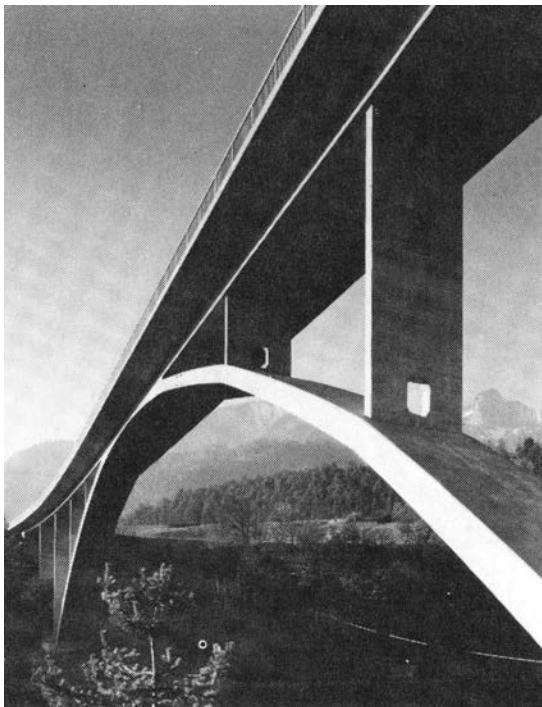


MIT Class 6.S080 (AUS)
Mechanical Invention through Computation

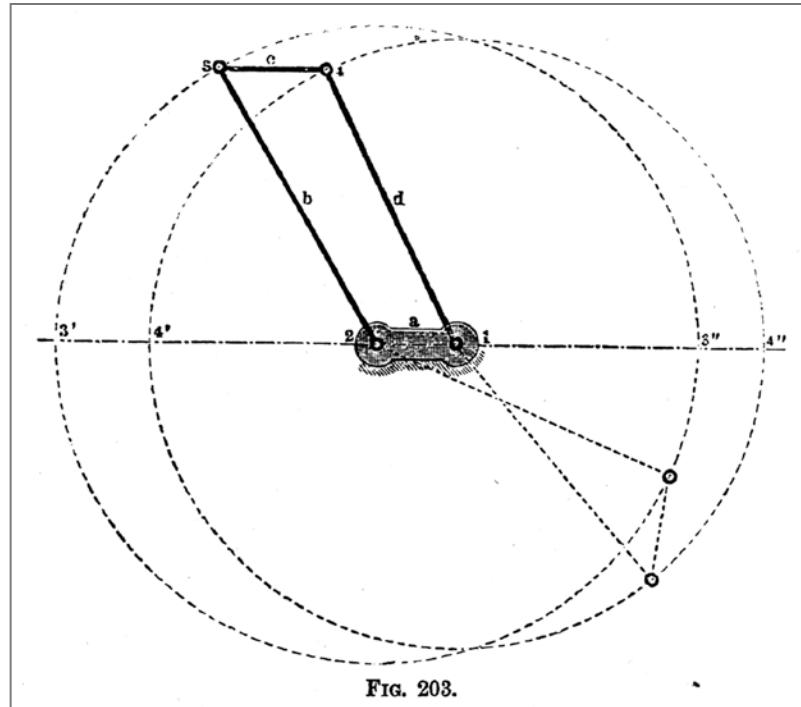
Mechanism Basics

Design Principles

Structure and Mechanism



Structure:
Force is resisted

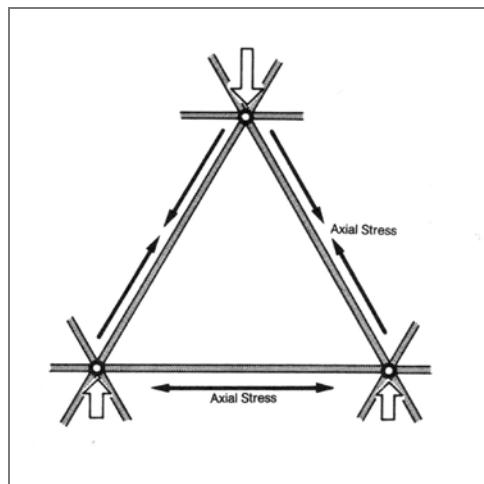


Mechanism:
Force flows into movement

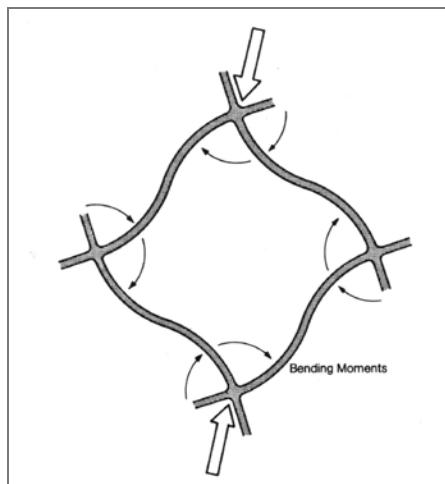
Design Principles

Structure and Mechanism

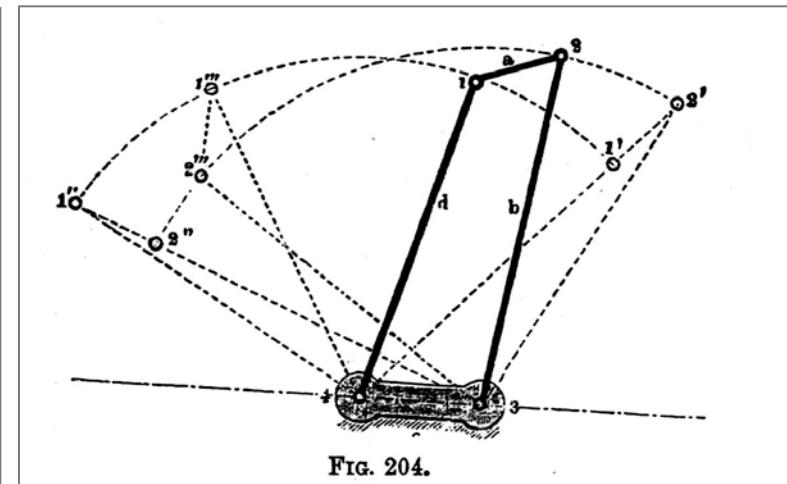
Possible Responses to Applied Force



Structural
Resistance



Structural Deflection
(elastic or Inelastic)

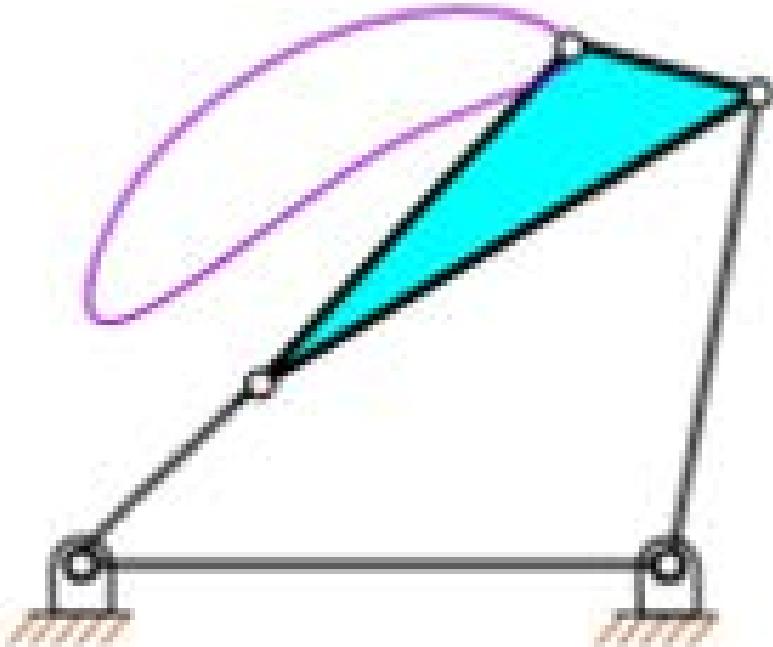


Kinematic deflection

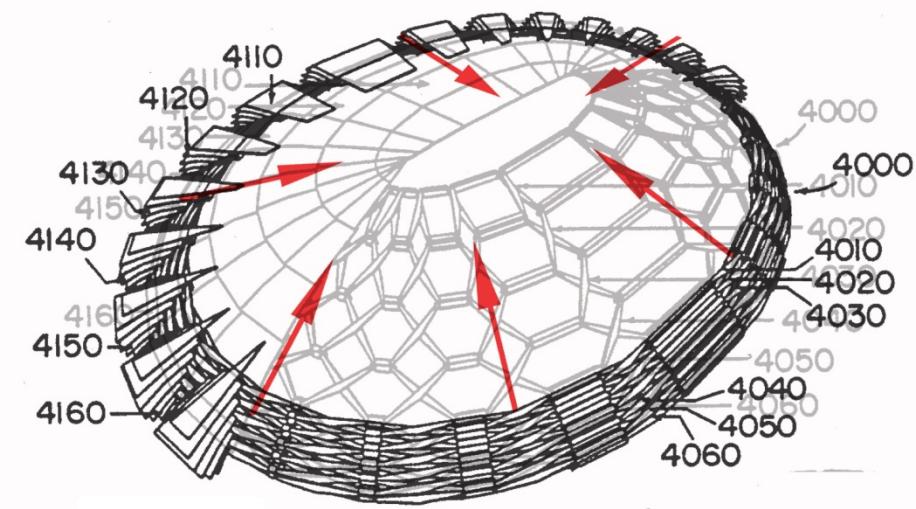
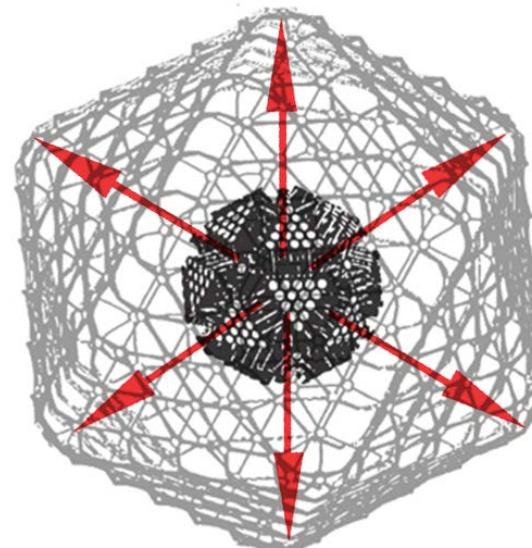
FIG. 204.

Mechanism paradigms

Synthesize a motion path



Synthesize a form change



Definitions

- **Kinematics:** the study of the motion of bodies without reference to mass or force
- **Links:** considered as rigid bodies
- **Kinematic pair:** a connection between two bodies that imposes constraints on their relative movement. (also referred to as a mechanical joint)
- **Ground:** static point of reference
- **Degree of freedom (DOF):** of a mechanical system is the number of independent parameters that define its configuration.

Links types

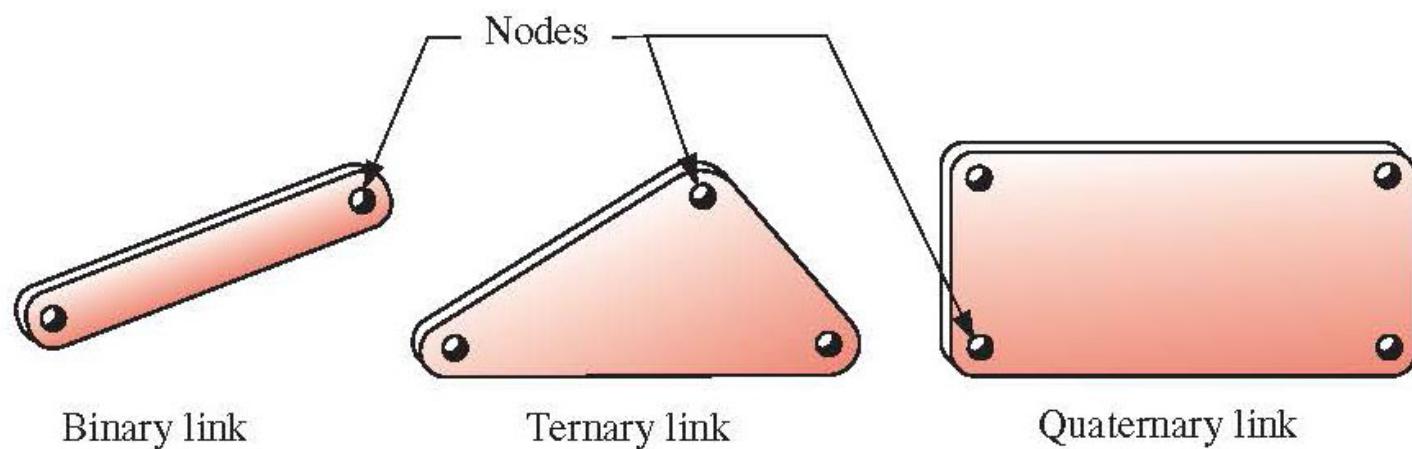


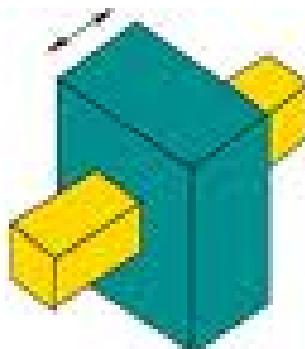
FIGURE 2-2

Links of different order

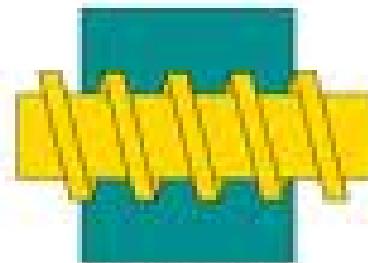
Kinematic pairs



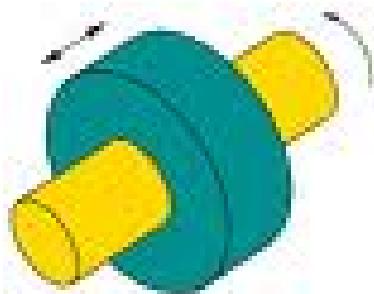
Revolute
1 Degree of Freedom



Prismatic
1 Degree of Freedom



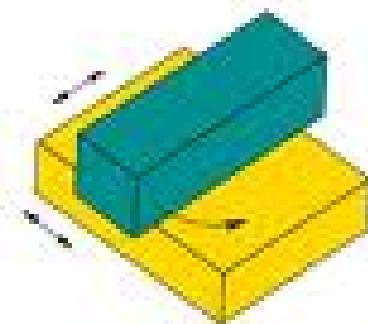
Screw
1 Degree of Freedom



Cylindrical
2 Degrees of Freedom



Spherical
3 Degrees of Freedom



Planar
3 Degrees of Freedom

Historic Mechanisms

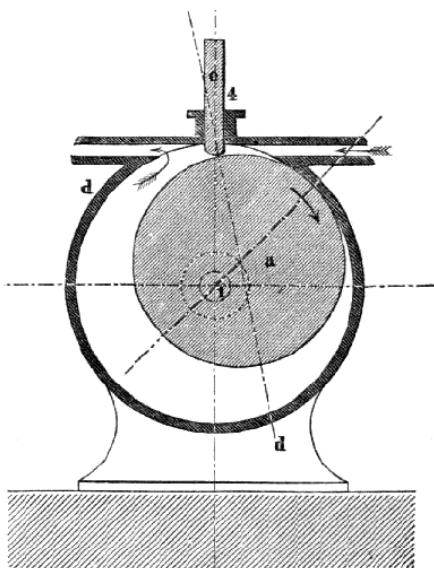
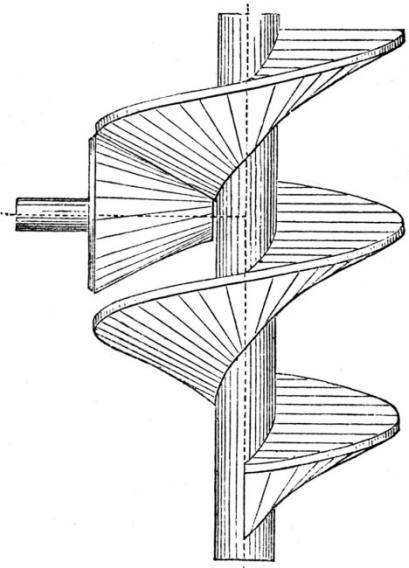
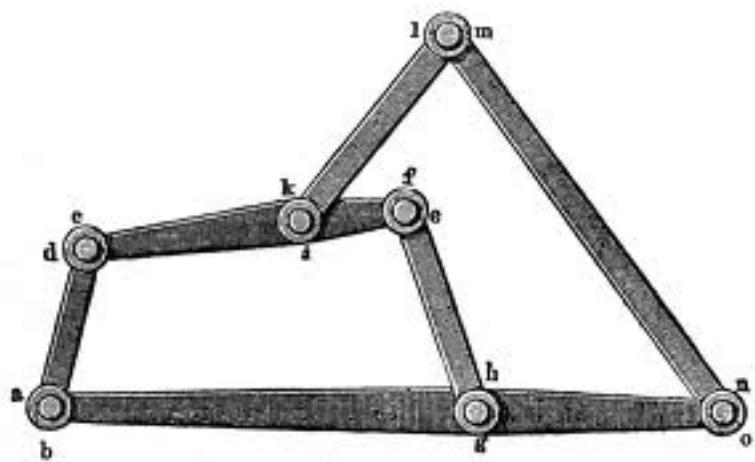


FIG. 4.—Yule, Hall. (Eng.)
 $(C''_3 P^{\perp})^{\frac{1}{2}} - b - \frac{c}{2}; (V^{\perp}) = a, d.$

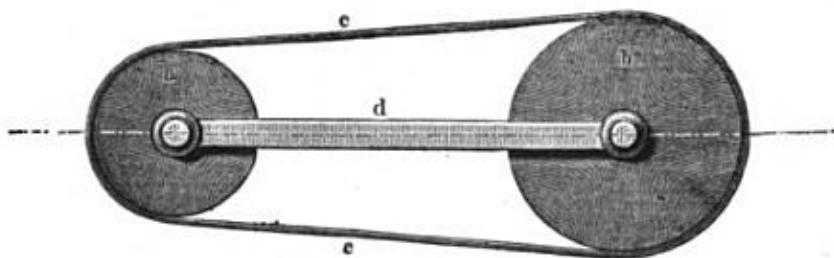
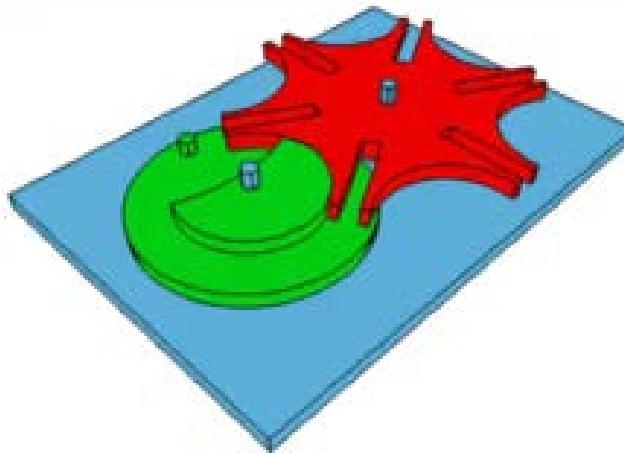


FIG. 182.

the same as that running off the other. The band for the pulley *a* is identical—coincident—with that for *b*, the corre-

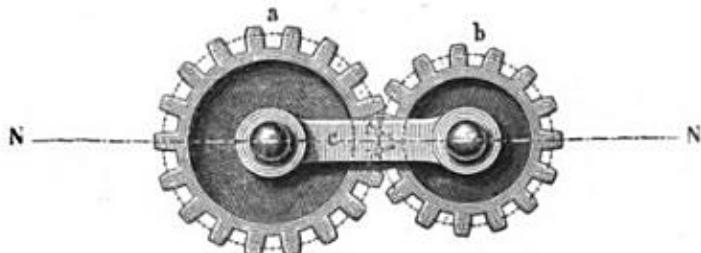
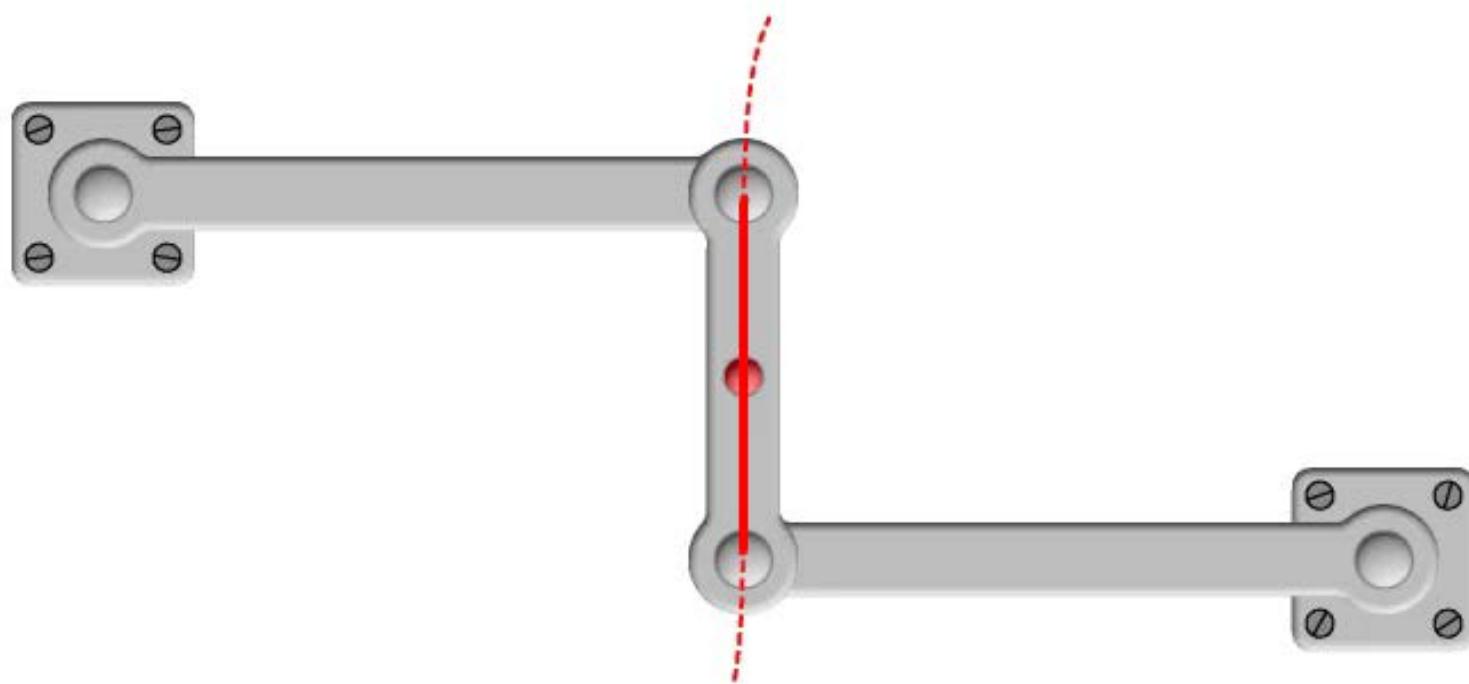
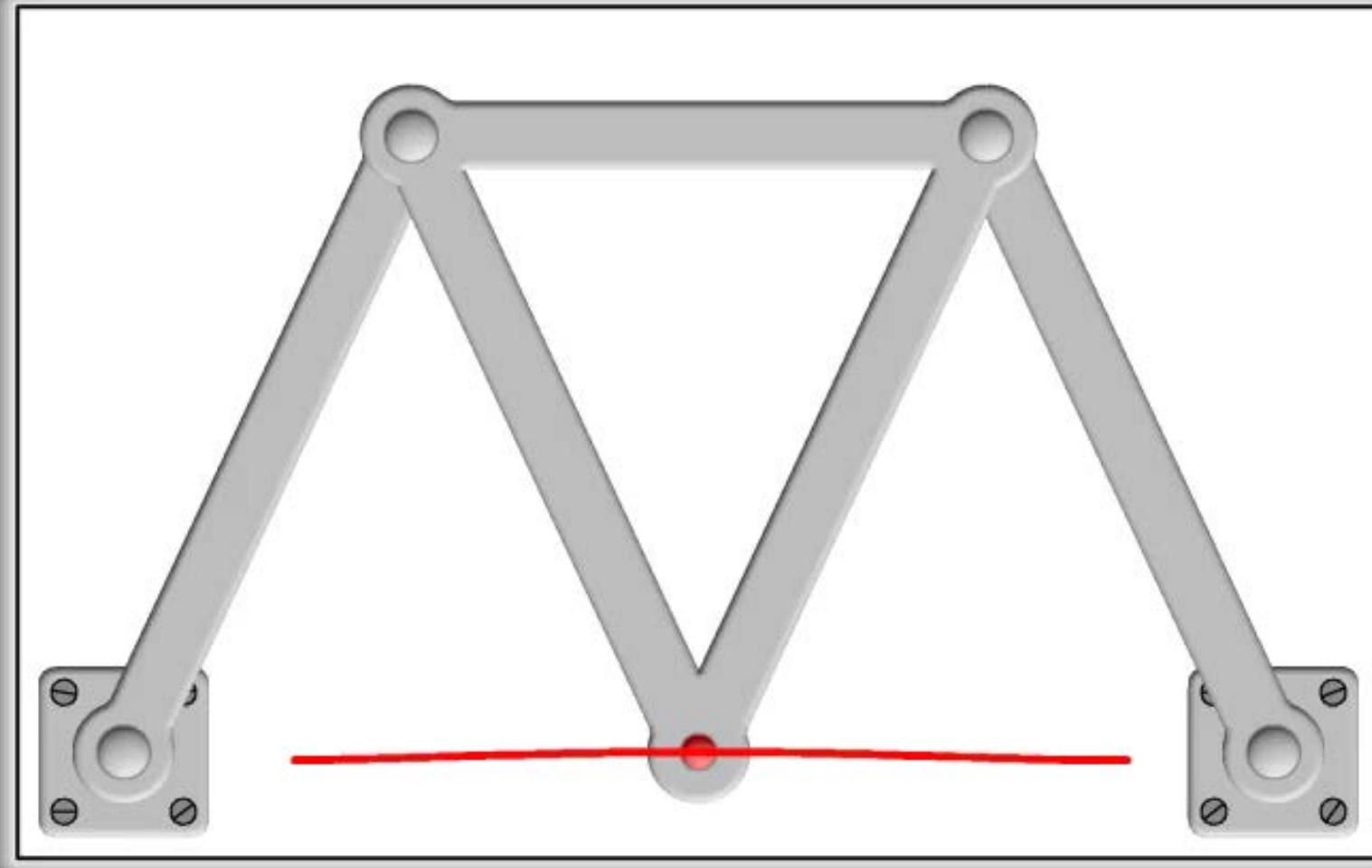


FIG. 183.

