力: $\frac{\partial p}{\partial \theta} = 0$



回顾:

柱坐标系—奈维-斯托克斯方程



$$\begin{aligned} r \text{ 方向: } & \rho \left(\frac{\partial u_r}{\partial t} + u_r \frac{\partial u_r}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_r}{\partial \theta} - \frac{u_\theta^2}{r} + u_z \frac{\partial u_r}{\partial z} \right) \\ & = -\frac{\partial p}{\partial r} + \rho X_r + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial r u_r}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial u_\theta}{\partial \theta} + \frac{\partial^2 u_r}{\partial z^2} \right] \end{aligned}$$

$$\begin{aligned} \theta \text{ 方向: } & \rho \left(\frac{\partial u_\theta}{\partial t} + u_r \frac{\partial u_\theta}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_r u_\theta}{r} + u_z \frac{\partial u_\theta}{\partial z} \right) \\ & = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \rho X_\theta + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial r u_\theta}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial u_r}{\partial \theta} + \frac{\partial^2 u_\theta}{\partial z^2} \right] \end{aligned}$$

$$\begin{aligned} z \text{ 方向: } & \rho \left(\frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_z}{\partial \theta} + u_z \frac{\partial u_z}{\partial z} \right) \\ & = -\frac{\partial p}{\partial z} + \rho X_z + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \theta^2} + \frac{\partial^2 u_z}{\partial z^2} \right] \end{aligned}$$

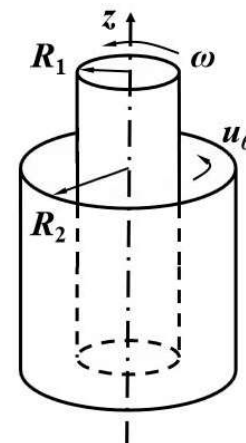
简化柱坐标系中的奈维-斯托克斯方程得:

$$\frac{d}{dr} \left(\frac{1}{r} \frac{dr u_\theta}{dr} \right) = 0$$

$$\text{边界条件: } \begin{cases} r = R_1, u_\theta = \omega R_1 \\ r = R_2, u_\theta = 0 \end{cases}$$

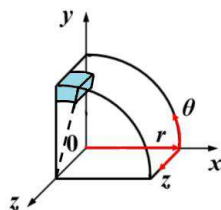
解得速度分布:

$$u_\theta = \frac{\omega R_1^2}{r} \frac{R_2^2 - r^2}{R_2^2 - R_1^2}$$



回顾:

柱坐标系中剪切应力与形变的关系:



$$\begin{aligned}\tau_{r\theta} = \tau_{\theta r} &= -\mu \left[r \frac{\partial}{\partial r} \left(\frac{u_\theta}{r} \right) + \frac{1}{r} \frac{\partial u_r}{\partial \theta} \right] & \tau_{rr} &= -2\mu \frac{\partial u_r}{\partial r} \\ \tau_{\theta z} = \tau_{z\theta} &= -\mu \left(\frac{\partial u_\theta}{\partial z} + \frac{1}{r} \frac{\partial u_z}{\partial \theta} \right) & \tau_{\theta\theta} &= -2\mu \left(\frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_r}{r} \right) \\ \tau_{zr} = \tau_{rz} &= -\mu \left(\frac{\partial u_z}{\partial r} + \frac{\partial u_r}{\partial z} \right) & \tau_{zz} &= -2\mu \frac{\partial u_z}{\partial z}\end{aligned}$$

柱坐标系中剪切应力 $\tau_{r\theta}$ 与形变的关系:

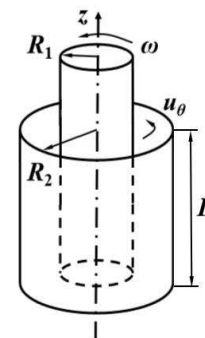
$$\tau_{r\theta} = -\mu \left[r \frac{\partial}{\partial r} \left(\frac{u_\theta}{r} \right) + \frac{1}{r} \frac{\partial u_r}{\partial \theta} \right]$$

