EE4002 Control Engineering

B) State-Space Control

Chapter 1. The State Space Modelling Approach

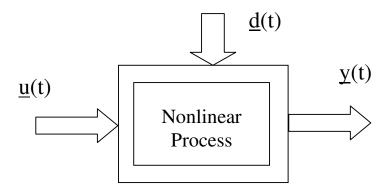
1.1 The State Space Model

Transformation of an Nth order multi-input-multi-output system to N first order differential equations:

Benefits

- Robust method of simulating high order differential equations
- Easy analysis of dynamics
- Allows for use of model reduction methods
- Can easily apply advanced control
- Used for estimation

Consider the Nth order nonlinear dynamical process:

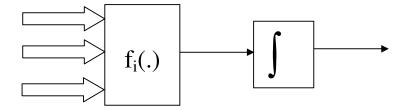


In general this process could be represented by a model consisting of :

It is possible to transform this set of coupled nonlinear differential equations to yield a set of N first order differential equations with states, $\{x_1(t), x_2(t),x_N(t)\}$.

The differential equation describing the dynamics of the i^{th} state $x_i(t)$ can be written as:

Which could be represented by the following simulation diagram:



The N first order differential equations could be written as:

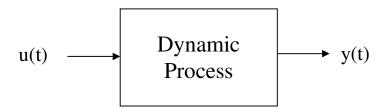
$$\frac{d}{dt} \begin{bmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_N(t) \end{bmatrix} = \dot{\underline{x}}(t) = \begin{bmatrix} f_1(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \\ f_2(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \\ \vdots \\ f_N(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \end{bmatrix}$$

The output $\underline{y}(t)$ is then generated by:

$$\underline{y}(t) = \begin{bmatrix} y_1(t) \\ y_2(t) \\ \vdots \\ y_P(t) \end{bmatrix} = \begin{bmatrix} h_1(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \\ h_2(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \\ \vdots \\ h_P(\underline{x}(t), \underline{u}(t), \underline{d}(t)) \end{bmatrix}$$

1.1.1 Some Example State-Space Systems:

i) Third Order Linear Process



Modelled by the following differential equation:

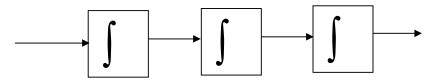
$$\frac{d^3}{dt^3}y(t) + 5\frac{d^2}{dt^2}y(t) + 3\frac{d}{dt}y(t) + y(t) = u(t)$$

Which could of course be represented by the transfer function model:

$$U(s) \longrightarrow Y(s)$$

Rearrange to yield an expression for the highest derivative:

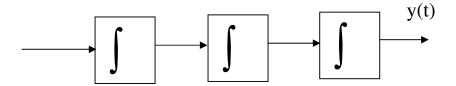
Form lower derivatives then by integration:



But we know that:

$$\ddot{y}(t) = u(t) - 5\ddot{y}(t) - 3\dot{y}(t) - y(t)$$

This yields the following simulation diagram:



Assign the state variables as the outputs of integrators:

We can now specify the state vector for the process as:

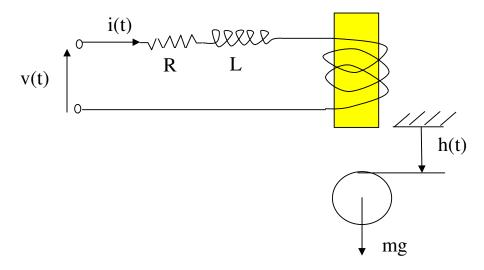
Now this could be written in matrix form as:

$$\frac{d}{dt} \begin{bmatrix} x_1(t) \\ x_2(t) \\ x_3(t) \end{bmatrix} = \begin{bmatrix} x_1(t) \\ x_2(t) \\ x_3(t) \end{bmatrix} + \begin{bmatrix} u(t) \\ u(t) \end{bmatrix}$$

$$y(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \\ x_3(t) \end{bmatrix}$$

ii) Third Order Nonlinear Process

Consider the following magnetic suspension system:



Can be modelled by:

$$m\frac{d^2h(t)}{dt^2} = mg - \frac{Ki^2(t)}{h^2(t)}$$

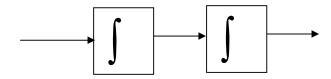
$$L\frac{di(t)}{dt} = v(t) - Ri(t)$$

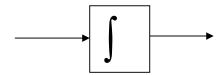
Rewritten as:

$$\frac{d^2h(t)}{dt^2} = g - \frac{Ki^2(t)}{mh^2(t)}$$

$$\frac{di(t)}{dt} = \frac{1}{L} (v(t) - Ri(t))$$

Could be represented by the following simulation diagram:





Note that three integrators are required – system is 3^{rd} order Now can arbitrarily assign the three states $\{x_1(t), x_2(t), x_3(t)\}$ And define the state vector:

The state-equations are now:

$$\dot{x}_1 = x_2(t)$$

$$\dot{x}_2 = g - \frac{Kx_3^2(t)}{mx_1^2(t)}$$

$$\dot{x}_3 = \frac{1}{L} (u(t) - Rx_3(t))$$

with output equation:

$$y(t) = x_1(t)$$

1.2 Derivation of the Linear State Space Model - by Linearisation

Consider first the linearisation of a multivariate function:

$$z = g(\underline{w}) \quad where \quad \underline{w} = \begin{bmatrix} w_1 \\ \vdots \\ w_N \end{bmatrix}$$

about the operating point vector:

$$\underline{w}_0 = \begin{bmatrix} w_1' \\ \vdots \\ w_N' \end{bmatrix}$$

Use the first order Taylor's series approximation:

$$z \approx g(\underline{w}_0) + \frac{\partial g}{\partial w_1}\bigg|_{w=w_0} (w_1 - w_1') + \frac{\partial g}{\partial w_2}\bigg|_{w=w_0} (w_2 - w_2') + \cdots + \frac{\partial g}{\partial w_N}\bigg|_{w=w_0} (w_N - w_N')$$

which could be written as:

$$(z-z_0) = \begin{bmatrix} \frac{\partial g}{\partial w_1} \Big|_{\underline{w}_0} & \frac{\partial g}{\partial w_2} \Big|_{\underline{w}_0} & \cdots & \frac{\partial g}{\partial w_N} \Big|_{\underline{w}_0} \end{bmatrix} (\underline{w} - \underline{w}_0)$$

Now for simplicity first consider a linearisation of the ith state equation:

$$\frac{dx_i(t)}{dt} = f_i(\underline{x}(t))$$

Could be linearised to yield:

$$(z-z_0) = \left[\frac{\partial f_i}{\partial x_1}\bigg|_{\underline{x}_0} \quad \frac{\partial f_i}{\partial x_2}\bigg|_{\underline{x}_0} \quad \cdots \quad \frac{\partial f_i}{\partial x_N}\bigg|_{\underline{x}_0}\right] (\underline{x}-\underline{x}_0)$$

But:

Hence the linearised equation becomes:

$$\frac{d}{dt}\Delta x_i(t) = \left[\frac{\partial f_i}{\partial x_1}\bigg|_{\underline{x}_0} \quad \frac{\partial f_i}{\partial x_2}\bigg|_{\underline{x}_0} \quad \cdots \quad \frac{\partial f_i}{\partial x_N}\bigg|_{\underline{x}_0}\right] \Delta \underline{x}(t)$$

Now let us expand to include input and disturbance vector:

$$\frac{dx_i(t)}{dt} = f_i(\underline{x}(t), \underline{u}(t), \underline{d}(t))$$

Which will have the linearisation about the operating point:

$$\frac{d}{dt}\Delta x_{i}(t) = \left[\frac{\partial f_{i}}{\partial x_{1}}\Big|_{op} \frac{\partial f_{i}}{\partial x_{2}}\Big|_{op} \cdots \frac{\partial f_{i}}{\partial x_{N}}\Big|_{op}\right] \Delta \underline{x}(t) + \left[\frac{\partial f_{i}}{\partial u_{1}}\Big|_{op} \frac{\partial f_{i}}{\partial u_{2}}\Big|_{op} \cdots \frac{\partial f_{i}}{\partial u_{m}}\Big|_{op}\right] \Delta \underline{u}(t) + \left[\frac{\partial f_{i}}{\partial d_{1}}\Big|_{op} \frac{\partial f_{i}}{\partial d_{2}}\Big|_{op} \cdots \frac{\partial f_{i}}{\partial d_{s}}\Big|_{op}\right] \Delta \underline{d}(t)$$

which could further be written as:

$$\frac{d}{dt}\Delta x_i(t) = \begin{bmatrix} a_{i1} & a_{i2} & \cdots & a_{iN} \end{bmatrix} \Delta \underline{x}(t) + \begin{bmatrix} b_{i1} & b_{i2} & \cdots & b_{im} \end{bmatrix} \Delta \underline{u}(t) + \begin{bmatrix} e_{i1} & e_{i2} & \cdots & e_{is} \end{bmatrix} \Delta \underline{d}(t)$$

