

GATE 2009 ME

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- 1) For a matrix $[M] = \begin{bmatrix} 4 & 3 \\ 5 & x \end{bmatrix}$, the transpose of the matrix is equal to the inverse of the matrix, i.e., $[M]^T = [M]^{-1}$. The value of x is given by:

- (a) 4
- (b) 5
- (c) $-\frac{3}{5}$
- (d) $\frac{3}{5}$
- (e) $\frac{4}{5}$

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- 2) The divergence of the vector field $3xz\hat{i} + 2xy\hat{j} - yz^2\hat{k}$ at a point (1, 1, 1) is equal to:

- (a) 7
- (b) 4
- (c) 3
- (d) 0

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- 3) The inverse Laplace transform of $\frac{1}{s^2+s}$ is:

- (a) $1 + e^t$
- (b) $1 - e^t$
- (c) $1 - e$
- (d) $1 + e$

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- 4) If three coins are tossed simultaneously, the probability of getting at least one head is:

- (a) $\frac{1}{8}$
- (b) $\frac{3}{8}$
- (c) $\frac{7}{8}$
- (d) $\frac{1}{2}$

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- 5) If a closed system is undergoing an irreversible process, the entropy of the system:

- (a) must increase
- (b) always remains constant
- (c) must decrease
- (d) can increase, decrease or remain constant

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- 6) A coolant fluid at 30°C flows over a heated flat plate maintained at a constant temperature of 100°C . The boundary layer temperature distribution at a given location may be approximated as $T = 30 + 70e^{-y}$ where y (in m) is the distance normal to the plate and T is

in $^{\circ}\text{C}$. If thermal conductivity is 1.0 W/mK , the local convective heat transfer coefficient ($\text{in W/m}^2\text{K}$) at that location will be:

- (a) 0.2
- (b) 1
- (c) 5
- (d) 10

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7) A frictionless piston-cylinder device contains a gas initially at 0.8 MPa and 0.015 m^3 . It expands quasi-statically at constant temperature to a final volume of 0.030 m^3 . The work output (in kJ) during this process will be:

- (a) 8.32
- (b) 12.00
- (c) 554.67
- (d) 8320.00

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8) In an ideal vapor compression refrigeration cycle, the specific enthalpy of refrigerant (in kJ/kg) at the following states is given as:

inlet of condenser: 283

exit of condenser: 116

exit of evaporator: 232.

The COP of this cycle is:

- (a) 2.27
- (b) 2.75
- (c) 3.27
- (d) 3.75

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9) A compressor undergoes a reversible, steady flow process. The specific work required to be supplied to the compressor for this gas compression process is:

- (a) $\int_1^2 P dv$
- (b) $\int_1^2 v dP$
- (c) $v_1 (P_2 - P_1)$
- (d) $-P_1 (v_1 - v_2)$

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10) A block weighing 981 N is resting on a horizontal surface. The coefficient of friction is $\mu = 0.2$. A vertical cable provides partial support. A man can pull horizontally with 100 N . What will be the tension T (in N) in the cable if the man is just able to move the block to the right?

- (a) 176.2
- (b) 196.0
- (c) 481.0
- (d) 981.0

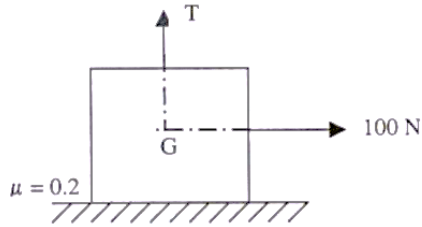


Fig. 10

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- 11) If the principal stresses in a plane stress problem are $\sigma_1 = 100\text{ MPa}$, $\sigma_2 = 40\text{ MPa}$, the magnitude of the maximum shear stress (*in MPa*) will be:

(a) 60
(b) 50
(c) 30
(d) 20

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- 12) A simple quick return mechanism is shown. If the forward to return ratio is 2:1, and radius of the crank O_1P is 125 mm, the distance d (*in mm*) between crank center to lever pivot center point should be

