

# FE Analysis – What goes in and what comes out



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## Input to and output from an FE program

### Structural analysis

- ★ Given a structure described by
    - ▶ geometry
    - ▶ physical characteristics
    - ▶ boundary conditions
  - ★ subjected to loads
  - ★ evaluate the response quantities
    - ▶ displacements
    - ▶ stresses
    - ▶ strains
    - ▶ support reactions etc.
- Input to a FE program**
- Output from a FE program**

The above is required for any kind of structural analysis and not just for FE

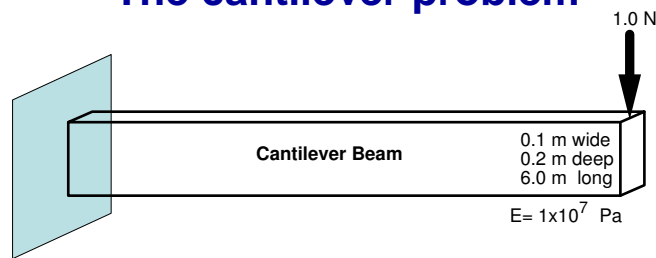
## Input to and output from an FE program

- ★ All FE programs require similar input – so if you understand one you know others
- ★ The manner in which input is provided varies from one program to another
- ★ We will use one of the most powerful commercial FE packages that we use both for teaching and research: ABAQUS
- ★ ABAQUS can solve many complex problems. The documentation in printed form exceeds 15000 pages. Online documentation is available at:

<http://www.see.ed.ac.uk/it/online/abaqus-681/v6.8/index.html>

We will discuss ABAQUS with reference to a simple problem that can easily be solved by hand

## The cantilever problem

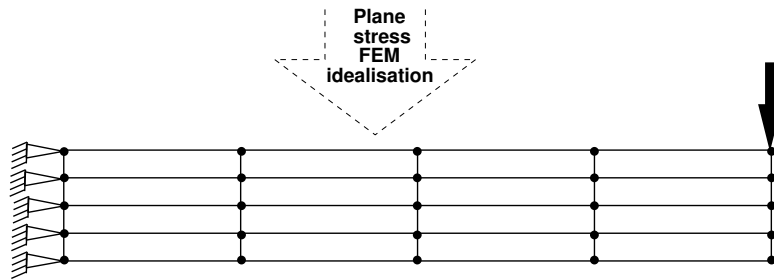


Consider a simple problem. For this problem, using elementary theory of beams you can easily work out:

- ★ Displacements – e.g. end displacement is given by  $PL^3/3EI$
- ★ Moments
- ★ Stresses and strains
- ★ Reactions at the support

We will consider what is required to solve this using ABAQUS

## FE input for the cantilever problem



We will:

- ★ Use plane stress idealisation (to be discussed in detail)
- ★ Discretise the continuum into 16 elements – choice arbitrary
- ★ Use 4-noded elements

## FE input for the cantilever problem

We must now prepare a datafile (using any text editor on the computer). All ABAQUS datafiles must be of the form **filename.inp** where, *filename* is any name chosen by the analyst. Lets choose the name **cantbeam.inp** for this problem. The following lines show the contents of this file.

```
*HEADING
CANTILEVER WITH CONTINUUM ELEMENTS--END SHEAR, CPS4I, 4 X 4 MESH
*NODE
1,0.,0.
17,6.,0.
401,0.,.2
417,6.,.2
*NGEN,NSET=FIX
1,401,100
*NGEN,NSET=END
17,417,100
*NFILL
FIX,END,4,4
```

These are  
here for  
identification  
and are not  
actually input

LINE 1  
LINE 2  
LINE 3  
LINE 4  
LINE 5  
LINE 6  
LINE 7  
LINE 8  
LINE 9  
LINE 10  
LINE 11  
LINE 12  
LINE 13

## FE input for the cantilever problem

```
*ELEMENT,TYPE=CPS4I
1,1,5,105,101
*ELGEN,ELSET=EALL
1,4,4,1,4,100,4
*MATERIAL,NAME=A1
*ELASTIC
1.E7,0.
*SOLID SECTION,MATERIAL=A1,ELSET=EALL
.1
*PREPRINT,ECHO=YES,MODEL=YES,HISTORY=YES
*STEP,PERTURBATION
*STATIC
*BOUNDARY
FIX,1,2
*CLOAD
17,2,-.5
417,2,-.5
```

