

THEO - JANSEN'S MECHANISM

ME2220

GROUP MEMBERS :-

ME21BTECH11001

- Abhishek Ghosh

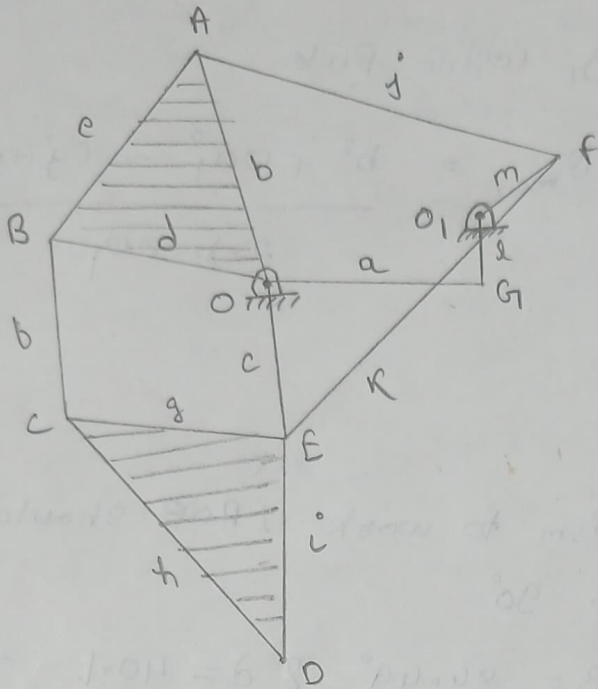
ME21BTECH11002

- Aditya Verma

ME21BTECH11028

- Loukik Kalbande

Dimensional Synthesis of Theo - Jansen Mechanism



The first step is to design of two four-bar subcomponents and the second is the design of a parallel four-bar mechanism.

for the first 4-bar-mech O_1FA is a crank rocker. (m is crank, b is rocker)

fig: Theo - Jansen model

i) O_1FA synthesis

Taking $a = 38$ and $d = 7.8$

$$O_1O_1 = \sqrt{a^2 + d^2} = 38.79$$

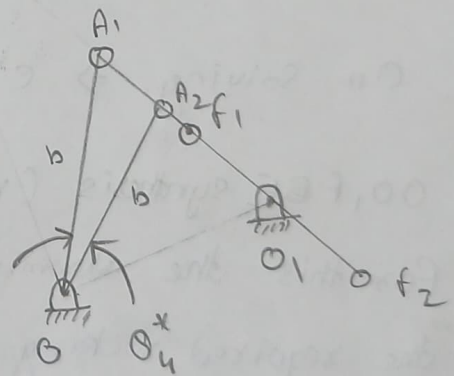
The required rocking angle

$$\theta_4^* = 42.37^\circ \quad (\text{Constraint})$$

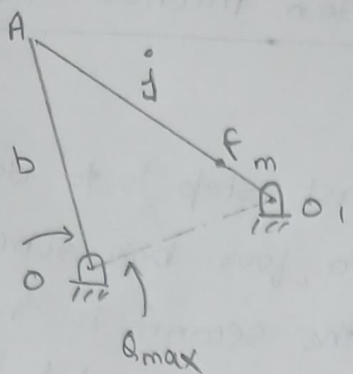
Also, $\frac{m}{b} = \sin\left(\frac{\theta_4}{2}\right)$

$$\frac{m}{b} = 0.361$$

Taking $m = 15 \Rightarrow b = \frac{15}{0.361}$
 $= 41.5$



$$A_1A_2 = 2m$$



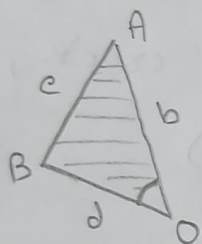
Assuming the max angle (θ_{max}) b/w rocker and OO_1 , to be 108.06°

