

## Centralized control

The manipulator's dynamics are

$$B(q) \ddot{q} + \underbrace{C(q, \dot{q}) \dot{q} + g(q)}_{n(q, \dot{q})} = \tau \Rightarrow B \ddot{q} + n = \tau.$$

Let the control action be  $\tau = \hat{n} + \hat{B}u$ . Then,  $B \ddot{q} + n = \hat{n} + \hat{B}u$ .

If the estimates  $\hat{B}$  and  $\hat{n}$  are exact, the dynamics reduce to

$$\ddot{q} = u.$$

The dynamics can now be controlled linearly, through

$$\ddot{e} + k_d \dot{e} + k_p e = 0, \text{ with } e = q - q_d.$$

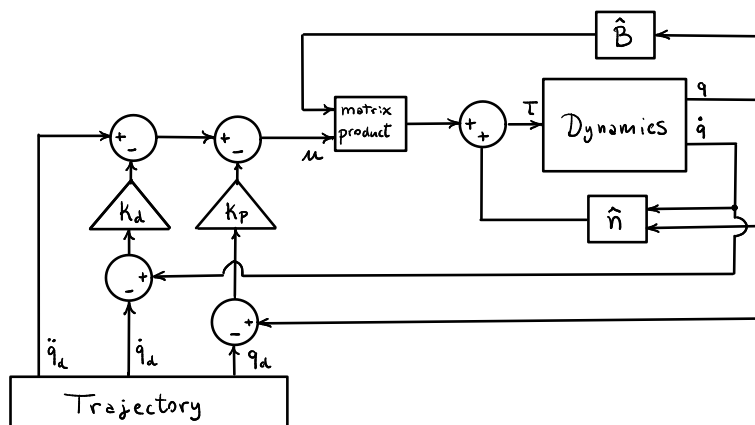
In which case, the input,  $u$ , must be

$$u = \ddot{q} = \ddot{q}_d - k_d(\dot{q} - \dot{q}_d) - k_p(q - q_d).$$

Here,  $q \in \mathbb{R}^n$ ,  $k_d \in \mathbb{R}^{n \times n}$  and  $k_p \in \mathbb{R}^{n \times n}$ .

- Example (11<sup>th</sup> Set of Problems, P1)

In block diagram form, the previous controller takes the following form.



For a 2 degrees of freedom manipulator ( $n=2$ ), the stiffness and damping matrices may be defined as

$$K_p = \begin{pmatrix} k_{p1} & 0 \\ 0 & k_{p2} \end{pmatrix}, \quad K_d = \begin{pmatrix} k_{d1} & 0 \\ 0 & k_{d2} \end{pmatrix}, \quad e = \begin{pmatrix} e_1 \\ e_2 \end{pmatrix} = \begin{pmatrix} q_1 - q_{1d} \\ q_2 - q_{2d} \end{pmatrix}.$$

The resulting linear system is then

$$\ddot{e}_i + k_{d,i} \dot{e}_i + k_{p,i} e_i = 0 \quad , \quad \text{for } i=1,2.$$

Taking its Laplace transform reveals  $E_i(s) (s^2 + k_{d,i} s + k_{p,i}) = 0$ , which corresponds to the dynamics of a **Damped Harmonic Oscillator**.

This can also be written in terms of **frequency** and **damping ratio**, as

$$E_i(s) (s^2 + 2\omega_{n,i} \xi_i s + \omega_{n,i}^2) = 0. \text{ Thus,}$$

$$K_p = \begin{pmatrix} \omega_{n1}^2 & 0 \\ 0 & \omega_{n2}^2 \end{pmatrix}, \quad K_d = \begin{pmatrix} 2\omega_{n1} \xi_1 & 0 \\ 0 & 2\omega_{n2} \xi_2 \end{pmatrix},$$

and

$$\tau = \hat{n} + \hat{B} (\ddot{q}_d - K_d (\dot{q} - \dot{q}_d) - K_p (q - q_d)).$$

### Decentralized Control (see the 10<sup>th</sup> set of Problems)

In the decentralized control the joints are controlled independently.

