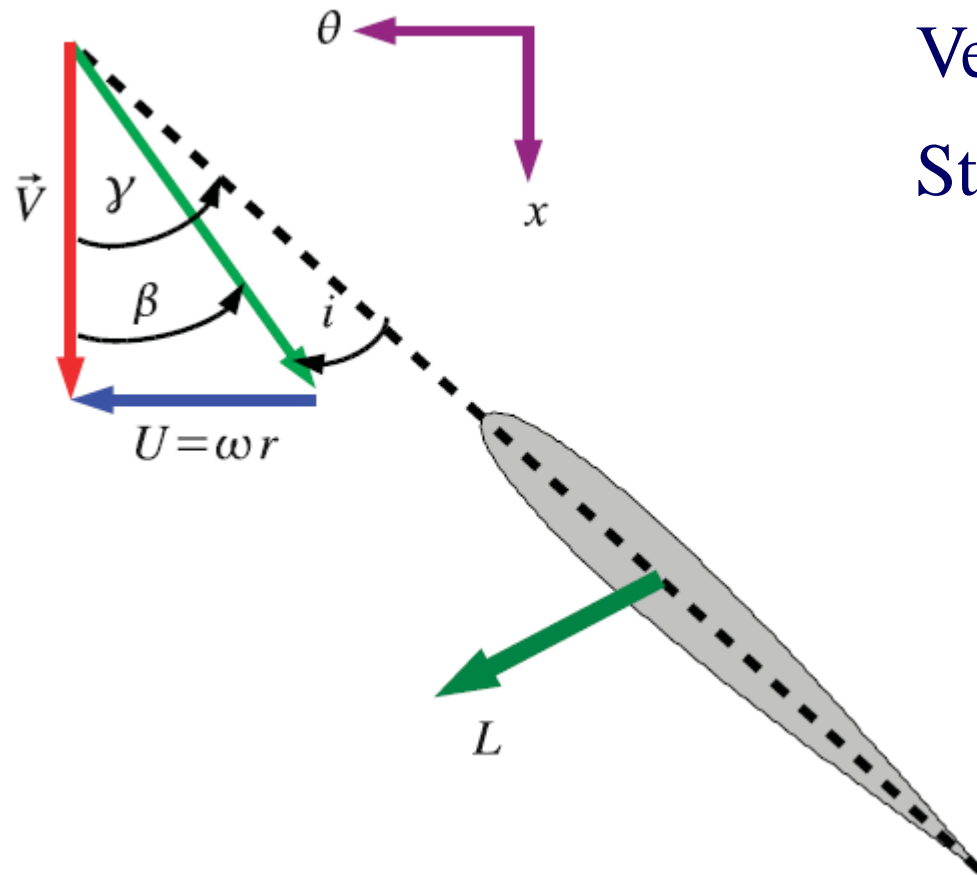




# Problem continued



Velocity triangle.

Steps to follow:

- ❖ Get  $V$ ,  $U$  and  $W$  magnitudes and directions.
- ❖ Get blade setting angle  $\gamma$  at different locations
- ❖ Find lift force and tangential force on blade per unit span at different locations.
- ❖ Use the per unit span-wise power estimates to get overall power produced.



# Lecture 6

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## Different Turbomachines and their operation

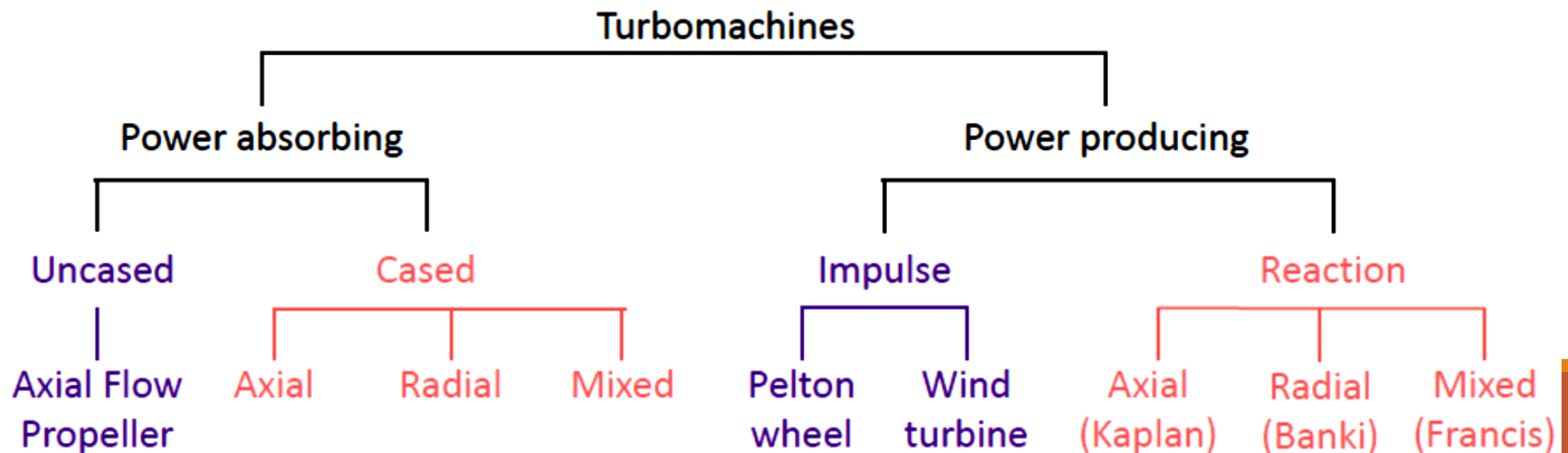
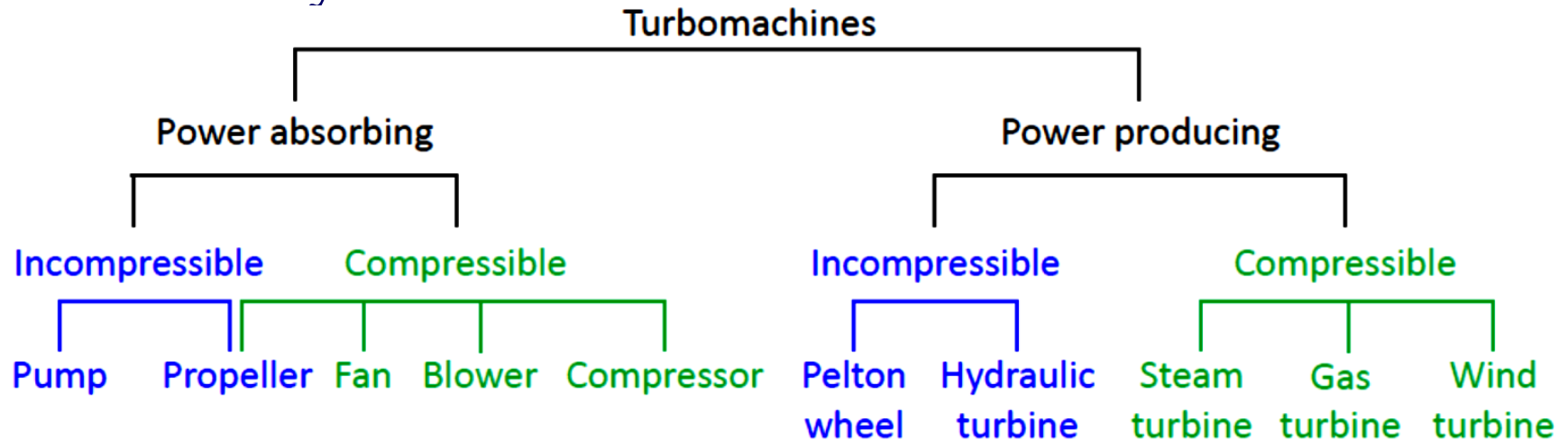
- ❖ *Radial flow turbomachines*
- ❖ *Centrifugal flow turbomachines*
- ❖ *Hydraulic turbines*