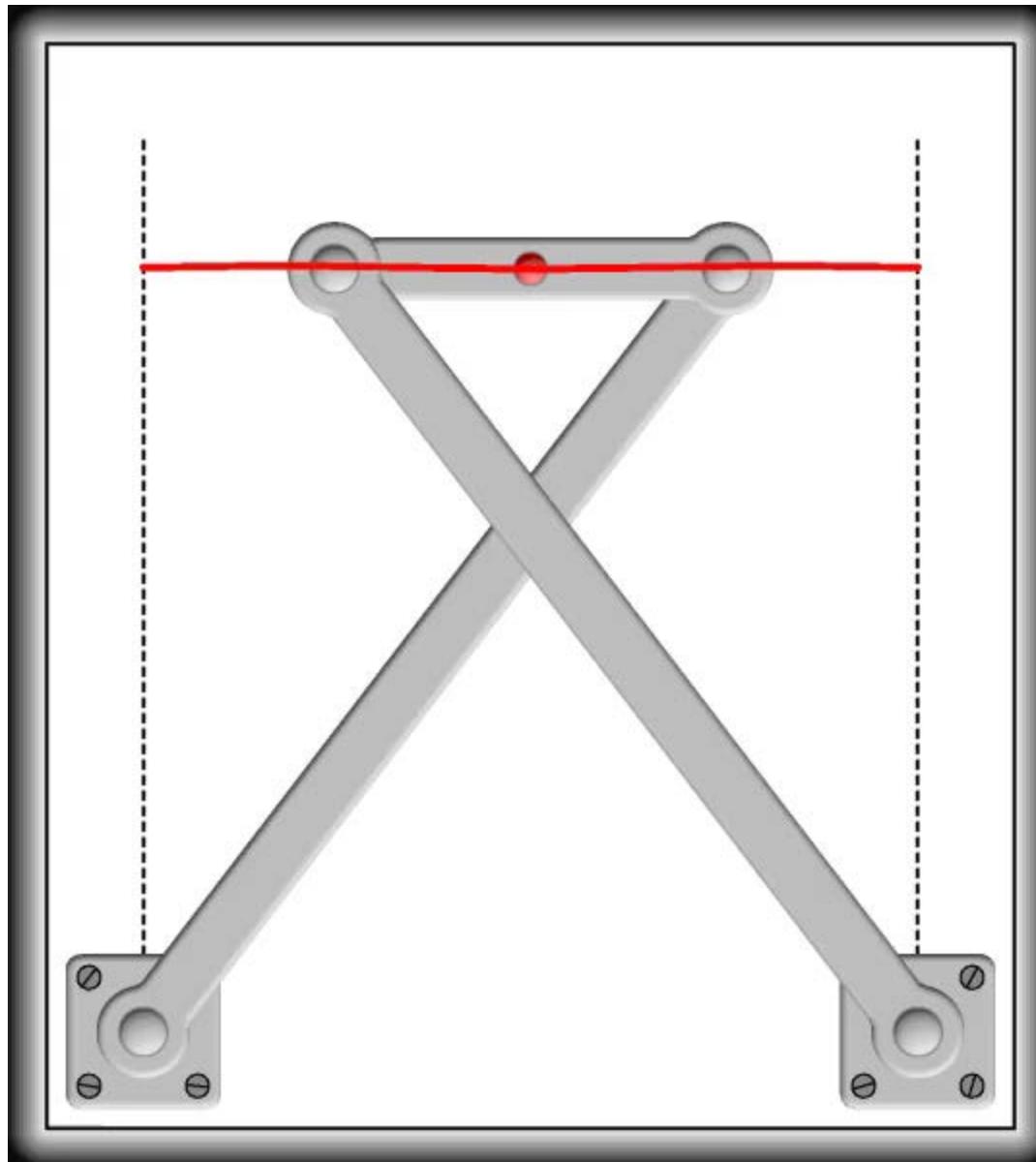
Straight-line linkages (James Watt)



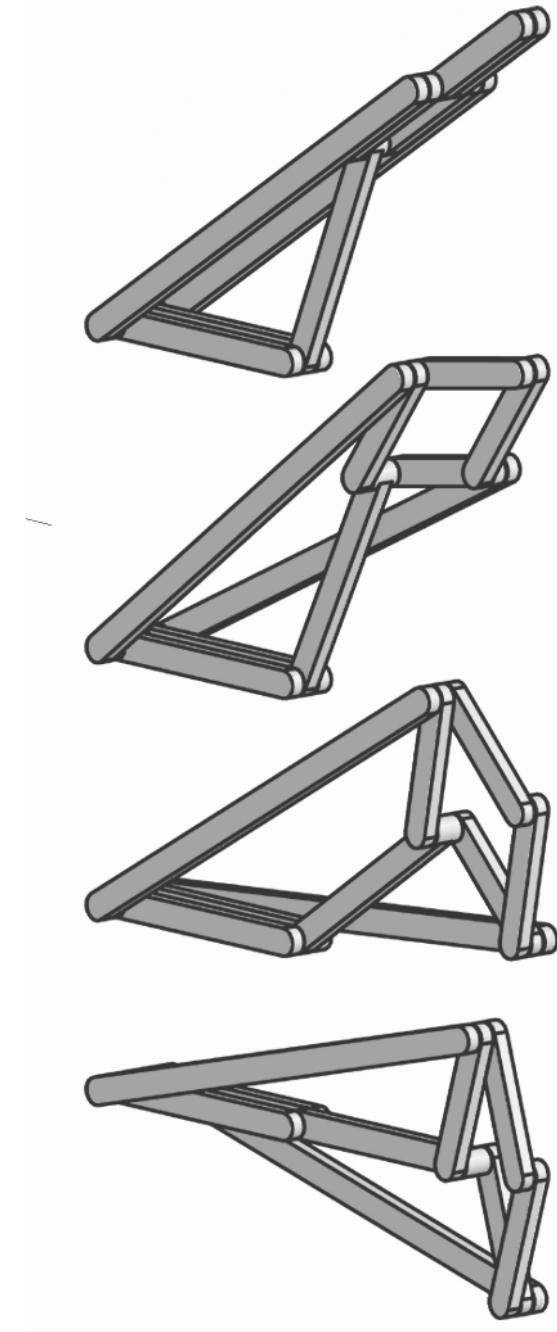
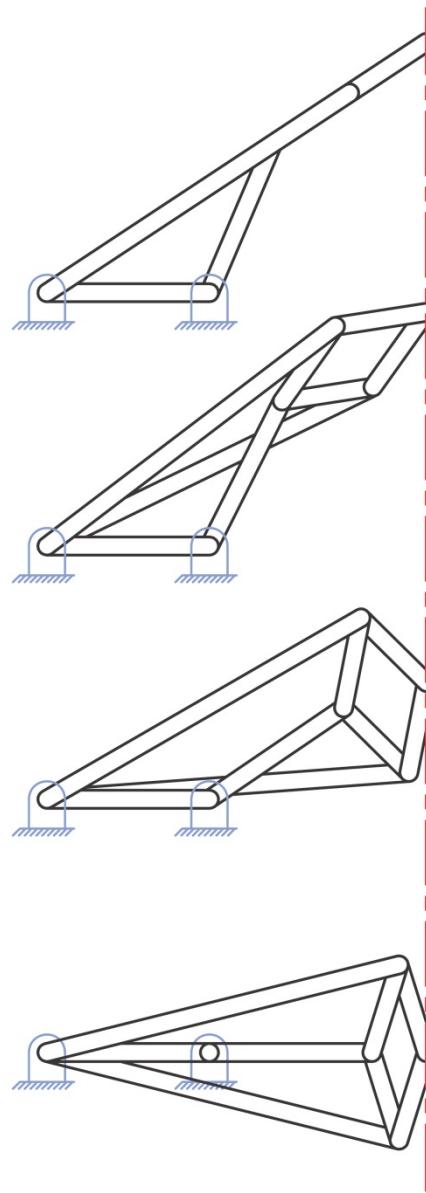
Straight-line linkages (Richard Roberts)



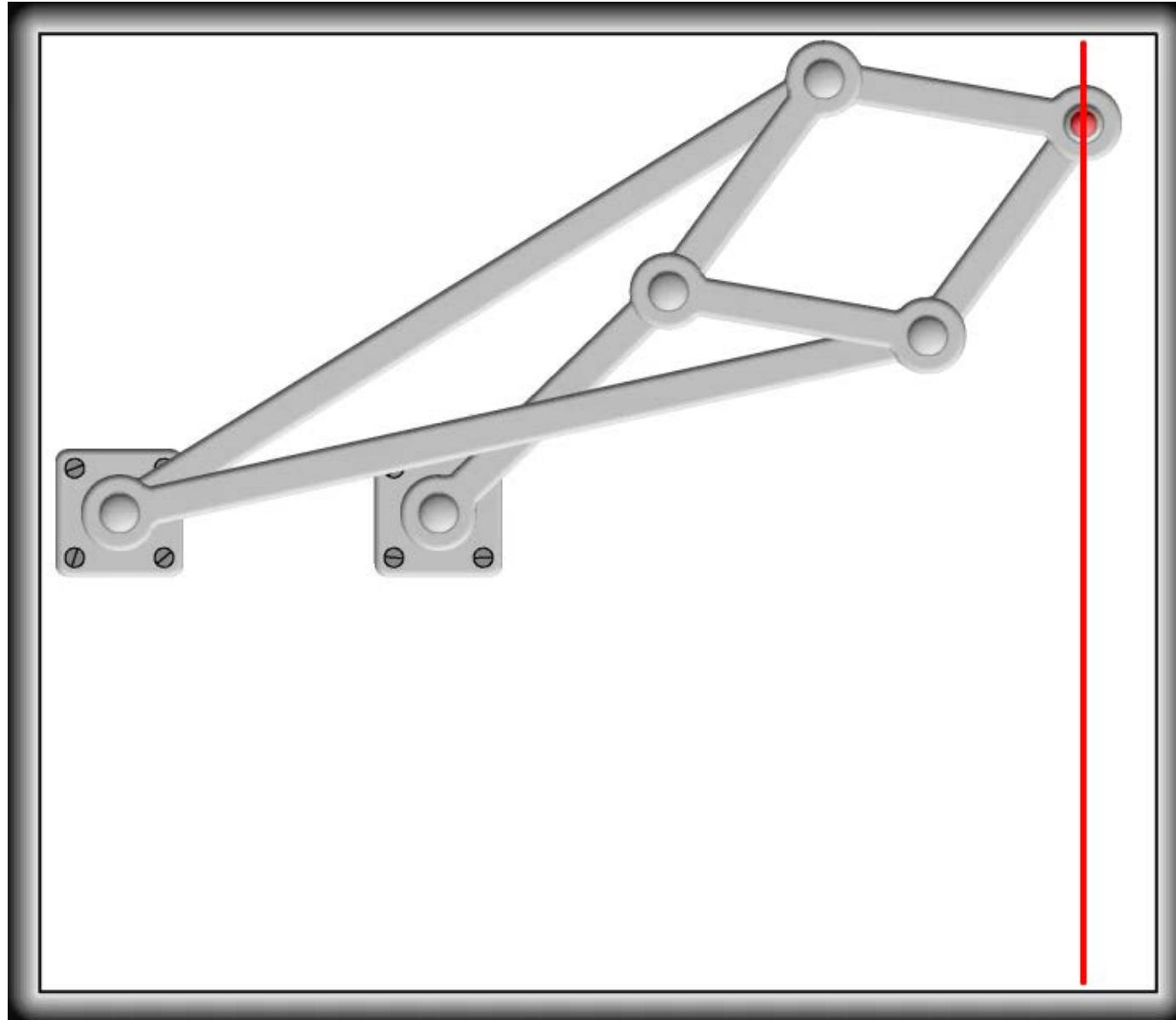
Straight-line linkages (Tchebicheff)



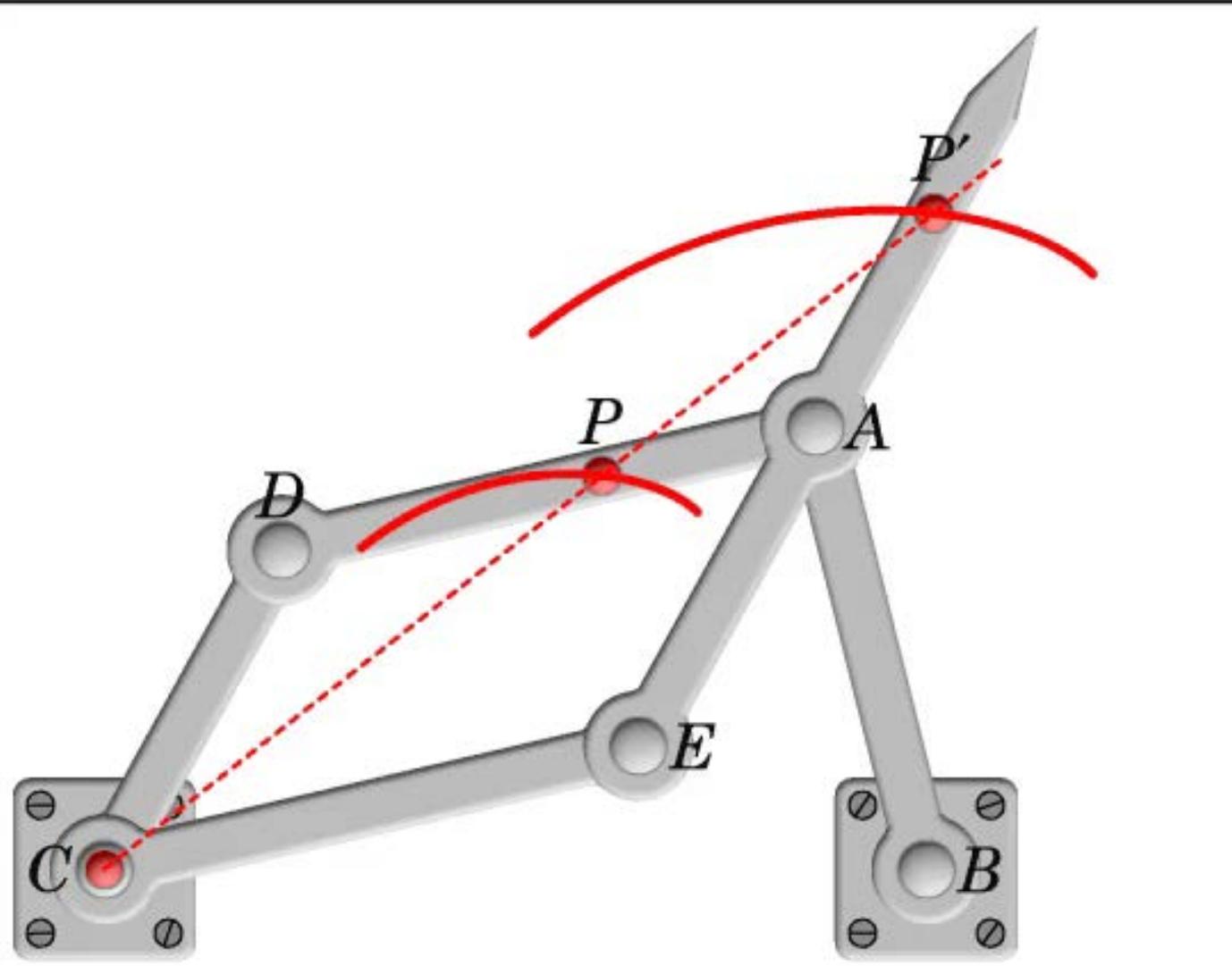
Peaucellier Linkage



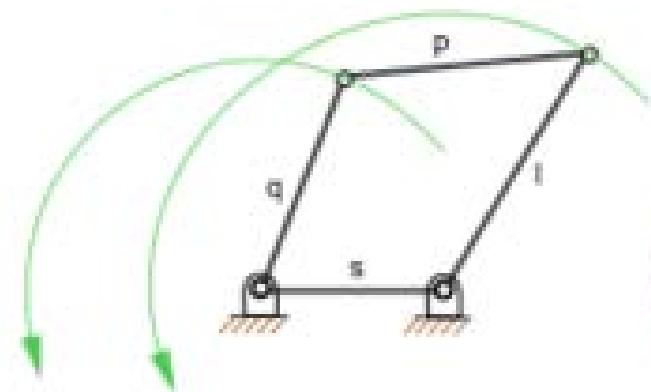
Straight-line linkages (Peaucellier-Lipkin)



Straight-line linkages (Kempe)

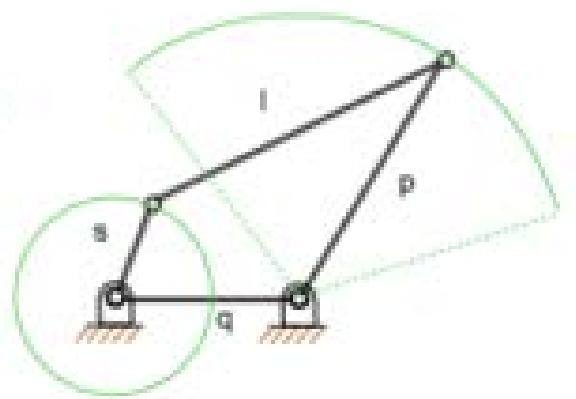


4-bar linkage types

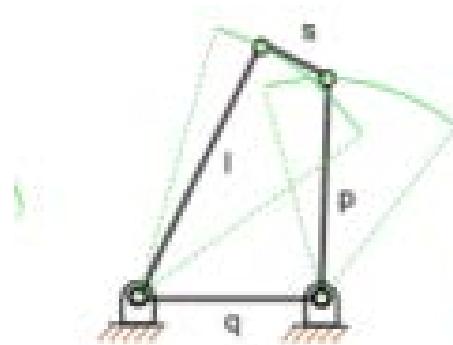


(full revolution,
both links)

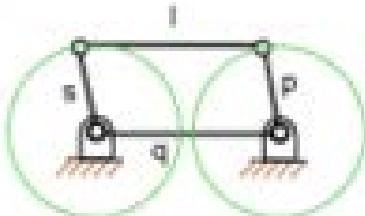
Drag-link
 $s+l > p+q$
(continuous motion)



Crank-rocker
 $s+l > p+q$
(continuous motion)

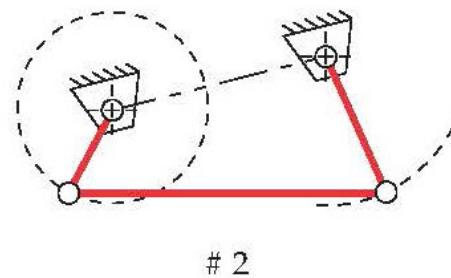
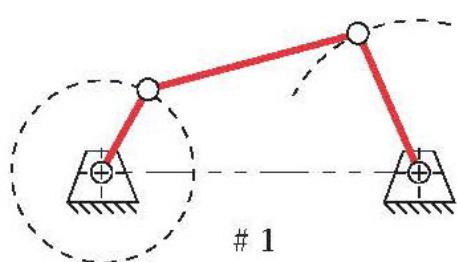


Double-rocker
 $s+l > p+q$
(no continuous motion)

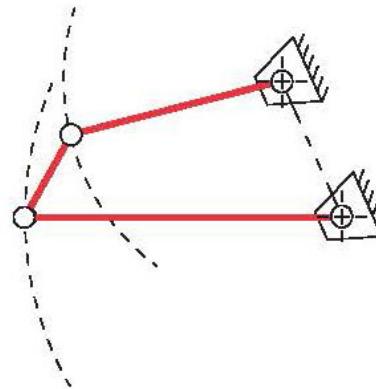
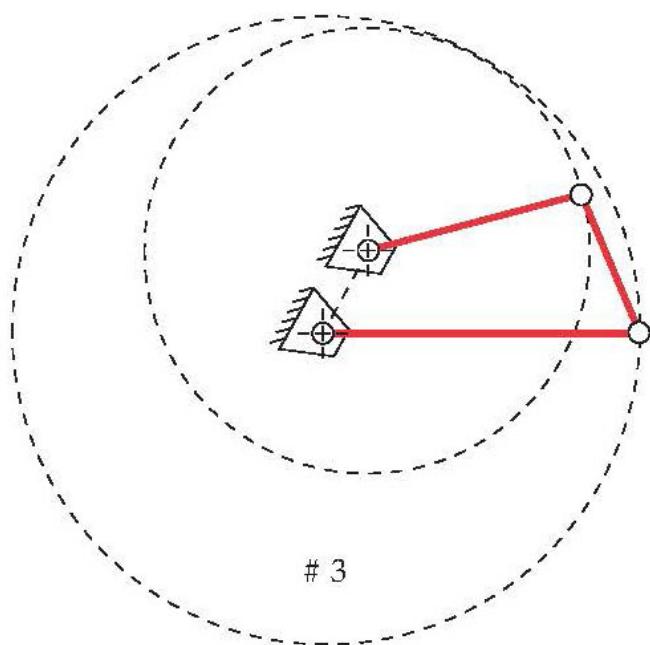


Parallelogram linkage
 $s+l > p+q$
(continuous motion)

Kinematic inversions



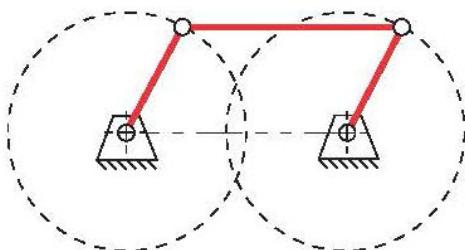
(a) Two non-distinct crank-rocker inversions (GCRR)



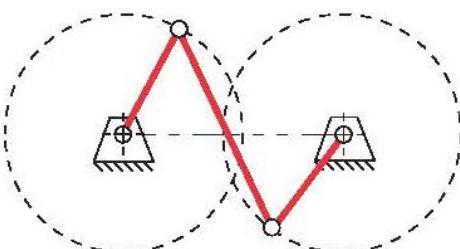
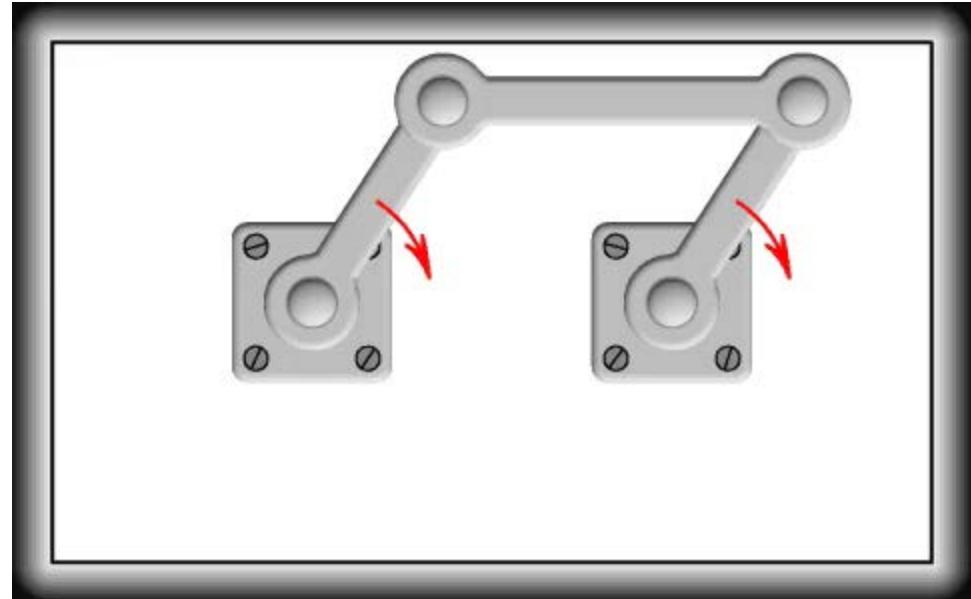
(b) Double-crank inversion (GCCC)
(drag link mechanism)

(c) Double-rocker inversion (GRCR)
(coupler rotates)

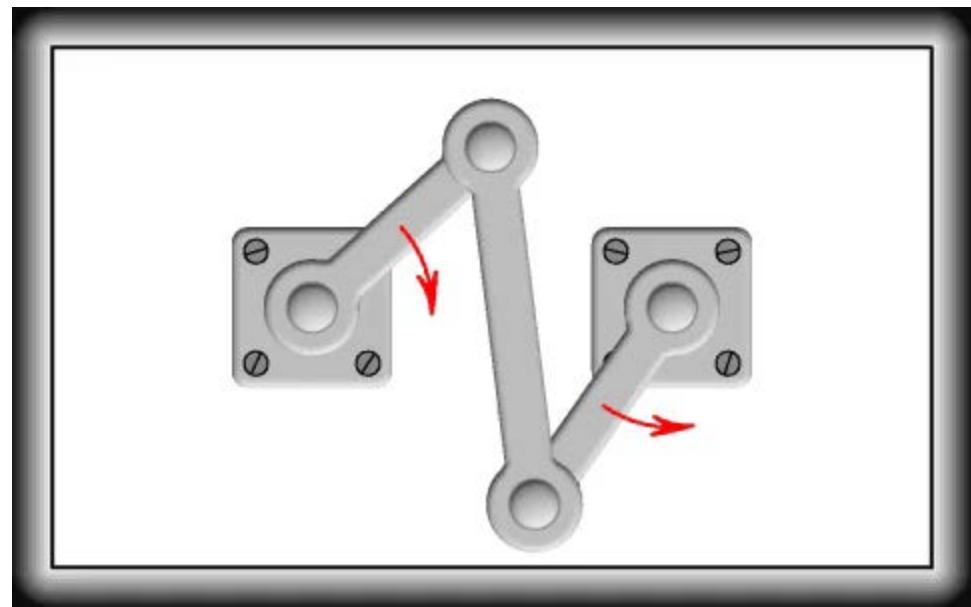
Four-bar linkage examples



Parallel 4-bar



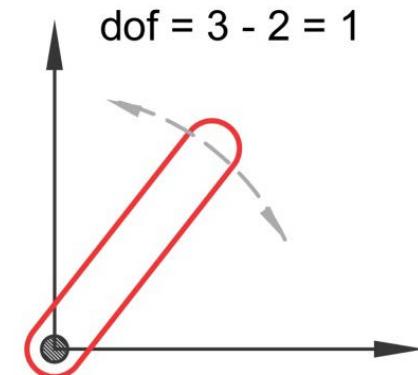
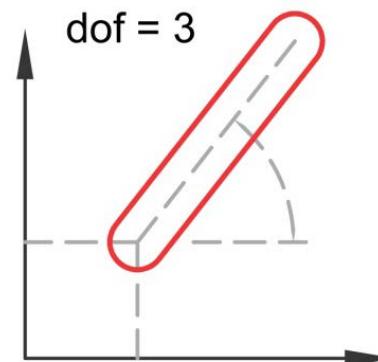
Anti-parallel 4-bar



