control



Robótica Móvel e Inteligente / Mobile and Intelligent Robotics

Academic year 2023-24

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Control systems



 Objective: to impose a given value of some physical quantity in a system by acting on some other physical quantity

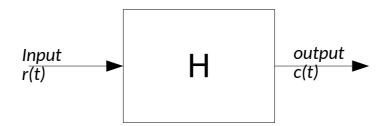


http://blog.caranddriver.com/nissan-develops-fully-electric-steerby-wire-system-will-go-on-sale-next-year/

basic concepts



- Systems approach:
 - Input signal
 - Output signal
 - Process, transforming input into output
- Objective: to impose a given value at a system's output, by acting in its input



• Example:

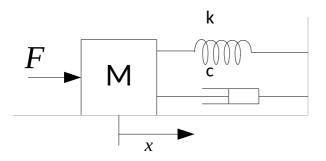
accelerator car speed

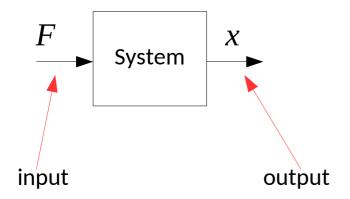
Input / output relationship



General case:

- input r(t) and output c(t) are related by differential equations
 - this is the "default" in physical systems...





Mathematical relation between input and output

$$F(t) = M a(t) + c v(t) + k x(t)$$

$$F(t) = M \frac{\mathrm{d}^2 x(t)}{\mathrm{d}t^2} + c \frac{\mathrm{d}x(t)}{\mathrm{d}t} + k x(t)$$

Input / output relationship



General case:

- input r(t) and output c(t) are related by differential equations

$$a_{n}\frac{d^{n}c(t)}{dt^{n}} + a_{n-1}\frac{d^{n-1}c(t)}{dt^{n-1}} + \dots + a_{0}c(t) = b_{m}\frac{d^{m}r(t)}{dt^{m}} + b_{m-1}\frac{d^{m-1}r(t)}{dt^{m-1}} + \dots + b_{0}r(t)$$

Combination of c(t) and its derivatives

Combination of r(t) and its derivatives

Difficult to solve and convert to a systems perspective

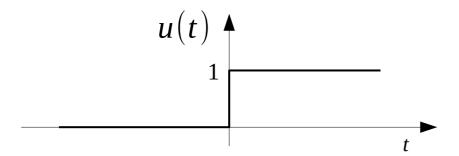




 Differential equations are simplified by the use of Laplace transforms.

$$L\{f(t)\} = F(s) = \int_{0}^{+\infty} e^{-st} f(t) dt$$

$$L^{-1}{F(s)} = \frac{1}{2\pi j} \int_{\sigma-j\infty}^{\sigma+j\infty} e^{st} F(s) dt = f(t) \cdot u(t)$$



u(t) is the unit step function. We only consider the function f(t) to have non-null values for t>0.

Computing the Laplace transform



Example for u(t)

$$L[u(t)] = \int_{0}^{+\infty} e^{-st} u(t) dt$$

$$= \int_{0}^{+\infty} e^{-st} dt$$

$$= \left[-\frac{1}{s} e^{-st} \right]_{0}^{+\infty}$$

$$= 0 - \left(-\frac{1}{s} \right)$$

$$= \frac{1}{s}$$



TABLE 2.1 Laplace transform table

Item no.	f(t)	F(s)
1.	$\delta(t)$	1
2.	u(t)	$\frac{1}{s}$
3.	tu(t)	$\frac{1}{s^2}$
4.	$t^n u(t)$	$\frac{n!}{s^n+1}$
5.	$e^{-at}u(t)$	$\frac{1}{s+a}$
6.	$\sin \omega t u(t)$	$\frac{\omega}{s^2 + \omega}$
7.	$\cos \omega t u(t)$	$\frac{s}{s^2 + \omega}$