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# Classification of Turbomachinery

Two ways of classification.





Engineering and  
Architecture School  
**University**Zaragoza

# Rotating Machinery MSc

## MODULE 1

### *Principles of turbomachinery*

Author: Francisco Alcrudo

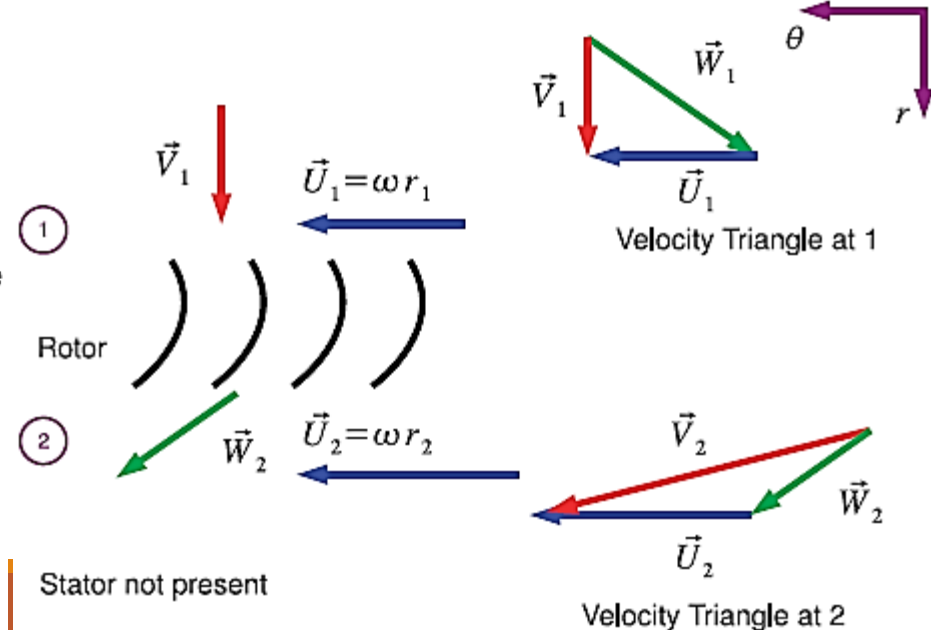
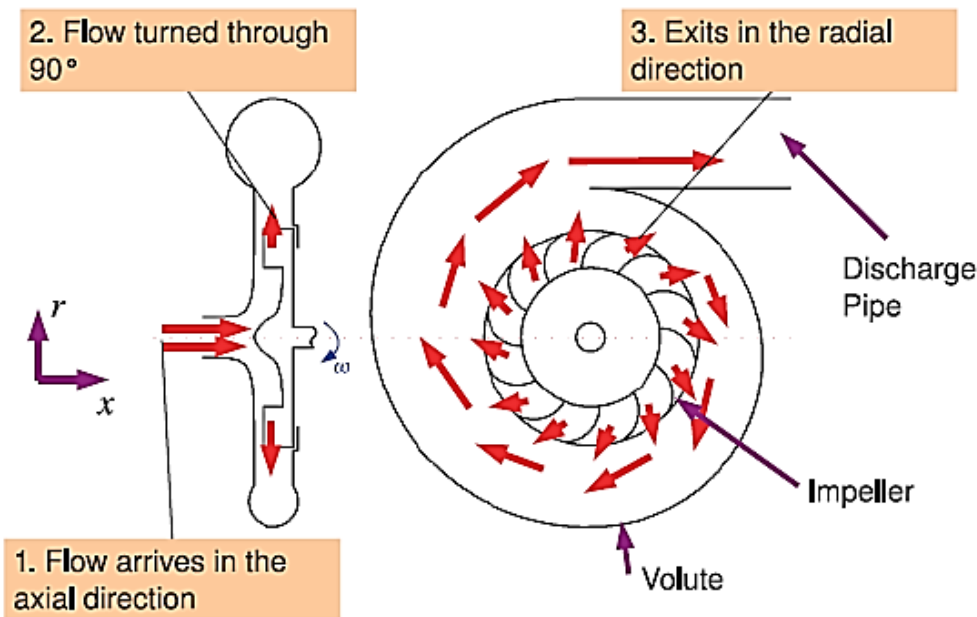
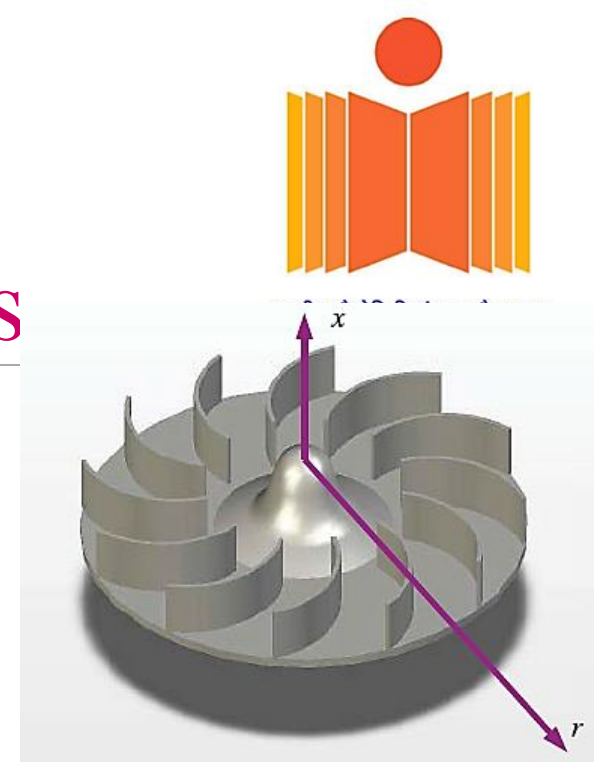




