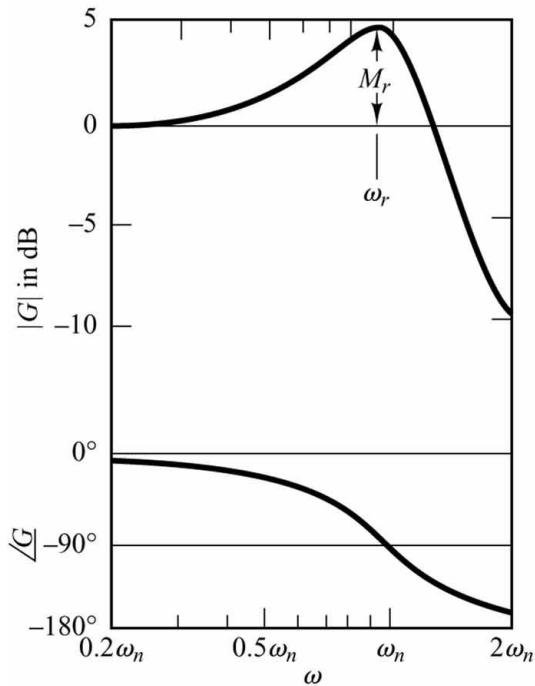


Automatic Control Systems

Lag-lead

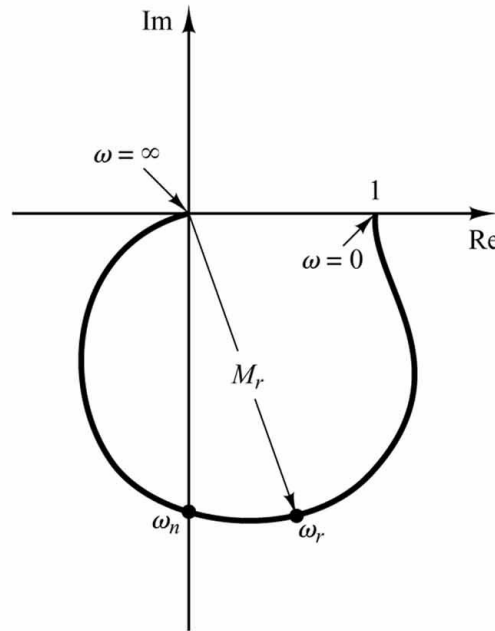
alghoniemy@alexu.edu.eg

Frequency Domain Representations



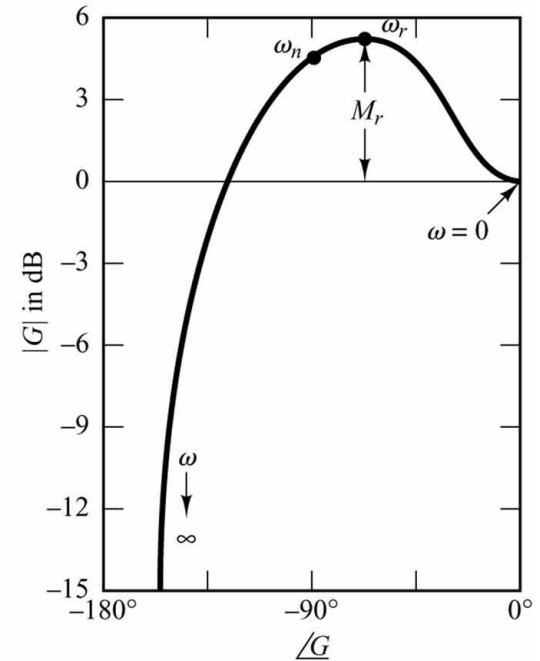
(a)

**Logarithmic plots
(Bode Plots)**



(b)

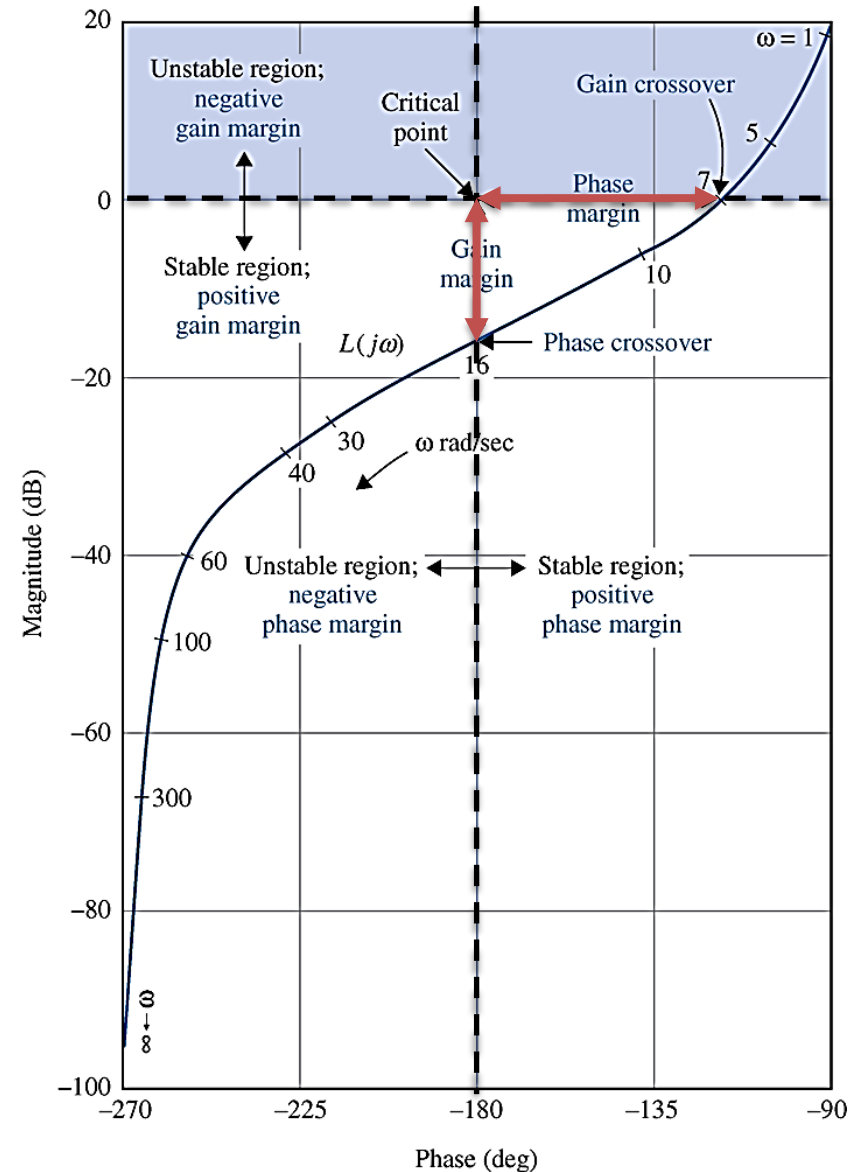
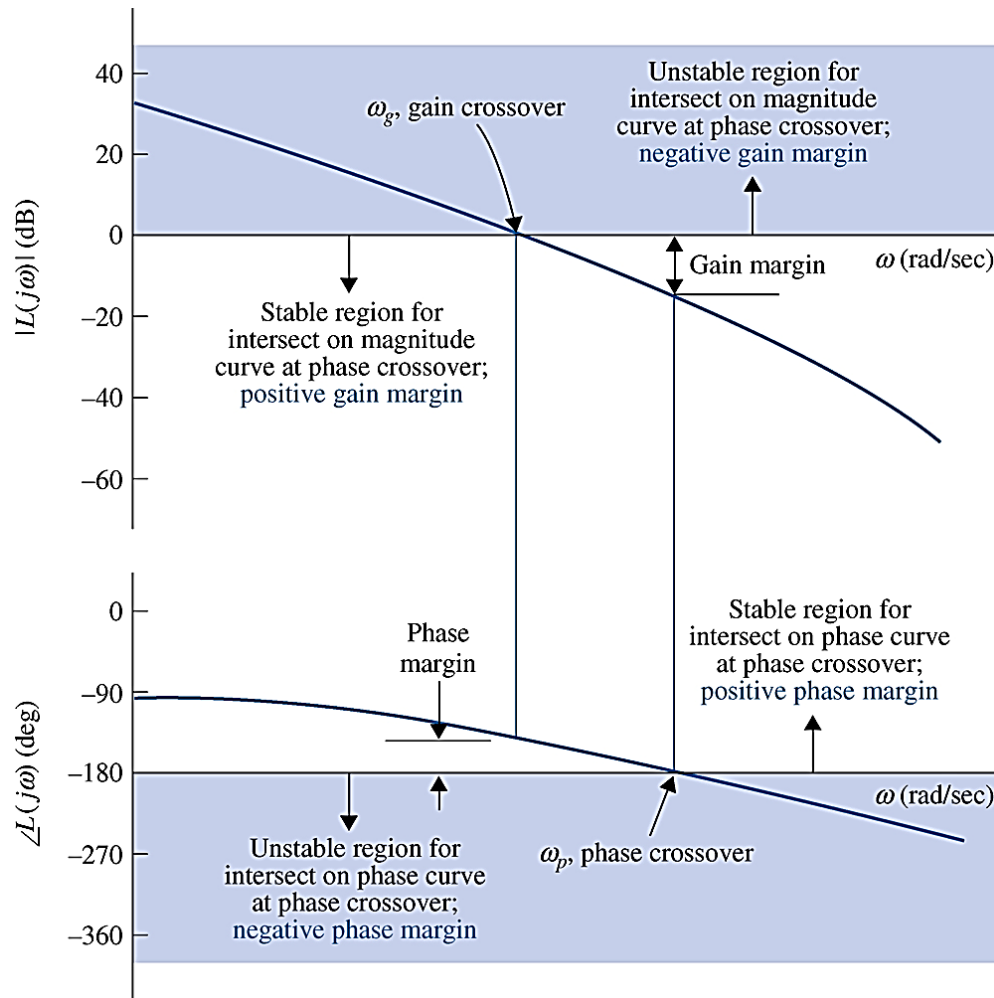
**Polar plots
(Nyquist Diagrams)**



(c)

**Log Magnitude-Phase
(Nichols Charts)**

Stability analysis with the log-magnitude and phase diagram

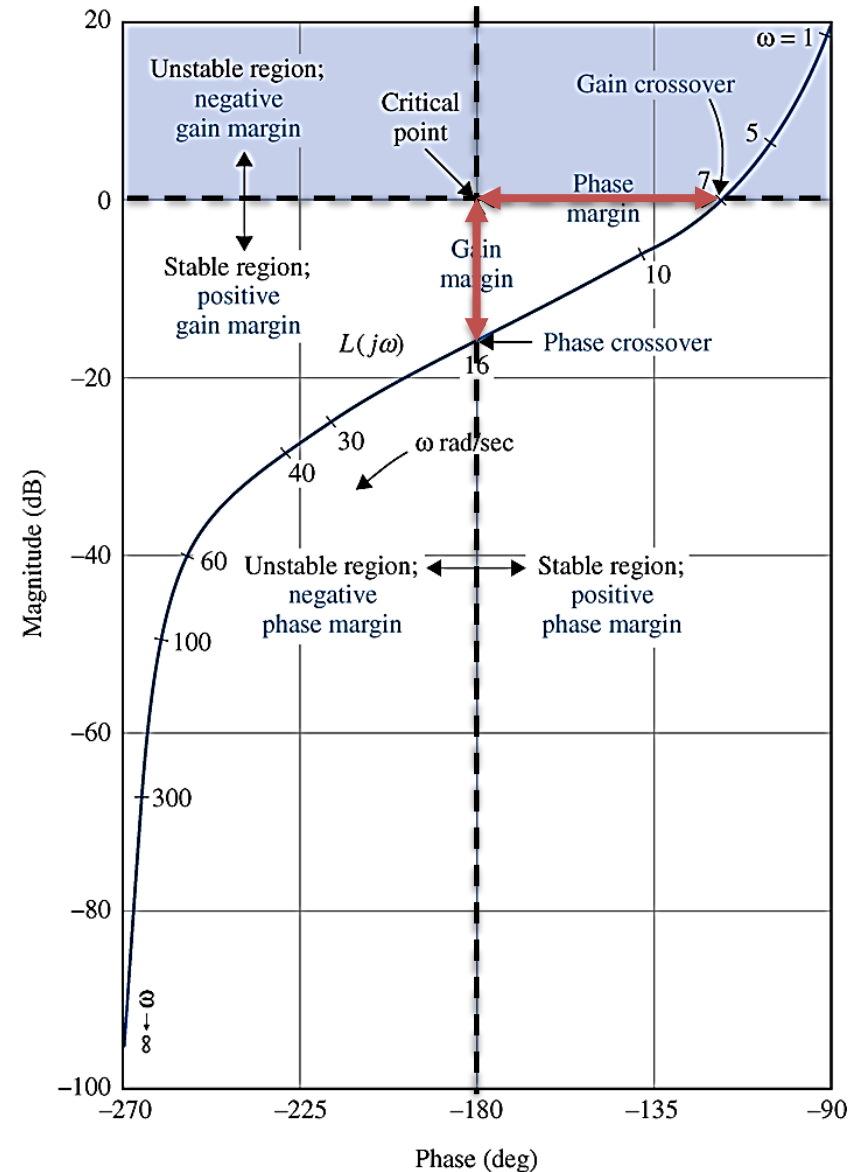


Stability analysis with the log-magnitude and phase diagram

- This is **open-loop** representation
- The critical point at $(0 \text{ dB}, -180^\circ)$

