

THE THEORY MECHANISMS OF

GROUP 4

ABESECH ME21B003

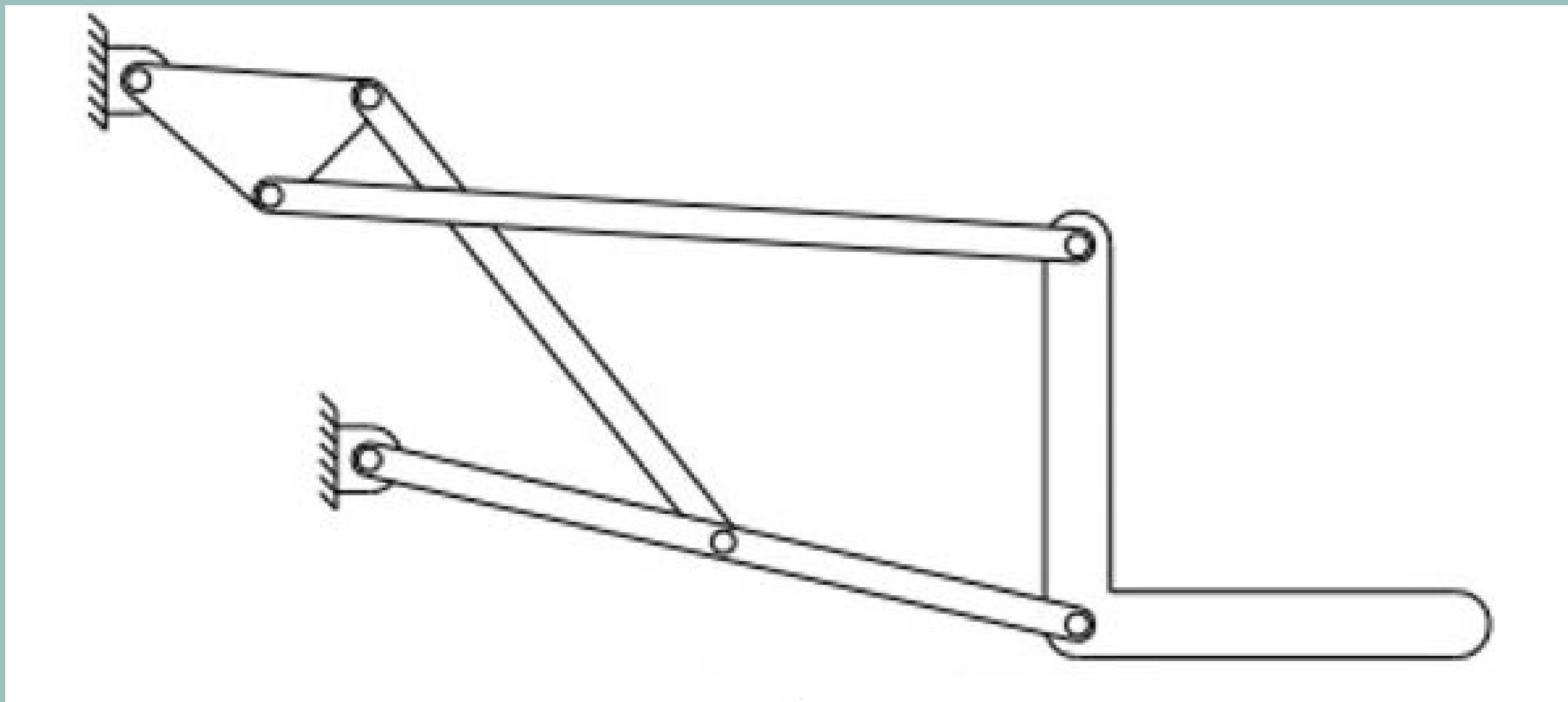
DARSHAN ME21B051

GYOTHIRRAM ME21B071

JAYARAM ME21B079

RITESH ME21B162

OUR MECHANISM

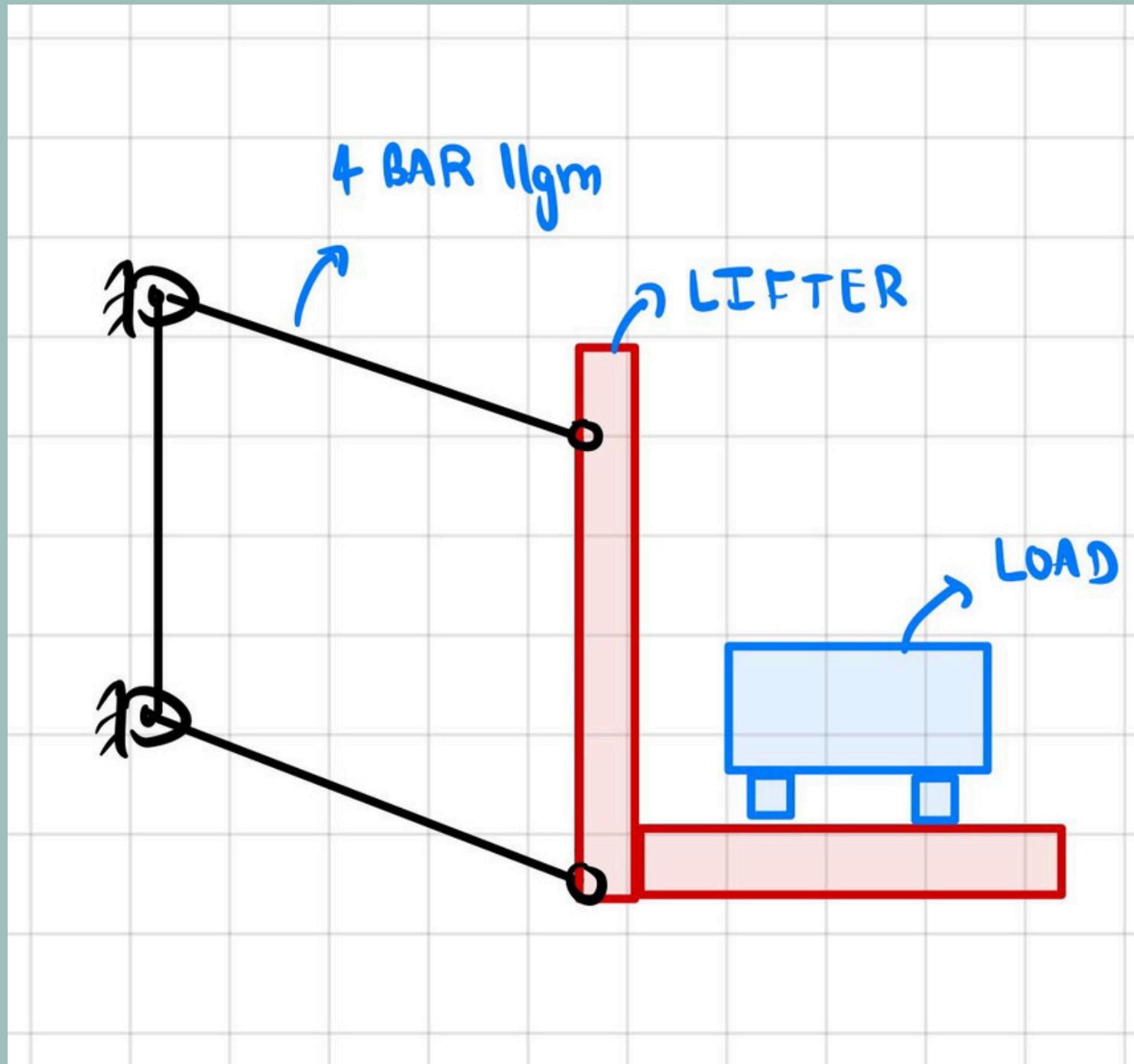


6 BAR LIFTER

Vertically lifts a
load using a
rotary input.

The linkage is in fact an
inversion of Stephenson's
mechanism, and one of the
ternary links acts as our
input.

Why are we not using a simple parallelogram?



