

Review

Unbalanced mass

$$m\ddot{x} + c\dot{x} + kx = m_0 e \omega_r^2 \sin \omega_r t$$

\uparrow total mass \uparrow offset mass \swarrow offset dist. $\underbrace{\quad\quad\quad}_{\text{driven freq}}$

steady state $x_p(t) = \bar{X} \sin(\omega_r t - \theta)$

$$\bar{X} = \frac{m_0 e}{m} \frac{r^2}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$

$$\theta = \arctan \frac{2\zeta r}{1-r^2}$$

If $\zeta > 1$: $\frac{m\bar{X}}{m_0 e} < 1$ - increase damping a lot and solve any issues

as $r \rightarrow \infty$: $\frac{m\bar{X}}{m_0 e} \rightarrow 1$ - at high freq mass imbalance doesn't matter so much

low damping + $r \approx 1$ - problematic!
 \hookrightarrow car speed

Equivalent mass, stiffness, damping

mass: equate kinetic energy

stiffness: equate potential energy

damping: equate "energy loss per cycle"

[in the context of forced vibrations]

$$\Delta E = \oint F_d dx$$
$$= \int_0^{\frac{2\pi}{\omega}} F_d \frac{dx}{dt} dt$$

Coulomb

$$C_{eq} = \frac{4\mu mg}{\pi \omega X}$$

Aero Drag

$$C_{eq} = \frac{8}{3\pi} \alpha \omega X$$

$$\alpha = \frac{C_D A}{C_0}$$

$$F_D \propto V^2$$

Hysteretic Damping

$$C_{eq} = \frac{kE}{\omega} \text{ or } \frac{h}{\omega}$$

area inside a hysteresis loop is ΔE

$$\pi C \omega X^2$$

General Force Response

- forces applied to system are rarely modeled well by sinusoids (harmonic excitation)
- develop methods to handle generic forces

Superposition

For linear systems the response is the sum ~~of~~ the responses from additive forcing functions.

e.g.

$$\ddot{x} + \omega_n^2 x = 0$$

if $x_1(t)$ and $x_2(t)$ are both solutions

then

$x = a_1 x_1 + a_2 x_2$ is also a solution

similarly

$$\ddot{x} + \omega_n^2 x = f_1$$

$$x_p = x_1$$

$$\ddot{x} + \omega_n^2 x = f_2$$

$$x_p = x_2$$

the solution to $\ddot{x} + \omega_n^2 x = f_1 + f_2$ is simply $x_1 + x_2$

Periodic : repeats in time

- harmonic: sinusoidal

Non periodic : doesn't repeat

- impulses, step, ramp, etc

- random

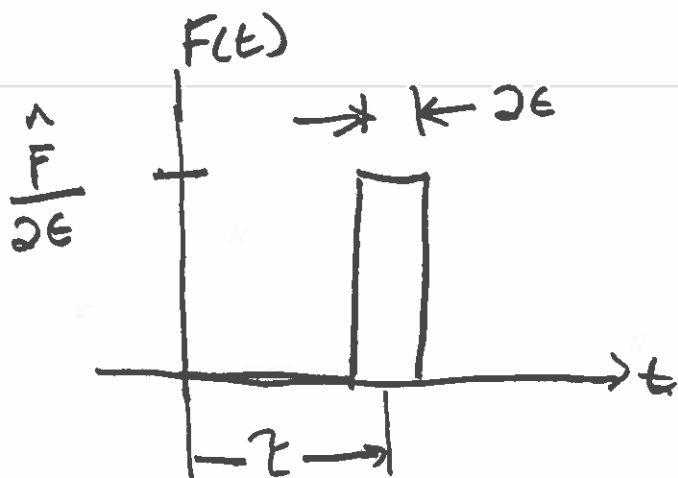
Transient : goes to zero in finite amount of time.

Impulse Response Function

Impulse: large force applied for a very short duration (non-periodic)

impulses cause "shock loading"

Key idea: response from an impulse is equivalent to the response from some initial conditions



$$F(t) = \begin{cases} 0 & t \leq \tau - \epsilon \\ \frac{\hat{F}}{2\epsilon} & \tau - \epsilon \leq t \leq \tau + \epsilon \\ 0 & t \geq \tau + \epsilon \end{cases}$$

ϵ is small and $\epsilon > 0$

Impulse of Force: measure of the forcing functions "strength"

$$I(\epsilon) = \int_{\tau-\epsilon}^{\tau+\epsilon} F(t) dt = \int_{-\infty}^{\infty} F(t) dt$$

↳ units N·s

$$I(\epsilon) = \int_{-\infty}^{\infty} F(t) dt = \frac{\hat{F}}{2\epsilon} 2\epsilon = \hat{F}$$