Gruebler's equation

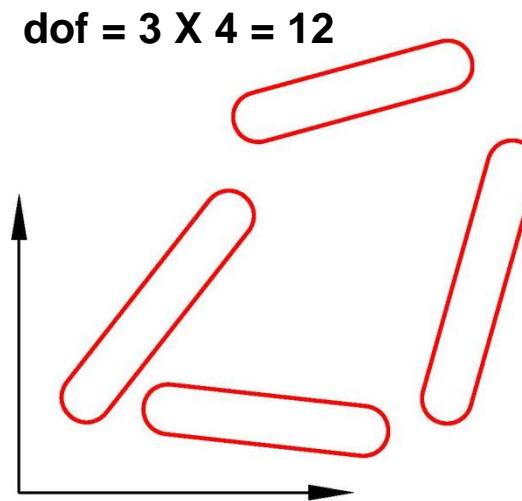
N = Number of Links (including ground link)

P = Number of Joints (pivot connections between links)

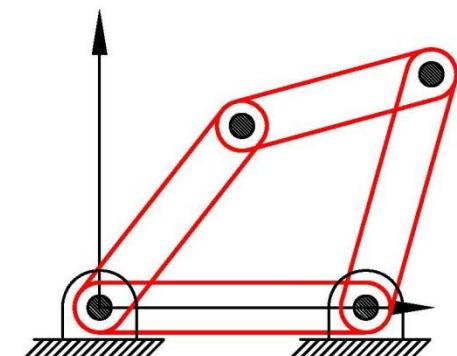


- Each link has 3 degrees of freedom
- Each pivot subtracts 2 degree of freedom

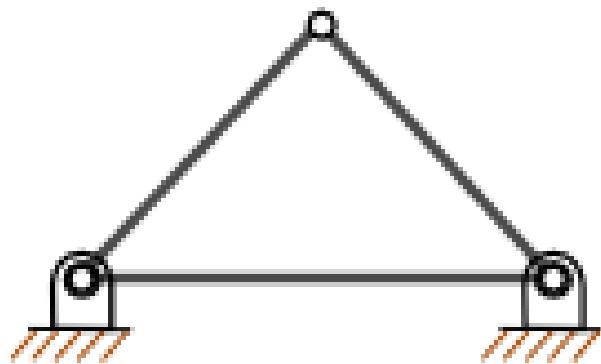
$$\text{DOF} = 3(N-1) - 2P$$



$$\text{dof} = 3 \times (4-1) - (2 \times 4) = 1$$



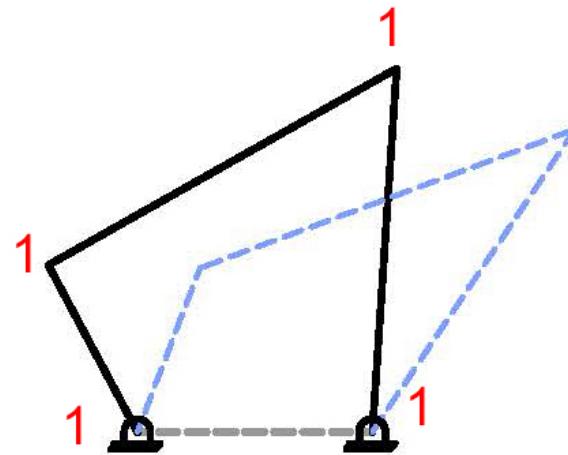
Examples



$$N = 3$$

$$P = 3$$

$$DOF = 3 \times (3-1) - (2 \times 3) = 0$$

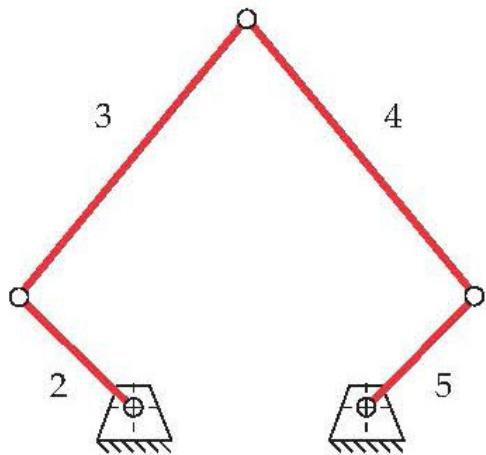


$$N = 4$$

$$P = 4$$

$$DOF = 3 \times (4-1) - (2 \times 4) = 1$$

Examples

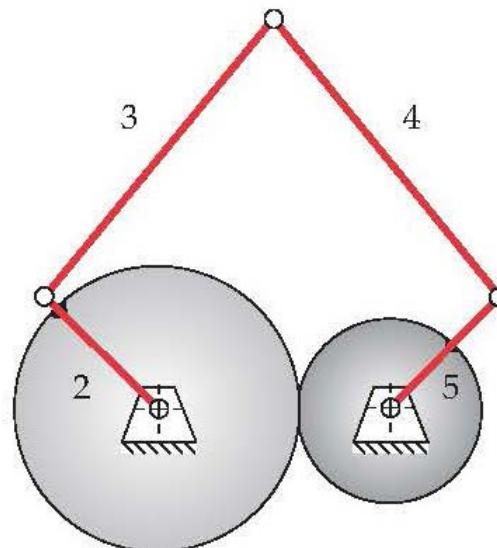


(a) Fivebar linkage—2 DOF

$$N = 5$$

$$P = 5$$

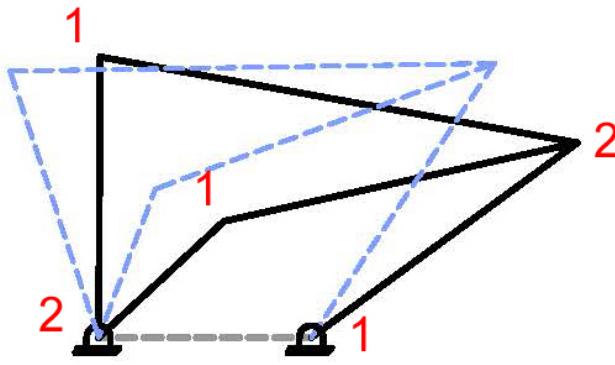
$$\text{DOF} = 2$$



(b) Geared fivebar linkage—1 DOF

Geared connection removes
one degree of freedom
DOF = 1

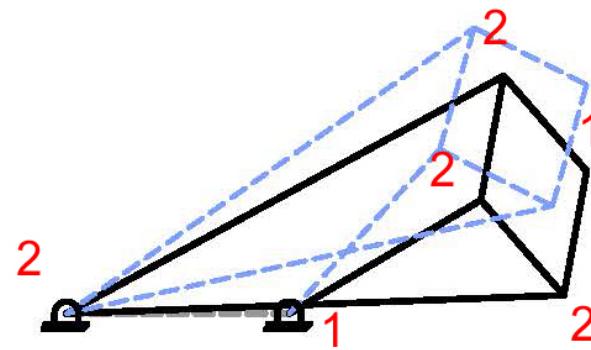
Examples



$$N = 6$$

$$P = 7$$

$$\text{DOF} = 3 \times (6-1) - (2 \times 7) = 1$$



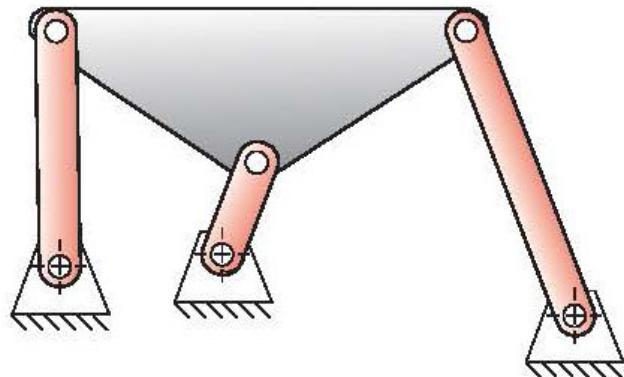
$$N = 8$$

$$P = 10$$

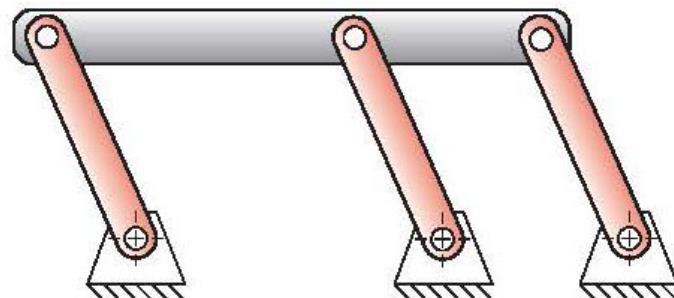
$$\text{DOF} = 3 \times (8-1) - (2 \times 10) = 1$$

Relation of DOF to special geometries

Agrees with Gruebler's equation (doesn't move)



Doesn't agree with Gruebler's equation (moves)



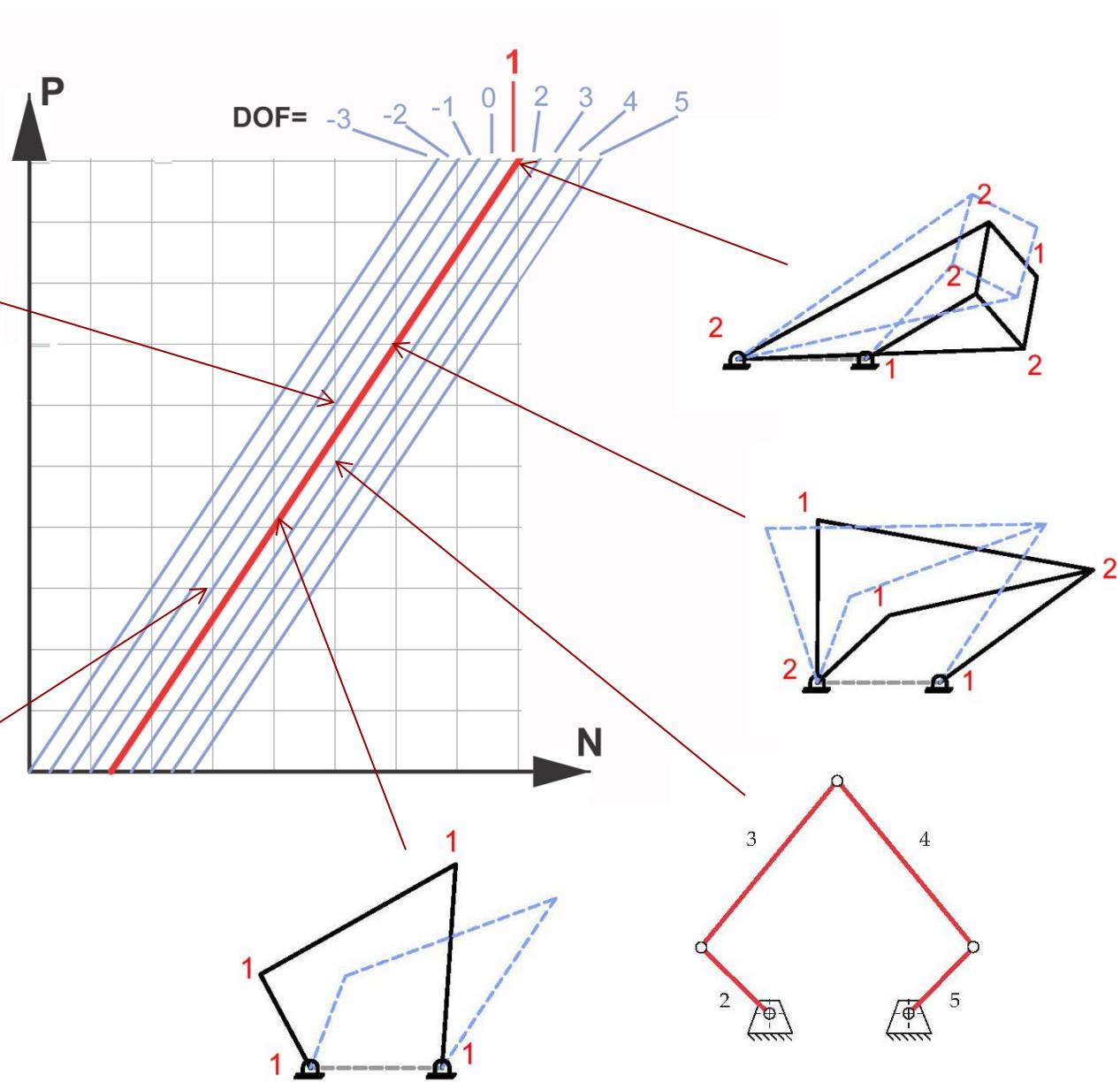
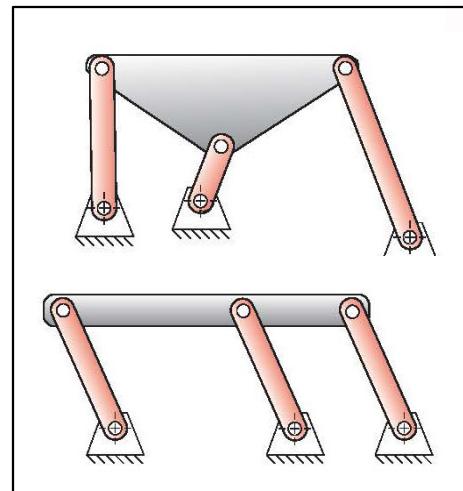
$$N = 5$$

$$P = 6$$

$$\text{DOF} = 3 \times (5-1) - (2 \times 6) = 0$$

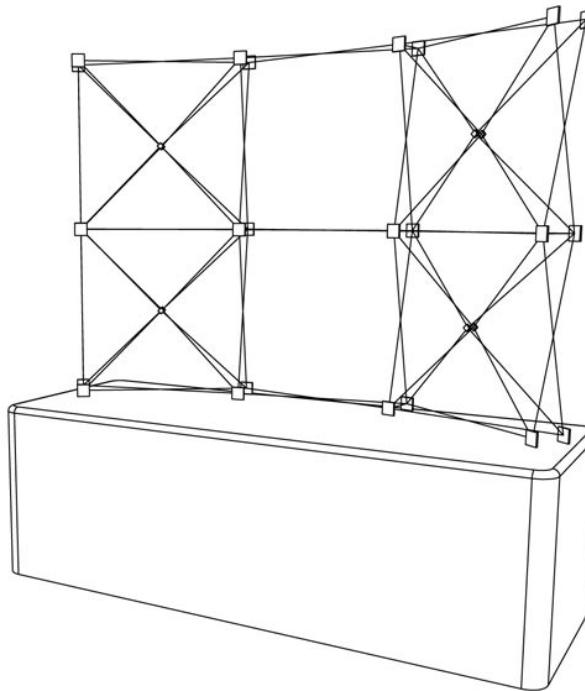
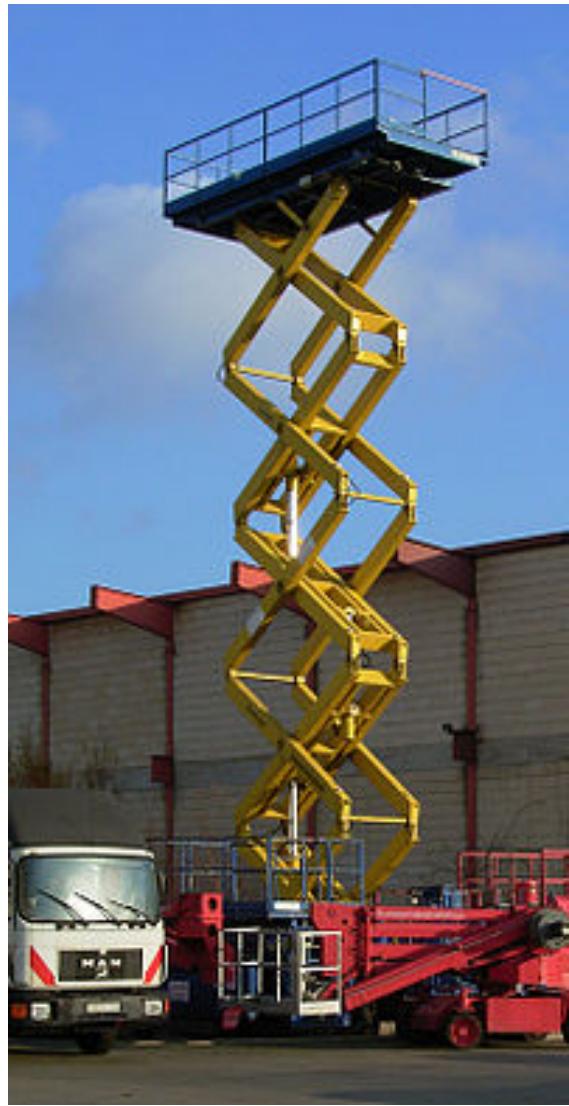
Graph of linkages

$$DOF = 3(N-1) - 2P$$

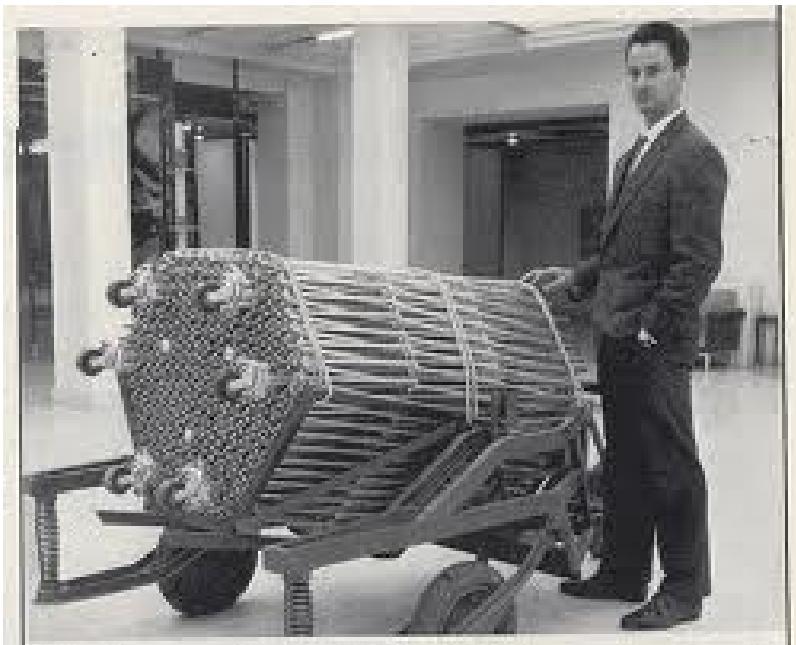


Scissor Linkages

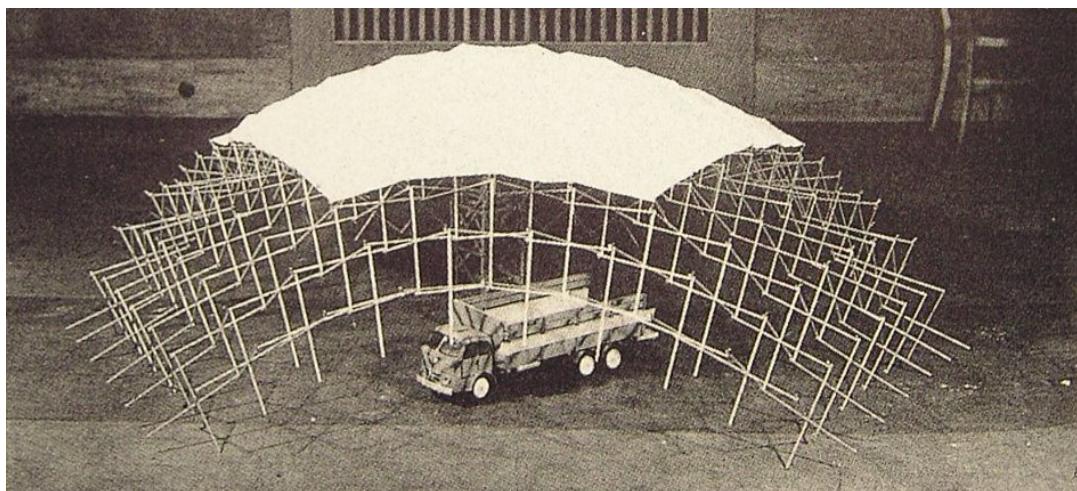
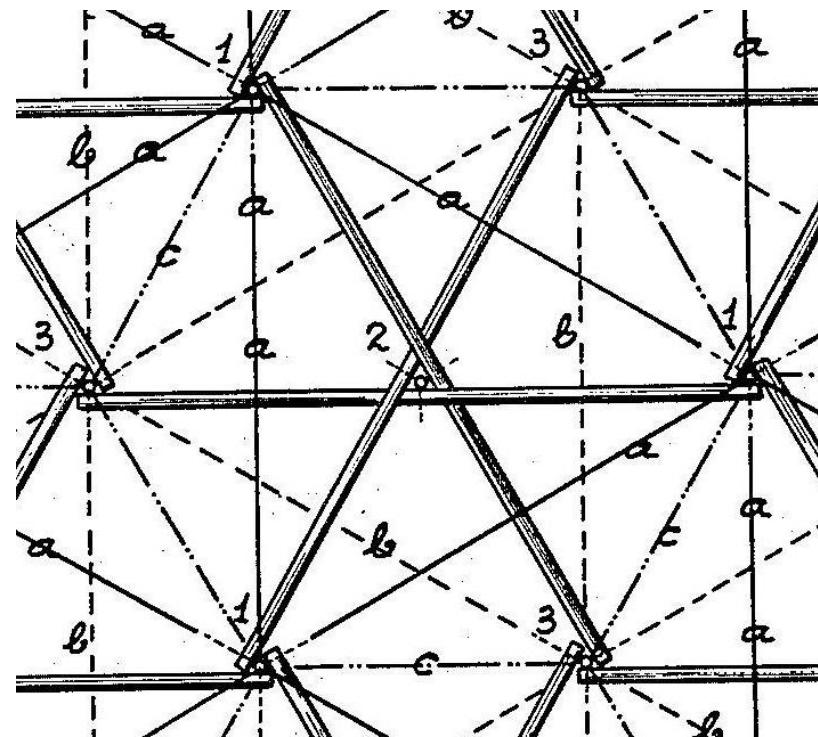
Scissor mechanisms



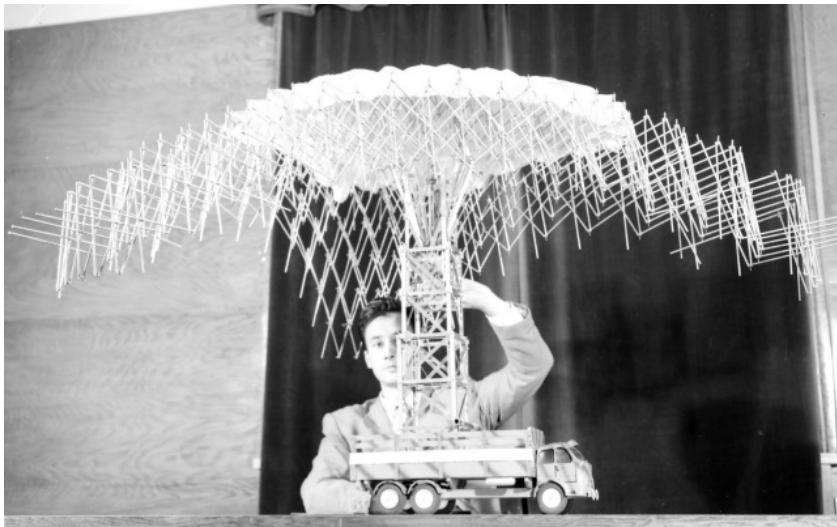
Historic examples of scissor mechanisms



Emilio Pinero



Historic examples of scissor mechanisms



Emilio Pinero



Examples of scissor mechanisms



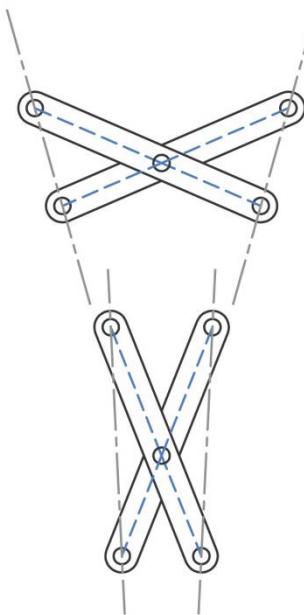
Sergio Pellegrino



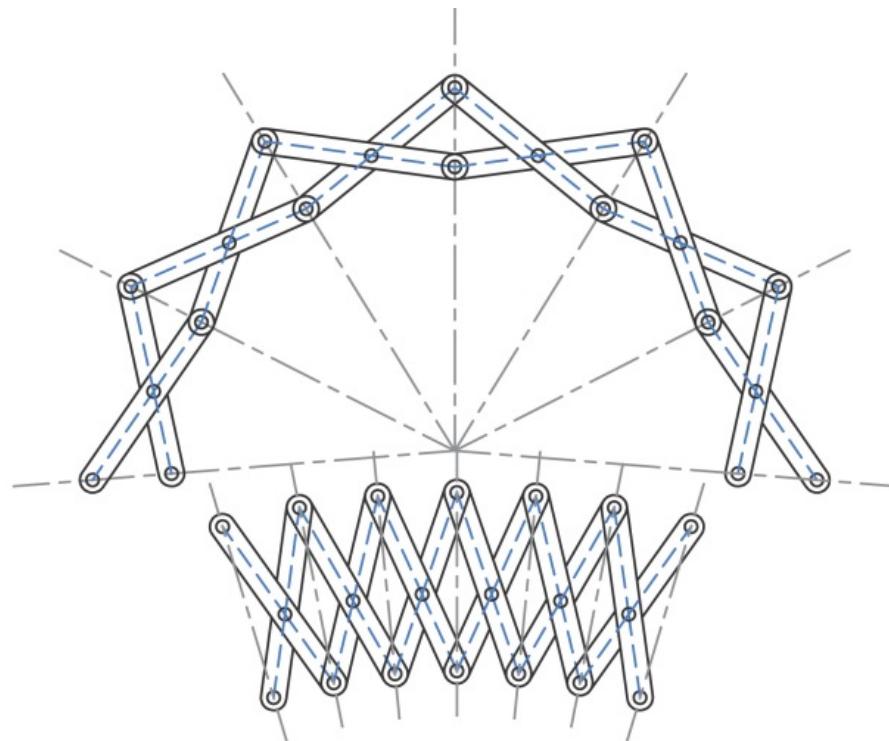
Felix Escrig

Curvature of scissor mechanisms

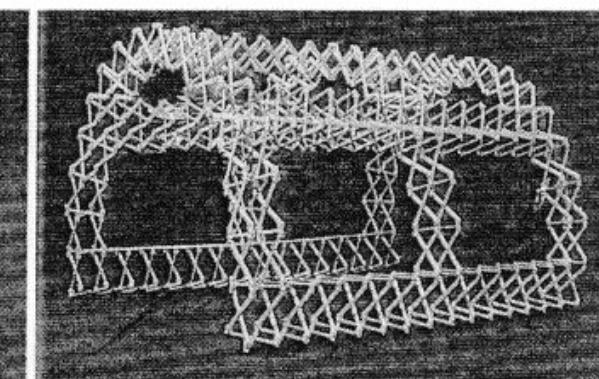
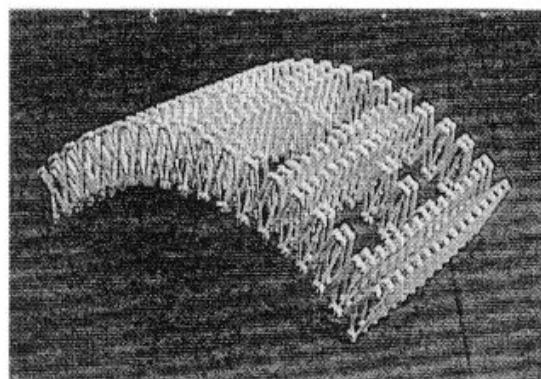
Off-center connection point => structures of variable curvature