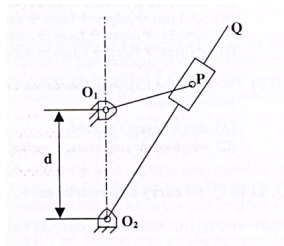


Fig. 12

(a) 144.3
(b) 216.5
(c) 240.0
(d) 250.0

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- 13) The rotor shaft of a large electric motor supported between short bearings at both ends shows a deflection of 1.8 mm. The likely critical speed (*in rpm*) is:

(a) 350
(b) 705
(c) 2810

(d) 430

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- 14) A solid circular shaft of diameter d is subjected to a combined bending moment M and torque T . The material property to be used for designing the shaft using the relation $\frac{16}{\pi d^3} \sqrt{M^2 + T^2}$ is:

- (a) ultimate tensile strength (S_u)
- (b) tensile yield strength (S_y)
- (c) torsional yield strength (S_{sy})
- (d) endurance strength (S_e)

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- 15) The effective number of lattice points in the unit cell of simple cubic, body centered cubic, and face centered cubic space lattices, respectively, are:

- (a) 1, 2, 2
- (b) 1, 2, 4
- (c) 2, 3, 4
- (d) 2, 4, 4

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- 16) Friction at the tool-chip interface can be reduced by:

- (a) decreasing the rake angle
- (b) increasing the depth of cut
- (c) decreasing the cutting speed
- (d) increasing the cutting speed

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- 17) Two streams of liquid metal which are not hot enough to fuse properly result in a casting defect known as:

- (a) cold shut
- (b) swell
- (c) sand wash
- (d) scab

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- 18) The expected time (t_e) of a PERT activity in terms of optimistic time (t_o), pessimistic time (t_p) and most likely time (t_m) is given by:

- (a) $t_e = \frac{t_o + 4t_m + t_p}{6}$
- (b) $t_e = \frac{t_o + 4t + t_p}{6}$
- (c) $t_e = \frac{t_o + 4t + t_p}{3}$
- (d) $t_e = \frac{t_o + 4t_m + t_p}{3}$

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- 19) Which of the following is the correct data structure for solid models?

- (a) solid part \rightarrow faces \rightarrow edges \rightarrow vertices

- (b) solid part \rightarrow edges \rightarrow faces \rightarrow vertices
- (c) vertices \rightarrow edges \rightarrow faces \rightarrow solid parts
- (d) vertices \rightarrow faces \rightarrow edges \rightarrow solid parts

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- 20) Which of the following forecasting methods takes a fraction of forecast error into account for the next period forecast?
- (a) simple average method
 - (b) moving average method
 - (c) weighted moving average method
 - (d) exponential smoothing method

GATE ME 2009**Q.21 TO Q.60 CARRY TWO MARKS EACH**

- 21) An analytic function of a complex variable $z = x + iy$ is expressed as $f(z) = u(x, y) + iv(x, y)$ where $i = \sqrt{-1}$. If $u = xy$, the expression for v should be

- (A) $\frac{(x+y)^2}{2} + k$
- (B) $\frac{x^2 - y^2}{2} + k$
- (C) $\frac{y^2 - x^2}{2} + k$
- (D) $\frac{(x-y)^2}{2} + k$

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- 22) The solution of $x \frac{dy}{dx} + y = x^4$ with the condition $y(1) = \frac{6}{5}$ is

- (A) $y = \frac{x^4}{5} + \frac{1}{x}$
- (B) $y = \frac{4x^4}{5} + \frac{4}{5x}$
- (C) $y = \frac{x^4}{5} + 1$
- (D) $y = \frac{x^5}{5} + 1$

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- 23) A path AB in the form of one quarter of a circle of unit radius is shown in the figure. Integration of $(x+y)^2$ on path AB traversed in a counter-clockwise sense is

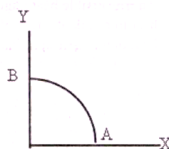


Fig. 23

- (A) $-\frac{\pi}{2} - 1$
 (B) $\frac{\pi}{2} + 1$
 (C) $\frac{\pi}{2}$
 (D) $\frac{\pi}{2}$

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24) The distance between the origin and the point nearest to it on the surface $z^2 = 1 + xy$ is

- (A) 1
 (B) $\frac{\sqrt{3}}{2}$
 (C) $\sqrt{3}$
 (D) 2

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25) The area enclosed between the curves $y^2 = 4x$ and $x^2 = 4y$ is

