

**IIT Hyderabad**

# **Mini Project Report**

**Submitted by:**

ME21BTECH11001 Abhishek Ghosh

ME21BTECH11002 Aditya Verma

ME21BTECH11012 C Bharath

ME21BTECH11019 Harshit Shambharkar

ME21BTECH11028 Loukik Kalbande

**ME3425: Mini Project**

Mechanical Engineering

30.04.2025

**Submitted to:**

Dr. Karri Badarinath

# 1 Shaft Analysis

## 1.1 Calculations

Power (P):

$$P = \frac{3}{4} \times 1hp = \frac{3}{4} \times 746 = 559.5W \quad (1)$$

Given:

$$N = 300 \text{ rpm}, \quad \rho = 786 \times 10^{-8} \text{ kg/mm}^3 \text{ (mild steel)} \quad (2)$$

Torque: (Considering 25% overloading)

$$T_{max} = \frac{P \times 60,000 \times 1.25}{2\pi N} \approx 22261.797 \text{ N-mm} \quad (3)$$

Maximum Shear Stress:

$$\tau_{s,max} = 40 \text{ N/mm}^2 \quad (4)$$

Shaft Diameter:

$$D_{shaft} = \left( \frac{16T_s}{\pi\tau_s} \right)^{\frac{1}{3}} \times 1.4 = 20 \text{ mm} \quad (5)$$

Shear Stress in Shaft:

$$\tau_{shaft} = \frac{16T}{\pi D^3} = 14.17 \text{ N/mm}^2 \quad (6)$$

$$\tau_{shaft} < \tau_{s,max} = 40 \text{ N/mm}^2 \text{ (within permissible limit)} \quad (7)$$

Bending Stress in Shaft:

$$\sigma_{b,shaft} = \frac{32(M_{shaft,max})}{\pi(D_{shaft})^3} = 0.764 \text{ N/mm}^2 \quad (8)$$

Von Mises Stress:

$$\sigma_{eq} = \sqrt{\sigma_b^2 + 3\tau^2} = 24.55 \text{ N/mm}^2 < 280 \text{ N/mm}^2 \text{ (yield permissible)} \quad (9)$$