The coupling between the joints motion is treated as a disturbance, together with the variations in the mass distribution. The approach can be summarized through approximation of the manipulator's dynamics, taking here the form of a linear system, as follows.

$$B_{\max} \ddot{q} \approx \tau, \quad \text{where } B_{\max} = \text{diag}(\max(B_{11}) \dots \max(B_{nn})).$$

The independent control of the joints can be achieved through a **P-D controller**, defined as

$$\tau = -K_d \dot{q} - K_p (q - q_d), \quad \text{with } K_d = \begin{pmatrix} k_{d1} & & 0 \\ & \ddots & \\ 0 & & k_{dn} \end{pmatrix}, \quad K_p = \begin{pmatrix} k_{p1} & & 0 \\ & \ddots & \\ 0 & & k_{pn} \end{pmatrix}.$$

The closed loop becomes

$$B_{\max} \ddot{q} + K_d \dot{q} + K_p q \approx K_p q_d,$$

or, in the frequency domain,

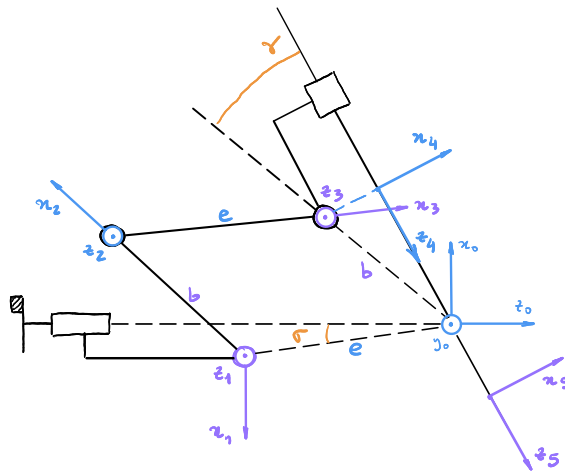
$$\left( s^2 + \frac{K_{d,i}}{B_{\max,ii}} s + \frac{K_{p,i}}{B_{\max,ii}} \right) Q_i(s) \approx \frac{K_{p,i}}{B_{\max,ii}} Q_{d,i}(s), \quad i=1, \dots, n.$$

The stiffness and damping are therefore related to the frequency and damping ratio through

$$k_{p,i} = B_{\max,ii} \omega_{n,i}^2, \quad k_{d,i} = 2 B_{\max,ii} \xi_i \omega_{n,i}.$$

# De Vinci's Equations of Motion

Recall the kinematics of the De Vinci Xi manipulator.



$d_i$	$\theta_i$	$a_i$	$\alpha_i$
$-e \cos(\sigma)$	$\theta_1' + \pi$	$e \sin(\sigma)$	$\pi/2$
0	$\theta_2' + \pi + \gamma$	$b$	0
0	$-\theta_2' - \gamma + \sigma - \frac{\pi}{2}$	$e$	0
0	$\theta_2' - \sigma$	$b \sin(\gamma)$	$\pi/2$
$d_5' + b \cos \gamma$	0	0	0

$$q = (\theta_1' + \pi, \theta_2' + \pi + \gamma, -\theta_2' - \gamma + \sigma - \frac{\pi}{2}, \theta_2' - \sigma, d_5' + b \cos \gamma)^T$$

$$\ddot{q} = (\ddot{\theta}_1', \ddot{\theta}_2', -\ddot{\theta}_2', \ddot{\theta}_2', \ddot{d}_5')^T$$

$$\bar{q} = (\theta_1' + \pi, \theta_2' + \pi + \gamma, d_5' + b \cos \gamma)^T$$

$$\ddot{\bar{q}} = (\ddot{\theta}_1', \ddot{\theta}_2', \ddot{d}_5')^T$$

Its dynamics may be obtained through Newton-Euler's recursive algorithm,

$$\tau = \begin{pmatrix} \tau_1 \\ \tau_2 \\ \tau_3 \\ \tau_4 \\ \tau_5 \end{pmatrix} = \text{tau-NewtonEuler}(\text{DH}, M, \dot{q}, \ddot{q}, \omega_0^o, \dot{\omega}_0^o, \ddot{p}_0^o(1g1)) .$$