- (A) $\frac{16}{3}$
 (B) 8
 (C) $\frac{32}{3}$
 (D) 16

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26) The standard deviation of a uniformly distributed random variable between 0 and 1 is

- (A) $\frac{1}{\sqrt{12}}$
 (B) $\frac{1}{\sqrt{3}}$
 (C) $\frac{1}{\sqrt{12}}$
 (D) $\frac{1}{\sqrt{12}}$

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27) Consider steady, incompressible and irrotational flow through a reducer in a horizontal pipe where the diameter is reduced from 20 cm to 10 cm. The pressure in the 20 cm pipe just upstream of the reducer is 150 kPa. The fluid has a vapour pressure of 50 kPa and a specific weight of 5 kN/m³. Neglecting frictional effects, the maximum discharge ($\text{in m}^3/\text{s}$) that can pass through the reducer without causing cavitation is

- (A) 0.05
 (B) 0.16
 (C) 0.27
 (D) 0.38

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28) In a parallel flow heat exchanger operating under steady state, the heat capacity rates of the hot and cold fluid are equal. The hot fluid, flowing at 1 kg/s with $C_p = 4 \text{ kJ/kgK}$, enters at 102°C while the cold fluid has an inlet temp of 15°C. The overall heat transfer coefficient is 1 kW/m²K and area is 5 m². Neglect ambient heat transfer. The exchanger is characterized by $2\varepsilon = 1 - \exp(-2NTU)$. The exit temperature (in °C) for the cold fluid is

- (A) 45
 (B) 55
 (C) 65

(D) 75

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- 29) In an air-standard Otto cycle, the compression ratio is 10. The condition at the beginning of the compression process is 100 kPa and 27°C . Heat added at constant volume is 1500 kJ/kg, while 700 kJ/kg of heat is rejected during the other constant volume process. Specific gas constant for air = 0.287 kJ/kgK. The mean effective pressure (in kPa) is

(A) 103
(B) 310
(C) 515
(D) 1032

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- 30) An irreversible heat engine extracts heat from a high temperature source at 100 kW and rejects heat at 50 kW. The entire work output is used to drive a reversible heat pump operating between 17°C and 75°C . The rate (in kW) at which the heat pump delivers heat to its high temp sink is

(A) 50
(B) 250
(C) 300
(D) 360

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- 31) You are asked to evaluate assorted fluid flows for their suitability in a given laboratory application. The following three flow choices, expressed in terms of the two-dimensional velocity fields in the xy -plane, are made available.

P. $u = 2y, v = -3x$

Q. $u = 3xy, v = 0$

R. $u = -2x, v = 2y$

Which flow(s) should be recommended when the application requires the flow to be incompressible and irrotational?

(A) P and R
(B) Q
(C) Q and R
(D) R

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- 32) Water at 25°C flows through a 1.0 km long G.I. pipe of 200 mm diameter at $0.07 \text{ m}^3/\text{s}$. Darcy friction factor is 0.02 and water density is 1000 kg/m^3 . The pumping power (in kW) required is

(A) 1.8
(B) 17.4
(C) 20.5
(D) 41.0

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- 33) Steady-state heat conduction across the thickness in a plane composite wall (see figure) exposed to convection on both sides

Given: $h_1 = 20 \text{ W/m}^2\text{K}$; $h_2 = 50 \text{ W/m}^2\text{K}$; $T_{\infty,1} = 20^{\circ}\text{C}$; $T_{\infty,2} = -2^{\circ}\text{C}$; $k_1 = 20 \text{ W/mK}$; $k_2 = 50 \text{ W/mK}$; $L_1 = 0.30 \text{ m}$, $L_2 = 0.15 \text{ m}$. Assuming negligible contact resistance between the wall surfaces, the interface temperature, T (in $^{\circ}\text{C}$), of the two walls will be

