

# Numerical computation of the macroscopic tangent stiffness obtained from a microstructure

## 1 Introduction

For a microstructure occupying the volume  $V$ , the stress tensor  $\boldsymbol{\sigma}$  at the scale of the microstructure should satisfy the equilibrium in weak form:

$$\int_V \boldsymbol{\epsilon}(\mathbf{v}) : \boldsymbol{\sigma} \, dV = \int_{\partial V} \mathbf{v} \cdot \mathbf{t} \, dS \quad \forall \mathbf{v} \quad (1)$$

where  $\mathbf{t} = \boldsymbol{\sigma} \cdot \mathbf{n}$  is the stress vector at the boundary of  $V$ . Eq. (1) is also satisfied for a small variation of the stress which at a given state is linked to the variation of strain through a tangent stiffness tensor  $\mathbf{C}$ ,  $\delta \boldsymbol{\sigma} = \mathbf{C} : \boldsymbol{\epsilon}(\delta \mathbf{u})$ , resulting in

$$\int_V \boldsymbol{\epsilon}(\mathbf{v}) : \delta \boldsymbol{\sigma} \, dV = \int_{\partial V} \mathbf{v} \cdot \delta \mathbf{t} \, dS \quad \forall \mathbf{v} \quad (2)$$

Equation (2) with  $\delta \boldsymbol{\sigma} = \mathbf{C} : \boldsymbol{\epsilon}(\delta \mathbf{u})$  is equivalent to

$$\int_V w(\boldsymbol{\epsilon}(\delta \mathbf{u})) \, dV - \int_{\partial V} \delta \mathbf{u} \cdot \delta \mathbf{t} \, dS \leq \int_V w(\boldsymbol{\epsilon}(\mathbf{v})) \, dV - \int_{\partial V} \mathbf{v} \cdot \delta \mathbf{t} \, dS \quad \forall \mathbf{v} \quad (3)$$

where  $w(\boldsymbol{\epsilon}) = \frac{1}{2} \boldsymbol{\epsilon} : \mathbf{C} : \boldsymbol{\epsilon}$ .

## 2 Periodic boundary conditions

For periodic boundary conditions the displacement vector is linked to the macroscopic strain tensor  $\mathbf{E}$ . Any displacement field kinematically admissible with the periodic boundary conditions belongs to the set

$$\kappa(\mathbf{E}) = \{\mathbf{u} : \mathbf{u}(\mathbf{x}) = \mathbf{E} \cdot \mathbf{x} + \tilde{\mathbf{u}}(\mathbf{x}) \text{ with } \tilde{\mathbf{u}} \text{ periodic}\}$$

Since  $\delta \mathbf{t}$  is anti-periodic we can show that

$$\int_V \boldsymbol{\epsilon}(\mathbf{v}) : \delta \boldsymbol{\sigma} \, dV = \int_{\partial V} \mathbf{v} \cdot \delta \mathbf{t} \, dS = \mathbf{E} \int_V \delta \boldsymbol{\sigma} \, dV \quad \forall \mathbf{v} \in \kappa(\mathbf{E}) \quad (4)$$

Since  $\delta \mathbf{u} \in \kappa(\delta \mathbf{E})$ , we have

$$\int_{\partial V} \delta \mathbf{u} \cdot \delta \mathbf{t} \, dS = \int_{\partial V} \mathbf{v} \cdot \delta \mathbf{t} \, dS \quad \forall \mathbf{v} \in \kappa(\delta \mathbf{E}) \quad (5)$$

and therefore from (3)

$$\int_V w(\boldsymbol{\epsilon}(\delta \mathbf{u})) \, dV \leq \int_V w(\boldsymbol{\epsilon}(\mathbf{v})) \, dV \quad \forall \mathbf{v} \in \kappa(\delta \mathbf{E}) \quad (6)$$

The solution then minimizes  $w$ :

$$\langle w(\boldsymbol{\epsilon}(\delta \mathbf{u})) \rangle = \min_{\mathbf{v} \in \kappa(\delta \mathbf{E})} \langle w(\boldsymbol{\epsilon}(\mathbf{v})) \rangle \quad (7)$$

