### LINEAR SYSTEMS

### PREAMBLE (CODE)

```
from numpy import *
from numpy.linalg import *
from matplotlib.pyplot import *
from mpl_toolkits.mplot3d import *
from scipy.integrate import solve_ivp
```

### PREAMBLE

### **INPUTS**

It's handy to introduce non-autonomous ODEs.

There are designated as

$$\dot{x} = f(x, u)$$

where  $x \in \mathbb{R}^n$  and  $u \in \mathbb{R}^m$ , that is

$$f: \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$$
.

The vector-valued u is the system input.

This quantity may depend on the time t

$$u: t \in \mathbb{R} \mapsto u(t) \in \mathbb{R}^m$$

(actually it may also depend on some state, but we will adress this later).

#### A solution of

$$\dot{x} = f(x, u) \text{ and } x(t_0) = x_0$$

is merely a solution of

$$\dot{x} = h(t, x)$$
 and  $x(t_0) = x_0$ ,

where

$$h(t,x) = f(x, u(t)).$$

### OUTPUTS

We may complement the system dynamics with an equation

$$y = g(x, u) \in \mathbb{R}^p$$

The vector y refers to the **systems output**, usually the quantities that we can effectively measure in a system (the state x itself may be unknown).

# WHAT ARE LINEAR SYSTEMS?

### STANDARD FORM

Input  $u \in \mathbb{R}^m$ , state  $x \in \mathbb{R}^n$ , output  $y \in \mathbb{R}^p$ .

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

### WHY LINEAR?

Assume that:

$$\dot{x}_1 = Ax_1 + Bu_1, x_1(0) = x_{10},$$

$$\dot{x}_2 = Ax_2 + Bu_2, x_2(0) = x_{20},$$

#### Set

$$\bullet x_3 = \lambda x_1 + \mu x_2,$$

• 
$$u_3 = \lambda u_1 + \mu u_2$$
 and

$$x_{30} = \lambda x_{10} + \mu x_{20}.$$

for some  $\lambda$  and  $\mu$ .

Then

$$\dot{x}_3 = Ax_3 + Bu_3, \ x_3(0) = x_{30}.$$

## INTERNAL + EXTERNAL DYNAMICS

Corollary: Since  $(x_0, u) = (x_0, 0) + (0, u)$  the solution of

$$\dot{x} = Ax + Bu, \ x(0) = x_0$$

is the sum of the solutions  $x_1$  and  $x_2$  of:

### the internal dynamics

$$\dot{x}_1 = Ax_1, \ x_1(0) = x_0$$

(behavior controlled by the initial value only, no input)

and the external dynamics:

$$\dot{x}_2 = Ax_2 + Bu, \ x_2(0) = 0$$

(behavior controlled by the input, the systems is initially at rest)

### MATRIX SIZE

$$A \in \mathbb{R}^{n \times n}, B \in \mathbb{R}^{n \times m}, C \in \mathbb{R}^{p \times n}, D \in \mathbb{R}^{p \times m}.$$

$$egin{bmatrix} A & B \ \hline C & D \end{bmatrix}$$

### LTI SYSTEMS

They are actually referred to as linear time-invariant (LTI) systems:

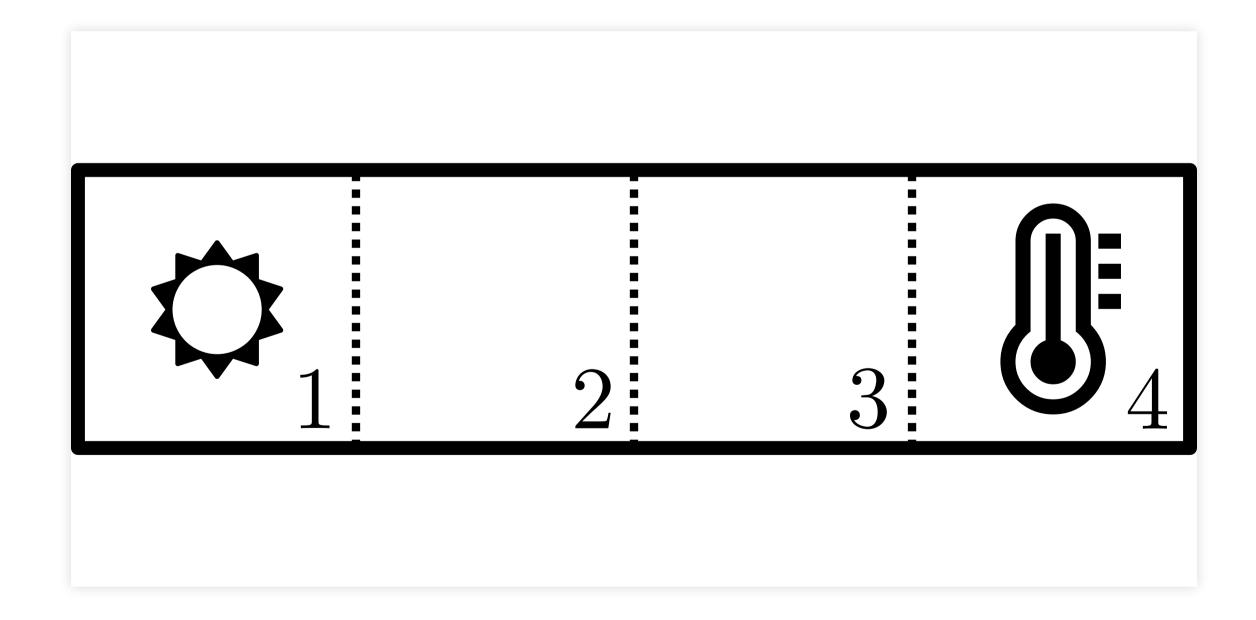
When x(t) is a solution of

$$\dot{x} = Ax + Bu, \ x(0) = x_0,$$

then  $x(t-t_0)$  is a solution of

$$\dot{x} = Ax + Bu(t - t_0), \ x(t_0) = x_0.$$

# — LINEAR SYSTEM / HEAT EQUATION



### SIMPLIFIED MODEL

- Four cells numbered 1 to 4 are arranged in a row.
- The first cell has a heat source, the last one a temperature sensor.
- The heat sink/source is increasing the temperature of its cell of *u* degrees by second.
- If the temperature of a cell is T and the one of a neighbor is  $T_n$ , T increases of  $T_n-T$  by second.

### Given the geometric layout:

• 
$$dT_1/dt = u + (T_2 - T_1)$$

• 
$$dT_2/dt = (T_1 - T_2) + (T_3 - T_2)$$

• 
$$dT_3/dt = (T_2 - T_3) + (T_4 - T_3)$$

• 
$$dT_4/dt = (T_3 - T_4)$$

• 
$$y = T_4$$

$$Set x = (T_1, T_2, T_3, T_4).$$

The model is linear and its standard matrices are:

$$A = \begin{bmatrix} -1 & 1 & 0 & 0 \\ 1 & -2 & 1 & 0 \\ 0 & 1 & -2 & 1 \\ 0 & 0 & 1 & -1 \end{bmatrix}$$

$$B = \begin{bmatrix} 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}, C = \begin{bmatrix} 0 & 0 & 0 & 1 \end{bmatrix}, D = [0]$$

### NONLINEAR TO LINEAR

Consider the nonlinear system

$$\dot{x} = f(x, u)$$

$$y = g(x, u)$$

Assume that  $x_e$  is an equilibrium when  $u = u_e$  (cst):

$$f(x_e, u_e) = 0$$

and let

$$y_e = g(x_e, u_e)$$

### Define the error variables

• 
$$\Delta x = x - x_e$$
,

• 
$$\Delta u = u - u_e$$
 and

• 
$$\Delta y = y - y_e$$
.

As long as the error variables stay small

$$f(x,u) \simeq \overbrace{f(x_e, u_e)}^{0} + \frac{\partial f}{\partial x}(x_e, u_e) \Delta x + \frac{\partial f}{\partial u}(x_e, u_e) \Delta u$$
$$g(x,u) \simeq \overbrace{g(x_e, u_e)}^{y_e} + \frac{\partial g}{\partial x}(x_e, u_e) \Delta x + \frac{\partial g}{\partial u}(x_e, u_e) \Delta u$$

Hence, the error variables satisfy approximately

$$d(\Delta x)/dt = A\Delta x + B\Delta u$$

$$\Delta y = C\Delta x + D\Delta u$$
with

$$\begin{bmatrix} A & B \\ \hline C & D \end{bmatrix} = \begin{bmatrix} \frac{\partial f}{\partial x} & \frac{\partial f}{\partial u} \\ \hline \frac{\partial g}{\partial x} & \frac{\partial g}{\partial u} \end{bmatrix} (x_e, u_e)$$

### **ASYMPTOTIC STABILITY**

The equilibrium  $x_e$  is locally asymptotically stable for

$$\dot{x} = f(x, u_e)$$
 $\iff$ 

The equilibrium  $\boldsymbol{0}$  is locally asymptotically stable for

$$\frac{d\Delta x}{dt} = A\Delta x$$
 where  $A = \partial f(x_e, u_e)/\partial x$ .