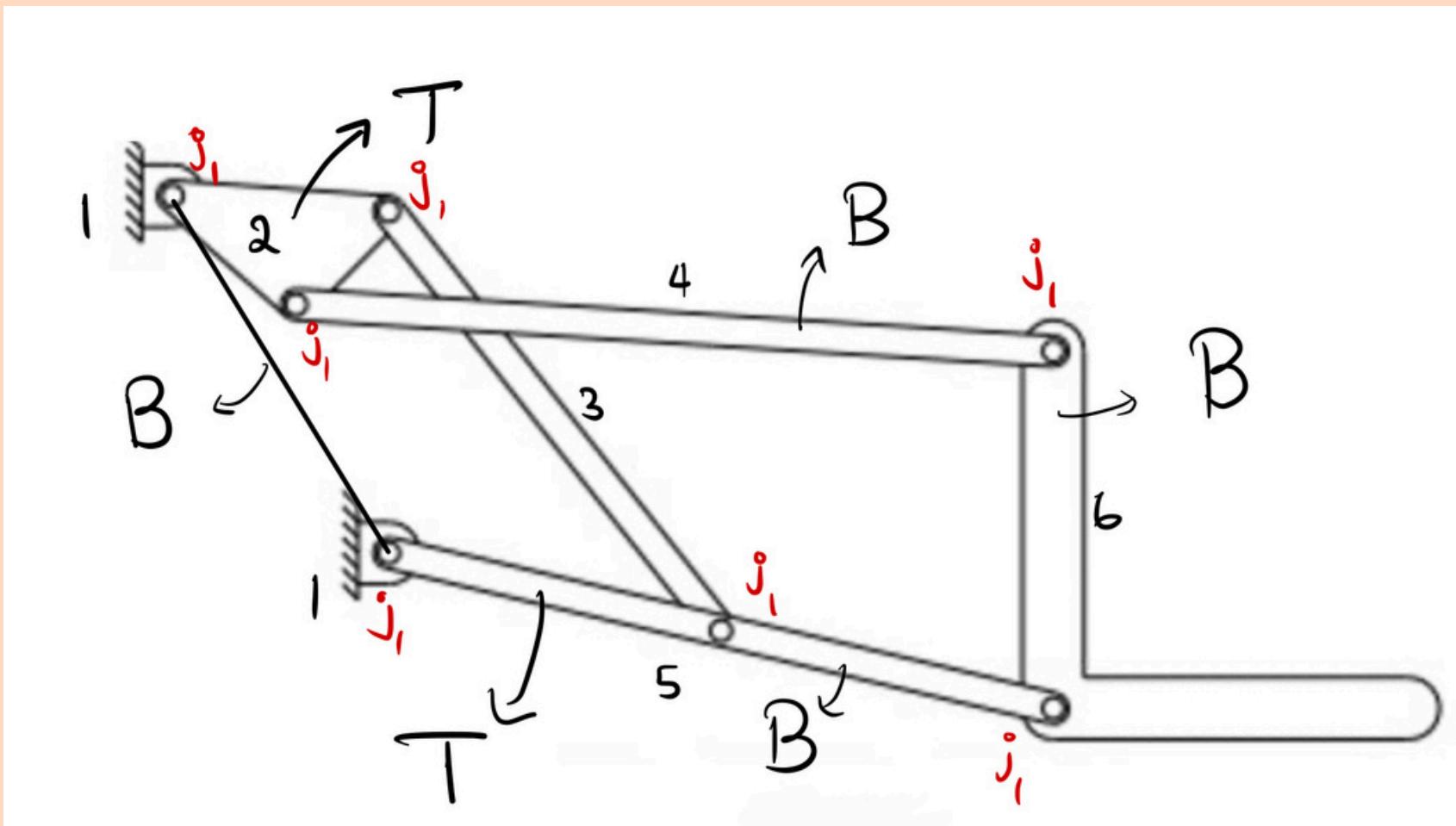
This limits the lifter's range of motion and its payload capacity.

This is harder to control.

MA is better with a 6 bar.

MOBILITY ANALYSIS

Our lifter has 2 ternary links, 4 binary links, and a mobility of 1.



T - ternary links, B - binary links

$$n_a = 4 \quad n_3 = 2 \Rightarrow n = 6$$

no. of j₁ joints = 7

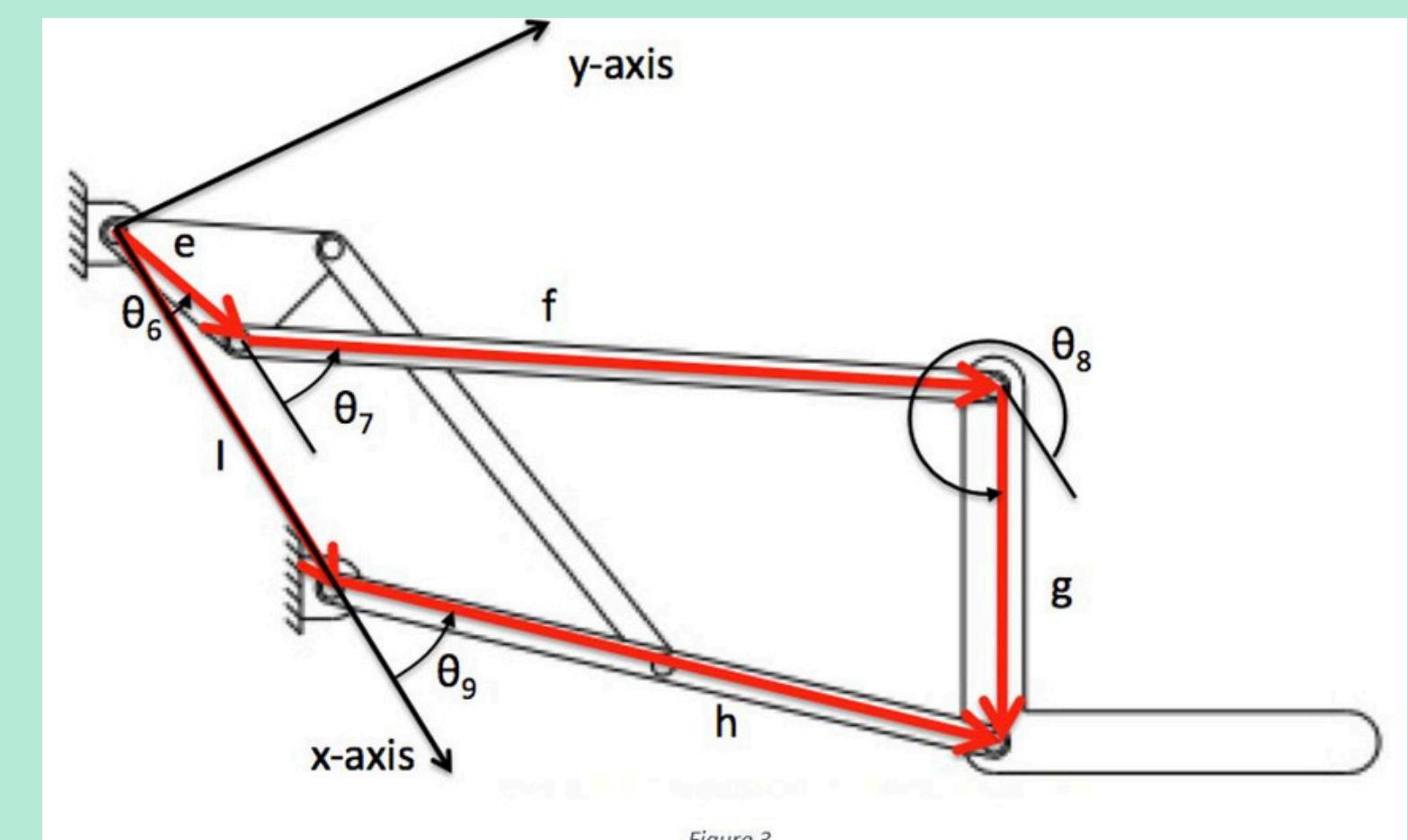
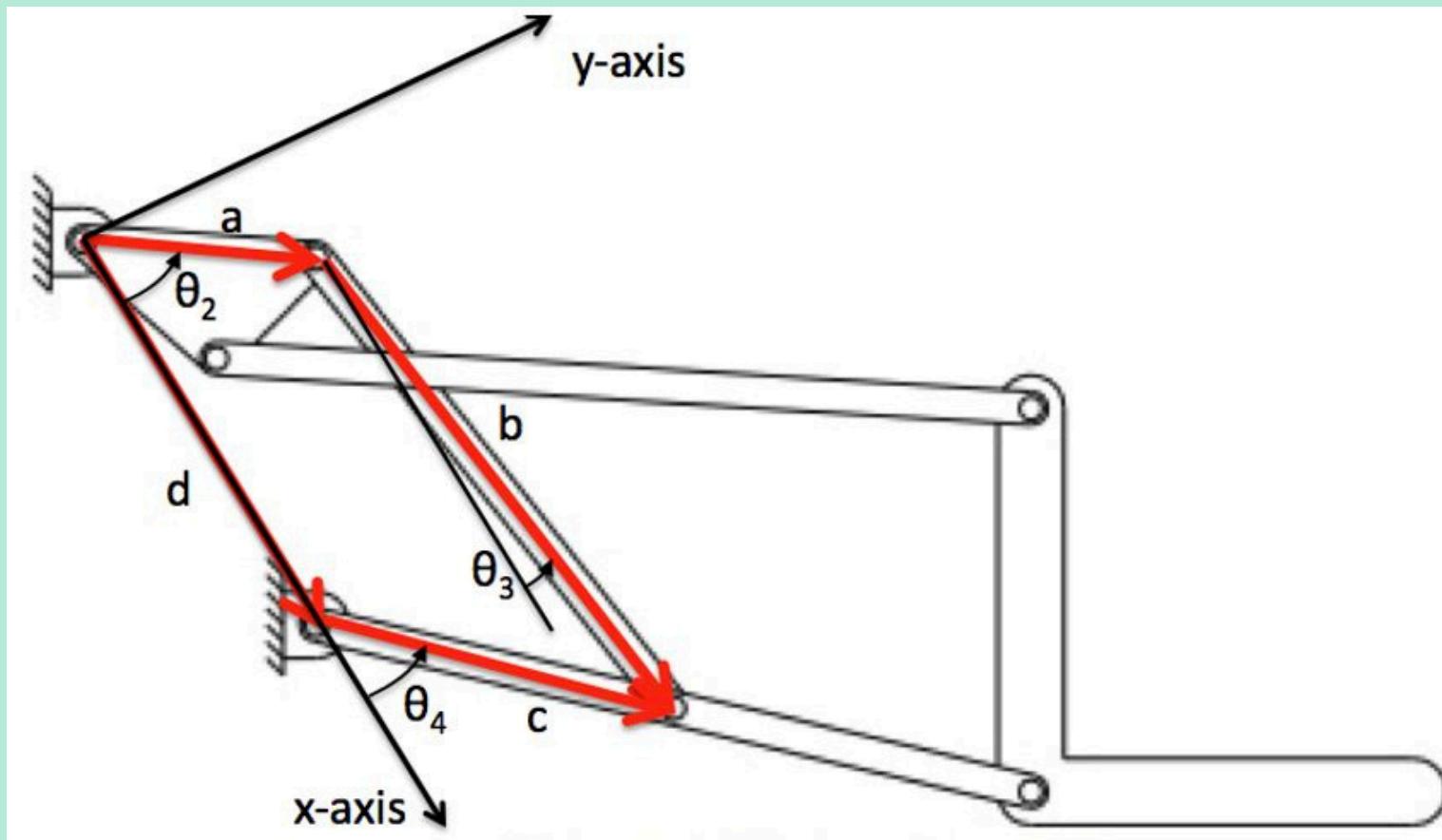
$$m = 3(n-1) - 2j_1$$