# Radial Flow Turbomachines

## Radial flow pump

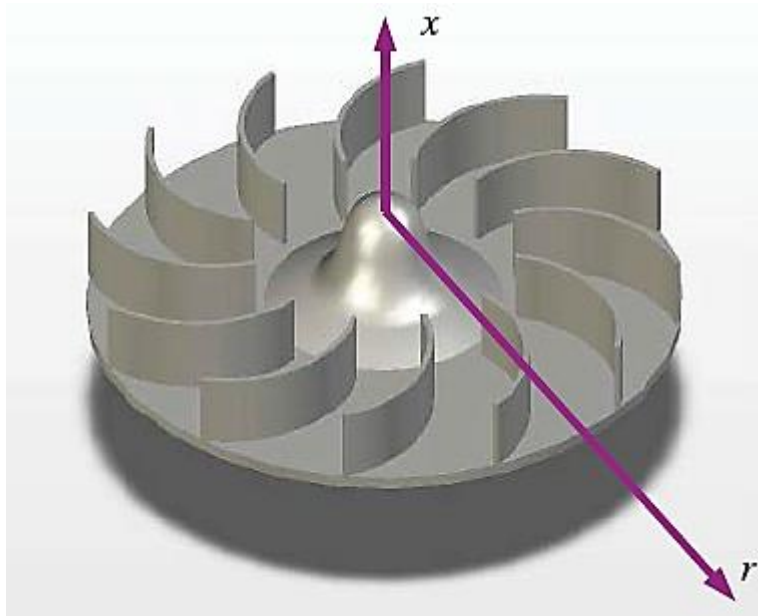
- ❖ *Volute collects the fluid and acts as a diffuser*
- ❖ *For large head built up stator between rotor and volute is employed*





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# Radial and centrifugal impeller