### **O** – LINEARIZATION

Consider

$$\dot{x} = -x^2 + u, \quad y = xu$$

If we set  $u_e=1$ , the system has an equilibrium at  $x_e=1$  (and also  $x_e=-1$  but we focus on the former) and the corresponding y is  $y_e=x_eu_e=1$ .

Around this configuration  $(x_e, u_e) = (1, 1)$ , we have

$$\frac{\partial(-x^2 + u)}{\partial x} = -2x_e = -2, \ \frac{\partial(-x^2 + u)}{\partial u} = 1,$$

and

$$\frac{\partial xu}{\partial x} = u_e = 1, \ \frac{\partial xu}{\partial u} = x_e = 1.$$

Thus, the approximate, linearized dynamics around this equilibrium is

$$d(x-1)/dt = -2(x-1) + (u-1)$$
$$y-1 = (x-1) + (u-1)$$

# ② – LINEARIZED DYNAMICS / PENDULUM

A pendulum submitted to a torque c is governed by

$$m\ell^2\ddot{\theta} + b\dot{\theta} + mg\ell\sin\theta = c.$$

We assume that only the angle  $\theta$  is effectively measured.

- What are the function f and g that determine the nonlinear dynamics of the pendulum when  $x = (\theta, \dot{\theta}), u = c$  and  $y = \theta$ ?
- Show that for any angle  $\theta_e$  we can find a constant value  $c_e$  of the torque such that  $x_e=(\theta_e,0)$  is an equilibrium.
- Compute the linearized dynamics of the pendulum around this equilibrium and put it in the standard form (compute A, B, C and D).

### INTERNAL DYNAMICS

We study the behavior of the solution

$$\dot{x} = Ax$$
,  $x(0) = x_0 \in \mathbb{R}^n$ 

We try to get some understanding with the simplest cases first.

### SCALAR CASE, REAL-VALUED

$$\dot{x} = ax$$

$$a \in \mathbb{R}, x(0) = x_0 \in \mathbb{R}.$$

#### Solution:

$$x(t) = e^{at}x_0$$

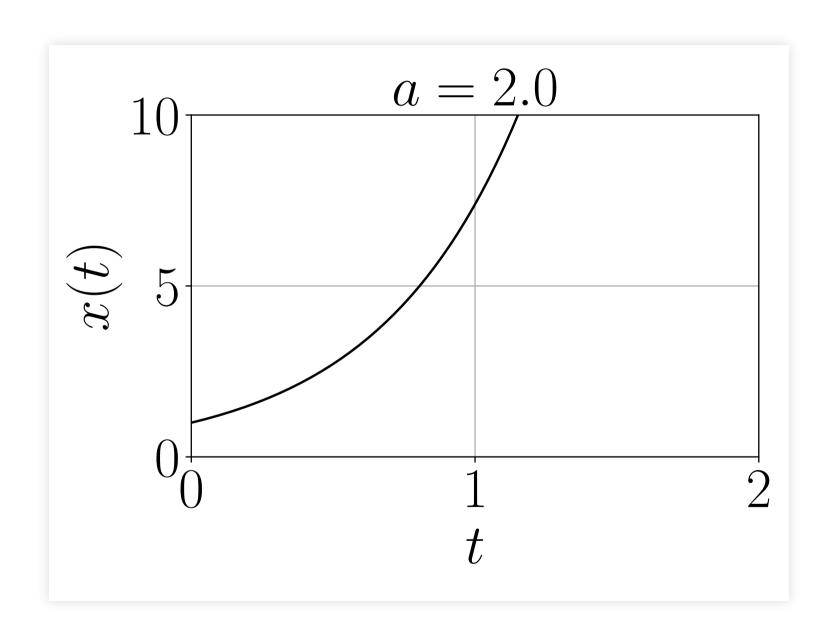
### **Proof:**

$$\frac{d}{dt}e^{at}x_0 = ae^{at}x_0 = ax(t)$$
and
$$x(0) = e^{a \times 0}x_0 = x_0.$$

### **TRAJECTORY**

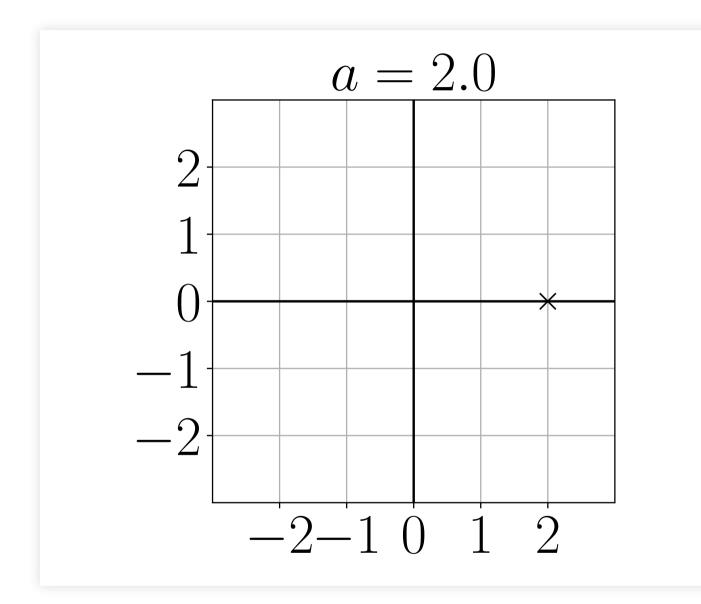
```
a = 2.0; x0 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
plot(t, exp(a*t)*x0, "k")
xlabel("$t$"); ylabel("$x(t)$"); title(f"$a={a}$")
grid(); axis([0.0, 2.0, 0.0, 10.0])
```

# **TRAJECTORY**



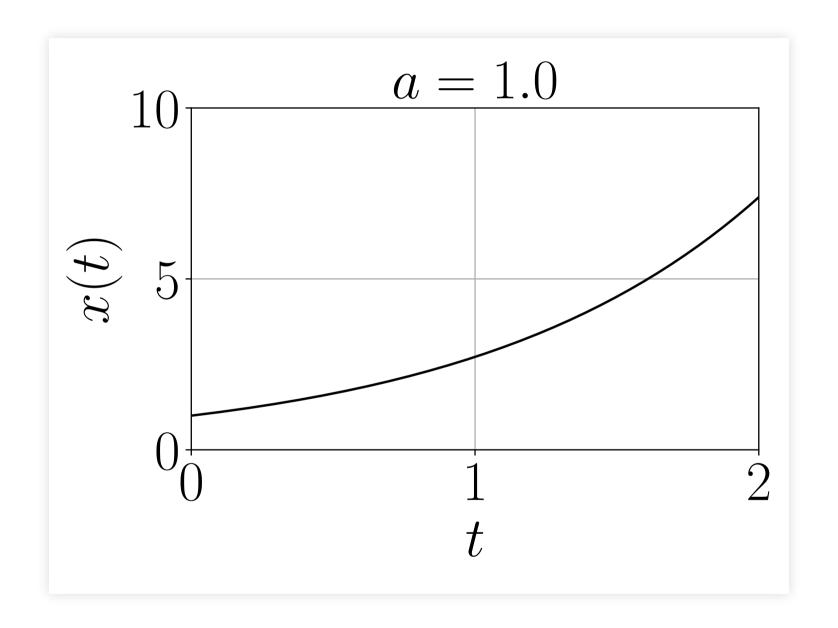


```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



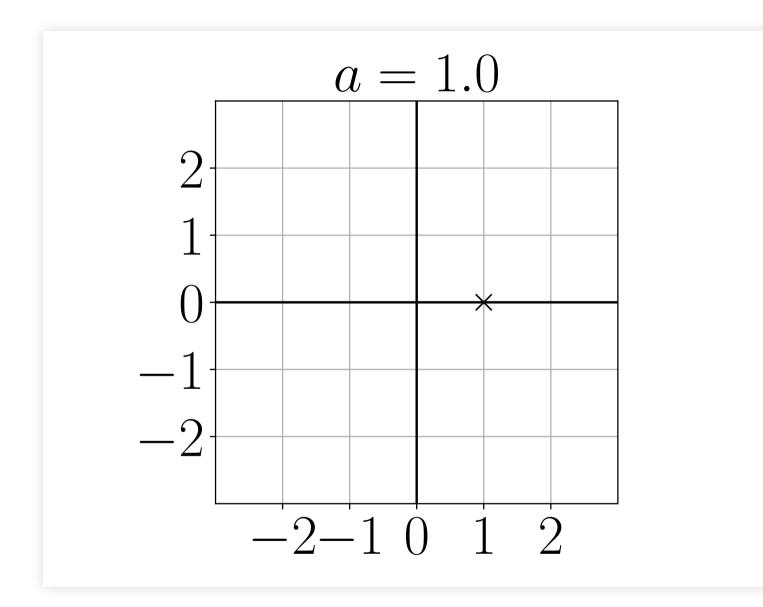


