$\underline{\underline{L}} := 1  \text{ft}$ $\underline{\underline{W}} := 1  \text{ft}$	Dimensions of Platform	rev	RPM of input power
$m_{r} := 20lbm = 9.072 kg$	Mass of Robot	$\omega_{\mathbf{m}} := 3200 \frac{\text{rev}}{\text{min}}$ $d_{\mathbf{S}} := \frac{3}{8} \text{in}$	Diameter of screw
$P_{S} := 1 in$	Pitch of screw	•	
$\eta_{\rm S} \coloneqq .86$	Efficiency of screw assembly	$R_s := \frac{d_s}{2}$	Radius of screw
oo to kg	Danaita of ataul	$L_S := 6in$	Length of screw
$\rho := 8050 \frac{\text{kg}}{\text{m}^3}$	Density of steel	$\theta := 7.5 \text{deg}$	Angle to sweep up to
$\omega := 50 \frac{\text{deg}}{\text{s}}$	Angular velocity	ī	
$\mu := .16$	Coefficient of friction b/n nut and screw	$\mathbf{S} := \frac{\mathbf{L}}{2} \cdot \sin(\theta) = 0.783 \cdot \text{in}$	Vertical travel distance

## Estimate the power required to lift mass:

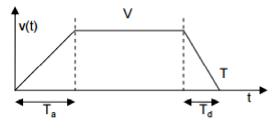
$$t := \frac{\theta}{\omega} = 0.15 \,\mathrm{s}$$

$$P := \frac{m_{r} \cdot g \cdot S}{t} = 11.798 \,\mathrm{W}$$

Time Required to sweep up to angle

Power required to lift mass in given time

Using a trapezoidal velocity profile calculate peak velocity and the minimum pitch required:



 Reduces peak velocity by 25% when compared to a triangular profile

 $v_{pk} := \frac{3 \cdot S}{2t} = 0.199 \cdot \frac{m}{s}$ 

Peak velocity following a 1/3-1/3-1/3 profile

 $P_{min} := \frac{v_{pk}}{\omega_m} = 0.147 \cdot in$ 

Minimum pitch required to sweep up to angle in given time and rpm

 $\label{eq:leadscrewspeed} \text{Leadscrewspeed}_{pk}(\text{Pitch}) \coloneqq \frac{v_{pk} \cdot 60}{\text{Pitch}}$ 

Recalculate speed based on selected pitch screw

 $Leadscrewspeed_{pk}(P_s) = 469.894 \cdot \frac{rev}{s}$ 

$$\omega_{pk} \coloneqq \text{Leadscrewspeed}_{pk}\!\!\left(P_{S}\right) \! \cdot \! \frac{2\pi}{60} = 2952.433 \! \cdot \! \frac{\text{rev}}{\text{min}}$$

Calculate the reflected inertia from load and screw to the input power:

$$m_s := \pi \cdot R_s^2 \cdot L_s \cdot \rho = 0.193 \cdot lbm$$

Mass of lead screw

$$J_s := m_s \cdot R_s^2 = 1.983 \times 10^{-6} \text{ m}^2 \cdot \text{kg}$$

Inertia from screw

$$J_L := m_r \cdot (0.063in)^2 = 2.323 \times 10^{-5} \text{ m}^2 \cdot \text{kg}$$

Inertia from Load

$$J_T := J_s + J_L = 2.521 \times 10^{-5} \text{ m}^2 \cdot \text{kg}$$

Total Inertia

Calculate the acceleration of the screw based on peak velocity:

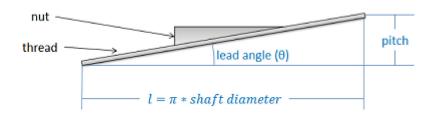
$$a := \frac{v_{pk}}{\left(\frac{1}{3} \cdot t\right)} = 3.978 \frac{m}{s^2}$$

Acceleration during 1/3 of the profile

$$\alpha := \frac{a \cdot 2\pi}{P_S} = 984.144 \cdot \frac{\text{rad}}{S^2}$$

Angular acceleration

Calculate Torques based on reflected inertia and weight being moved:



Unwrapping the threads, the lead angle can then be calculated

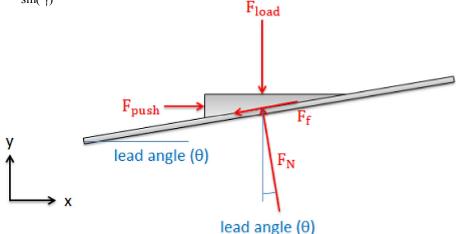
$$\gamma := \operatorname{atan}\left(\frac{P_{s}}{\pi \cdot d_{s}}\right) = 40.325 \cdot \operatorname{deg}$$

$$x := \frac{\left(\frac{S}{P_{s}}\right) \cdot P_{s}}{\sin(\alpha)} = 1.21 \cdot \operatorname{in}$$

Need to check if screw is self locking later

$$x := \frac{\left(\frac{S}{P_s}\right) \cdot P_s}{\sin(\gamma)} = 1.21 \cdot in$$

Distance traveled along incline



$$T_{f} := \frac{m_{f} \cdot \mu \cdot g}{\cos(\gamma)} \cdot R_{g} = 0.089 \cdot N \cdot m$$

Torque to overcome friction

$$T_I := J_T \cdot \alpha = 0.025 \cdot N \cdot m$$

Torque to accelerate load

$$T_a := \frac{T_J + T_f}{\eta_s} = 0.132 \cdot N \cdot m$$

Total torque

## Based on total torque, calculate the input power required:

$$P_{pk} := T_a \cdot \omega_{pk} = 6.508 \,\mathrm{W}$$

Power input required at the shaft

$$P_{avg} := \frac{\left(\frac{m_r \cdot g \cdot S}{t}\right)}{\eta_S} = 13.719 \,\text{W}$$

Average Power

$$T_{rms} := \sqrt{\frac{\frac{1}{3} \cdot t \cdot T_J^2 + \frac{1}{3} \cdot t \cdot T_f^2 + \frac{1}{3} \cdot t \cdot T_J^2}{t}} = 0.055 J$$

Will be used later to check heat build up

## Motor-Cam Design

$$d_{rod} := \frac{3}{8}in$$
  $L_2 := 3in$ 

Linkage arm dimensions

$$y := \frac{L}{4} \cdot \sin(\theta) = 0.392 \cdot \text{in}$$
  $L_1 := y$ 

$$L_1 := y$$

Moment arm

$$J_{r} := \left[\pi \cdot \left(\frac{d_{rod}}{2}\right)^{2} \cdot L_{2} \cdot \rho\right] \cdot \left(\frac{L_{2}^{2}}{3}\right) = 8.46 \times 10^{-5} \,\mathrm{m}^{2} \cdot \mathrm{kg}$$

$$J_{p} := \frac{m_{r}}{12} \cdot \left[4 \cdot L^{2} + (W)^{2}\right]$$

Inertias of the linkages and platform with the robot standing on top

$$J_{\text{TW}} := J_{\text{p}} + J_{\text{r}} = 0.351 \,\text{m}^2 \cdot \text{kg}$$

$$\alpha := \frac{\omega}{t} = 5.818 \cdot \frac{\text{rad}}{s^2}$$

Angular acceleration required

$$M := m_r \cdot \frac{g}{\cos(\theta)} \cdot L_1 = 0.892 \cdot N \cdot m$$

$$T_r := J_T \cdot \alpha = 2.044 \cdot N \cdot m$$

$$T_t := T_r + M = 2.936 J$$