Using, Cosine Rule

$$\cos \theta_{max} = \frac{b^2 + OO_1^2 - (y' + m)^2}{2 \cdot b \cdot OO_1}$$

On solving, $\Rightarrow y' = 50$

ii) ΔOAB



for the mechanism to work $\angle AOB$ should be b/w 0° to 90°

Assuming $\angle AOB = 84.49^\circ$ & $d = 40.1$

Using cosine Rule

$$\cos 84.49^\circ = \frac{b^2 + d^2 - c^2}{2 \cdot b \cdot d}$$

On Solving $\Rightarrow c = 55.8$

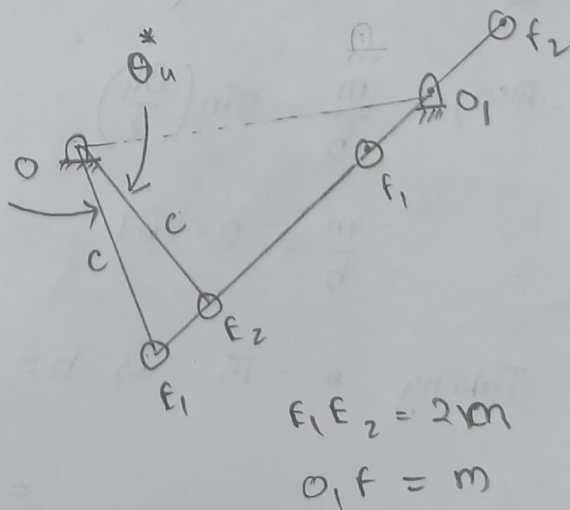
iii) OO_1FBE synthesis (m is crank, c is rocker)

For this the assuming the required rocking angle

$\theta_4^* = 44.87^\circ$ (Constraint)

from ΔOE_1E_2

$$\frac{m}{c} = \sin \left(\frac{\theta_4^*}{2} \right)$$



$$E_1E_2 = 2m$$

$$O_1F = m$$

$$\frac{m}{c} = 0.381$$

$$\Rightarrow c = \frac{m}{0.381} = \frac{15}{0.381}$$

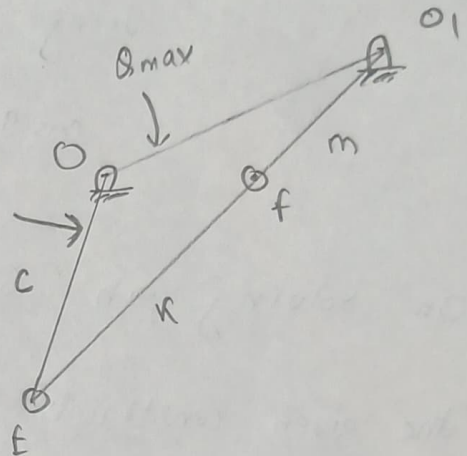
$$\Rightarrow c = 39.3$$

Assuming max angle θ_{max} b/w rocker & OO_1 to be 160°

Using cosine Rule

$$\cos 160^\circ = \frac{c^2 + OO_1^2 - (K+m)^2}{2 \cdot c \cdot OO_1}$$

On Solving $\Rightarrow K = 61.9$



iv) OBCE synthesis

Since this is a parallel four bar mech with $OB \parallel CE$

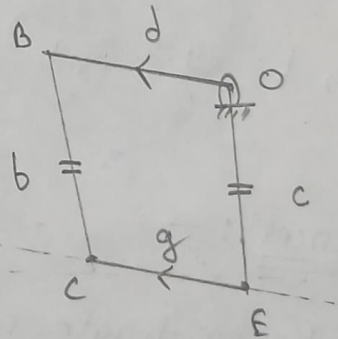
$$\& BC = OE$$

$$b = c = 39.3$$

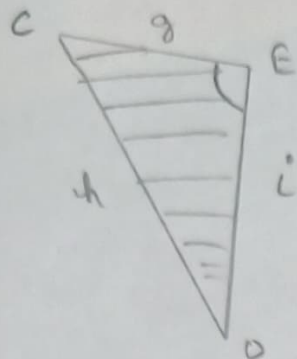
With OE known, draw $CE \parallel BO$

now construct BC of length 39.3

This gives length of $CE = g = 36.7$.



