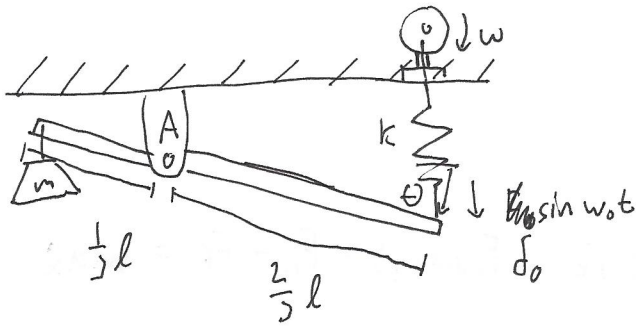


20-R-VII-DY-19 Intermediate

A straight bar, of length  $l$ , is pinned to the ceiling at point A. On one end, there is a ~~10kg~~ weight and on the other, there is a spring with a  $k$  value of  $150 \text{ N/m}$ .

A motor applies a periodic ~~force~~ <sup>displacement</sup>  $\delta = 0.5 \sin \omega t$  to the spring. What is the maximum velocity of the steady-state vibration?



Solution: FBD

$\sum M_A$ :  ~~$2k y_z$~~

$$\left(\frac{2}{3}l\right)k(y_z - \delta_0 \sin \omega t) = -ma\left(\frac{l}{3}\right)$$

( $mg$  &  $kx_p$  cancel out)

~~$$2k y_z = \dots$$~~

$$2k(y_z - \delta_0 \sin \omega t) = -ma$$

$$2k y_z + ma = 2k \delta_0 \sin \omega t$$

$$2k\left(\frac{2l}{3}\right)\theta + m\left(\frac{l}{3}\ddot{\theta}\right) = 2k \delta_0 \sin \omega t$$

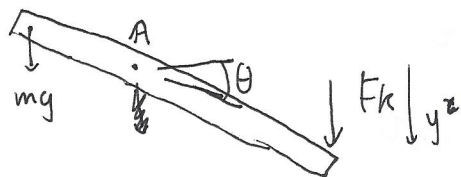
$$\ddot{\theta} + \frac{4k}{m}\theta = \frac{6k \delta_0 \sin \omega t}{lm}$$

$$-C\omega_0^2 \sin \omega t + \frac{4k}{m}(C \sin \omega t) = \frac{6k \delta_0 \sin \omega t}{lm}$$

$$C\left(\frac{4k}{m} - \omega_0^2\right) = \frac{6k \delta_0}{lm}$$

$$x_p = C \sin \omega t = \theta_p$$

$$\ddot{x}_p = -C\omega_0^2 \sin \omega t = \ddot{\theta}_p$$

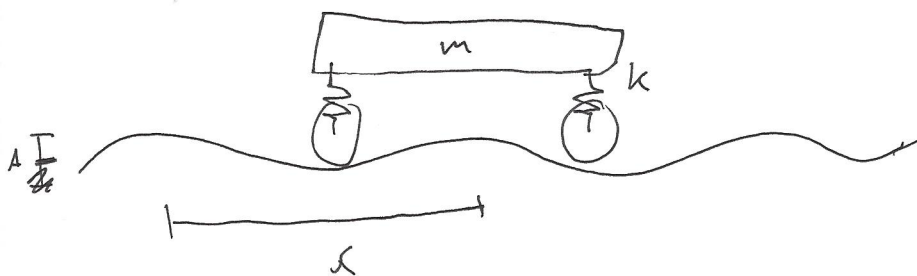


$$y = \frac{2l}{3} \sin \theta \approx \frac{2l}{3} \theta$$

$$a = \alpha \frac{l}{3} \quad \alpha = \ddot{\theta}$$

20-R-VIB-DY-20 Intermediate

A 2000 kg car is driving across a bumpy road which can be described as a sinusoidal wave with an amplitude of 0.1 m and a wave length of 6 m. There are 4 springs, one for each wheel, and they all have a spring constant of 1000 N/m. Find the velocity of the car that will produce the greatest vibration.



~~6/3~~ Solution:  $\delta_0 = 0.1 \text{ m}$   $\lambda = 6 \text{ m}$

$$k = 4(1000) = 4000$$

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{2} \quad \tau = \frac{2\pi}{\omega_n} = \frac{2\pi}{\sqrt{2}}$$

$\omega_0 = \omega_n$  for resonance

$$\lambda = \frac{6 \text{ m}}{\text{in}} \quad \frac{2\pi}{\sqrt{2}} \text{ s}$$

$$v = \frac{\lambda}{\tau} = 1.35 \text{ m/s}$$

$$C = \frac{6k\delta_0}{lm(\frac{4k}{m} - \omega_0^2)} = \frac{6k\delta_0}{l(4k - \omega_0^2 m)}$$

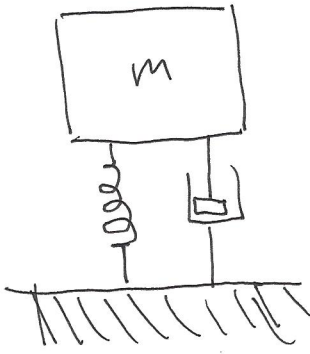
$$\ddot{\theta}_p = m(\omega_0 \cos \omega_0 t) = \frac{6k\delta_0 \omega_0}{l(4k - \omega_0^2 m)} \cos \omega_0 t$$

$$v_{\max} = \dot{\theta}_p \cdot r = \frac{6k\delta_0 \omega_0 \cancel{l}}{l^3(4k - \omega_0^2 m)} = \frac{2k\delta_0 \omega_0}{(4k - \omega_0^2 m)}$$

20-R-VIB-DY-21

Beginner

A simple spring, damper, and load system is configured as shown. Given that the  $k = 15 \text{ N/m}$  and  $m = 10 \text{ kg}$ , what damping constant will make the system critically damped?



Solution:  $c_c = 2m\omega_n = 2m\sqrt{\frac{k}{m}} = 24.49 \text{ Ns/m}$