

Flash Card

Definitions

Rise time (t_r) : It is the time taken by the step response to rise from 10% to 90% of final value.

$$t_r = \frac{1.8}{\omega_n}$$

Peak Overshoot (M_p) : Largest deviation of step response above the step input.

$$M_p = e^{\frac{-\pi \epsilon}{\sqrt{1-\epsilon^2}}}$$

Peak time (t_p) : The time at which peak overshoot occurs.

$$t_p = \frac{\pi}{\omega_d}$$

Settling time (t_s) : The time taken by the step response to rise and stay at 1% tolerance.

$$t_s = \frac{4.6}{\epsilon \omega_n}$$

Poles of a second order system:

$$S = -\sigma \pm j \omega_d$$

$$\sigma = \epsilon \omega_n$$
$$\omega_d = \omega_n \sqrt{1 - \epsilon^2}$$

Standard form of systems

1st order:

$$\frac{Y(S)}{R(S)} = \frac{1}{S + \sigma}$$

2nd Order:

$$\frac{Y(S)}{R(S)} = \frac{\omega_n^2}{S^2 + 2\epsilon\omega S + 1}$$

Derivation: For the derivation of the above parameters:

- $y(t) = 1 - e^{-\sigma t} \cos \omega_d t - \frac{\sigma}{\omega_d} \sin \omega_d t$
- $a \cos \alpha + b \sin \alpha = \sqrt{a^2 + b^2} \cos(\alpha - \beta)$ or $\frac{1}{\sqrt{a^2 + b^2}} \sin(\alpha + \beta)$
 $\beta = \tan^{-1} \left(\frac{b}{a} \right)$
- t_p :
 - $\frac{dy(t)}{dt} = 0$
 - Equate $\omega_d t_p = m\pi$
- % Overshoot:
 - $M_p = \frac{y_{max} - y_{ss}}{y_{ss}}$; $y_{ss} = 1$ and $y_{max} = y(t_p)$
- t_s :
 - $e^{-\sigma t}$ is called damping coefficient. Equate this to the requirement.
 - Consider damping ratio to be negligible
 - Consider damping frequency to be equal to 1

Note: $\sigma = \text{damping ratio}$

$\omega_d = \text{damping frequency}$

Root Locus

- Find the no. of poles and zeros.
- Find the region where the root locus exists.
- Find the centroid $\sigma = \frac{\sum Re(poles) - \sum Re(zeroes)}{P - Z}$
- Find the angle of asymptotes $\theta_q = \frac{(2q+1)180}{P-Z}$

- Find the break-away or break-in point/s using CE: $1 + kL(s) = 0$ and $\frac{dK}{ds} = 0$, for **TWO ADJACENT POLES OR ZEROS**
- Find the intersection with imaginary axis using RH criteria using the CE equation.
- Find the angle of departure = $180 - \theta_p + \theta_z$ or the angle of arrival = $180 - \theta_z + \theta_p$ in the case of **COMPLEX ZEROS**.

Types of systems and its stability:

Type 0: No poles at $s = 0$

Type 1: 1 pole at $s = 0$

Type 2: 2 poles at $s = 0$

D → Controller gain

G → Plant gain

H → Feedback gain

$$K_p = \lim_{s \rightarrow 0} DGH; \text{ Step : } e_{ss} = \frac{1}{1 + K_p}$$

$$K_v = \lim_{s \rightarrow 0} SDGH; \text{ Ramp : } e_{ss} = \frac{1}{K_v}$$

$$K_u = \lim_{s \rightarrow 0} S^2 DGH; \text{ Parabola : } e_{ss} = \frac{1}{K_u}$$

	Step	Ramp	Parabola
Type 0	$\frac{1}{1+K_p}$	∞	∞
Type 1	0	$\frac{1}{K_v}$	∞
Type 2	0	0	$\frac{1}{K_u}$

Design of Controllers

Controller Signal: It is obtained by multiplying the error signal with a constant K_p

Closed Loop Transfer Function:

$$T(s) = \frac{GD}{1 + GDH}$$

Open Loop Transfer Function:

$$T(s) = GDH$$

1. Proportional Controller:

- $D = K_P$
- Find the characteristic equation $CE : 1 + DGH = 0$
- Compare with standard second order equation $s^2 + 2 \zeta \omega_n s + \omega_n^2 = 0$
- Find the required values.
- Find the error constant e_{ss}

2. Proportional Integral Controller:

- $D = K_P + \frac{K_I}{s}$
- Follow the procedure given above, but just change the value of D.