$$m = 3(6-1) - 2(7)$$

$$\boxed{m=1}$$

POSITION ANALYSIS

The ground link supports a 4 bar and a 5 bar



the 4 bar drives the 5 bar

PART 1

DYAD FORM SYNTHESIS

Synthesis using
dyad form
equations and
free choices.

Steps Involved

Dyad Form - write the dyad form and
loop closure equations.

Constraints and Free Choices - the
lifter's orientation does not change

Results - solving equations with our
choices and constraints

DYAD FORM EQUATIONS

Dyad form Synthesis:

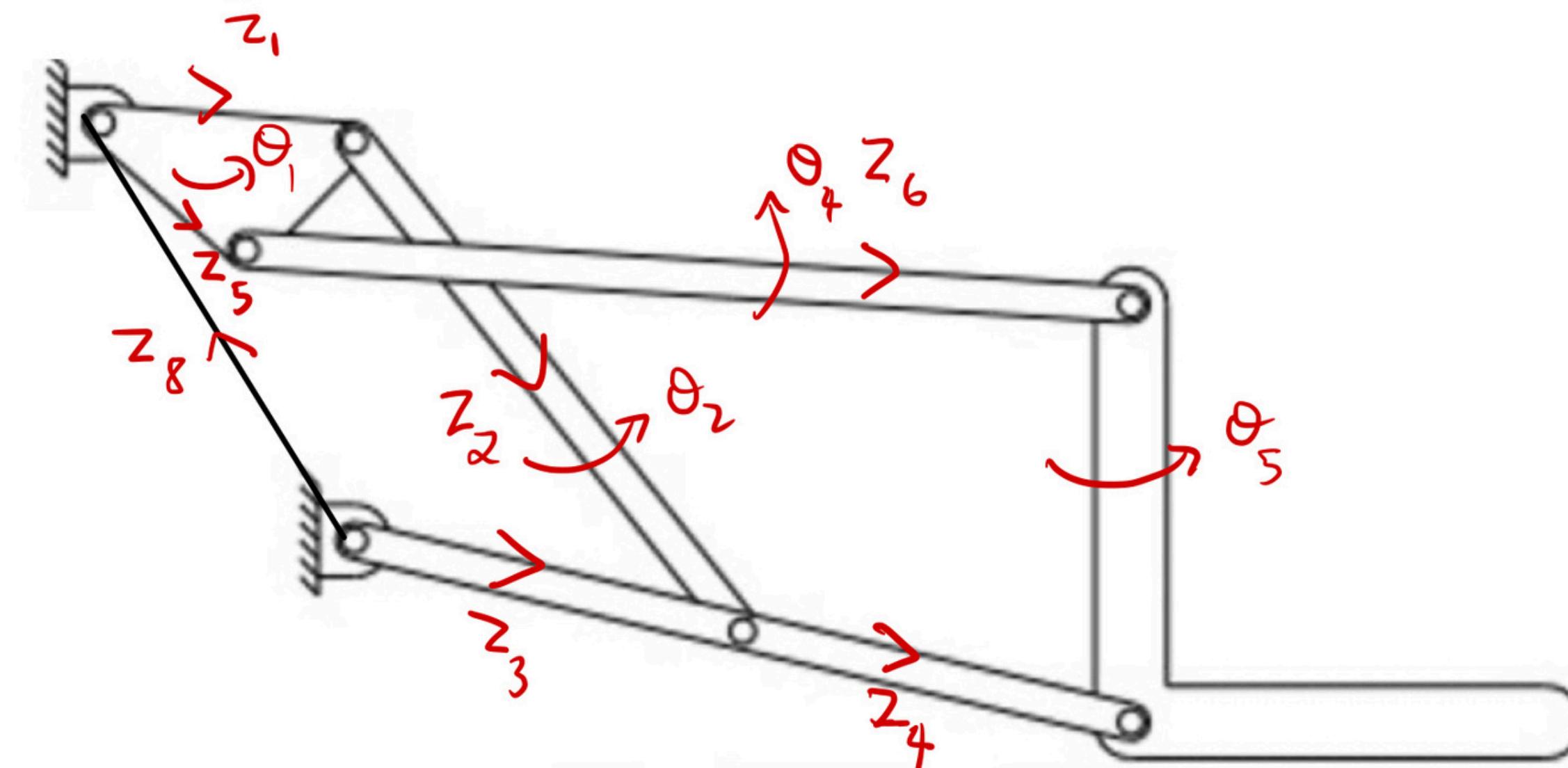


Figure 1

DYAD FORM EQUATIONS

Dyad form Equations:

$$\text{loop 1 : } z_1(e^{i\theta_1}-1) + z_2(e^{i\theta_2}-1) + z_4(e^{i\theta_3}-1) = \delta$$

$$\text{loop 2 : } z_5(e^{i\theta_1}-1) + z_6(e^{i\theta_4}-1) + z_7(e^{i\theta_5}-1) = \delta$$

$$\text{loop 3 : } z_3(e^{i\theta_3}-1) + z_4(e^{i\theta_3}-1) = \delta \xrightarrow{*} \text{Redundant}$$