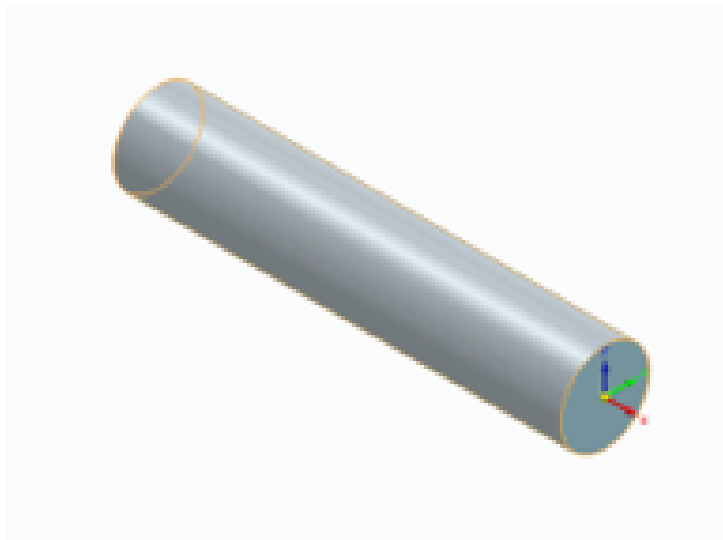


Figure 1: Shaft

## 1.2 Plots

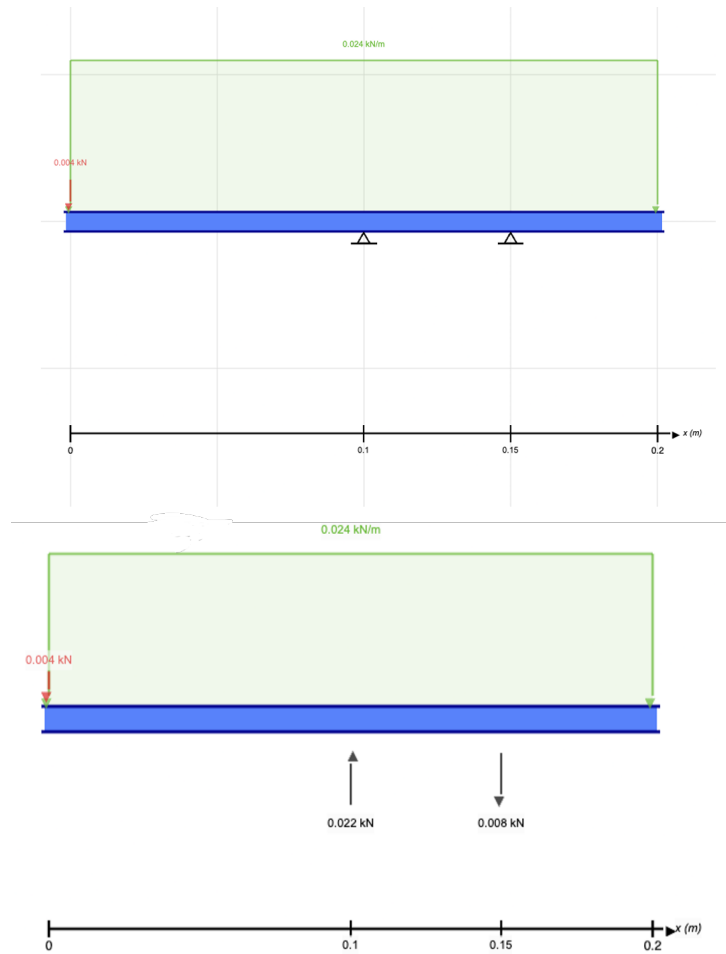


Figure 2: Free Body Diagram

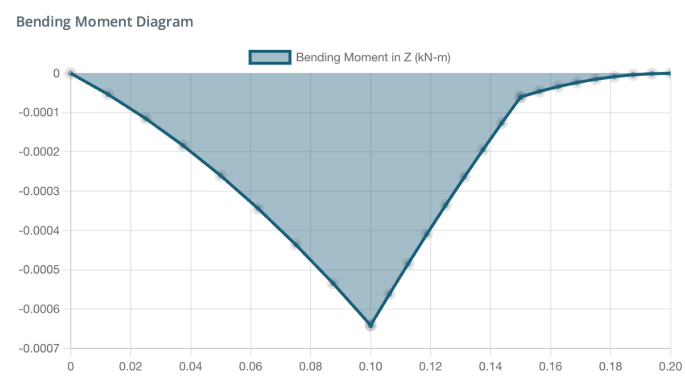


Figure 3: Bending Moment Diagram

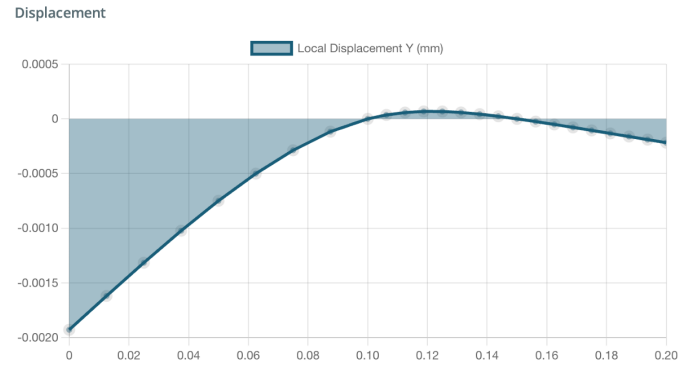


Figure 4: Displacement Diagram

## 2 Hub

Given:

$$D_o = 85 \text{ mm}, \quad D_i = 20 \text{ mm} \quad (10)$$

Shear Stress at Hub:

$$\tau_{Hub} = \frac{16T_{max}D_o}{\pi(D_o^4 - D_i^4)} = 0.185 \text{ N/mm}^2 \quad (11)$$