Laplace transform theorems

	Theorem	Name
$\mathscr{L}[f(t)] = F(s)$	$\int_{0-}^{\infty} f(t)e^{-st}dt$	Definition
$\mathscr{L}[kf(t)]$	=kF(s)	Linearity theorem
$\mathcal{L}[f_1(t) + f_2(t)]$	$[F] = F_1(s) + F_2(s)$	Linearity theorem
$\mathcal{L}[e^{-at}f(t)]$	=F(s+a)	Frequency shift theorem
$\mathscr{L}[f(t-T)]$	$=e^{-sT}F(s)$	Time shift theorem
$\mathcal{L}[f(at)]$	$=\frac{1}{a}F\left(\frac{s}{a}\right)$	Scaling theorem
$\mathscr{L}\left[\frac{df}{dt}\right]$	= sF(s) - f(0-)	Differentiation theorem
$\mathscr{L}\left[\frac{d^2f}{dt^2}\right]$	$= s^2 F(s) - sf(0-) - f'(0-)$	Differentiation theorem
$\mathscr{L}\left[\frac{d^nf}{dt^n}\right]$	$= s^{n}F(s) - \sum_{k=1}^{n} s^{n-k}f^{k-1}(0-)$	Differentiation theorem
$\mathscr{L}ig[\int_{0-}^t f(au)d auig]$	$=\frac{F(s)}{s}$	Integration theorem
$f(\infty)$	$=\lim_{s\to 0} sF(s)$	Final value theorem ¹
f(0+)	$=\lim_{s\to\infty} sF(s)$	Initial value theorem ²



$$\left(a_{n}\frac{d^{n}c(t)}{dt^{n}}\right)+a_{n-1}\frac{d^{n-1}c(t)}{dt^{n-1}}+\ldots+a_{0}c(t)=b_{m}\frac{d^{m}r(t)}{dt^{m}}+b_{m-1}\frac{d^{m-1}r(t)}{dt^{m-1}}+\ldots+b_{0}r(t)$$

Computing the Laplace transform

$$L\left\{a_n \frac{d^n c(t)}{dt^n}\right\} = a_n L\left\{\frac{d^n c(t)}{dt^n}\right\}$$
 Linearity
$$= a_n s^n L\{c(t)\}$$
 Differentiation theorem
$$= a_n s^n C(s)$$
 Definition of Laplace transform



$$a_{n} \frac{d^{n} c(t)}{dt^{n}} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \dots + a_{0} c(t) = b_{m} \frac{d^{m} r(t)}{dt^{m}} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \dots + b_{0} r(t)$$

$$a_n s^n C(s) + a_{n-1} s^{n-1} C(s) + \dots + a_0 C(s) = b_m s^m R(s) + b_{m-1} s^{m-1} R(s) + \dots + b_0 R(s)$$

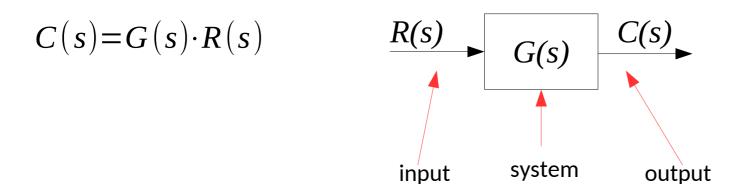
$$(a_n s^n + a_{n-1} s^{n-1} + \dots + a_0) C(s) = (b_m s^m + b_{m-1} s^{m-1} + \dots + b_0) R(s)$$

$$\frac{C(s)}{R(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \dots + a_0} = G(s)$$

Transfer function

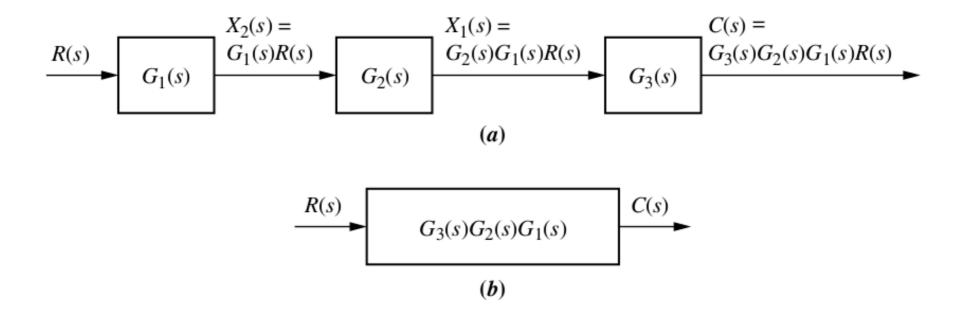


$$\frac{C(s)}{R(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \dots + a_0} = G(s)$$
Transfer function