Loop Closure Equation:

$$\text{loop 1 : } z_1 + z_2 - z_3 + z_8 = 0$$

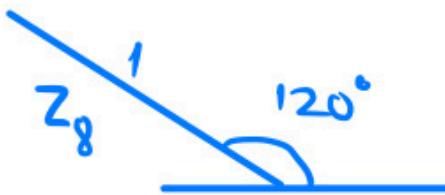
$$\text{loop 2 : } z_1 e^{i\theta_1} + z_2 e^{i\theta_2} - z_3 e^{i\theta_3} + z_8 = 0$$

SOLVING...

$$\text{loop 2} - \text{loop 1} = z_1(e^{i\theta_1} - 1) + z_2(e^{i\theta_2} - 1) - z_3(e^{i\theta_3} - 1) = 0$$

Assuming $z_8 = e^{i\frac{2\pi}{3}}$

$$z_3 = z_4$$



$$\delta' = z_3(e^{i\theta_3} - 1)$$

$$z_1(e^{i\theta_{1i}} - 1) + z_2(e^{i\theta_{2i}} - 1) = \delta'_i \rightarrow \text{for } i^{\text{th}} \text{ position}$$

On Representing as a matrix:

$$(z_1 \ z_2 \ 1) \begin{pmatrix} e^{i\theta_{11}} - 1 & e^{i\theta_{21}} - 1 & 1 \\ e^{i\theta_{12}} - 1 & e^{i\theta_{22}} - 1 & 1 \\ -\delta'_1 & -\delta'_2 & -z_3 + z_8 \end{pmatrix} = 0$$

$$\therefore A Z = 0$$

$$|A| = 0$$

SOLVING...

$$\begin{vmatrix} e^{i\theta_{11}} - 1 & e^{i\theta_{21}} - 1 & 1 \\ e^{i\theta_{12}} - 1 & e^{i\theta_{22}} - 1 & 1 \\ -\delta'_1 & -\delta'_2 & -z_3 + z_8 \end{vmatrix} = 0$$

Assumptions:

$$\theta_{11} = 20^\circ \quad \theta_{31} = 18^\circ$$

$$\Rightarrow \delta'_1 = z_3(e^{i18^\circ} - 1) = 0.156 + 0.215i$$

$$\theta_{12} = 30^\circ \quad \theta_{32} = 27^\circ$$

$$\Rightarrow \delta'_2 = z_3(e^{i27^\circ} - 1) = 0.208 + 0.339i$$

$$z_3 = 0.853 e^{-i\frac{\pi}{4}}$$

On solving (2 eqn's 2 unknowns) we get: $\theta_{21} = 3.194^\circ$ $\theta_{22} = 25.383^\circ$

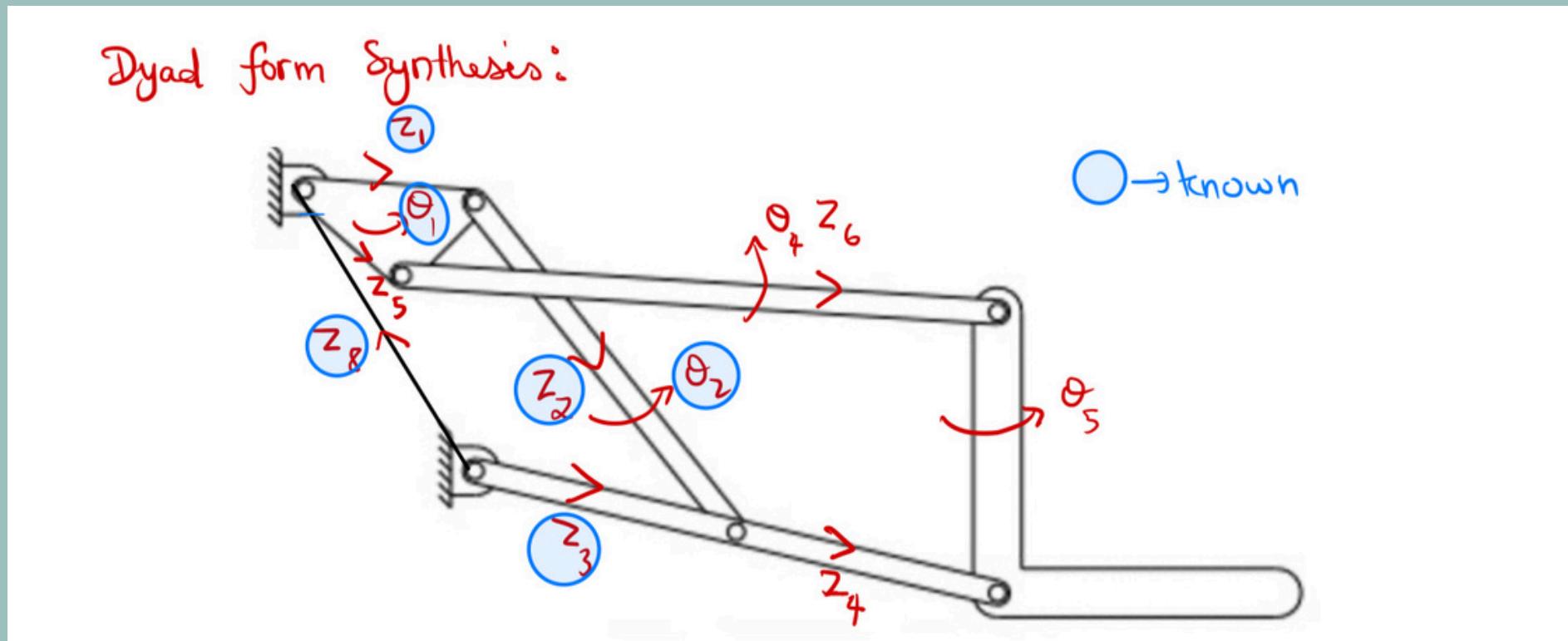
$$\Rightarrow z_1(e^{i20^\circ} - 1) + z_2(e^{i\theta_{21}} - 1) = \delta'_1 = 0.156 + 0.215i$$

$$z_1(e^{i30^\circ} - 1) + z_2(e^{i\theta_{22}} - 1) = \delta'_2 = 0.208 + 0.339i$$

$i(-8^\circ)$ $i(-52^\circ)$

On solving (4 eqn's 4 unknowns) we get: $z_1 = 0.532 e^{i(-8^\circ)}$ $z_2 = 1.386 e^{i(-52^\circ)}$

SOLVING...



Similar to the previous method, we solve for the remaining unknowns using the following determinant:

$$\begin{pmatrix} e^{i\theta_{11}} - 1 & e^{i\theta_{41}} - 1 & 1 \\ e^{i\theta_{12}} - 1 & e^{i\theta_{42}} - 1 & 1 \\ -\delta_1 & -\delta_2 & z_7 - z_3 - z_4 + z_8 \end{pmatrix} = 0$$