```
a = 1.0; x0 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
plot(t, exp(a*t)*x0, "k")
xlabel("$t$"); ylabel("$x(t)$"); title(f"$a={a}$")
grid(); axis([0.0, 2.0, 0.0, 10.0])
```



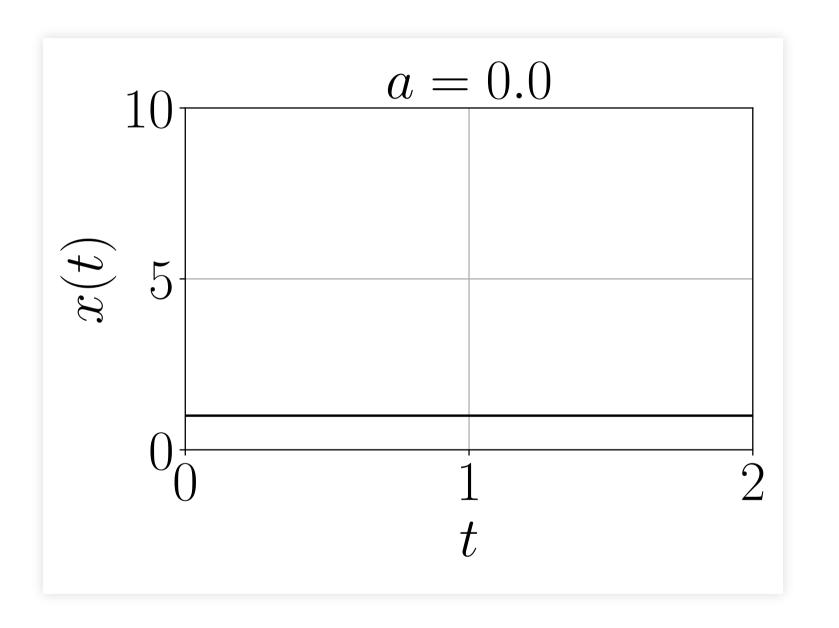


```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



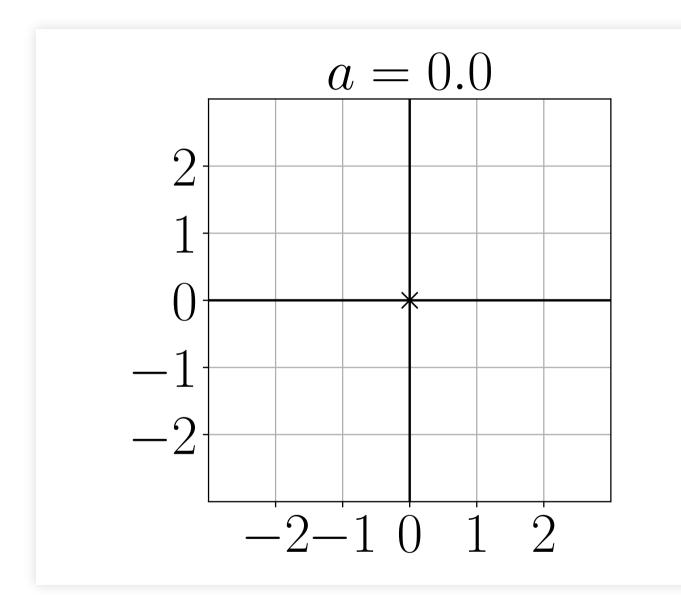


```
a = 0.0; x0 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
plot(t, exp(a*t)*x0, "k")
xlabel("$t$"); ylabel("$x(t)$"); title(f"$a={a}$")
grid(); axis([0.0, 2.0, 0.0, 10.0])
```



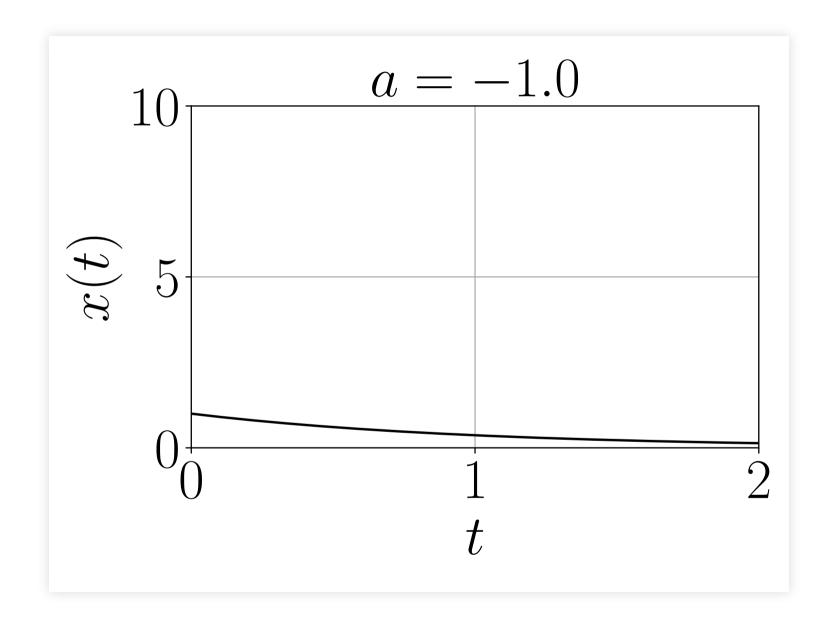


```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



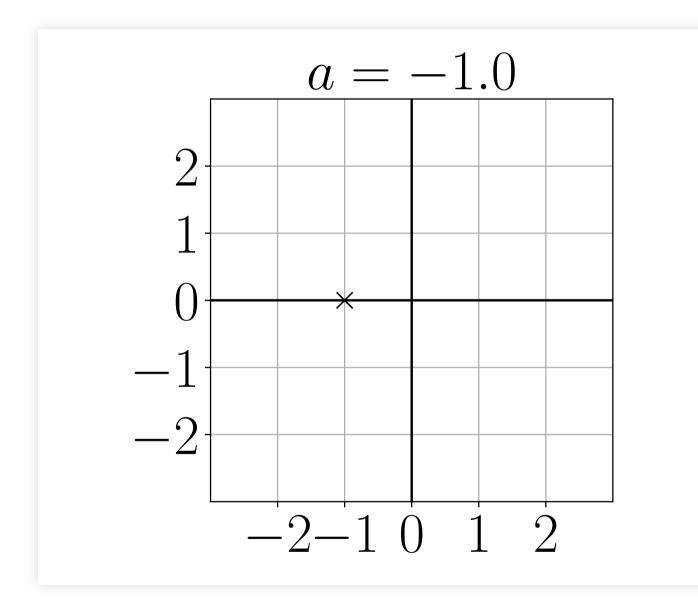


```
a = -1.0; x0 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
plot(t, exp(a*t)*x0, "k")
xlabel("$t$"); ylabel("$x(t)$"); title(f"$a={a}$")
grid(); axis([0.0, 2.0, 0.0, 10.0])
```



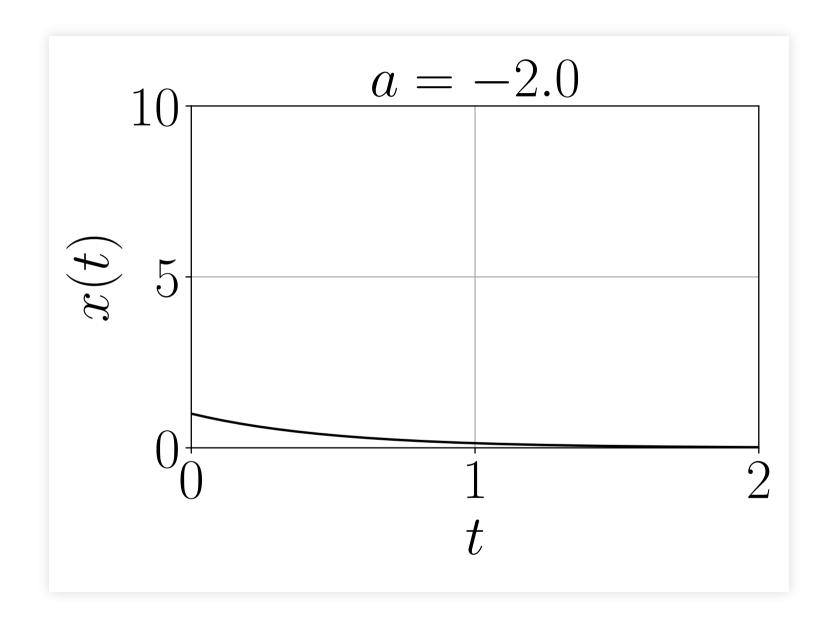