This could of course be repeated for all N state equations:

$$\frac{d}{dt} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \vdots \\ \Delta x_i \\ \vdots \\ \Delta x_N \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1N} \\ a_{21} & a_{22} & \cdots & a_{2N} \\ \vdots & \vdots & \ddots & \vdots \\ a_{i1} & a_{i2} & \cdots & a_{iN} \\ \vdots & \vdots & \ddots & \vdots \\ a_{N1} & a_{N2} & \cdots & a_{NN} \end{bmatrix} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \vdots \\ \Delta x_i \\ \vdots \\ \Delta x_N \end{bmatrix} + \begin{bmatrix} b_{11} & b_{12} & \cdots & b_{1m} \\ b_{21} & b_{22} & \cdots & b_{2m} \\ b_{21} & b_{22} & \cdots & b_{2m} \\ \vdots & \vdots & \ddots & \vdots \\ b_{i1} & b_{i2} & \cdots & b_{im} \\ \vdots & \vdots & \ddots & \vdots \\ b_{N1} & b_{N2} & \cdots & b_{Nm} \end{bmatrix} \begin{bmatrix} \Delta u_1 \\ \Delta u_2 \\ \vdots \\ \Delta u_m \end{bmatrix}$$

$$+ \begin{bmatrix} e_{11} & e_{12} & \cdots & e_{1s} \\ e_{21} & e_{22} & \cdots & e_{2s} \\ \vdots & \vdots & \ddots & \vdots \\ e_{i1} & e_{i2} & \cdots & e_{is} \\ \vdots & \vdots & \ddots & \vdots \\ e_{N1} & e_{N2} & \cdots & e_{Ns} \end{bmatrix} \begin{bmatrix} \Delta d_1 \\ \Delta d_2 \\ \vdots \\ \Delta d_s \end{bmatrix}$$

Or in more compact form as:

Now consider the static output equation for the ith output:

$$y_i(t) = h_i(x(t), u(t), d(t))$$

The linearisation about the operating point $\{\underline{x}_0,\underline{u}_0,\underline{d}_0\}$ is:

$$\Delta y_{i}(t) = \left[\frac{\partial h_{i}}{\partial x_{1}} \Big|_{op} \frac{\partial h_{i}}{\partial x_{2}} \Big|_{op} \cdots \frac{\partial h_{i}}{\partial x_{N}} \Big|_{op} \right] \Delta \underline{x}(t) + \left[\frac{\partial h_{i}}{\partial u_{1}} \Big|_{op} \frac{\partial h_{i}}{\partial u_{2}} \Big|_{op} \cdots \frac{\partial h_{i}}{\partial u_{m}} \Big|_{op} \right] \Delta \underline{u}(t) + \left[\frac{\partial h_{i}}{\partial d_{1}} \Big|_{op} \frac{\partial h_{i}}{\partial d_{2}} \Big|_{op} \cdots \frac{\partial h_{i}}{\partial d_{s}} \Big|_{op} \right] \Delta \underline{u}(t)$$

which could be written as:

$$\Delta y_i(t) = \begin{bmatrix} c_{i1} & c_{i2} & \cdots & c_{iN} \end{bmatrix} \Delta \underline{x}(t) + \begin{bmatrix} d_{i1} & d_{i2} & \cdots & d_{im} \end{bmatrix} \Delta \underline{u}(t) + \begin{bmatrix} f_{i1} & f_{i2} & \cdots & f_{is} \end{bmatrix} \Delta \underline{d}(t)$$

