



# PSV SIZING USING DWSIM: A FREE AND OPEN-SOURCE CHEMICAL PROCESS SIMULATOR



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## PREFACE

The manual “Process Simulation Using DWSIM” presents a set of PSV Sizing exercises using a free and open-source chemical process simulator “DWSIM” and can be utilized to establish process simulation laboratory as part of undergraduate chemical engineering degree or in allied degree curriculum. The problem statements are of intermediate level of some advanced theories using API 520 and API 521.

### Prerequisite

- Must know about DWSIM UI/UX.
- Flow sheeting in DWSIM
- Selection of Thermodynamic Packages.
- Manipulating variables

Thanks

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P.E. O&G

### Disclaimer

All the exercises are strictly restricted to learning only and not meant to be used in real world application.



## PROCESS SIMULATION USING DWSIM: A FREE AND OPEN-SOURCE CHEMICAL PROCESS SIMULATOR

### PREAMBLE

DWSIM is an open-source CAPE-OPEN compliant chemical process simulator. It features a Graphical User Interface (GUI), advanced thermodynamics calculations, reactions support and petroleum characterization / hypothetical component generation tools. DWSIM can simulate steady-state, vapor–liquid, vapor–liquid-liquid, solid–liquid and aqueous electrolyte equilibrium processes and has built-in thermodynamic models and unit operations (<https://en.wikipedia.org/wiki/DWSIM>). It is available for Windows, Linux and Mac OS.

The objective of the course is to create awareness of the open-source process simulator “DWSIM” among prospective graduates and practicing process engineers. The course will cover Intermediate aspects of create flow sheet in DWSIM and simulation of simple units such as Mixer, Splitters, CSTR, Distillation column, Pumps, Turbines, Compressors, etc.

### Target Audience

- Junior Interns in Process Firms
- III / Final year B. Tech. Chemical Engineering students
- M. Tech. Chemical Engineering students
- Practicing Process Engineers



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## 1 BACKGROUND

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Pressure relief valves or other relieving devices are used to protect piping and equipment against excessive over-pressure. Proper selection, use, location, and maintenance of relief devices are essential to protect personnel and equipment as well as to comply with codes and laws.

Determination of the maximum relief required may be difficult. Loads for complex systems are determined by conservative assumptions and detailed analysis. By general assumption, two unrelated emergency conditions caused by unrelated equipment failures or operator error will not occur simultaneously (no double jeopardy). The sequence of events must be considered. The development of relief loads requires the engineer to be familiar with overall process design, including the type of pump drives used, cooling water source, spares provided, plant layout, instrumentation, and emergency shut-down philosophy.

This section suggests methods to calculate relief capacity for most emergency conditions, including fire. A common reference for determining individual relieving rates is contained in Section 3 of API RP 521.1 The design of the proper relieving device must take into consideration all of the following upset conditions for the individual equipment item if such upset can occur. Each upset condition must be carefully evaluated to determine the "worst case" condition which will dictate the relieving device capacity.

### *Blocked Discharge*

The outlet of almost any vessel, pump, compressor, fired heater, or other equipment item can be blocked by mechanical failure or human error. In this case, the relief load is usually the maximum flow which the pump, compressor, or other flow source produces at relief conditions.

### *Fire Exposure*

Fire is one of the least predictable events which may occur in a gas processing facility but is a condition that may create the greatest relieving requirements. If fire can occur on a plant-wide basis, this condition may dictate the sizing of the entire relief system; however, since equipment may be dispersed geographically, the effect of fire exposure on the relief system may be limited to a specific plot area. Vapor generation will be higher in any area which contains a large number of uninsulated vessels. Various empirical equations have been developed to determine relief loads from vessels exposed to fire. Formula selection varies with the system and fluid considered. Fire conditions may overpressure vapor-filled, liquid-filled, or mixed-phase systems.

### *Tube Rupture*

When a large difference exists between the design pressure of the shell and tube sides of an exchanger (usually a ratio of 1.5 to 1 or greater), provisions are required for relieving the low-pressure side. Normally, for design, only one tube is considered to rupture. Relief volume for one tube rupture can be calculated using appropriate sizing equations in this section. When a cool media contacts a hot stream, the effects of flashing should be considered. Also, the possibility of a transient over-pressure caused by the sudden release of vapor into an all-liquid system should be considered.