The **upper-right** corner is the **low frequency**,
the **lower-left** is the **high frequency**

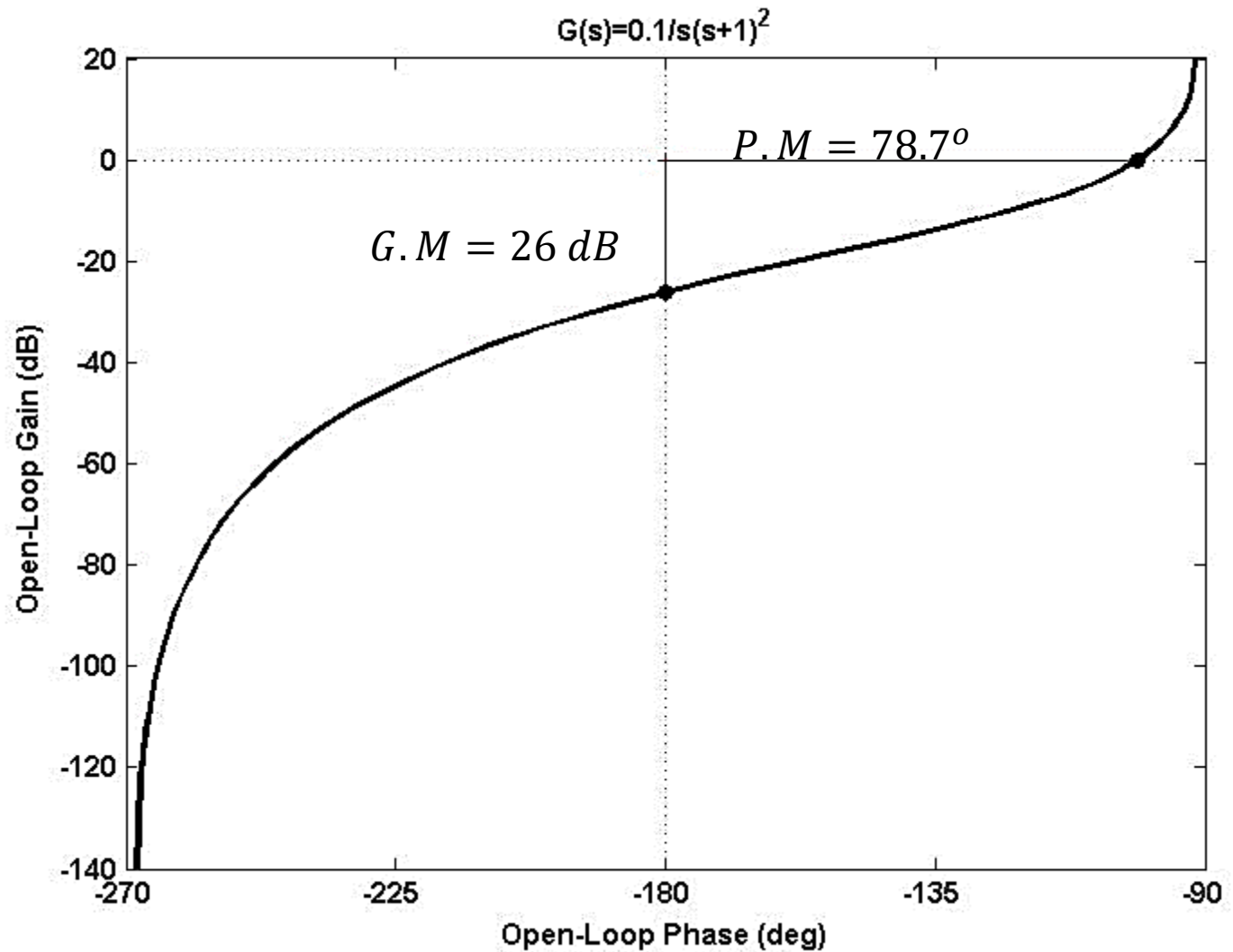
- The system is stable if, in the direction of increasing frequency, the curve intersects the 0 dB line before the -180° line
- Changing the gain K shifts the curve **vertically**
- Changing the **phase ϕ** shifts the curve **horizontally**



- example

$$G(s) = \frac{0.1}{s(s+1)^2}$$

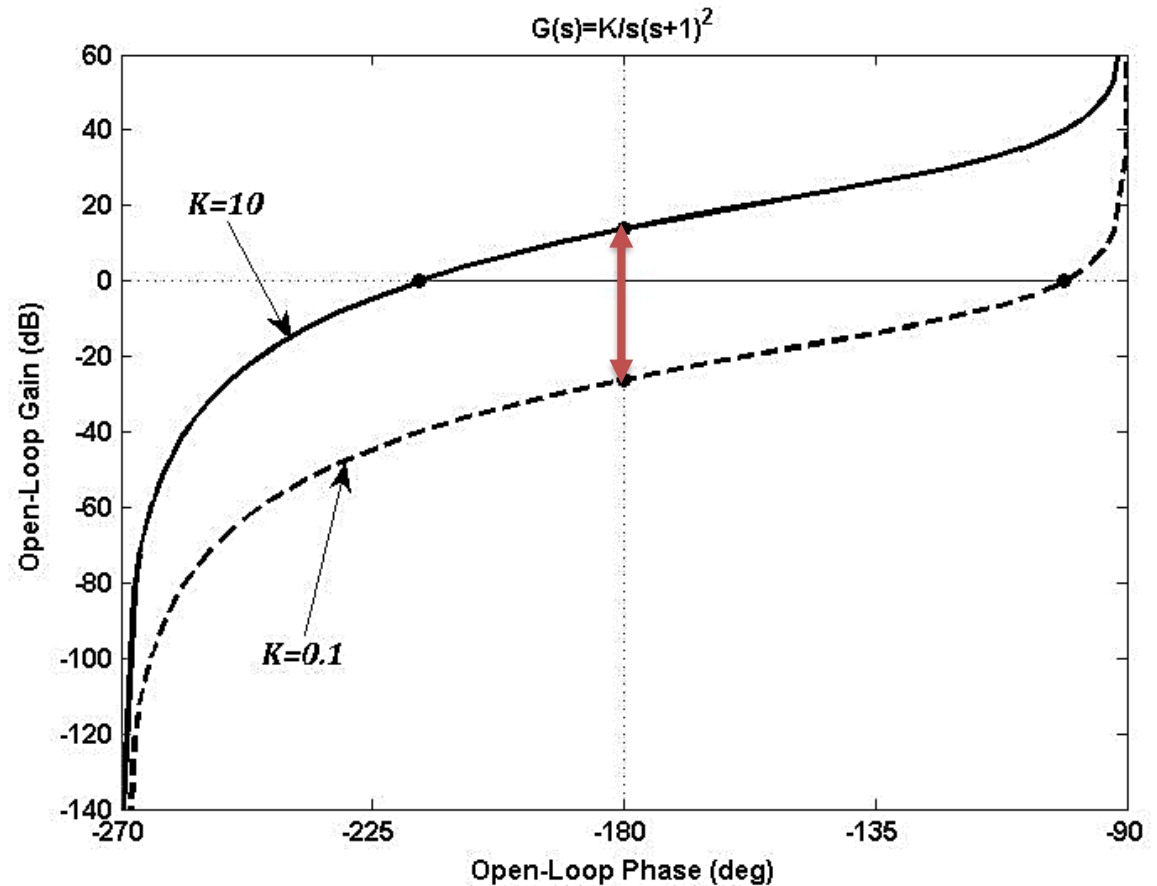
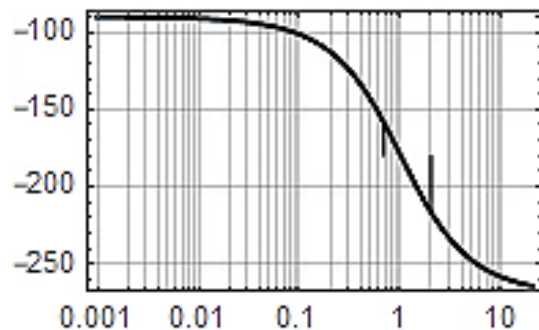
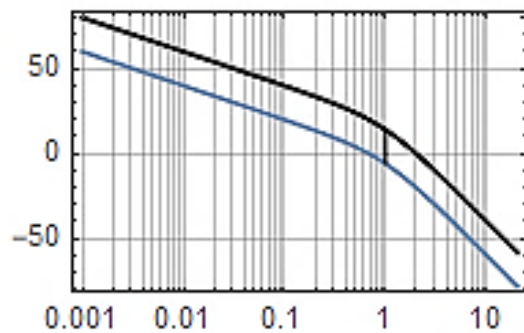
- The system is stable.



- Increasing the gain shifts the curve vertically

$$G(s) = \frac{K}{s(s+1)^2}$$

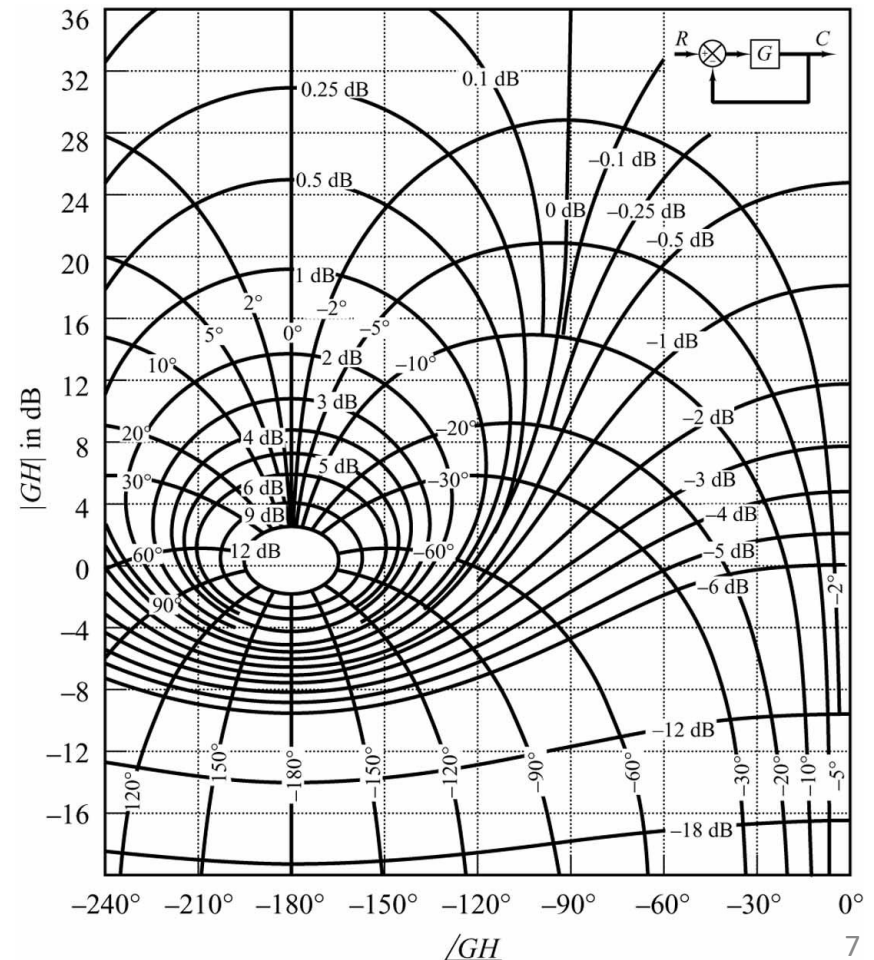
K	stability
$K = 0.1$	stable
$K = 10$	unstable