This could be repeated for all the P outputs:

$$\begin{bmatrix} \Delta y_1 \\ \Delta y_2 \\ \vdots \\ \Delta y_i \\ \vdots \\ \Delta y_P \end{bmatrix} = \begin{bmatrix} c_{11} & c_{12} & \cdots & c_{1N} \\ c_{21} & c_{22} & \cdots & c_{2N} \\ \vdots & \vdots & \ddots & \vdots \\ c_{i1} & c_{i2} & \cdots & c_{iN} \\ \vdots & \vdots & \ddots & \vdots \\ c_{P1} & c_{P2} & \cdots & c_{PN} \end{bmatrix} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \vdots \\ \Delta x_i \\ \vdots \\ \Delta x_N \end{bmatrix} + \begin{bmatrix} d_{11} & d_{12} & \cdots & d_{1m} \\ d_{21} & d_{22} & \cdots & d_{2m} \\ \vdots & \vdots & \ddots & \vdots \\ d_{i1} & d_{i2} & \cdots & d_{im} \\ \vdots & \vdots & \ddots & \vdots \\ d_{P1} & d_{P2} & \cdots & d_{Pm} \end{bmatrix} \begin{bmatrix} \Delta u_1 \\ \Delta u_2 \\ \vdots \\ \Delta u_m \end{bmatrix}$$

$$+\begin{bmatrix} f_{11} & f_{12} & \cdots & f_{1s} \\ f_{21} & f_{22} & \cdots & f_{2s} \\ \vdots & \vdots & \ddots & \vdots \\ f_{i1} & f_{i2} & \cdots & f_{is} \\ \vdots & \vdots & \ddots & \vdots \\ f_{P1} & f_{P2} & \cdots & f_{Ps} \end{bmatrix} \begin{bmatrix} \Delta d_1 \\ \Delta d_2 \\ \vdots \\ \Delta d_s \end{bmatrix}$$

which again could be written in more compact form as:

This yields the following linear state-space model for, a process about a particular operating point:

$$\frac{d}{dt}\underline{x}(t) = A\underline{x}(t) + B\underline{u}(t) + E\underline{d}(t)$$

$$\underline{y}(t) = C\underline{x}(t) + D\underline{u}(t) + F\underline{d}(t)$$

1.2.1 Linearisation Examples:

i) The magnetic suspension system

$$m\frac{d^2h(t)}{dt^2} = mg - \frac{Ki^2(t)}{h^2(t)}$$

$$L\frac{di(t)}{dt} = v(t) - Ri(t)$$

Find a linear model about the desired operating airgap h=0.01m

The process parameters are:

L=10mH M=0.05 Kg g=10ms $^{-2}$ R=1 ohm K=2x10 $^{-4}$ Nm 2 /A 2 First find the operating point,

From the force equation:

$$m\frac{d^2h(t)}{dt^2} = mg - \frac{Ki^2(t)}{h^2(t)} =$$

From the electrical equation:

$$L\frac{di(t)}{dt} = v(t) - Ri(t) =$$

The operating point vector will now be defined as:

$$\underline{x}_0 = \begin{bmatrix} h_0 \\ \dot{h}_0 \\ i_0 \end{bmatrix} = \begin{bmatrix} & & \\ & & \end{bmatrix}$$

Now linearise about this operating point vector:

The state equations can be written as:

$$\dot{x}_1 = x_2(t)$$

$$\dot{x}_2 = g - \frac{Kx_3^2(t)}{mx_1^2(t)}$$

$$\dot{x}_3 = \frac{1}{L} (u(t) - Rx_3(t))$$

Now define deviations of the states from their operating point values as:

Hence the linearised model is:

$$\frac{d}{dt}\Delta x_1(t) = \frac{\partial f_1}{\partial x_2} \bigg|_{OP} \Delta x_2(t)$$

$$\frac{d}{dt}\Delta x_2(t) = \frac{\partial f_2}{\partial x_1} \bigg|_{OP} \Delta x_1(t) + \frac{\partial f_2}{\partial x_3} \bigg|_{OP} \Delta x_3(t)$$

$$\frac{d}{dt}\Delta x_3(t) = \frac{\partial f_3}{\partial x_3} \bigg|_{OP} \Delta x_3(t) + \frac{\partial f_2}{\partial u} \bigg|_{OP} \Delta u(t)$$

