

## Introduction to Other Formulations

1. Joint Coordinate Method (Chapter 13)
2. Featherstone's Formulation

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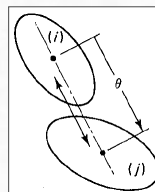
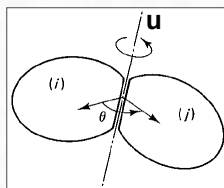


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## Joint Coordinates

- If bodies  $i$  and  $j$  are connected by a kinematic joint having relative (joint) coordinate,  $\theta_{ij}$ , then the coordinates  $\mathbf{q}_j$  can be determined from  $\mathbf{q}_i$  and  $\theta_{ij}$

$$\mathbf{q}_j = f^{(c)}(\mathbf{q}_i, \theta_{ij})$$



$$\dot{\mathbf{q}}_j = f^{(v)}(\dot{\mathbf{q}}_i, \dot{\theta}_{ij})$$

$$Ex: \omega_j = \omega_i + \dot{\theta} \mathbf{u}$$

$$\ddot{\mathbf{q}}_j = f^{(a)}(\dot{\mathbf{q}}_i, \ddot{\mathbf{q}}_i, \dot{\theta}_{ij}, \ddot{\theta}_{ij})$$

$$Ex: \dot{\omega}_j = \dot{\omega}_i + \ddot{\theta} \mathbf{u} + \dot{\theta} \dot{\mathbf{u}} = \dot{\omega}_i + \ddot{\theta} \mathbf{u} + \dot{\theta} \tilde{\omega}_i \mathbf{u}$$

$$\ddot{\theta}_{ij} = f^{(a)}(\ddot{\mathbf{q}}_i, \ddot{\mathbf{q}}_j)$$

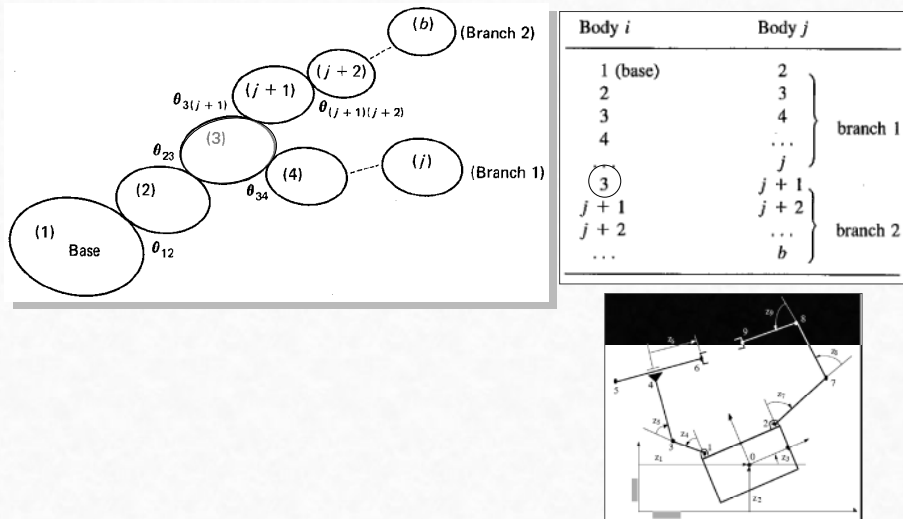
注意: 爲了可進行加減運算, 故各向量必須使用同一座標系加以表示, 通常先將各向量轉換至機架(座)之座標系才作加減。

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## Open-Chain Systems: System Topology



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## Open-Chain Systems: Solving Procedures

$$\begin{bmatrix} \mathbf{M} & \Phi_q^T \\ \Phi_q & \mathbf{0} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ -\lambda \end{bmatrix} = \begin{bmatrix} \mathbf{g} \\ \gamma \end{bmatrix} \quad \mathbf{y} = \begin{bmatrix} \mathbf{q}_1 \\ \boldsymbol{\theta} \\ \dot{\mathbf{q}}_1 \\ \dot{\boldsymbol{\theta}} \end{bmatrix} \quad \dot{\mathbf{y}} = \begin{bmatrix} \dot{\mathbf{q}}_1 \\ \dot{\boldsymbol{\theta}} \\ \ddot{\mathbf{q}}_1 \\ \ddot{\boldsymbol{\theta}} \end{bmatrix}$$

### (a) Main routine

- (a.1) Specify initial conditions for  $\mathbf{q}$  and  $\dot{\mathbf{q}}$ .
- (a.2) Specify (or automatically determine) the topology of the system.
- (a.3) Compute initial conditions for  $\boldsymbol{\theta}$  and  $\dot{\boldsymbol{\theta}}$ .
- (a.4) Transfer the initial values to  $\mathbf{y}$  (Eq. 13.39 or 13.40).