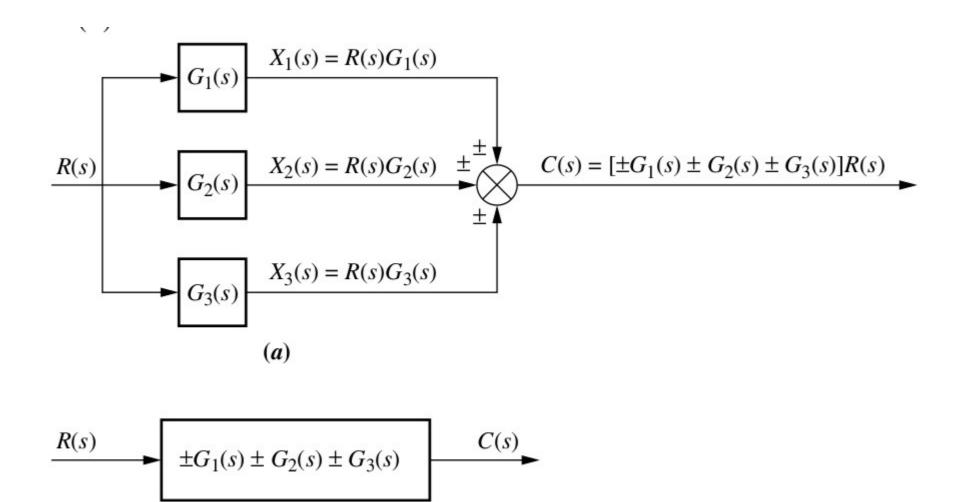
- A relation expressed originally in terms of a differential equation is expressed as a product
- the physical nature of input/output relationship is irrelevant; only mathematical relationship matters --> abstraction



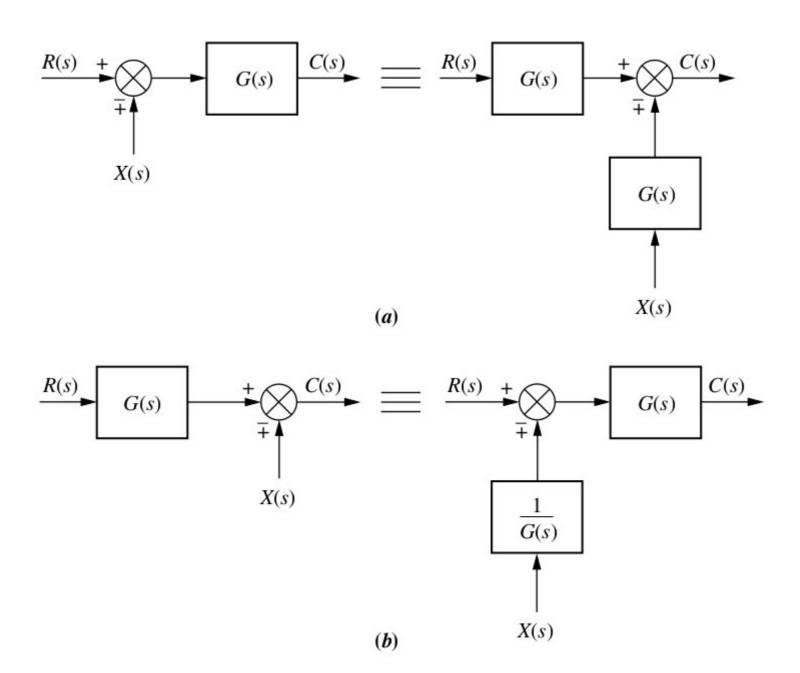


(b)