But we know that:

$$f_1(x_2) = x_2$$

$$f_2(x_1, x_3) = g - \frac{Kx_3^2}{Mx_1^2}$$

$$f_3(x_3, u) = \frac{u}{L} - \frac{R}{L}x_3$$

This yields the following linearised model about the operating point:

$$\frac{d}{dt}\Delta x_1(t) = \Delta x_2(t)$$

$$\frac{d}{dt}\Delta x_2(t) =$$

$$\frac{d}{dt}\Delta x_3(t) = -100\Delta x_3(t) + 100\Delta u(t)$$

And of course:

$$\Delta y(t) = h(t) - h_0 = \Delta x_1(t)$$

This then could be written in matrix form as:

$$\frac{d}{dt} \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \Delta x_3 \end{bmatrix} = \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \Delta x_3 \end{bmatrix} + \begin{bmatrix} \Delta u(t) \\ \Delta x_2 \\ \Delta x_3 \end{bmatrix}$$

$$\Delta y(t) = \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \\ \Delta x_3 \end{bmatrix}$$

ii) Permanent magnet DC motor

Could be modelled by the following coupled equations:

$$\frac{di}{dt} = \frac{1}{L}(v(t) - Ri(t) - K_m \omega(t))$$

$$\frac{d\omega}{dt} = \frac{1}{J}(K_M i(t) - B\omega(t) - T_L(t))$$

As you can see these are linear differential equations:

If a tachometer of gain K_T V/rads⁻¹ is used to measure the speed, then the output equation could be written as:

Consider now that the motor is driving a nonlinear fan load:

$$T_L(t) = K_F \omega^2(t)$$

The process then would be modelled as:

$$\frac{di}{dt} = \frac{1}{L} (v(t) - Ri(t) - K_m \omega(t))$$

$$\frac{d\omega}{dt} = \frac{1}{J} (K_M i(t) - B\omega(t) - K_F \omega^2(t))$$

Generate a linear state-space model which describes the dynamics of this process close to the operating speed:

First find the operating point:

$$\frac{d\omega}{dt} = \frac{1}{J} (K_M i(t) - B\omega(t) - K_F \omega^2(t)) =$$

Now from the electrical equation:

$$\frac{di}{dt} = \frac{1}{L}(v(t) - Ri(t) - K_m \omega(t)) =$$

Assign the states:
$$\underline{x}(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} i(t) \\ \omega(t) \end{bmatrix}$$

and input and output: u(t) = v(t) and $y(t) = v_T(t) = K_T \omega(t)$

hence:

$$\dot{x}_1(t) = \frac{1}{L}(u(t) - Rx_1(t) - K_m x_2(t)) = f_1(x_1, x_2, u)$$

$$\dot{x}_2(t) = \frac{1}{J}(K_M x_1(t) - Bx_2(t) - K_F x_2^2(t)) = f_2(x_1, x_2)$$

$$y(t) = K_T x_2(t) = h(x_2)$$

The linearised model about the operating point is:

$$\frac{d}{dt}\Delta x_{1}(t) = \frac{\partial f_{1}}{\partial x_{1}}\Big|_{OP} \Delta x_{1}(t) + \frac{\partial f_{1}}{\partial x_{2}}\Big|_{OP} \Delta x_{2}(t) + \frac{\partial f_{1}}{\partial u}\Big|_{OP} \Delta u(t)$$

$$\frac{d}{dt}\Delta x_{2}(t) = \frac{\partial f_{2}}{\partial x_{1}}\Big|_{OP} \Delta x_{1}(t) + \frac{\partial f_{2}}{\partial x_{2}}\Big|_{OP} \Delta x_{2}(t)$$

$$\Delta y(t) = \frac{\partial h}{\partial x_{2}}\Big|_{OP} \Delta x_{2}(t)$$

Obviously the electrical equation is linear:

$$\dot{x}_1(t) = \frac{1}{L}(u(t) - Rx_1(t) - K_m x_2(t)) = f_1(x_1, x_2, u)$$

Now concentrating on the mechanical equation:

$$\dot{x}_2(t) = \frac{1}{J} (K_M x_1(t) - Bx_2(t) - K_F x_2^2(t)) = f_2(x_1, x_2)$$

And simply for the output equation:

$$y(t) = K_T x_2(t) = h(x_2)$$

This yields the following linear state-space model: