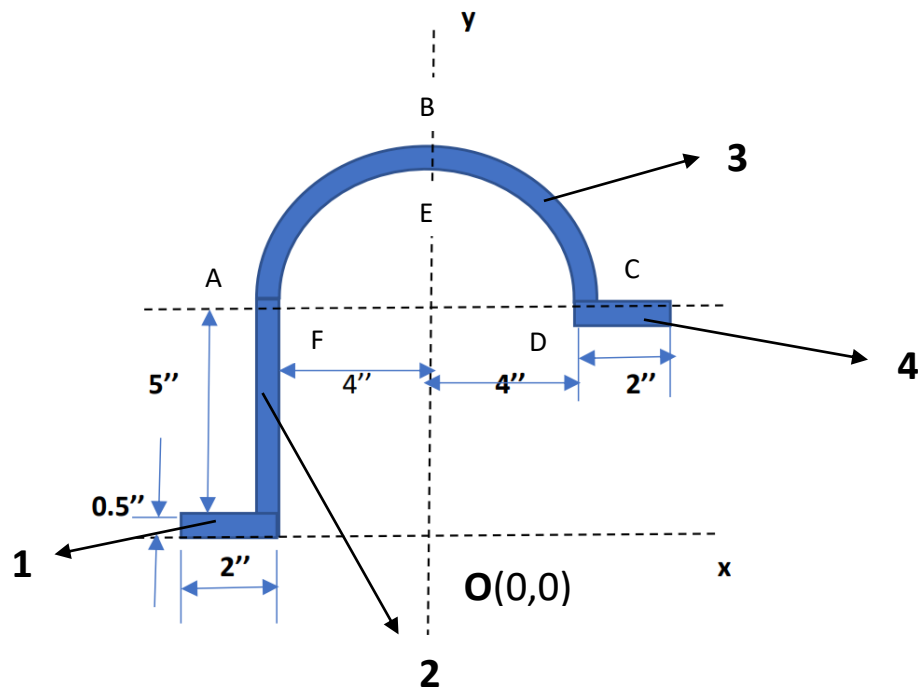


# ME1020 Homework 4

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ME21BTECH11001

## Question 1:-



Finding Centroids for respective labelled parts by considering O as origin

- 1)  $A_1 = 0.5 \times 2 = 1 \text{ inch}^2$   
 $x_1 = -5 ; y_1 = 0.25$
- 2)  $A_2 = 0.5 \times 5 = 2.5 \text{ inch}^2$   
 $x_2 = -4.25 ; y_2 = 3$
- 3)  $A_3 = \frac{\pi}{2} (4.5^2 - 4^2) = 2.125\pi \text{ inch}^2$   
 $x_3 = 0$

Quarter-circular area		$\frac{4r}{3\pi}$	$\frac{4r}{3\pi}$	$\frac{\pi r^2}{4}$
Semicircular area		0	$\frac{4r}{3\pi}$	$\frac{\pi r^2}{2}$

Centroid for symmetric disc is  $\frac{4R}{3\pi}$  ;

$\Rightarrow ABCDEF = \text{Disc}_{ABC} - \text{Disc}_{DEF}$

$y$  is  $y$  coordinate for centroid of disc

$$\Rightarrow y_3 = \frac{A_{ABC}y_{ABC} - A_{DEF}y_{DEF}}{A_{ABC} - A_{DEF}} + 5.5 = \frac{\frac{\pi}{2} (4.5)^2 \left( \frac{4}{3\pi} (4.5) \right) - \frac{\pi}{2} (4)^2 \left( \frac{4}{3\pi} (4) \right)}{\frac{\pi}{2} (4.5^2 - 4^2)} + 5.5$$