(A) -0.50
(B) 2.75

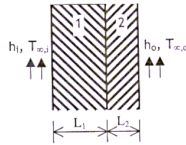


Fig. 33

(C) 3.75

(D) 4.50

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- 34) The velocity profile of a fully developed laminar flow in a straight circular pipe, as shown in the figure, is given by the expression

$$u(r) = \frac{R^2}{4\mu} \frac{dp}{dx} \left(1 - \frac{r^2}{R^2} \right)$$

where is a constant. The average velocity of fluid in the pipe is

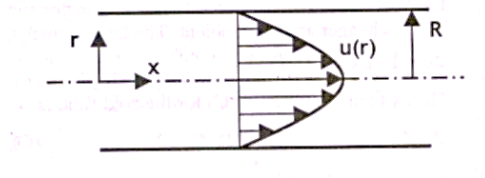


Fig. 34

(A) $\frac{R^2}{8\mu} \frac{dp}{dx}$

(B) $\frac{R^2}{4\mu} \frac{dp}{dx}$

(C) $\frac{R^2}{6\mu} \frac{dp}{dx}$

(D) $\frac{R^2}{2\mu} \frac{dp}{dx}$

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- 35) A solid shaft of diameter, d and length, L is fixed at both the ends. A torque, T_0 is applied at a distance, $L/4$ from the left end as shown in the figure given below.

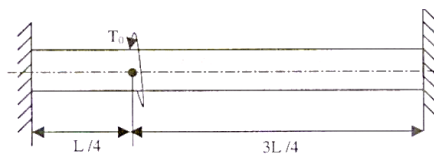


Fig. 35

The maximum shear stress in the shaft is

- (A) $\frac{16T_0}{3\pi d^3}$
- (B) $\frac{12T_0}{\pi d^3}$
- (C) $\frac{8T_0}{\pi d^3}$
- (D) $\frac{4T_0}{3\pi d^3}$

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- 36) An epicyclic gear train is shown schematically in the adjacent figure. The sun gear 2 on the input shaft is a 20 teeth external gear. The planet gear 3 is a 40 teeth external gear. The ring gear 5 is a 100 teeth internal gear. The ring gear 5 is fixed and gear 2 is rotating at 60 rpm ccw. The arm 4 attached to the output shaft will rotate at

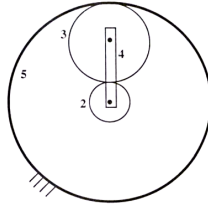


Fig. 36

- (A) 10 rpm ccw
- (B) 10 rpm cw
- (C) 12 rpm cw
- (D) 12 rpm ccw

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- 37) A forged steel link with uniform diameter of 30 mm at the centre is subjected to an axial force that varies from 40 kN in compression to 160 kN in tension. The tensile (S_u), yield (S_y) and corrected endurance (S_e) strengths of the steel are 600 MPa, 420 MPa, and 240 MPa respectively. The factor of safety against fatigue endurance as per Soderberg's criterion is

- (A) 1.26
- (B) 1.37
- (C) 1.45
- (D) 2.00

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- 38) An automotive engine weighing 240 kg is supported on four springs with linear characteristics. Each of the front two springs have a stiffness of 16 MN/m while the stiffness of each rear spring is 32 MN/m. The engine speed (in rpm), at which resonance is likely to occur, is

- (A) 6040
- (B) 3020
- (C) 1424
- (D) 955

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- 39) A vehicle suspension system consists of a spring and a damper. The stiffness of the spring is 3.6 kN/m and the damping constant is 400 Ns/m. If the mass is 50 kg, then the damping

factor (d) and damped natural frequency (f_n), respectively, are

- (A) 0.471 and 1.19 Hz
- (B) 0.471 and 7.48 Hz
- (C) 0.666 and 1.35 Hz
- (D) 0.666 and 8.50 Hz

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- 40) A frame of two arms of equal length L is shown in the adjacent figure. The flexural rigidity of each arm of the frame is EI . The vertical deflection at the point of application of load P is

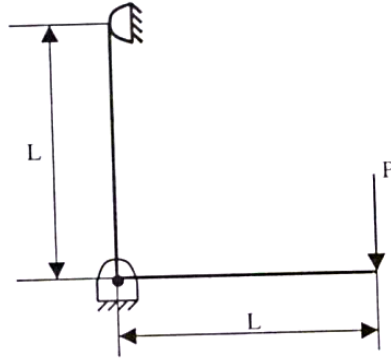


Fig. 40

- (A) $\frac{PL^3}{3EI}$
- (B) $\frac{2PL^3}{3EI}$
- (C) $\frac{PL^3}{EI}$
- (D) $\frac{4PL^3}{3EI}$

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- 41) A uniform rigid rod of mass M and length L is hinged at one end as shown in the adjacent figure. A force P is applied at a distance $\frac{2L}{3}$ from the hinge so that the rod swings to the right. The reaction at the hinge is

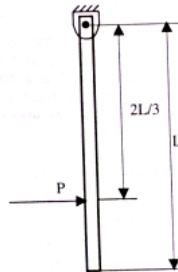


Fig. 41

- (A) $-P$
 (B) 0
 (C) $\frac{P}{3}$
 (D) $\frac{2P}{3}$

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42) Match the approaches given below to perform stated kinematics / dynamics analysis of machine.