```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



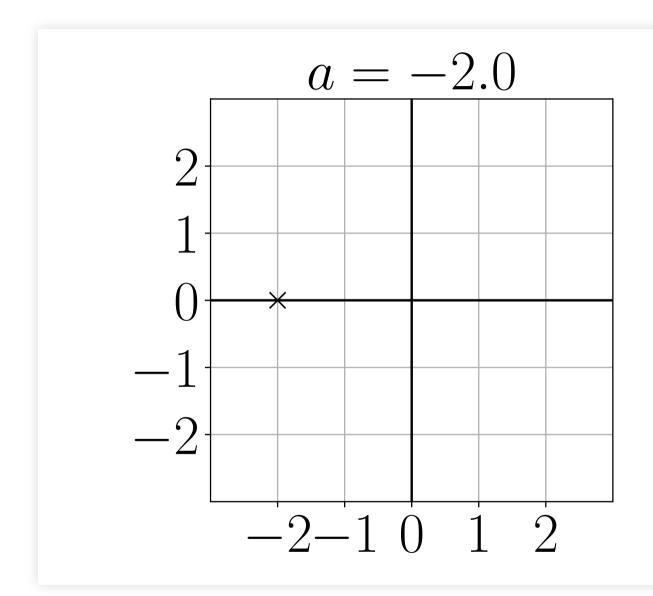


```
a = -2.0; x0 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
plot(t, exp(a*t)*x0, "k")
xlabel("$t$"); ylabel("$x(t)$"); title(f"$a={a}$")
grid(); axis([0.0, 2.0, 0.0, 10.0])
```





```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



## **ANALYSIS**

- The origin is globally asymptotically stable when a < 0.0:
  - a is in the open left-hand plane,
- In this case, define the time constant  $\tau = -1/a$ :

$$x(t) = e^{at}x_0 = e^{-t/\tau}x_0$$

au controls the time it take for the solution to (almost) reach to the origin:

- when  $t = \tau$ , |x(t)| is  $\approx 33\%$  of  $|x_0|$ ;
- when  $t = 3\tau$ , |x(t)| is  $\simeq 5\%$  of  $|x_0|$ .

# VECTOR CASE, DIAGONAL, REAL-VALUED

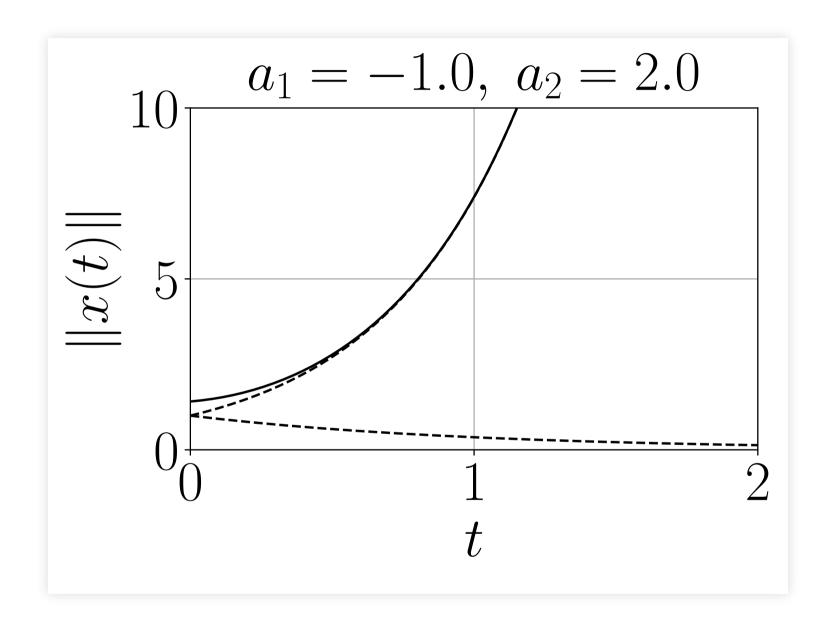
$$\dot{x}_1 = a_1 x_1, \ x_1(0) = x_{10}$$
 $\dot{x}_2 = a_2 x_2, \ x_2(0) = x_{20}$ 
i.e.
$$A = \begin{bmatrix} a_1 & 0 \\ 0 & a_2 \end{bmatrix}$$

#### Solution: by linearity

$$x(t) = e^{a_1 t} \begin{bmatrix} x_{10} \\ 0 \end{bmatrix} + e^{a_2 t} \begin{bmatrix} 0 \\ x_{20} \end{bmatrix}$$

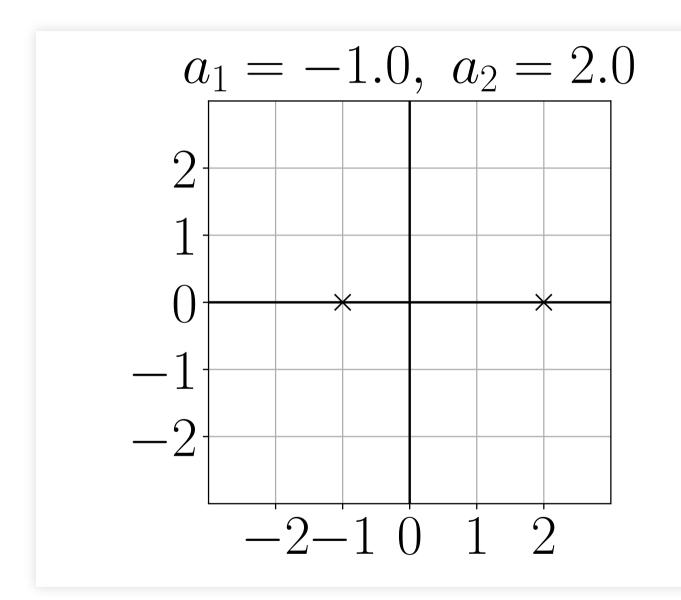


```
a1 = -1.0; a2 = 2.0; x10 = x20 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
x1 = \exp(a1*t)*x10; x2 = \exp(a2*t)*x20
xn = sqrt(x1**2 + x2**2)
plot(t, xn , "k")
plot(t, x1, "k--")
plot(t, x2 , "k--")
```



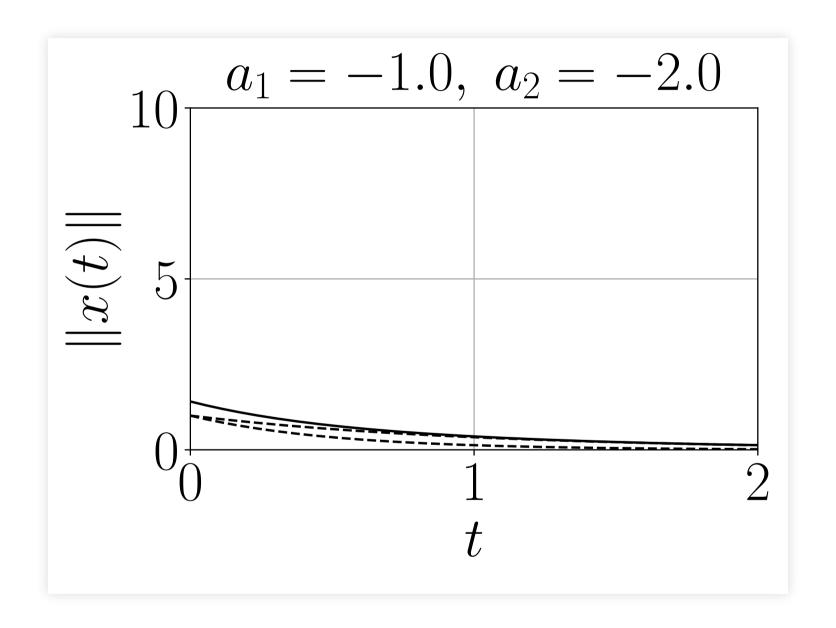


```
figure()
plot(real(a1), imag(a1), "x", color="k", ms=10.0)
plot(real(a2), imag(a2), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
```



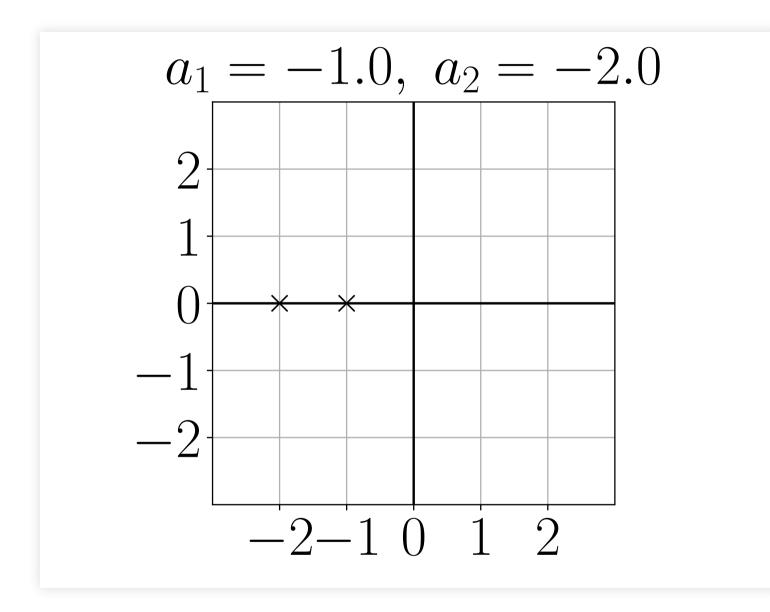


```
a1 = -1.0; a2 = -2.0; x10 = x20 = 1.0
figure()
t = linspace(0.0, 3.0, 1000)
x1 = \exp(a1*t)*x10; x2 = \exp(a2*t)*x20
xn = sqrt(x1**2 + x2**2)
plot(t, xn , "k")
plot(t, x1, "k--")
plot(t, x2, "k--")
```