因为 $u_r=0$, 所以: $\tau_{r\theta} = -\mu r \frac{\partial}{\partial r} \left(\frac{u_\theta}{r} \right)$

剪切应力分布: $\tau_{r\theta} = \frac{2\mu\omega R_1^2 R_2^2}{R_2^2 - R_1^2} \frac{1}{r^2}$

作用在内筒壁上的摩擦力 F 为:

$$F = \tau_{r\theta} \Big|_{r=R_1} 2\pi R_1 L = \frac{4\pi\mu\omega R_1 R_2^2 L}{R_2^2 - R_1^2}$$

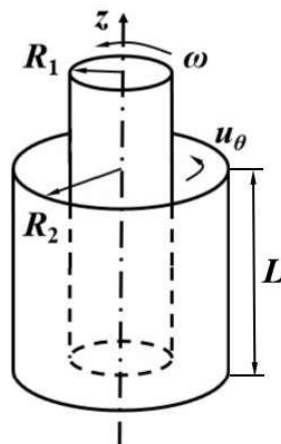


作用在内外筒壁上的力矩相等，即：

$$M_{or} = FR_1 = \frac{4\pi\mu\omega R_1^2 R_2^2 L}{R_2^2 - R_1^2}$$

$$\mu = \frac{M_{or} (R_2^2 - R_1^2)}{4\pi\omega R_1^2 R_2^2 L}$$

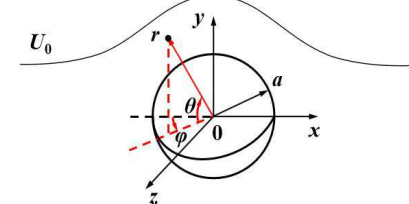
上式是旋转粘度计的测粘度的原理



2.2.5 低雷诺数下绕球爬流—斯托克斯阻力定律

爬流是 $Re < 1$ 的极慢运动 $Re = \frac{\text{惯性力}}{\text{粘性力}} \ll 1$

忽略惯性力，选用球坐标系



定常: $\frac{\partial(\quad)}{\partial t} = 0$

二维流动: $\begin{cases} u_r \neq 0 \\ u_\theta \neq 0 \\ u_\phi = 0 \end{cases} \quad \begin{cases} \frac{\partial(\quad)}{\partial r} \neq 0 \\ \frac{\partial(\quad)}{\partial \theta} \neq 0 \\ \frac{\partial(\quad)}{\partial \phi} = 0 \end{cases}$

忽略重力: $X_r = X_\theta = X_\phi = 0$

简化球坐标系中的连续性方程和奈维-斯托克斯方程，
可得：

连续性方程：

$$\frac{1}{r^2} \frac{\partial r^2 u_r}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial u_\theta \sin \theta}{\partial \theta} = 0$$

奈维-斯托克斯方程 r 方向：

$$\frac{\partial p}{\partial r} = \mu \left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial u_r}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial u_r}{\partial \theta} \right) - \frac{2}{r^2} u_r - \frac{2}{r^2} \frac{\partial u_\theta}{\partial \theta} - \frac{2}{r^2} u_\theta \cot \theta \right]$$

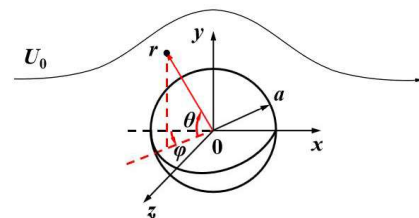
奈维-斯托克斯方程 θ 方向：

$$\frac{1}{r} \frac{\partial p}{\partial \theta} = \mu \left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial u_\theta}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial u_\theta}{\partial \theta} \right) + \frac{2}{r^2} \frac{\partial u_r}{\partial \theta} - \frac{u_\theta}{r^2 \sin^2 \theta} \right]$$

3个方程，3个未知量 u_r, u_θ, p ，可求出解析解

边界条件：

$$\begin{cases} r = a, u_r = 0, u_\theta = 0 \\ r \rightarrow \infty, u_r = U_0 \cos \theta, u_\theta = -U_0 \sin \theta, p = p_0 \end{cases}$$

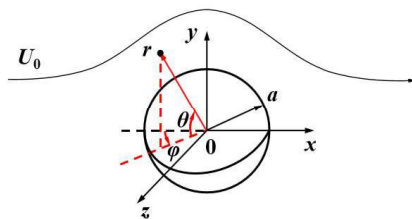


方程组为线性偏微分方程组，可以用分离变量
法解得速度分布和压力分布为：

$$\begin{cases} u_r = U_0 \left[1 - \frac{3a}{2r} + \frac{1}{2} \left(\frac{a}{r} \right)^3 \right] \cos \theta \\ u_\theta = -U_0 \left[1 - \frac{3a}{4r} - \frac{1}{4} \left(\frac{a}{r} \right)^3 \right] \sin \theta \\ p = p_0 - \frac{3}{2} \frac{\mu U_0}{a} \left(\frac{a}{r} \right)^2 \cos \theta \end{cases}$$

球表面压力分布：

$$r = a \quad p = p_0 - \frac{3}{2} \frac{\mu U_0}{a} \cos \theta$$



剪切应力分布：

$$\tau_{r\theta} = \mu \left[r \frac{\partial}{\partial r} \left(\frac{u_\theta}{r} \right) + \frac{1}{r} \frac{\partial u_r}{\partial \theta} \right]$$

代入 u_r, u_θ 得

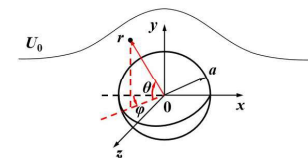
$$\tau_{r\theta} = -\frac{3}{2} \mu U_0 \frac{a^3}{r^4} \sin \theta$$

球表面剪切应力分布：

$$r = a \quad \tau_{r\theta} = -\frac{3}{2a} \mu U_0 \sin \theta$$

球表面总阻力：

$$D = \int_0^{2\pi} d\varphi \int_0^\pi (-p \cos \theta - \tau_{r\theta} \sin \theta) a^2 \sin \theta d\theta$$



$$D = 2\pi\mu a U_0 + 4\pi\mu a U_0$$

压差阻力 摩擦阻力

$$D = 6\pi\mu a U_0$$

斯托克斯阻力定律

适用条件 $Re < 1$ 的爬流

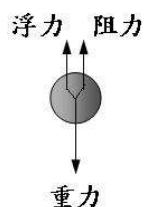
例2-5 落球法测粘度

解：测定小球在静止流体中匀速下降速度 u ,根据力平衡有：

$$\frac{1}{6}\pi d^3 \rho_s g = \frac{1}{6}\pi d^3 \rho g + 6\pi\mu \frac{d}{2} u$$

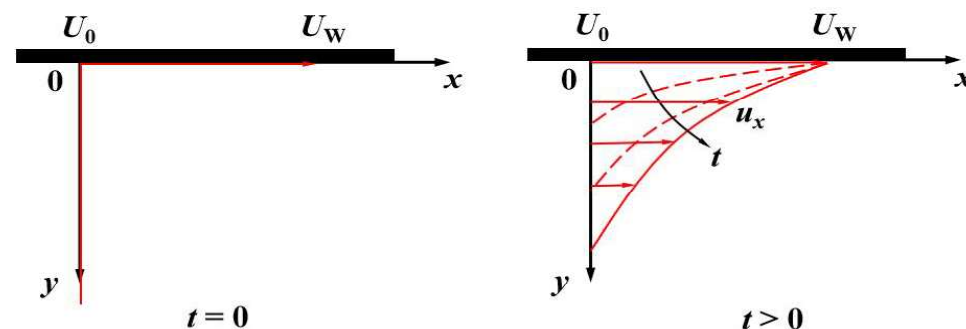
$$\mu = \frac{(\rho_s - \rho)gd^2}{18u}$$