$$\Rightarrow y_3 = 8.21$$

$$4) A_4 = 0.5 \times 2 = 1 \text{ inch}^2$$

$$x_4 = 5 ; y_4 = 5.5$$

For whole figure let centroid be  $(\bar{x}, \bar{y})$

$$\Sigma A_i = A_1 + A_2 + A_3 + A_4 = 4.5 + 2.125\pi = 11.1725 \text{ inch}^2$$

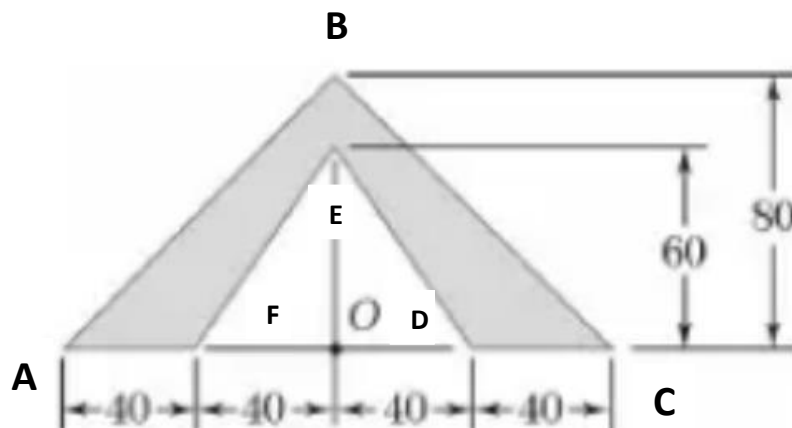
$$\Rightarrow \bar{x} = \frac{\Sigma A_i x_i}{\Sigma A_i} = \frac{-5 \times 1 - 4.25 \times 2.5 + 0 \times 2.125\pi + 5 \times 1}{11.1725} = -0.950 \text{ inch}$$

Similarly,

$$\Rightarrow \bar{y} = \frac{\Sigma A_i y_i}{\Sigma A_i} = \frac{1 \times 0.25 + 2.5 \times 3 + 8.21 \times 2.125\pi + 1 \times 5.5}{11.1725} = 6.08 \text{ inch}$$

Centroid at  $(-0.950, 6.08) \text{ inches}$

## Question 2:-



a) Given figure  $ABCDEF = \text{Triangle}_{ABC} - \text{Triangle}_{DEF}$

For ABC:- ( AC is x axis and OB is y axis )

For Triangle with base b and height h moment of inertia with centroid as origin of axes is

$$I_{xx} = \frac{bh^3}{36}$$

Using parallel axis theorem to find  $I_{xx}$  along base

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**Isosceles triangle** (Origin of axes at centroid.)

$$A = \frac{bh}{2} \quad \bar{x} = \frac{b}{2} \quad \bar{y} = \frac{h}{3}$$

$$I_x = \frac{bh^3}{36} \quad I_y = \frac{hb^3}{48} \quad I_{xy} = 0$$

$$I_p = \frac{bh}{144} (4h^2 + 3b^2) \quad I_{np} = \frac{bh^3}{12}$$

(Note: For an equilateral triangle,  $h = \sqrt{3}b/2$ )

$$\Rightarrow I_{xx} = \frac{bh^3}{36} + \frac{bh}{2} \left(\frac{h}{3}\right)^3 = \frac{1}{12}bh^3$$

$$\Rightarrow I_{xx} = \frac{1}{12}(160)(80)^3 = 6.8267 \times 10^6 \text{ mm}^4$$

About y axis  $ABC = AOB + COB = 2 \times MOI_{AOB}$

And for a right Angled triangle moment of inertia about axis ( origin of axes at vertex ) is given by

$$I_{yy} = \frac{hb^3}{12}$$

$$\Rightarrow I_{yy} = 2 \left( \frac{1}{12}(80)(80)^3 \right) = 6.8267 \times 10^6 \text{ mm}^4$$

Polar moment of inertia :-

$$\Rightarrow J_o = I_{xx} + I_{yy} = 2 \times 6.8267 \times 10^6 \text{ mm}^4 = 13.65 \times 10^6 \text{ mm}^4$$