(a)

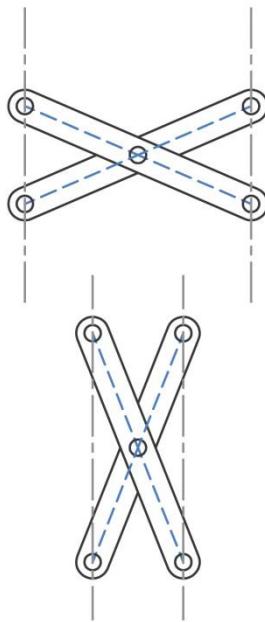


(b)



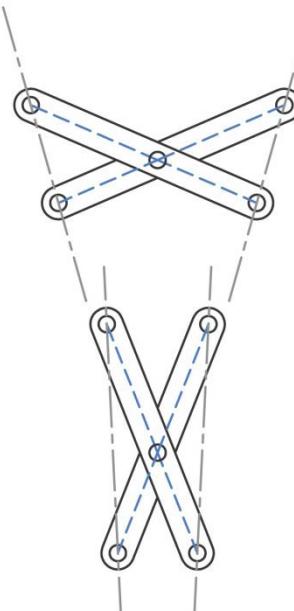
Scissor Types

Parallel / symmetric



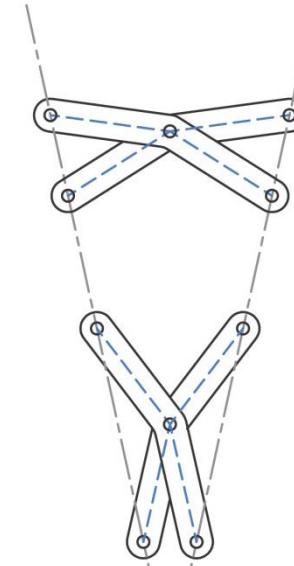
No curvature

Parallel / asymmetric

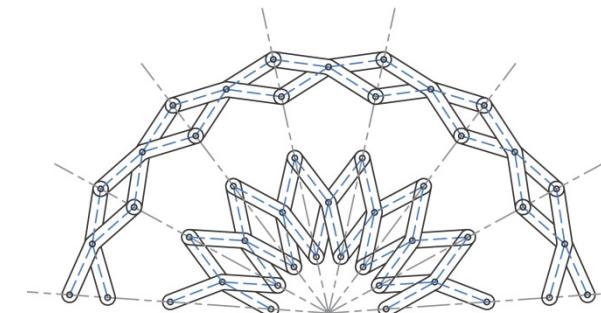
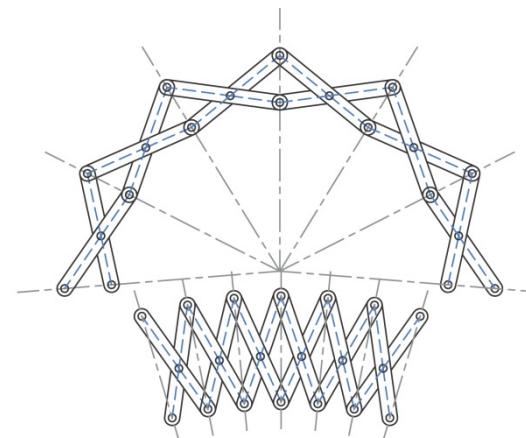
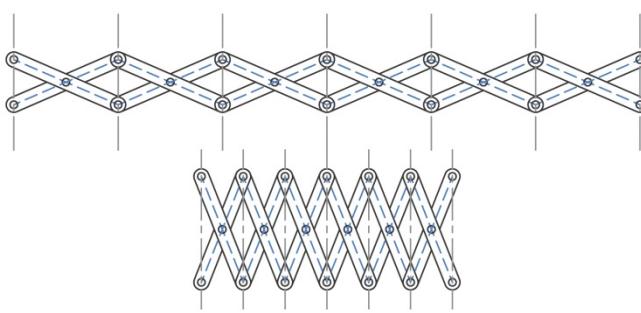


Variable curvature

Angulated

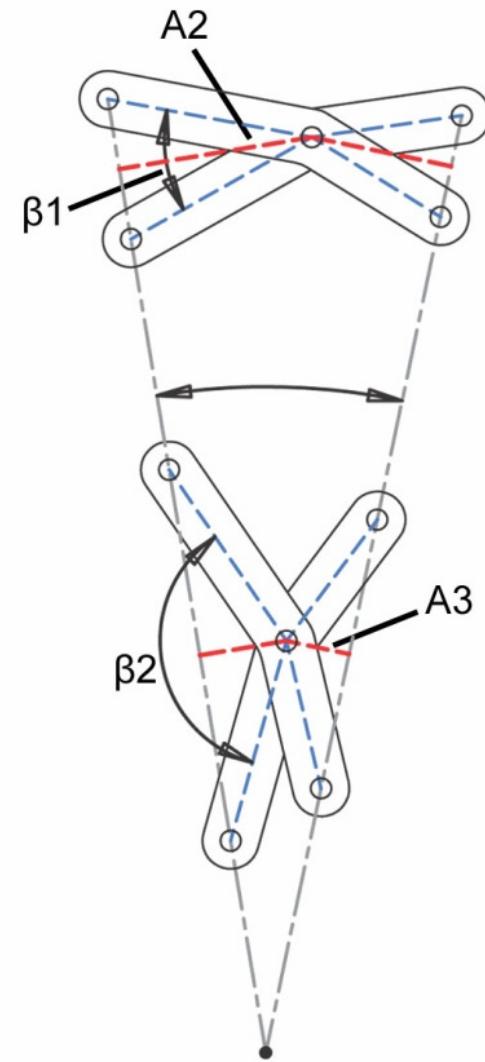
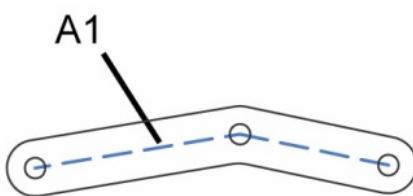
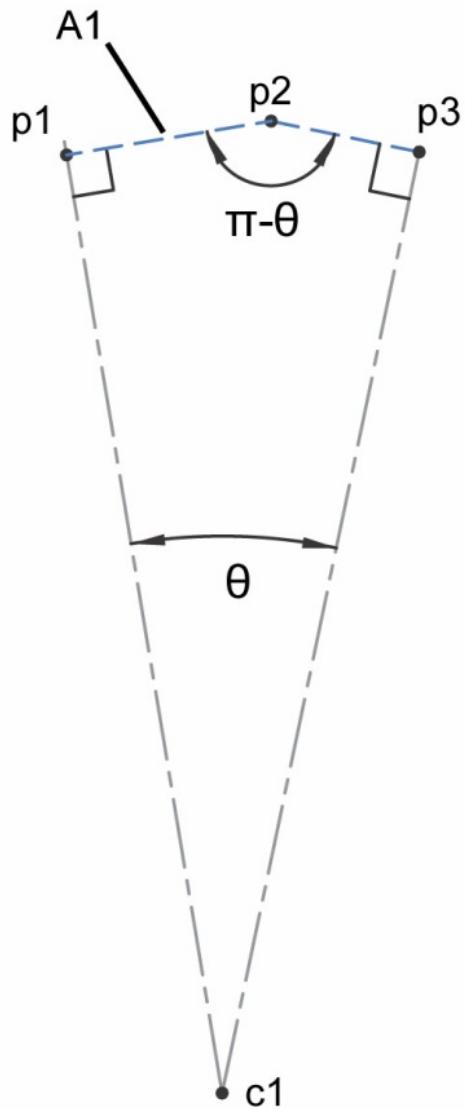


Constant curvature



Angulated scissors

Provides invariant angle during deployment



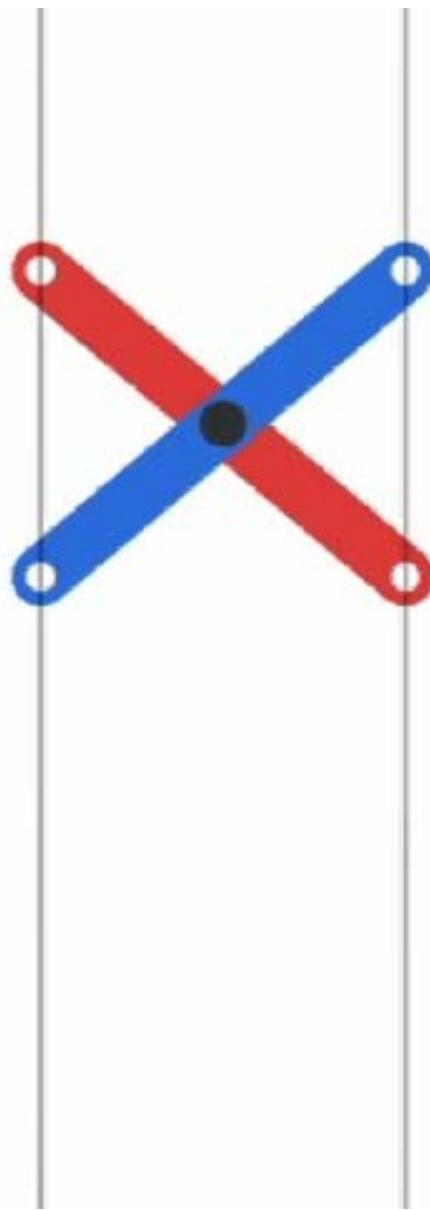
Scissor mechanism: demonstration

Parallel / Symmetric



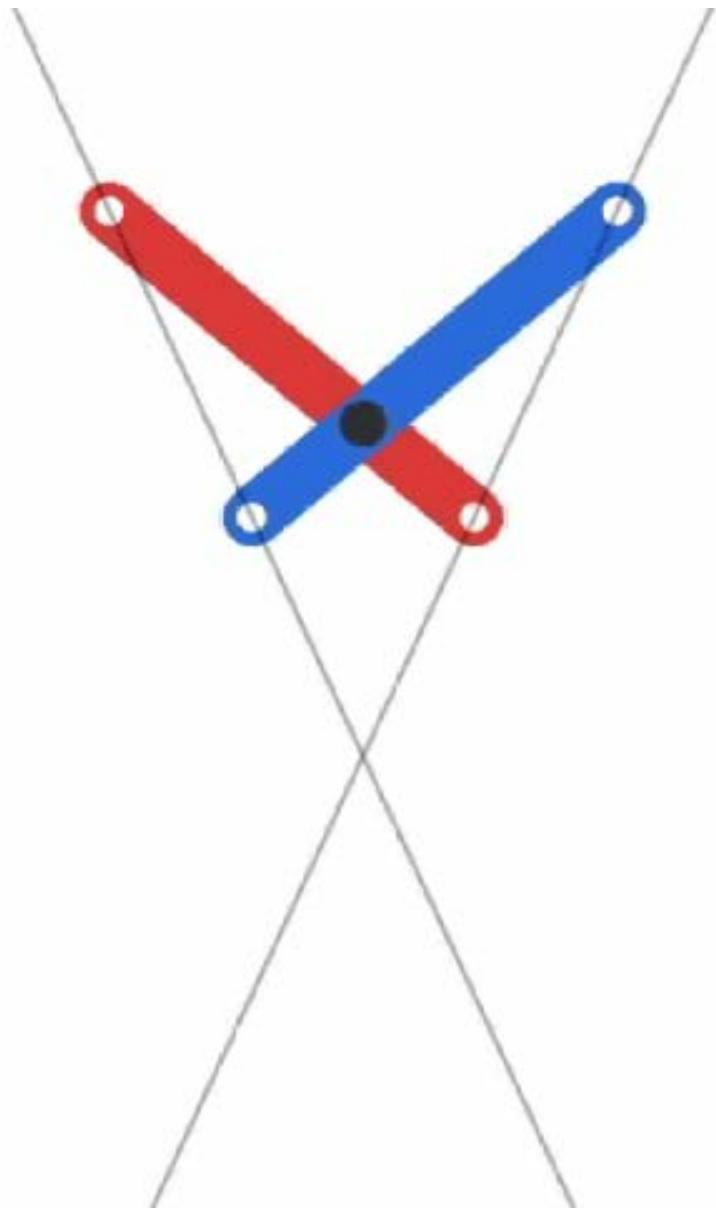
Scissor mechanism: demonstration

Off-center connection

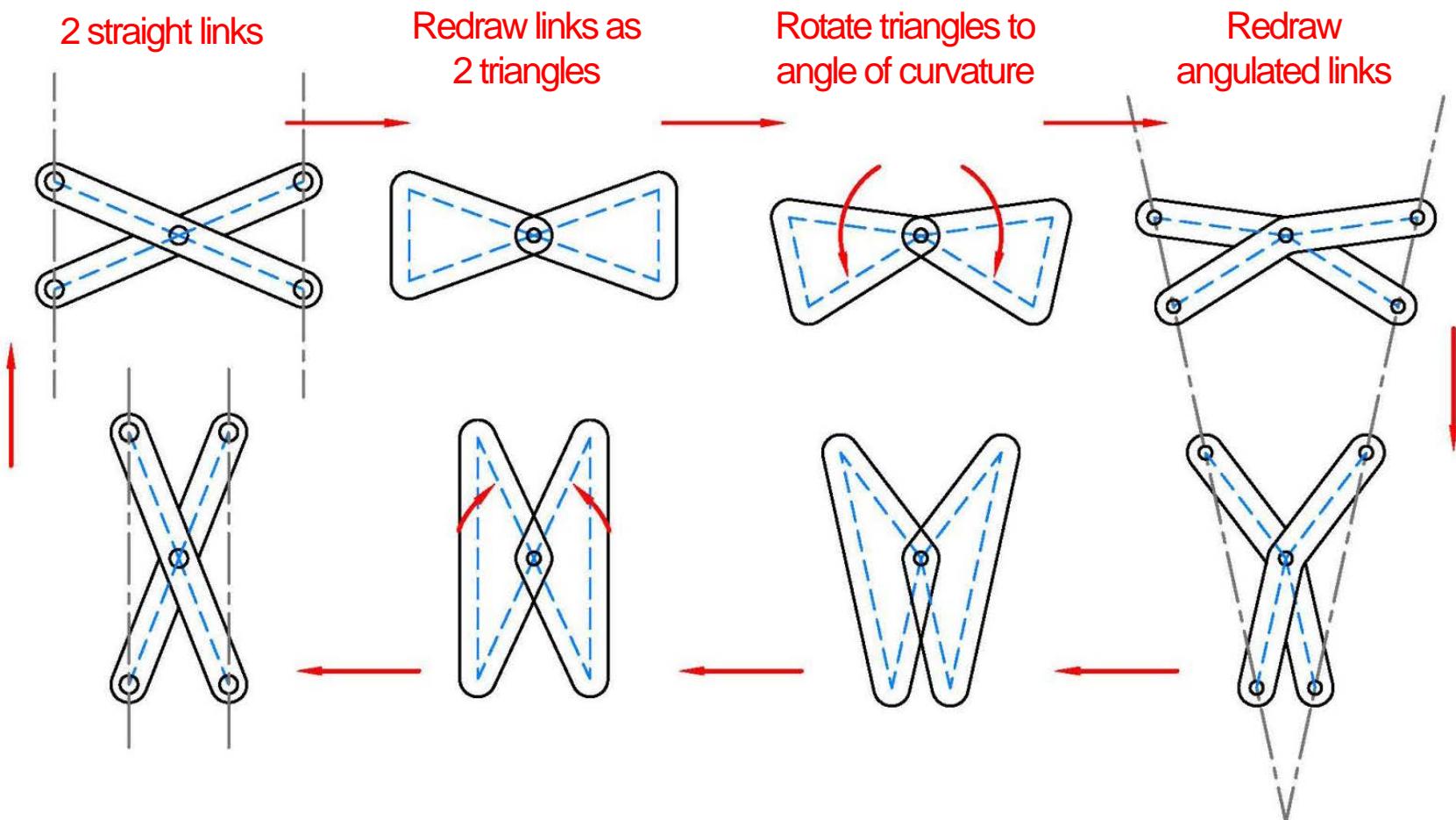


Scissor mechanism: demonstration

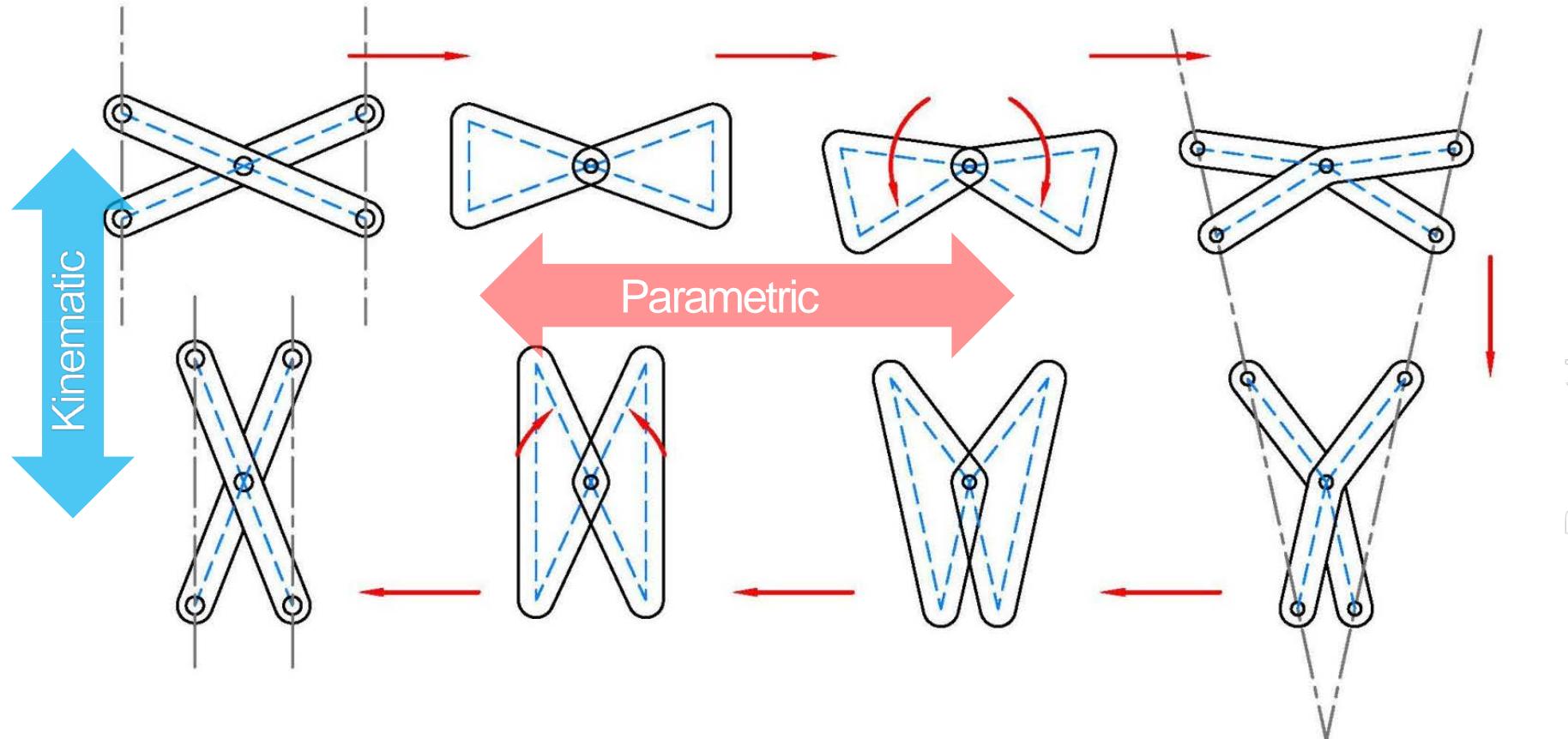
Angulated



Angulated link: geometric construction



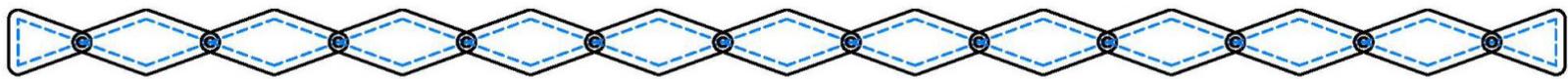
Angulated link: geometric construction



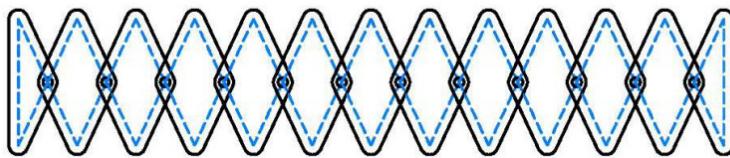
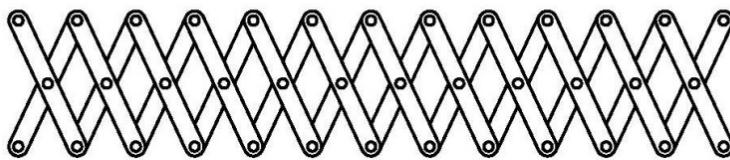
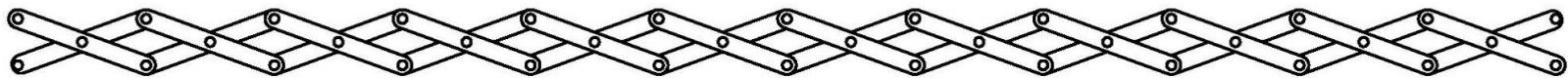
Parametric

