

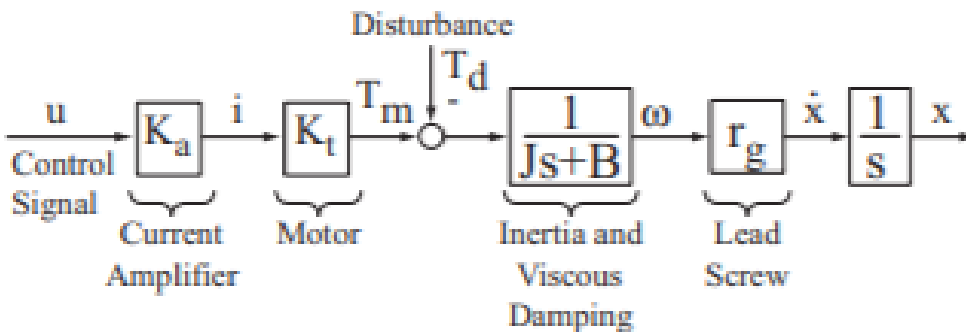
# Me2400-Project-2

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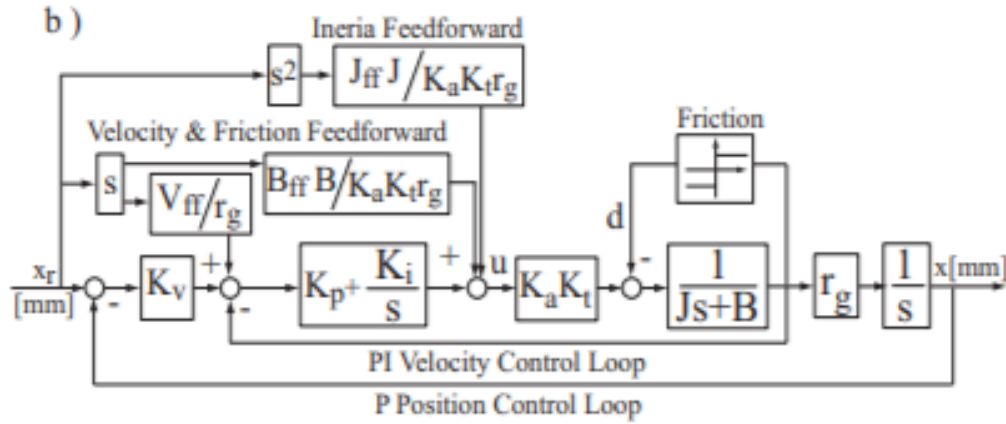
Through this project, we simulate the closed loop transfer function of 5-axis Computer Numerical Controlled (CNC) drives. Here the model considers the drive to be controlled by P-PI Cascade with feed forward dynamics and friction compensation. Study and analysis is conducted based on the paper “Identification of 5-Axis Machine Tools Feed Drive Systems for Contouring Simulation”.

The rigid body motion of a feed drive mechanism is shown in the below figure.



Here  $J \text{ kgm}^2$  is the total inertia and  $B \text{ kgm}^2/\text{s}$  is the viscous damping of the axis. In this model, motor and the amplifier are assumed to operate within their linear range, where  $u \text{ V}$  is the control voltage command to the current amplifier modeled by a gain factor,  $K_a \text{ A/V}$ , and the corresponding torque delivered to the drive is obtained by multiplying the current with the motor torque constant  $K_t \text{ A/V}$ , and  $r_g \text{ mm/rad}$  is transmission gain of the ball-screw mechanism. torque delivered to the drive is obtained by multiplying the current with the motor torque constant  $K_t \text{ A/V}$ , and  $r_g \text{ mm/rad}$  is transmission gain of the ball-screw mechanism

P-PI controller scheme is one of the most commonly used position control structure in commercial CNC drives. It is depicted below:-



Here, the velocity loop is closed using Proportional-Integral (PI) control by adjusting the gains  $K_p$ ,  $K_i$ , and the position loop is closed by a proportional control (P) using  $K_v$ . Here the feed-forward compensation of axis dynamics is applied to increase the servo tracking bandwidth where  $V_f f \in [0,1]$ ,  $B_f f \in [0,1]$ ,  $J_f f \in [0,1]$  are the respective gains used to cancel the dynamics in P-PI controller.

The generalised axis model between commanded ( $x_r$  mm) and actual axis position ( $x$  mm) can be written as;

$$[s^2 + a_1 s + a_2 + a_3 \frac{1}{s}] x(s) = [b_0 s^2 + b_1 s + b_2 + a_3 \frac{1}{s}] x_r(s) - \text{sign}(\dot{x}) d_c. \dots\dots (1)$$

where  $d_c$  Nm is the average Coulomb friction torque and  $\text{sign}()$  is the signum function. The model parameters can be expressed in terms of  $B$ ,  $K_a$ ,  $K_p$ ,  $K_t$ ,  $J$ , etc. These parameters are obtained using an objective function which is solved using Particle Swarm optimisation (PSO) method. Using these parameters we can simulate the axis motion by a closed loop drive system.

The true closed loop parameters for the P-PI controlled virtual drive is given in the table below;

Model Parameters	Actual value
$a_1$	238.4335
$a_2$	48421.5859
$a_3$	2487140.3
$b_0$	0
$b_1$	117.5424
$b_2$	37258.9796
$d_c$	2.6815

The model is simulated by substituting the value of the parameters from the table in equation (1).