*Control Valve Failure*

The failure positions of instruments and control valves must be carefully evaluated. In practice, the control valve may not fail in the desired position. A valve may stick in the wrong position, or a control loop may fail. Relief protection for these factors must be provided. Relief valve sizing requirements for these conditions should be based on flow coefficients (manufacturer data) and pressure differentials for the specific control valves and the facility involved.

*Thermal Expansion*

If isolation of a process line on the cold side of an exchanger can result in excess pressure due to heat input from the warm side, then the line or cold side of the exchanger should be protected by a relief valve. If any equipment item or line can be isolated while full of liquid, a relief valve should be provided for thermal expansion of the contained liquid. Low process temperatures, solar radiation, or changes in atmospheric temperature can necessitate thermal protection. Flashing across the relief valve needs to be considered.

*Utility Failure*

Loss of cooling water may occur on an area-wide or plant-wide basis. Affected are fractionating columns and other equipment utilizing water cooling. Cooling water failure is often the governing case in sizing flare systems. Electric power failure, similar to cooling water failure, may occur on an area-wide or plant-wide basis and may have a variety of effects. Since electric pump and air cooler fan drives are often employed in process units, a power failure may cause the immediate loss of reflux to fractionators. Motor driven compressors will also shut down. Power failures may result in major relief loads. Instrument air system failure, whether related to electric power failure or not, must be considered in sizing of the flare system since pneumatic control loops will be interrupted. Also control valves will assume the position as specified on "loss of air" and the resulting effect on the flare system must be considered.

## 2 PSV SIZING

---

### Objective

Find the appropriate orifice areas of the PSV for the given problem statement

### Data

- Protected equipment: Reboiler
- Relief service: Steam
- Reason for relief: Blocked steam discharge
- Relieving rate: 10000 kg/hr.
- Gas Density: 2.5 kg/m<sup>3</sup>
- Ratio of specific heats for gas ( $C_p/C_v$ ): 1.3
- Compressibility factor: 1.1
- Molecular weight of gas = 250 °C
- Set pressure: 5 barg
- Accumulation: 10%
- Back pressure at relief valve discharge: 0.5 barg
- Type of relief valve: Conventional pressure relief valve
- Inlet stream 2: 80 mol % Methanol solutions flowing at 10 kmol/h
- Both the streams are at 30 °C and at 1 bar pressure
- The liquid streams can be considered as ideal

### DWSIM Blocks Used

- Valve
- Material Stream
- Indicators (Digital or Analog)

### Procedure

1. Start a new DWSIM Simulation (DWSIM VER 8.0 - CLASSIC UI). Click on “New steady state Simulation” as a template for new simulation
2. The simulation configuration window will be opened. It shows a specification page. Add components required to solve the problem statement. In the present case, add water. Ensure all components are added from the same property database. For instance, in this case, both components are added from “ChemSep” database.
3. Specify the thermodynamic package as Steam table (IAPWS-IF97).
4. Customize the system of units for the simulation and click “Next”.
5. The flow sheeting section of simulation window will be opened. First, let provide input and output streams for the unit operation to be performed. Drag and drop two Material streams available at the right, in the object palette. Rename them stream as “Inlet-Stream-1” and “Inlet-Stream-2”. These serve as input streams.
6. On clicking the “Inlet Stream” stream, general information about the stream will be displayed on the left side of screen. Specify the feed compositions, temperature, and



pressure for the inlet streams. Once credentials are specified for the inlet streams, the color of stream turns blue.

**Inlet stream (Material Stream)**

**Information** | Connections

**General Info**

Object: Inlet stream

Status: Calculated (20-08-2022 14:40:22)

Linked to:

**Property Package Settings**

Property Package: Steam Tables (IAPWS-IF97) (1)

**Input Data** | Results | Annotations | Dynamics | Floating Tables

**Stream Conditions** | Compound Amounts

Flash Spec: Temperature and Pressure (TP)

Temperature: 250 C

Pressure: 6.01325 bar

Mass Flow: 10000 kg/h

Molar Flow: 555.084 kmol/h

Volumetric Flow: 3930.12 m3/h

Specific Enthalpy: 2957.61 kJ/kg

Specific Entropy: 7.1823 kJ/[kg.K]

Vapor Phase Mole Fraction: 1

Figure 1 Inlet stream Specs

- Below the Unit Operation tab on left, locate the valve block. Drag and drop into the flow sheet. Rename it as "PSV".

