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			DIMENSIONS ARE IN INCHES TOLERANCES:		DRAWN	07/31/2025				
			LINEAR: ± 0.001" ANGULAR (MACHINED): ± 0.001° ANGULAR (BENT): ± 0.1°		CHECKED	N/A				
					ENG APPR.	N/A				
					MFG APPR.	N/A				
			GEOMETRIC TOLERANCING INTERPRETED PER ASME Y14.5-2009		Q.A.	N/A				
			MATERIAL ASTM A 106 GRADE B		COMMENTS:					
			FINISH FLANGES - AS MACHINED OTHER - AS FABRICATED	(A) - 1/32" x 45° CHAMFER (B) - 1/16" W x 1/16" DEEP x R4.625" CIRCULAR CX GASKET GROOVE CONCENTRIC WITH DATUM E OR H (C) - 0.100" X 45° CHAMFER (D) WALL THICKNESS UNIFORM		SIZE A	DWG. NO. 1	REV 0001		
		NEXT ASSY	USED ON					SCALE: 1:8	WEIGHT: 56.96 LB	SHEET 1 OF 1
		APPLICATION		DO NOT SCALE DRAWING						

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Project: 4" Flanged Pipe 90° Elbow

Engineer: Alan C. Longfellow

Date: 31 July 2025

Calculation Type: Internal Pressure, Hoop Stress Verification

References: ASME B31.3, Lamé equations

Given:

- Pipe ID = 4.000 in
- Wall thickness (t) = 0.500 in
- Max design internal pressure (P_{inner}) = 5.000 ksi
- External pressure (P_{outer}) = STP = 0.0147 ksi
- Safety factor (FS) = 1.5

Material:

- ASTM A106 Grade B
- Yield strength (σ_y) = 35.0 ksi

Calculated geometry:

- $r_{\text{inner}} = 2.000$ in
- $r_{\text{outer}} = 2.500$ in
- $r_{\text{mean}} = 0.5 \cdot (2.000 \text{ in} + 2.500 \text{ in}) = 2.250$ in

Lamé equation for hoop stress σ_θ in a thick-walled vessel ($r_{\text{outer}}/t < 10$):

$$\sigma_\theta = \frac{P_{\text{inner}} r_{\text{inner}}^2 - P_{\text{outer}} r_{\text{outer}}^2}{r_{\text{outer}}^2 - r_{\text{inner}}^2} - \frac{r_{\text{inner}}^2 r_{\text{outer}}^2 (P_{\text{inner}} - P_{\text{outer}})}{(r_{\text{outer}}^2 - r_{\text{inner}}^2) r_{\text{mean}}^2}$$

Solution for hoop stress:

$$\begin{aligned} \sigma_\theta &= \frac{(5.000 \text{ ksi}) \cdot (2.000 \text{ in})^2 - (0.0147 \text{ ksi}) \cdot (2.500 \text{ in})^2}{(2.500 \text{ in})^2 - (2.000 \text{ in})^2} - \\ &\frac{(2.000 \text{ in})^2 \cdot (2.500 \text{ in})^2 \cdot (5.000 \text{ ksi} - 0.0147 \text{ ksi})}{((2.500 \text{ in})^2 - (2.000 \text{ in})^2) \cdot (2.250 \text{ in})^2} = 19.790 \text{ ksi} \end{aligned}$$

Safety factor verification:

$$\begin{aligned} \sigma_{\text{max}} &= \frac{\sigma_y}{\text{FS}} = \frac{35.0 \text{ ksi}}{1.5} = 23.333 \text{ ksi} \\ \frac{\sigma_\theta}{\sigma_{\text{max}}} &= \frac{23.333 \text{ ksi}}{19.790 \text{ ksi}} = 117.9\% \text{ FS, } \quad \mathbf{ACCEPTABLE} \end{aligned}$$