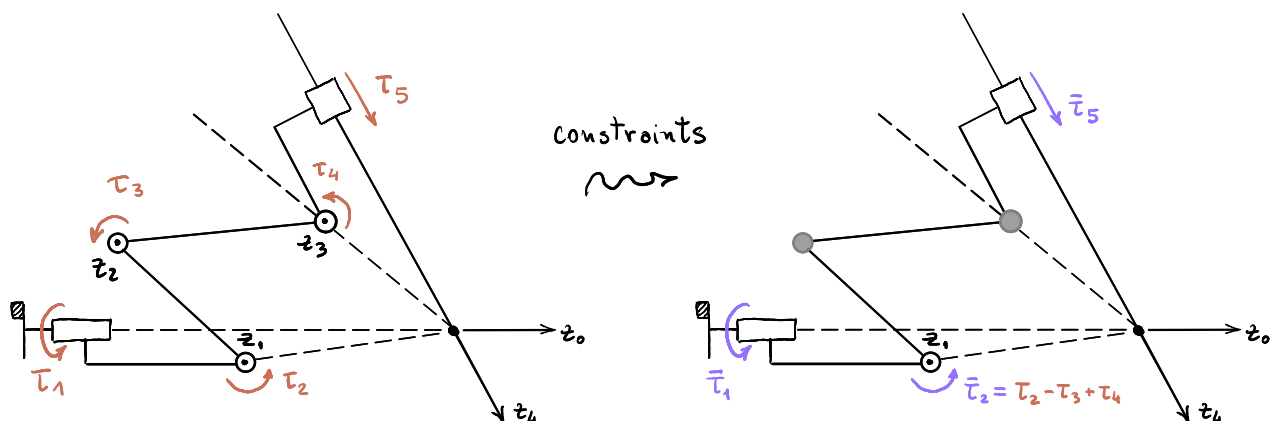
This allows to obtain the equations of motion in the form of

$$B \ddot{\bar{q}} + n = \tau, \text{ with } n = C \dot{\bar{q}} + g .$$

Because of the kinematic constraint, we find that the mass matrix is not square,

$$B(\bar{q}) = \frac{\partial \tau}{\partial \ddot{\bar{q}}} \in \mathbb{R}^{5 \times 3} .$$

An additional step is therefore required, in order to complete the transition between the  $q$  and  $\bar{q}$  spaces. This transformation step can be found by noting that the input actions  $\tau_3$  and  $\tau_4$  have been relinquished. Thus, a corresponding input space  $\bar{\tau} \in \mathbb{R}^3$  must also be found.



Suppose an input  $\tau = (\tau_1 \tau_2 \tau_3 \tau_4 \tau_5)^T$  is applied to the joints of the unconstrained manipulator (space  $q$ ), as illustrated. Then, after the constraint is applied, the inputs  $\tau_1$  and  $\tau_5$  remain unchanged in the constrained input space, termed  $\bar{\tau} = (\bar{\tau}_1 \bar{\tau}_2 \bar{\tau}_5)^T$ . In other words,  $\tau_1 = \bar{\tau}_1$  and  $\tau_5 = \bar{\tau}_5$ . Now, because the motion in  $\theta_4$  is coupled in the same direction of  $\theta_2$ , and  $\theta_3$  is coupled in the opposite direction, it must be true that  $\bar{\tau}_2 = \tau_2 - \tau_3 + \tau_4$ .

The previous description can be written in matrix form, as

$$\begin{pmatrix} \bar{\tau}_1 \\ \bar{\tau}_2 \\ \bar{\tau}_5 \end{pmatrix} = \begin{pmatrix} 1 & 0 & 0 & 0 & 0 \\ 0 & 1 & -1 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} \tau_1 \\ \tau_2 \\ \tau_3 \\ \tau_4 \\ \tau_5 \end{pmatrix} \Leftrightarrow \bar{\tau} = S^T \tau.$$

Thus, the equations of motion are obtained through

$$S^T B \ddot{q} + S^T n = S^T \tau \Rightarrow \bar{B} \ddot{q} + \bar{n} = \bar{\tau}.$$

This result is equivalent to the application of **D'Alembert's principle**. Through which it can be found that the generalized forces can be transformed between two spaces of coordinates  $\alpha$  and  $\beta$  as  $\xi_\alpha = \left(\frac{\partial \beta}{\partial \alpha}\right)^T \xi_\beta$ . Thus, in the present example,  $\bar{\tau} = \left(\frac{\partial q}{\partial \bar{q}}\right)^T \tau$  and  $S = \frac{\partial q}{\partial \bar{q}} = \frac{\partial \dot{q}}{\partial \dot{\bar{q}}}$ .