### (b) Numerical integration routine

### (c) DIFEQN routine

- (c.1) Transfer the contents of  $\mathbf{y}$  to  $\mathbf{q}_{\text{base}}$ ,  $\dot{\mathbf{q}}_{\text{base}}$  (if there is a floating base body),  $\boldsymbol{\theta}$ , and  $\dot{\boldsymbol{\theta}}$ .
- (c.2) Compute  $\mathbf{q}$  and  $\dot{\mathbf{q}}$  for all of the bodies (Eqs. 13.36 and 13.37).
- (c.3) Evaluate  $\mathbf{M}$ ,  $\Phi_q$ ,  $\mathbf{g}$ , and  $\gamma$ .
- (c.4) Solve Eq. 13.16 for  $\ddot{\mathbf{q}}$  and  $\lambda$ .
- (c.5) Compute  $\ddot{\boldsymbol{\theta}}$  (Eq. 13.38).
- (c.6) Transfer  $\dot{\mathbf{q}}_{\text{base}}$ ,  $\ddot{\mathbf{q}}_{\text{base}}$  (if there is a floating base body),  $\dot{\boldsymbol{\theta}}$ , and  $\ddot{\boldsymbol{\theta}}$  to  $\dot{\mathbf{y}}$ .
- (c.7) Return.

Fixed constraints  $\rightarrow \lambda$

$$\mathbf{q}_j = f^{(c)}(\mathbf{q}_i, \boldsymbol{\theta}_{ij})$$

$$\dot{\mathbf{q}}_j = f^{(v)}(\dot{\mathbf{q}}_i, \dot{\boldsymbol{\theta}}_{ij})$$

$$\ddot{\boldsymbol{\theta}}_{ij} = f^{(a)}(\ddot{\mathbf{q}}_i, \ddot{\mathbf{q}}_j)$$

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## Close-Chain Systems: Cut-Joint method

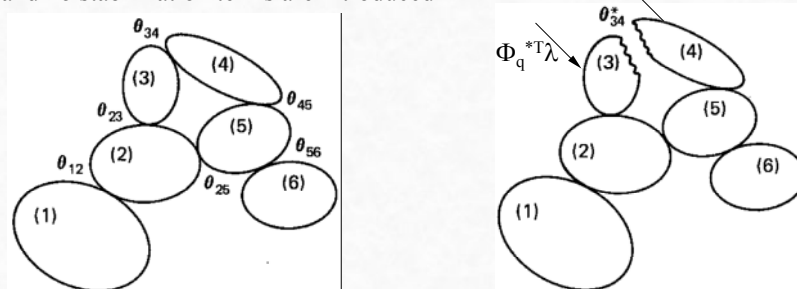
$$\begin{bmatrix} \mathbf{M} & \Phi_q^T \\ \Phi_q & \mathbf{0} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ -\lambda \end{bmatrix} = \begin{bmatrix} \mathbf{g} \\ \gamma \end{bmatrix}$$

$$\begin{bmatrix} \mathbf{M} & \Phi_q^T & \Phi_q^{*T} \\ \Phi_q & \mathbf{0} & \mathbf{0} \\ \Phi_q^* & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ -\lambda \\ -\lambda^* \end{bmatrix} = \begin{bmatrix} \mathbf{g} \\ \gamma \\ \gamma^* - 2\alpha\dot{\Phi}^* - \beta^2\Phi^* \end{bmatrix} \quad \leftarrow \quad \begin{cases} \mathbf{M}\ddot{\mathbf{q}} - \Phi_q^T\lambda - \Phi_q^{*T}\lambda^* = \mathbf{g} \\ \Phi_q\ddot{\mathbf{q}} = \gamma \\ \Phi_q^*\ddot{\mathbf{q}} = \gamma^* - 2\alpha\dot{\Phi}^* - \beta^2\Phi^* \end{cases}$$

$\Phi(q)$  : constraint equations for uncut edges

$\Phi^*(q)$  : constraint equations for cut edges

※There is no constraint violation at uncut edges,  
and no stabilization terms are introduced



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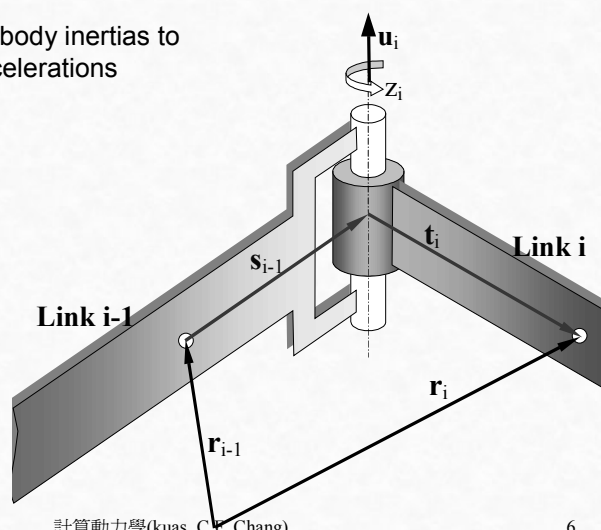
## Basic Concept for Featherstone's Formulation

- Calculating a series of articulated-body inertias, one for each link
- Using the articulated-body inertias to calculate the joint accelerations

首先, 由內而外求各桿件之  
位置和速度.

其次, 由外而內求複合桿件  
之相關性質, 例如慣性矩陣  
和作用力;

最後, 由基座開始逐一向外  
計算各接頭之相對加速度



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## Featherstone's Formulation

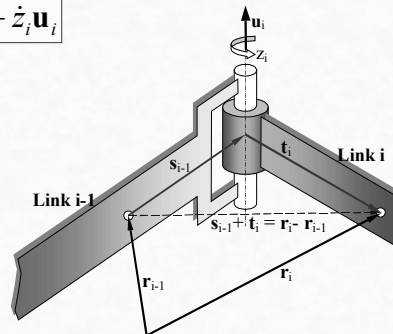
Ref: R. Featherstone, 1987, Robot Dynamics Algorithms, Kluwer Academic Publishers

- Here we adopt the notation introduced by Bae and Haug (1987) instead of the notation of spatial vector (Bae, D.-S. and Haug, E.J., "A Recursive Formulation for Constrained Mechanical System Dynamics. Part I: Open-Loop Systems, Part II: Closed-Loop Systems," Mechanics of Structures and Machines, Vol. 15, pp. 359-382, pp. 481-506.)
- In this method, the type of joint is arbitrary. The relative positions of adjacent links can be described by using the four parameters introduced by Denavit & Hartenberg
- For simplicity we will assume that all the joints are of revolute type**

$$\dot{\mathbf{r}}_i = \dot{\mathbf{r}}_{i-1} + \tilde{\boldsymbol{\omega}}_{i-1} \mathbf{s}_{i-1} + \tilde{\boldsymbol{\omega}}_i \mathbf{t}_i \quad \boldsymbol{\omega}_i = \boldsymbol{\omega}_{i-1} + \dot{z}_i \mathbf{u}_i$$

$$\begin{aligned} \dot{\mathbf{r}}_i &= \dot{\mathbf{r}}_{i-1} + \tilde{\boldsymbol{\omega}}_{i-1} \mathbf{s}_{i-1} + (\tilde{\boldsymbol{\omega}}_{i-1} + \dot{z}_i \tilde{\mathbf{u}}_i) \mathbf{t}_i \\ &= \dot{\mathbf{r}}_{i-1} + \tilde{\boldsymbol{\omega}}_{i-1} (\mathbf{s}_{i-1} + \mathbf{t}_i) + \dot{z}_i \tilde{\mathbf{u}}_i \mathbf{t}_i \\ &= \dot{\mathbf{r}}_{i-1} + \tilde{\boldsymbol{\omega}}_{i-1} (\mathbf{r}_i - \mathbf{r}_{i-1}) + \dot{z}_i \tilde{\mathbf{u}}_i \mathbf{t}_i \end{aligned}$$