The minization can also be formulated by a minization with constraint

$$\langle w(\boldsymbol{\epsilon}(\delta \mathbf{u})) \rangle = \min_{\mathbf{e}, \mathbf{v} \text{ periodic}} \langle w(\mathbf{e}) \rangle \quad \text{subject to} \quad \delta \mathbf{E} + \boldsymbol{\epsilon}(\mathbf{v}) - \mathbf{e} = 0 \quad (8)$$

Introduce the lagrangian

$$\mathcal{L}(\mathbf{e}, \mathbf{v}, \boldsymbol{\lambda}; \delta \mathbf{E}) = \langle w(\mathbf{e}) \rangle + \langle \boldsymbol{\lambda}(\delta \mathbf{E} + \boldsymbol{\epsilon}(\mathbf{v}) - \mathbf{e}) \rangle \quad (9)$$

The solution turns into finding the stationay point of  $\mathcal{L}$  on the set of  $(\mathbf{e}, \mathbf{v} \text{ periodic}, \boldsymbol{\lambda})$

$$\mathcal{L}(\mathbf{e}^*, \mathbf{v}^*, \boldsymbol{\lambda}^*; \delta \mathbf{E}) = \langle w(\mathbf{e}^*) \rangle = \text{stationary point of } \mathcal{L}(\mathbf{e}, \mathbf{v}, \boldsymbol{\lambda}; \delta \mathbf{E}) \quad (10)$$

$\mathbf{e}, \mathbf{v} \text{ periodic}, \boldsymbol{\lambda}$

with  $\boldsymbol{\epsilon}(\delta \mathbf{u}) = \mathbf{e}^*$ ,  $\delta \mathbf{u}(\mathbf{x}) = \delta \mathbf{E} \cdot \mathbf{x} + \mathbf{v}^*(\mathbf{x})$  and  $\delta \boldsymbol{\sigma} = \boldsymbol{\lambda}^*$ . Due to the linearity of the problem  $\boldsymbol{\epsilon}(\delta \mathbf{u}(\mathbf{x})) = \mathbf{A}(\mathbf{x}) : \delta \mathbf{E}$  and thus

$$\mathcal{L}(\mathbf{e}^*, \mathbf{v}^*, \boldsymbol{\lambda}^*; \delta \mathbf{E}) = \langle w(\mathbf{e}^*) \rangle = \frac{1}{2} \delta \mathbf{E} : \langle {}^t \mathbf{A} : \mathbf{C} : \mathbf{A} \rangle : \delta \mathbf{E} \quad (11)$$

The tensor  $\mathbf{C}^{\text{hom}} = \langle {}^t \mathbf{A} : \mathbf{C} : \mathbf{A} \rangle$  is the homogenized tangent stiffness tensor. It is also given by  $\mathbf{C}^{\text{hom}} = \langle \mathbf{C} : \mathbf{A} \rangle$  through the identity

$$\langle \boldsymbol{\epsilon}(\mathbf{v}) : \delta \boldsymbol{\sigma} \rangle = \mathbf{E} : \langle \delta \boldsymbol{\sigma} \rangle \quad \forall \mathbf{v} \in \kappa(\mathbf{E}) \quad (12)$$

Therefore it can be shown that

$$\mathbf{C}^{\text{hom}} = \langle \mathbf{C} \rangle - \langle ({}^t \mathbf{A} - \mathbf{1}) : \mathbf{C} : (\mathbf{A} - \mathbf{1}) \rangle \quad (13)$$

### 3 FE discretization

After a FE discretization (2) turns into:

$$\mathbf{v} \mathbf{K} \delta \mathbf{u} = \mathbf{v} \delta \mathbf{f} \quad \forall \mathbf{v} \quad (14)$$

where  $\delta \mathbf{u}$  and  $\mathbf{v}$  are nodal displacement vectors and  $\mathbf{K}$  the tangent FE matrix.