V) $\triangle ECD$



for given mechanism $\angle CED$ should range from 0° to 180°

Assuming $\angle CED = 99.1^\circ$ & $i = 49$

Using cosine Rule

$$\cos 99.1^\circ = \frac{g^2 + i^2 - h^2}{2 \cdot g \cdot i}$$

On Solving $\Rightarrow h = 65.7$

With the given constraints of Rocking angle and assuming

max angle $(\angle AOD_1 = 108.06^\circ$ & $\angle CED = 99.1^\circ$ & lengths

$a = 38$, $d = 7.8$, $d = 40.1$, $i = 49$ the following

are found out to be

$b = 41.5$, $j = 50$, $e = 55.8$, $e = 39.3$, $k = 61.9$, $g = 36.7$

, $b = 39.3$, $h = 65.7$

To prove Trajectory of point T only depends on input angle :—

Theory :—

Theo - Jansen schematic diagram is shown below. It consists of two stiff triangles AON, FBT and rods KA = a, KB = z, NF = g, OB = d, O₁K = r.

Hinges \rightarrow A, O, N, B, F, K, O₁

with O and O₁ being fixed firmly.

The mechanism is set to motion by moving the rod O₁K (r). The vertex T of triangle FBT follows a certain trajectory.

Let AN = w, OA = v, ON = u, FB = b, BT = p, FT = s, $\angle AON = \alpha$, $\angle FBT = \beta$ & $OO_1 = d$.

In order for T to follow optimal trajectory, we need to find numerical values of the parameters \rightarrow a, z, d, g, v, u, w, b, p, s, r.