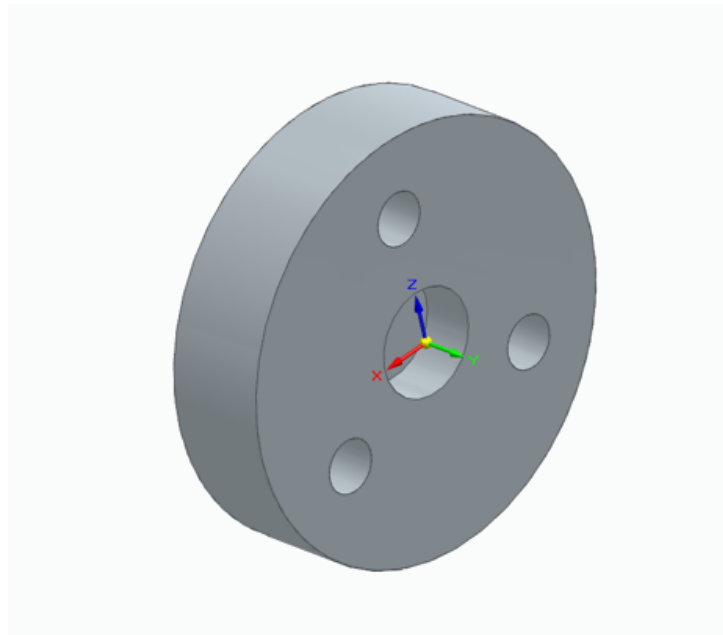


Figure 5: Hub

### 3 Elbow Rod

#### 3.1 Calculations

Torque in Rod:

$$T_{rod} = \frac{T_{max}}{3} = 7420.5 \text{ N-mm} \quad (12)$$

Evaluating at  $D = 10\text{mm}$ ,  $L_{rod} = 90\text{mm}$

Moment at Elbow:

$$M_{elbow} = 24.9 \text{ N-mm} \quad (13)$$

Bending Stress at Elbow:

$$\sigma_{b,elbow,self-wt} = \frac{32M_{elbow}}{\pi D_{elbow}^3} = 0.25 \text{ N/mm}^2 \quad (14)$$

Axial Force in Rod:

$$F_{rod} = \frac{T_{rod}}{r} = 296.82 \text{ N}, \text{ where } r = 25 \text{ mm} \quad (15)$$

Axial Stress in Rod:

$$\sigma_{axial} = \frac{4F_{rod}}{\pi \times D^2} = 3.77 \text{ N/mm}^2 \quad (16)$$

Moment Due to Rod Force:

$$M_{hub,rxn} = F_{rod}L_{rod} = 26714.16 \text{ N-mm} \quad (17)$$

Bending Stress at Hub:

$$\sigma_{b,Hub-rxn} = \frac{32M_{hub,rxn}}{\pi D_{elbow}^3} = 272.11 \text{ N/mm}^2 \quad (18)$$

Total Bending Stress:

$$\sigma_{b,total} = \sigma_{b,elbow,self-wt} + \sigma_{b,Hub-rxn} = 272.36 \text{ N/mm}^2 \quad (19)$$

Equivalent Bending Stress

$$\sigma_{eq} = \sigma_{b,total} + \sigma_{axial} = 276.14 \text{ N/mm}^2 < 280 \text{ N/mm}^2 \quad (20)$$