These are  
here for  
identification  
and are not  
actually input

LINE 14  
LINE 15  
LINE 16  
LINE 17  
LINE 18  
LINE 19  
LINE 20  
LINE 21  
LINE 22  
LINE 23  
LINE 24  
LINE 25  
LINE 26  
LINE 27  
LINE 28  
LINE 29  
LINE 30

## FE input for the cantilever problem

```
*EL PRINT
COORD
S
E
*EL PRINT,POSITION=AVERAGED AT NODES
S
*NODE PRINT
U
RF
*END STEP
```

These are  
here for  
identification  
and are not  
actually input

LINE 31  
LINE 32  
LINE 33  
LINE 34  
LINE 35  
LINE 36  
LINE 37  
LINE 38  
LINE 39  
LINE 40

We will now consider what these input lines mean

## FE input for the cantilever problem

### LINES 1 and 2

- ★ All lines beginning with a single \* are keyword lines.
- ★ The first line indicates that a heading is to follow.
- ★ The heading is for our own convenience so that we can identify the problem at a later date.

\*HEADING  
CANTILEVER WITH CONTINUUM ELEMENTS--END SHEAR, CPS4I, 4 X 4 MESH

Indicates a heading or a title is to follow

A title of our choice

## FE input for the cantilever problem

### LINES 3 to 7

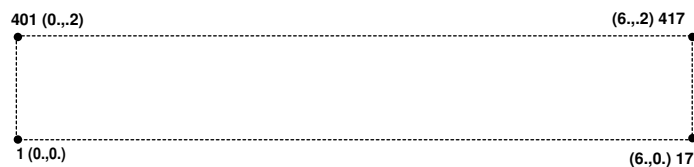
- ★ The \*NODE keyword in line 3 indicates that data related to nodes is to follow.
- ★ Lines 4 to 7 have a node number followed by the co-ordinates of the node.

\*NODE  
1,0.,0.  
17,6.,0.  
401,0.,.2  
417,6.,.2

Indicates node coordinates follow

LINE 3  
LINE 4  
LINE 5  
LINE 6  
LINE 7

y-coordinate  
x-coordinate  
node number



Information provided so far

## FE input for the cantilever problem

### LINES 8 to 11

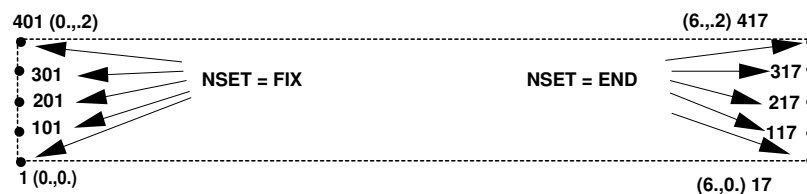
- ★ The \*NGEN keyword generates nodes.
- ★ Parameters are sometimes required on keyword lines. Many parameters are optional.
- ★ On line 8 an optional parameter NSET=FIX has been used. This implies that the generated nodes will form a set called FIX.
- ★ In line 9 we ask ABAQUS to generate nodes from node no. 1 to node no. 401 with an increment of 100.

|                |   |         |
|----------------|---|---------|
| *NGEN,NSET=FIX | Generate a set of nodes and call them FIX | LINE 8  |
| 1,401,100      | Generate a set of nodes and call them END | LINE 9  |
| *NGEN,NSET=END |   | LINE 10 |
| 17,417,100     |   | LINE 11 |

|  |                     |
|--|---------------------|
|  | Node no. increments |
|  | End node number     |
|  | Start node number   |

## FE input for the cantilever problem



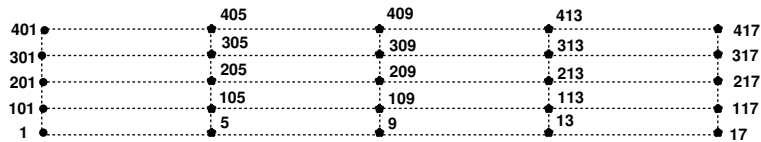
Information provided so far

## FE input for the cantilever problem

LINES 12 and 13

- ★ The \*NFILL keyword fills nodes between two bounds.
- ★ The first two entries on line 12 give the names of the node sets defining the first and the second bounds of the region respectively.
- ★ The third entry gives the number of intervals between bounding nodes, which is 4 in our case.
- ★ The fourth entry gives the increment in node numbers from the node number at the first bound set.