Similarly for DEF :-

$$\Rightarrow I_{xx} = \frac{1}{12}(80)(60)^3 = 1.44 \times 10^6 \text{ mm}^4$$

Triangle DEF = DOE + FOE = 2 × MOI<sub>DOE</sub>

$$\Rightarrow I_{yy} = 2 \left( \frac{1}{12}(60)(40)^3 \right) = 0.64 \times 10^6 \text{ mm}^4$$

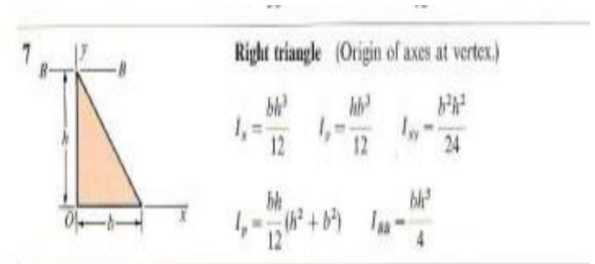
Polar moment of inertia :-

$$\Rightarrow J_o = I_{xx} + I_{yy} = (1.44 + 0.64) \times 10^6 \text{ mm}^4 = 2.08 \times 10^6 \text{ mm}^4$$

For entire ABCDEF = Triangle<sub>ABC</sub> − Triangle<sub>DEF</sub>

$$\Rightarrow (J_o)_{ABCDEF} = (J_o)_{ABC} - (J_o)_{DEF}$$

$$\Rightarrow (J_o)_{ABCDEF} = (13.65 - 2.08) \times 10^6 = 11.57 \times 10^6 \text{ mm}^4$$



**b) Centroid of ABCDEF = Triangle<sub>ABC</sub> − Triangle<sub>DEF</sub>**

For triangle centre of mass is at h/3 from base

By symmetry  $x_{ABC} = x_{DEF} = 0$

For ABC

$$A_{ABC} = \frac{1}{2}(160)(80) = 6400 \text{ mm}^2$$

$$y_{ABC} = \frac{80}{3}$$

For DEF

$$A_{DEF} = -\frac{1}{2}(80)(60) = -2400 \text{ mm}^2$$

$$y_{ABC} = \frac{60}{3} = 20$$

For whole figure let centroid be at  $(\bar{x}, \bar{y})$

$$\Sigma A_i = A_{ABC} + A_{DEF} = 6400 - 2400 = 4000 \text{ mm}^2$$

$$\Rightarrow \bar{y} = \frac{\Sigma A_i y_i}{\Sigma A_i} = \frac{6400\left(\frac{80}{3}\right) - 2400(20)}{4000} = 30.67$$

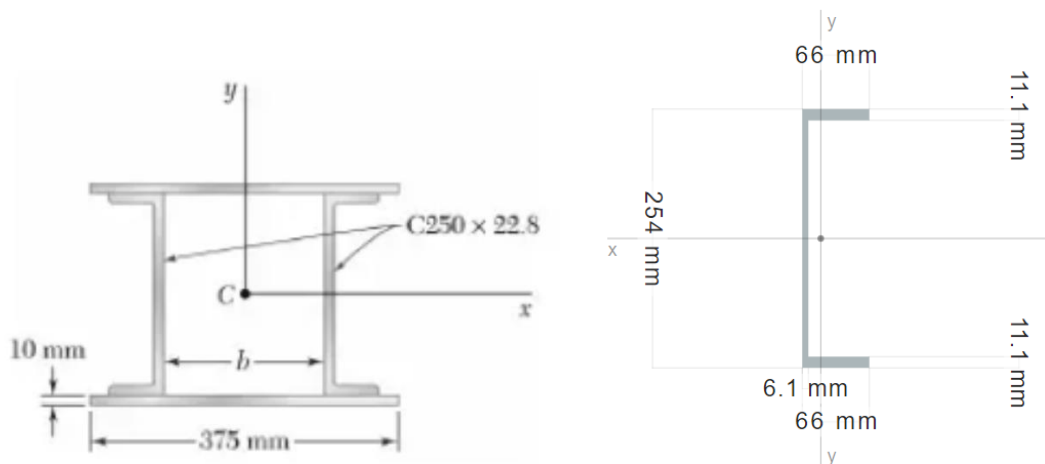
For finding MOI about centroid we use parallel axis theorem

