## 2017-ME-27-39

## EE24BTECH11001 - ADITYA TRIPATHY

2) A parametric curve defined by  $x = \cos\left(\frac{\pi u}{2}\right)$ ,  $y = \sin\left(\frac{\pi u}{2}\right)$  in the range  $0 \le u \le 1$  is rotated about the *x*-axis by 360 degrees. Area of the surface generated is

3) P(0,3), Q(0.5,4), and R(1,5) are three points on the defined by f(x). Numerical integration is carried out using bothe trapezoidal rule and Simpson's rule within limits x = 0 and x = 1 for the

c)  $2\pi$ 

(2017 - ME)

(2017 - ME)

d)  $4\pi$ 

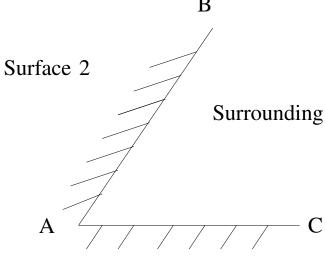
1) For the vector  $\mathbf{V} = 2yz\mathbf{i} + 3xz\mathbf{j} + 4xy\mathbf{k}$ , the value of  $\nabla \cdot (\nabla \times \mathbf{V})$  is

b) π

a)  $\frac{\pi}{2}$ 

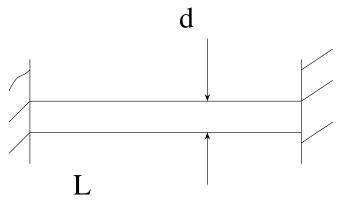
curve. The difference between the two results will be			(20	(2017 - ME)	
a) 0	b) 0.25	c) 0.5	d) 1		
4) The velocity prostream velocity the value of $\frac{\delta^*}{\delta}$	and $\delta$ is the local bounda	ayer flow for flow over a flar ry layer thickness, If $\delta^*$ is t	the local displacement	is the free thickness, $017 - ME$ )	
a) $\frac{2}{\pi}$	b) $1 - \frac{2}{\pi}$	c) $1 + \frac{2}{\pi}$	d) 0		
$d_1$ and $d_2$ arranged The friction factor known positive	ged in series. Both pipes tor for turbulent flow thor constants and <i>Re</i> is the R	ble fluid through two long are of equal length and the rught pipes is of the form $j$ eynolds number. Neglectin in pipe 2, $\frac{\Delta P_1}{\Delta P_2}$ , is given by	e flow is turbulent in b $f = K(Re)^{-n}$ , where $K$ g minor losses, the ra	ooth pipes. and <i>n</i> are	
a) $\left(\frac{d_2}{d_1}\right)^{(5-n)}$	b) $\left(\frac{d_2}{d_1}\right)^5$	c) $\left(\frac{d_2}{d_1}\right)^{(3-n)}$	d) $\left(\frac{d_2}{d_1}\right)^{(5+n)}$		
6) For a steady flo of a particle at	w, the velocity field is $\mathbf{V}$ $(1,-1)$ is	$= \left(-x^2 + 3z\right)\mathbf{i} + (2xy)\mathbf{j}. \text{ Th}$	ne magnitude of the ac	ecceleration 117 – <i>ME</i> )	
a) 2	b) 1	c) $2\sqrt{5}$	d) 0		
at 1 bar, and 30	00K is contained in a sea	400J/kg K; specific heat a led rigid cylinder. During The increase in entropy of	an adiabatic processs.		
	et temperature to the turb	air, specific heat at constaine is 1200 K and the isoe	ntropic efficiency is 0		
At a location, the	ne total pressure is $100kP$	of water vapor and dry air ( $a$ , the temperature is $30^{\circ}$ a er at $30^{\circ}$ is $42646Pa$ , the m	nd the relative humidi ass of water vapor per	ty is 55%.	

- 10) Air contains  $79\%N_2$  and  $21\%O_2$  on a molar basis. Methane  $(CH_4)$  is burned with 50% excess air than required stoichiometrically. Assuming complete combustion of methane, the molar percentage of  $N_2$  in the products is (2017 ME)
- 11) Two black surfaces, AB and BC, of lengths 5m amd 6m, respectively, are oriented as shown. Both surfaces extend infinitely into the third dimension. Given that view factor  $F_{12} = 0.5$ ,  $T_1 = 800K$ ,  $T_2 = 600K$ ,  $T_{surrounding} = 300K$  and Stefan Boltzman constant  $\sigma = 5.67 \times 10^{-8} W/(m^2 K^4)$  the heat transfer from Surface 2 to the surrounding environment is (inkW) (2017 ME)



## Surface 1

- 12) Heat is generated uniformly in a long solid cylindrical rod (diameter = 10mm) at the rate of  $4 \times 10^7 W/m^3$ . The thermal conductivity of the rod material is 25 W/mK. Under steady state conditions, the temperature difference between the centre and the surface of the rod is (in  $^{\circ}C$ ) (2017 ME)
- 13) An initially stress free massless elastic beam of length L and circular cross-section with diameter d (d << L) is held fixed between two walls as shown. The beam material has Young's modulus E and coefficient of thermal expansion  $\alpha$ . If the beam is slowly and uniformly heated, the temperatture rise required to cause the beam to buckle is proportional to



(2017 - ME)

a) *d* 

b)  $d^2$ 

c)  $d^3$ 

d)  $d^4$