各向量必須使用同一座標系加以表示才能相加減  
此處使用慣性座標系(全域座標系)  
 $dz/dt$ 為桿*i*相對於桿*i-1*之角速度



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## Recursive Relation Between the Velocities of Two Consecutive Bodies

$$\boldsymbol{\omega}_i = \boldsymbol{\omega}_{i-1} + \dot{z}_i \mathbf{u}_i \quad \dot{\mathbf{r}}_i = \dot{\mathbf{r}}_{i-1} + \tilde{\boldsymbol{\omega}}_{i-1} (\mathbf{r}_i - \mathbf{r}_{i-1}) + \dot{z}_i \tilde{\mathbf{u}}_i \mathbf{t}_i$$

$$\mathbf{Y}_i \equiv \begin{bmatrix} \dot{\mathbf{r}}_i \\ \boldsymbol{\omega}_i \end{bmatrix} = \begin{bmatrix} \mathbf{I} & \tilde{\mathbf{r}}_{i-1} - \tilde{\mathbf{r}}_i \\ \mathbf{0} & \mathbf{I} \end{bmatrix} \begin{bmatrix} \dot{\mathbf{r}}_{i-1} \\ \boldsymbol{\omega}_{i-1} \end{bmatrix} + \begin{bmatrix} \tilde{\mathbf{u}}_i \mathbf{t}_i \\ \mathbf{u}_i \end{bmatrix} \dot{z}_i$$

$$\mathbf{B}_i \equiv \begin{bmatrix} \mathbf{I} & \tilde{\mathbf{r}}_{i-1} - \tilde{\mathbf{r}}_i \\ \mathbf{0} & \mathbf{I} \end{bmatrix}$$

$$\mathbf{Y}_i = \mathbf{B}_i \mathbf{Y}_{i-1} + \mathbf{b}_i \dot{z}_i$$

$$\mathbf{b}_i \equiv \begin{bmatrix} \tilde{\mathbf{u}}_i \mathbf{t}_i \\ \mathbf{u}_i \end{bmatrix}$$

$$\dot{\mathbf{Y}}_i = \begin{bmatrix} \ddot{\mathbf{r}}_i \\ \dot{\boldsymbol{\omega}}_i \end{bmatrix} = \dot{\mathbf{B}}_i \mathbf{Y}_{i-1} + \mathbf{B}_i \dot{\mathbf{Y}}_{i-1} + \dot{\mathbf{b}}_i \dot{z}_i + \mathbf{b}_i \ddot{z}_i$$

$$\dot{\mathbf{Y}}_i = \mathbf{B}_i \dot{\mathbf{Y}}_{i-1} + \dot{\mathbf{b}}_i \dot{z}_i + \mathbf{d}_i \quad \mathbf{d}_i = \dot{\mathbf{B}}_i \mathbf{Y}_{i-1} + \dot{\mathbf{b}}_i \dot{z}_i$$

Thus, one can compute recursively forward the velocity and acceleration of links from  $i=1$  to  $i=N$  if knowing  $dz/dt$ ,  $d^2z/dt^2$ , as well as the velocity and acceleration of the **base body**. Note: Accelerations yield inertia forces and inertia torques

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**Relative Angular Acceleration at the Last Joint**

$$\mathbf{Y}_i = \begin{bmatrix} \dot{\mathbf{r}}_i \\ \boldsymbol{\omega}_i \end{bmatrix} = \mathbf{B}_i \mathbf{Y}_{i-1} + \mathbf{b}_i \dot{z}_i$$

$$\mathbf{B}_i = \begin{bmatrix} \mathbf{I} & \tilde{\mathbf{r}}_{i-1} - \tilde{\mathbf{r}}_i \\ \mathbf{0} & \mathbf{I} \end{bmatrix}, \quad \mathbf{b}_i = \begin{bmatrix} \tilde{\mathbf{u}}_i \mathbf{t}_i \\ \mathbf{u}_i \end{bmatrix}$$

$$\mathbf{d}_i = \dot{\mathbf{B}}_i \mathbf{Y}_{i-1} + \dot{\mathbf{b}}_i \dot{z}_i$$

$$\dot{\mathbf{Y}}_i = \mathbf{B}_i \dot{\mathbf{Y}}_{i-1} + \mathbf{b}_i \ddot{z}_i + \mathbf{d}_i$$