$$\Rightarrow J_o = J_{Centroid} + A_{ABCDEF} y^2 \quad (y = \text{distance between o and centroid})$$

$$\Rightarrow J_{Centroid} = J_o - A_{ABCDEF} y^2 = 11.57 \times 10^6 - (4000)(30.67)^2$$

$$\Rightarrow J_{Centroid} = 7.807 \times 10^6 \text{ mm}^4$$

### Question 3:-



**TABLE B-4** Properties of Channel Sections: SI Units

Designation	Mass (kg/m)	Area (mm <sup>2</sup> )	Depth (mm)	Flange		Web thickness (mm)	Axis X-X			Axis Y-Y			x (mm)
				Width (mm)	Thickness (mm)		I (10 <sup>6</sup> mm <sup>4</sup> )	S = I/c (10 <sup>3</sup> mm <sup>3</sup> )	r = √I/A (mm)	I (10 <sup>6</sup> mm <sup>4</sup> )	S = I/c (10 <sup>3</sup> mm <sup>3</sup> )	r = √I/A (mm)	
C380 × 74	74.0	9480	381	94.5	16.5	18.2	168	882	133	4.58	61.8	22.0	20.3
× 60	60.0	7610	381	89.4	16.5	13.2	145	762	138	3.82	54.7	22.4	19.8
× 50.4	50.4	6450	381	86.4	16.5	10.2	131	688	143	3.36	50.6	22.9	20.0
C310 × 45	45.0	5680	305	80.5	12.7	13.0	67.4	442	109	2.13	33.6	19.4	17.1
× 37	37.0	4740	305	77.5	12.7	9.83	59.9	393	113	1.85	30.6	19.8	17.1
× 30.8	30.8	3920	305	74.7	12.7	7.16	53.7	352	117	1.61	28.2	20.2	17.7
C250 × 45	45.0	5680	254	77.0	11.1	17.1	42.9	339	86.9	1.64	27.0	17.0	16.5
× 37	37.0	4740	254	73.4	11.1	13.4	37.9	298	89.4	1.39	24.1	17.1	15.7
× 30	30.0	3790	254	69.6	11.1	9.63	32.8	259	93.0	1.17	21.5	17.5	15.4
→ C250 × 22.8	22.8	2890	254	66.0	11.1	6.10	28.0	221	98.3	0.945	18.8	18.1	16.1

a)

From ASTM data for C250 X 22.8 we obtain

$$A = 2890 \text{ mm}^2; I_{xx} = 28 \times 10^6 \text{ mm}^4; I_{yy} = 0.945 \times 10^6 \text{ mm}^4;$$

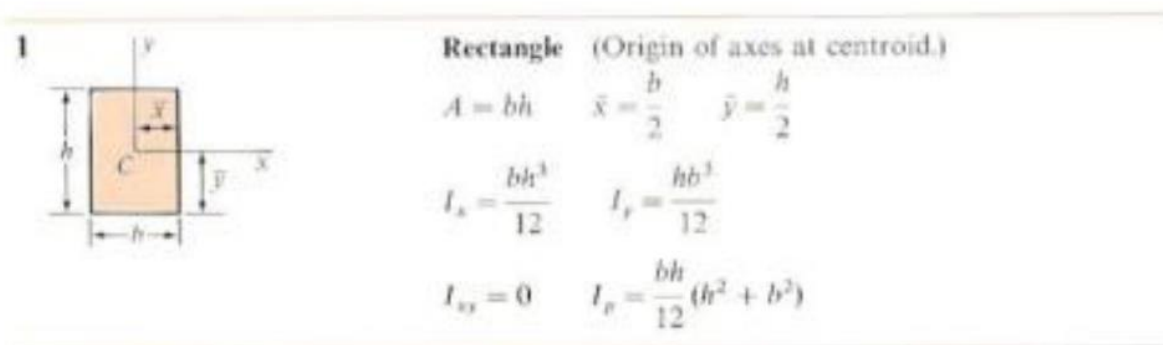
Distance between edge to centroidal axis,  $x_1 = 16.10 \text{ mm}$