For the matrix column  $\tilde{\mathbf{u}}$  associated to the periodic displacement field  $\tilde{\mathbf{u}}(\mathbf{x})$ , there is a  $n \times m$  matrix  $\mathbf{P}$  ( $n = \text{nb of dof}$ ,  $m = \text{nb of relationships between displacement components}$ ) so that

$$\mathbf{P} \tilde{\mathbf{u}} = 0 \quad (15)$$

The traction field  $\delta \mathbf{t}(\mathbf{x})$  being anti-periodic, we have

$$\tilde{\mathbf{v}}\mathbf{K}\delta\mathbf{u} = 0 \quad \forall \tilde{\mathbf{v}} : \mathbf{P}\tilde{\mathbf{v}} = 0 \quad (16)$$

Hence there are  $m$  Lagrange multipliers or a  $m$ -components vector  $\lambda$

$$\mathbf{K}\delta\mathbf{u} = {}^t\mathbf{P}\lambda \quad (17)$$

$$\mathbf{P}\delta\mathbf{u} = \mathbf{P}(\delta\mathbf{E} \cdot \mathbf{x}) \quad (18)$$

where  $\delta\mathbf{E} \cdot \mathbf{x}$  is the matrix column formed with the displacement field obtained from the homogeneous deformation gradient  $\delta\mathbf{E} \cdot \mathbf{x}$  so that  $\delta\mathbf{u} = \delta\mathbf{E} \cdot \mathbf{x} + \tilde{\mathbf{u}}$ .

The number of the reduced unknowns is  $n - m$ . We note  $\tilde{\mathbf{u}}_r$  the reduced unknown vector and there is a  $(n - m) \times n$  matrix  $\mathbf{B}$ :

$$\tilde{\mathbf{u}} = \mathbf{B}\tilde{\mathbf{u}}_r \quad (19)$$

resulting in

$$\mathbf{P}\mathbf{B} = 0 \quad (20)$$

From the above equations

$$\mathbf{K}_r\tilde{\mathbf{u}}_r = \mathbf{f}_r = -{}^t\mathbf{B}\mathbf{K}(\delta\mathbf{E} \cdot \mathbf{x}) \quad (21)$$

with  $\mathbf{K}_r = {}^t\mathbf{B}\mathbf{K}\mathbf{B}$ . The homogenized tangent stiffness tensor  $\mathbf{C}^{\text{hom}}$  can be identified through the equivalence of the work forms

$$V\delta\mathbf{E} : \mathbf{C}^{\text{hom}} : \delta\mathbf{E}' = \delta\mathbf{u}\mathbf{K}\delta\mathbf{u}' \quad (22)$$

where  $\delta\mathbf{u}$  (resp.  $\delta\mathbf{u}'$ ) is the solution associated with  $\delta\mathbf{E}$  (resp.  $\delta\mathbf{E}'$ ), so

$$V\delta\mathbf{E} : \mathbf{C}^{\text{hom}} : \delta\mathbf{E}' = (\delta\mathbf{E} \cdot \mathbf{x})\mathbf{K}(\delta\mathbf{E}' \cdot \mathbf{x}) - \tilde{\mathbf{u}}_r\mathbf{f}'_r \quad (23)$$

which is the discretized form of (13). If  $\mathbf{K}$  is symmetric we can ensure the symmetry of  $\mathbf{C}^{\text{hom}}$  as follows:

$$V\delta\mathbf{E} : \mathbf{C}^{\text{hom}} : \delta\mathbf{E}' = (\delta\mathbf{E} \cdot \mathbf{x})\mathbf{K}(\delta\mathbf{E}' \cdot \mathbf{x}) - \frac{1}{2}(\tilde{\mathbf{u}}_r\mathbf{f}'_r + \tilde{\mathbf{u}}'_r\mathbf{f}_r) \quad (24)$$

The computation of  $\mathbf{C}^{\text{hom}}$  can be performed by the successive use of Eqs (21) and (23 or 24). Indeed during the iterative process of the FE method, at each iteration the matrices  $\mathbf{K}_r$  and  $\mathbf{K}$  are built and linear sets of equations with left hand sides similar to that of (21) are solved. Then Eq (21) can be solved for  $\tilde{\mathbf{u}}_r$  six times with right hand side obtained by setting  $\delta\mathbf{E} = 1/2(\mathbf{e}_k \otimes \mathbf{e}_l + \mathbf{e}_l \otimes \mathbf{e}_k)$ . So six resolutions should be performed. Eq (23 or 24) is then used to calculate the corresponding components of  $\mathbf{C}^{\text{hom}}$ .