\*NFILL                      Fill nodes between bounds                      LINE 12  
 FIX,END,4,4                      Node no. increments                      LINE 13  
                                     No. of intervals between bounding nodes  
                                     2<sup>nd</sup> bound  
                                     1<sup>st</sup> bound



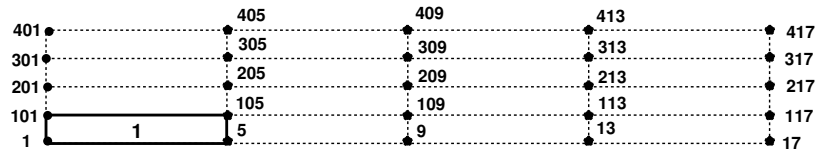
## FE input for the cantilever problem

Lines 14 and 15

- ★ The \*ELEMENT is a keyword used to define an element directly by specifying its nodes
- ★ This has a required parameter TYPE that is required to be set equal to the element type. In our case we shall use a 4-noded element that has a name CPS4I in ABAQUS.
- ★ In line 15 one element is defined. The first entry on this line gives this element a number and entries 2 to 5 give it a “connectivity”. Connectivity refers to the set of nodes connected to the element given in a specific order

                                    Indicates that an element is to be specified  
                                     Specifies the element type. CPS4I is ABAQUS terminology  
 \*ELEMENT,TYPE=CPS4I                      LINE 14  
 1,1,5,105,101                      LINE 15  
                                     Connectivity i.e. node numbers comprising the element in order  
                                     Element number

## FE input for the cantilever problem



Information provided so far

## FE input for the cantilever problem

### LINES 16 and 17

The `*ELGEN` keyword generates elements. We use an optional parameter `ELSET` to give the set of generated elements a name (we call them `EALL`). The entries on line 17 are:

1. Master element number — 1 in our case as this is the master element that we shall use to generate other similar elements.
2. Number of elements to be defined in the first row generated, including the master element — 4 in our problem.
3. Increment in node numbers of corresponding nodes from element to element in the row — 4 in our case.
4. Increment in element numbers in the row — let us choose 1.
5. Number of rows to be defined including the master row — we need 4.
6. Increment in node numbers of corresponding nodes from row to row — 100 in our case.
7. Increment in element numbers of corresponding elements from row to row — we use 4.

```
*ELGEN,ELSET=EALL
1,4,4,1,4,100,4
```

LINE 16  
LINE 17



## FE input for the cantilever problem

\*ELGEN,ELSET=EALL      Generate elements and call the set EALL      LINE 16  
 1,4,4,1,4,100,4      LINE 17

Increment in corresponding element nos. from row to row  
 Increment in corresponding node nos. from row to row  
 No. of rows to be defined including the master row  
 Increment in element nos. in the row  
 Increment in corresponding node nos. in the row  
 Number of elements to be defined in the first row generated, including the master element  
 Master element number

|     |    |     |    |     |    |     |    |     |
|-----|----|-----|----|-----|----|-----|----|-----|
| 401 |    | 405 |    | 409 |    | 413 |    | 417 |
|     | 13 | 305 | 14 | 309 | 15 | 313 | 16 |     |
| 301 | 9  | 205 | 10 | 209 | 11 | 213 | 12 | 317 |
| 201 | 5  | 105 | 6  | 109 | 7  | 113 | 8  | 217 |
| 101 | 1  | 5   | 2  | 9   | 3  | 13  | 4  | 117 |
| 1   |    |     |    |     |    |     |    | 17  |

All elements in ELSET = EALL

## FE input for the cantilever problem

LINES 18 to 22

- ★ The \*MATERIAL keyword indicates the start of a material definition and has a required parameter NAME associated to it – let us call it A1.
- ★ The \*ELASTIC option in line 19 is used to define linear elastic moduli to the material A1. Young's modulus,  $E$  and Poisson's ratio,  $\nu$  are specified in line 20.
- ★ The \*SOLID SECTION keyword has been used here to define thickness – 0.1m specified in line 22). This thickness is valid for all elements ELSET=EALL and these elements have elastic properties of MATERIAL=A1.

