History of Vibrations

I+ (1736 -1813)

Galileo Father of experimental science (1564-1641) - simple prendulum - resonance frequencies (Fr) - telescope: onoons jupiter Joseph Saweur - Vibratity String mode shapes (1653-1716) - frequency VS kength - harmonics of beats Newton (En) (1642-1727) - Calculus, gravational equations - Laws of motion => FEMA - Principin 1687 - Taylor's thereom -Taylor Series 1685-1731 - theoretical solution vibrating string Daniel Bernoulli Guiss) Bernoulli-Euler 1766-1782 Beam Theory Learhard Euler S 1767-1783 - Newton-Euler equations - generalized coordinates Juseph-Louis Lagrange

①

- alternative way to form equations

Charles-Augustin de Colulomb (Fr) (1736-180E) - Friction, torsional Oscillations E.F.F Chedri (German) 1756-187 - vibrating plates Sophie Germain (1776-1831) . plate theory (1809) Kirchoff - corrected plate boundary (1824-1887) conditions Poisson (Fr) - restangular membrane (178-1846) Vibration Frahm (1909) vibration absorber. Stephen Timoskenko (Ukraine) - Improved bean vibration theory significantly - Father applied mechanics (1878-1972) J.P. Den Hartog (1901-1989) - The primary authority on applied vibrations.

Mechanical Vibrations

Three essential factors:

- (Kinetic energy storage)
- (Potential energy Storage)
- 3 Diss apative mechanism (Energy loss)

Degrees of Freedom: the number of independent coordinates to describe the motion of a system

Classify diff. Vibration systems

Discrete US Continous systems Finite #40 t DoF US Infinite # of DoF

(II) Free of Excited by externel external forces VS Forces

Damped VS Undamped has energy disappation

3

Linear

ODE

ODE

ODE

ODE

ODE

System

F=ma are enough to predictable in the statistical sense matron

Analysis Procedure

1) Mathematical model off reality

(ODE/PDE that determine the system)

State

2) Derive the Equations of Motion

3) Seek Solutions to the equations

4) Interpret results

Mathematical Models of Reality

F=ma Second order ordinary
differential equation

Prescribe F, M, a or T, I, I

to precisely describe a system's
motion.

Concepts:
-Right bodies Champed masses with
inertia, not flexible)
- dof

- reference frames (need inertial frame for EoM derivation) - coordinates describer config of sys. - Genelarized coordinates Ly are minimum set of coordinates that uniquely define the config.

Derive the EoM of a simple pendulum (I) Direct Method (F.B.D + Newton-Eule-Equations) I Indirect Method (Lagrange, Hamilton's, Kane' nethod, etc) Lagranges Method" " Energy method" Kinetic energy: $T = \sum_{i=1}^{N} \pm m_i v_i^2$ = & = mixi Momentum of particle 21 = Mivi = Pi $\frac{d!}{d!} \left(\frac{2T}{2x_i} \right) = m_i x_i$ For conserta forces Fi = - 24- potential ener

$$\frac{d}{dt}\left(\frac{2T}{2x_i}\right) = -\frac{24}{2x_i}$$

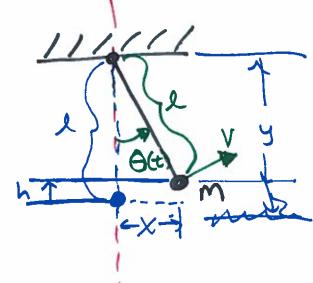
$$\frac{\partial T}{\partial x_i} = 0 \qquad \frac{\partial U}{\partial \dot{x}_i} = 0 \qquad (3)$$

$$\frac{d}{dt}\left(\frac{2(T-u)}{2xi}\right) = \frac{2(T-u)}{2xi}$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{x}_i} \right) = \frac{\partial L}{\partial x_i}$$

Lagrange Equation (of the second Kind)

Example 11



$$T = \frac{1}{2}mv^2 = \frac{1}{2}m(\theta R)^2$$
 $U = mgh = mg(R(1-cos\theta))$
 $L = T - U = \frac{1}{2}m\theta R^2 - mgR(1-cos\theta)$

Second order ODE in O

$$\Theta = -\frac{9}{2} \sin \Theta \qquad \text{Non-1}$$

linearized