3D View



Plan View

Figure 4.3: Centrifugal Impeller



# Radial impellers

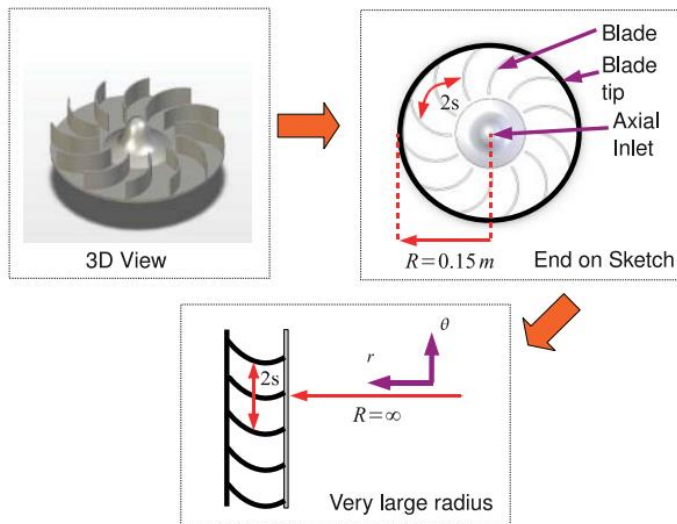
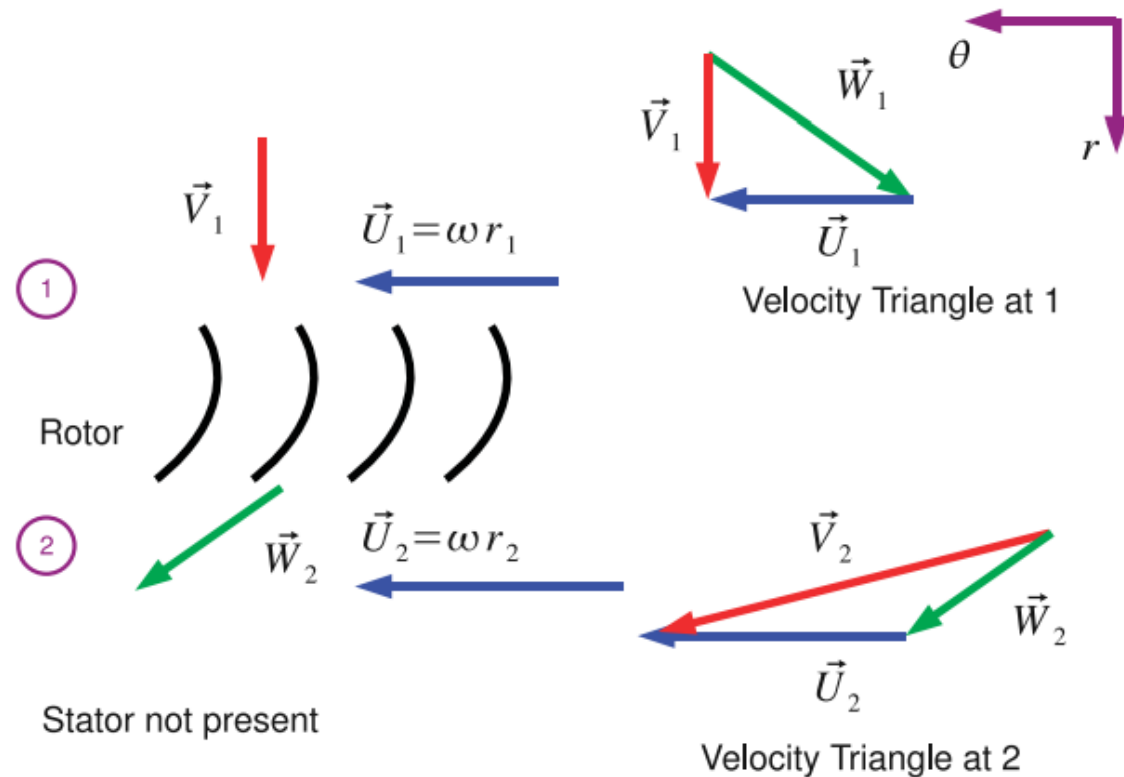
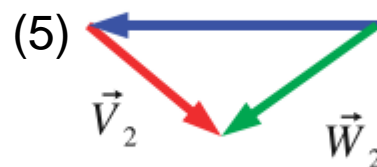
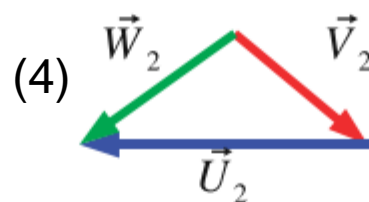
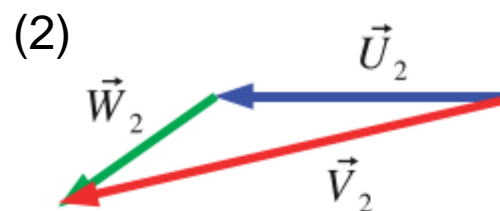
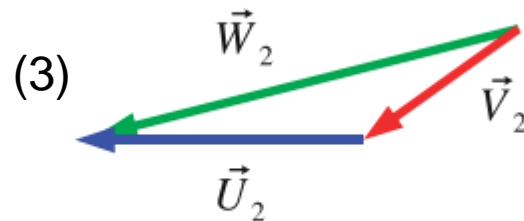
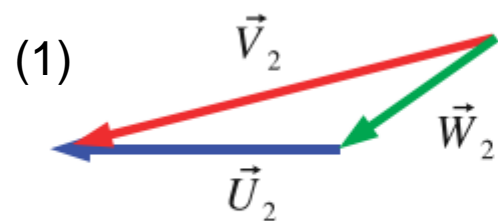