RESULTS

We took the ground link to be 30° to vertical
Scaling to a reasonable level

$$Z_1=25.76$$

$$Z_2=67.075$$

$$Z_3=41.303$$

$$Z_3+Z_4=82.587 \text{ (link 5)}$$

$$Z_5=20.244$$

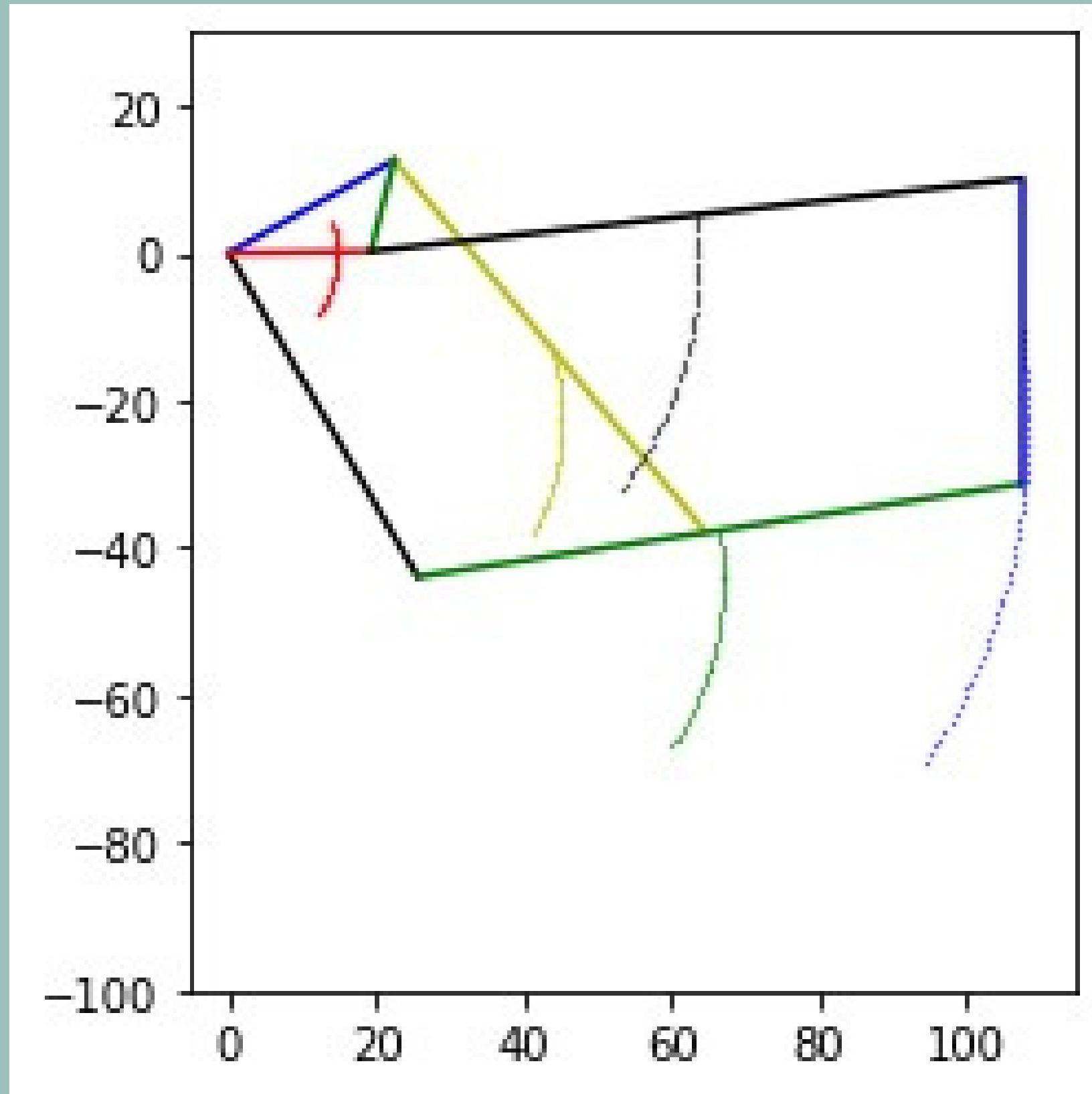
$$Z_6=86.755$$

$$Z_7=42.414$$

$$Z_8=48.393$$

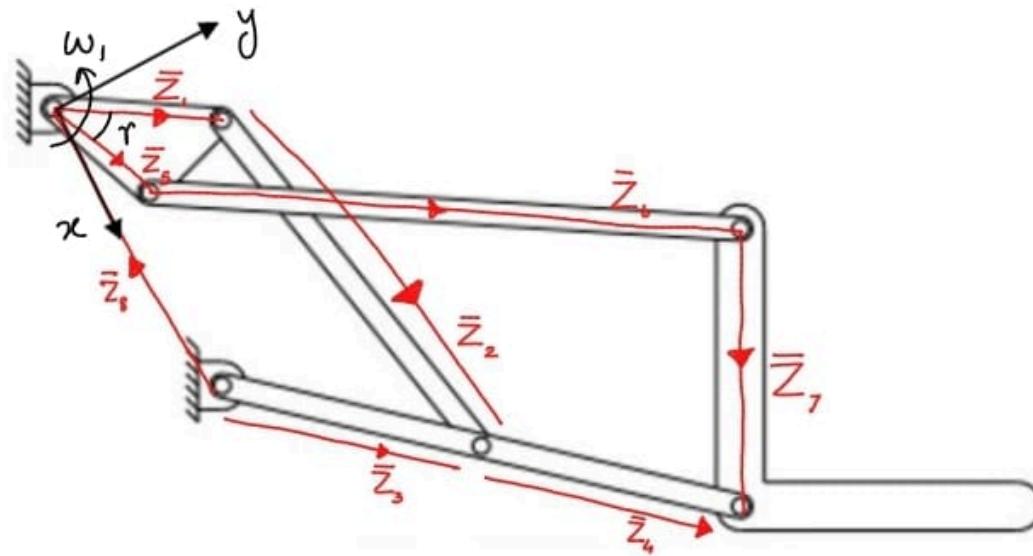
(all units are in cm)

COG TRAJECTORIES



VELOCITY ANALYSIS

Velocity Analysis



$$Z_j = \eta_j e^{i\theta_j}$$

loop-1

$$Z_1 + Z_2 - Z_3 + Z_8 = 0$$

$$\eta_1 e^{i\theta_1} + \eta_2 e^{i\theta_2} - \eta_3 e^{i\theta_4} + \eta_8 = 0 \quad (\theta_3 = \theta_4)$$

d. $w \cdot \eta \cdot t$

$$i\eta_1 w_1 e^{i\theta_1} + i\eta_2 w_2 e^{i\theta_2} - i\eta_3 w_4 e^{i\theta_4} + 0 = 0 \quad \text{--- (1)}$$

loop-2

$$Z_5 + Z_6 + Z_7 - Z_4 - Z_3 + Z_8 = 0$$

$$\eta_5 e^{i\theta_5} + \eta_6 e^{i\theta_6} + \eta_7 e^{i\theta_7} - (\eta_4 + \eta_3) e^{i\theta_4} + \eta_8 = 0$$

differentiate with respect to time

$$i\eta_5 w_5 e^{i\theta_5} + i\eta_6 w_6 e^{i\theta_6} + i\eta_7 w_7 e^{i\theta_7} - i(\eta_4 + \eta_3) w_4 e^{i\theta_4} = 0$$

$$i\eta_5 w_1 e^{i(\theta_1 - \gamma)} + i\eta_6 w_6 e^{i\theta_6} + i\eta_7 w_7 e^{i\theta_7} - i(\eta_3 + \eta_4) w_4 e^{i\theta_4} = 0 \quad \text{--- (2)}$$