```
figure()
plot(real(a1), imag(a1), "x", color="k", ms=10.0)
plot(real(a2), imag(a2), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
```



## **ANALYSIS**

- The rightmost  $a_i$  determines the asymptotic behavior,
- The origin is globally asymptotically stable only when every  $a_i$  is in the open left-hand plane.

# SCALAR CASE, COMPLEX-VALUED

$$\dot{x} = ax$$

$$a \in \mathbb{C}, x(0) = x_0 \in \mathbb{C}.$$

#### Solution: formally, the same old solution

$$x(t) = e^{at}x_0$$

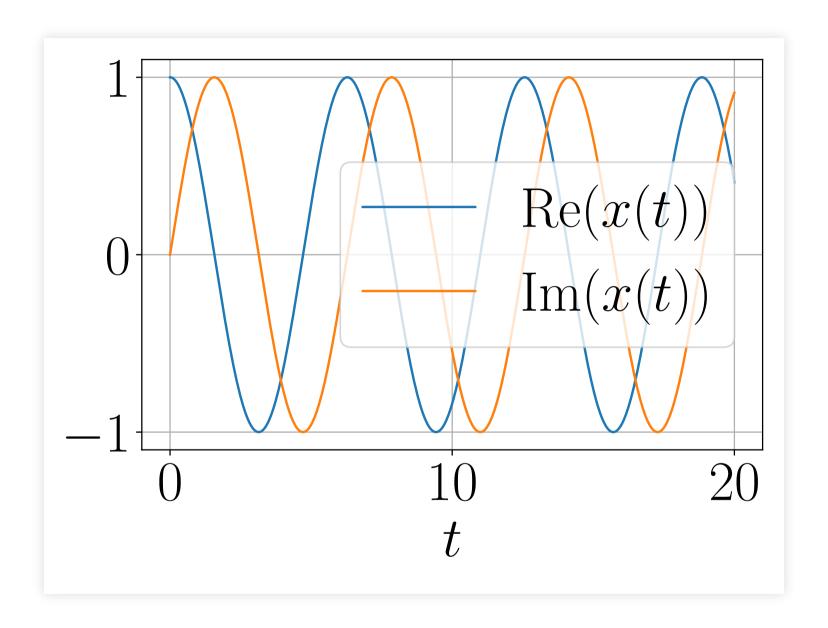
But now,  $x(t) \in \mathbb{C}$ :

if 
$$a = \sigma + i\omega$$
 and  $x_0 = |x_0|e^{i\angle x_0}$ 

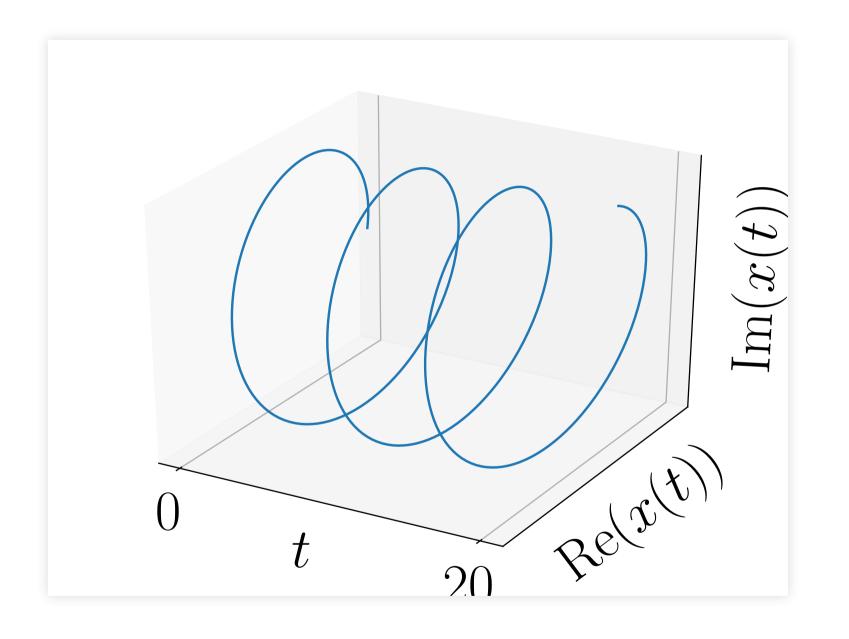
$$|x(t)| = |x_0|e^{\sigma t}$$
 and  $\angle x(t) = \angle x_0 + \omega t$ .



```
a = 1.0j; x0=1.0
figure()
t = linspace(0.0, 20.0, 1000)
plot(t, real(exp(a*t)*x0), label="$\mathrm{Re}
(x(t))$")
plot(t, imag(exp(a*t)*x0), label="$\mathrm{Im}
(x(t))$")
xlabel("$t$")
```

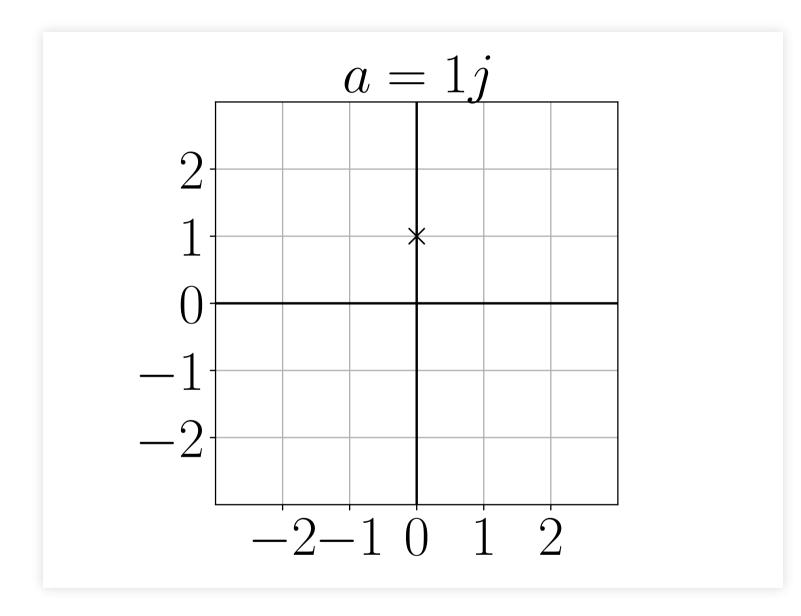


```
fig = figure()
ax = fig.add_subplot(111, projection="3d")
zticks = ax.set zticks
ax.plot(t, real(exp(a*t)*x0), imag(exp(a*t)*x0))
xticks([0.0, 20.0]); yticks([]); zticks([])
ax.set_xlabel("$t$")
ax.set_ylabel("$\mathrm{Re}(x(t))$")
ax.set_zlabel("$\mathrm{Im}(x(t))$")
```



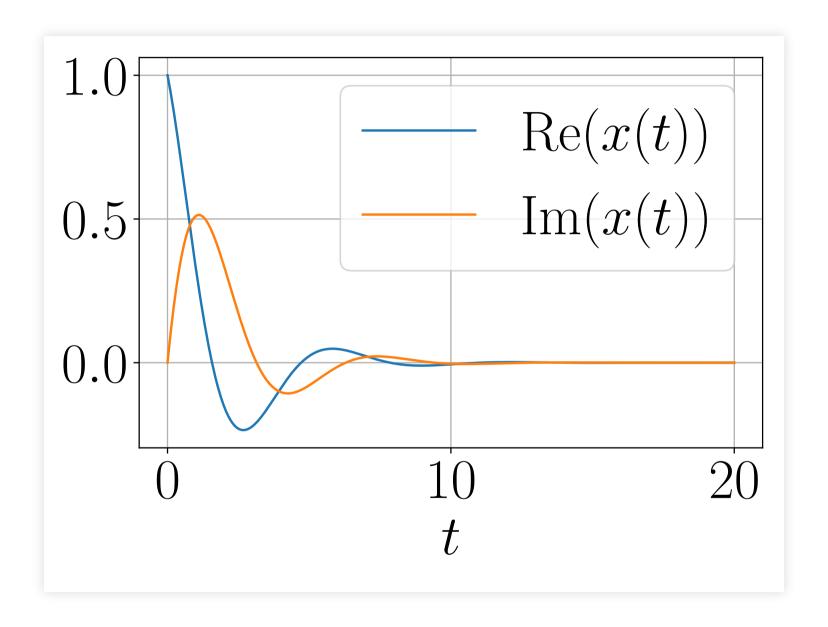


```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



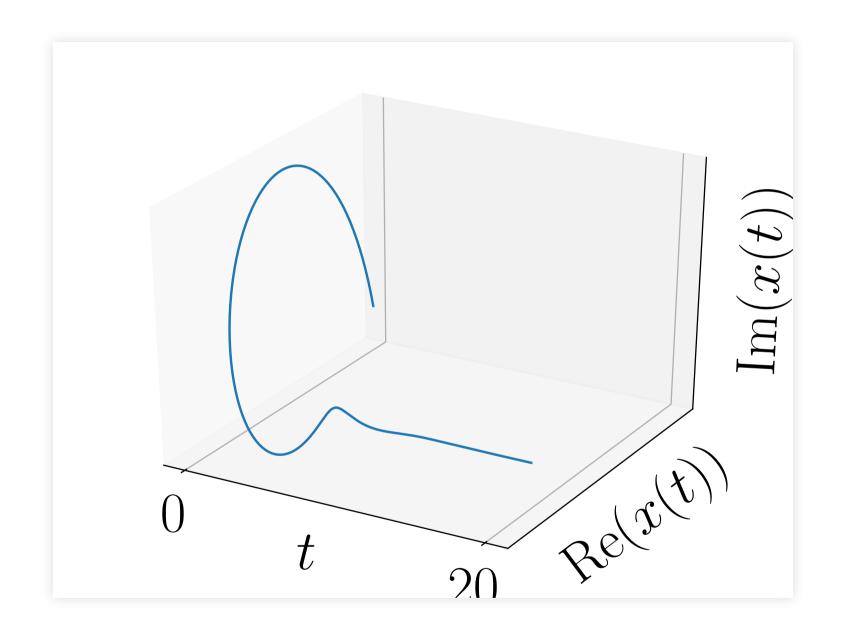


```
a = -0.5 + 1.0j; x0=1.0
figure()
t = linspace(0.0, 20.0, 1000)
plot(t, real(exp(a*t)*x0), label="$\mathrm{Re}
(x(t))$")
plot(t, imag(exp(a*t)*x0), label="$\mathrm{Im}
(x(t))$")
xlabel("$t$")
```



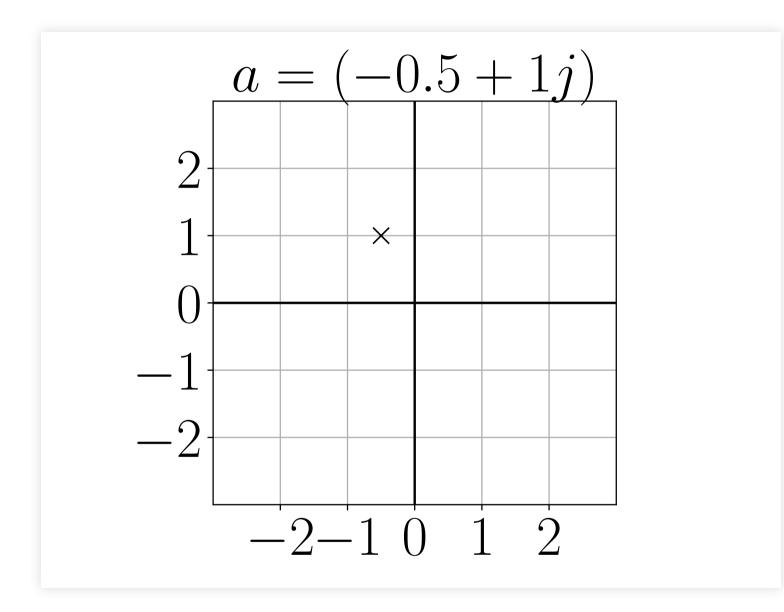


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fig = figure()
ax = fig.add_subplot(111, projection="3d")
zticks = ax.set zticks
ax.plot(t, real(exp(a*t)*x0), imag(exp(a*t)*x0))
xticks([0.0, 20.0]); yticks([]); zticks([])
ax.set_xlabel("$t$")
ax.set_ylabel("$\mathrm{Re}(x(t))$")
ax.set_zlabel("$\mathrm{Im}(x(t))$")
```