Figure 4.4: The Cascade View for a Radial Impeller





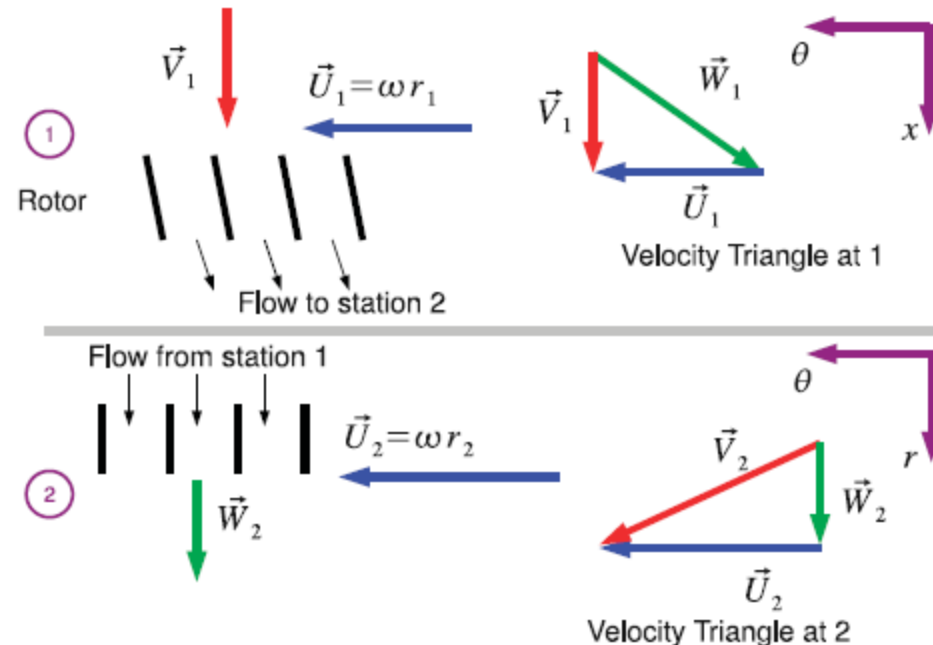
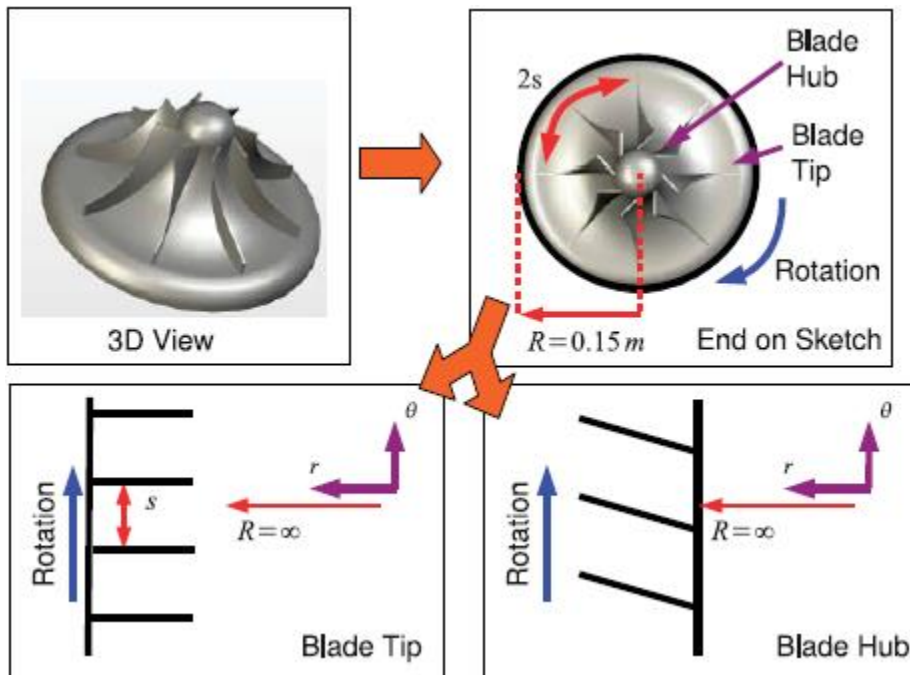




# Radial Flow Turbomachines

## Centrifugal pump

- ❖ *Note the change in impeller shape and co-ordinates at inlet and exit. (axial inlet, radial outlet).*





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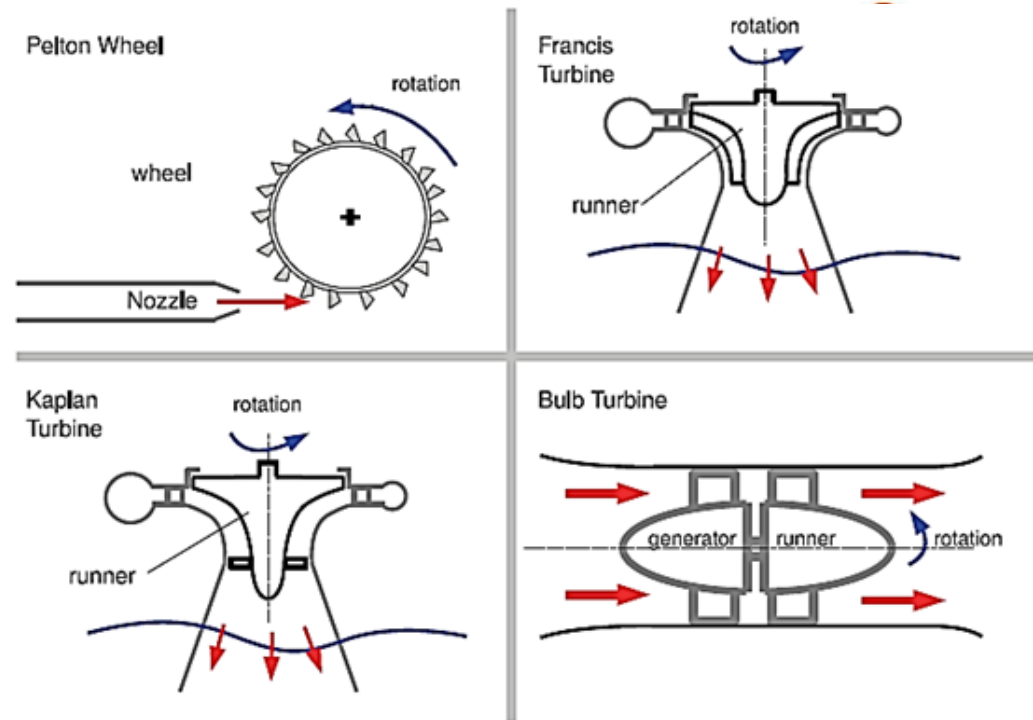
# Centrifugal Pump Working