$$\frac{10}{s(s+1)^2}$$

The Nichols chart

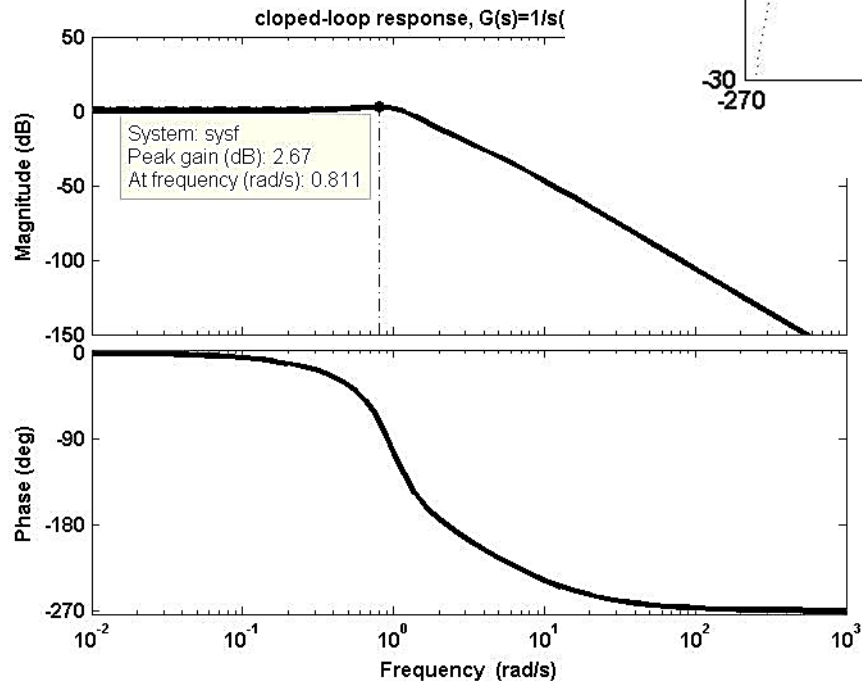
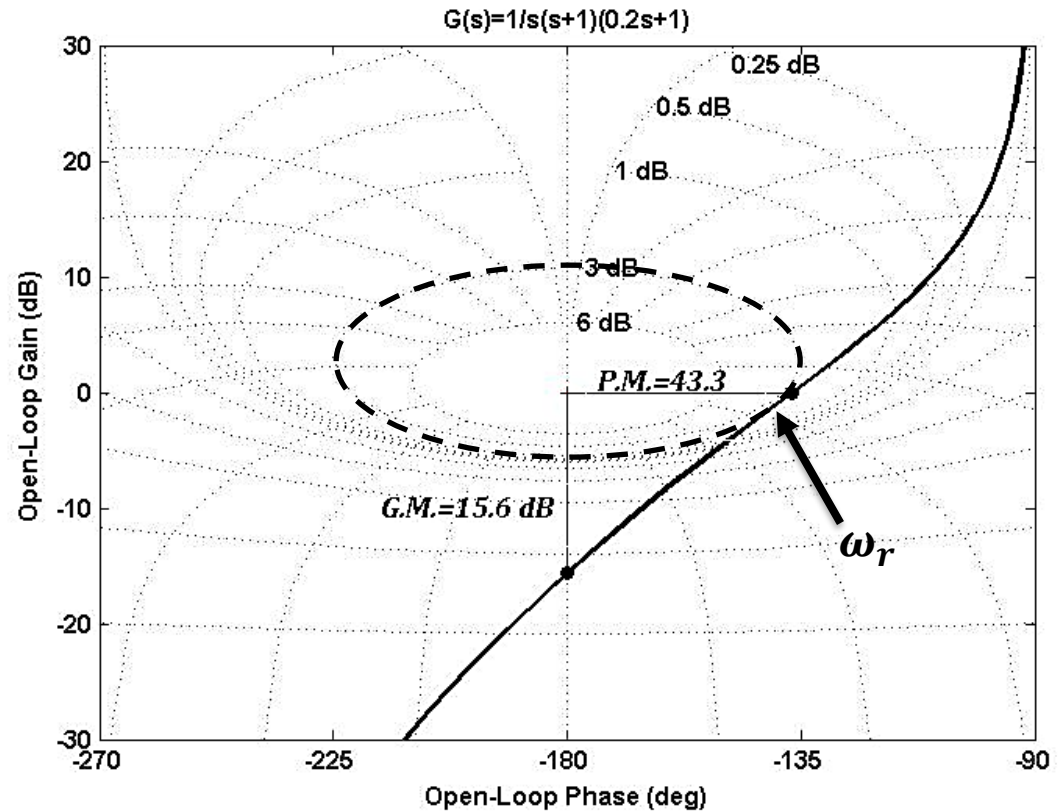
- Constant M -circles and N -circles in the **magnitude-phase plane**
- The intersects between the constant M -loci and the $G(j\omega)$ trajectory gives the value of **closed-loop** $|M(j\omega)|$
- The **resonance peak** M_r and the **resonant frequency** ω_r are found by locating the **smallest** M -locus that is **tangent** to the $G(j\omega)$ trajectory
- The **bandwidth** is the frequency at which $G(j\omega)$ curve intersects $M = 0.707$ locus



- example

$$G(s) = \frac{1}{s(s+1)(0.2s+1)}$$

- Closed-loop response



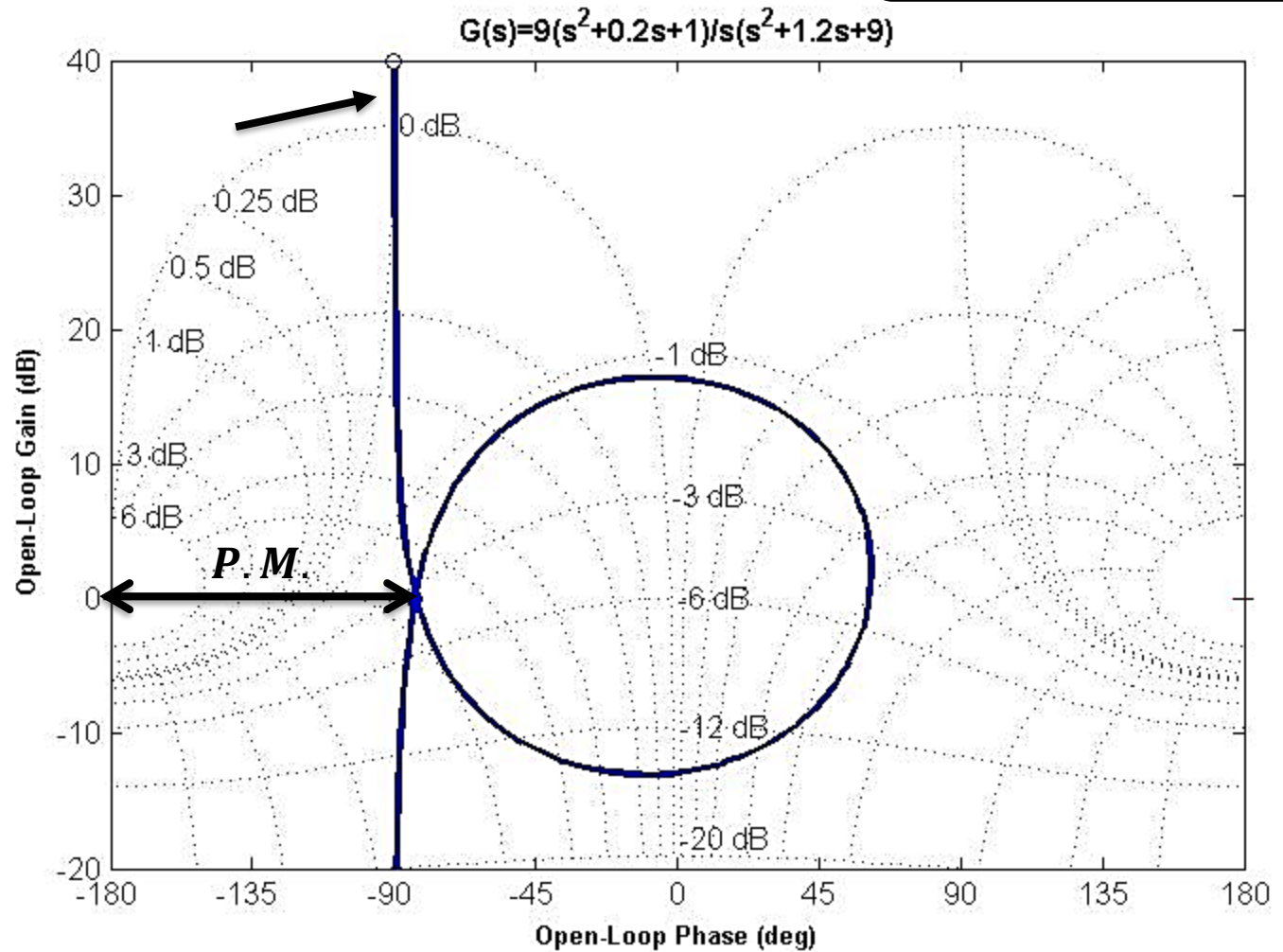
M_r	2.67 dB
ω_r	0.811 rad/sec
ω_B	1.33 rad/sec

- example

$$G(s) = \frac{9(s^2 + 0.2s + 1)}{s(s^2 + 1.2s + 9)}$$

- $G.M = \infty?$

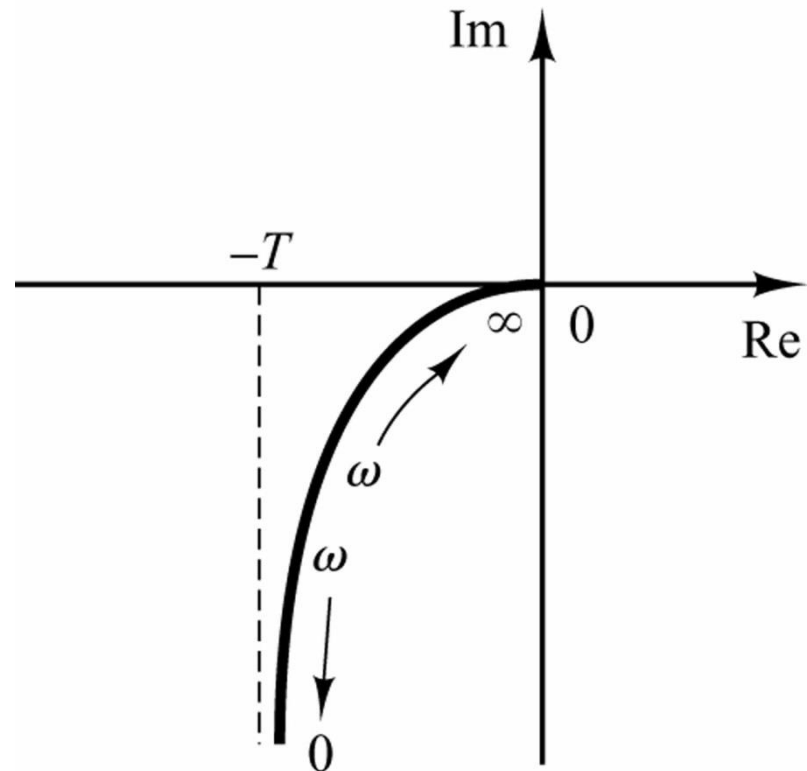
- $M_r = 0 \text{ dB}$



```
b=9*[1 0.2 1]; a=[1 1.2 9 0]; sys=tf(b,a); nichols(sys)
```

Frequency domain design of control systems

- The **low-frequency** region of the open-loop locus indicates the **steady state** behavior of the closed-loop system
- The **medium-frequency** region of the locus indicates the **relative stability**
- The **high-frequency** region indicates the **transient response**



- example:

$$G(s) = \frac{K}{s(1+s)(1+0.0125s)}$$

- We need the steady state error for a unit ramp must be less than 0.01
- We need to realize a resonant peak $M_p = 1.25$

$$e_{ss} = \lim_{s \rightarrow 0} \frac{1}{sG(s)} = \frac{1}{K} \leq 0.01$$

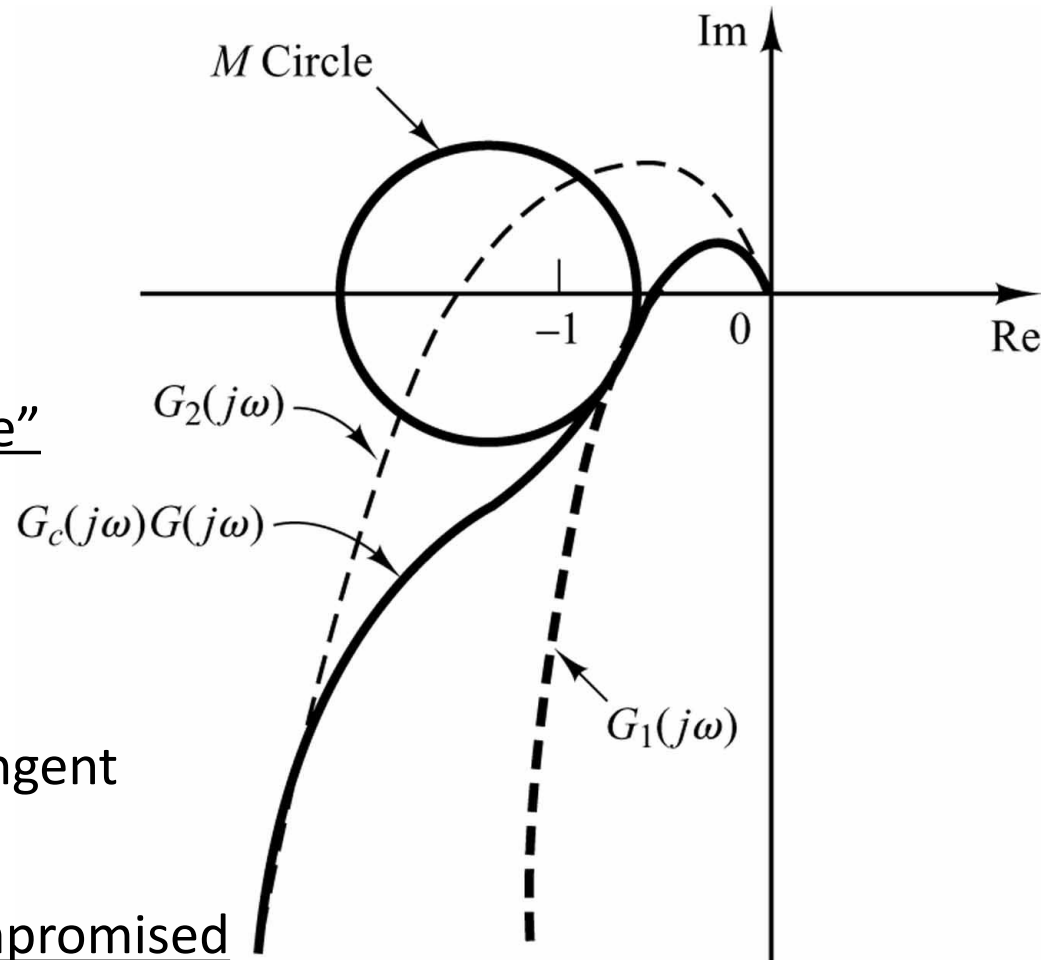
$$K > 100$$

for $K > 100 \Rightarrow G_2(j\omega)$ “unstable”