- The principle of virtual power (虛功率原理) for an  $N$ -link serial manipulator can be expressed as (作用於剛體上之外力和慣性力的合力)

$$W^* \equiv \sum_{i=1}^N \dot{\mathbf{q}}_i^{*T} \mathbf{F}_i = \sum_{i=1}^N \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) = 0$$

$$\mathbf{M}_i = \begin{bmatrix} m_i \mathbf{I}_i & \mathbf{0} \\ \mathbf{0} & \mathbf{J}_i \end{bmatrix}, \quad \dot{\mathbf{Y}}_i = \begin{bmatrix} \dot{\mathbf{r}}_i \\ \dot{\boldsymbol{\omega}}_i \end{bmatrix}, \quad \mathbf{Q}_i = \begin{bmatrix} \mathbf{f}_i \\ \mathbf{n}_i - \tilde{\boldsymbol{\omega}}_i \mathbf{J}_i \boldsymbol{\omega}_i \end{bmatrix}$$

$$\sum_{i=1}^{N-2} \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) + \mathbf{Y}_{N-1}^{*T} (\mathbf{M}_{N-1} \dot{\mathbf{Y}}_{N-1} - \mathbf{Q}_{N-1}) + \mathbf{Y}_N^{*T} (\mathbf{M}_N \dot{\mathbf{Y}}_N - \mathbf{Q}_N) = 0$$

Substituting  $\mathbf{Y}_N^*$  and  $d\mathbf{Y}_N/dt$

$$\sum_{i=1}^{N-2} \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) + \mathbf{Y}_{N-1}^{*T} (\mathbf{M}_{N-1} \dot{\mathbf{Y}}_{N-1} - \mathbf{Q}_{N-1}) + (\mathbf{Y}_{N-1}^{*T} \mathbf{B}_N^T + \mathbf{b}_N^T \dot{z}_N^*) (\mathbf{M}_N (\mathbf{B}_N \dot{\mathbf{Y}}_{N-1} + \mathbf{b}_N \ddot{z}_N + \mathbf{d}_N) - \mathbf{Q}_N) = 0$$

$$\sum_{i=1}^{N-2} \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) + \mathbf{Y}_{N-1}^{*T} \{ (\mathbf{M}_{N-1} + \mathbf{B}_N^T \mathbf{M}_N \mathbf{B}_N) \dot{\mathbf{Y}}_{N-1} - (\mathbf{Q}_{N-1} + \mathbf{B}_N^T \mathbf{Q}_N - \mathbf{B}_N^T \mathbf{M}_N \mathbf{d}_N) + \mathbf{B}_N^T \mathbf{M}_N \mathbf{b}_N \ddot{z}_N \} + \dot{z}_N^* \{ \mathbf{b}_N^T \mathbf{M}_N (\mathbf{B}_N \dot{\mathbf{Y}}_{N-1} + \mathbf{b}_N \ddot{z}_N + \mathbf{d}_N) - \mathbf{b}_N^T \mathbf{Q}_N \} = 0$$

Note that the value of an independent virtual velocity can be arbitrarily specified. If choosing  $\dot{z}_N/dt=0$  the sum of all terms but the last one equals to zero. Consequently, the last term should be zero

$$\ddot{z}_N = (\mathbf{b}_N^T \mathbf{M}_N \mathbf{b}_N)^{-1} \{ \mathbf{b}_N^T \mathbf{Q}_N - \mathbf{b}_N^T \mathbf{M}_N \mathbf{B}_N \dot{\mathbf{Y}}_{N-1} - \mathbf{b}_N^T \mathbf{M}_N \mathbf{d}_N \}$$

This equation can be used to determine the relative angular acceleration at joint  $N$  if  $\mathbf{M}_N$  and  $\mathbf{Q}_N$  are known

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**Relative Angular Acceleration at Joint  $i$**

$$\ddot{z}_N = (\mathbf{b}_N^T \mathbf{M}_N \mathbf{b}_N)^{-1} \{ \mathbf{b}_N^T \mathbf{Q}_N - \mathbf{b}_N^T \mathbf{M}_N \mathbf{B}_N \dot{\mathbf{Y}}_{N-1} - \mathbf{b}_N^T \mathbf{M}_N \mathbf{d}_N \}$$