8. Under specification for valve add the data as follows.

PSV (Valve)

General Info

Object: PSV

Status: Calculated (20-08-2022 14:40:38)

Linked to:

Connections

Inlet Stream: Inlet stream

Outlet Stream: Outlet stream

Calculation Parameters

Calculation Type: Outlet Pressure

Pressure Drop: 4.5 bar

Outlet Pressure: 1.51325 bar

Flow Coefficient Type: ☒ Kv ☐ Cv

Kv[Cv](max) (IEC 60534): 100 Calculate

☐ Use Opening (%) versus Kv[Cv]/Kv[Cv]max (%) relationship

Kv[Cv]/Kv[Cv]max (%) = f(OP(%)) expression: 1.0\*OP

Valve Opening (%): 50

Property Package Settings

Property Package: Steam Tables (IAPWS-IF97) (1)

Notes

Figure 2 PSV Specs

9. Click on the add specs as shown in the image

PSV (Valve)

General Info

Object: PSV

Status: Calculated (20-08-2022 14:40:38)

Linked to:

Connections

Inlet Stream: Inlet stream

Outlet Stream: Outlet stream

Calculation Parameters

Calculation Type: Outlet Pressure

Pressure Drop: 4.5 bar

Outlet Pressure: 1.51325 bar

Flow Coefficient Type: ☒ Kv ☐ Cv

Kv[Cv](max) (IEC 60534): 100 Calculate

☐ Use Opening (%) versus Kv[Cv]/Kv[Cv]max (%) relationship

Kv[Cv]/Kv[Cv]max (%) = f(OP(%)) expression: 1.0\*OP

Valve Opening (%): 50

Property Package Settings

Property Package: Steam Tables (IAPWS-IF97) (1)

Notes

Attach Utility

Figure 3 Add PSV Sizing/ Evaluation

PressureSafetyValveSizing1 X Material Streams Spreadsheet Charts Flowsheet Dynamics Manager

Docking [Icons] Close

Name: PressureSafetyValveSizing1

Attached to: PSV

☐ Update automatically

**Sizing Method**

Relieved Phase (Fluid): Vapor

Sizing Method: API RP 520

**Relieving Conditions**

Temperature: 250 C Allowed Overpressure: 10 %

Pressure: 6.01325 bar Total Backpressure: 1.51325 bar

**Relieving Flow Rates**

Total: 3930.12 m3/h

Liquid: 0 m3/h

Vapor: 3930.12 m3/h

**Constants**

Kd: 0.85

Kb: 1.00

Kc: 1.00

**Sizing**

Calculate Orifice Area

PSV Orifice Area: in. <sup>2</sup>

API Standard Orifice / Nominal Area: in. <sup>2</sup>

Figure 4 PSV Sizing Calculator

10. Add digital and analog indicators for the material streams and valve as shown in the figure below and give targeting properties.
11. Add the Kd, Kc and Kb values and press on calculate orifice area the required orifice area will be calculated as shown.

