**Results**

For this experiment, pressure measurements at two separate points (P1 and P2), compressor shaft RPM measurements (N), and force measurements (Scale Reading) were collected for three separate back plate configurations. The static pressure and temperature of the room were also recorded, for use in finding the density of the air in Oliver Hall. Each pressure measurement (besides the static pressure of Oliver Hall) was recorded in inches of water, with the former being a measurement of the pressure difference between the atmosphere and the larger of the two tube diameters within the compressor, and the latter being a measurement of the pressure difference between the smaller of the two tube diameters within the compressor and the first pressure reading.

With these values collected, the shaft horse power of the compressor was calculated through the use of Equation (1) below:

(1)

This value, whose units were in horse power, was then converted to W through the use of Equation (2) below:

(2)

Once this was completed, the pressure values, which were recorded in inches of water, were converted to Pascals through the use of Equation (3) below:

(3)

Where is the density of water (1000 kg/m3), h is the height of the water in meters, and g is the acceleration of gravity (9.81 m/s2).

With all of these values calculated, the flow rate through the compressor was then found. This was accomplished by assuming a different Reynold’s Number for each RPM increment in each trial. Based on this Reynold’s Number, a coefficient of drag was calculated for each trial based on Equation (4) below:

(4)

Where β is given as 0.5 and L2, the location of PT2 is 0.5. The results of this work are listed in Table (1) on the following page.

This value was then used to calculate the volume flow rate through the compressor, as described in Equation (5) below:

(5)

Where A2 is the smallest area within the compressor, which was found to be 0.041 m2.

The Power Output of the compressor (Po) was then found by multiplying Q by P2. With all of these values calculated, the coefficients associated with the flow through the compressor were then found through the use of Equations (6-9) below:

Power Coefficient:

Pressure Coefficient:

Flow Coefficient:

Reynold’s Number:

(6-9)

Where µ is the kinematic viscosity of air, for which a value of 1.983 x 10-5 kg/ms was used. Once the Re of each trial was found, the initial Re used to calculated CD was changed until the two Re’s were reasonably close for each case (Within 1000 of each other). Tables 4-1 through 4-3 list all of the data collected, used, and collected over the course of the experiment.

**Table 4-1. Shaft RPM, Force, Power, and Compressor Pressures**

|  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- |
| **Test No** | **N (RPM)** | **Back Plate Location (in)** | **Scale Reading** | **HP** | **PT1 (m)** | **PT2 (m)** |
| 1 | 631 | 3 | 0.5 | 0.078875 | 0.0089 | 0.0000 |
| 2 | 1204 | 3 | 1.75 | 0.52675 | 0.0356 | 0.0000 |
| 3 | 1823 | 3 | 3.9 | 1.777425 | 0.0978 | 0.0000 |
| 4 | 625 | 2 | 0.5 | 0.078125 | 0.0089 | 0.0025 |
| 5 | 1227 | 2 | 1.6 | 0.4908 | 0.0394 | 0.0127 |
| 6 | 1809 | 2 | 3.6 | 1.6281 | 0.0991 | 0.0330 |
| 7 | 625 | 1 | 0.1 | 0.015625 | 0.0089 | 0.0051 |
| 8 | 1286 | 1 | 1.3 | 0.41795 | 0.0432 | 0.0203 |
| 9 | 1812 | 1 | 2.5 | 1.1325 | 0.0914 | 0.0419 |