$$\mathbf{Q}_i = \begin{bmatrix} \mathbf{f}_i \\ \mathbf{n}_i - \tilde{\boldsymbol{\omega}}_i \mathbf{J}_i \boldsymbol{\omega}_i \end{bmatrix}, \quad \mathbf{B}_i = \begin{bmatrix} \mathbf{I} & \tilde{\mathbf{r}}_{i-1} - \tilde{\mathbf{r}}_i \\ \mathbf{0} & \mathbf{I} \end{bmatrix}, \quad \mathbf{b}_i = \begin{bmatrix} \tilde{\mathbf{u}}_i \mathbf{t}_i \\ \mathbf{u}_i \end{bmatrix}$$

$$\mathbf{M}_i = \begin{bmatrix} m_i \mathbf{I}_i & \mathbf{0} \\ \mathbf{0} & \mathbf{J}_i \end{bmatrix}, \quad \mathbf{Y}_i = \begin{bmatrix} \dot{\mathbf{r}}_i \\ \boldsymbol{\omega}_i \end{bmatrix}, \quad \mathbf{d}_i = \dot{\mathbf{B}}_i \mathbf{Y}_{i-1} + \dot{\mathbf{b}}_i \dot{z}_i$$

$$\sum_{i=1}^{N-2} \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) + \mathbf{Y}_{N-1}^{*T} \{ (\mathbf{M}_{N-1} + \mathbf{B}_N^T \mathbf{M}_N \mathbf{B}_N) \dot{\mathbf{Y}}_{N-1} - (\mathbf{Q}_{N-1} + \mathbf{B}_N^T \mathbf{Q}_N - \mathbf{B}_N^T \mathbf{M}_N \mathbf{d}_N) + \mathbf{B}_N^T \mathbf{M}_N \mathbf{b}_N \ddot{z}_N \} = 0$$

Define

$$\bar{\mathbf{M}}_{N-1} \equiv \mathbf{M}_{N-1} + \mathbf{B}_N^T (\mathbf{M}_N - \mathbf{M}_N \mathbf{b}_N (\mathbf{b}_N^T \mathbf{M}_N \mathbf{b}_N)^{-1} \mathbf{b}_N^T \mathbf{M}_N) \mathbf{B}_N$$

$$\bar{\mathbf{Q}}_{N-1} \equiv \mathbf{Q}_{N-1} + (\mathbf{B}_N^T - \mathbf{B}_N^T \mathbf{M}_N \mathbf{b}_N (\mathbf{b}_N^T \mathbf{M}_N \mathbf{b}_N)^{-1} \mathbf{b}_N^T) (\mathbf{Q}_N - \mathbf{M}_N \mathbf{d}_N)$$

$d^2 z_i / dt^2$  為桿  $i$  相對於桿  $i-1$  之角速度

$$\sum_{i=1}^{N-2} \mathbf{Y}_i^{*T} (\mathbf{M}_i \dot{\mathbf{Y}}_i - \mathbf{Q}_i) + \mathbf{Y}_{N-1}^{*T} (\bar{\mathbf{M}}_{N-1} \dot{\mathbf{Y}}_{N-1} - \bar{\mathbf{Q}}_N) = 0$$

Repeat the same process of eliminating the last two terms yields

$$\bar{\mathbf{M}}_{i-1} \equiv \mathbf{M}_{i-1} + \mathbf{B}_i^T (\bar{\mathbf{M}}_i - \bar{\mathbf{M}}_i \mathbf{b}_i (\mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i)^{-1} \mathbf{b}_i^T \bar{\mathbf{M}}_i) \mathbf{B}_i$$

$$\bar{\mathbf{Q}}_{i-1} \equiv \mathbf{Q}_{i-1} + (\mathbf{B}_i^T - \mathbf{B}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i (\mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i)^{-1} \mathbf{b}_i^T) (\bar{\mathbf{Q}}_i - \bar{\mathbf{M}}_i \mathbf{d}_i)$$