```
figure()
plot(real(a), imag(a), "x", color="k", ms=10.0)
gca().set_aspect(1.0)
xlim(-3,3); ylim(-3,3);
plot([-3,3], [0,0], "k")
plot([0, 0], [-3, 3], "k")
xticks([-2,-1,0,1,2]); yticks([-2,-1,0,1,2])
title(f"$a={a}$")
```



### **ANALYSIS**

• the origin is globally asymptotically stable if  $\alpha$  is in the open left-hand plane:

- if  $a = \sigma + i\omega$ ,
  - $\tau = -1/\sigma$  is the time constant related of the speed of convergence,
  - $\omega$  the (rotational) frequency of the (damped) oscillations.

Only one step left before the (almost) general case ...

#### EXPONENTIAL MATRIX

If  $M \in \mathbb{C}^{n \times n}$ , the **exponential** is defined as:

$$e^{M} = \sum_{i=0}^{+\infty} \frac{M^{n}}{n!} \in \mathbb{C}^{n \times n}$$



The exponential of a matrix M is *not* the matrix with elements  $e^{M_{ij}}$  (the elementwise exponential).

- elementwise exponential: exp (numpy module),
- exponential: expm (scipy.linalg module).

### **?** EXPONENTIAL MATRIX

Let

$$M = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}$$

•  $[\mathbf{x}^2]$  Compute the exponential of M.

**Q.** Hint: 
$$\cosh x = \frac{e^x + e^{-x}}{2}$$
,  $\sinh x = \frac{e^x - e^{-x}}{2}$ .

• [△] Check the results with expm.

#### Note that

$$\frac{d}{dt}e^{At} = \frac{d}{dt} \sum_{n=0}^{+\infty} \frac{A^n}{n!} t^n$$

$$= \sum_{n=1}^{+\infty} \frac{A^n}{(n-1)!} t^{n-1}$$

$$= A \sum_{n=1}^{+\infty} \frac{A^{n-1}}{(n-1)!} t^{n-1} = Ae^{At}$$

Thus, for any  $A \in \mathbb{C}^{n \times n}$  and  $x_0 \in \mathbb{C}^n$ ,

$$\frac{d}{dt}(e^{At}x_0) = A(e^{At}x_0)$$

### INTERNAL DYNAMICS

The solution of

$$\dot{x} = Ax$$
 and  $x(0) = x_0$ 

is

$$x(t)=e^{At}x_0.$$

### STABILITY CRITERIA

Let  $A \in \mathbb{C}^{n \times n}$ .

The origin of  $\dot{x} = Ax$  is globally asymptotically stable



all eigenvalues of A have a negative real part.

### ② G.A.S. ⇔L.A.

Show that for a linear systems  $\dot{x} = Ax$ , it is enough that the origin is locally attractive for the system to be globally asymptotically stable.

#### WHY DOES THIS CRITERIA WORK?

Assume that A is diagonalizable with eigenvalues  $\{\lambda_1,\ldots,\lambda_n\}.$ 

(Very likely unless A has some special structure)

# Then, there is an invertible matrix $P \in \mathbb{C}^{n \times n}$ such that

$$P^{-1}AP = \operatorname{diag}(\lambda_1, \dots, \lambda_n) = \begin{bmatrix} \lambda_1 & 0 & \cdots & 0 \\ 0 & \lambda_2 & \cdots & 0 \\ \vdots & \vdots & \vdots & \vdots \\ 0 & \cdots & \cdots & \lambda_n \end{bmatrix}$$

Thus, if 
$$y = P^{-1}x$$
,  $\dot{x} = Ax$  is equivalent to

$$\begin{vmatrix} \dot{y}_1 & = & \lambda_1 y_1 \\ \dot{y}_2 & = & \lambda_2 y_2 \\ \vdots & = & \vdots \\ \dot{y}_n & = & \lambda_n y_n \end{vmatrix}$$

The system is G.A.S. iff each component of the system is, which holds iff  $\operatorname{Re}\lambda_i < 0$  for each i.

## ② STABILITY / 2ND-ORDER SYSTEM

Consider the scalar ODE

$$\ddot{x} + kx = 0$$
, with  $k > 0$ 

- [ $\mathbf{x}^2$ ] Determine the representation of this system as a first-order ODE with state  $(x, \dot{x})$ .
- [**?**, **x**<sup>2</sup>] Is this system asymptotically stable?

- [ $\mathbf{\hat{y}}, \mathbf{x}^2$ ] If its solutions oscillate, determine its (rotational) frequency  $\boldsymbol{\omega}$ ?
- [ $\mathbf{\hat{y}}, \mathbf{x}^2$ ] Characterize the asymptotic behavior of x(t) when  $\ddot{x} + b\dot{x} + kx = 0$  for some b > 0.

## **?** STABILITY / INTEGRATORS

Consider the system

$$\dot{x} = Jx \text{ with } J = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \vdots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ 0 & 0 & 0 & \dots & 0 \end{bmatrix}$$

• [\$\overline{\mathbb{X}}, \mathbb{x}^2] Compute the solution when

$$x(0) = \begin{bmatrix} 0 \\ \vdots \\ 0 \end{bmatrix},$$

then for any initial condition.

- [ $\mathbb{Q}, \mathbf{x}^2$ ] Same questions when  $\dot{x} = (\lambda I + J)x$  for some  $\lambda \in \mathbb{C}$ .
- [2] Is the system asymptotically stable? Why does it matter in general?

# I/O BEHAVIOR

### CONTEXT

Assume that the system is "initially at rest":

$$x(0) = 0$$

- Forget about the state x(t) (may be unknown)
- Study the input/output (I/O) relationship:

$$u \rightarrow y$$

In this context, we have:

$$y(t) = \int_0^t Ce^{A(t-\tau)} Bu(\tau) d\tau + Du(t)$$

#### CAUSAL SIGNALS

- extend u(t) and y(t) by 0 when t < 0 (as causal signals).
- introduce the Heaviside function defined by

$$e(t) = \begin{vmatrix} 1 & \text{if } t \ge 0, \\ 0 & \text{if } t < 0. \end{vmatrix}$$

### IMPULSE RESPONSE

The system **impulse response** is defined by:

$$H(t) = (Ce^{At}B) \times e(t) + D\delta(t) \in \mathbb{R}^{p \times m}$$

- works for general or MIMO systems.

  MIMO = multiple-input & multiple-output systems.
- $\blacksquare \delta(t)$  is the **unit impulse**, we'll get back to it (in the meantime, you may assume that D=0).

### SISO SYSTEMS

When

$$p = m = 1$$

(single-input & single-output or SISO systems),

the  $1 \times 1$  matrix H(t) is identified with a scalar h(t):

$$H(t) = [h(t)]$$

Then, we have:

$$y(t) = \int_{-\infty}^{+\infty} H(t - \tau)u(\tau) d\tau$$

and denote \* this operation between H and u:

$$y(t) = (H * u)(t)$$

It's called a convolution.

### **© IMPULSE RESPONSE**

Consider the SISO system

$$\begin{vmatrix} \dot{x} & = ax + u \\ y & = x \end{vmatrix}$$
  
where  $a \neq 0$ .

#### We have

$$H(t) = (Ce^{At}B) \times e(t) + D\delta(t)$$
$$= [1]e^{[a]t}[1]e(t) + [0]\delta(t)$$
$$= [e(t)e^{at}]$$

When u(t) = e(t) for example,

$$y(t) = \int_{-\infty}^{+\infty} e(t - \tau)e^{a(t - \tau)}e(\tau) d\tau$$

$$= \int_{0}^{t} e^{a(t - \tau)} d\tau$$

$$= \int_{0}^{t} e^{a\tau} d\tau$$

$$= \frac{1}{a}(e^{at} - 1)$$

## ② IMPULSE RESPONSE / INTEGRATOR

• [x²] Compute the impulse response of the system

$$\begin{vmatrix} \dot{x} & = & u \\ y & = & x \end{vmatrix}$$

where  $u \in \mathbb{R}, x \in \mathbb{R}$  and  $y \in \mathbb{R}$ .

# ② IMPULSE RESPONSE / DOUBLE INTEGRATOR

• [x²] Compute the impulse response of the system

$$\begin{vmatrix} \dot{x}_1 & = & x_2 \\ \dot{x}_2 & = & u \\ y & = & x_1 \end{vmatrix}$$

where  $u \in \mathbb{R}$ ,  $x = (x_1, x_2) \in \mathbb{R}^2$  and  $y \in \mathbb{R}$ .

## **②IMPULSE RESPONSE / GAIN**

• [x²] Compute the impulse response of the system

$$y = Ku$$

where  $u \in \mathbb{R}^m$ ,  $y \in \mathbb{R}^p$  and  $K \in \mathbb{R}^{p \times m}$ .

## ② IMPULSE RESPONSE / MIMO SYSTEM

•  $[\mathbf{x^2}]$  Find a linear system with matrices A,B,C,D whose impulse response is

$$H(t) = \begin{bmatrix} e^t e(t) & e^{-t} e(t) \end{bmatrix}$$

•  $[\mathbf{x}^2]$  Is there another set of matrices A, B, C, D with the same impulse response? With a matrix A of a different size?

#### LAPLACE TRANSFORM

Associate to a scalar signal  $x(t) \in \mathbb{R}, t \in \mathbb{R}$ , the function of a complex argument  $s \in \mathbb{C}$ :

$$x(s) = \int_{-\infty}^{+\infty} x(t)e^{-st} dt.$$

defined when  $\operatorname{Re}(s) > \sigma \operatorname{if} ||x(t)|| \leq Ke^{\sigma t}$ .

### **A** NOTATION

We use the same symbol (here "x") to denote:

- a signal x(t) and
- its Laplace transform x(s)

They are two equivalent representations of the same "object", but different mathematical "functions".

If you fear some ambiguity, use named variables, e.g.:

$$x(t = 1)$$
 or  $x(s = 1)$  instead of  $x(1)$ .

## VECTOR/MATRIX-VALUED SIGNALS

The Laplace transform

- of a vector-valued signal  $x(t) \in \mathbb{R}^n$  or
- of a matrix-valued signals  $X(t) \in \mathbb{R}^{m \times n}$

are computed elementwise.

$$x_i(s) = \int_{-\infty}^{+\infty} x_i(t)e^{-st} dt.$$

$$X_{ij}(s) = \int_{-\infty}^{+\infty} X_{ij}(t)e^{-st} dt.$$

#### RATIONAL & CAUSAL SIGNALS

We will only deal with rational & causal signals:

$$x(t) = \left(\sum_{\lambda \in \Lambda} p_{\lambda}(t)e^{\lambda t}\right) e(t)$$

where:

- $\Lambda$  is a finite subset of  $\mathbb{C}$ ,
- for every  $\lambda \in \Lambda$ ,  $p_{\lambda}(t)$  is a polynomial in t.

- Such signals are causal since
  - x(t) = 0 when t < 0.
  - Causality  $\Leftrightarrow$  deg  $n(s) \leq$  deg d(s).
- They are rational since

$$x(s) = \frac{n(s)}{d(s)}$$

where n(s) and d(s) are polynomials.

## LAPLACE TRANSFORM / EXPONENTIAL

$$Set x(t) = e(t)e^{at}$$

$$x(s) = \int_0^{+\infty} e^{at} e^{-st} dt = \int_0^{+\infty} e^{(a-s)t} dt$$
$$= \left[ \frac{e^{(a-s)t}}{a-s} \right]_0^{+\infty} = \frac{1}{s-a}$$

 $(\operatorname{lf}\operatorname{Re}(s) \ge \operatorname{Re}(a) + \epsilon, \operatorname{then}|e^{(a-s)t}| \le e^{-\epsilon t})$ 

#### SYMBOLIC COMPUTATIONS

```
import sympy
from sympy.abc import t, s, a
from sympy.integrals.transforms import
laplace_transform
def L(f):
    return laplace_transform(f, t, s)[0]
```