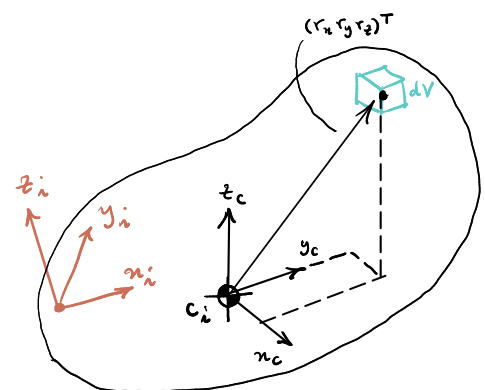
## Inertia Tensor

Consider a rigid-body termed **link  $i$**  (or body  $i$ ). Then, its equations of motion, in **decoupled form**, may be written as

$$\begin{cases} m_i \ddot{p}_{c_i} = \sum f, \\ I_i \dot{\omega}_i + \omega_i \times (I_i \omega_i) = \sum \tau. \end{cases}$$

Here,  $I_i$  is the inertia tensor relative to the center of

mass of link  $i$ . Its coordinates are here expressed in the inertial frame,  $I_i = I_i^0$ .



This inertia tensor can be evaluated around a body frame fixed to the mass center, as follows.

$$\mathbf{I}_i^c = \begin{pmatrix} \int (r_y^c)^2 + (r_z^c)^2 \rho dV & - \int r_x^c r_y^c \rho dV & - \int r_x^c r_z^c \rho dV \\ \text{sym.} & \int (r_x^c)^2 + (r_z^c)^2 \rho dV & - \int r_y^c r_z^c \rho dV \\ & & \int (r_y^c)^2 + (r_x^c)^2 \rho dV \end{pmatrix}.$$

The coordinates of this tensor can be expressed in the inertial frame through

$$\mathbf{I}_i = \mathbf{R}_c \mathbf{I}_i^c \mathbf{R}_c^T, \text{ or in any body-fixed frame through}$$

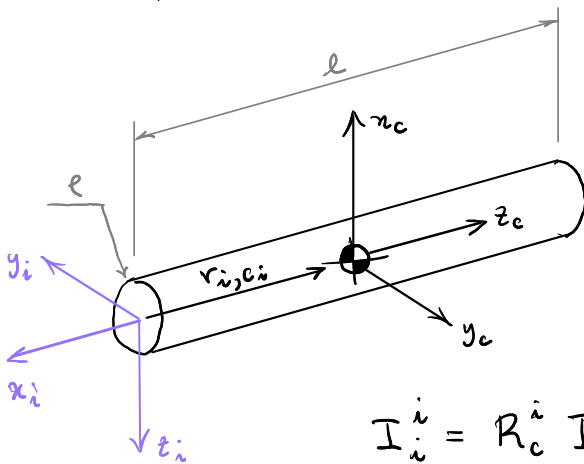
$$\mathbf{I}_i^{\hat{i}} = \mathbf{R}_c^{\hat{i}} \mathbf{I}_i^c (\mathbf{R}_c^{\hat{i}})^T.$$

Note that this transformation does not change the fact that the inertia tensor is still expressed **relative to the center of mass** of the link.

For complex shapes,  $\mathbf{I}_i^c$  must be evaluated numerically (solidworks, meshlab, etc.).

For simple shapes,  $\mathbf{I}_i^c$  can usually be found in tables, as follows.

#### • Example - Cylinder



$$\mathbf{I}_i^c = \begin{pmatrix} \frac{1}{12} m_i (3r_i^2 + l^2) & 0 & 0 \\ 0 & \frac{1}{12} m_i (3r_i^2 + l^2) & 0 \\ 0 & 0 & \frac{1}{2} m_i r_i^2 \end{pmatrix}$$

$$\mathbf{R}_c^{\hat{i}} = (\hat{x}_c \ \hat{y}_c \ \hat{z}_c) = \begin{pmatrix} 0 & 0 & -1 \\ 0 & -1 & 0 \\ -1 & 0 & 0 \end{pmatrix}$$

$$\mathbf{I}_i^{\hat{i}} = \mathbf{R}_c^{\hat{i}} \mathbf{I}_i^c (\mathbf{R}_c^{\hat{i}})^T = \begin{pmatrix} \frac{1}{2} m_i r_i^2 & 0 & 0 \\ 0 & \frac{1}{12} m_i (3r_i^2 + l^2) & 0 \\ 0 & 0 & \frac{1}{12} m_i (3r_i^2 + l^2) \end{pmatrix}$$