Tong linkage

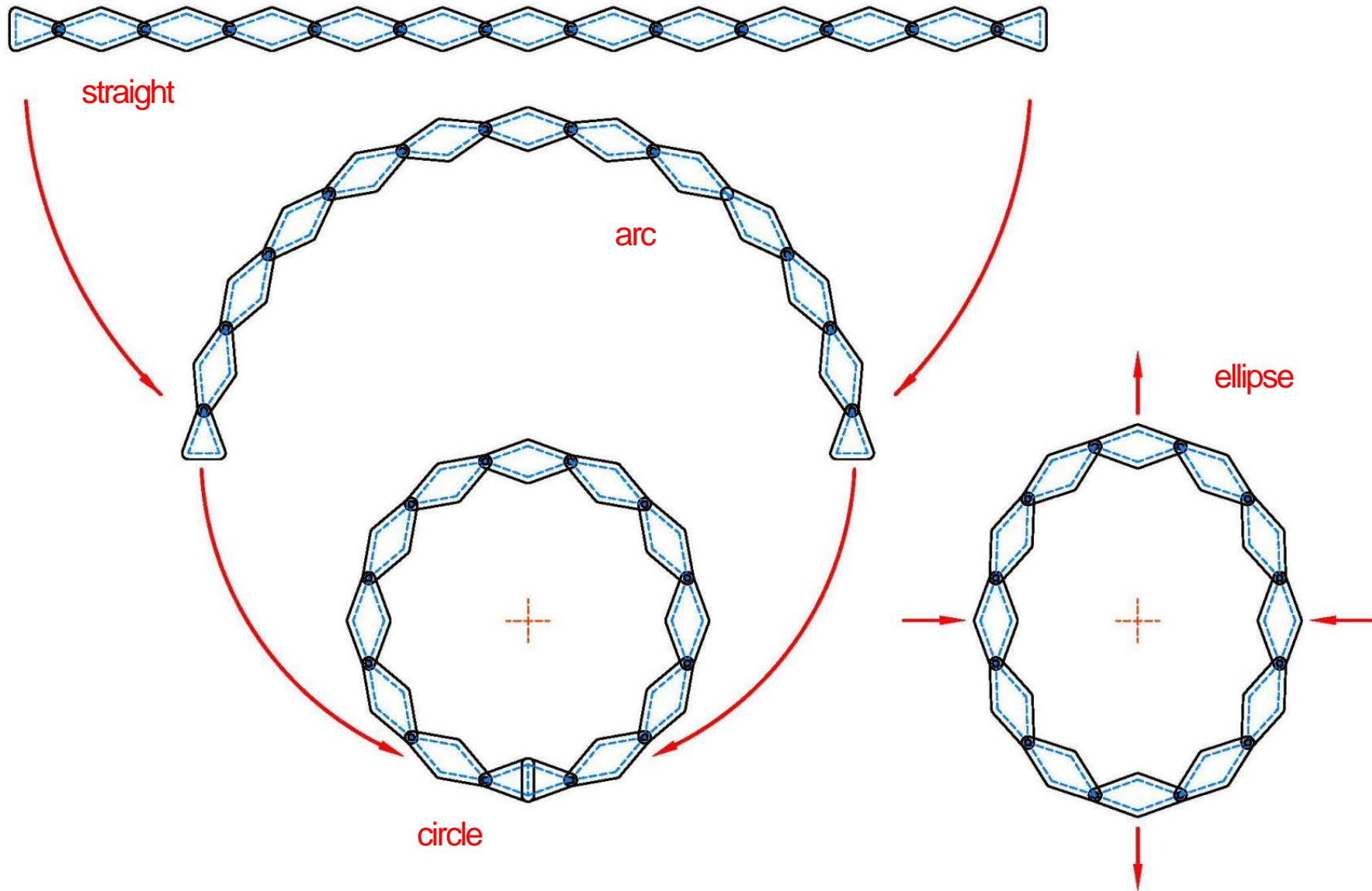
Hinged
rhombs



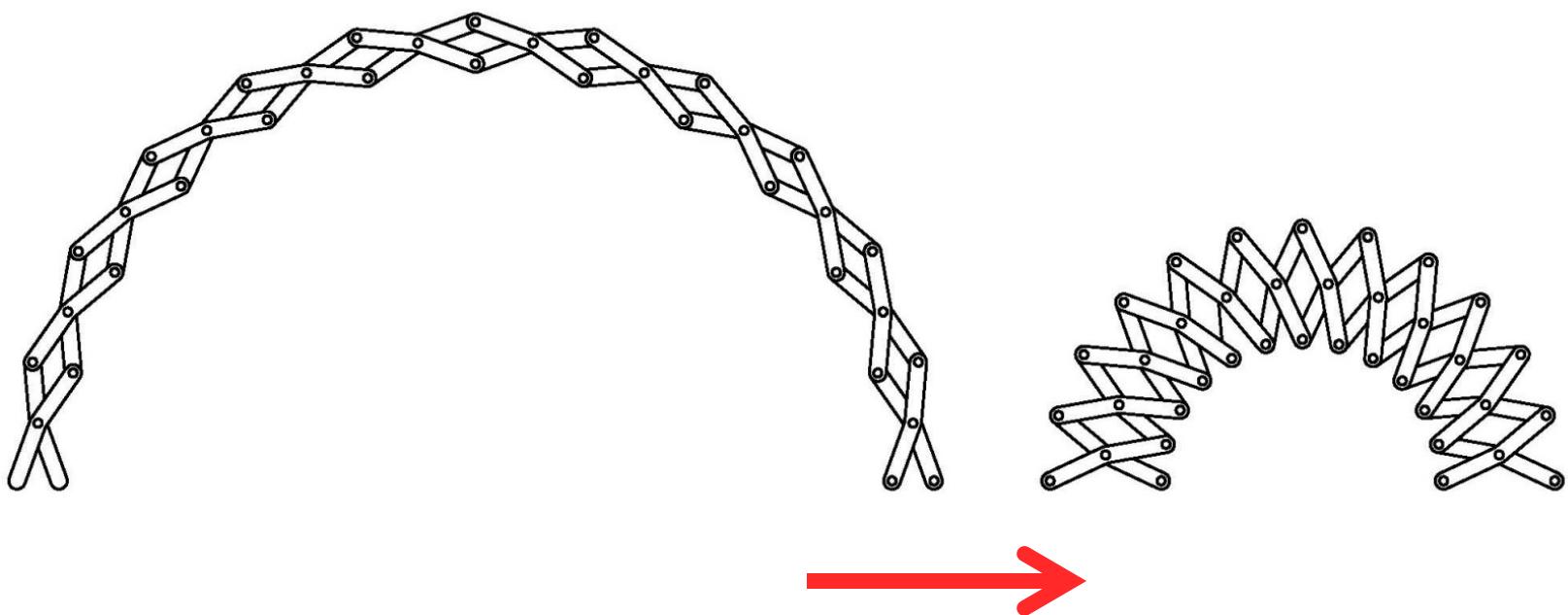
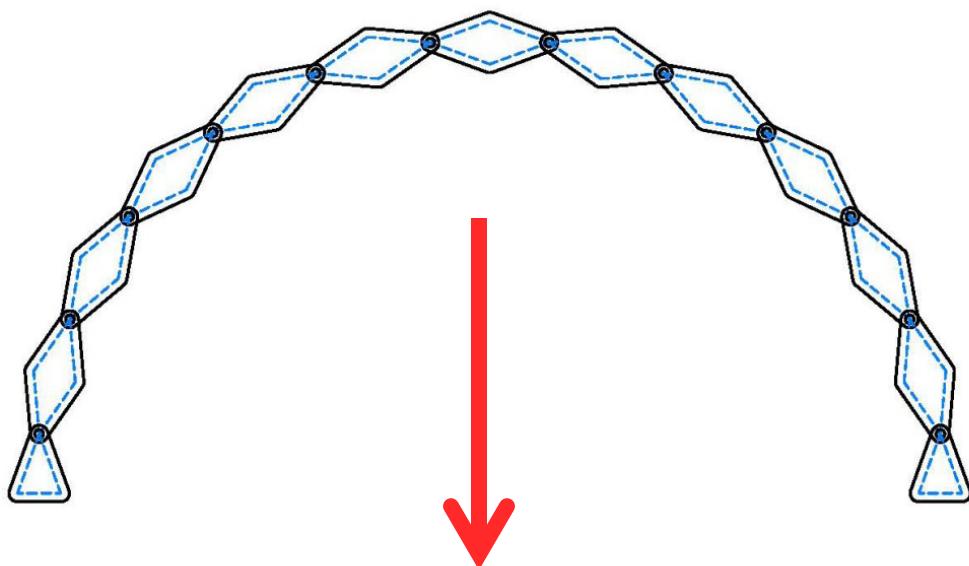
Tong
linkage



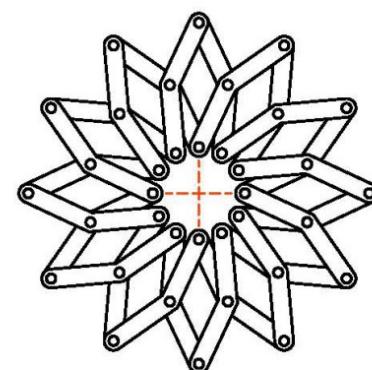
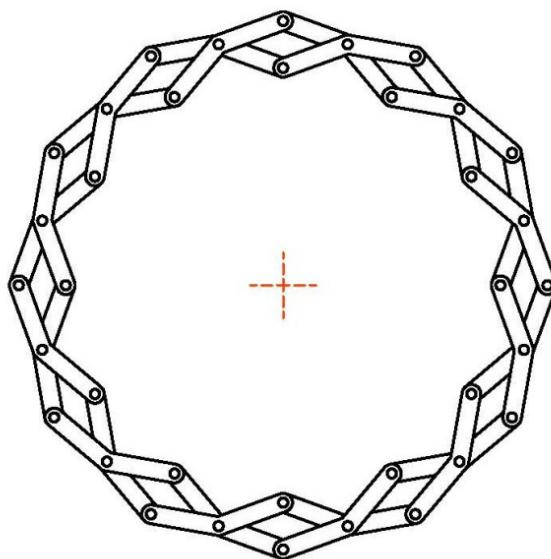
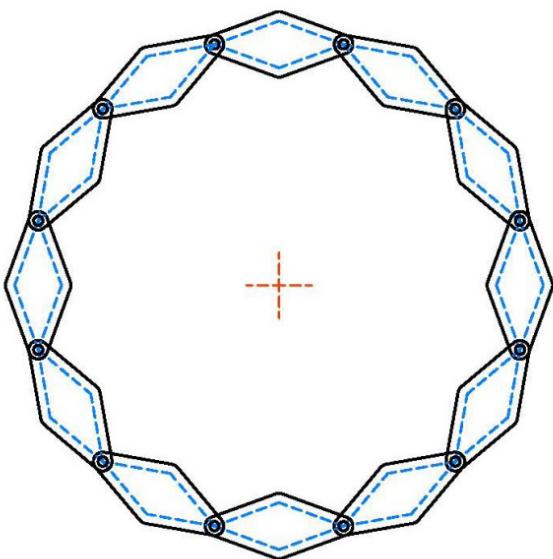
Hinged rhombs – transforming between configurations



Arc - geometric construction



Circle - geometric construction



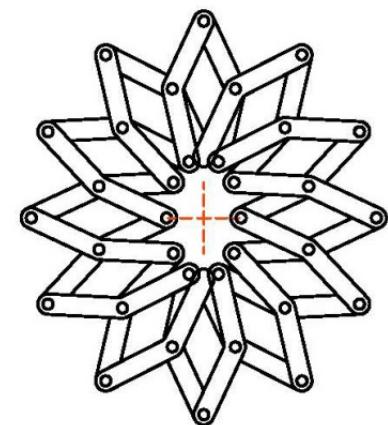
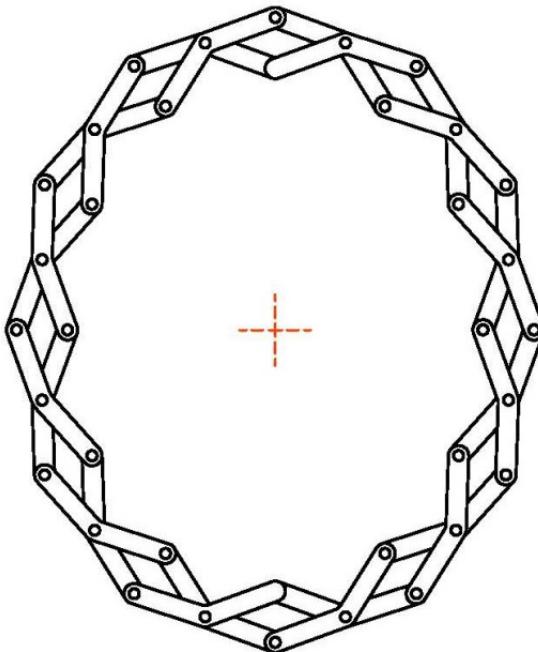
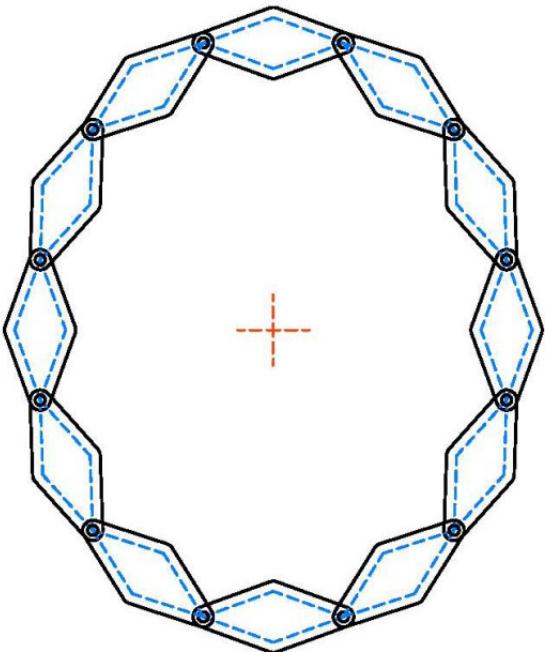
Ring linkages



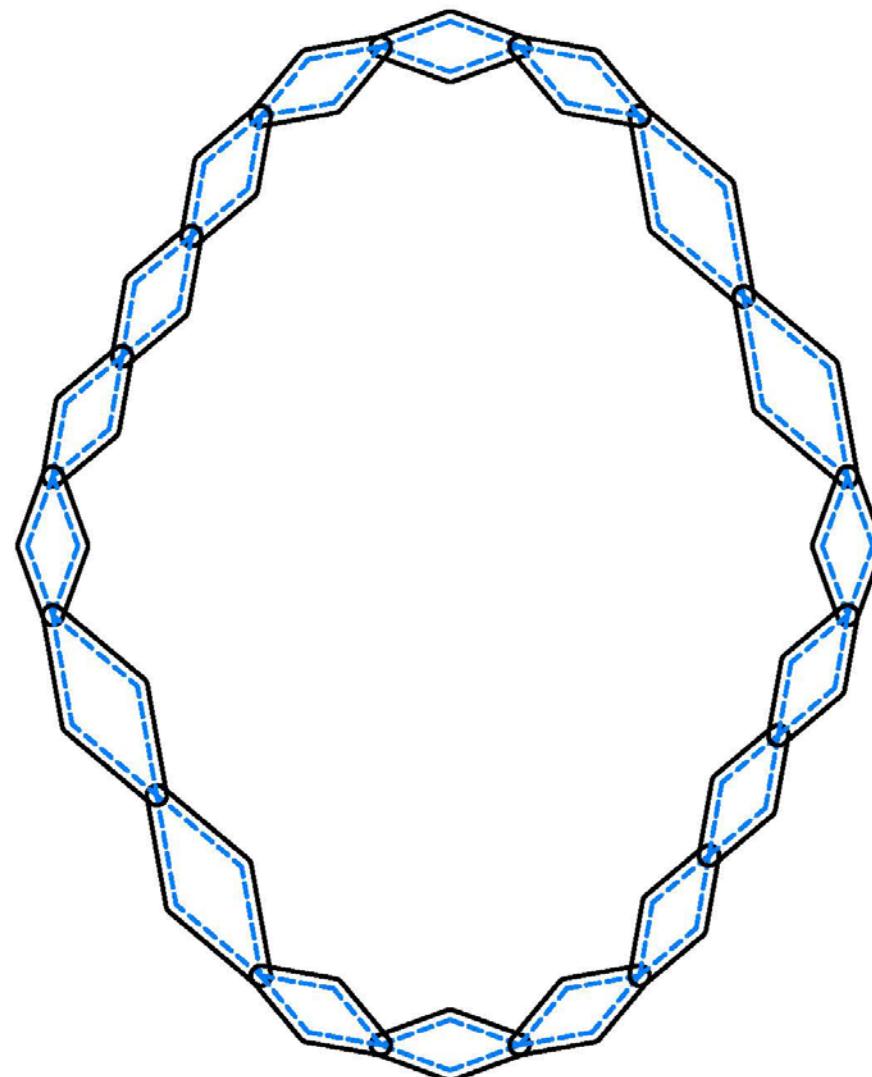
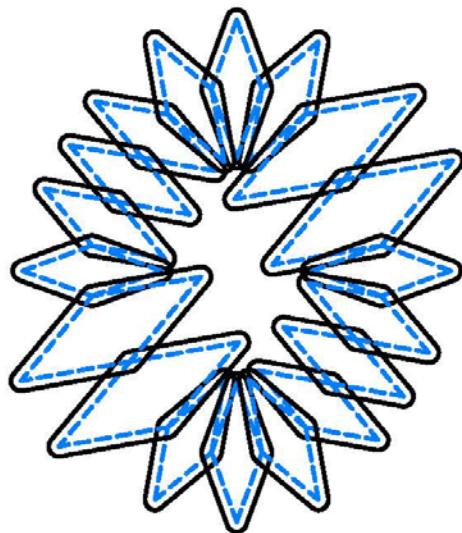
Ring linkages



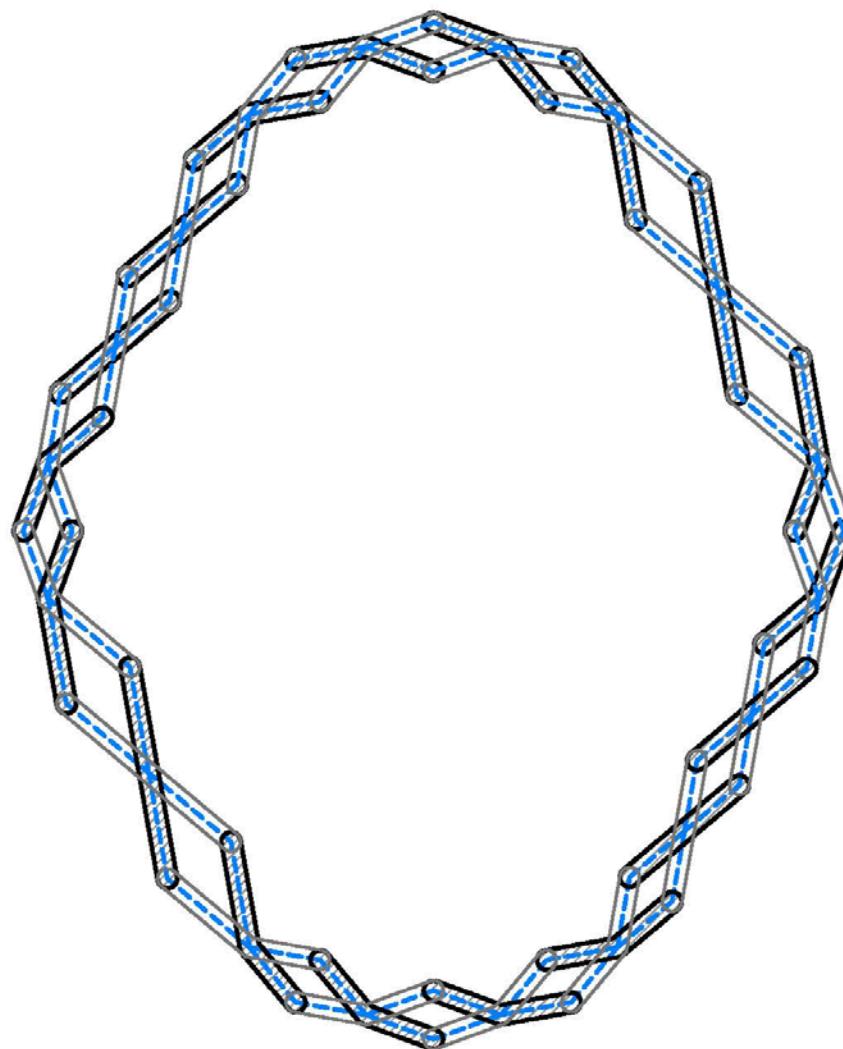
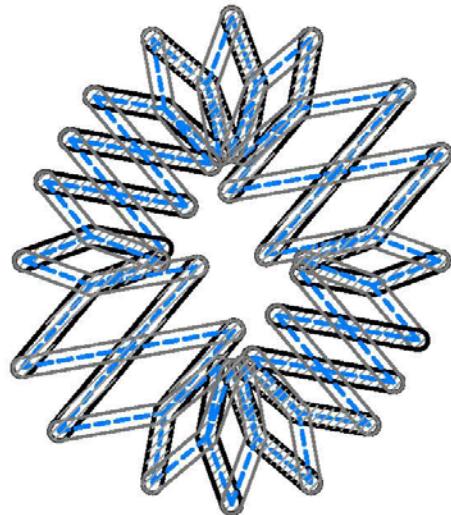
Ellipse - geometric construction



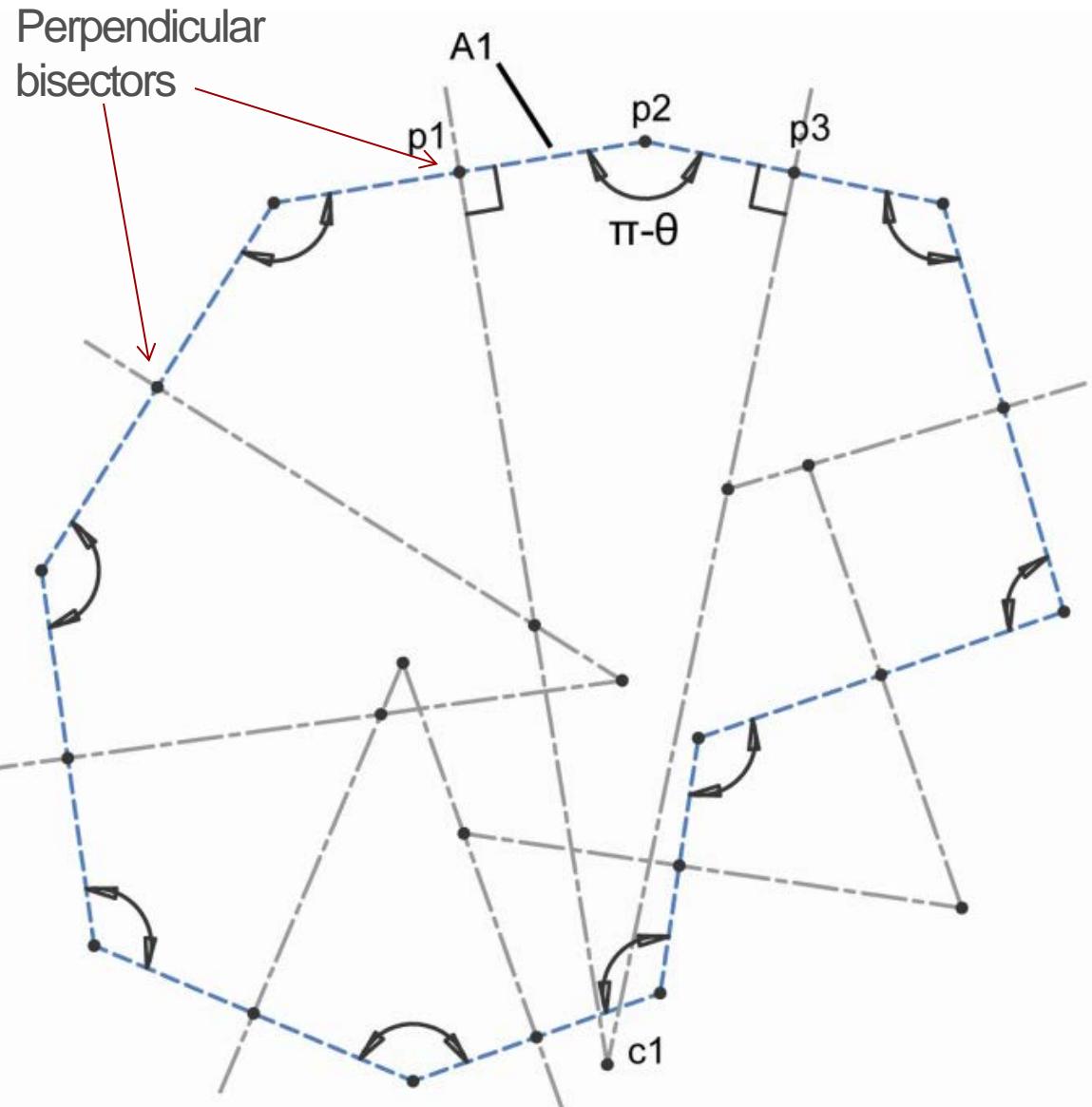
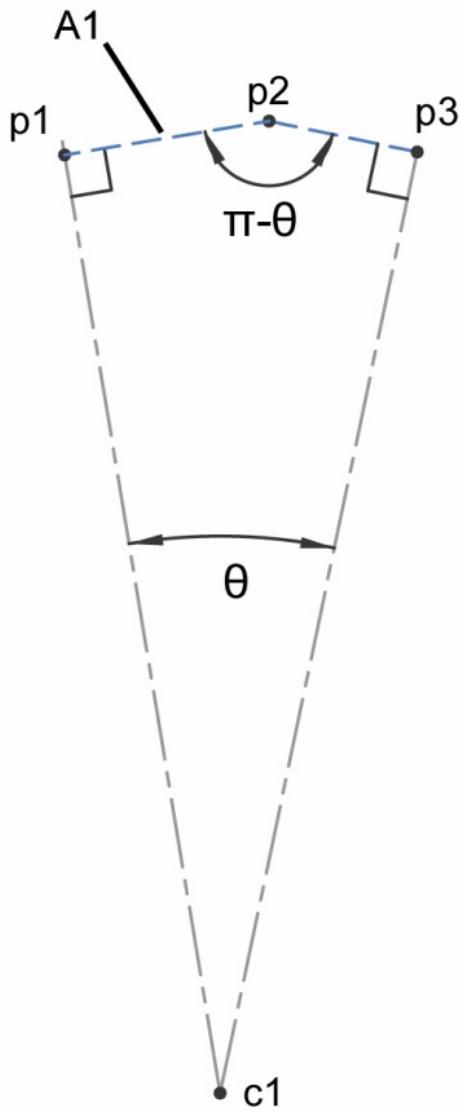
Unequal rhombs