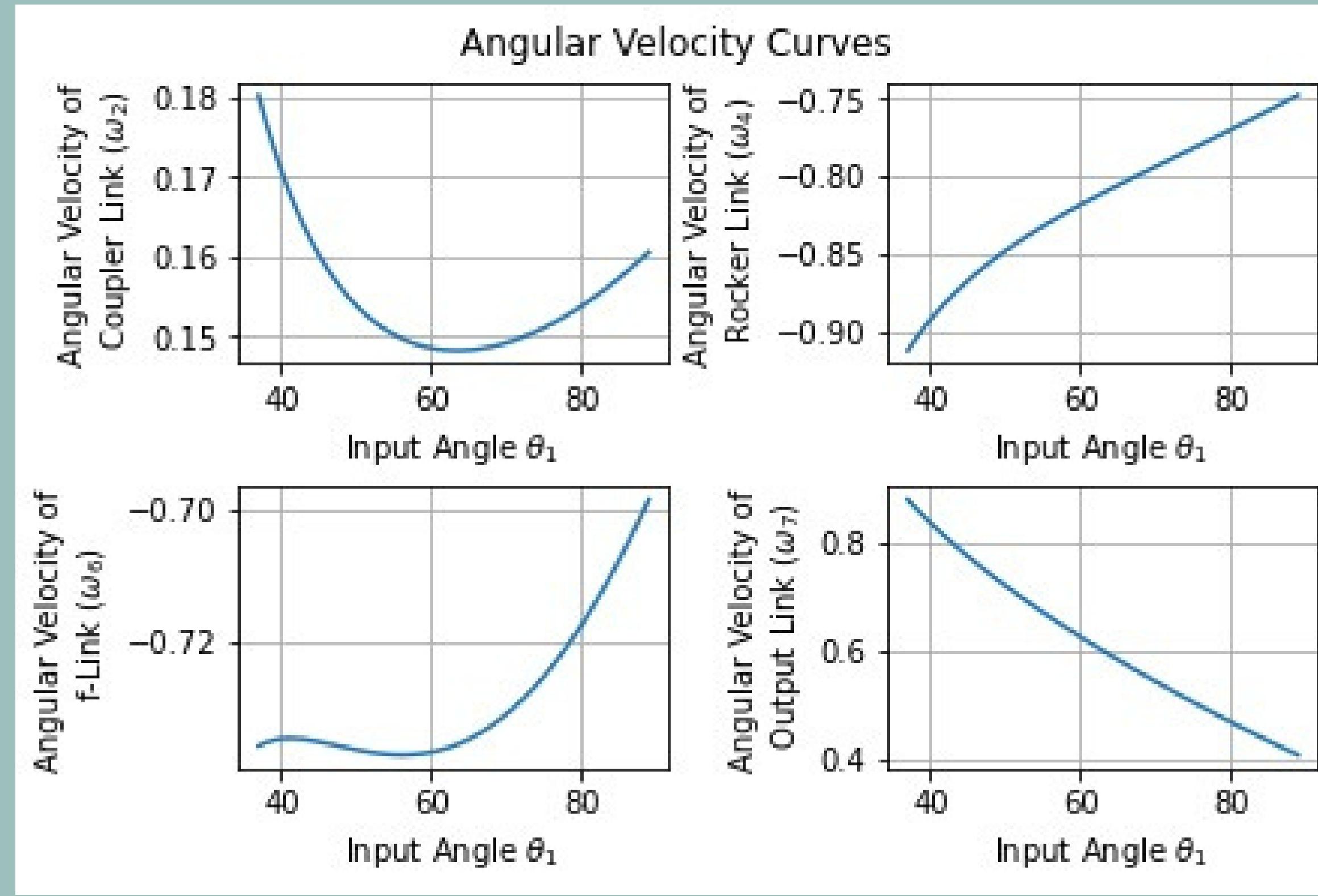
VELOCITY ANALYSIS

known : ω_1 unknown $\omega_2 \omega_4 \omega_6 \omega_7$
 + eqn + unknown

$$\begin{bmatrix} i\omega_1 r_1 \cos\theta_1 \\ i\omega_1 r_1 \sin\theta_1 \\ i\omega_1 r_5 \cos(\theta_1 - \gamma) \\ i\omega_1 r_5 \sin(\theta_1 - \gamma) \end{bmatrix}$$

$$= \begin{bmatrix} -ir_2 \cos\theta_2 & -ir_3 \cos\theta_4 & 0 & 0 \\ -ir_2 \sin\theta_2 & -ir_3 \sin\theta_4 & 0 & 0 \\ 0 & +i(r_3+r_4) \cos\theta_4 & -ir_6 \cos\theta_6 & -ir_7 \cos\theta_7 \\ 0 & +i(r_3+r_4) \sin\theta_4 & -ir_6 \sin\theta_6 & -ir_7 \sin\theta_7 \end{bmatrix} \begin{bmatrix} \omega_2 \\ \omega_4 \\ \omega_6 \\ \omega_7 \end{bmatrix}$$

VELOCITY ANALYSIS



ACCELERATION ANALYSIS

Acceleration Analysis :

Loop closure :

$$\textcircled{1} \quad Z_1 + Z_2 - Z_3 + Z_8 = 0$$

$$g_1 e^{i\theta_1} + g_2 e^{i\theta_2} - g_3 e^{i\theta_4} + g_8 = 0 \quad (\theta_3 = \theta_4)$$

$$\textcircled{2} \quad Z_5 + Z_6 + Z_7 - Z_4 - Z_3 + Z_8 = 0$$

$$g_5 e^{i\theta_5} + g_6 e^{i\theta_6} + g_7 e^{i\theta_7} - (g_4 + g_3) e^{i\theta_4} + g_8 = 0$$

Velocity eqns :

$$\textcircled{1} \quad i g_1 \omega_1 e^{i\theta_1} + i g_2 \omega_2 e^{i\theta_2} - i g_3 \omega_4 e^{i\theta_4} + 0 = 0$$

$$\textcircled{2} \quad i g_5 \omega_5 e^{i\theta_5} + i g_6 \omega_6 e^{i\theta_6} + i g_7 \omega_7 e^{i\theta_7} - i(g_4 + g_3) \omega_4 e^{i\theta_4} = 0$$

$$i g_5 \omega_1 e^{i(\theta_1 - r)} + i g_6 \omega_6 e^{i\theta_6} + i g_7 \omega_7 e^{i\theta_7} - i(g_4 + g_3) \omega_4 e^{i\theta_4} = 0$$

Acceleration eqns

$$\textcircled{1} \quad (i\alpha_1 - \omega_1^2) g_1 e^{i\theta_1} + (i\alpha_2 - \omega_2^2) g_2 e^{i\theta_2} - (i\alpha_4 - \omega_4^2) g_3 e^{i\theta_4} = 0$$

$$\textcircled{2} \quad (i\alpha_1 - \omega_1^2) g_5 e^{i(\theta_1 - r)} + (i\alpha_6 - \omega_6^2) g_6 e^{i\theta_6} + (i\alpha_7 - \omega_7^2) g_7 e^{i\theta_7} - (i\alpha_4 - \omega_4^2) (g_4 + g_3) e^{i\theta_4} = 0$$

ACCELERATION ANALYSIS

knowns α_1 Unknowns $\rightarrow \alpha_2 \alpha_4 \alpha_6 \alpha_7$

$$(i\alpha_1 - \omega_1^2) \alpha_1 e^{i\theta_1} = (i\alpha_4 - \omega_4^2) \alpha_3 e^{i\theta_4} - (i\alpha_2 - \omega_2^2) \alpha_2 e^{i\theta_2}$$

$$(i\alpha_1 - \omega_1^2) \alpha_1 e^{i\theta_1} + \omega_4^2 \alpha_3 e^{i\theta_4} - \omega_2^2 \alpha_2 e^{i\theta_2} = i\alpha_4 \alpha_3 e^{i\theta_4} - i\alpha_2 \alpha_2 e^{i\theta_2}$$

$$(i\alpha_1 - \omega_1^2) \alpha_5 e^{i(\theta_1 - r)} = (i\alpha_4 - \omega_4^2) (\alpha_3 + \alpha_4) e^{i\theta_4} - (i\alpha_6 - \omega_6^2) \alpha_6 e^{i\theta_6}$$

$$- (i\alpha_7 - \omega_7^2) \alpha_7 e^{i\theta_7}$$

$$(i\alpha_1 - \omega_1^2) \alpha_5 e^{i(\theta_1 - r)} + \omega_4^2 (\alpha_4 + \alpha_3) e^{i\theta_4} - \omega_6^2 \alpha_6 e^{i\theta_6} - \omega_7^2 \alpha_7 e^{i\theta_7}$$