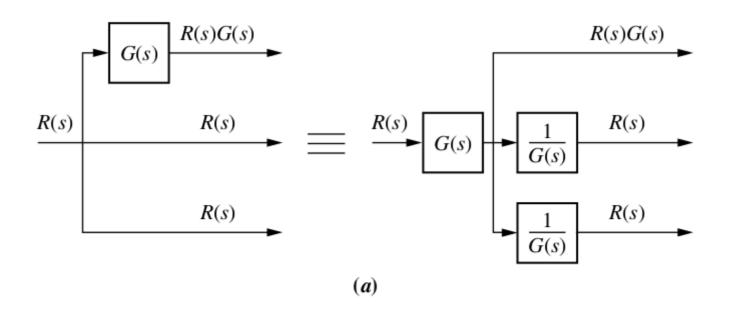


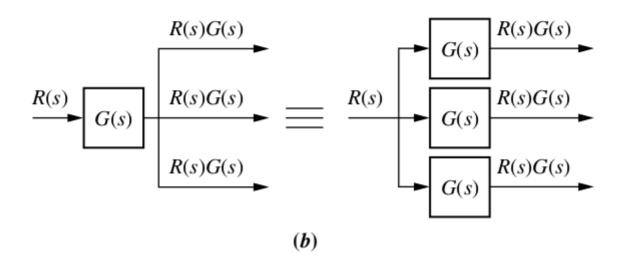






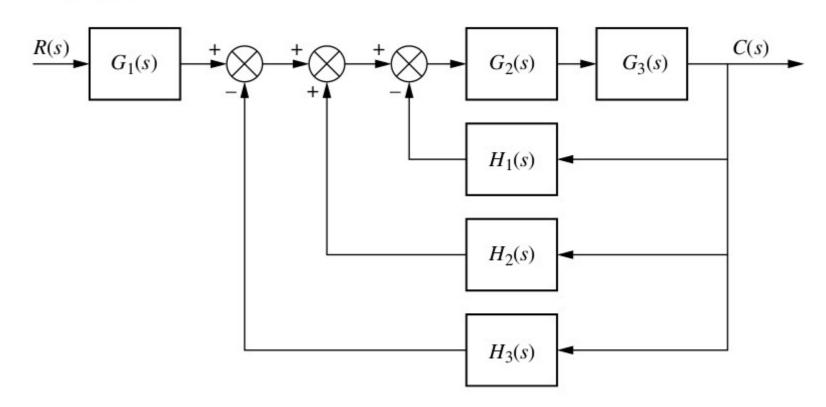








PROBLEM: Reduce the block diagram shown in Figure 5.9 to a single transfer function.