\*MATERIAL,NAME=A1      We are defining a material called A1      LINE 18  
 \*ELASTIC      The elastic properties of A1 follow      LINE 19  
 1.E7,0.      Young's modulus and Poisson's ratio      LINE 20  
 \*SOLID SECTION,MATERIAL=A1,ELSET=EALL      LINE 21  
 .1      Thickness of A1 to follow. All elements in EALL are of material A1      LINE 22  
 Thickness

Model now fully defined. Loading and BCs to be specified

## FE input for the cantilever problem

### LINE 23

After you have run ABAQUS the numerical results are printed in the *filename.dat* file. This line indicates that you would like your input file (*filename.inp*), the history data and the model data to be included in the results file.

```
*PREPRINT, ECHO=YES, MODEL=YES, HISTORY=YES
```

LINE 23

### LINES 24 and 40

These two lines indicate the beginning and the end of a step. We have already provided the model data. The program now requires boundary conditions and loading. The parameter **PERTURBATION** indicates that we are conducting a linear analysis in which we shall be providing load and boundary changes.

```
*STEP, PERTURBATION
```

LINE 24

```
.  
.
.
```

```
*END STEP
```

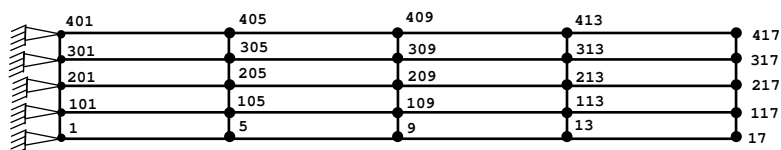
LINE 40

## FE input for the cantilever problem

### LINES 25 to 27

- ★ **\*STATIC** indicates that the step should be analysed as a static load step.
- ★ **\*BOUNDARY** (line 26) is used to prescribe boundary conditions.
- ★ The first entry on data line (line 27) indicates that the boundary conditions relate to node set **FIX**. The following two entries on line 27 restrain degrees of freedom 1 and 2. Thus all nodes of the node set **FIX** are now restrained from displacement in both *x*- and *y*-directions.

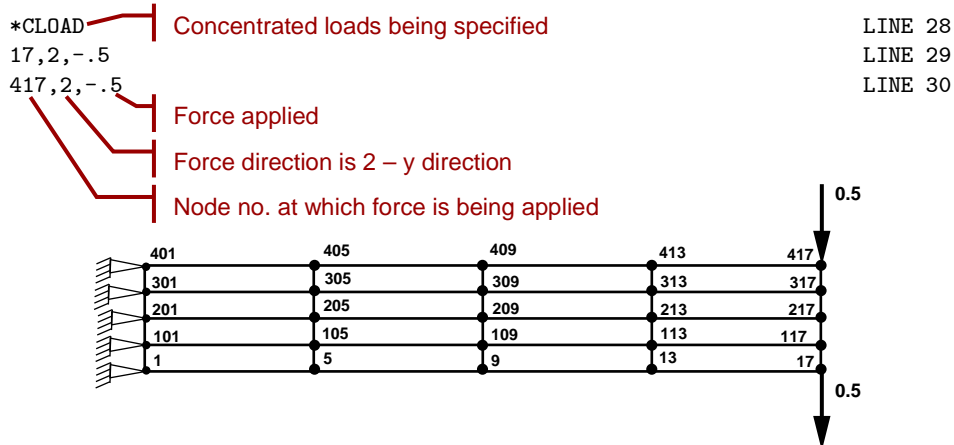
|                  |   |         |
|------------------|---|---------|
| <b>*STATIC</b>   | Load step is static   | LINE 25 |
| <b>*BOUNDARY</b> | Boundary conditions follow  | LINE 26 |
| <b>FIX, 1, 2</b> | Restrainted in direction 2 – y direction                          | LINE 27 |
|                  | Restrainted in direction 1 – x direction                          |         |
|                  | Boundary conditions relate to all nodes in NSET called <b>FIX</b> |         |



## FE input for the cantilever problem

LINES 28 to 30

- ★ \*CLOAD indicates concentrated loads are being specified.
- ★ In lines 29 and 30 concentrated loads have been applied on nodes 17 and 417, in the  $y$ -direction (degree of freedom – 2), and the magnitude of these loads is 0.5. The minus sign indicates that the load acts downwards ( $y$  is positive upwards).



