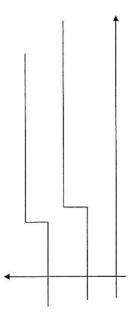
22/2/22

Chapter 4. Design of State-Space Servo-Controllers

4.1 Introducing the Reference Signal

It is desired that the process output vector should follow a specified vector of setpoints:



Consider the SISO process:

$$\dot{\underline{x}} = A\underline{x} + Bu$$

$$v = Cx$$

For a steady state desired output yss we have:

Propose the control-law:

$$u(t) = u_{ss} - K(\underline{x}(t) - \underline{x}_{ss})$$

In the steady state:

$$\underline{O} = A\underline{x}_{ss} + Bu_{ss}$$
$$y_{ss} = C\underline{x}_{ss}$$

Let us propose the simple relationships:

$$u_{ss} = N_u r_{ss} \qquad \underline{x}_{ss} = N_x r_{ss}$$

This implies:

$$\underline{O} = AN_x r_{ss} + BN_u r_{ss}$$

$$y_{ss} = CN_x r_{ss}$$

Of course if we want the steady state output to be rss:

Then the gains N_x and N_u can be determined as:

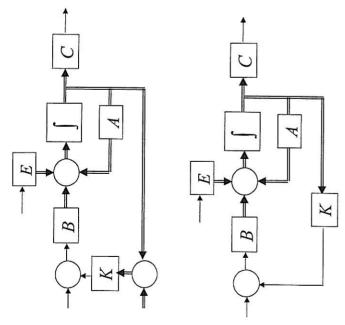
$$\begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} A & B \\ C & 0 \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

The control-law is then:

$$u(t) = N_u r(t) - K \big(x(t) - N_x r(t) \big)$$

Which could of course be rewritten as:

This yields the following control structures:



<u>Tutorial:</u> Introduce a setpoint signal to the motor speed controller developed earlier – test in Simulink.

Some of the problems of this technique include:

- Does not increase the system type:
- Gains designed to reduce ess for setpoint changes:

4.2 State-Space Control with Integral Action

Consider again the SISO process:

$$\underline{\dot{x}}(t) = A\underline{x}(t) + Bu(t)$$
$$y(t) = C\underline{x}(t)$$

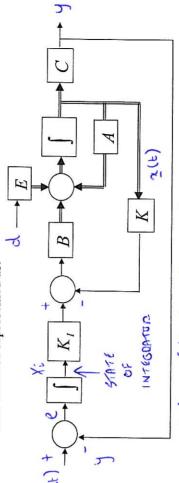
The state control-law with integral action is:

$$u(t) = -K\underline{x}(t) + K_I \int_0^t e(\tau) d\tau$$

where: 2(t)=2(t)-4(t)

we want y(t) to touch setpoint a(t)

This could be represented as:



we have introduced an outer feedback loop

Introduce another state:

$$x_{I}(t) = \int_{0}^{t} e(\tau)d\tau \qquad \text{Output of}$$

CONTROLLED INTEGRATOR

The control-law become:

$$u(t) = -K\underline{x}(t) + K_Ix_I$$

This yields the closed-loop state equation:

$$\underline{\dot{x}}(t) = A\underline{x}(t) + B\left(-K\underline{x}(t) + K_Ix_I(t)\right)$$

But we know:

$$x_I(t) = \int_{0}^{t} e(\tau) d\tau$$

Hence the complete closed loop system can be represented by the coupled equations:

N+(
$$\int \underline{\dot{x}}(t) = (A - BK)\underline{x}(t) + BK_{I}x_{I}(t) \leftarrow \text{Paccess}$$

$$\leq \overline{\dot{x}}(t) = -C\underline{x}(t) + r(t) \qquad \leftarrow \text{contratile}$$

Assign a new state vector:

$$\underline{z}(t) = \left[\frac{\underline{x}(t)}{x_1(t)}\right] \Leftrightarrow \lambda$$

The closed-loop equations can be written more compactly as:

CLOSED
$$\frac{d}{dt} \left[\frac{\chi(t)}{\lambda_{z}} \right] = \begin{bmatrix} \beta - 8\kappa & 8\kappa_{z} \\ -C & O \end{bmatrix} \frac{\chi(t)}{\lambda_{z}} + \begin{bmatrix} O \\ -C \end{bmatrix} r(t)$$
The poles of the closed-loop system are given by the roots of:

$$\det(sI - A_2) = 0$$

N.B. There are (N+1) closed loop pales

Determine the gains to place the N+1 closed-loop poles to obtain the following characteristic equation:

$$C_{des}(s) = s^{(N+1)} + C_N s^N + \cdots C_1 s + C_0$$

Proof of Integral Action: Asseme Joseph Joseph stability

Consider an asymptotically constant setpoint signal: $\lim_{s \neq 0} \lim_{k \to \infty} \lim_{k \to \infty} |k| = |k| = |k|$

Since the closed-loop system is stable – the states must converge to steady-state values:

$$\lim_{t \to \infty} \dot{x}_{z} = O \qquad \Rightarrow \underline{0} = (A - BK)\underline{x}_{ss} + BK_{I}x_{Iss}$$

$$\lim_{t \to \infty} \dot{x}_{z} = O \Rightarrow 0 = -C\underline{x}_{ss} + r_{ss}$$

x(t) + 255

2(4) + R55

y(1)7yss

EXAMPLE: The DC Motor

$$\frac{d}{dt} \begin{bmatrix} \omega(t) \\ i(t) \end{bmatrix} = \begin{bmatrix} 0 & 50 \\ -200 & -200 \end{bmatrix} \begin{bmatrix} \omega(t) \\ i(t) \end{bmatrix} + \begin{bmatrix} 0 \\ 200 \end{bmatrix} v(t) + \begin{bmatrix} -50 \\ 0 \end{bmatrix} T_L(t)$$

$$y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \qquad \text{SPEED CC}$$

$$y(t) = L \qquad \text{speed of } t \in L$$

Use the control-law

$$v(t) = -\left[k_1 \quad k_2 \left[\frac{\omega(t)}{i(t)}\right] + K_I \left[\left(r_o(\tau) - y(\tau)\right)\right] d\tau$$

Hence the closed loop state-equation is:

$$\frac{d}{dt} \begin{bmatrix} \omega(t) \\ \frac{i(t)}{x_I(t)} \end{bmatrix} = \begin{bmatrix} 0 & 50 & 0 \\ -200 - 200k_1 & -200 - 200k_2 \\ -1 & 0 & 0 \end{bmatrix} \begin{bmatrix} \omega(t) \\ i(t) \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ i(t) \\ 1 \end{bmatrix} r(t) + \begin{bmatrix} -50 \\ 0 \\ 0 \end{bmatrix} T_L(t)$$

The poles of the closed-loop system are given by roots of:

$$\det\begin{bmatrix} \begin{bmatrix} s & 0 & 0 \\ 0 & s & 0 \end{bmatrix} & \begin{bmatrix} & \theta_2 \\ 0 & & 50 \\ 0 & s & 0 \end{bmatrix} & \begin{bmatrix} & 0 \\ -200 - 200k_1 & -200 - 200k_2 \\ & & & 1 \end{bmatrix} = 0$$

which yields:

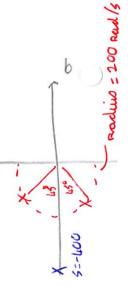
order

$$\det \left[\begin{bmatrix} s & -50 & 0 \\ 200 + 200k_1 & s + 200 + 200k_2 & -200K_t \\ 1 & + & 0 & - & s + 1 \end{bmatrix} \right] = 0$$

the closed-loop characteristic equation is then:

This can be achieved by placing the controller pole further out left:

CONTROLLER
POLE 15
FURTHER BUT
LEFT



The desired closed-loop characteristic equation is:

FAST SLOW
$$C_{des}(s) = (s + 400)(s^2 + 282.8s + 40000)$$

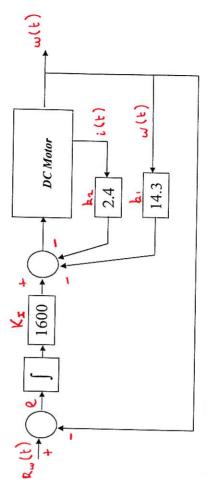
$$= s^3 + 682.8s^2 + 153120s + 16000000$$

Compare with the closed-loop characteristic equation:

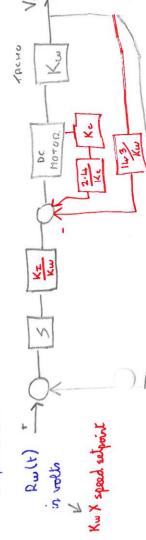
This yields the controller:

$$v(t) = -14.3\omega(t) - 2.4i(t) + 1600 \int_{0}^{t} (r_{o}(\tau) - \omega(\tau)) d\tau$$

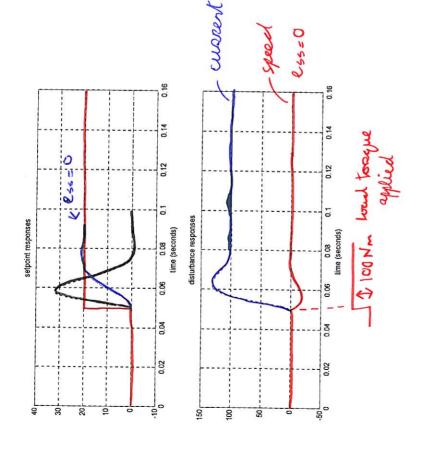
which could be built as follows:



In pareliee



The closed-loop responses are:



4.2.1 Use of Ackermans Method to Design Controllers with Integral Action

Define the open-loop equations as:

$$\dot{\underline{x}}(t) = A\underline{x}(t) + Bu(t)$$
$$\dot{x}_I(t) = r(t) - C\underline{x}(t)$$

Or in more compact form as:

$$\frac{\partial PEN}{\partial Q} = \frac{1}{2} \left[\begin{array}{c|c} & & & \\ & & & \\ & & & \\ \hline & &$$

 $\frac{Z}{z} = A_2 \frac{Z}{z} + B_2 u + G_R$ Which is under the control:

$$u(t) = -K\underline{x}(t) + K_I x_I$$
 $u = -k_2 \mathbf{y}(t)$

or:

$$u(t) = -\left[K \mid -K_I\right]\underline{z}(t)$$

We can then use Ackermann's formula:

$$k_{z} = [K \mid -K_{I}] = [0 \quad 0 \quad \cdots \quad 0 \quad 1]C_{z}^{-1}C_{des}(A_{z})$$

where:

$$C_{des}(s) = s^{N+1} + C_N s^N + \cdots C_1 s + C_0 = 0$$
 $\beta_2 =$

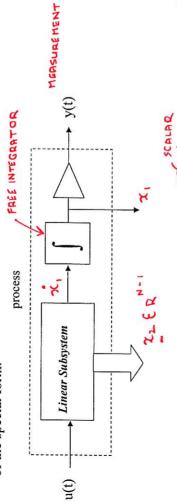
and:
$$C_2 = \begin{bmatrix} B_2 & A_2B_2 & A_2^2B_2 & \cdots & A_2^NB_2 \end{bmatrix}$$

Use of Inherent Integral Action 4.2.2

Consider the SISO process model:

$$\frac{\dot{x}}{\dot{x}} = A\underline{x} + Bu$$
$$y = C\underline{x}$$

of the special form:



 $\underline{X}(t) = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$ The state of the Nth order process is then:

The output is: $y(t) = [c_1 \mid 0 \quad 0 \quad \dots \quad 0] \underline{x}(t)$

We want the output y(t) to track the setpoint r(t). Consider that

Consider the standard tracking controller structure:

$$u(t) = \underbrace{N_u r(t)}_{\mathbf{M}_{\mathbf{X}}(t)} - K(\underline{x}(t) - [N_{\mathbf{X}}r(t)])$$

with the choice:

The control-law becomes:

$$u(t) = -K \left(\frac{1/c_1}{2} \right)$$

$$u(t) = -K \left(\frac{1/c_1}{2} \right)$$

$$uib cop_{x_1}(t)$$

$$uib cop_{x_2}(t)$$

$$V = K$$

$$V = \frac{x_1 + C_2}{2}$$

$$V = \frac{x_2 + C_2}{2}$$

$$V = \frac{x_1 + C_2}{2}$$

Or:

Which could be written as: $\frac{y(t)}{c}$

 $u(t) = -K \begin{vmatrix} \frac{y(t) - r(t)}{x_1^2(t)} \\ \vdots \\ \vdots \\ \frac{k_1}{z_1} \end{vmatrix} K_1 \left[\frac{y(t) - r(t)}{\underline{x}_1(t)} \right]$ is team of output soften x_1 New Kmolbux

This could be represented by: = Ry (B-y)-K2 22