**Constants**

Kd: 0.85

Kb: 1.00

Kc: 1.00

**Sizing**

Calculate Orifice Area

PSV Orifice Area: 4.79 in. <sup>2</sup>

API Standard Orifice / Nominal Area: P / 6.38 in. <sup>2</sup>

Figure 5 Defining Constants

12. Run the simulation by pressing “Solve flow sheet” button on the top corner of the screen.

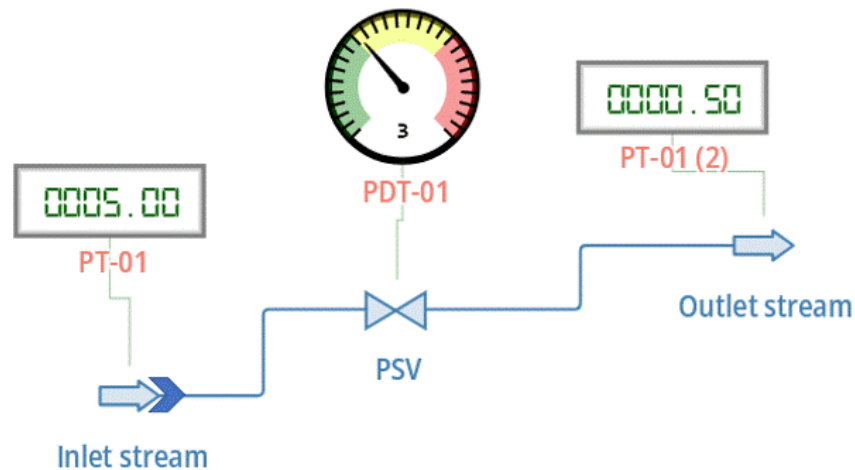


Figure 6 Flow Sheet

	Outlet stream	Inlet stream
Temperature (C)	242.368	250
Pressure (bar)	1.51325	6.01325
Mass Flow (kg/h)	10000	10000
Molar Flow (kmol/h)	555.084	555.084
Volumetric Flow (m <sup>3</sup> /h)	15636	3930.12
Density (Mixture) (kg/m <sup>3</sup> )	0.639549	2.54445
Molecular Weight (Mixture) (kg/kmol)	18.0153	18.0153
Specific Enthalpy (Mixture) (kJ/kg)	2957.61	2957.61
Specific Entropy (Mixture) (kJ/[kg.K])	7.81162	7.1823
Molar Enthalpy (Mixture) (kJ/kmol)	53282.1	53282.1
Molar Entropy (Mixture) (kJ/[kmol.K])	140.729	129.391
Thermal Conductivity (Mixture) (W/[m.K])	0.0375384	0.0394454

### 3 MANUAL CALCULATION IN SMATH STUDIO

31 Aug 2022 09:23:29 - 14 PSV Cal.sm

<b>Valve Data</b>		
Discharge to	Atmosphere	Protected equipment : Reboiler
Set pressure	$P_{set} := 5 \text{ bar}$	Relief service : Steam
Back Pressure	$P_2 := 0.5 \text{ bar}$	Reason for relief : Blocked steam discharge
Accumulation	$a_g := 10 \%$	Relieving Rate : 10,000 kg/hr
Atmosphere pressure	$P := 1.01325 \text{ bar}$	Gas Density : 2.5 kg/m <sup>3</sup>
Accumulated Pressure	$P_1 := P_{set} \cdot \left( 1 + \left( \frac{a_g}{100} \right) \right) + P = 6.5132 \text{ bar}$	Ratio of specific heats for the gas ( $C_P/C_V$ ) : 1.3
Pressure ratio	$r := \frac{P_2}{P_1} = 0.0768$	Compressibility factor of gas = 1.1
		Molecular weight of gas = 18 gm/mole
		Relieving temperature = 250°C
		Set pressure : 5.0 barg
		Accumulation : 10%
		Back pressure at relief valve discharge : 0.5 barg
		Type of relief valve : Conventional pressure relief valve

<b>Process data</b>	
Vapor (V)/ Steam (S)	Enter the condition of fluid for vapour "V" and for steam "S" $Fluid := "S"$
Relieving temperature	$T := 503.15 \text{ K}$
Molecular weight	$M := 18 \frac{\text{kg}}{\text{kmol}}$
Compressibility	$Z := 1.1$
Specific heat capacity ratio	$k := 1.3$
Required relief load	$W := 10000 \frac{\text{kg}}{\text{hr}}$

Figure 7 SMath Page 1

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**Relief valve orifice calculation**

Check for vapor or steam

```

F:=if Fluid="v"
    "X"
else
    if Fluid="s"
        "Q"
    else
        "Enter spec for vap or steam"
F="Q"

```

**Correction factors**

Discharge coeff

 $K_d := 0.975$ 

Back Pressure coeff

 $K_b := 1$ 

Comb Rupture Disk Factor

 $K_c := 1$ 

Superheat steam corr factor

 $K_{sh} := 1$ 

Napier correction factor

```

K_n:=if Fluid="s"
     $\frac{2.764 \cdot P_1 - 1000 \text{ bar}}{3.324 \cdot P_1 - 1061 \text{ bar}}$ 
else
    "_"

```

Calculated  $F_2$  with

$$F_2 = \sqrt{\left(\frac{k}{k-1}\right) \cdot (r)^{2/k} \cdot \left(\frac{1-r^{(k-1)/k}}{1-r}\right)}$$

 $K_n = 0.9$ 

Figure 8 SMath Page 2

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**Calculated data****Type of flow**

Check for subcritical flow using the following formula

$$\frac{P_2}{P} \geq \left( \frac{2}{k+1} \right)^{\frac{k}{k-1}}$$

Shows whether the fluid (vapour) is at critical or subcritical stage

```
Flow:=if F="X"
  if  $\left( \frac{2}{k+1} \right)^{\frac{k}{k-1}} \cdot P_1 > P_2$ 
    "Critical"
  else
    "Sub Critical"
  else
    "-"
Flow="-"
```

**K value coeff**

Calculated C with

$$C = 520 \cdot \sqrt{k \cdot \left( \frac{2}{k+1} \right)^{\frac{k+1}{k-1}}}$$

```
C:=if (F="X")^(Flow="Critical")
   $520 \cdot \sqrt{k \cdot \left( \frac{2}{k+1} \right)^{\frac{k+1}{k-1}}}$ 
  else
    "-"
C="-"
```