## FE input for the cantilever problem

LINES 31 to 39

- ★ \*EL PRINT is used to define output requests for element variables such as stresses and strains. FE does all calculations at integration points within elements.
- ★ Line 32 to 34 request output of the coordinates of integration points (COORD), stresses (S), and strains (E). By default option \*EL PRINT outputs values at integration points. If we want averages of values extrapolated at the nodes, we can use the parameter POSITION=AVERAGED AT NODES. Here we have requested stresses (line 36).
- ★ \*NODE PRINT is used to define output requests for nodal variables such as displacements and reactions
- ★ Here we want nodal displacements which are indicated by the letter U in line 38 and reaction forces indicated by RF in line 39.

## FE input for the cantilever problem

|                                       |   |         |
|---------------------------------------|---|---------|
| *EL PRINT                             | Print request for element variables           | LINE 31 |
| COORD                                 | Print integration point coordinates           | LINE 32 |
| S                                     | Print stresses                                | LINE 33 |
| E                                     | Print strains                                 | LINE 34 |
| *EL PRINT ,POSITION=AVERAGED AT NODES | Print element variables extrapolated at nodes | LINE 35 |
| S                                     | Print stresses                                | LINE 36 |
| *NODE PRINT                           |   | LINE 37 |
| U                                     | Print displacements                           | LINE 38 |
| RF                                    | Print reactions at restrained nodes           | LINE 39 |

ABAQUS outputs results in a number of different files.  
*filename.dat* is a readable file in which you can see the above output

## FE output for the cantilever problem

Consider partial results from the *filename.dat* file

| E L E M E N T   D E F I N I T I O N S |       |                       |                       |    |     |     |
|---------------------------------------|-------|-----------------------|-----------------------|----|-----|-----|
| NUMBER                                | TYPE  | PROPERTY<br>REFERENCE | NODES FORMING ELEMENT |    |     |     |
| 1                                     | CPS4I | 1                     | 1                     | 5  | 105 | 101 |
| 2                                     | CPS4I | 1                     | 5                     | 9  | 109 | 105 |
| 3                                     | CPS4I | 1                     | 9                     | 13 | 113 | 109 |
| .                                     |       |                       |                       |    |     |     |
| .                                     |       |                       |                       |    |     |     |
| .                                     |       |                       |                       |    |     |     |

Element nos. → TYPE → PROPERTY REFERENCE → Element property → Element connectivity

## FE output for the cantilever problem

### M A T E R I A L   D E S C R I P T I O N

MATERIAL NAME: A1

|         |                    |                    |
|---------|--------------------|--------------------|
| ELASTIC | YOUNG'S<br>MODULUS | POISSON'S<br>RATIO |
|         | 1.00000E+07        | 0.00000E+00        |

Values for material A1 as input

## FE output for the cantilever problem

### N O D E   D E F I N I T I O N S

| NODE<br>NUMBER | COORDINATES |             |             | SINGLE POINT CONSTRAINTS |      |     |
|----------------|-------------|-------------|-------------|--------------------------|------|-----|
|                |             |             |             | TYPE                     | PLUS | DOF |
| 1              | 0.00000E+00 | 0.00000E+00 | 0.00000E+00 |                          | 1    | 2   |
| 5              | 1.5000      | 0.00000E+00 | 0.00000E+00 |                          |      |     |
| 9              | 3.0000      | 0.00000E+00 | 0.00000E+00 |                          |      |     |
| 13             | 4.5000      | 0.00000E+00 | 0.00000E+00 |                          |      |     |
| .              |             |             |             |                          |      |     |
| .              |             |             |             |                          |      |     |
| .              |             |             |             |                          |      |     |

Nodal coordinates as input or generated by ABAQUS in accordance to our instructions in the input file. Note the constraints included.

## FE output for the cantilever problem

### BOUNDARY CONDITIONS

| NODE | DOF | AMP.<br>REF. | MAGNITUDE   | NODE | DOF | AMP.<br>REF. | MAGNITUDE   |
|------|-----|--------------|-------------|------|-----|--------------|-------------|
| 1    | 1   | (RAMP)       | 0.00000E+00 | 1    | 2   | (RAMP)       | 0.00000E+00 |
| 101  | 1   | (RAMP)       | 0.00000E+00 | 101  | 2   | (RAMP)       | 0.00000E+00 |
| 201  | 1   | (RAMP)       | 0.00000E+00 | 201  | 2   | (RAMP)       | 0.00000E+00 |
| 301  | 1   | (RAMP)       | 0.00000E+00 | 301  | 2   | (RAMP)       | 0.00000E+00 |
| 401  | 1   | (RAMP)       | 0.00000E+00 | 401  | 2   | (RAMP)       | 0.00000E+00 |