```
xt = sympy.exp(a*t)
xs = L(xt) # 1/(-a + s)
```

### **?** LAPLACE TRANSFORM / RAMP

Compute the Laplace Transform of

$$x(t) = te(t)$$

#### **CONVOLUTION & LAPLACE**

Let H(t) be the impulse response of a system.

Its Laplace transform H(s) is called the system transfer function.

For LTI systems in standard form, we have

$$H(s) = C[sI - A]^{-1}B + D$$

#### **OPERATIONAL CALCULUS**

The Laplace transform turns convolution into products:

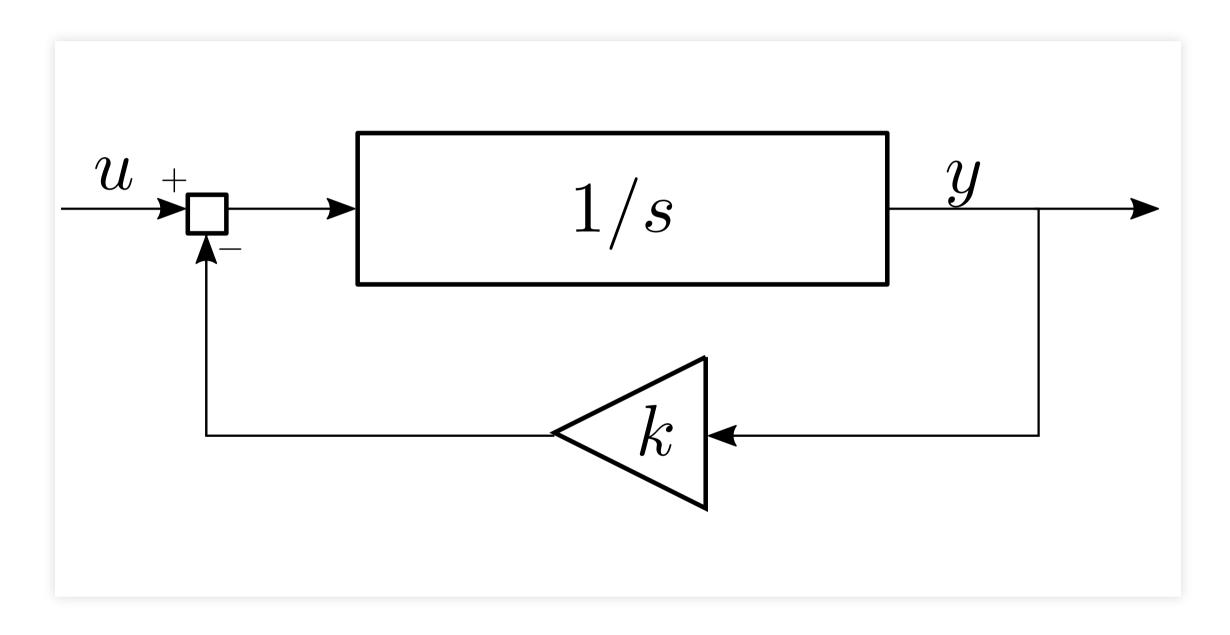
$$y(t) = (H * u)(t) \iff y(s) = H(s) \times u(s)$$

#### GRAPHICAL LANGUAGE

Control engineers used *block diagrams* to describe (combinations of) dynamical systems, with

- "boxes" to determine the relation between input signals and output signals and
- "wires" to route output signals to inputs signals.

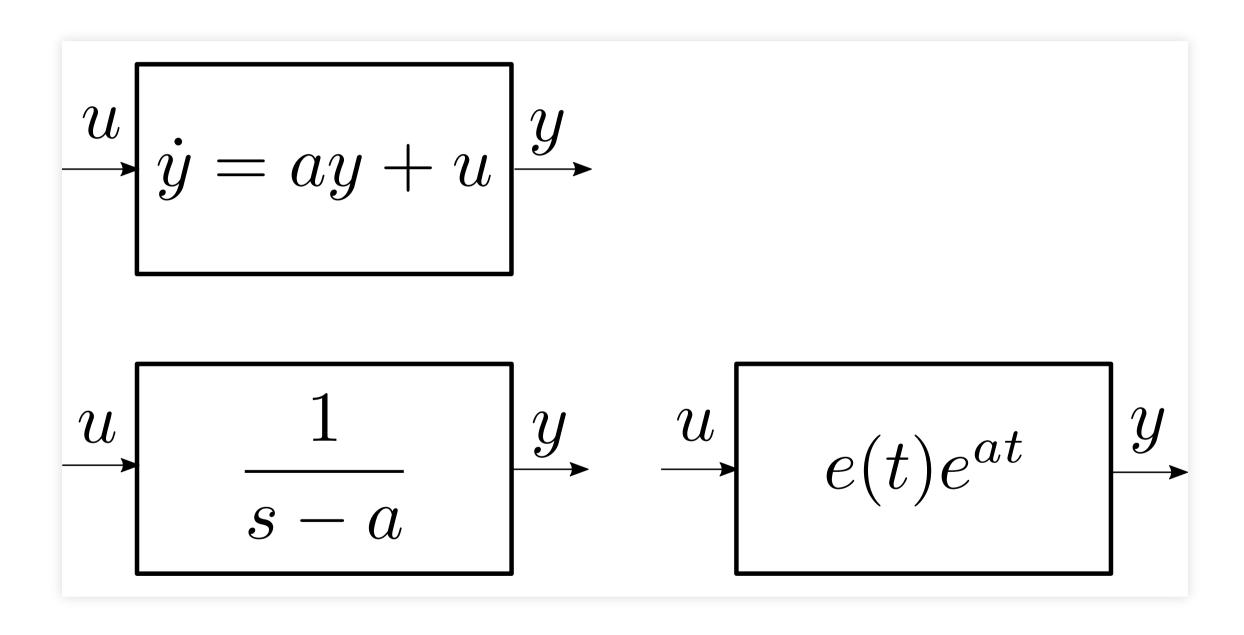
## **® BLOCK-DIAGRAM / FEEDBACK**



- Triangles denote gains (scalar or matrix multipliers),
- Adders sum (or substract) signals.

- LTI systems can be specified with:
  - (differential) equations,
  - the impulse response,
  - the transfer function,

### **EQUIVALENT SYSTEMS**



## ③ BLOCK-DIAGRAM / FEEDBACK

Compute the transfer function H(s) of the system depicted in the feedback block-diagram example.

#### IMPULSE RESPONSE

Why refer to h(t) as the system "impulse response"?

By the way, what's an impulse?

### **© IMPULSES APPROXIMATIONS**

Pick a time constant  $\epsilon > 0$  and define

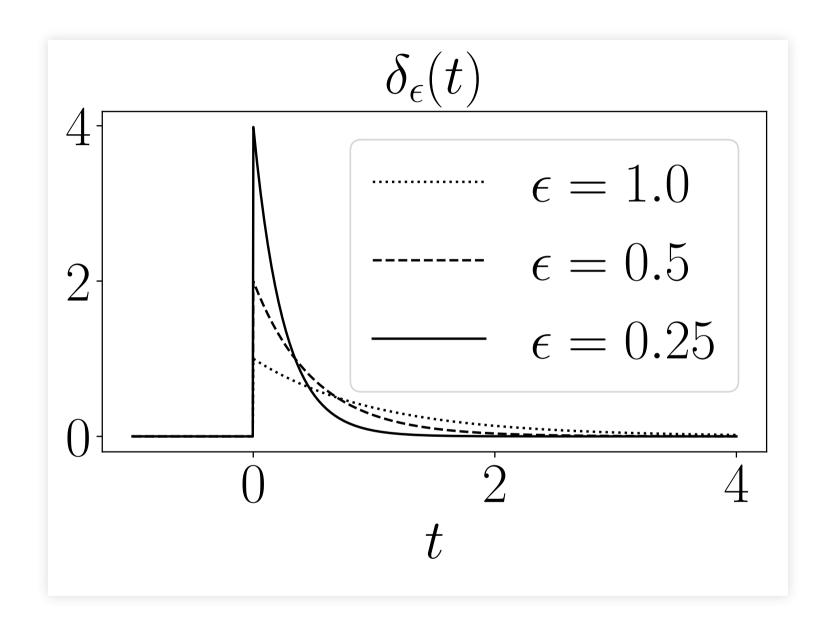
$$\delta_{\epsilon}(t) = \frac{1}{\epsilon} e^{-t/\epsilon} e(t)$$

```
def delta(t, eps=1.0):

return exp(-t / eps) / eps * (t >= 0)
```



```
figure()
t = linspace(-1, 4, 1000)
plot(t, delta(t, eps=1.0), "k:",
label="$\epsilon=1.0$")
plot(t, delta(t, eps=0.5), "k--",
label="$\epsilon=0.5$")
plot(t, delta(t, eps=0.25), "k",
label="$\epsilon=0.25$")
```



## IMPULSES IN THE LAPLACE DOMAIN

$$\delta_{\epsilon}(s) = \int_{-\infty}^{+\infty} \delta_{\epsilon}(t)e^{-st} dt$$

$$= \frac{1}{\epsilon} \int_{0}^{+\infty} e^{-(s+1/\epsilon)t} dt$$

$$= \frac{1}{\epsilon} \left[ \frac{e^{-(s+1/\epsilon)t}}{-(s+1/\epsilon)} \right]_{0}^{+\infty} = \frac{1}{1+\epsilon s}$$

- The "limit" of the signal  $\delta_{\epsilon}(t)$  when  $\epsilon \to 0$  is not defined as a function (issue for t=0) but as a generalized function  $\delta(t)$ , the unit impulse.
- This technicality can be avoided in the Laplace domain where

$$\delta(s) = \lim_{\epsilon \to 0} \delta_{\epsilon}(s) = \lim_{\epsilon \to 0} \frac{1}{1 + \epsilon s} = 1.$$

Thus, if 
$$y(t) = (h * u)(t)$$
 and

- 1.  $u(t) = \delta(t)$  then
- $2. y(s) = h(s) \times \delta(s) = h(s) \times 1 = h(s)$
- 3. and thus y(t) = h(t).

Conclusion: the impulse response h(t) is the output of the system when the input is the unit impulse  $\delta(t)$ .

### I/O STABILITY

A system is I/O-stable if there is a  $K \geq 0$  such that

for any 
$$t \ge$$
,  $||y(t)|| \le KM$ 

whenever

for any 
$$t \ge$$
,  $||u(t)|| \le M$ 

There is a bound on the amplification of the input signal that the system can provide.

Also called BIBO-stability (for "bounded input, bounded output")

#### TRANSFER FUNCTION POLES

A **pole** of the transfer function H(s) is a  $s \in \mathbb{C}$  such that for at least one element  $H_{ij}(s)$ ,

$$|H_{ij}(s)| = +\infty.$$

### I/O-STABILITY CRITERIA

A system is I/O-stable if and only if all its poles are in the open left-plane, i.e. such that

## INTERNAL STABILITY VS I/O-STABILITY

If the system  $\dot{x}=Ax$  is asymptotically stable, then for any matrices B,C,D of appropriate sizes,

$$\dot{x} = Ax + Bu$$
 $\dot{y} = Cx + Du$ 
is I/O-stable.

## FULLY ACTUATED & MEASURED SYSTEM

If 
$$B=I, C=I$$
 and  $D=0$ , that is  $\dot{x}=Ax+u, \ y=x$  then  $H(s)=[sI-A]^{-1}$ .

Therefore, s is a pole of H iff it's an eigenvalue of A.

Thus, in this case, asymptotic stability and I/O-stability are equivalent.

This equivalence holds under much weaker conditions.