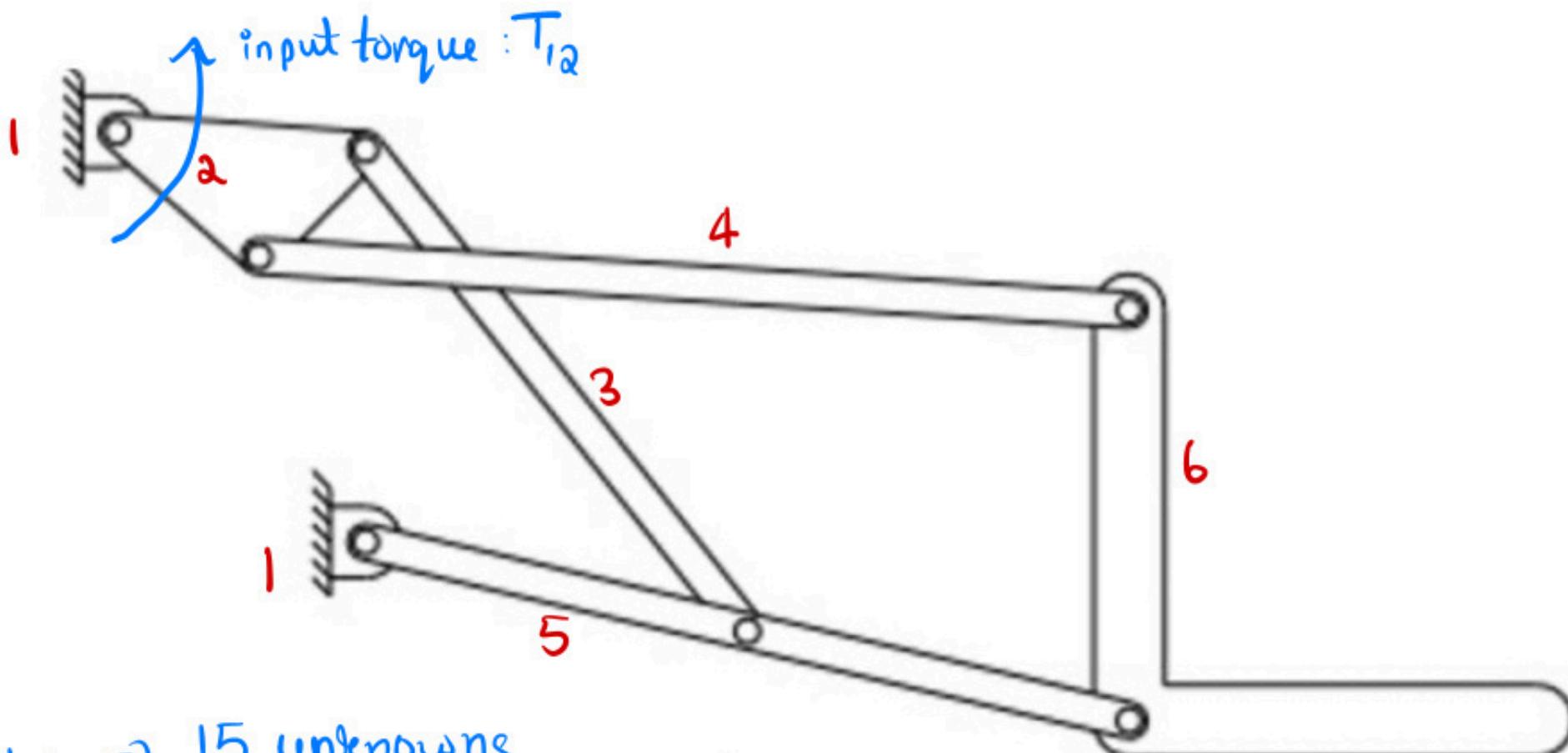
$$= i\alpha_4 (\alpha_3 + \alpha_4) e^{i\theta_4} - i\alpha_6 \alpha_6 e^{i\theta_6}$$

$$- i\alpha_7 \alpha_7 e^{i\theta_7}$$

$$\begin{bmatrix} (i\alpha_1 - \omega_1^2) \alpha_1 \cos\theta_1 & + \omega_4^2 \alpha_3 \cos\theta_4 & - \omega_2^2 \alpha_2 \cos\theta_2 \\ (i\alpha_1 - \omega_1^2) \alpha_1 \sin\theta_1 & + \omega_4^2 \alpha_3 \sin\theta_4 & - \omega_2^2 \alpha_2 \sin\theta_2 \\ (i\alpha_1 - \omega_1^2) \alpha_5 \cos(\theta_1 - r) & + \omega_4^2 (\alpha_3 + \alpha_4) \cos\theta_4 & - \omega_6^2 \alpha_6 \cos\theta_6 - \omega_7^2 \alpha_7 \cos\theta_7 \\ (i\alpha_1 - \omega_1^2) \alpha_5 \sin(\theta_1 - r) & + \omega_4^2 (\alpha_3 + \alpha_4) \sin\theta_4 & - \omega_6^2 \alpha_6 \sin\theta_6 - \omega_7^2 \alpha_7 \sin\theta_7 \end{bmatrix}$$

$$= \begin{bmatrix} -i\alpha_2 \cos\theta_2 & i\alpha_3 \cos\theta_4 & 0 & 0 \\ -i\alpha_2 \sin\theta_2 & i\alpha_3 \sin\theta_4 & 0 & 0 \\ 0 & i(\alpha_3 + \alpha_4) \cos\theta_4 & -i\alpha_6 \cos\theta_6 & -i\alpha_7 \cos\theta_7 \\ 0 & i(\alpha_3 + \alpha_4) \sin\theta_4 & -i\alpha_6 \sin\theta_6 & -i\alpha_7 \sin\theta_7 \end{bmatrix} \begin{bmatrix} \alpha_2 \\ \alpha_4 \\ \alpha_6 \\ \alpha_7 \end{bmatrix}$$

FORCE ANALYSIS



5 links \Rightarrow 15 unknowns

Each link: $\sum F_x = \sum F_y = \sum \gamma = 0 \Rightarrow 3 \text{ eqn's}$

\therefore 15 unknowns 15 equations

N_x, N_y from ground $\Rightarrow 4$

$F_{ij} \& -F_{ji}$ b/w ith & jth link. $\Rightarrow 10$

$T_{12} \Rightarrow 1$

PART 2

SIMULATION

Synthesis
using iterative
methods.

Steps Involved

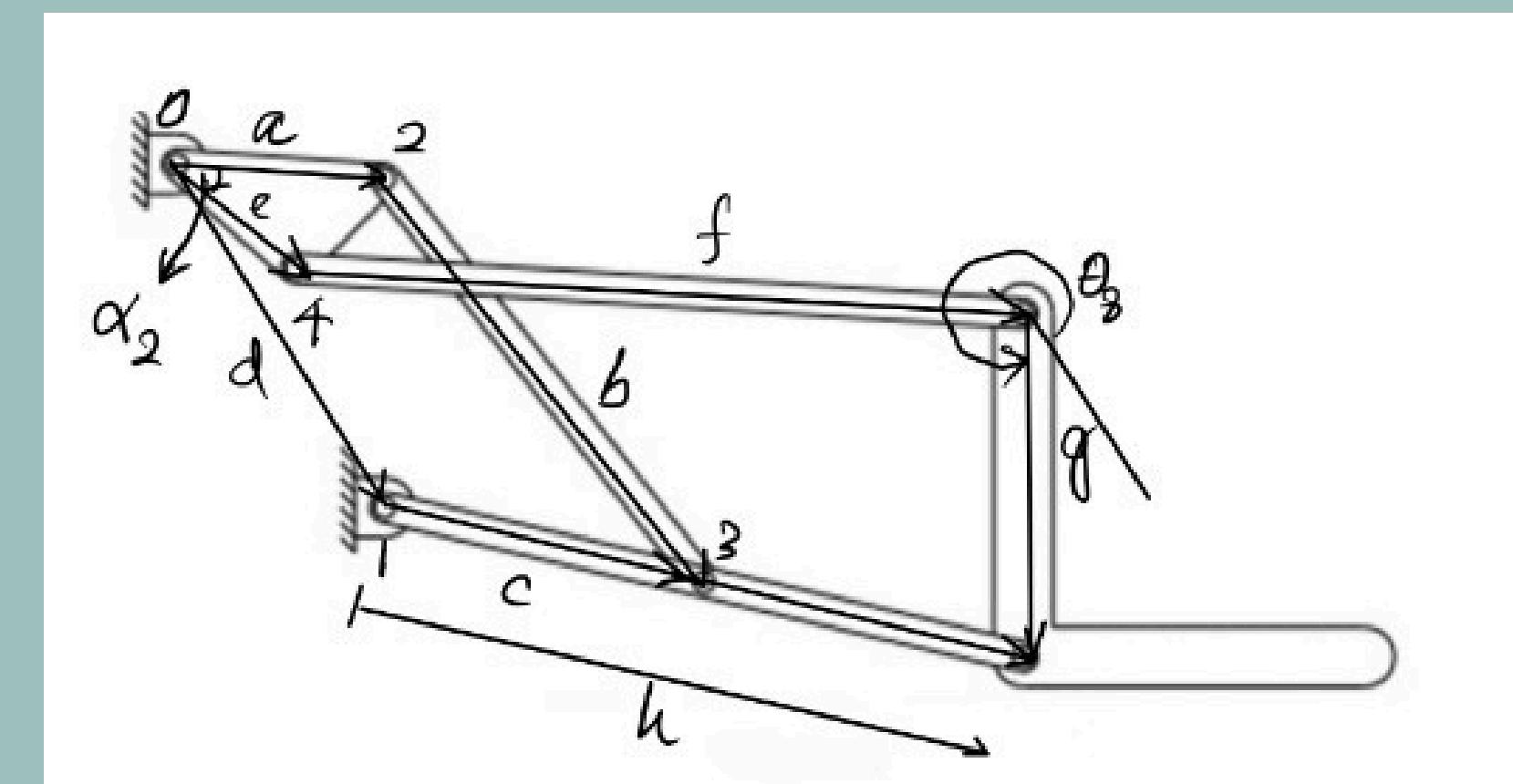
Geometrical Analysis - understanding
how the mechanism is spaced out