- (RAMP) OR (STEP) - INDICATE USE OF DEFAULT AMPLITUDES ASSOCIATED WITH THE STEP

### CONCENTRATED LOADS

| NODE | DOF | AMP. | AMPLITUDE | NODE | DOF | AMP. | AMPLITUDE |
|------|-----|------|-----------|------|-----|------|-----------|
| 17   | 2   |      | -0.50000  | 417  | 2   |      | -0.50000  |

## FE output for the cantilever problem

### ELEMENT OUTPUT

THE FOLLOWING TABLE IS PRINTED FOR ALL ELEMENTS  
WITH TYPE CPS4I AT THE INTEGRATION POINTS

| ELEMENT | PT | FOOT-<br>NOTE | COORD1 | COORD2     |
|---------|----|---------------|--------|------------|
| 1       | 1  |               | 0.3170 | 1.0566E-02 |
| 1       | 2  |               | 1.183  | 1.0566E-02 |
| 1       | 3  |               | 0.3170 | 3.9434E-02 |
| 1       | 4  |               | 1.183  | 3.9434E-02 |

Integration points are points within the elements. You can verify from the above coordinates that for element 1 the points are approximately located as:

|     |     |
|-----|-----|
| ☆ 3 | ☆ 4 |
| ☆ 1 | ☆ 2 |

## FE output for the cantilever problem

THE FOLLOWING TABLE IS PRINTED FOR ALL ELEMENTS WITH  
TYPE CPS4I AT THE INTEGRATION POINTS

| ELEMENT | PT | FOOT-<br>NOTE | S11    | S22         | S12    |
|---------|----|---------------|--------|-------------|--------|
| 1       | 1  |               | -7044. | -3.8691E-02 | -31.41 |
| 1       | 2  |               | -7044. | -0.1444     | -31.41 |
| 1       | 3  |               | -4769. | -3.8691E-02 | -31.41 |
| 1       | 4  |               | -4769. | -0.1444     | -31.41 |

Element nos.

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## FE output for the cantilever problem

THE FOLLOWING TABLE IS PRINTED FOR ALL ELEMENTS WITH TYPE CPS4I  
AVERAGED AT THE NODES

| NODE | FOOT-<br>NOTE | S11    | S22        | S12    |
|------|---------------|--------|------------|--------|
| 1    |               | -7877. | 7.8724E-11 | -31.41 |
| 5    |               | -6750. | -0.1831    | -31.27 |
| 9    |               | -4500. | 0.7042     | -31.23 |
| 13   |               | -2250. | -2.615     | -31.27 |

## FE output for the cantilever problem

N O D E O U T P U T

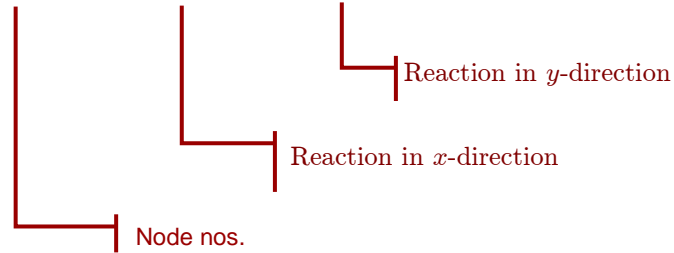
THE FOLLOWING TABLE IS PRINTED FOR ALL NODES

| NODE | FOOT-<br>NOTE | U1          | U2          |
|------|---------------|-------------|-------------|
| 5    |               | -1.1815E-03 | -8.8765E-03 |
| 9    |               | -2.0250E-03 | -3.2940E-02 |
| 13   |               | -2.5315E-03 | -6.7129E-02 |
| 17   |               | -2.7000E-03 | -0.1064     |



## FE output for the cantilever problem

| NODE | FOOT- | RF1         | RF2        |
|------|-------|-------------|------------|
| 1    | NOTE  | 18.76       | 8.3114E-02 |
| 101  |       | 22.47       | 0.2474     |
| 201  |       | -8.5938E-12 | 0.3389     |
| 301  |       | -22.47      | 0.2474     |
| 401  |       | -18.76      | 8.3114E-02 |



What should the total reactions in the two directions be?

THE ANALYSIS HAS BEEN COMPLETED