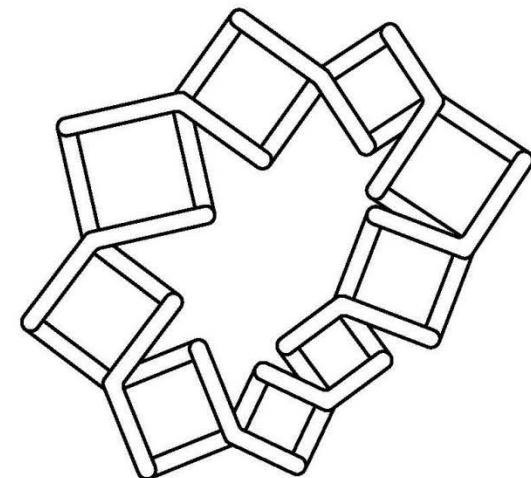
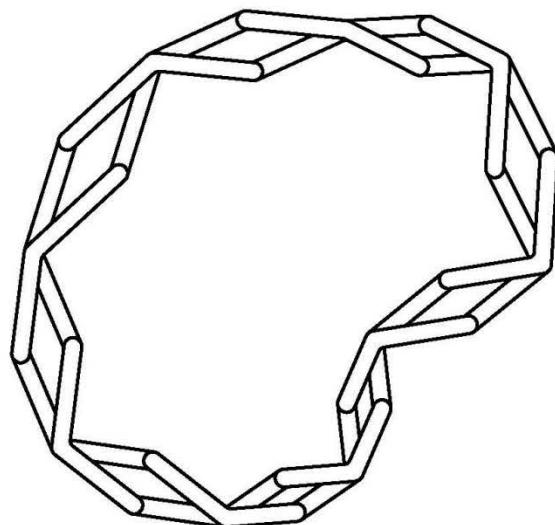
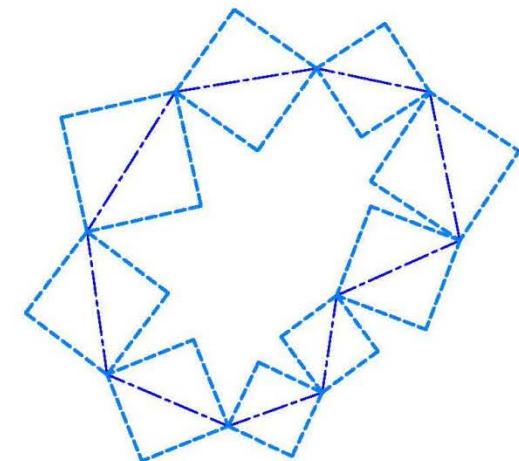
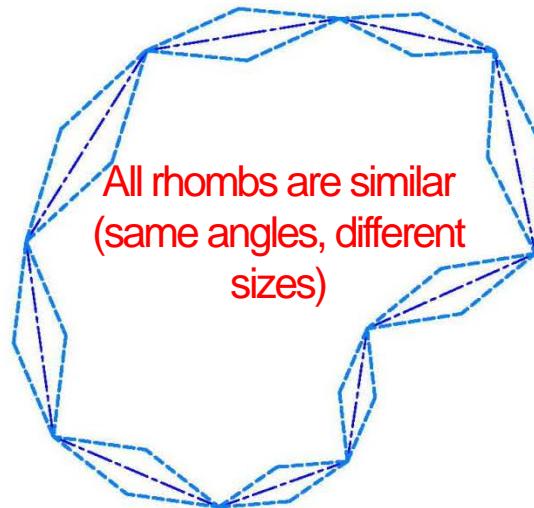
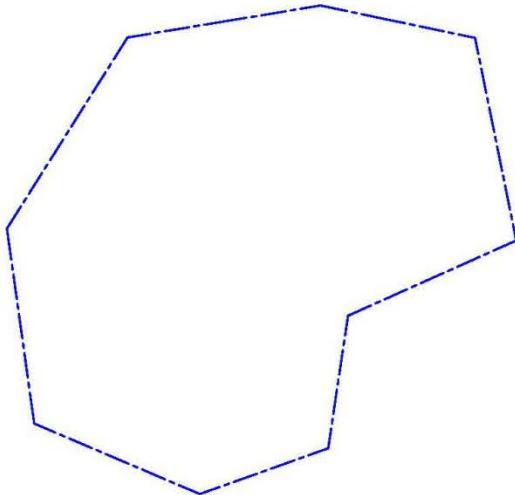
Unequal rhombs



Constructing expanding polygons



Irregular polygon – geometric construction

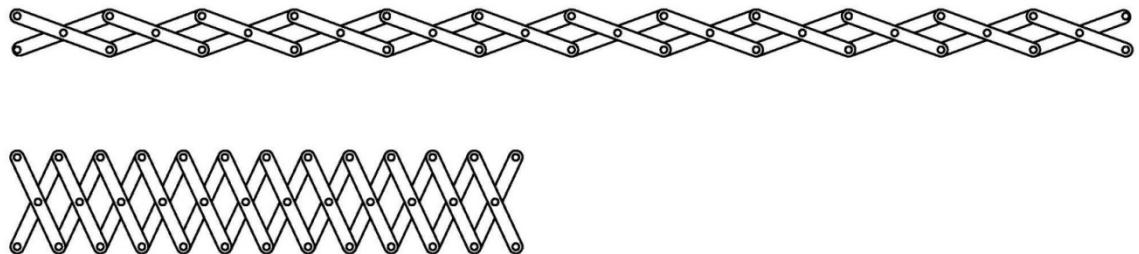


Degrees of freedom of a tong linkage

Number of pivots for a tong linkage:

$$P = 3N/2 - 2$$

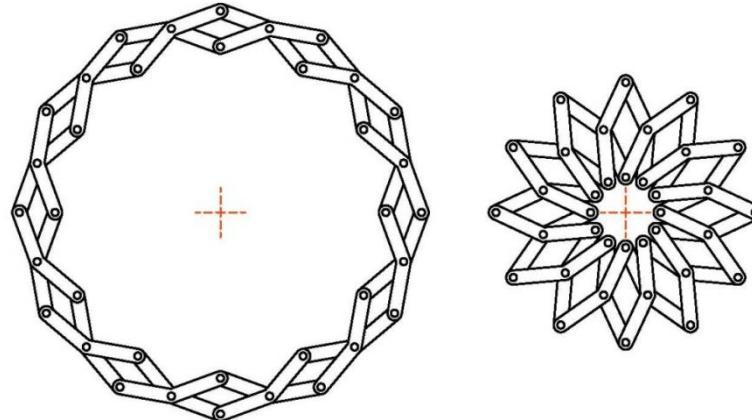
$$\begin{aligned} \text{DOF} &= 3 \times (N-1) - 2P \\ &= 3N - 3 - (3N - 4) = \mathbf{1} \end{aligned}$$



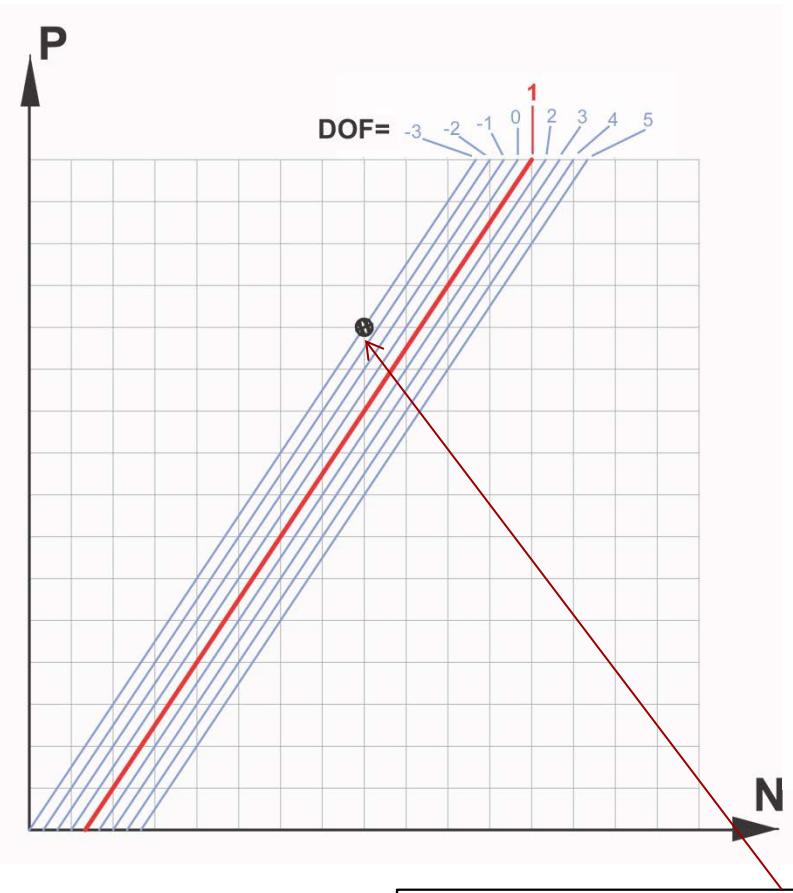
Number of pivots for a closed tong linkage:

$$P = 3N/2$$

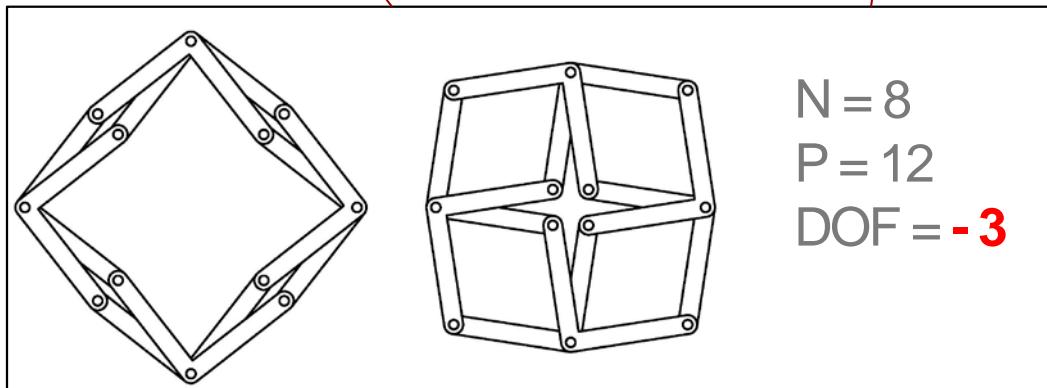
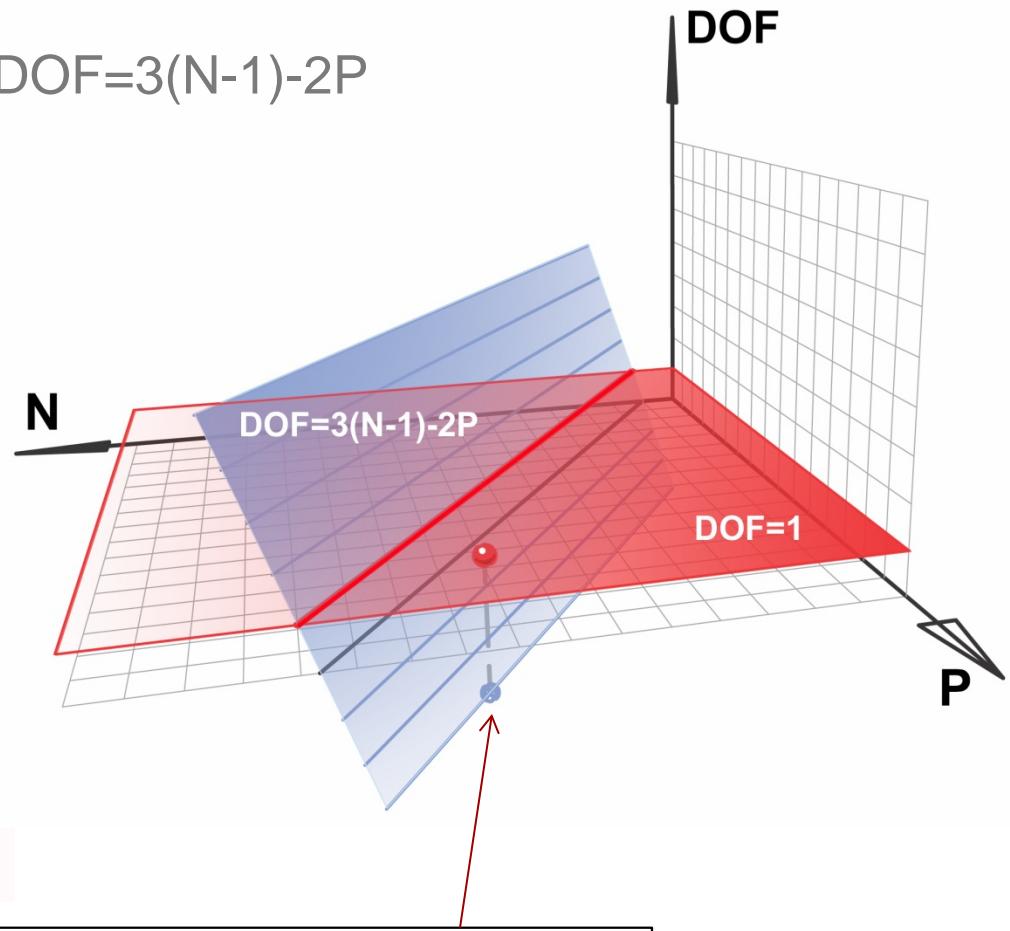
$$\begin{aligned} \text{DOF} &= 3 \times (N-1) - 2P \\ &= 3N - 3 - 3N = \mathbf{-3} \end{aligned}$$



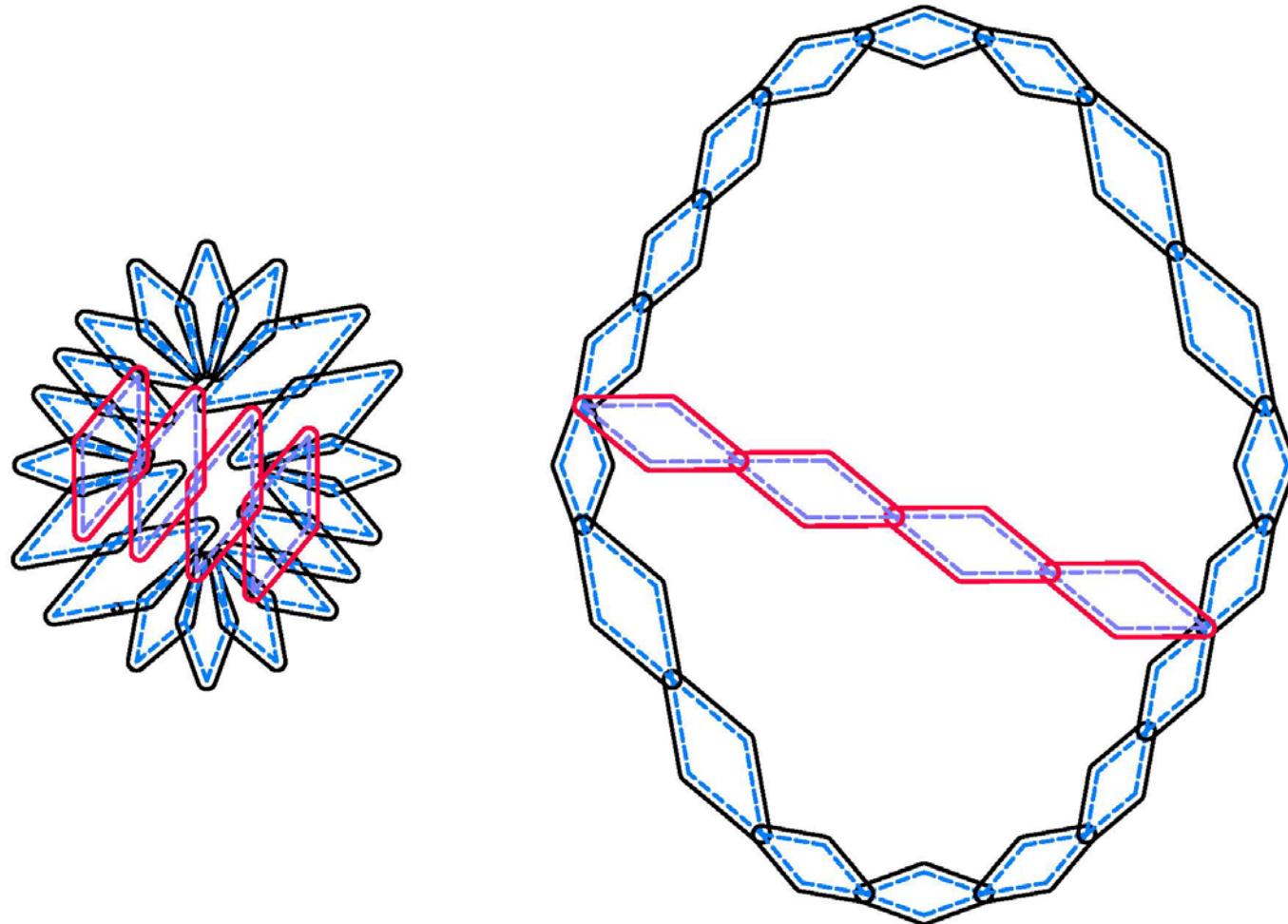
Spatial interpretation of Gruebler's equation



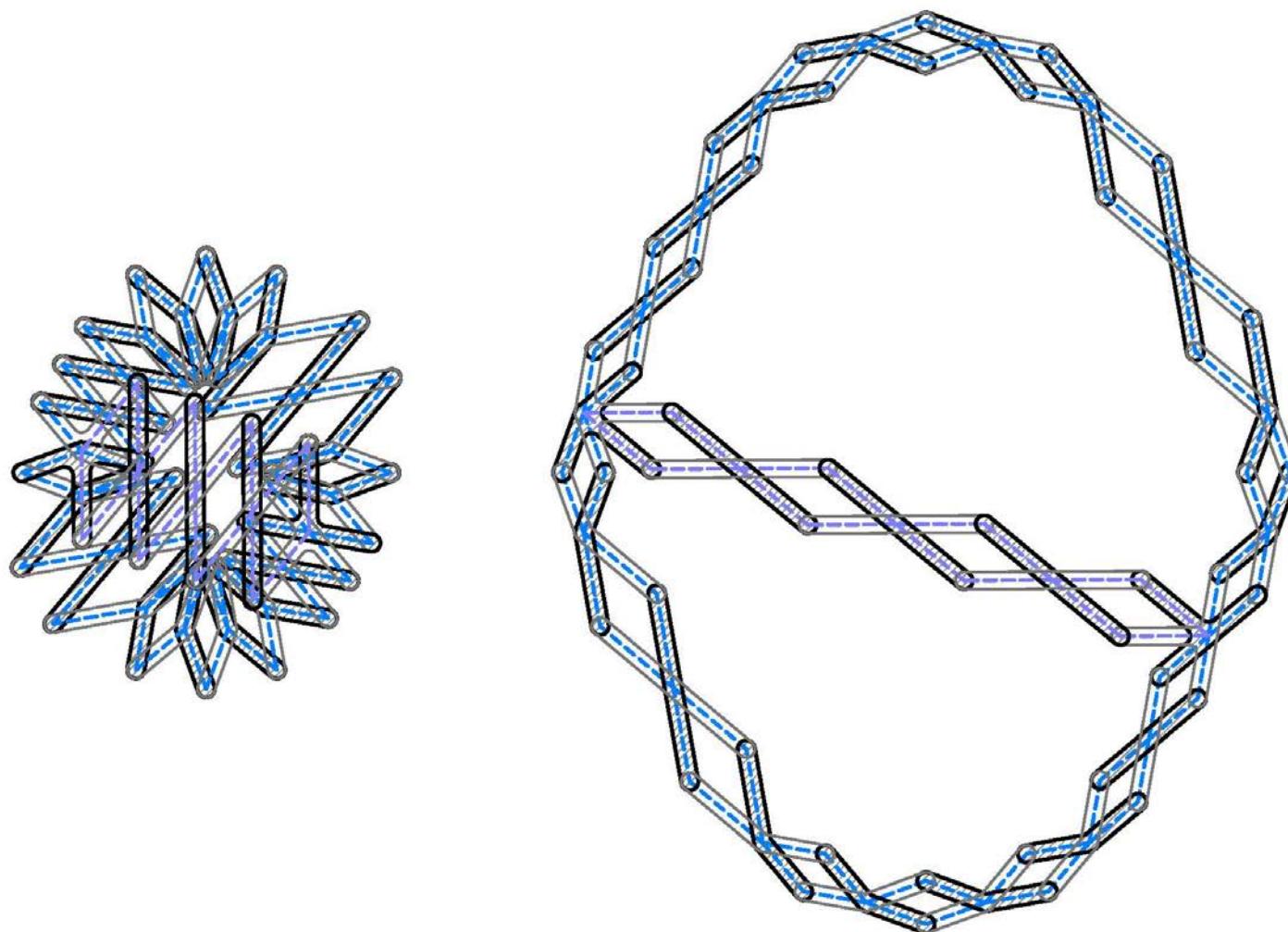
$$DOF = 3(N-1)-2P$$



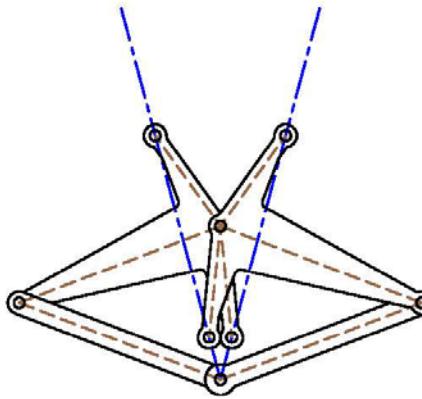
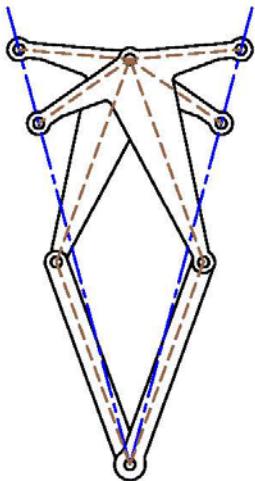
Unequal rhombs with crossing connection



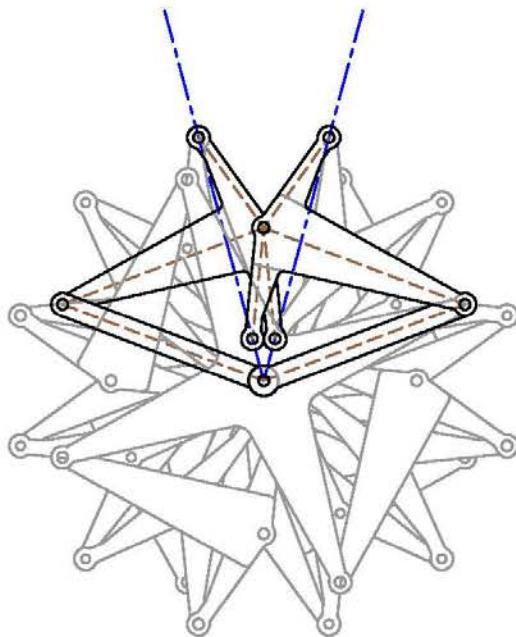
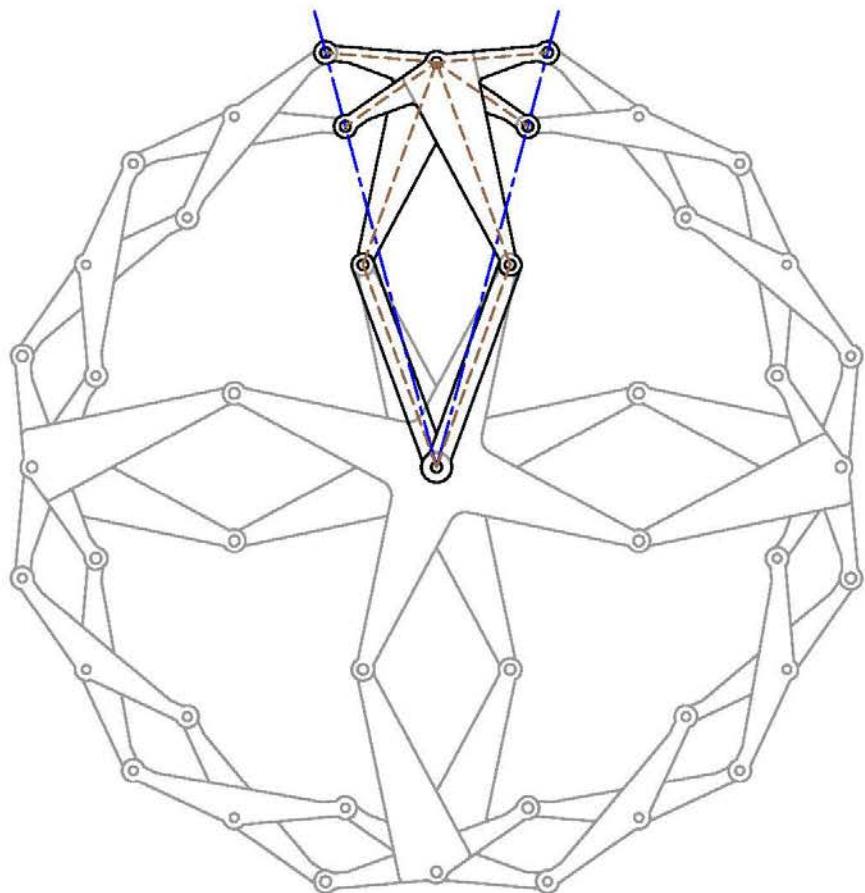
Unequal rhombs with crossing connection



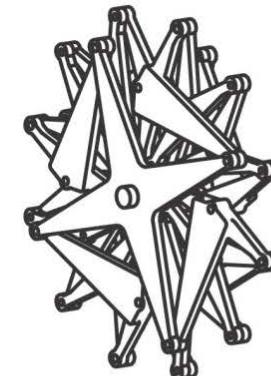
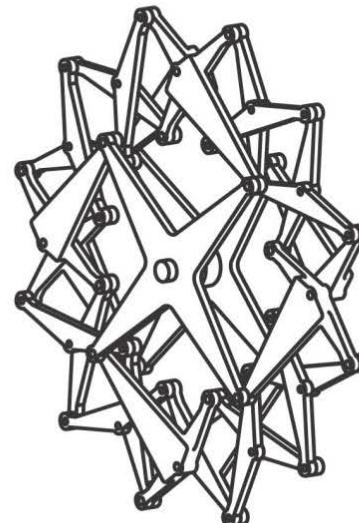
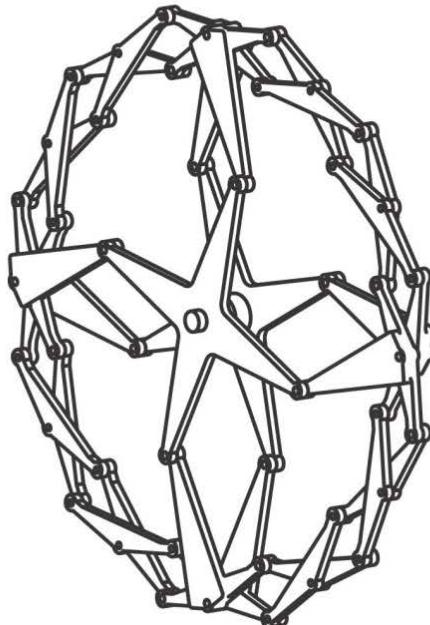
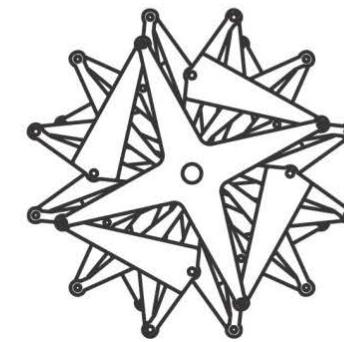
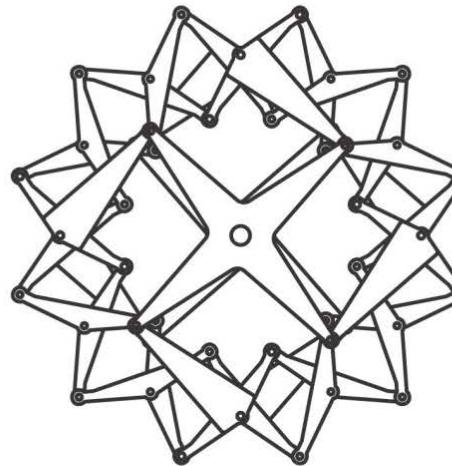
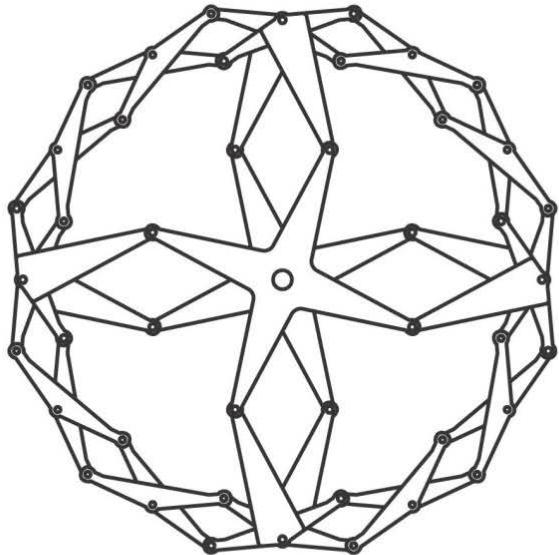
Polygon linkages with fixed centers