适用条件 $Re < 1$ 的爬流



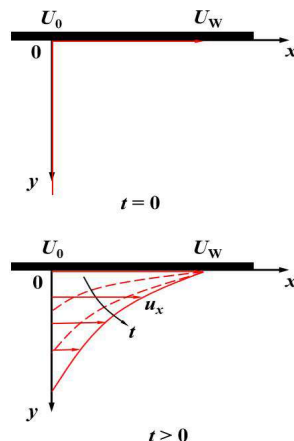
2.2.6 静止流体中的平板启动

静止的水面上有一块无限大平板，初始速度为 $U_0=0$ ，突然以 U_w 速度运动，并维持不变。平板下水中的速度分布 u_x 随时间也发生变化。



非定常: $\frac{\partial u_x}{\partial t} \neq 0$

一维流动: $\begin{cases} u_x \neq 0 \\ u_y = 0 \\ u_z = 0 \end{cases} \begin{cases} \frac{\partial u_x}{\partial x} = 0 \\ \frac{\partial u_x}{\partial y} \neq 0 \\ \frac{\partial u_x}{\partial z} = 0 \end{cases} \begin{cases} \frac{\partial^2 u_x}{\partial x^2} = 0 \\ \frac{\partial^2 u_x}{\partial y^2} \neq 0 \\ \frac{\partial^2 u_x}{\partial z^2} = 0 \end{cases}$

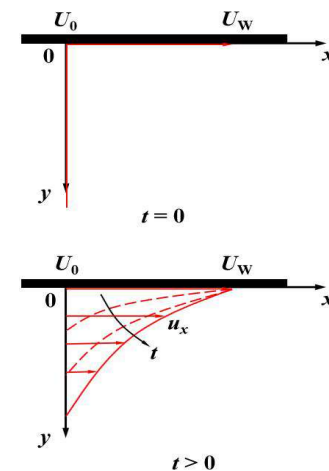


简化奈维-斯托克斯方程, 可得:

$$\frac{\partial u_x}{\partial t} = \nu \frac{\partial^2 u_x}{\partial y^2}$$

初始条件: $t = 0, u_x = 0$

边界条件: $t > 0, \begin{cases} y = 0, u_x = U_w \\ y \rightarrow \infty, u_x = U_0 = 0 \end{cases}$



x方向没有重力: $X = 0$

x方向无压差力: $\frac{\partial p}{\partial x} = 0$

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \rho X + \mu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right)$$

方程为一维非定常偏微分方程 $\frac{\partial u_x}{\partial t} = \nu \frac{\partial^2 u_x}{\partial y^2}$

令 $\eta = \frac{y}{\sqrt{4\nu t}}$ 则 $\frac{\partial u_x}{\partial t} = \frac{\partial u_x}{\partial \eta} \frac{\partial \eta}{\partial t} = -\frac{\eta}{2t} \frac{\partial u_x}{\partial \eta}$

$$\frac{\partial u_x}{\partial y} = \frac{\partial u_x}{\partial \eta} \frac{\partial \eta}{\partial y} = \frac{1}{\sqrt{4\nu t}} \frac{\partial u_x}{\partial \eta}$$

$$\frac{\partial^2 u_x}{\partial y^2} = \frac{1}{\sqrt{4\nu t}} \frac{\partial}{\partial \eta} \frac{\partial u_x}{\partial \eta} \frac{\partial \eta}{\partial y} = \frac{1}{4\nu t} \frac{\partial^2 u_x}{\partial \eta^2}$$

代入原方程可得: $\frac{d^2 u_x}{d\eta^2} + 2\eta \frac{du_x}{d\eta} = 0$

$$\frac{d^2 u_x}{d\eta^2} + 2\eta \frac{du_x}{d\eta} = 0$$

边界条件: $\begin{cases} \eta = 0, u_x = U_w \\ \eta \rightarrow \infty, u_x = U_0 = 0 \end{cases}$