Constraints - the lifter's orientation
is constant

Python Code - iteratively solving
equations with the conditions

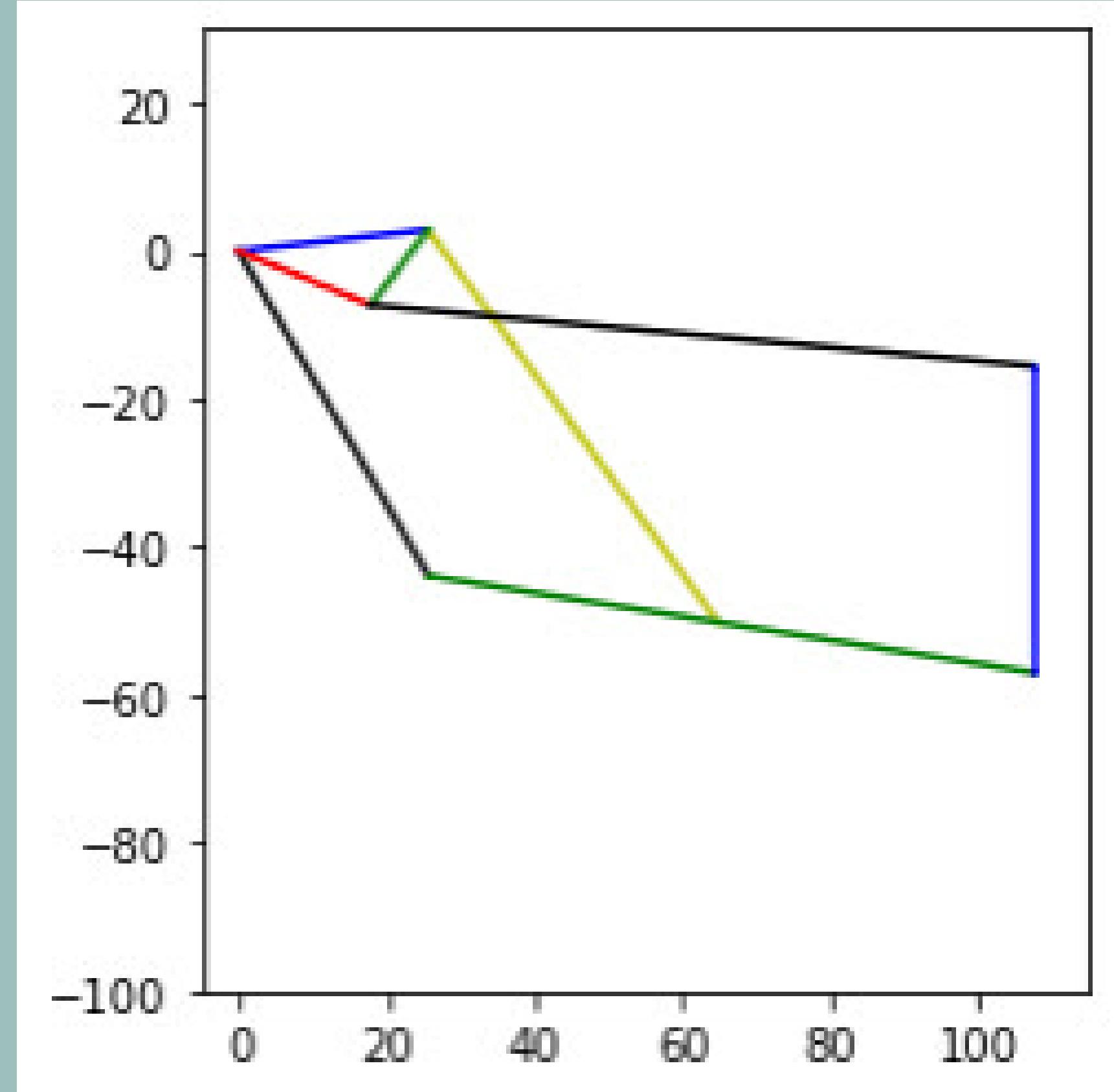
METHOD OF SIMULATION

The code iteratively generates 100 sets of 6-bar linkage link lengths within 5 units of the original lengths, validates them through position synthesis, and selects the configuration with the highest output link range of motion as the optimal solution.

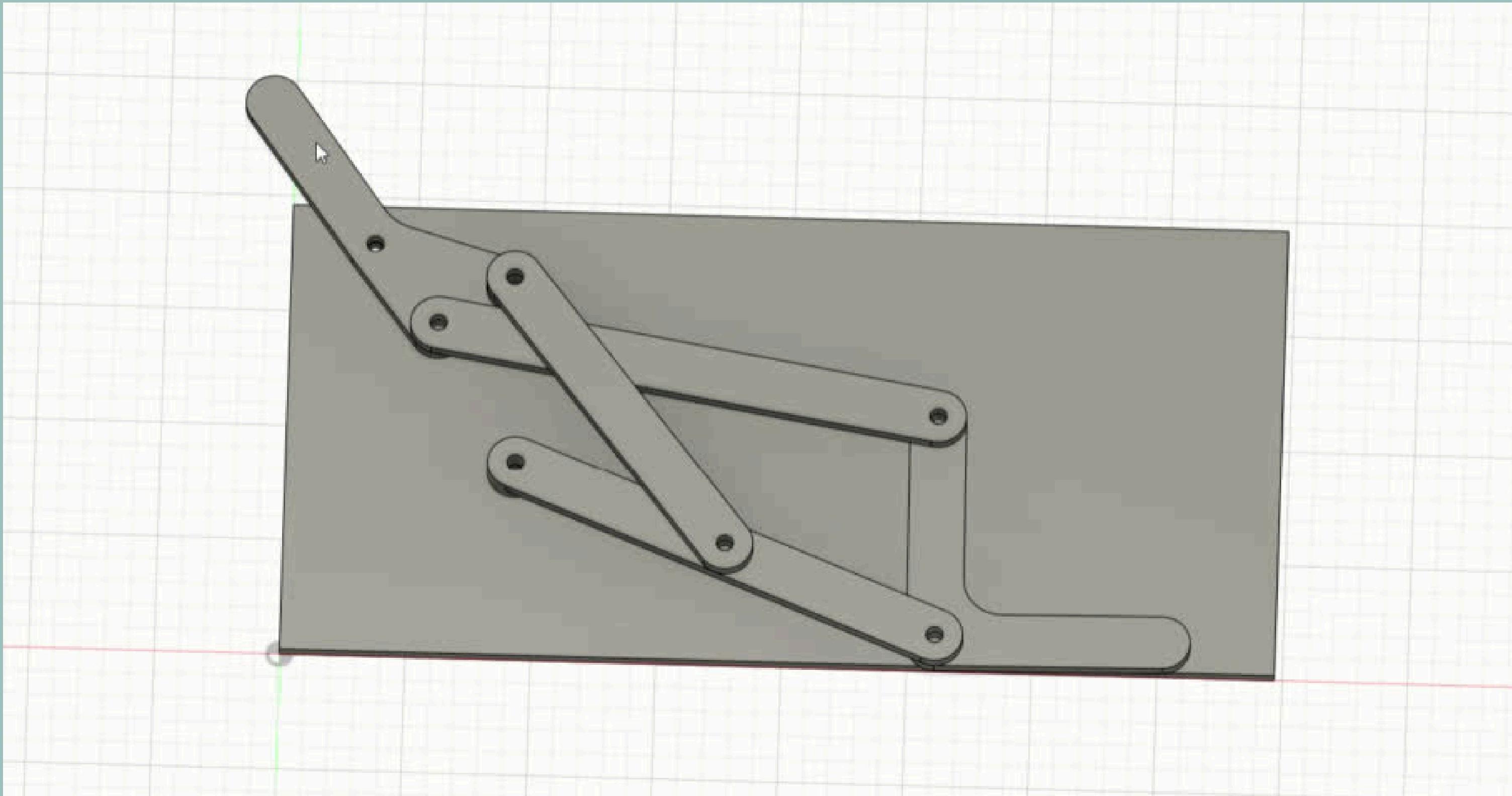


SIMULATION

With the results obtained in dyad form synthesis we used an iterative process to simulate the mechanism as shown in the figure.



CAD MODEL



Thank You