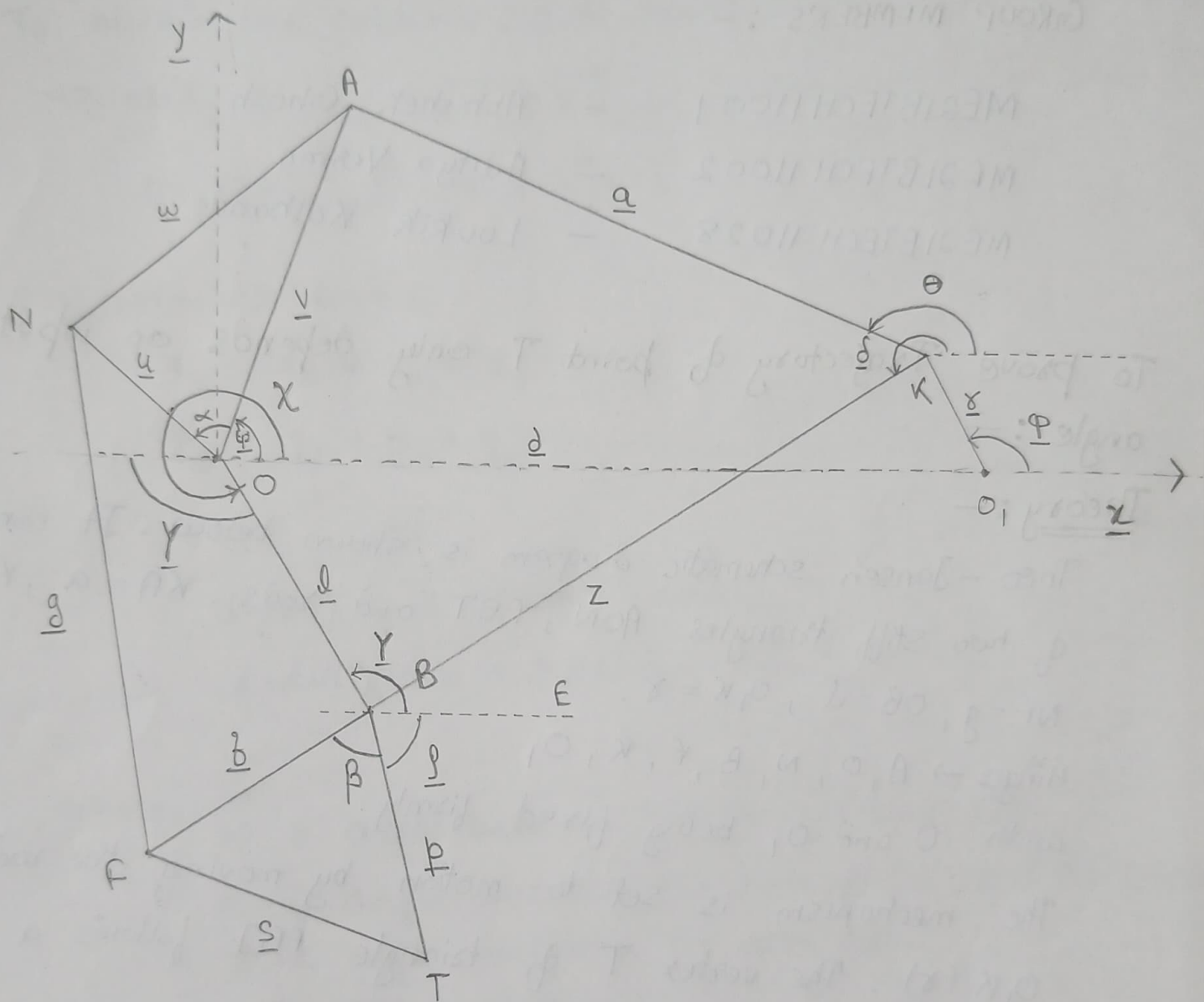


Fig: Schematic Diagram of Theo Jansen Mechanism

The coordinates of point T =:

$$x_T = x_B + p \cos \phi$$

$$y_T = y_B + p \sin \phi$$

—(1) where x_B, y_B are the coordinates of point B.

ϕ = angle between BE and BT.

To obtain the coordinates of point B, we used the vector equality, $\vec{OB} = \vec{OK} + \vec{KB}$ —(2) (where, $\vec{OK} = \vec{OO_1} + \vec{O_1K}$)

$$\text{or say, } \vec{l} = (\vec{d} + \vec{r}) + \vec{z} \quad \text{—(2)}$$

In coordinate form:

$$x_B = x_d + x_r + x_z$$

$$y_B = y_d + y_r + y_z \quad \text{—(4)}$$

or

$$l \cdot \cos \chi = d + r \cos \phi + z \cos \delta$$

$$l \cdot \sin \chi = 0 + r \sin \phi + z \sin \delta$$

where,

ϕ = angle between horizontal and O_1K .

δ = angle between the horizontal and KB (figure)

χ = angle between Ox and OB .

from eq —(4), it follows

$$z \cdot \sin \delta = l \sin \chi - r \sin \phi$$

$$z \cdot \cos \delta = l \cos \chi - r \cos \phi - d$$

—(5)

by ratio;

$$(z \sin \delta)^2 + (z \cos \delta)^2 = z^2 \quad \text{—(6)}$$

from eqⁿ - (6) we found -:

$$A \sin \chi + B \cos \chi = C \quad - (7)$$

$$A = 2\ell r \sin \phi$$

$$B = 2\ell r \cos \phi + 2d\ell$$

$$C = \ell^2 + r^2 + d^2 + 2rd \cos \phi - z^2$$

- (8)

$$\text{let, } \sin \chi = \frac{2t}{1+t^2}, \quad \cos \chi = \frac{1-t^2}{1+t^2} \quad - (9)$$

substituting the values of $\sin \chi$ and $\cos \chi$ in eqⁿ - (7)

$$(B+C)t^2 - 2At + (C-B) = 0 \quad - (10)$$

the roots of above equation will be real and at a positive discriminant.

$$\text{ie. } A^2 + B^2 - C^2 > 0$$

roots of equation - (10) \Rightarrow

$$t_{1,2} = \frac{2A \pm \sqrt{4A^2 - 4(B+C)(C-B)}}{2(B+C)} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B+C} \quad - (12)$$

$$\chi_{1,2} = 2 \tan^{-1}(t)$$

- (13)

Since x_2 depends on angle ψ . So,

The coordinates of point B :

$$x_B = 1 \cdot \cos(x_2(\psi)) \quad - (14)$$

$$y_B = 1 \sin(x_2(\psi))$$

Similarly, as we did for coordinates of point B. The coordinate of point A are -:

$$\vec{OA} = \vec{OK} + \vec{KA} \quad - (15)$$

$$\vec{v} = \vec{d} + \vec{r} + \vec{a} \quad - (16)$$

in coordinate form,

$$x_A = x_d + x_r + x_a$$

$$y_A = y_d + y_r + y_a$$

$$\begin{aligned} v \cos \psi &= d + r \cos \phi + a \cos \theta \\ \text{or } v \sin \psi &= 0 + r \sin \phi + a \sin \theta \end{aligned}$$