Test waveforms

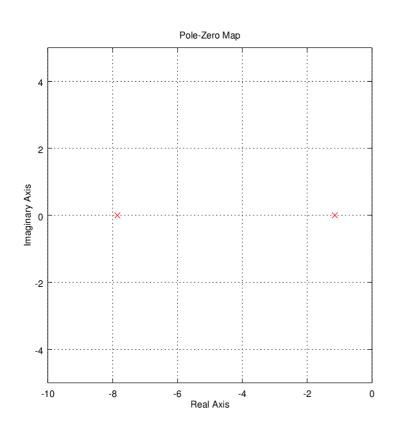


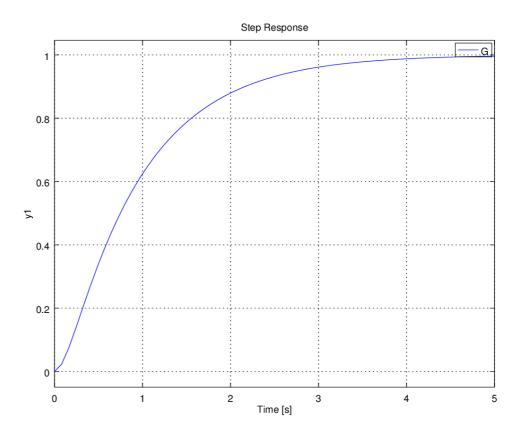
Function	Description	Sketch	Use
Impulse $\delta(t)$	$\delta(t) = \infty \text{ for } 0 - < t < 0 +$ = 0 elsewhere	f(t)	Transient response Modeling
	$\int_{0-}^{0+} \delta(t) dt = 1$	δ(t)	
u(t)	u(t) = 1 for t > 0 $= 0 for t < 0$	f(t)	Transient response Steady-state error
t.(0)	$tu(t) = t \text{ for } t \ge 0$		Standy atota array
u(t)	= 0 elsewhere		Steady-state error
$\frac{1}{2}t^2u(t)$	$\frac{1}{2}t^2u(t) = \frac{1}{2}t^2 \text{ for } t \ge 0$ $= 0 \text{ elsewhere}$	f(t)	Steady-state error
sin ωt		1	Transient response
			Modeling Steady-state error
	$\delta(t)$ $u(t)$ $tu(t)$	$\delta(t) = \infty \text{ for } 0 - \langle t \langle 0 + \\ = 0 \text{ elsewhere}$ $\int_{0-}^{0+} \delta(t) dt = 1$ $u(t) \qquad u(t) = 1 \text{ for } t > 0 \\ = 0 \text{ for } t < 0$ $tu(t) \qquad tu(t) = t \text{ for } t \geq 0 \\ = 0 \text{ elsewhere}$ $\frac{1}{2}t^{2}u(t) \qquad \frac{1}{2}t^{2}u(t) = \frac{1}{2}t^{2} \text{ for } t \geq 0 \\ = 0 \text{ elsewhere}$	$\delta(t) \qquad \delta(t) = \infty \text{ for } 0 - \langle t < 0 + t \rangle = 0 \text{ elsewhere}$ $\int_{0^{-}}^{0^{+}} \delta(t) dt = 1$ $u(t) \qquad u(t) = 1 \text{ for } t > 0$ $= 0 \text{ for } t < 0$ $tu(t) \qquad tu(t) = t \text{ for } t \geq 0$ $= 0 \text{ elsewhere}$ $\int_{0^{-}}^{0^{+}} \delta(t) dt = 1$