Analysis	Approach
P. Continuous relative rotation	1. D'Alembert's principle
Q. Velocity and acceleration	2. Grubler's criterion
R. Mobility	3. Grashoff's law
S. Dynamic-static analysis	4. Kennedy's theorem

- (A) P-1, Q-2, R-3, S-4
 (B) P-3, Q-4, R-2, S-1
 (C) P-2, Q-3, R-4, S-1
 (D) P-4, Q-2, R-1, S-3

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43) A company uses 2555 units of an item annually. Delivery lead time is 8 days. The reorder point (in number of units) to achieve optimum inventory is

- (A) 7
 (B) 8
 (C) 56
 (D) 60

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44) Consider the following Linear Programming Problem (LPP):

$$\text{Maximize } z = 3x_1 + 2x_2$$

$$\text{subject to } x_1 \leq 4,$$

$$x_2 \leq 6,$$

$$3x_1 + 2x_2 \leq 18,$$

$$x_1, x_2 \geq 0. \text{ The nature of this LPP is}$$

- (A) The LPP has a unique optimal solution.
 (B) The LPP is infeasible.
 (C) The LPP is unbounded.
 (D) The LPP has multiple optimal solutions.

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45) Six jobs arrived in a sequence as given below:

Average flow time (in days) for the above jobs using Shortest Processing Time rule is

- (A) 20.83
 (B) 23.16
 (C) 125.00
 (D) 139.00

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Jobs	Processing Time(days)
I	4
II	9
III	5
IV	10
V	6
VI	8

46) Minimum shear strain in orthogonal turning with a cutting tool of zero rake angle is

- (A) 0.0
- (B) 0.5
- (C) 1.0
- (D) 2.0

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47) Electrochemical machining is performed to remove material from an iron surface of 20 mm × 20 mm under the following conditions:

Inter electrode gap = 0.2 mm

Supply voltage = 12 V

Specific resistance of electrolyte = 2Ωcm

Atomic weight of iron = 55.85

Valency of Iron = 2

Faraday's constant = 96540 C

The material removal rate (ing/s) is

- (A) 0.3471
- (B) 3.471
- (C) 34.71
- (D) 347.1

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48) Match the following:

NC Code	Definition
P. M05	1. Absolute coordinate system
Q. G01	2. Dwell
R. G04	3. Spindle stop
S. G90	4. Linear interpolation

- (A) P-2, Q-3, R-4, S-1
- (B) P-3, Q-4, R-2, S-1
- (C) P-3, Q-4, R-1, S-2
- (D) P-4, Q-3, R-2, S-1

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49) What are the upper and lower limits of the shaft represented by 60 fg?

Use the following data:

Diameter 60 lies in the diameter step of 50-80 mm.

Fundamental tolerance unit i in $\mu\text{m} = 0.45D^{1/3} + 0.001D$, where D is the representative

size in mm;

Tolerance value for IT8 = $25i$. Fundamental deviation for 'f' shaft = $-5.5D^{0.41}$

- (A) Lower limit = 59.924 mm, Upper Limit = 59.970 mm
- (B) Lower limit = 59.954 mm, Upper Limit = 60.000 mm
- (C) Lower limit = 59.970 mm, Upper Limit = 60.016 mm
- (D) Lower limit = 60.000 mm, Upper Limit = 60.046 mm

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50) Match the items in Column I and Column II:

Column I	Column II
P. Metallic Chills	1. Support for the core
Q. Metallic Chaplets	2. Reservoir of the molten metal
R. Riser3.	3. Control cooling of critical sections
S. Exothermic Padding	4. Progressive solidification

- (A) P-1, Q-3, R-2, S-4
- (B) P-3, Q-4, R-2, S-1
- (C) P-1, Q-4, R-2, S-3
- (D) P-4, Q-1, R-2, S-3

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Common Data Questions

Common Data for Questions 51 and 52:

The inlet and the outlet conditions of steam for an adiabatic steam turbine are as indicated in the

The notations are as usually followed.