$$\begin{aligned} (a \sin \theta &= v \sin \psi - r \sin \phi \\ a \cos \theta &= v \cos \psi - d - r \cos \phi) \quad - (17) \end{aligned}$$

from ratio :

$$(v \sin \psi - r \sin \phi)^2 + (v \cos \psi - (r \cos \phi + d))^2 = a^2 \quad - (18)$$

$$A_1 \sin \psi + B_1 \cos \psi = C_1 \quad - (19)$$

where

$$A_1 = 2vr \sin \phi$$

$$B_1 = 2vr \cos \phi + 2dv$$

$$C_1 = v^2 + r^2 + d^2 + 2rd \cos \phi - a^2$$

- (20)

$$\text{let say, } \sin \psi = \frac{2\tau}{1+\tau^2}, \quad \cos \psi = \frac{1-\tau^2}{1+\tau^2} \quad - (21)$$

Substituting the value of $\sin\varphi$ and $\cos\varphi$ in equation- (19)

$$(B_1 + C_1)T^2 - 2A_1T + (C_1 - B_1) = 0 \quad \text{--- (23)}$$

roots of above quadratic equation \Rightarrow

$$T_{1,2} = \frac{2A_1 \pm \sqrt{4A_1^2 - 4(B_1 + C_1)(B_1 - B_1)}}{2(B_1 + C_1)}$$

$$= \frac{A_1 \pm \sqrt{A_1^2 + B_1^2 - C_1^2}}{B_1 + C_1} \quad \text{--- (24)}$$

Hence,

$$\Psi_{1,2} = 2 \tan^{-1}(T_{1,2}) \quad \text{--- (25)}$$

Since Ψ_1 depends on φ

Hence, the coordinates of point A -;

$$x_A = v \cos(\Psi_1(\varphi))$$

--- (26)

$$y_B = v \sin(\Psi_1(\varphi))$$

\Rightarrow Movement of Point T only depends on initial input angle.

Calculation of Degree of freedom :-

$$n = 8$$

$$j_1 = 4$$

$$j_2 = 3$$

$$j_3 = 0$$

$$j = j_1 + 2j_2 + j_3$$

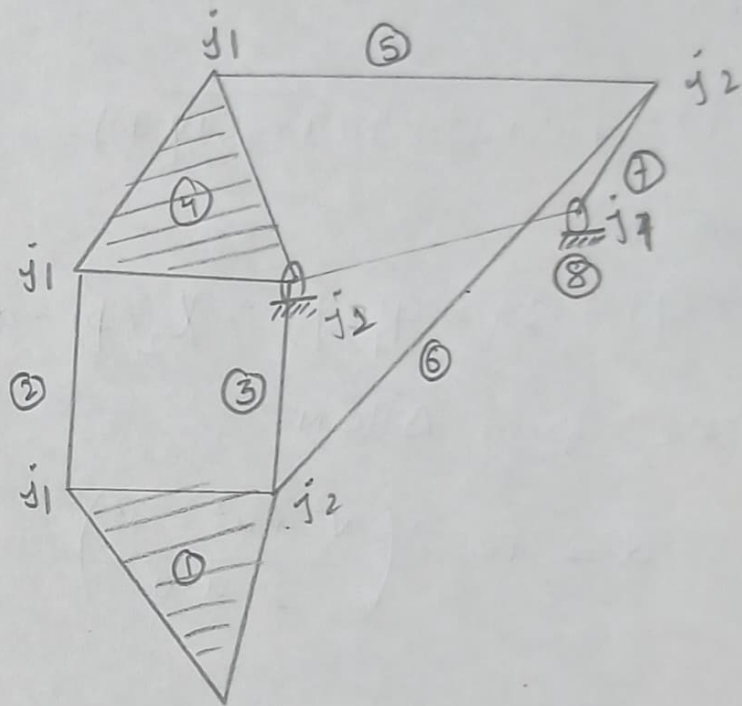
$$= 4 + 6 = 10$$

$$b = 3(n-1) - 2j$$

$$= 3(8-1) - 2(10)$$

$$= 21 - 20$$

$$= \underline{\underline{1}}$$



References :-

- 1) Researchgate.net
- 2) sciencedirect.com

Note :- All assumptions have been made with help of Geo gebra model.

LINK: <https://www.geogebra.org/classic/wvxbctek>