$$r(t)=u(t)$$
 $G_1(s)=\frac{9}{s^2+9s+9}$

Overdamped

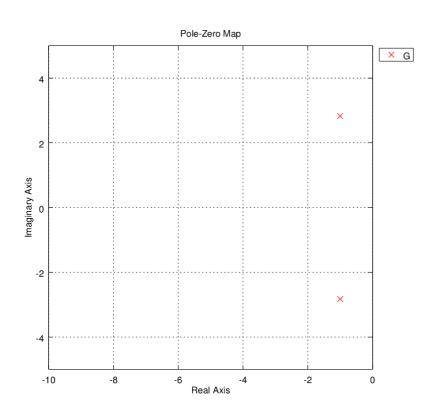


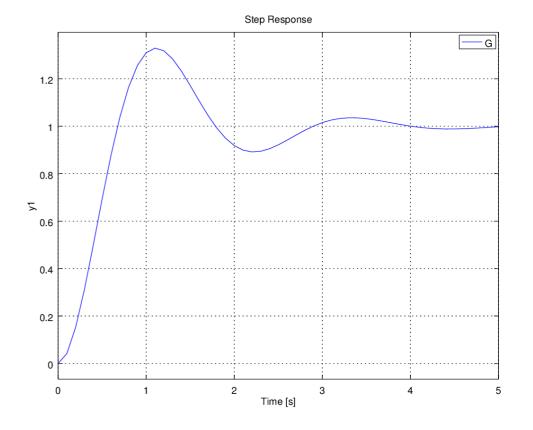




$$r(t)=u(t)$$
 $G_2(s)=\frac{9}{s^2+2s+9}$

Underdamped

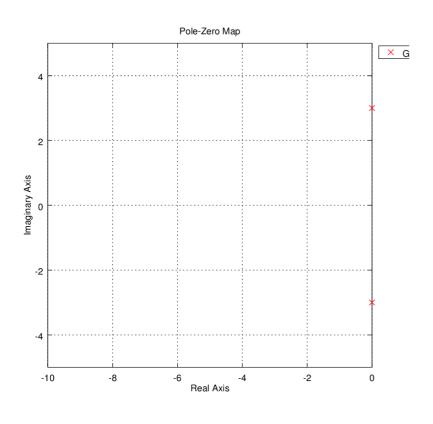


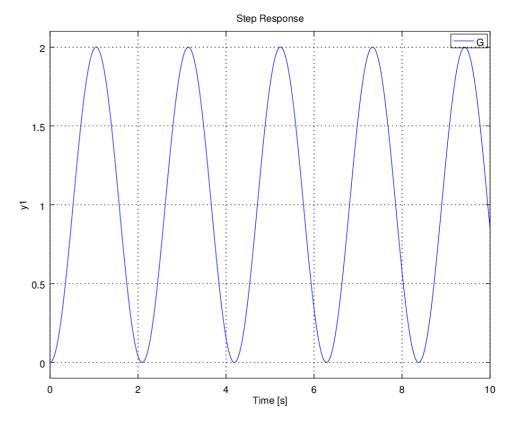




$$r(t)=u(t)$$
 $G_3(s)=\frac{9}{s^2+9}$

Undamped

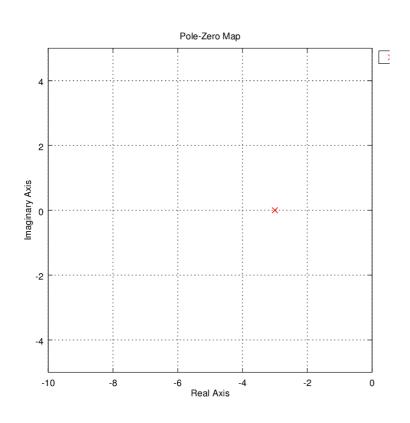


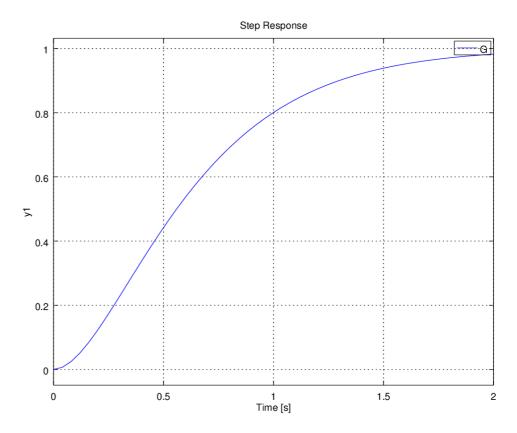




$$r(t)=u(t)$$
 $G_4(s)=\frac{9}{s^2+6s+9}$

Critically damped





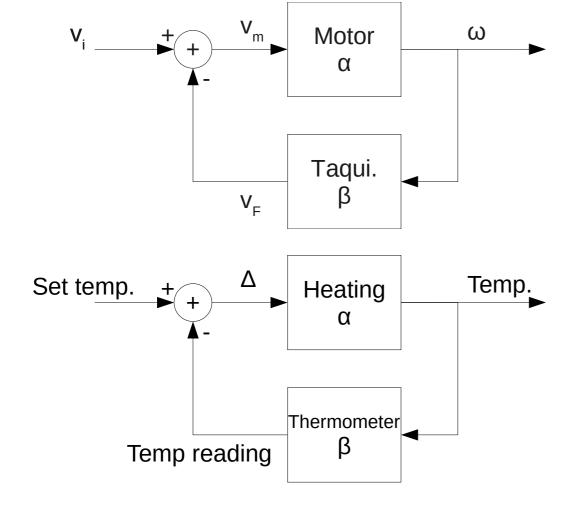
Video



feedback

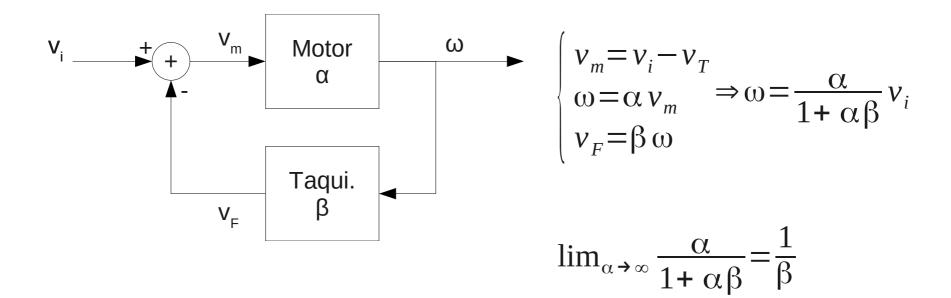


- a complex system is represented as a collection of interconnected set of simpler systems
 - each simple system has a known transfer function



feedback



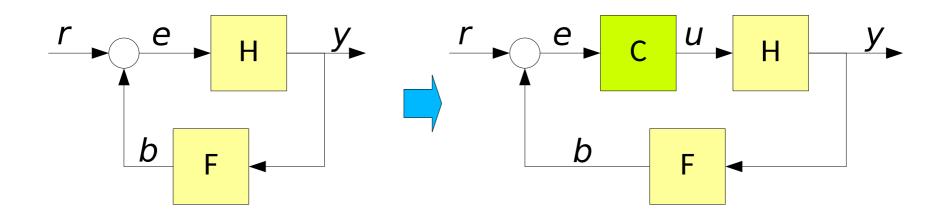


• for high values of α , the output value will depend mainly on the feedback