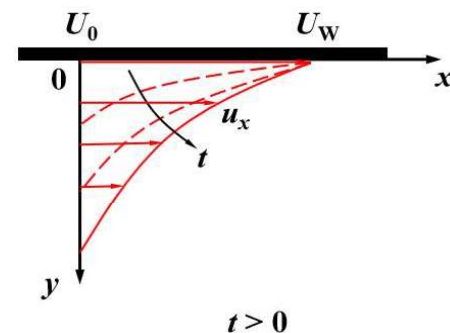
设: $\frac{du_x}{d\eta} = p$

$$\frac{dp}{d\eta} + 2\eta p = 0$$

积分: $p = C_1 e^{-\eta^2}$

$$\frac{du_x}{d\eta} = C_1 e^{-\eta^2}$$

再积分: $\int_{U_w}^{u_x} du_x = C_1 \int_0^\eta e^{-\eta^2} d\eta$



$$u_x - U_w = C_1 \int_0^\eta e^{-\eta^2} d\eta$$

边界条件: $\eta \rightarrow \infty, u_x = U_0 = 0$

$$C_1 = \frac{U_0 - U_w}{\int_0^\infty e^{-\eta^2} d\eta} \quad \text{其中: } \int_0^\infty e^{-\eta^2} d\eta = \frac{\sqrt{\pi}}{2}$$

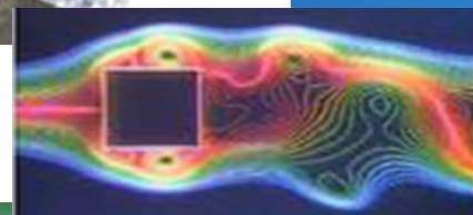
解方程得速度分布:

$$\frac{u_x - U_w}{U_0 - U_w} = \frac{2}{\sqrt{\pi}} \int_0^\eta e^{-\eta^2} d\eta$$

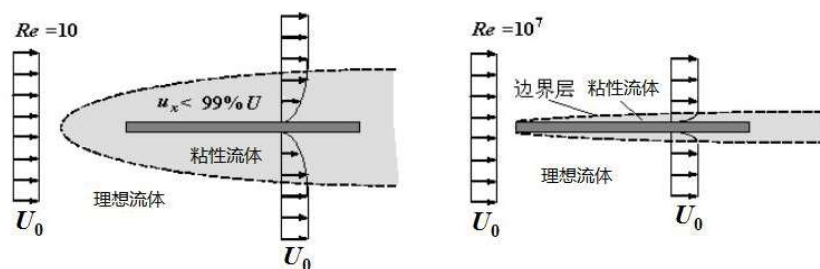
高斯误差函数

$$\frac{2}{\sqrt{\pi}} \int_0^\eta e^{-\eta^2} d\eta = \operatorname{erf}(\eta) = \frac{u_x - U_w}{U_0 - U_w} \quad \text{其中: } \eta = \frac{y}{\sqrt{4\nu t}}$$

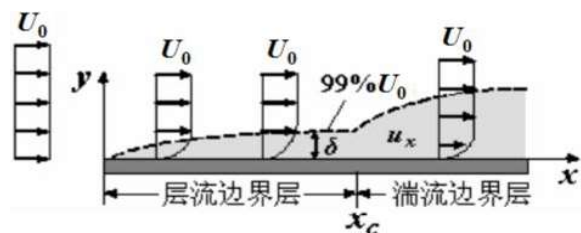
2.3 边界层理论



2.3.1 概念和特点



1904年，普朗特提出“边界层”概念



流体区域以 $99\%U$ 作为边界，内层区为粘性流体 u_x ，外层区为理想流体 U_0 。

特点

①. 慢：边界层内 $u_x < U_0$ ，壁面 $u_x = 0$ 。

②. 薄： $\delta \ll x$ 。

③. 陡： $\frac{du_x}{dy}$ 很大。

④. 增： $x \uparrow$ ， $\delta \uparrow$ 。

⑤. 旋：微团有旋。



⑥. 惯、粘同量级：惯性力与粘性力在边界层内量级相当。

⑦. 截面等压力：无压差流动。

⑧. 流型会转变： $x > x_c$ 时，层流 \rightarrow 湍流。

⑨. 逆压，失速会分离（绕曲面流动时的表现）。