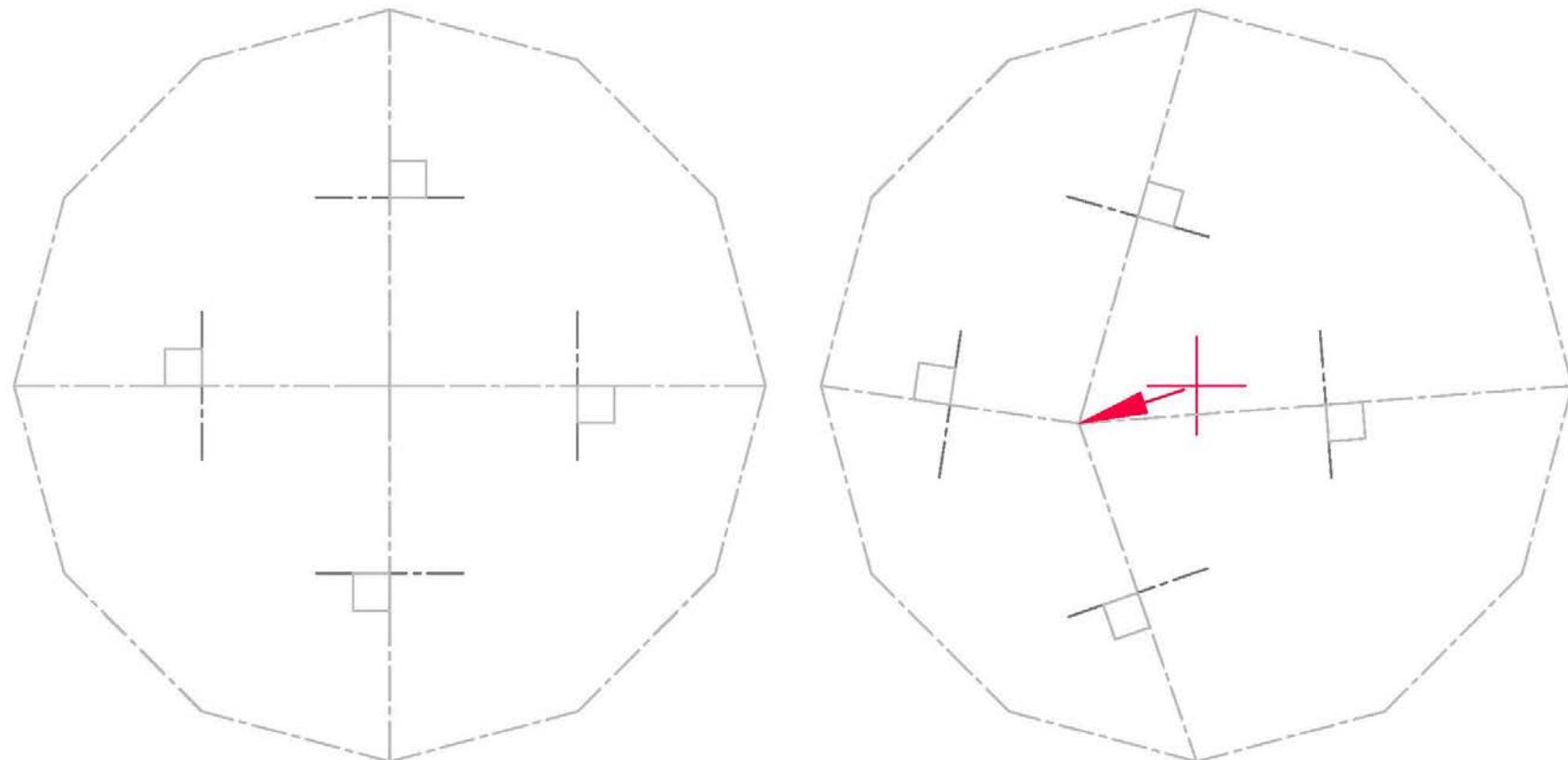
circular linkage with fixed center (four spokes)



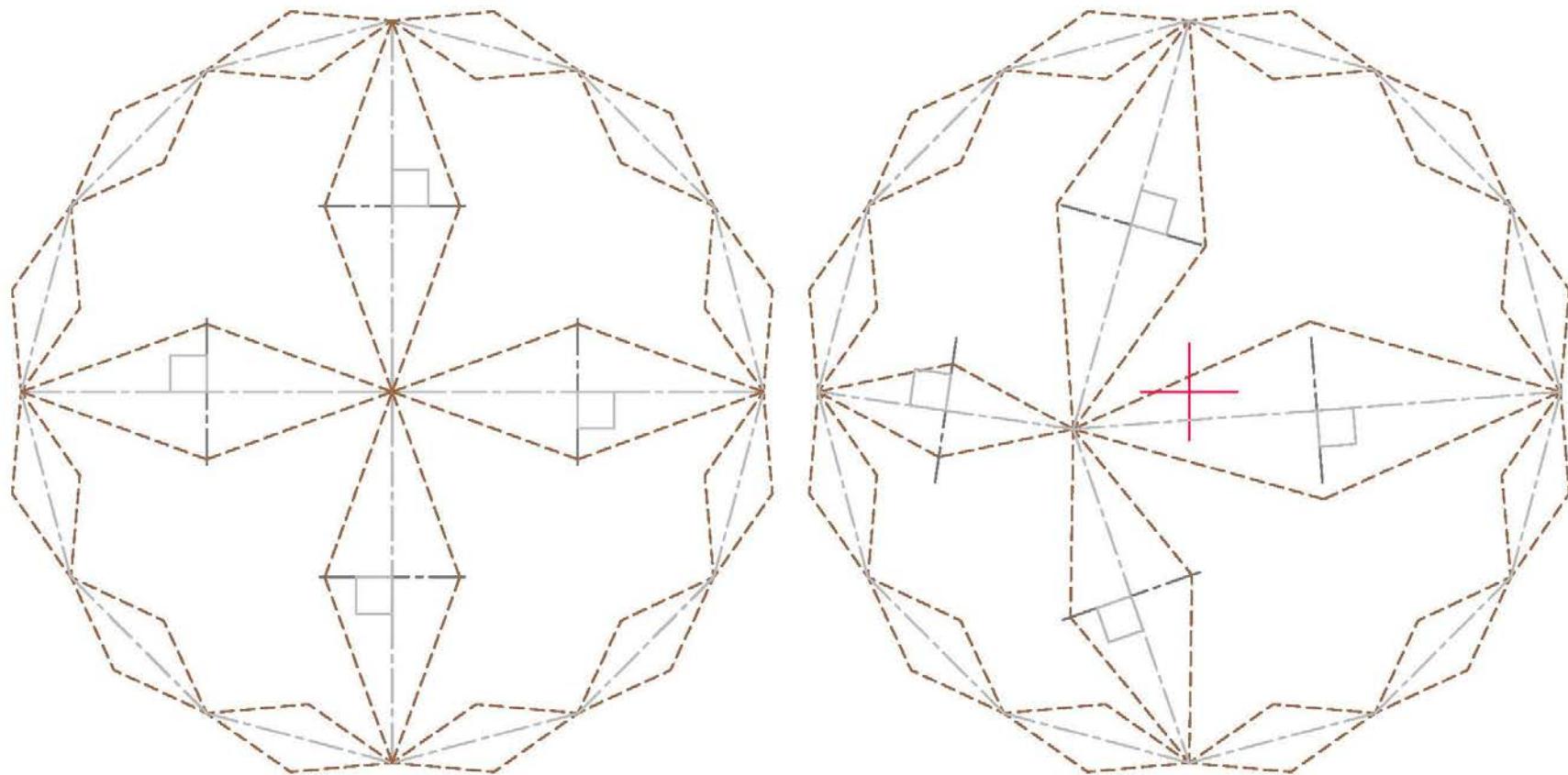
Circular linkages with fixed center



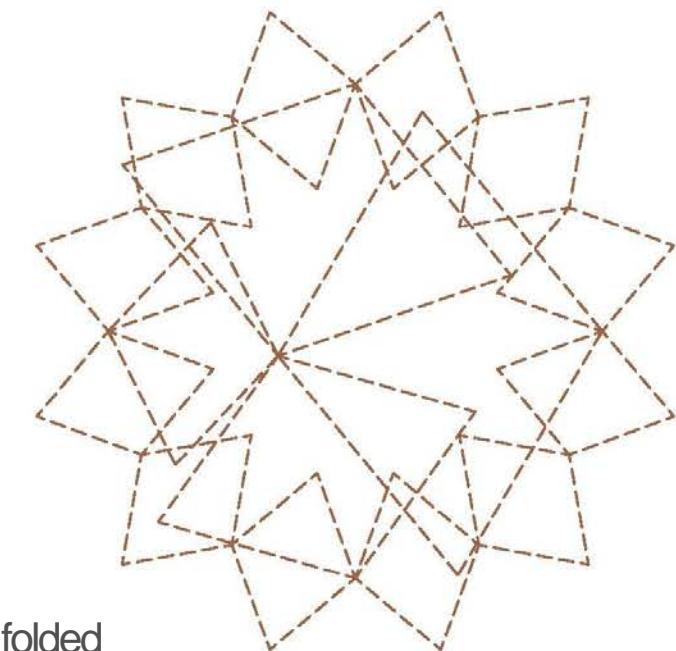
Construction for off-center fixed point



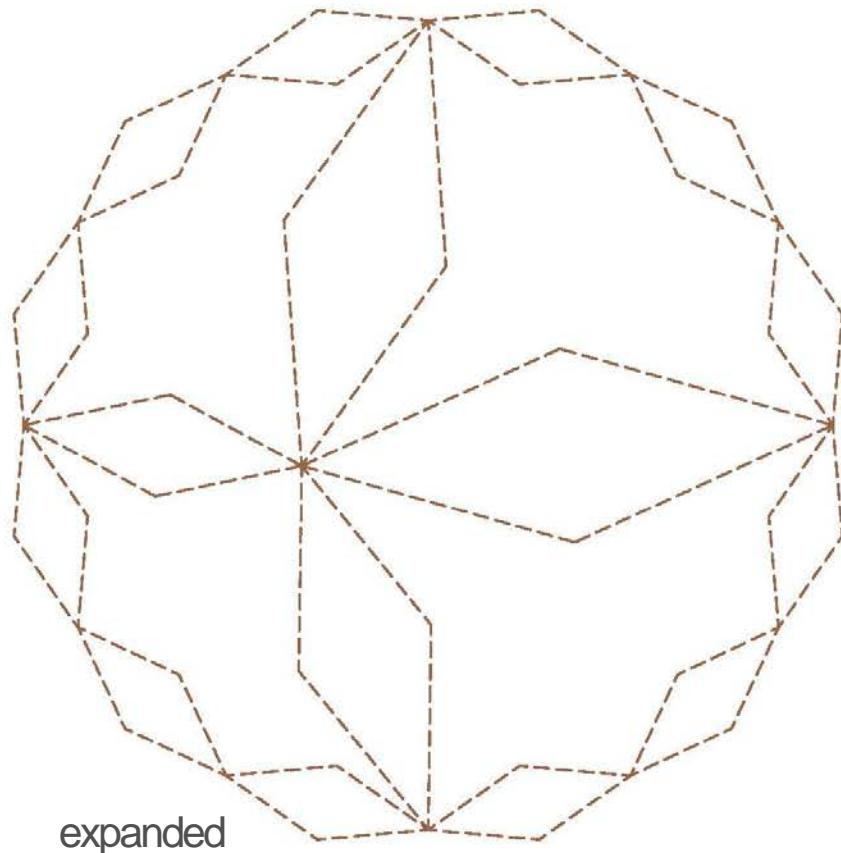
Polygon linkages – off-center fixed point



Polygon linkages – off-center fixed point

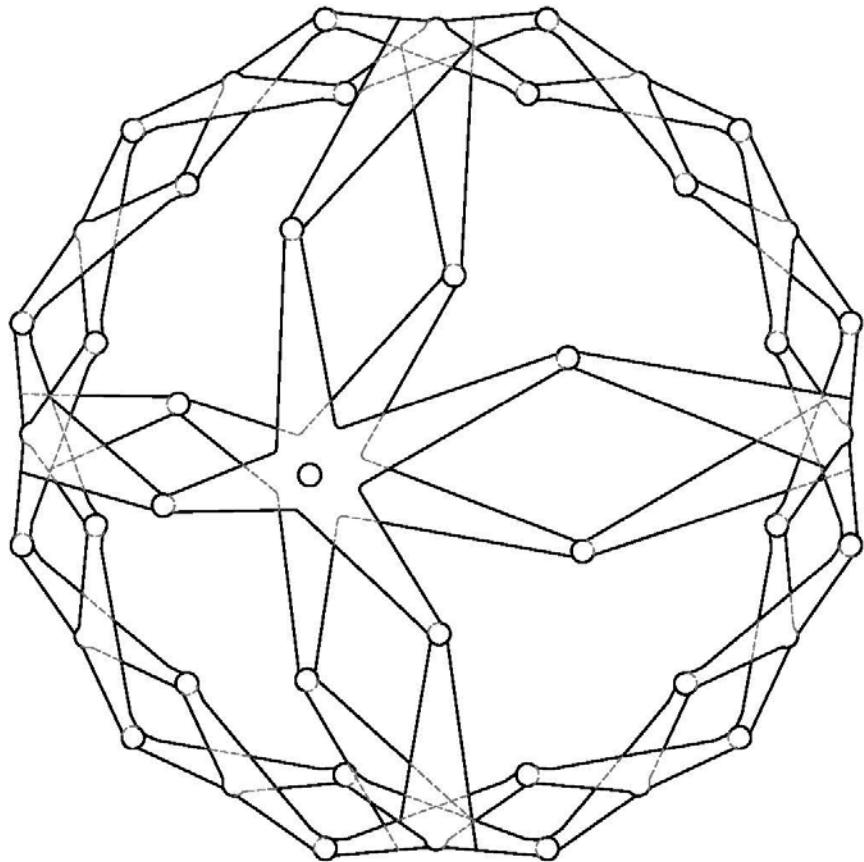
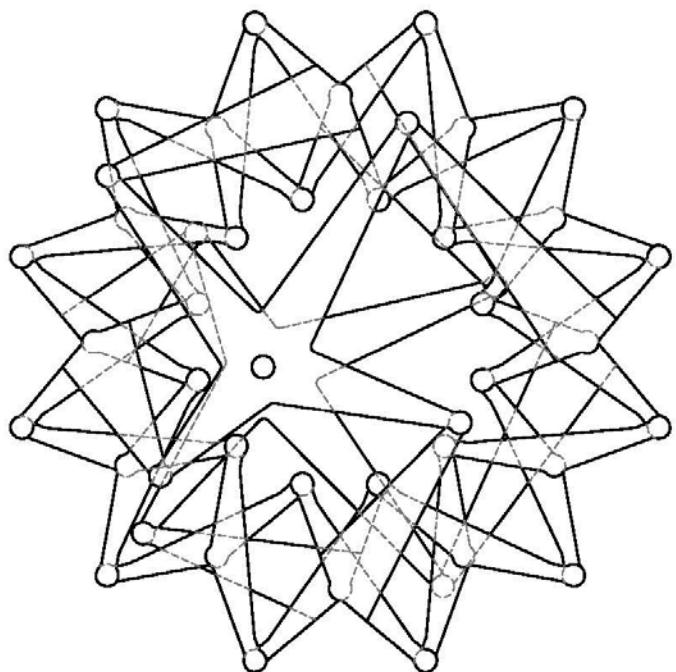


folded

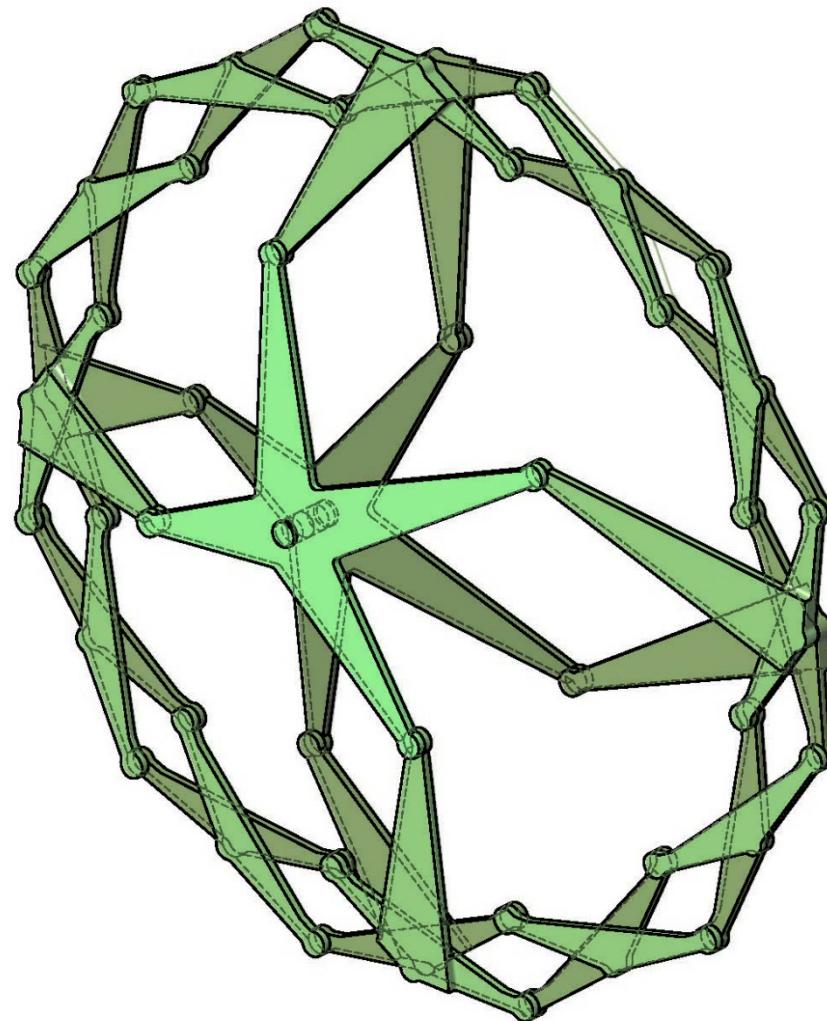
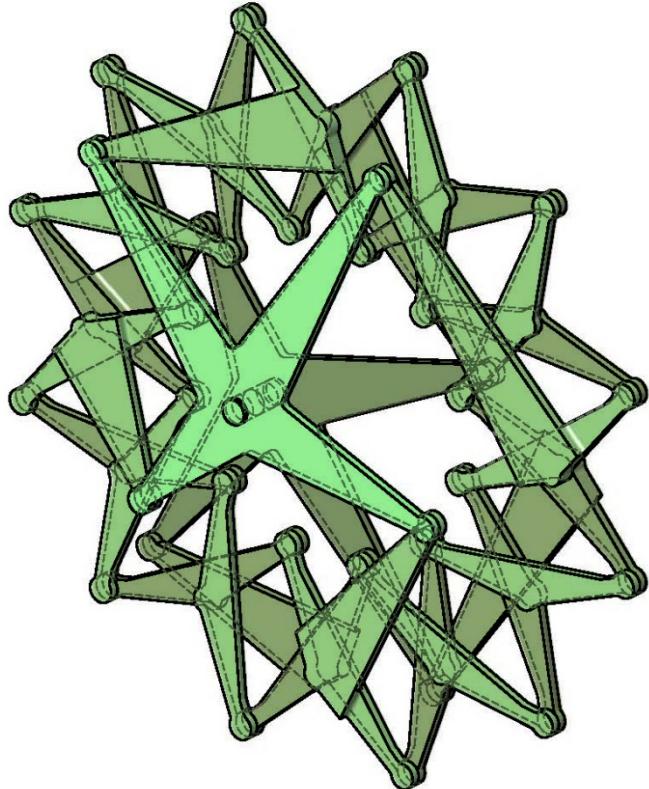


expanded

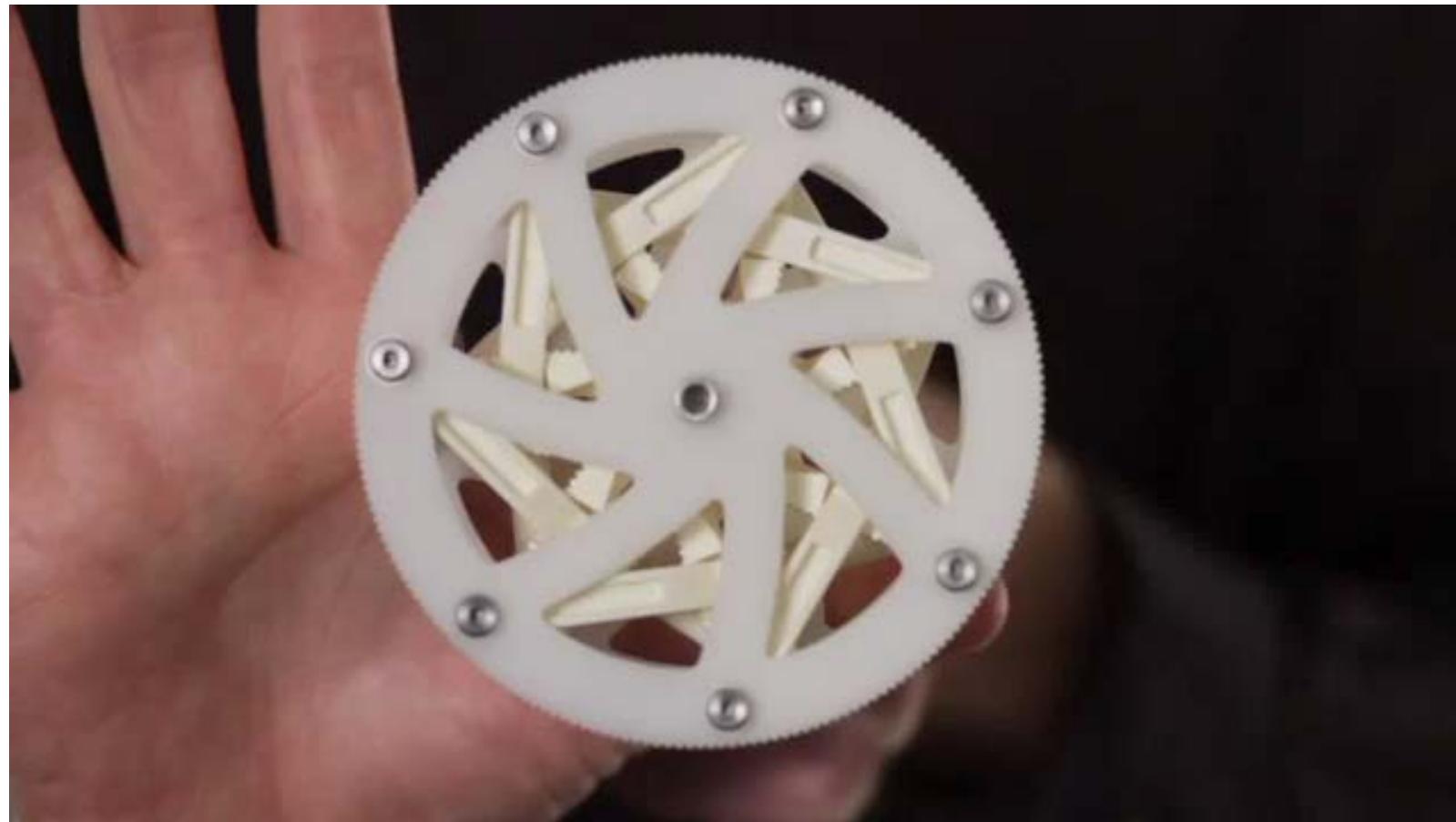
Polygon linkages – off-center fixed point



Polygon linkages – off-center fixed point



Ring linkages



Degrees of freedom (Graver formulation)

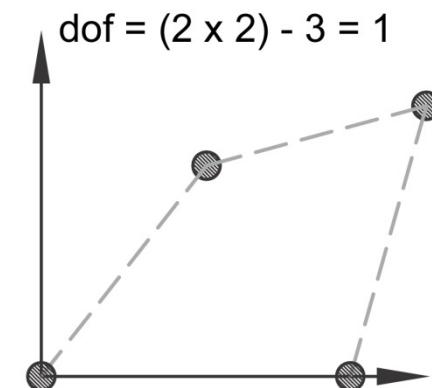
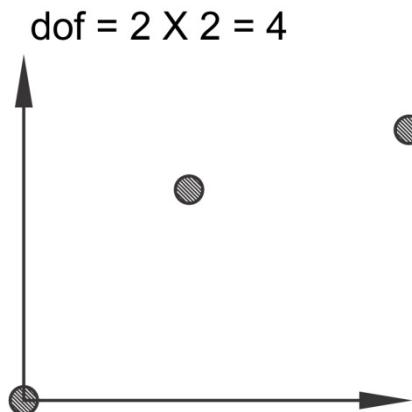
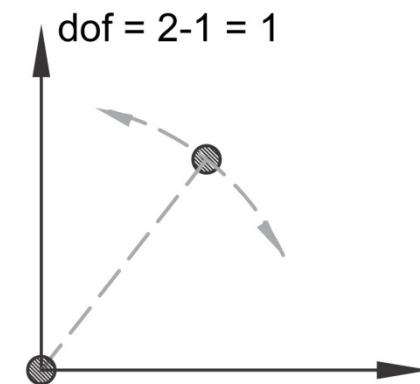
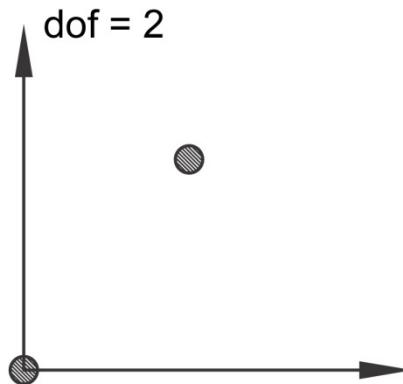
Each point (pivot) has 2 degrees of freedom