controller



•Controller C: included to improve the system response



r: reference (input)

e: error

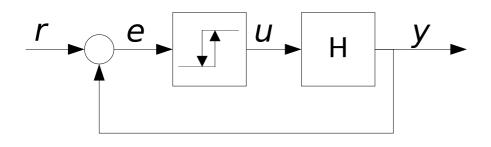
u: system input / control

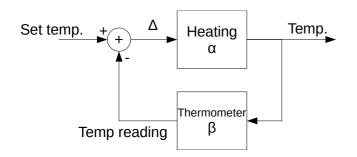
y: system output

b: feedback

On-Off (Bang-bang)







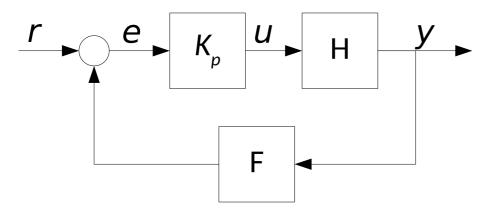
- Acting signal u goes on and off depending on error
- Example:
 - home temperature control
- May have hysteresis
- Usually not so good behaviour, but...
- Easy to implement (switch on and off...)

P controller



• P = "proportional". Simplest form of linear controller $u = K_p e$

• Gain: K_p



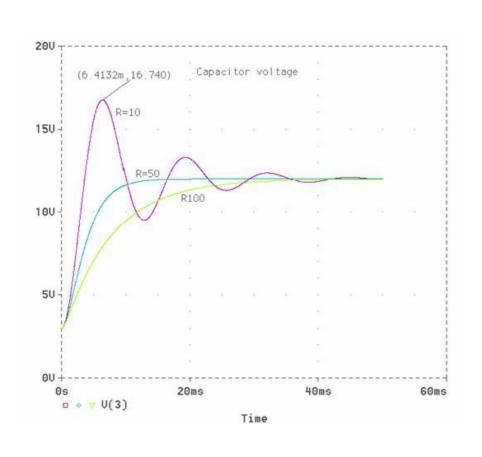
The larger K_p , the smaller e for the same output.

- Kp low: soft system
 - it takes a large value of error for system to react
- Kp high: hard system
 - strong reaction, even with small values of *e*.