Ideal Yield Strength of Mild Steel lies between 250 to 350 GPa

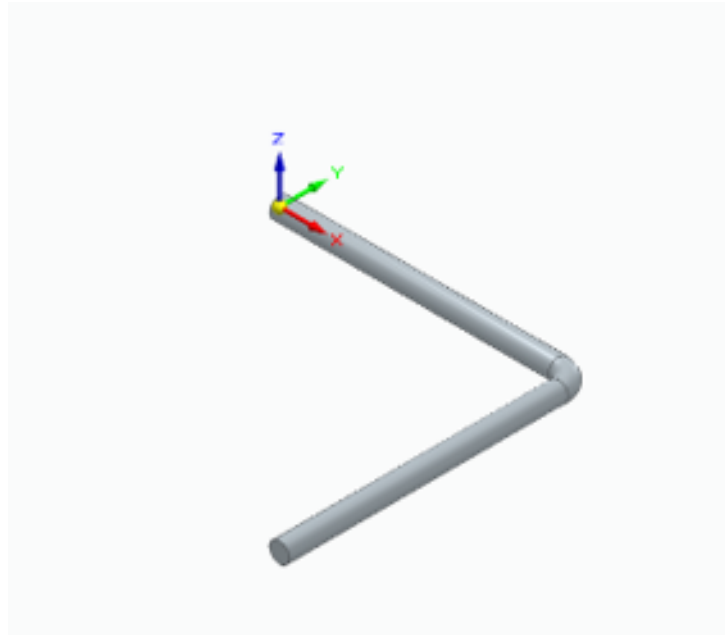


Figure 6: Elbow Rod

## 3.2 Plots

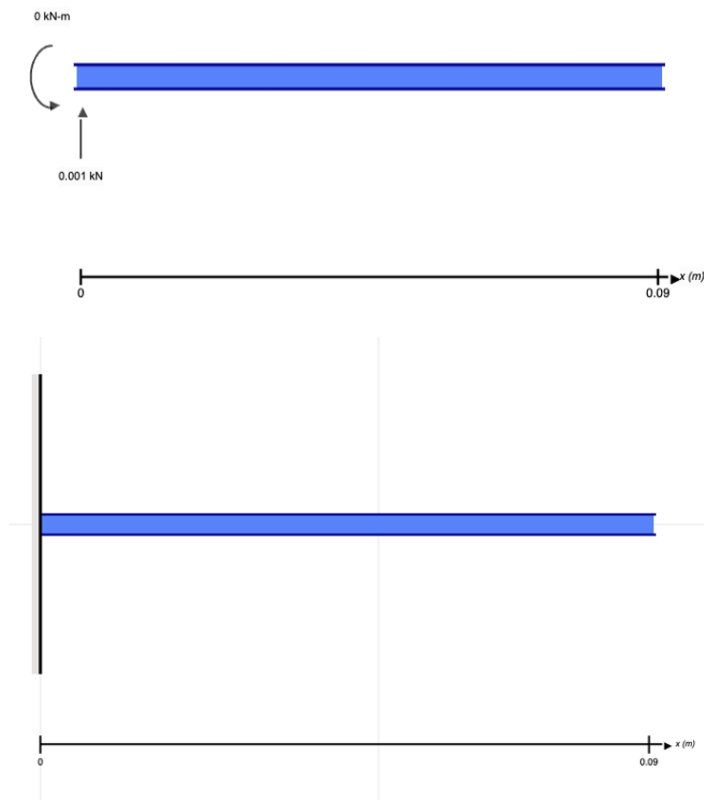


Figure 7: Free Body Diagram

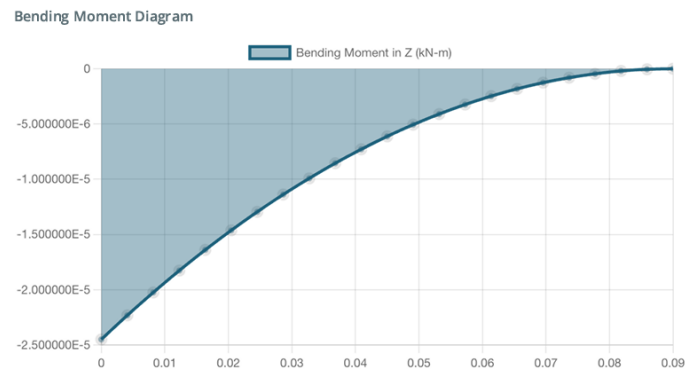


Figure 8: Bending Moment Diagram

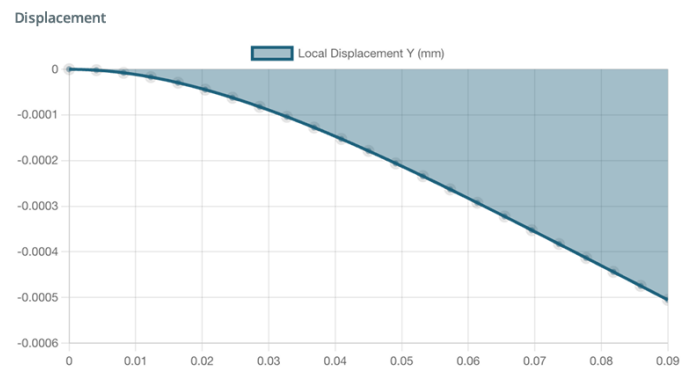


Figure 9: Displacement Diagram