$I(\epsilon)$ is independent of ϵ if $\epsilon \neq 0$

Define "impulse function" with two properties:

$$F(t-\tau) = 0 \quad t \neq \tau$$

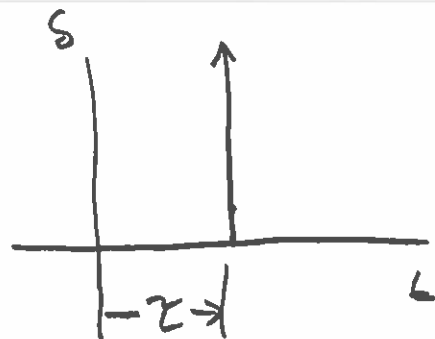
and

$$\int_{-\infty}^{\infty} F(t-\tau) dt = \hat{F}$$

unit impulse function (Dirac Delta Function)

$$\delta(t-\tau) = 0 \quad t \neq \tau$$

$$\int_{-\infty}^{\infty} \delta(t-\tau) dt = 1$$



response to $\hat{F}(t)$

$$m\ddot{x}(t) + c\dot{x}(t) + kx(t) = \hat{F}(t)$$

Recall that an impulse imparts a change in momentum.

$$\Delta p = \hat{F}$$

applying \hat{F} impulse is equivalent to $x_0 = 0$ and $v_0 = ?$

$$m v_0 = F \Delta t$$

$$v_0 = \frac{F \Delta t}{m}$$

If $x_0 = 0$ and $v_0 = \frac{F \Delta t}{m}$ and $0 < \zeta < 1$

$$x(t) = \frac{v_0}{\omega_d} e^{-\zeta \omega_n t} \sin \omega_d t$$

$$x(t) = \frac{\hat{F} e^{-\zeta \omega_n t}}{m \omega_d} \sin \omega_d t$$

$$x(t) = \hat{F} \underbrace{h(t)}$$

response to unit impulse

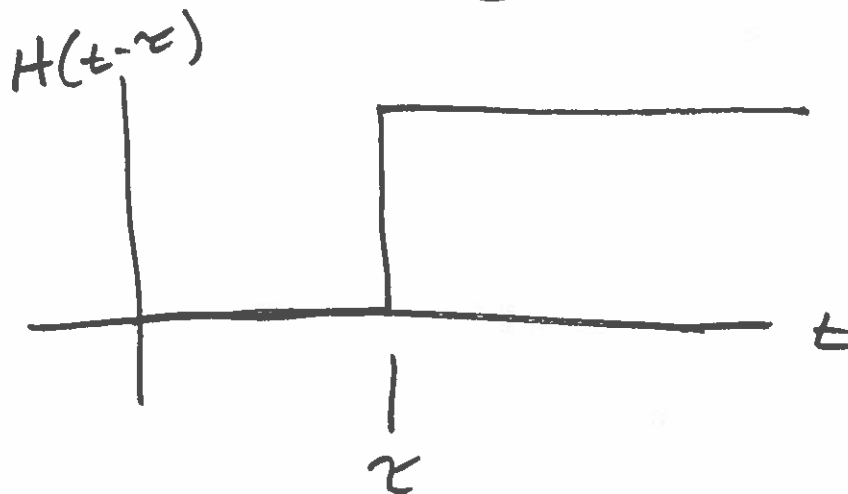
$$h(t-\tau) = \frac{1}{m\omega_d} e^{-\zeta\omega_n(t-\tau)} \sin\omega_d(t-\tau)$$

valid $t > \tau$

$$t < \tau \quad h(t-\tau) = 0$$

Heaviside Function

$$H(t-\tau) = \begin{cases} 0 & t < \tau \\ 1 & t \geq \tau \end{cases}$$



Model is only valid for large forces and short durations.

$$\text{if } \Delta t \ll T = \frac{2\pi}{\omega_n}$$

Example

Consider a linear system with

$$m = 0.1 \text{ kg}$$

$$c = 0.25 \text{ Ns/m}$$

$$k = 0.5 \text{ N/m}$$

- At $t = 2 \text{ s}$ the system is struck by a force of 10 N for a duration of 0.01 s .
- 1) What is the response function?
 - 2) sketch the response from $t = 0$ to $t = 10 \text{ s}$.

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{0.5 \frac{N}{m}}{0.1 \text{ kg}}} = 2.24 \frac{\text{rad}}{\text{s}} \quad \hat{F} = (10 \text{ N})(0.01 \text{ s})$$

$$\zeta = \frac{c}{2 m \cdot \omega_n} = \frac{0.25 \frac{Ns}{m}}{2(0.1 \text{ kg})(2.24 \frac{\text{rad}}{\text{s}})} \quad \hat{F} = 0.1 \text{ Ns}$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = (2.24 \frac{\text{rad}}{\text{s}}) \sqrt{1 - (0.559)^2} = 1.857 \frac{\text{rad}}{\text{s}}$$

$$x(t) = \frac{\hat{F}}{m \omega_d} e^{-\zeta \omega_n (t - \tau)} \sin \omega_d (t - \tau) \quad \tau = 2 \text{ s}$$

$$= \frac{(0.1 \text{ Ns}) e^{-(0.559 \cdot 2.24 \frac{\text{rad}}{\text{s}})(t - 2)}}{(0.1 \text{ kg})(1.86 \text{ rad/s})} \sin 1.86(t - 2)$$

$$= 0.538 e^{-1.25(t - 2s)} \sin 1.86(t - 2s)$$

$$T = \frac{2\pi}{\omega_d} = 3.38 \text{ s}$$

$$e \quad \begin{aligned} x(0) &= 0 \\ x(2) &= 0 \\ x(3.7) &= 0 \end{aligned}$$