## CENTRIFUGAL PUMPS

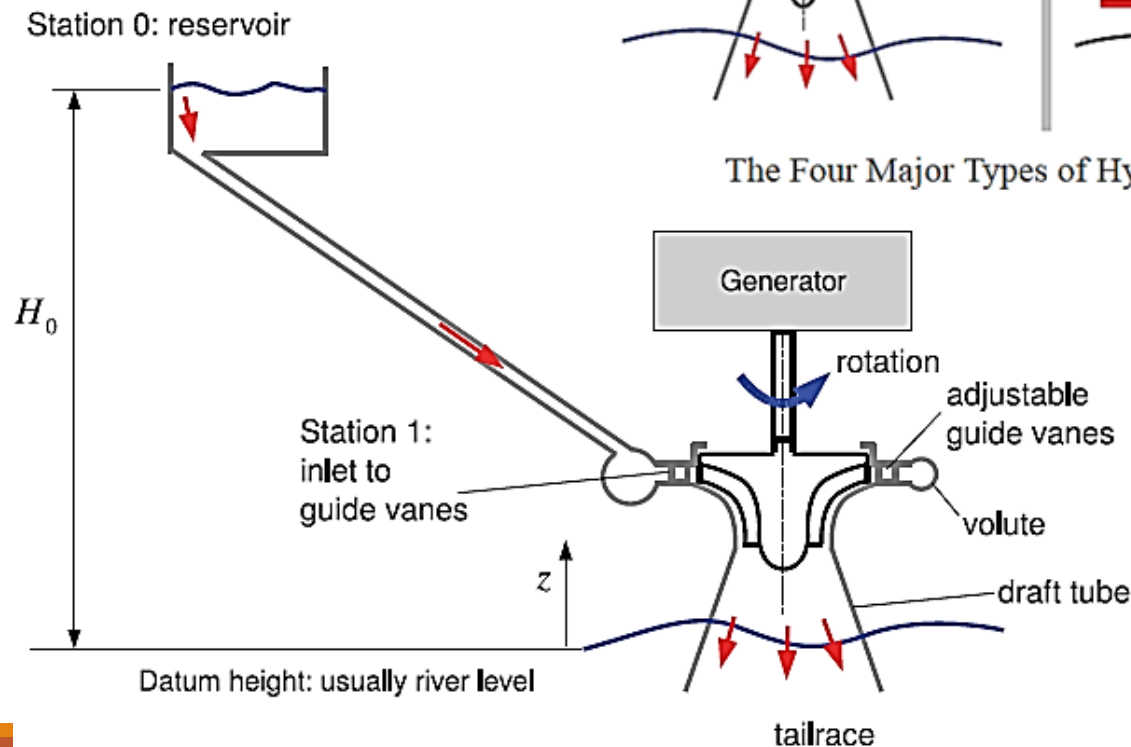


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# Hydraulic Turbines



The Four Major Types of Hydraulic Turbine



Schematic of Hydro-Electric Scheme



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# Pelton Turbine

## PELTON TURBINE

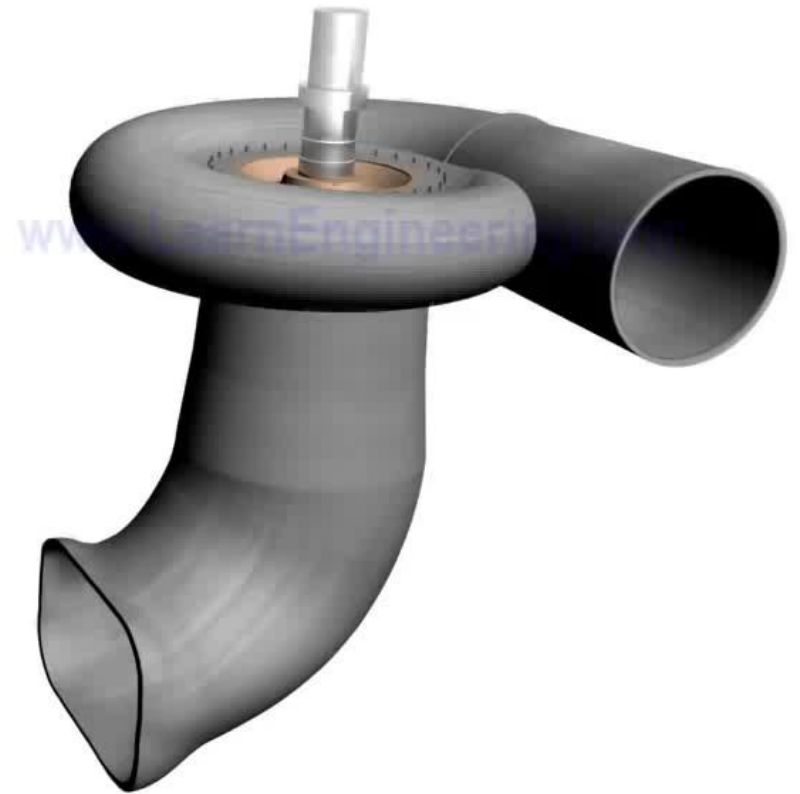


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# Francis Turbines





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# Kaplan Turbine

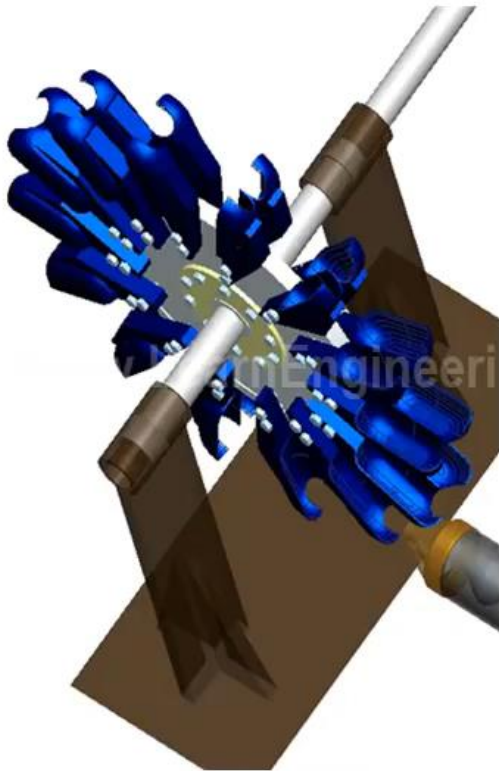
## KAPLAN TURBINE



# Comparison of Pelton, Francis & Kaplan Turbines



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# Lecture 7

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## Applications of Equations of Motion