$$\text{Total Area, } A_{tot} = 2(2890 + (10)(375)) \text{ mm}^2 = 13.28 \times 10^3 \text{ mm}^2$$

Distance between centre of plate to C,  $d = 127 + 5 = 132 \text{ mm}$

For a rectangular plate MOI with centre of axis about centroid is given by

$$I_{xx} = \frac{1}{12}bh^3; I_{yy} = \frac{1}{12}hb^3$$



Also base  $b = 250 \text{ mm}$  ; finding MOI about C :-

$$I_{xx} = 2(I_{xx})_{channel} + 2((I_{xx})_{plate} + A_{plate}d^2) \quad \{\text{parallel axis theorem}\}$$

$$= 2(28 \times 10^6) + 2\left(\frac{1}{12}(375)(10) + (375)(10)(132)^2\right)$$

$$I_{xx} = 186.74 \times 10^6 \text{ mm}^4$$

$$I_{yy} = 2((I_{yy})_{channel} + A_{plate}(\frac{250}{2} + x_1)^2) + 2(I_{yy})_{plate} \quad \{\text{parallel axis theorem}\}$$

$$= 2\left(0.945 \times 10^6 + (2890)\left(\frac{250}{2} + 16.10\right)^2\right) + 2\left(\frac{1}{12}(10)(375)^3\right)$$

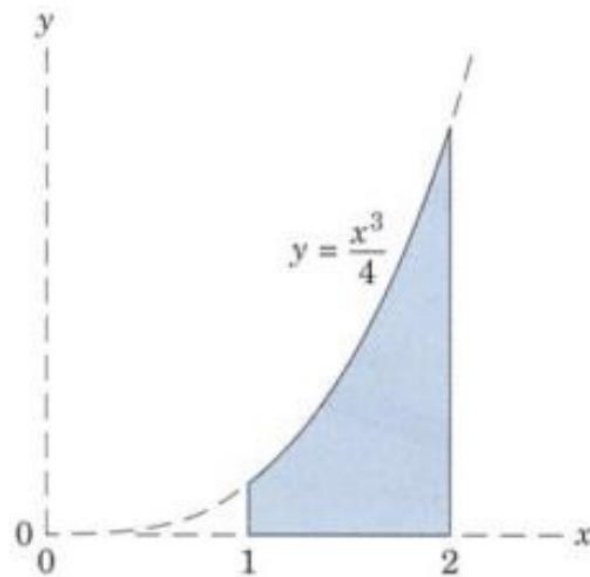
$$I_{yy} = \mathbf{204.84 \times 10^6 \text{ mm}^4}$$

**b)**

$$k_x = \sqrt{\frac{I_{xx}}{A_{tot}}} = \sqrt{14.06 \times 10^3} = \mathbf{118.6 \text{ mm}}$$

$$k_y = \sqrt{\frac{I_{yy}}{A_{tot}}} = \sqrt{15.42 \times 10^3} = \mathbf{124.19 \text{ mm}}$$