For base 1:  $\mathbf{Y}_1^{*T} (\bar{\mathbf{M}}_1 \dot{\mathbf{Y}}_1 - \bar{\mathbf{Q}}_1) = 0$

$$\dot{\mathbf{Y}}_1 = \bar{\mathbf{M}}_1^{-1} \bar{\mathbf{Q}}_1$$

物理意義: 將桿  $i$  及其後之桿件 ( $i+1 \sim N$ ) 視為一支剛桿再求解此桿與前一桿 ( $i-1$ ) 之共同接頭的相對角加速度

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## Solving Procedures

$$\mathbf{Q}_i = \begin{bmatrix} \mathbf{f}_i \\ \mathbf{n}_i - \tilde{\boldsymbol{\omega}}_i \mathbf{J}_i \boldsymbol{\omega}_i \end{bmatrix} \quad \mathbf{M}_i = \begin{bmatrix} m_i \mathbf{I}_i & 0 \\ 0 & \mathbf{J}_i \end{bmatrix} \quad \mathbf{Y}_i = \begin{bmatrix} \dot{\mathbf{r}}_i \\ \boldsymbol{\omega}_i \end{bmatrix}$$

$$\mathbf{B}_i = \begin{bmatrix} \mathbf{I} & \tilde{\mathbf{r}}_{i-1} - \tilde{\mathbf{r}}_i \\ 0 & \mathbf{I} \end{bmatrix} \quad \mathbf{b}_i = \begin{bmatrix} \tilde{\mathbf{u}}_i \mathbf{t}_i \\ \mathbf{u}_i \end{bmatrix} \quad \mathbf{d}_i = \dot{\mathbf{B}}_i \mathbf{Y}_{i-1} + \mathbf{b}_i \dot{\mathbf{z}}_i$$

1. Knowing the position and velocity of the base body and the joint relative positions  $z$  and velocity  $dz/dt$ , one can compute recursively forward the Cartesian position and velocity of all the links from  $i=1$  to  $i=N$ .

Step 1 根據各接頭之相對角位置和相對角速度, 由基座開始逐一求各桿件質心之位置, 速度和角速度( $\mathbf{Y}_i$ ).

$$\mathbf{Y}_i = \mathbf{B}_i \mathbf{Y}_{i-1} + \mathbf{b}_i \dot{\mathbf{z}}_i$$

2. The articulated inertias  $\bar{\mathbf{M}}$ , the forces  $\bar{\mathbf{Q}}$ , and the coefficients for determining  $d^2z_i/dt^2$  are then computed recursively backwards from  $i=N$  to  $i=1$ , using the following equations

$$\bar{\mathbf{M}}_{i-1} \equiv \mathbf{M}_{i-1} + \mathbf{B}_i^T (\bar{\mathbf{M}}_i - \bar{\mathbf{M}}_i \mathbf{b}_i (\mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i)^{-1} \mathbf{b}_i^T \bar{\mathbf{M}}_i) \mathbf{B}_i$$

$$\bar{\mathbf{Q}}_{i-1} \equiv \mathbf{Q}_{i-1} + (\mathbf{B}_i^T - \mathbf{B}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i (\mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i)^{-1} \mathbf{b}_i^T) (\bar{\mathbf{Q}}_i - \bar{\mathbf{M}}_i \mathbf{d}_i)$$

Step 2 由最末端之桿件開始, 計算下一步驟所需之參數, 例如複合桿件之慣性矩陣 $\bar{\mathbf{M}}$ 和作用力 $\bar{\mathbf{Q}}$ .

3. Finally, the acceleration of the base body ( $d\mathbf{Y}_1/dt$ ) is computed and then the relative accelerations  $d^2z_i/dt^2$  are determined recursively forward from  $i=1$  to  $i=N$ , using equation

$$\dot{\mathbf{Y}}_1 = \bar{\mathbf{M}}_1^{-1} \bar{\mathbf{Q}}_1 \quad \dot{\mathbf{Y}}_i = \mathbf{B}_i \dot{\mathbf{Y}}_{i-1} + \mathbf{b}_i \dot{\mathbf{z}}_i + \mathbf{d}_i$$

$$\ddot{\mathbf{z}}_i = (\mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{b}_i)^{-1} (\mathbf{b}_i^T \bar{\mathbf{Q}}_i - \mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{B}_i \dot{\mathbf{Y}}_{i-1} - \mathbf{b}_i^T \bar{\mathbf{M}}_i \mathbf{d}_i)$$

Step 3 由基座開始, 先求加速度 $d\mathbf{Y}_1/dt$ , 再根據前一步驟已求得之 $\bar{\mathbf{M}}$ 和 $\bar{\mathbf{Q}}$ 求桿件受力後各接頭之相對角加速度 $d^2z_i/dt^2$ .

<<將目前之角加速度和角速度積分後可得新角速度和角位置, 以確定各桿之新狀態>>

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