- ❖ *Conservation of Mass*
- ❖ *Conservation of Momentum*
- ❖ *Cascade of Blades*
- ❖ *Conservation of Energy & Rothalpy*





# Applications of Equations of Motion

Conservation laws when applied to turbomachines give insight into energy interaction mechanism.

## Conservation of Mass:

❖ Applying continuity to axial flow turbomachines  
we can write

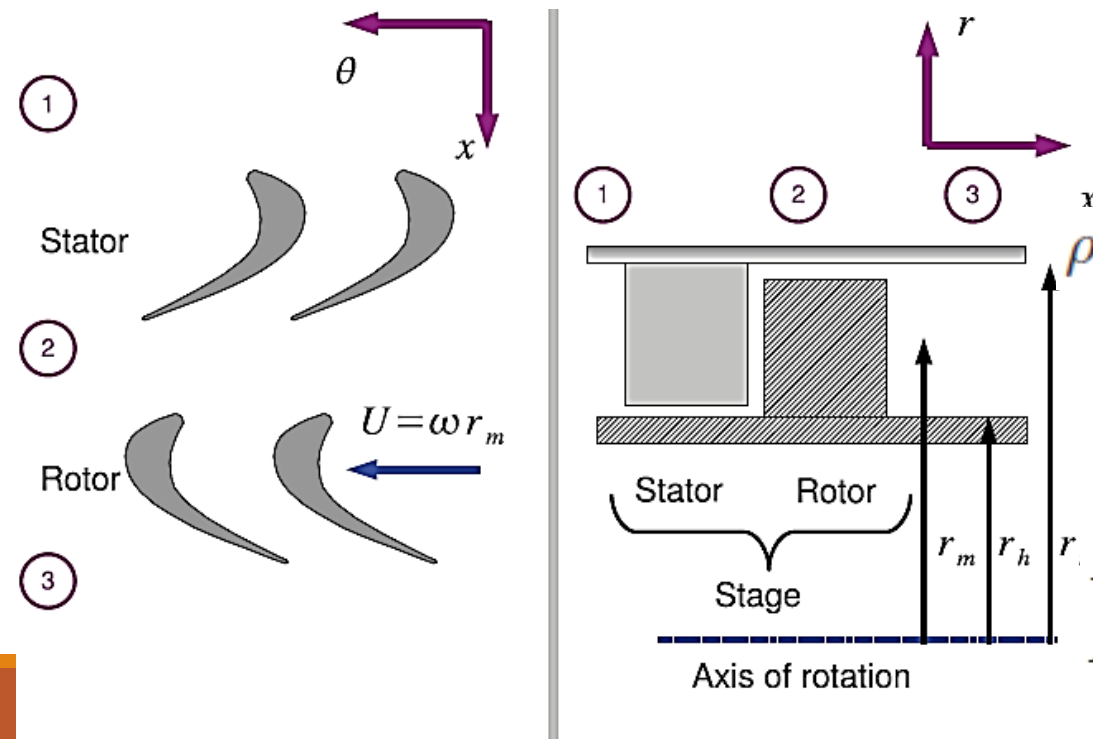
$$\rho_1 A_1 V_{1x} = \rho_2 A_2 V_{2x} = \rho_3 A_3 V_{3x}$$

➤ Note that area is perpendicular to axial velocity component & is given by

$$A = \pi(r_t^2 - r_h^2) = \pi(r_t + r_h)(r_t - r_h)$$

$$A = \pi(2r_m)(b) = 2\pi r_m b$$

$$2r_m = r_t + r_h \quad b = r_t - r_h$$





**Example** An industrial turbine operates at an 8.8:1 pressure ratio and a mass flow of  $77 \text{ kg/s}$  using air as the working fluid. The exhaust temperature is at  $43^\circ \text{C}$  and the inlet temperature to the machine is around  $1000^\circ \text{C}$ . The mean blade radius is  $0.4 \text{ m}$ . The machine is to be designed for a constant axial velocity of  $200 \text{ m/s}$ . Estimate the blade heights at entry and exit of the turbine.

At entry to the turbine:  $p_1 = 8.8 \text{ bar} = 8.8 \times 10^5 \text{ Pa}$  and  $T_1 = 1000 + 273 = 1273 \text{ K}$

At exit from the turbine:  $p_2 = 1.0 \text{ bar} = 1.0 \times 10^5 \text{ Pa}$  and  $T_2 = 473 + 273 = 730 \text{ K}$