Each link subtracts 1 degree of freedom

J = Number of joints (2D points in the plane)

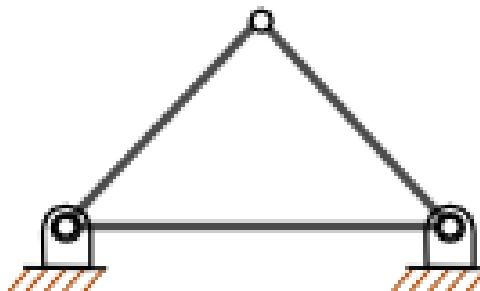
R = Number of links (not including ground link)

$$\text{DOF} = 2J - R$$

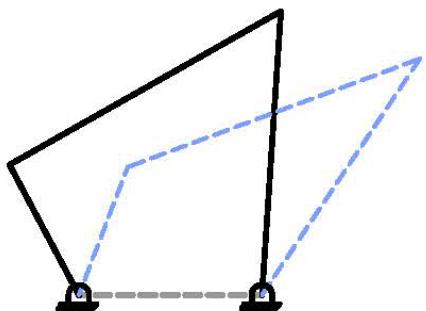


Examples

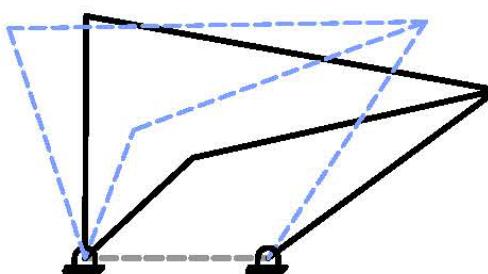
$$\text{DOF} = 2J - R$$



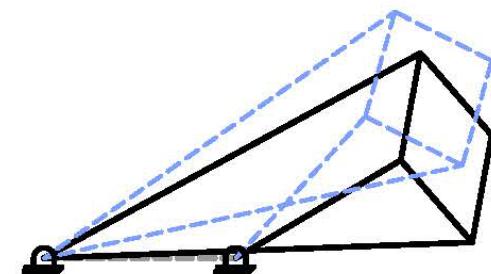
$$\begin{aligned} J &= 1 \\ R &= 2 \\ \text{DOF} &= (2 \times 1) - (1 \times 2) = 0 \end{aligned}$$



$$\begin{aligned} J &= 2 \\ R &= 3 \\ \text{DOF} &= (2 \times 2) - (3 \times 1) = 1 \end{aligned}$$



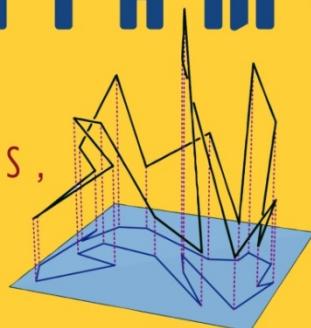
$$\begin{aligned} J &= 3 \\ R &= 5 \\ \text{DOF} &= (2 \times 3) - (5 \times 1) = 1 \end{aligned}$$



$$\begin{aligned} J &= 4 \\ R &= 7 \\ \text{DOF} &= (2 \times 4) - (7 \times 1) = 1 \end{aligned}$$

Geometric Folding algorithms

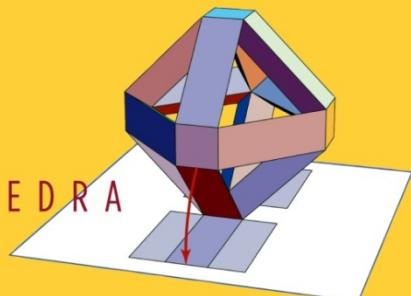
LINKAGES,



ORIGAMI,



& POLYHEDRA



ERIK D. Demaine & JOSEPH O'ROURKE

幾何的な FOLDING ALGORITHMS 折りアルゴリズム

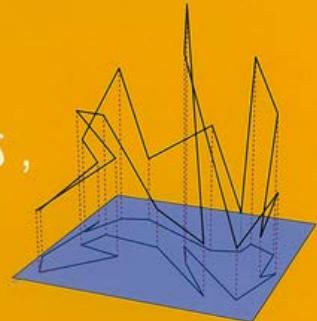
リンクエージ, 折り紙, 多面体

エリック・D・ドメイン & ジョセフ・オルーク 著

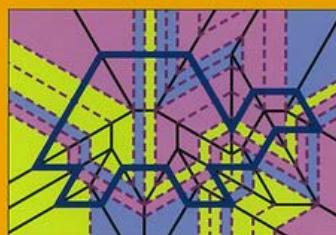
ERIK D. DEMAIN & JOSEPH O'ROURKE

上原隆平 訳

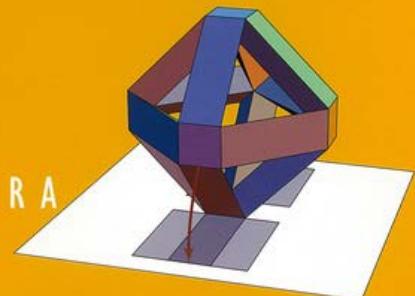
LINKAGES,



ORIGAMI,



POLYHEDRA



近代科学社

 Figure 2.1

Configuration Space

configuration \rightarrow point

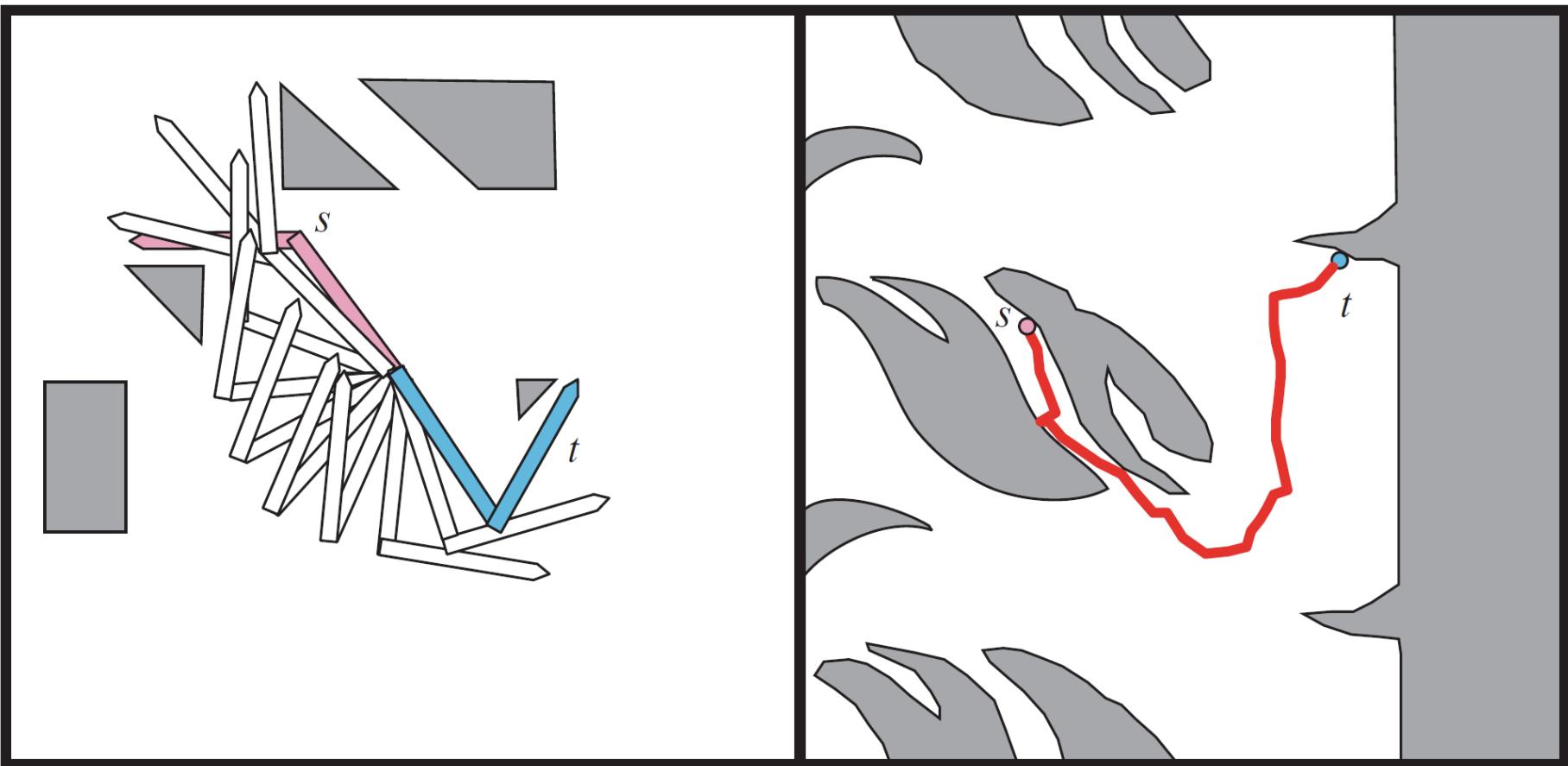
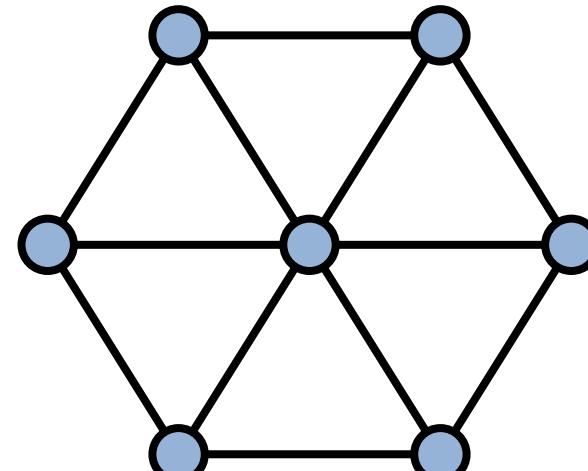
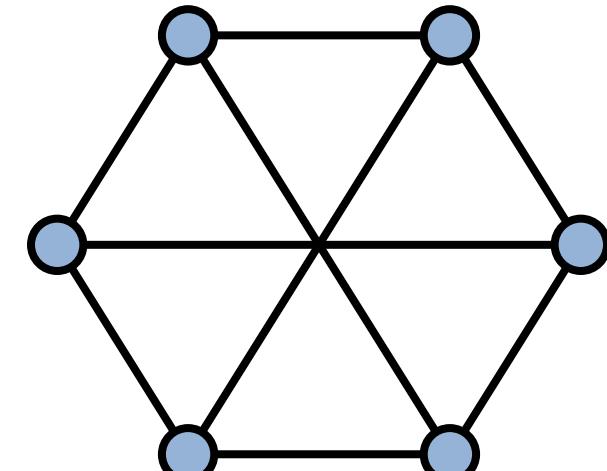
 θ_2  θ_1

Figure 4.1

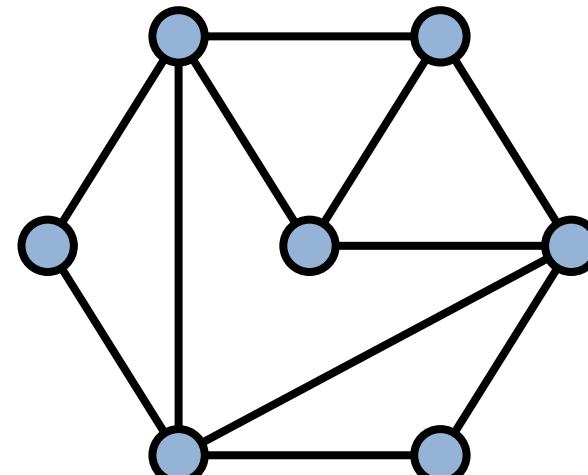
Rigidity



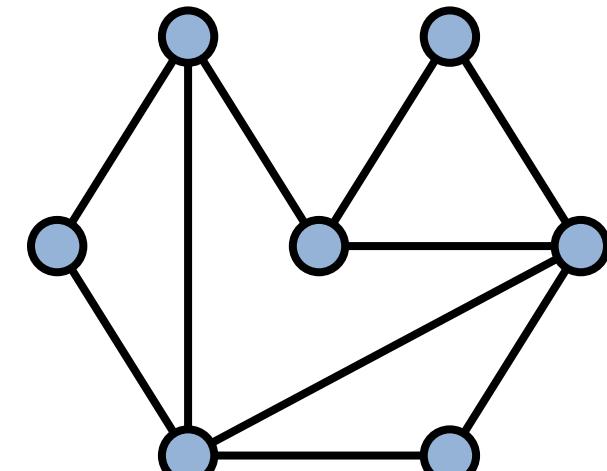
rigid



rigid



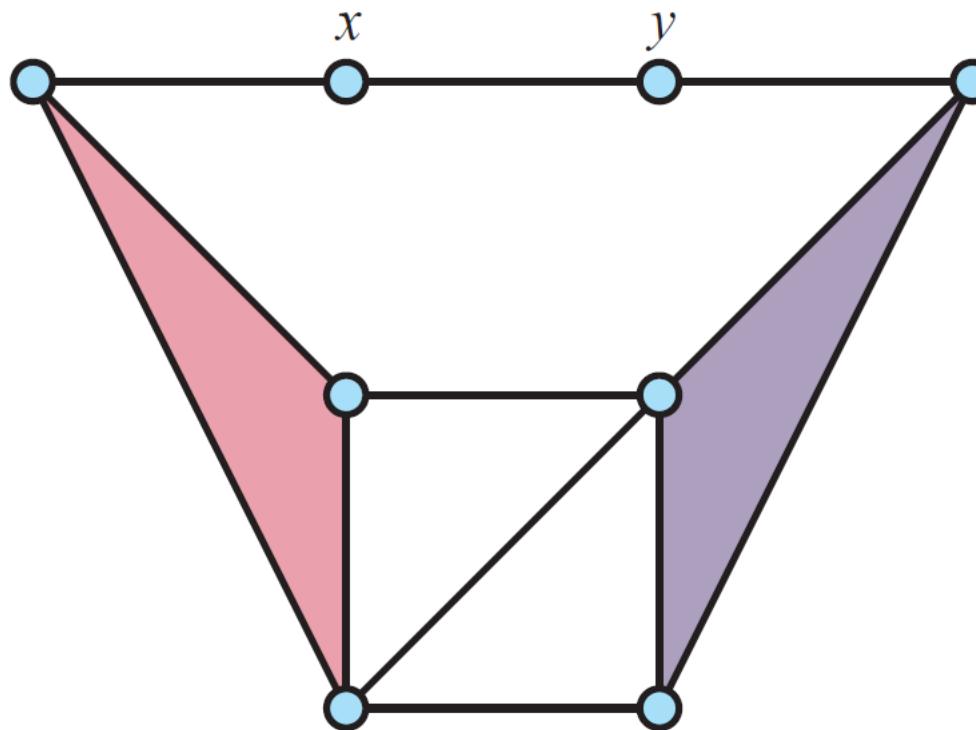
rigid



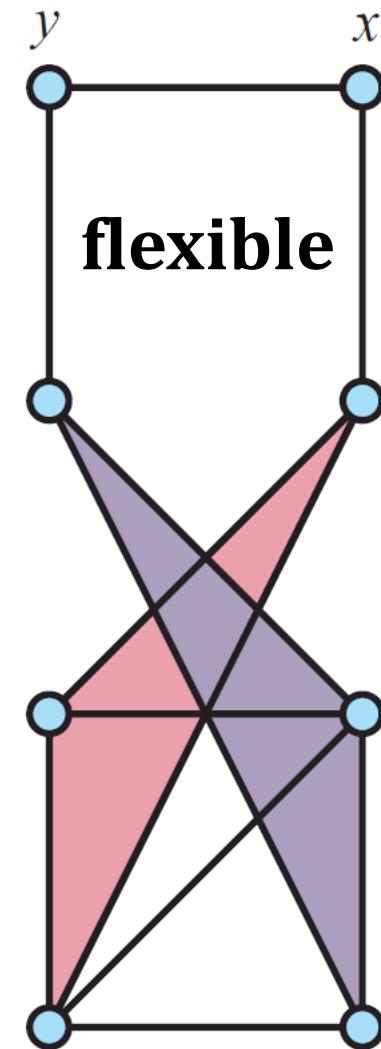
flexible

Rigidity Depends on Configuration

rigid

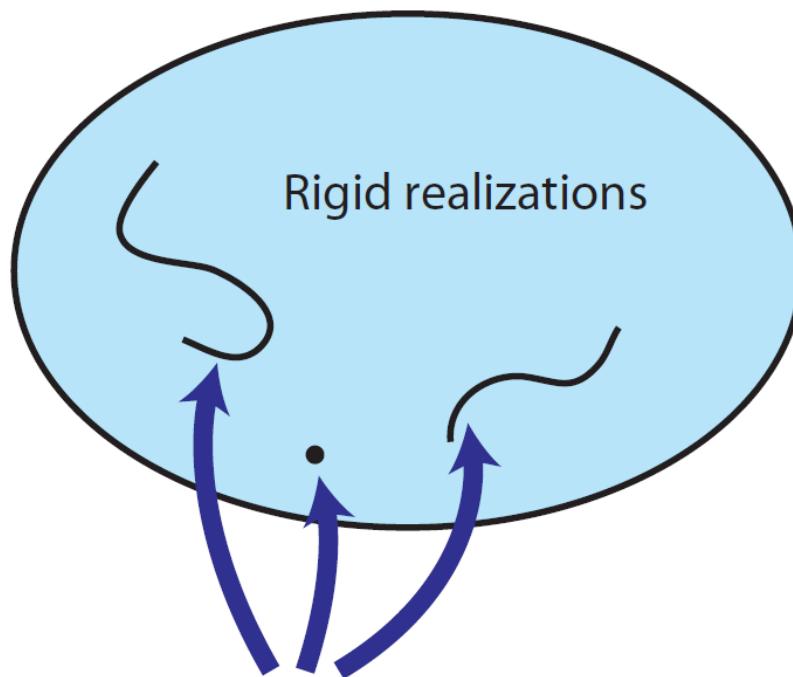


flexible



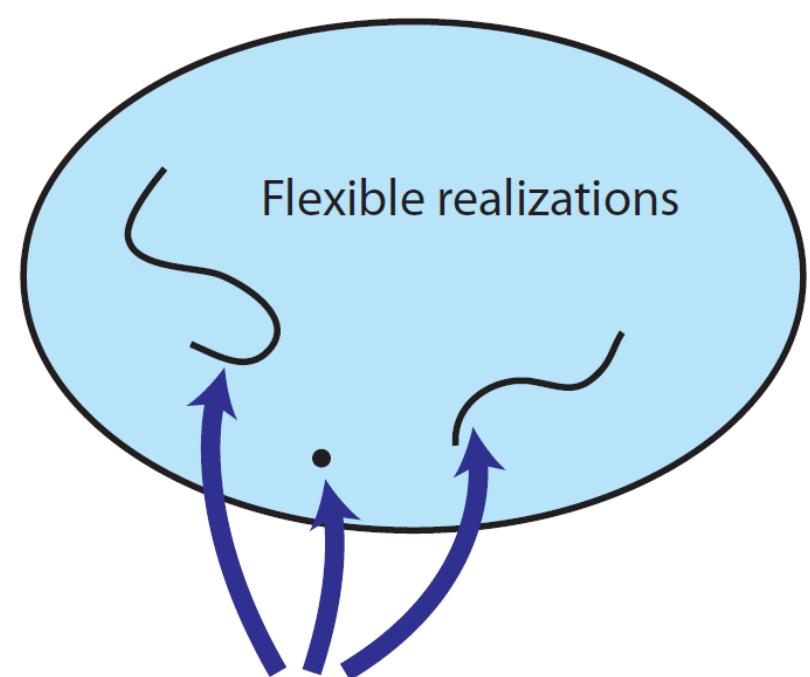
Generic Rigidity

generically rigid



(a) Flexible realizations

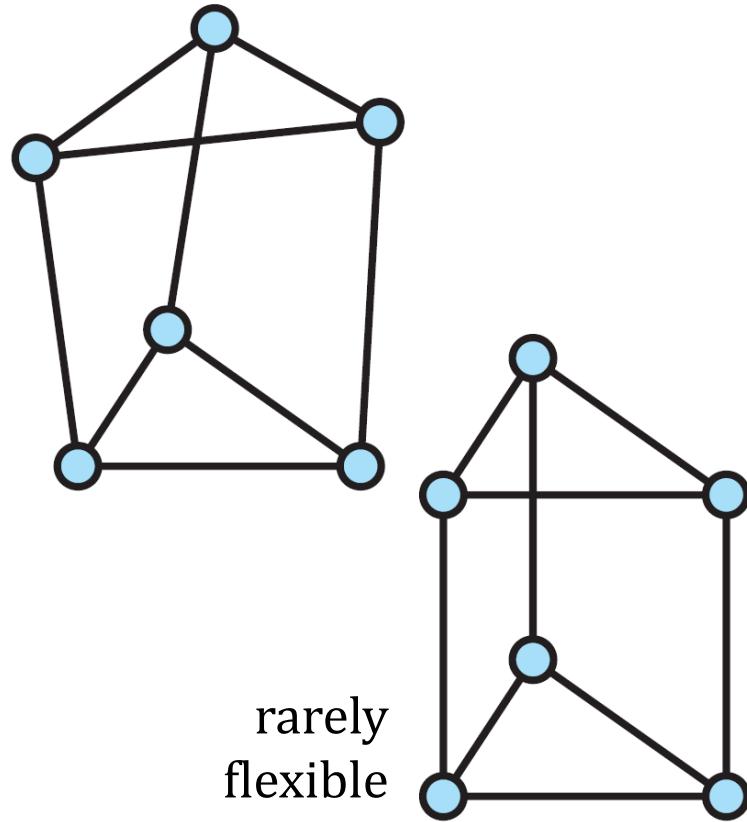
generically flexible



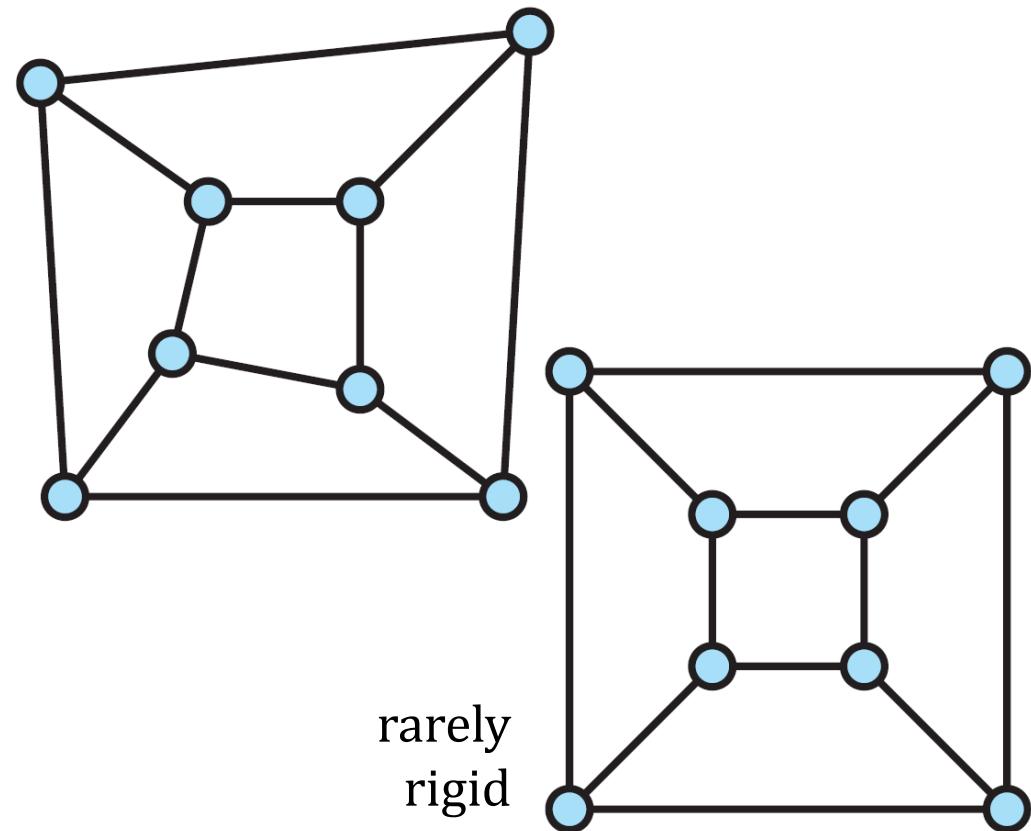
(b) Rigid realizations

Generic Rigidity

generically rigid

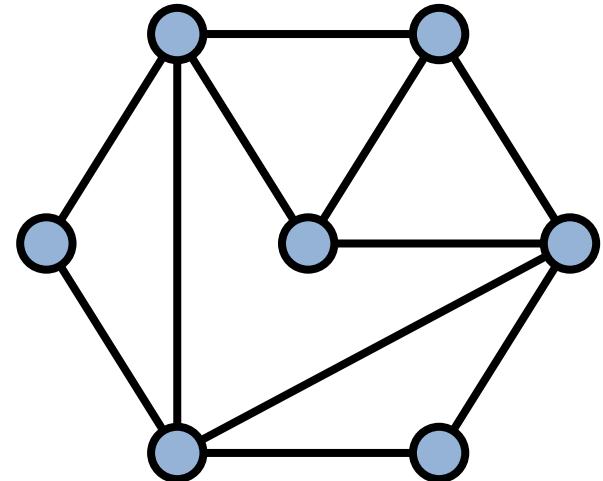


generically flexible



Laman's Theorem [1970]

- Generically rigid in 2D if and only if you can remove some extra bars to produce a **minimal graph** with
 - $2J - 3$ bars total, and
 - at most $2k - 3$ bars between every subset of k joints



Laman's Theorem [1970]

- **Generically rigid in 2D** if and only if you can remove some extra bars to produce a **minimal graph** with
 - $2J - 3$ bars total, and
 - at most $2k - 3$ bars between every subset of k joints