#### Question 4:-



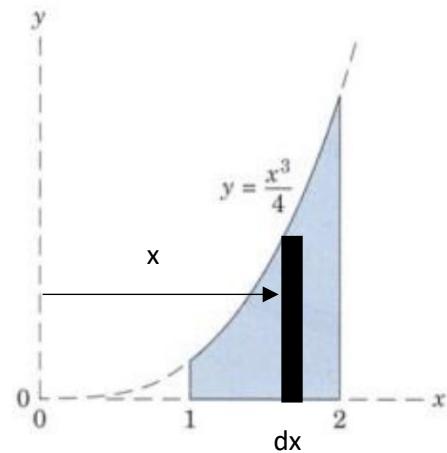
About y axis

$$dI_{yy} = x^2 dA$$

$$\Rightarrow I_{yy} = \int x^2 dA$$

$$\Rightarrow I_{yy} = \int_1^2 x^2 dA = \int_1^2 x^2 \frac{x^3}{4} dx = \int_1^2 \frac{x^5}{4} dx$$

$$\Rightarrow I_{yy} = \frac{[x^6]_1^2}{24} = \frac{2^6 - 1}{24} = \frac{63}{24} = 2.625 \text{ mm}$$



$$dA = y dx \Rightarrow dA = \frac{x^3}{4} dx$$

About x axis

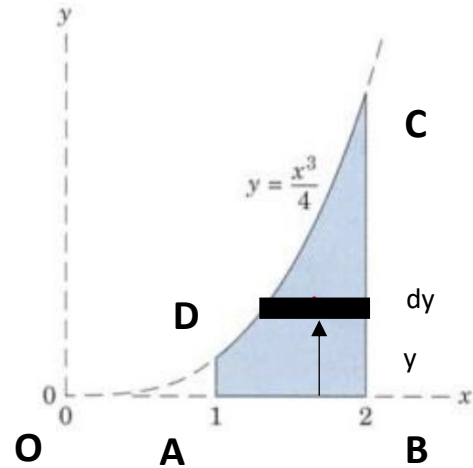
$$dI_{xx} = y^2 dA$$

For given figure

$$(I_{xx})_{ABCD} = (I_{xx})_{OBC} - (I_{xx})_{OAD}$$

For OBC  $dA_1 = xdy \Rightarrow dA = \left(2 - (4y)^{\frac{1}{3}}\right) dy$

For OAD  $dA_2 = xdy \Rightarrow dA = \left(1 - (4y)^{\frac{1}{3}}\right) dy$



$$I_{xx} = \int y^2 dA_1 - \int y^2 dA_2$$

$$\Rightarrow I_{xx} = \int_0^2 y^2 \left(2 - (4y)^{\frac{1}{3}}\right) dy - \int_0^{\frac{1}{4}} y^2 \left(1 - (4y)^{\frac{1}{3}}\right) dy$$

$$= \left[ \frac{2y^3}{3} - 4^{\frac{1}{3}} \frac{y^{\frac{10}{3}}}{\frac{10}{3}} \right]_0^2 - \left[ \frac{y^3}{3} - 4^{\frac{1}{3}} \frac{y^{\frac{10}{3}}}{\frac{10}{3}} \right]_0^{\frac{1}{4}}$$

$$= \left( \frac{16}{3} - \frac{(2^3)3}{5} \right) - \left( \frac{1}{3(2^6)} - \frac{3}{(2^6)10} \right) = \frac{2^{10}-1}{15(2^7)} = 0.533$$

$$\text{Area, } A = \int_1^2 dA = \int_1^2 y dx = \int_1^2 \frac{x^3}{4} dx = \frac{[x^4]_1^2}{16} = \frac{15}{16} = 0.9375$$

Radius of gyration:-

$$I_{yy} = k_y^2 A \Rightarrow k_y = \sqrt{\frac{I_{yy}}{A}} = \sqrt{2.8} = 1.673 \text{ mm}$$

$$I_{xx} = k_x^2 A \Rightarrow k_x = \sqrt{\frac{I_{xx}}{A}} = \sqrt{0.568} = 0.753 \text{ mm}$$

$$J_o = I_{xx} + I_{yy} = (k_x^2 + k_y^2)A = k_o^2 A$$

$$\Rightarrow k_o = \sqrt{\frac{I_{xx} + I_{yy}}{A}} = \sqrt{3.368} = 1.835 \text{ mm}$$