



**KEEPING MOBILE PHONE/SMART WATCH, EVEN IN 'OFF' POSITION, IS EXAM MALPRACTICE**

**General Instructions:**

- Use of Steam tables, Refrigeration tables, Psychrometric chart and Heat & Mass Transfer data book are permitted
- Missing data if any, may be reasonably assumed

**Answer any TEN Questions  
(10 X 10 = 100 Marks)**



- A three-stage reciprocating air-compressor delivers  $5.2 \text{ m}^3$  of free air per minute at the rated inlet condition of 1.03 bar and  $20^\circ\text{C}$ . The suction pressure and temperature are 1 bar and  $30^\circ\text{C}$  respectively. The air is cooled to  $30^\circ\text{C}$  after each stage of compression. The delivery pressure of the compressor is 150 bar when it is rotating at 300 rpm. Assume the clearance volume of all three cylinders is 5% of respective stroke volumes. If the index of compression and expansion in all the three stages is 1.35, find the power required to run the compressor. [5]
    - The capacity of a saturation vapour compression refrigeration cycle with R-12 as refrigerant is 10 tonnes. [5]  
This refrigeration unit is operating between  $-6^\circ\text{C}$  and  $36^\circ\text{C}$ . For this unit, find the mass flow rate of refrigerant. If the compressor employed in this used is single cylinder and single-acting which is running at 300 rpm, then find its cylinder diameter. Assume the ratio between stroke length and cylinder diameter as 1.2 and volumetric efficiency as 0.9.
  - An office is to be air-conditioned for 50 members when the outdoor conditions are  $30^\circ\text{C}$  dry bulb temperature and 75 % relative humidity. If the quantity of air supplied is  $0.4 \text{ m}^3/\text{min}/\text{person}$ , find the following:
    - Capacity of cooling coil in tonnes of refrigeration
    - Capacity of heating coil in kW
    - Amount of water vapour removed per hourAssume that required air inlet conditions are  $20^\circ\text{C}$  dry bulb temperature and 60 % relative humidity. Air is conditioned first by cooling and dehumidifying and then by heating
    - If the heating coil surface temperature is  $20^\circ\text{C}$ , find the by-pass factor of the heating coil.
- Determine the thermal conductivity of asbestos powder packed in between two concentric copper pipes of 25 mm inner diameter and 36 mm outer diameter. The inner pipe has a heating coil which has a rated power of 120 W/m. Assume negligible thermal contact resistance between the heating coil and inner surface of the pipe. The average temperature of inner and outer pipes are measured to be  $42.4^\circ\text{C}$  and  $27.9^\circ\text{C}$  respectively [5]
  - A copper wire of 40 mm diameter carries a current of 250 A and has an electrical resistance of  $0.25 \times 10^{-4} \Omega/\text{cm}$  per unit length. The surface temperature of copper wire is  $250^\circ\text{C}$  and the ambient air temperature is  $10^\circ\text{C}$ . If the thermal conductivity of the copper wire is  $175 \text{ W/m-K}$ , calculate maximum temperature and heat transfer co-efficient between wire surface and ambient air [5]
- Alloy steel ball of 12 mm diameter heated to  $800^\circ\text{C}$  is quenched in a bath at  $100^\circ\text{C}$ . Consider the following material properties for the ball:
  - Thermal conductivity =  $57 \text{ W/m-K}$
  - Density =  $7860 \text{ kg/m}^3$
  - Specific heat capacity =  $450 \text{ J/kg-K}$The convective heat transfer coefficient between the fluid in the quenching bath and steel ball is  $41.7 \text{ W/m}^2\text{-K}$ . For this heat treatment process, determine (i) temperature of the ball after 10 seconds (ii) Time taken for the steel ball to cool to  $400^\circ\text{C}$



5. Consider the cylinder wall of an IC engine as a plane wall. The initial temperature of the wall before combustion process is  $25^{\circ}\text{C}$ . It is given that the temperature of the combustion mixture is  $800^{\circ}\text{C}$ . Assume the convective heat transfer coefficient between the combustion mixture and wall to be  $20 \text{ W/m}^2\text{-K}$ . For design consideration, the maximum allowable temperature for the wall is  $400^{\circ}\text{C}$ . To meet the design criteria, determine the permissible time of exposure to the combustion mixture. Consider the following thermo-physical properties for cylinder wall
- Thermal conductivity =  $57 \text{ W/m-K}$
  - Thermal diffusivity =  $1.66 \times 10^{-5} \text{ m}^2/\text{s}$
6. Air flows over a flat plate at a velocity of  $3 \text{ m/s}$  and at temperature  $15^{\circ}\text{C}$ . The plate is maintained at  $85^{\circ}\text{C}$ . If the length of the plate is  $100 \text{ cm}$  long along the flow of air, find the heat lost by the plate per unit width.
7. A vertical plate of  $0.75 \text{ m}$  height is at  $170^{\circ}\text{C}$  and is exposed to air at a temperature of  $105^{\circ}\text{C}$  and  $1 \text{ bar}$  atmospheric pressure. Calculate mean heat transfer coefficient and rate of heat transfer per unit width of the plate
8. In a double pipe heat exchanger, hot fluid with a specific heat of  $2300 \text{ J/kg-K}$  at  $380^{\circ}\text{C}$  and leaves at  $300^{\circ}\text{C}$ . Cold fluid enters at  $25^{\circ}\text{C}$  and leaves at  $210^{\circ}\text{C}$ . Calculate the heat exchanger area required for counter flow and what would be the percentage increase in area if the fluid flows were parallel. Take overall heat transfer coefficient to be  $750 \text{ W/m}^2\text{-K}$  and mass flow rate hot water to be  $1 \text{ kg/s}$ .
9. Derive the expression for pressure ratio for maximum discharge from steam nozzle, starting from steady flow energy equation. Also, find the numerical value of critical pressure ratio for dry saturated and superheated steam from the derived expression.
10. Steam at a pressure of  $10.5 \text{ bar}$  and  $0.95$  dry is expanded through a convergent divergent nozzle. The pressure of the steam leaving the nozzle is  $0.85 \text{ bar}$ . Find the velocity of steam at throat for maximum discharge. Take index of expansion,  $n = 1.135$ . Also, find the throat area at the exit and the steam discharge if the throat area is  $1.2 \text{ cm}^2$ . Assume the flow is isentropic and there are no friction losses.
11. Ten thin brass fins of  $1 \text{ mm}$  thick,  $5 \text{ cm}$  wide and length  $7 \text{ cm}$  are placed axially on an engine cylinder at equidistance of  $1 \text{ cm}$ . The temperature of the fluid surrounding the fin is  $27^{\circ}\text{C}$  with a convective heat transfer coefficient of  $15 \text{ W/m}^2\text{-K}$ . Consider the thermal conductivity of the fin as  $100 \text{ W/m-K}$  and the base temperature of the fin as  $760^{\circ}\text{C}$ . Calculate the total heat transfer from the engine cylinder. Assume negligible heat transfer from the fin tip.
12. Discuss in detail the role of following in automobiles
- Extended surfaces
  - Heat exchangers
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