The corresponding densities are therefore:

$$\rho_1 = \frac{p_1}{RT_1} = \frac{8.8 \times 10^5}{287 \times 1273} = 2.41 \text{ kg/m}^3$$

$$\rho_2 = \frac{p_2}{RT_2} = \frac{1.0 \times 10^5}{287 \times 730} = 0.49 \text{ kg/m}^3$$

$$\dot{m} = \rho A V_x = \rho 2\pi r_m b V_x \implies b = \frac{\dot{m}}{\rho 2\pi r_m V_x}$$

$$b_1 = \frac{77}{2.41 \times 2\pi \times 0.4 \times 200} = 0.06 \text{ m}$$

$$b_2 = \frac{77}{0.49 \times 2\pi \times 0.4 \times 200} = 0.31 \text{ m}$$

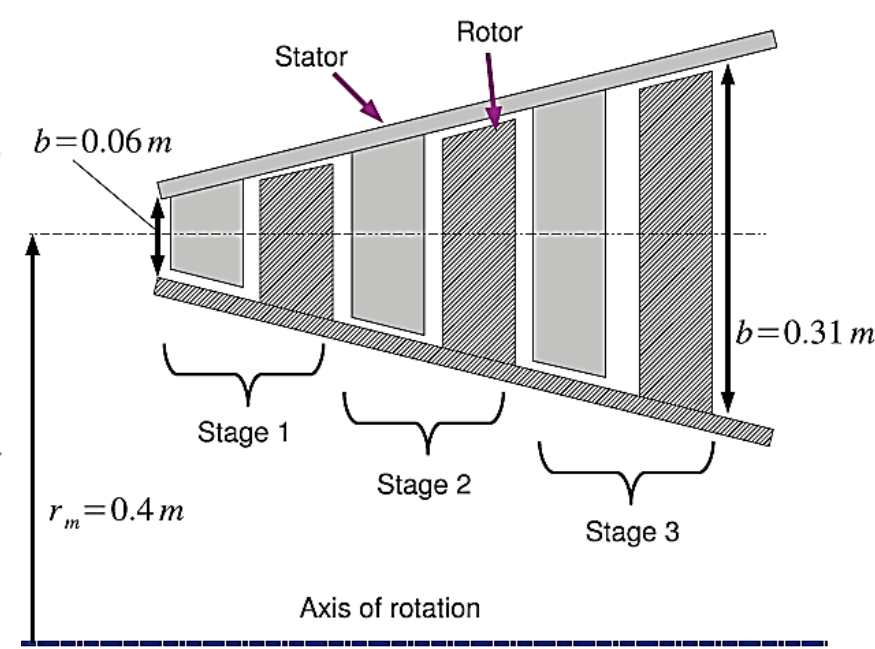


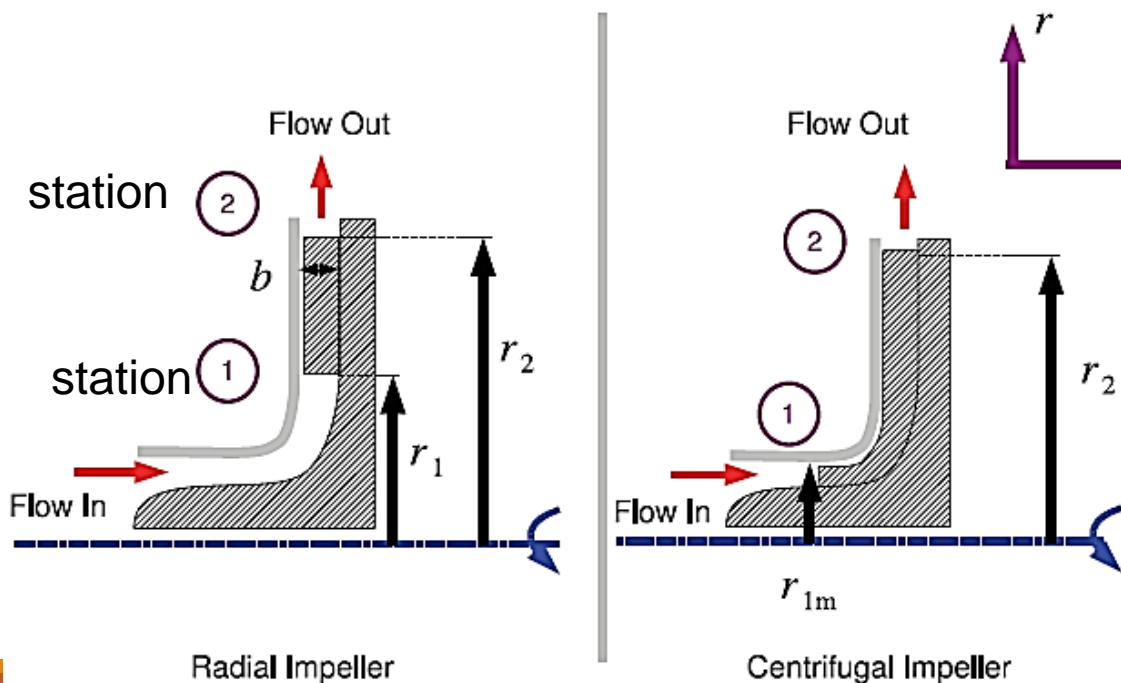
Figure 5.1: Meridional View of a Gas Turbine

# Application of Continuity to Radial Flow Turbomachines

For Radial flow turbomachines

$$\rho_1 A_1 V_{1r} = \rho_2 A_2 V_{2r} \implies \rho_1 2\pi r_1 b_1 V_{1r} = \rho_2 2\pi r_2 b_2 V_{2r}$$

For Centrifugal Flow Turbomachines



$$\dot{m}_1 = \rho_1 A_1 V_{1x} = \rho_1 2\pi r_{1m} b V_{1x}$$

$$\dot{m}_2 = \rho_2 A_2 V_{2r} = \rho_2 2\pi r_2 b V_{2r}$$

❖ *Most useful application of these equations is to find flow velocities given the mass flow rate and geometry.*

Figure 5.2: Meridional Views of Radial and Centrifugal Machines