



Mechanics of Materials II:

Thin-Walled Pressure Vessels and Torsion

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Module 18 Learning Outcome

- Solve for the angle of twist for elastic torsion of a straight cylindrical shaft that is non-prismatic

Elastic torsion of Straight Cylindrical Shafts that are non-prismatic

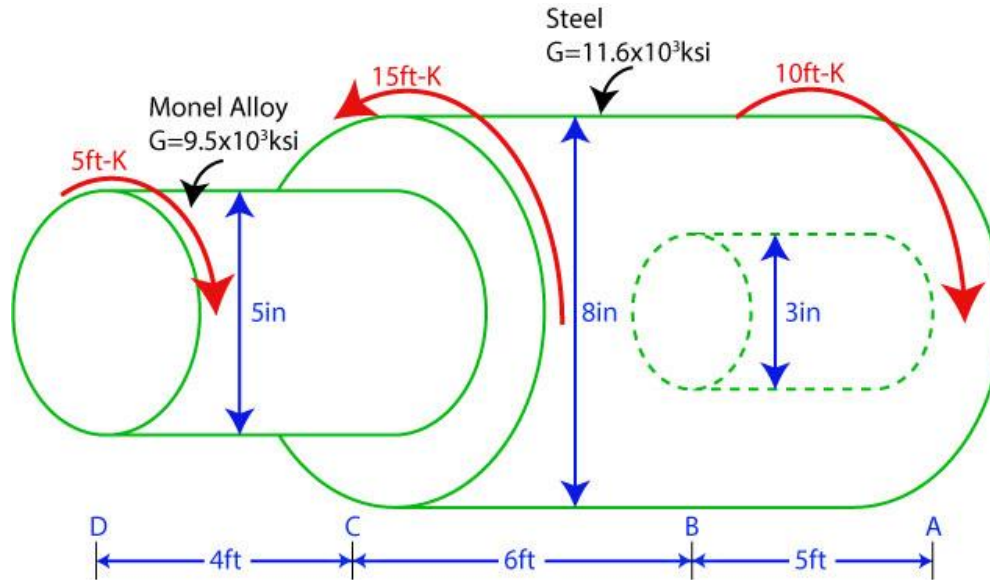
(prismatic is a straight engineering member with the same cross-section throughout its length)

Worksheet:

The non-prismatic cylindrical bar below is subject to torques as shown.

A portion of the steel section is hollow.

- Determine the maximum shear stress in each section.
- Determine the angle of twist of end D with respect to end A.

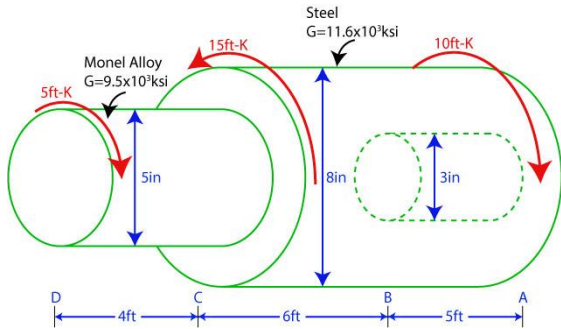


Elastic torsion of Straight Cylindrical Shafts that are non-prismatic (prismatic is a straight engineering member with the same cross-section throughout its length)

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$$\vec{T}_{CD} = 5 \text{ ft} \cdot k \hat{k}$$

$$\vec{T}_{BC} = -10 \text{ ft} \cdot k \hat{k}$$

$$\vec{T}_{AB} = -10 \text{ ft} \cdot k \hat{k}$$

$$\phi_{BC} = \frac{10 \text{ ft} \cdot k \left(\frac{12 \text{ in}}{\text{ft}} \right) 6 \text{ ft} \left(\frac{12 \text{ in}}{\text{ft}} \right)}{11.6 \times 10^3 \text{ ksi} \left[\frac{\pi (4 \text{ in})^4}{2} \right]} = 1.85 \times 10^{-3} \text{ rad } \hat{k}$$

(twist of C wrt B)

$$\phi_{AB} = \frac{10 \text{ ft} \cdot k \left(\frac{12 \text{ in}}{\text{ft}} \right) 5 \text{ ft} \left(\frac{12 \text{ in}}{\text{ft}} \right)}{11.6 \times 10^3 \text{ ksi} \left[\frac{\pi (4 \text{ in})^4}{2} - \frac{\pi (1.5 \text{ in})^4}{2} \right]} = 1.575 \times 10^{-3} \text{ rad } \hat{k}$$

(twist of B wrt A)