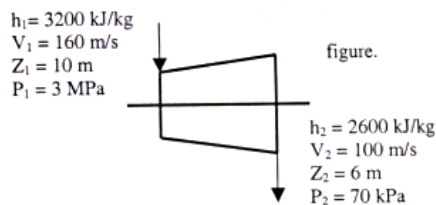


Fig. 50

- 51) If mass flow rate of steam through the turbine is 20 kg/s, the power output of the turbine (in MW) is
- (A) 12.157
 - (B) 12.941
 - (C) 168.001
 - (D) 168.785

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- 52) Assume the above turbine to be part of a simple Rankine cycle. The density of water at the inlet to the pump is 1000 kg/m^3 . Ignoring kinetic and potential energy effects, the specific work (in kJ/kg) supplied to the pump is

- (A) 0.293
- (B) 0.351
- (C) 2.930
- (D) 3.510

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Common Data for Questions 53 and 54:

Radiative heat transfer is intended between the inner surfaces of two very large isothermal parallel metal plates. The upper plate (plate 1) is a black surface and is the warmer, maintained at 727°C . The lower plate (plate 2) is diffuse and gray with emissivity 0.7 and kept at 227°C . Assume sufficiently large surfaces and steady-state; Stefan-Boltzmann constant is $5.67 \times 10^{-8} \text{ W/m}^2\text{K}^4$.

53) The irradiation (inkW/m^2) for the upper plate (plate 1) is

- (A) 2.5
- (B) 3.6
- (C) 17.0
- (D) 19.5

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54) If plate 1 is also diffuse and gray with emissivity 0.8, the net radiation heat exchange (inkW/m^2) between plates 1 and 2 is

- (A) 17.0
- (B) 19.5
- (C) 23.0
- (D) 31.7

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Common Data for Questions 55 and 56 :

Consider the following PERT network: The optimistic time, most likely time and

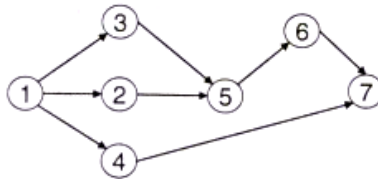


Fig. 54

pessimistic time of all the activities are given in the table below:

Column I	Column II
P. Metallic Chills	1. Support for the core
Q. Metallic Chaplets	2. Reservoir of the molten metal
R. Riser3.	3. Control cooling of critical sections
S. Exothermic Padding	4. Progressive solidifition

55) The critical path duration of the network (in days) is

- (A) 11

(B) 14

(C) 17

(D) 18

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56) The standard deviation of the critical path is

(A) 0.33

(B) 0.55

(C) 0.77

(D) 1.66

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Linked Answer Questions

Statement for Linked Answer Questions 57 and 58:

In a machining experiment, tool life was found to vary with the cutting speed in the following manner:

Cutting speed(m/min)	Tool life(minutes)
60	81
90	36

57) The exponent (n) and constant (k) of Taylor's tool life equation are

- (A) $n = 0.5$ and $k = 540$
- (B) $n = -1$ and $k = 0.74$
- (C) $n = 1$ and $k = 4860$
- (D) $n = -0.5$ and $k = 1.155$

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58) What is the percentage increase in tool life when cutting speed is halved?

- (A) 50%
- (B) 200%
- (C) 300%
- (D) 400%

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Statement for Linked Answer Questions 59 and 60:

A 20° full depth involute spur pinion of 4 mm module and 21 teeth is to transmit 15 kW at 960 rpm. Its face width is 25 mm.

59) The tangential force transmitted (in N) is

- (A) 3552
- (B) 2611
- (C) 1776
- (D) 1305

60) Given the tooth geometry factor is 0.32 and combined effect of dynamic load and allied factors intensifying stress is 1.5, the minimum allowable stress (in MPa) for the gear material is

- (A) 242.0
- (B) 166.5
- (C) 121.0
- (D) 74.0

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