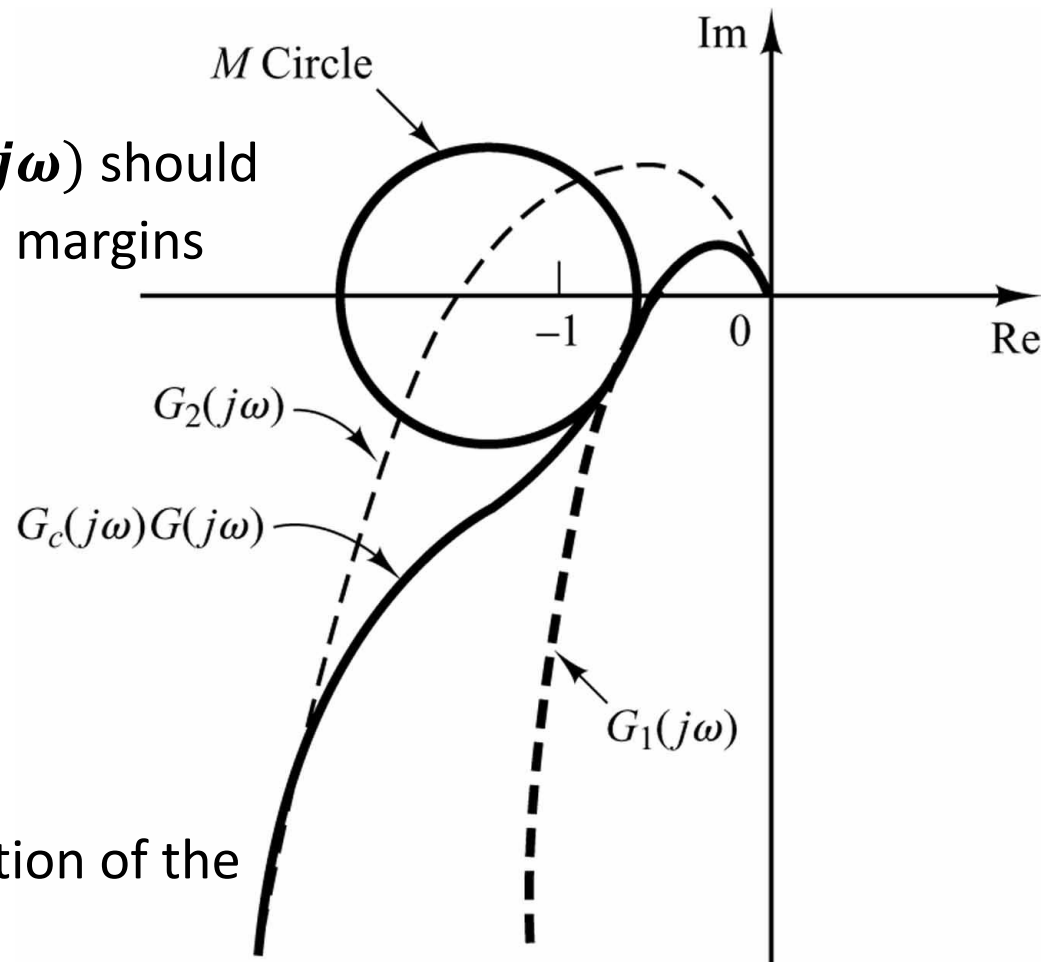
for stability, we need $K < 81$

- In order for the curve to be tangent to $M = 1.25 \Rightarrow K = 1.2$

then the steady state error is compromised



- we need to obtain a **compromise** between **steady state accuracy** and **relative stability**.
- This is obtained by **reshaping** the open-loop frequency response plot.
- The reshaped locus $G_c(j\omega)G(j\omega)$ should have a reasonable phase and gain margins or should be tangent to a certain M circle
- Reshaping is performed using **compensation**
- Reshaping is performed such that the **high-frequency** portion of the locus follows the $G_1(j\omega)$ while the **low-frequency** portions follows the $G_2(j\omega)$ locus

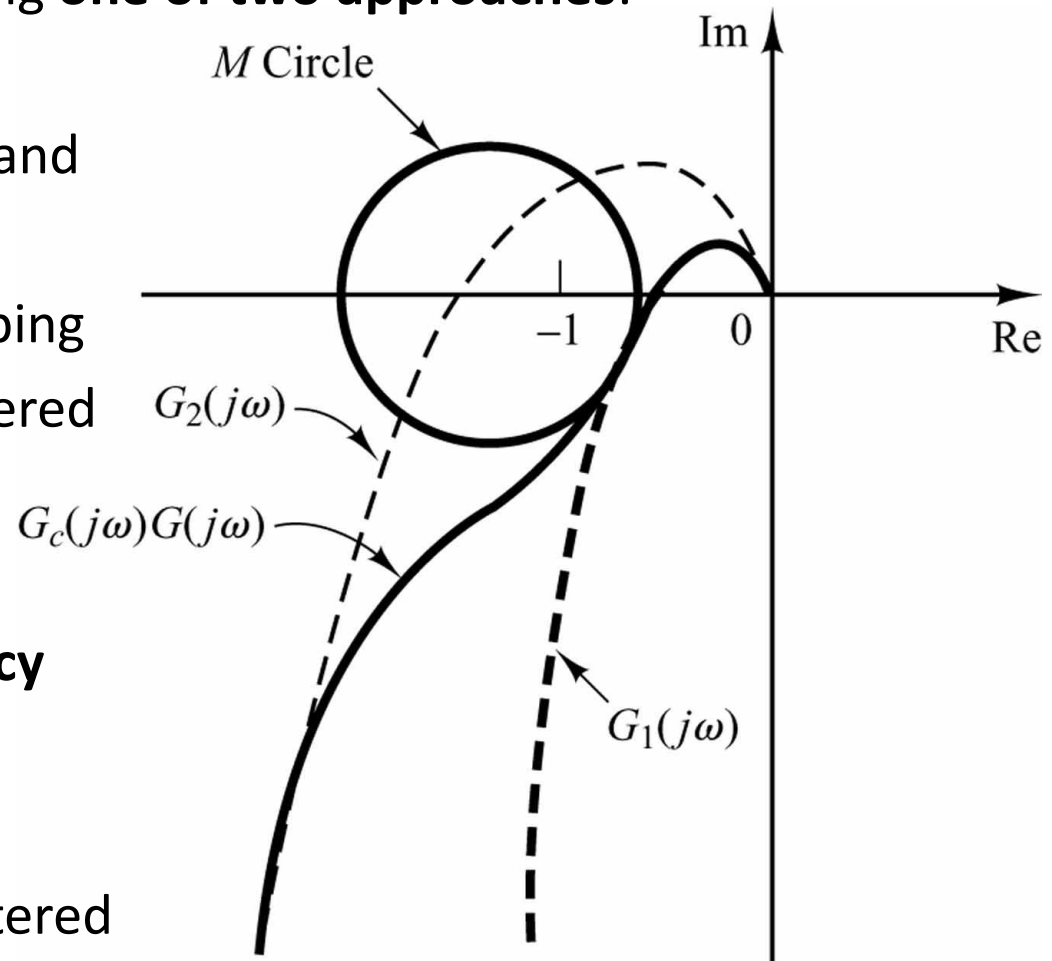


- then the Nyquist plot should be reshaped to follow $G_1(j\omega)$ in the **high-frequency** region

- this can be performed in using **one of two approaches**:

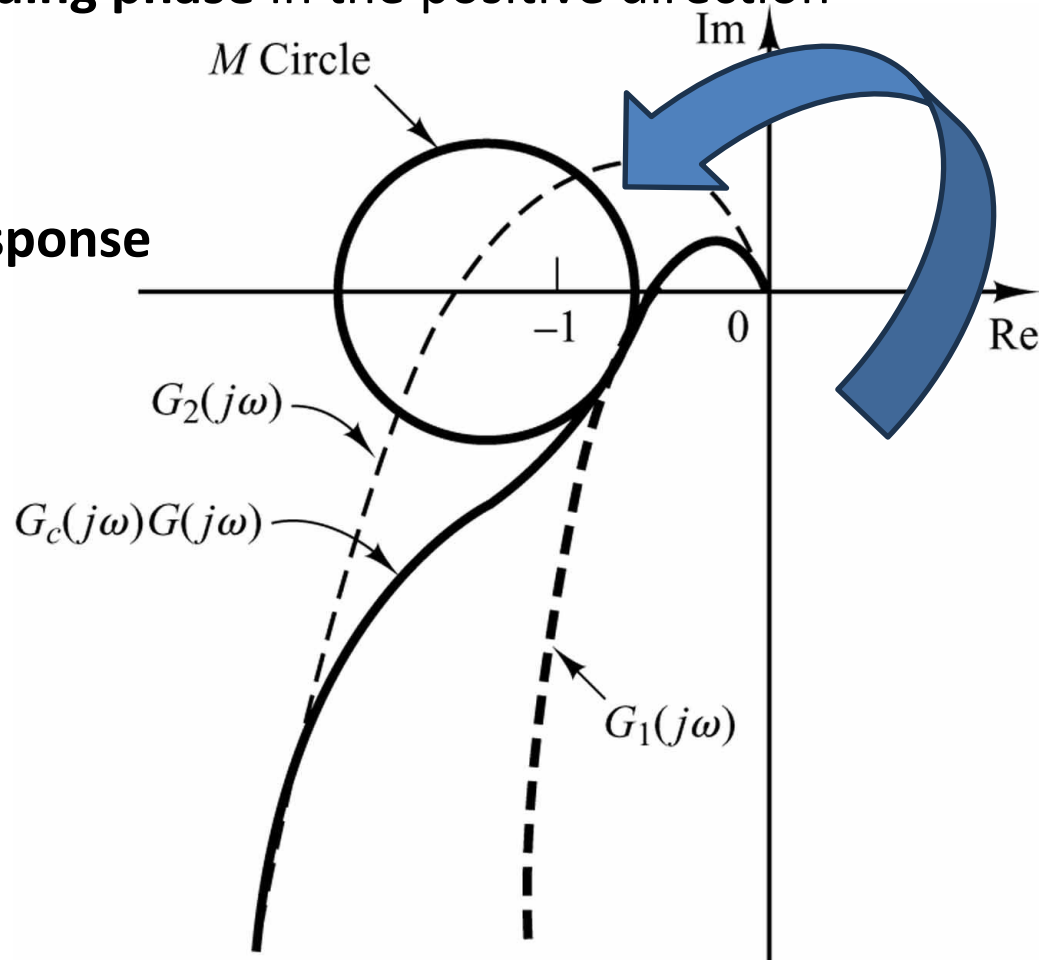
1. Starting with $G_2(j\omega)$ locus and reshaping the locus in the **high-frequency** region and keeping the low-frequency region unaltered

2. Starting with $G_1(j\omega)$ locus and reshaping the **low-frequency** portion to obtain a ramp-error constant of 100 while keeping the high-frequency region unaltered



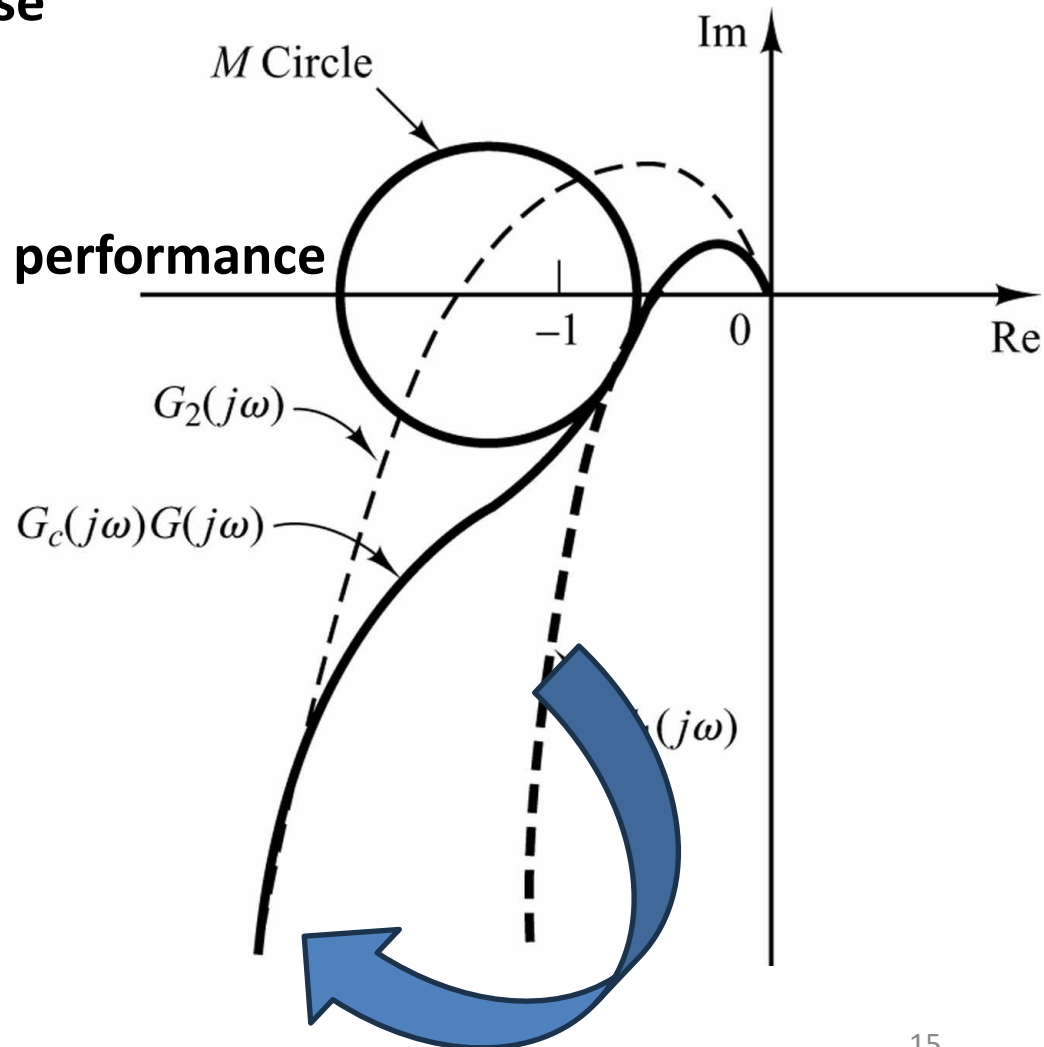
Phase-lead compensation

- the **high-frequency** portion of $G_2(j\omega)$ is rotated in the **counter-clockwise** direction, i.e., by **adding phase** in the positive direction
- Lead compensation yields an **improvement in the transient response** while and a **small change in steady state accuracy**
- lead compensation **increases** the system order by **one-degree** (unless cancellation occurs)



Phase-lag compensation

- the **low-frequency** portion of $G_1(j\omega)$ is shifted in the **clockwise** direction, i.e., by **subtracting phase**
- Lag compensation yields an **improvement in the steady-state performance** at the expense of **increasing the transient response time**
- lag compensation **increases** the system order by **one-degree** (unless cancellation occurs)



Lag-lead compensation

- Combines the characteristics of both lag and lead compensation
- The lag-lead compensation increases the system order by **two-degrees** (unless cancellation occurs), which increases system complexity.
- Rarely used

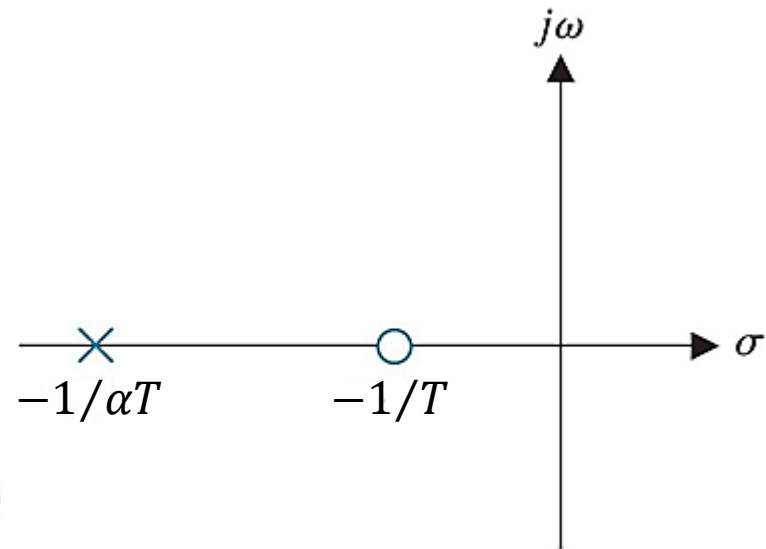
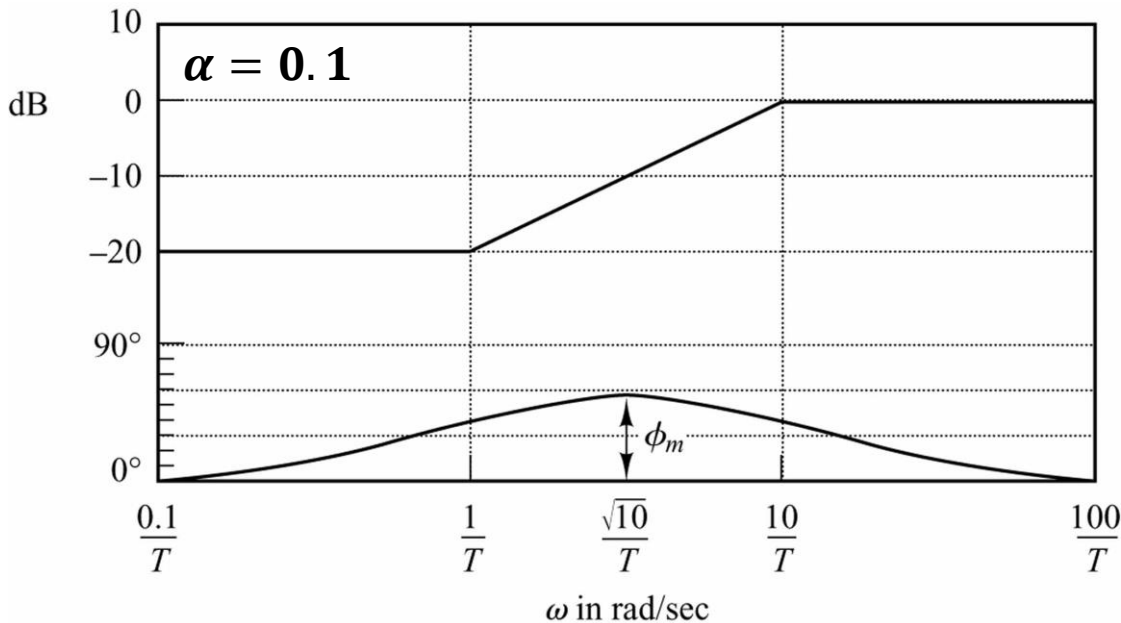
Phase-lead compensation

- Phase-lead adds a **zero**
- Phase lead is a **high-pass filter (PD)**
- may amplify high-frequency noise

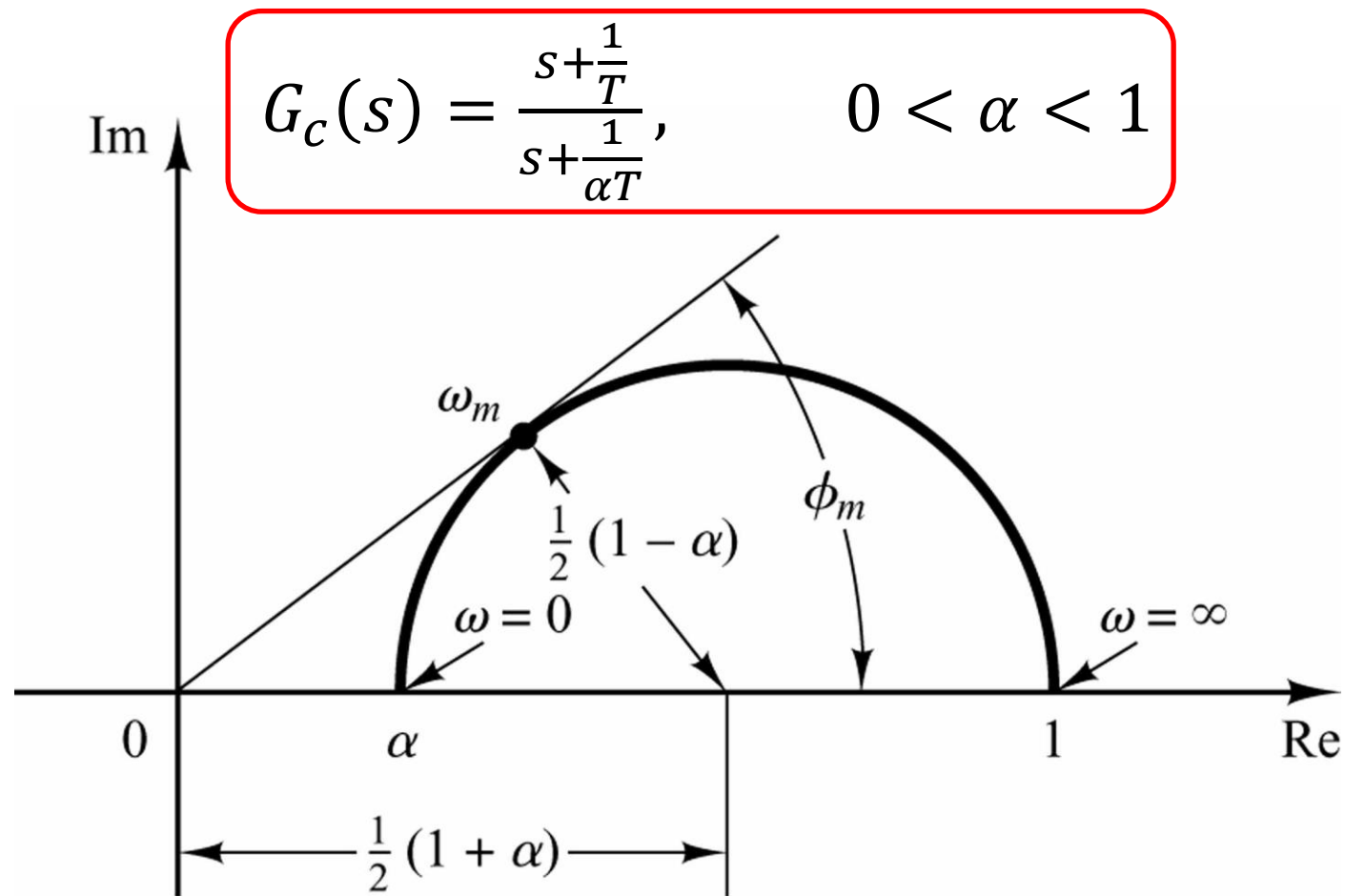
$$G_c(s) = K_c \alpha \frac{Ts + 1}{\alpha Ts + 1} = K \frac{Ts + 1}{\alpha Ts + 1}$$

$$G_c(s) = K_c \frac{s + \frac{1}{T}}{s + \frac{1}{\alpha T}} \quad (0 < \alpha < 1)$$

- The zero is closer to the origin than the pole (**strong zero**)



- Polar plot of the phase lead compensator,



- The maximum phase ϕ_m the compensator can provide is given by

$$\sin \phi_m = \frac{1 - \alpha}{1 + \alpha} \quad (0 < \alpha < 1)$$

- The minimum value of the attenuation factor $\alpha \cong 0.05$
- The **maximum phase-lead** that the compensator can provides is $\phi_m \cong 65^\circ$

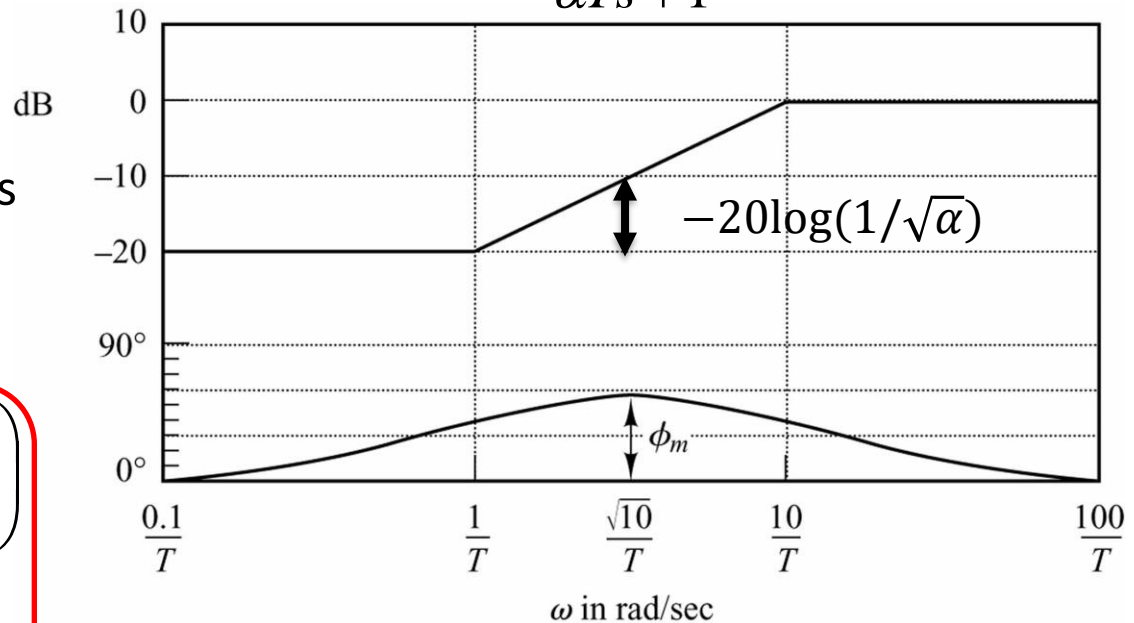
$$\sin \phi_m = \frac{1 - \alpha}{1 + \alpha} \quad (0 < \alpha < 1)$$

- the maximum phase happens at ω_m :

$$\log \omega_m = \frac{1}{2} \left(\log \frac{1}{T} + \log \frac{1}{\alpha T} \right)$$

$$\Rightarrow \omega_m = \frac{1}{T \sqrt{\alpha}}$$

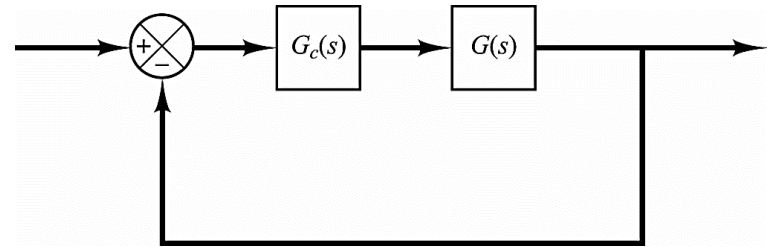
$$G_c(s) = K_c \alpha \frac{Ts + 1}{\alpha Ts + 1} \quad (0 < \alpha < 1)$$



Design steps using the phase-lead compensator

$$G_c(s)G(s) = \frac{Ts + 1}{\alpha Ts + 1} KG(s)$$

$$(0 < \alpha < 1)$$



1. Determine the value of **K** for a specific **steady state error**.
2. Evaluate the **phase margin** from Bode plot using the determined gain.
3. Determine the **necessary phase-lead angle to be added** to the system.
4. Add an additional **5° to 12°** to be added to the required phase.

5. Determine the **attenuation factor α** from

$$\sin \phi_m = \frac{1 - \alpha}{1 + \alpha}$$

$$(0 < \alpha < 1)$$

6. the new gain crossover frequency $\omega_m = 1/T\sqrt{\alpha}$ happens at the magnitude of the uncompensated system $KG(s) = -20\log(1/\sqrt{\alpha})$.
7. Determine the corner frequencies:

$$\text{zero @ } \omega = 1/T = \omega_m\sqrt{\alpha}, \quad \text{pole @ } \omega = 1/\alpha T = \omega_m/\sqrt{\alpha}$$

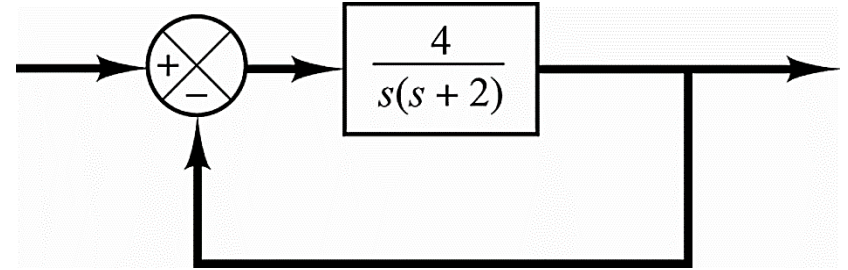
8. Determine the constant **$K_c = K/\alpha$**

- Example:

$$G(s) = \frac{4}{s(s+2)}$$

- Design specs:

1. velocity error constant $K_v = 20 \text{ sec}^{-1}$
2. $P.M. \geq 50^\circ, G.M > 10 \text{ dB}$



1. The velocity error constant

$$G_c(s)G(s) = \frac{Ts + 1}{\alpha Ts + 1} \frac{4K}{s(s+2)}$$

$$K_v = \lim_{s \rightarrow 0} sG_c(s)G(s) = 2K = 20$$

$$\Rightarrow K = 10 \Rightarrow KG(s) = \frac{40}{s(s+2)}$$

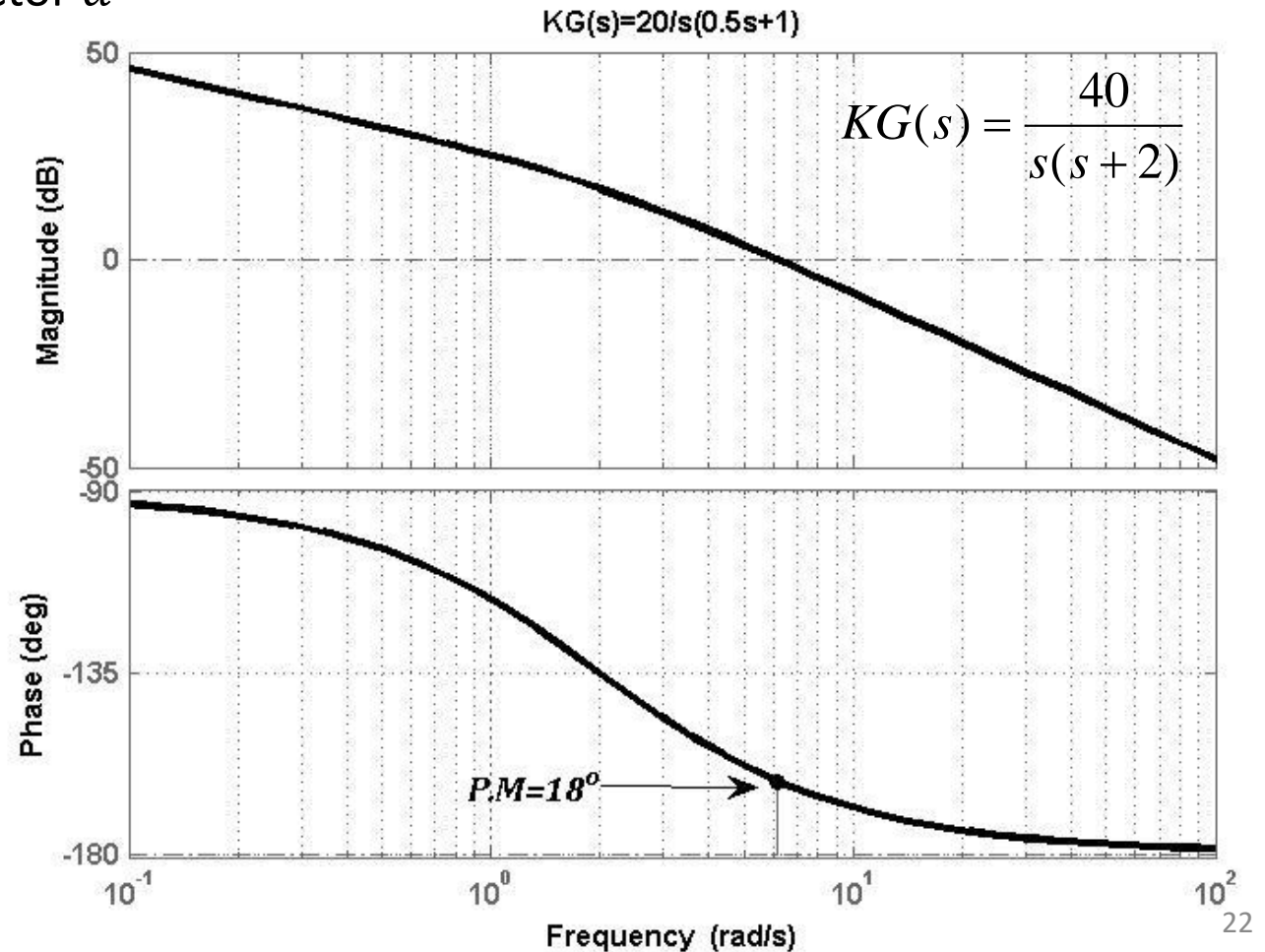
- The phase margin from Bode-plot $P.M. = 18^\circ$

- The maximum phase to be added $\phi_m = (50^\circ - 18^\circ) + 5^\circ = 37^\circ$

- the attenuation factor α

$$\sin \phi_m = \frac{1 - \alpha}{1 + \alpha}$$

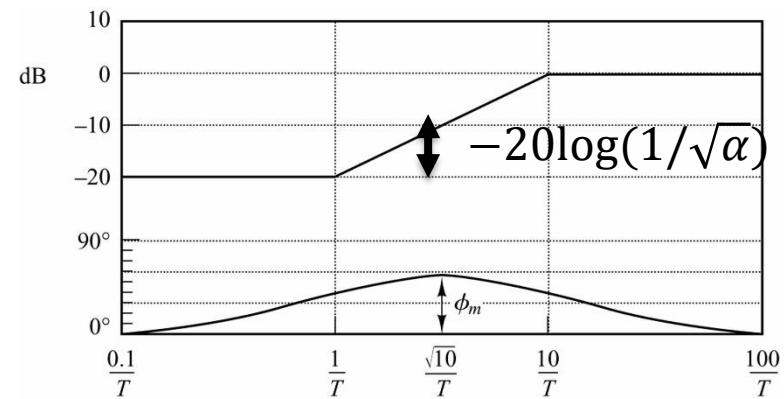
$$\Rightarrow \alpha = 0.24$$



- the **new gain crossover** frequency

$$-20\log(1/\sqrt{\alpha}) = -6.2 \text{ dB},$$

happens at $\omega_m = 1/T\sqrt{\alpha} = 9 \text{ rad/sec}$



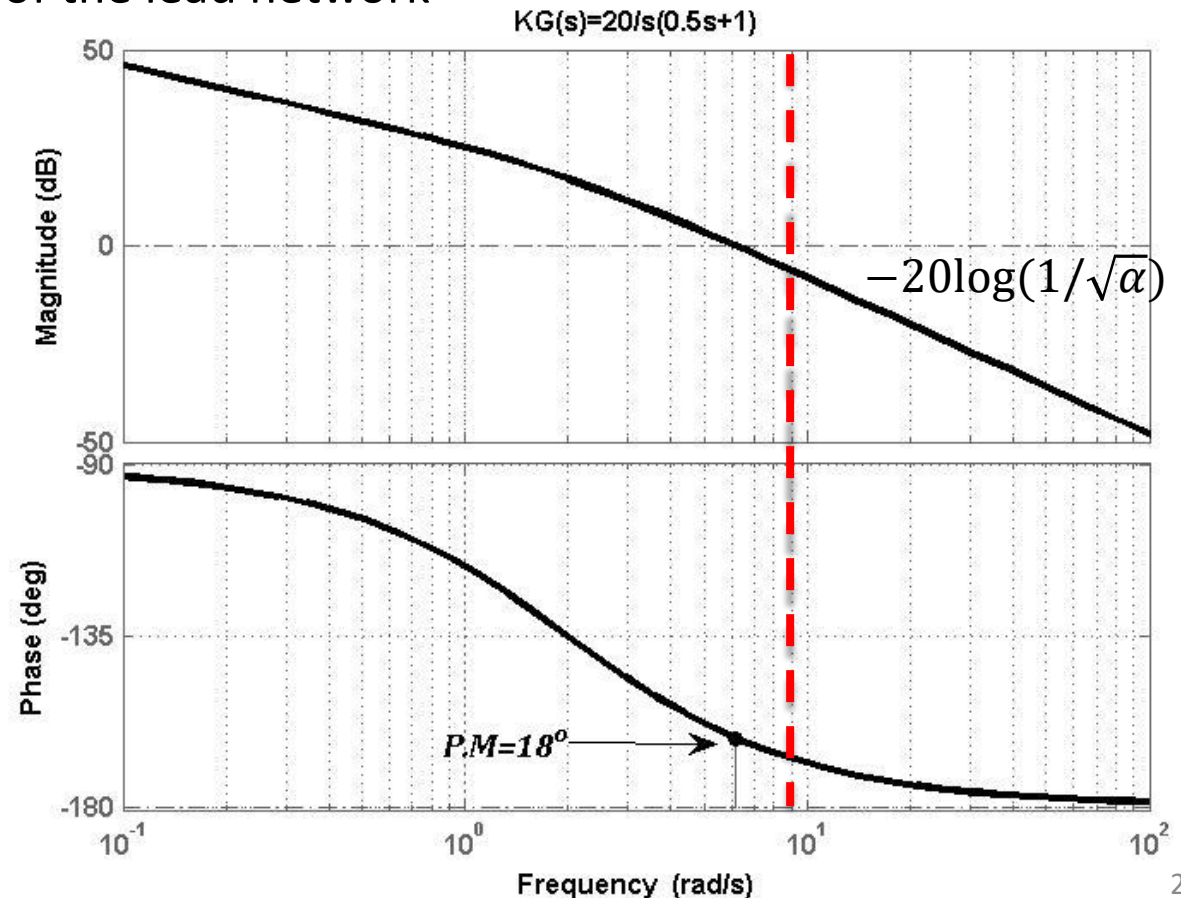
- The pole and the zero of the lead network

- Zero:**

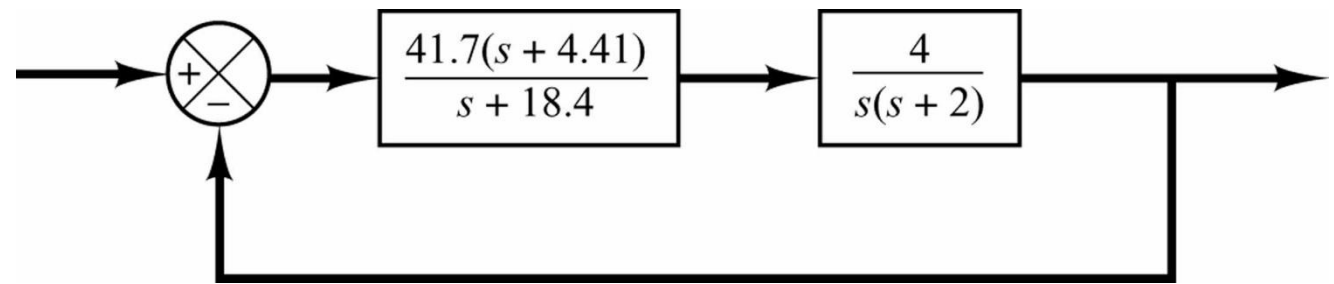
$$\begin{aligned} @ 1/T &= \omega_m \sqrt{\alpha} \\ &= 4.41 \text{ rad/sec} \end{aligned}$$

- Pole:**

$$\begin{aligned} @ 1/\alpha T &= \omega_m / \sqrt{\alpha} \\ &= 18.4 \text{ rad/sec} \end{aligned}$$



- **Bode-diagram**

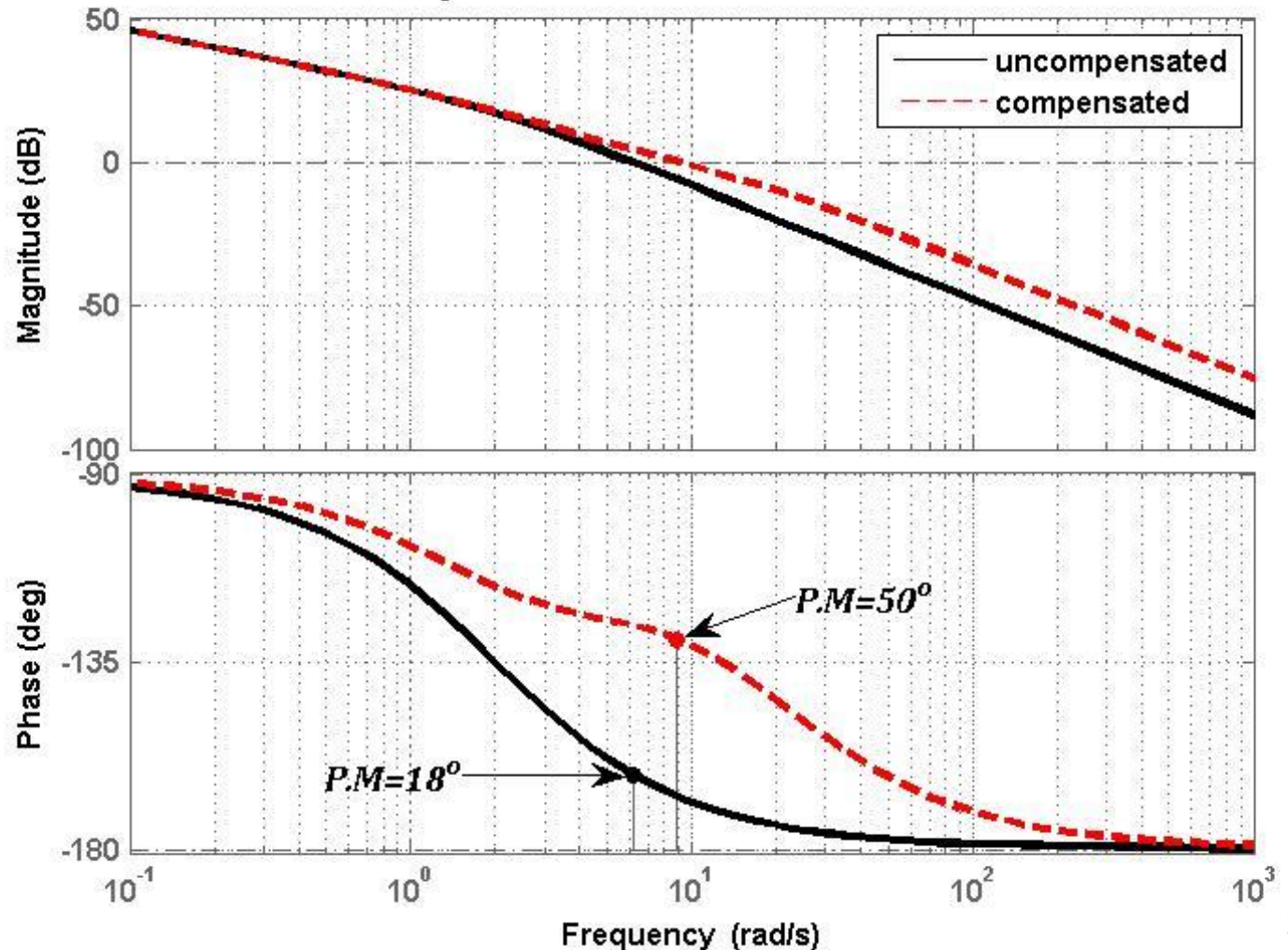


- The bandwidth has increased

$$G_c(s)G(s) = 4 \cdot 41.7 (s + 4.41) / s(s + 2)(s + 18.4)$$

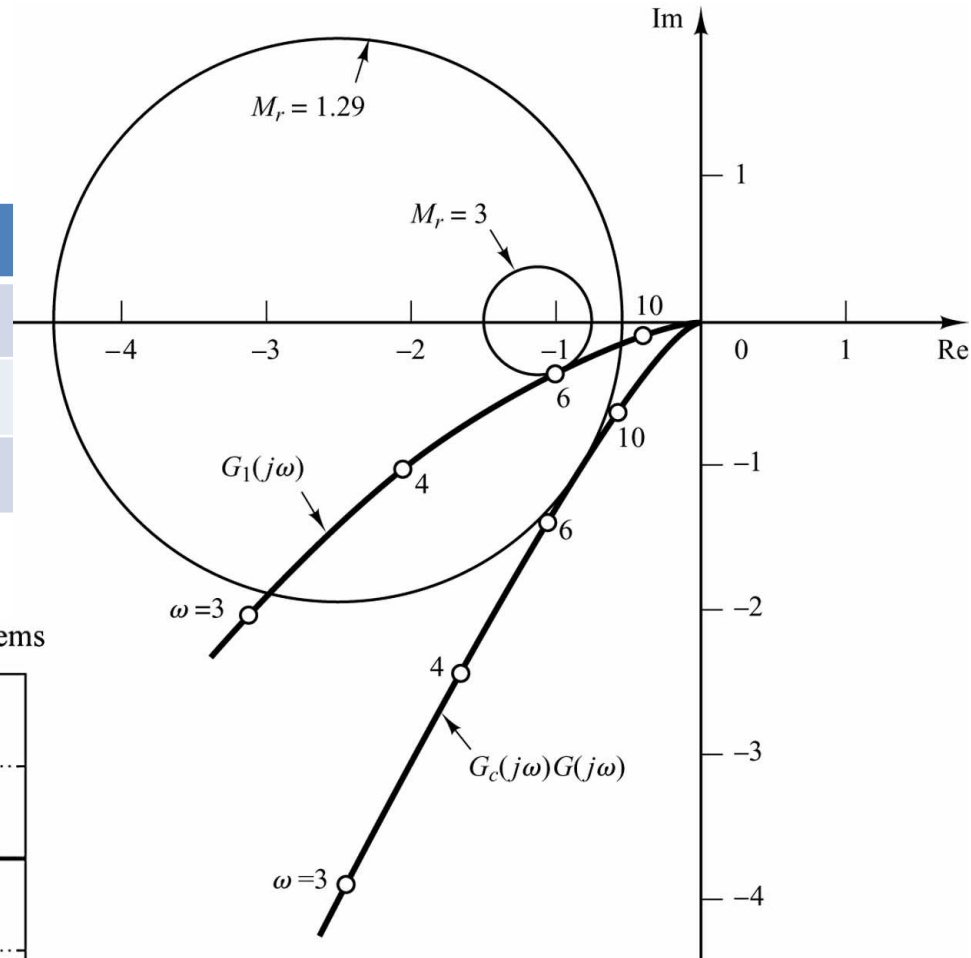
$$P.M. = 50^\circ$$

$$G.M. = \infty$$

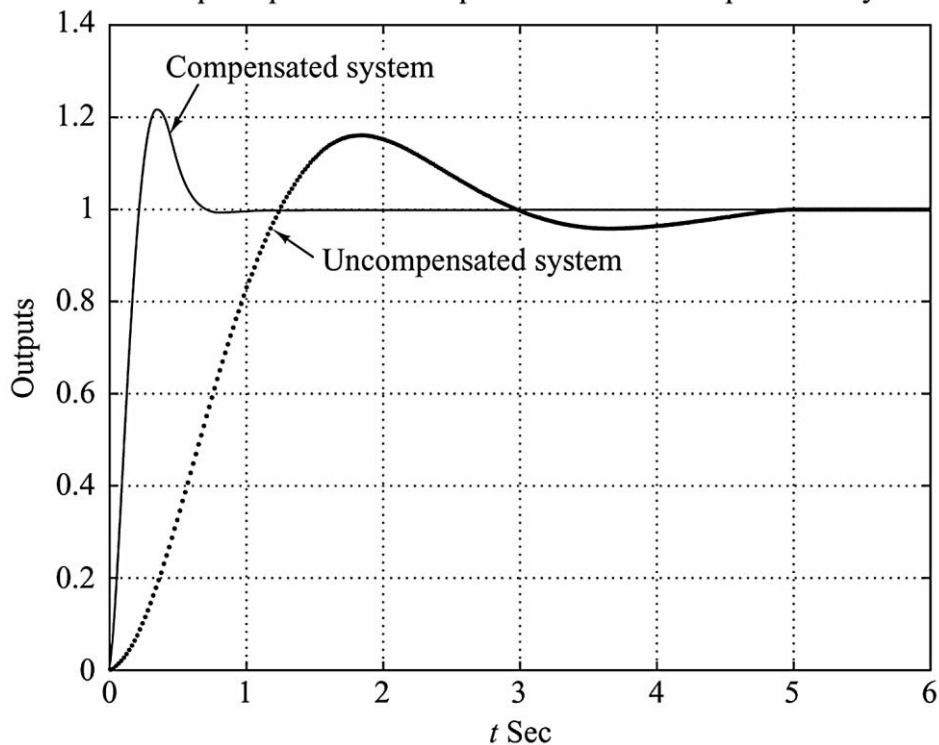


Nyquist plot:

	uncompensated	compensated
M_r	3	1.29
ω_r	6 rad/sec	7 rad/sec
ω_B	6.3 rad/sec	9 rad/sec



Unit-Step Responses of Compensated and Uncompensated Systems

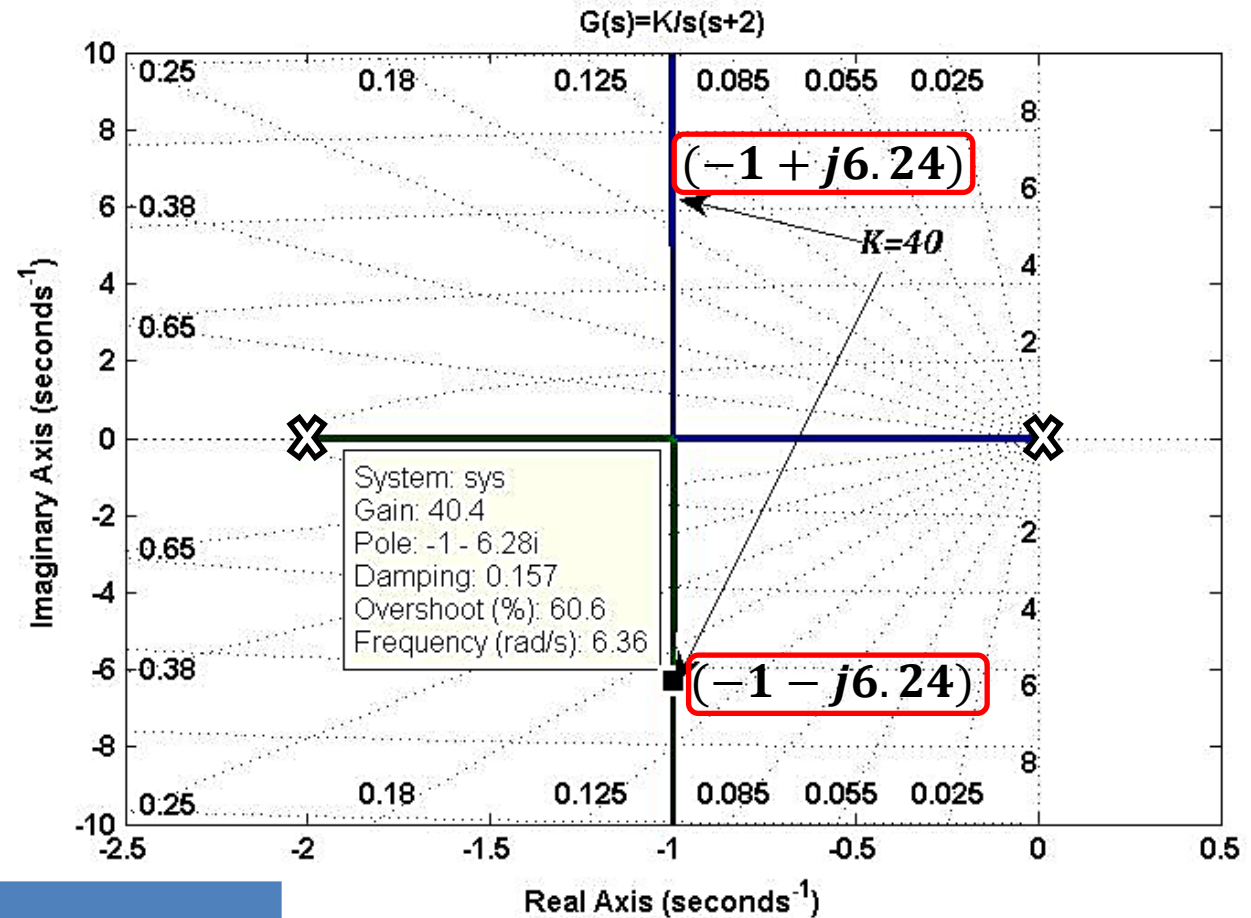


$$\frac{C(s)}{R(s)} = \frac{4}{s^2 + 2s + 4}$$

$$\frac{C(s)}{R(s)} = \frac{166.8s + 735.588}{s^3 + 20.4s^2 + 203.6s + 735.588}$$

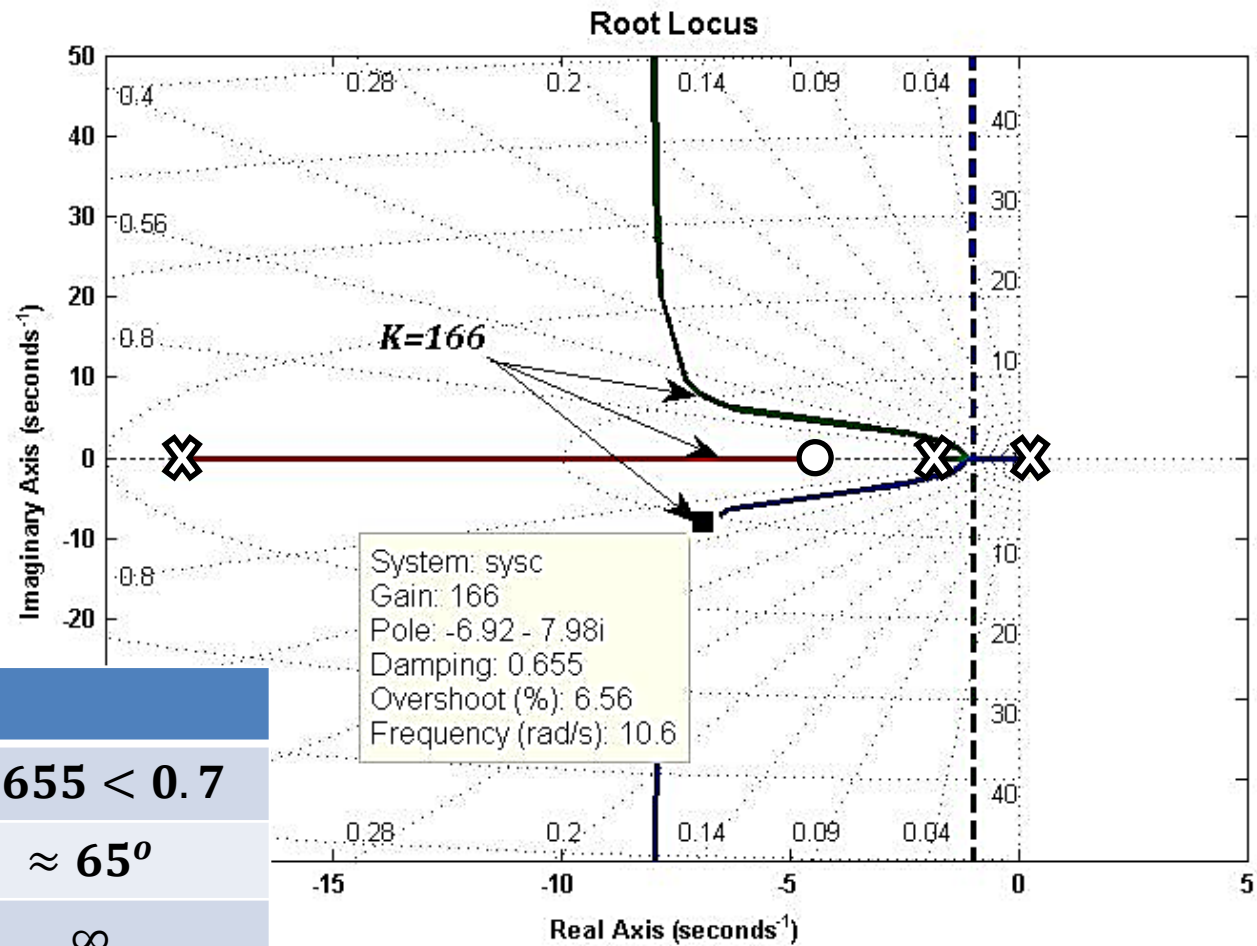
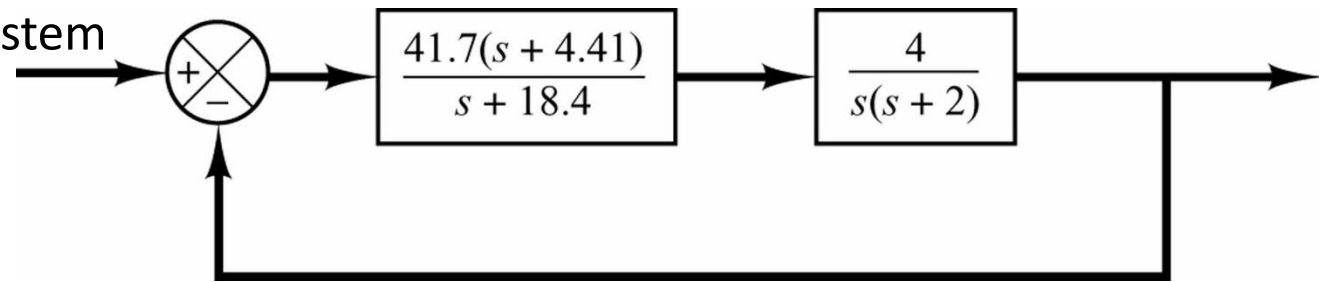
- The original system that satisfies the steady state error

$$KG(s) = \frac{40}{s(s+2)}$$



ξ	$0.157 < 0.7$
$P.M. \cong 100\xi$	$\approx 16^\circ$
$G.M.$	∞
$B.W.$	6.3 rad/sec

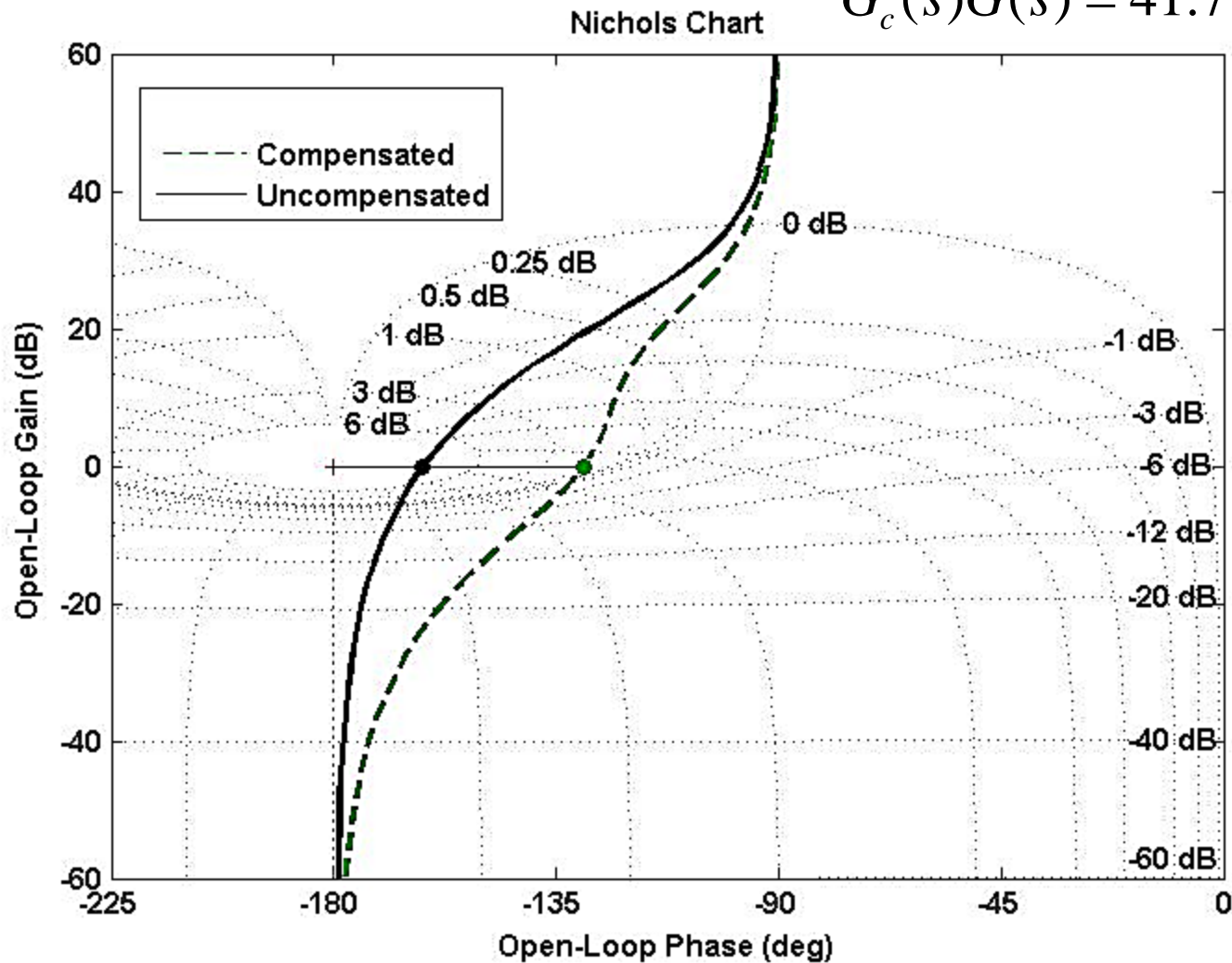
- The compensated system



ξ	$0.655 < 0.7$
$P.M. \cong 100\xi$	$\approx 65^\circ$
$G.M.$	∞
$B.W.$	10.6 rad/sec

- Nichols chart**

$$G_c(s)G(s) = 41.7 \frac{(s + 4.41)}{(s + 18.4)} \frac{4}{s(s + 2)}$$



	uncompensated	compensated
<i>P.M.</i>	$18^\circ @ \omega = 6.17 \text{ rad/sec}$	$50.5^\circ @ \omega = 8.89 \text{ rad/sec}$
<i>G.M.</i>	∞	∞