3. Proportional Integrator Differentiator Controller:

- $D = K_P + \frac{K_I}{s} + s K_U$
- Follow the procedure given above, but just change the value of D.

NOTE: If the order of the system is ≥ 2 then use R-H criteria to find the value of K, using the **characteristic equation**.

Design of **Lead Compensator** using root locus:

- angle of poles - angle of zeros = 180

- Calculate the angle of poles and zeros using the requirements given in the question
- $\theta = \cos^{-1}(\epsilon)$
- $r = \frac{1}{2}[180 - \theta - \phi]$
- $Z = \frac{\omega_n \sin r}{\sin(\theta + \phi)}$
- $P = \frac{\omega_n \sin(r + \phi)}{\sin(r + \theta + \phi)}$
- $D(s) = \frac{s + z}{s + p}$

State Space Analysis: The state of a system which is defined as minimum no. of interconnection that must be specified at any initial time t_c so that the complete dynamic behavior of a system at any time $t > t_o$ is determined when the i/p $u(t)$ is known

$$\frac{y(s)}{u(s)} = C [SI - A]^{-1} B + D$$

$$\phi(s) = [SI - A]^{-1}$$

$$x_n(t) = L^{-1}[\phi(s)X(0)]$$

$$x_f(t) = L^{-1}[\phi(s)BU(s)]$$

$$x(t) = x_n(t) + x_f(t)$$

Canonical Forms

Controllable Form:

$$\begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_0 & -a_1 & -a_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} [U]$$

$$y = [b'_0 \quad b'_1 \quad b'_2] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + b_3 U$$

- To **check** if a system is controllable:
 - Make sure the matrices are in **controller canonical form**.

$$Q = [B \quad AB \quad \dots \quad A^{n-1}B]$$

$|Q| \neq 0$, then the system is controllable

- To **design** a controller using pole-placement technique:
 - Compare the below two equations to get the required gains, for the given poles.

$$|\lambda I - (A - BK)| = 0$$

$$(\lambda - \lambda_1)(\lambda - \lambda_2) = 0$$

Observable Form:

$$\begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix} = \begin{bmatrix} -a_2 & 1 & 0 \\ -a_1 & 0 & 1 \\ -a_0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} b'_2 \\ b'_1 \\ b'_0 \end{bmatrix} [U]$$

$$y = [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + b_3 U$$

- To **check** if a system is observable:
 - Make sure the matrices are in **observer canonical form**.

$$S = [C^T \quad A^T C^T \quad \dots \quad (A^T)^{n-1} C^T]$$

$|s| \neq 0$, then the system is observable

- To **design** an observer using pole-placement technique:
 - Compare the below two equations to get the required gains, for the given poles.

$$|\lambda I - (A - LC)| = 0$$

$$(\lambda - \lambda_1)(\lambda - \lambda_2) = 0$$

Similarity Transformation:

$$\begin{aligned} A_w &= P^{-1}AP; B_w = P^{-1}B \\ C_w &= CP; D_w = D \end{aligned}$$

Bode Plot

- Plot the bode plot using the transfer function.
- Find ω_{gc} and the corresponding angle to ω_{gc}
- Find the required angle to obtain the given phase. Add 7-12 degrees to make it a whole number. This is θ_m

$$D(S) = \frac{\alpha(1 + \tau S)}{(1 + \alpha\tau S)}$$

$$\begin{aligned} \alpha &= \frac{1 - \sin\theta_m}{1 + \sin\theta_m} \\ \text{At } \omega_m, M_{db} &= 20\log(\sqrt{\alpha}) \\ \tau &= \frac{1}{\omega_m\sqrt{\alpha}} \end{aligned}$$

Mathematical modelling

Speaker:

$$\begin{aligned} F &= M\frac{d^2x}{dt^2} + b\frac{dx}{dt} \\ F &= BIL \\ L &= N\pi d \\ V &= Ri + L\frac{di}{dt} + e_{coil}; e_{coil} = Blv \end{aligned}$$

Cruise Control:

$$U = m \frac{d^2 x}{dt^2} + b \frac{dx}{dt}$$

DC motor:

$$e_a = i_a R_a + L_a \frac{di_a}{dt} + e_b$$

$$e_b = K_f \frac{d\theta}{dt}$$

$$T_a = k_t i_a = J \frac{d^2 \theta}{dt^2} + b \frac{d\theta}{dt}$$