 $u\neq 0$ iif $e\neq 0$ To reduce error, a high value of Kp is required

P controller



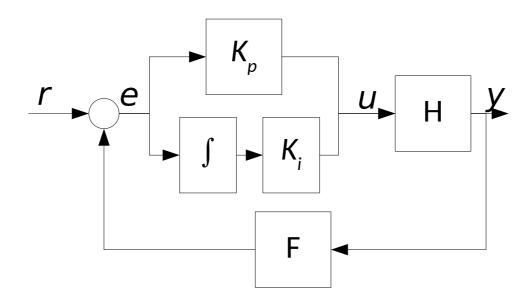


- Increasing gain Kp reduces error, but...
- High values of gain Kp may cause the system to be unstable

PI controller



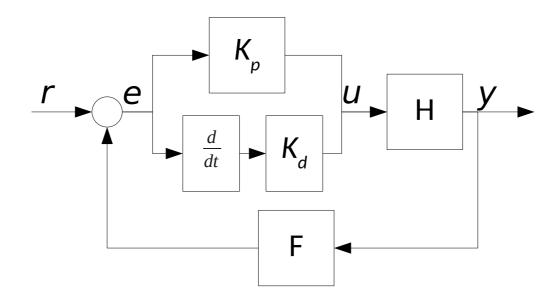
- PI: Proportional + Integral
 - Adding a term $K_i \int e \, dt$.
 - PI controller allows for systems with e=0
 - Problem: inertia (memory effect)
 - with rapid changes in the input, u may be at a value when the error e would require to be different



PD controller



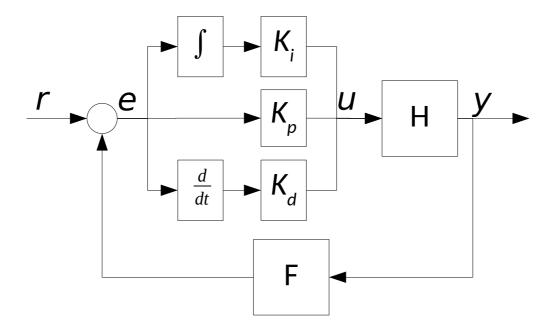
- PD = Proportional + Derivative
 - adding a term proportional to the error derivative
 - In general, it has the effect of reducing oscillations (damper)

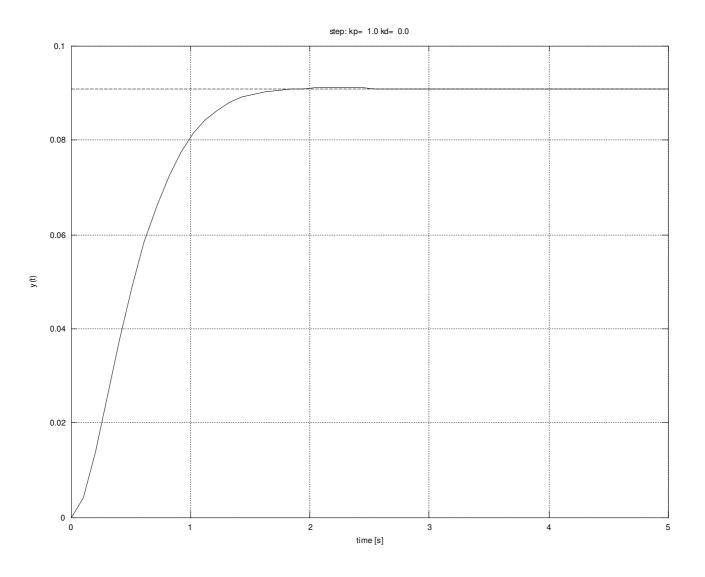


PID controller



- PID = Proportional + Integral + Derivative
 - Reunion of previous controllers
 - One of the most popular controllers





Coding a PID controller



Controller demo

BANGH



NONE P PID BANG, BANG2,

> git@github.com:iris-ua/ciberRatoTools.git branch: speedc 38