**Coeff Sub Critical Flow**Calculated  $F_2$  with

$$F_2 = \sqrt{\left( \frac{k}{k-1} \right) \cdot (r)^{2/k} \cdot \left( \frac{1-r^{(k-1)/k}}{1-r} \right)}$$

```
F2:=if (F="X")^(Flow="Sub Critical")
   $\sqrt{\left( \frac{k}{k-1} \right) \cdot (r)^{\frac{2}{k}} \cdot \left( \frac{1-r^{\frac{k-1}{k}}}{1-r} \right)}$ 
  else
    "-"
F2="-"
```

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Figure 9 SMath Page 3

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Calculated orifice

$$A = \frac{131.6 \cdot W}{C \cdot K_d \cdot P_1 \cdot K_b \cdot K_c} \cdot \sqrt{\frac{T \cdot Z}{M}}$$

$$A = \frac{0.179 \cdot W}{F_2 \cdot K_d \cdot K_c} \cdot \sqrt{\frac{T \cdot Z}{M \cdot P_1 \cdot (P_1 - P_2)}}$$

$$A = \frac{1.904 \cdot W}{P_1 \cdot K_d \cdot K_b \cdot K_c \cdot K_n \cdot K_{sh}}$$

```

A := if Fluid = "V"
    if Flow = "Critical"
        (131.6 * W) * sqrt(T * Z / M) * (kg / m^2)^-1
    else
        (0.179 * W) * sqrt(T * Z / (M * P_1 * (P_1 - P_2))) * (kg / m^2)^-1 * 10^-6
    else
        (1.904 * W) / (P_1 * K_d * K_b * K_c * K_n * K_sh) * (kg / m^2)^-1 * 10^-6

```

$$A = 3173.336 \text{ mm}^2$$

Relief Valve Designations

Standard Orifice Designation	Orifice Area													
	cm <sup>2</sup>	(in. <sup>2</sup> )												
D	0.710	0.110	*	*	*									
E	1.265	0.196	*	*	*									
F	1.981	0.307	*	*	*									
G	3.245	0.503			*	*								
H	5.085	0.785				*	*							
J	8.303	1.287					*	*						
K	11.658	1.838						*	*					
L	15.406	2.353							*	*				
M	23.226	3.60								*	*			
N	28.000	4.34									*	*		
P	41.161	6.38										*	*	
Q	71.290	11.05											*	*
R	103.226	16.0												*
T	167.742	26.0												*
	1 in.		1 × 2	1.5 × 2	1.5 × 2.5	1.5 × 3	2 × 3	2.5 × 4	3 × 4	4 × 6	6 × 8	6 × 10	8 × 10	
	mm		25 × 50	38 × 50	38 × 62	38 × 75	50 × 75	38 × 100	75 × 100	100 × 150	150 × 200	150 × 250	200 × 250	
Valve Body Size (Inlet Diameter × Outlet Diameter)														

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Figure 10 SMath Page 4



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$$f(\text{Recommended}_{\text{Orifice}}) := A$$

$$\text{STD}_{\text{Orifice}} := \begin{bmatrix} \text{"D"} & \text{"E"} & \text{"F"} & \text{"G"} & \text{"H"} & \text{"J"} & \text{"K"} & \text{"L"} & \text{"M"} & \text{"N"} & \text{"P"} & \text{"Q"} & \text{"R"} & \text{"T"} \\ 0.5 \text{ mm}^2 & 71.5 \text{ mm}^2 & 126.5 \text{ mm}^2 & 198.5 \text{ mm}^2 & 324.5 \text{ mm}^2 & 506.5 \text{ mm}^2 & 830.5 \text{ mm}^2 & 1186.5 \text{ mm}^2 & 1841.5 \text{ mm}^2 & 2323.5 \text{ mm}^2 & 2800.5 \text{ mm}^2 & 4116.5 \text{ mm}^2 & 7129.5 \text{ mm}^2 & 10332.5 \text{ mm}^2 \\ 71 \text{ mm}^2 & 126 \text{ mm}^2 & 198 \text{ mm}^2 & 325 \text{ mm}^2 & 506 \text{ mm}^2 & 830 \text{ mm}^2 & 1186 \text{ mm}^2 & 1841 \text{ mm}^2 & 2323 \text{ mm}^2 & 2800 \text{ mm}^2 & 4116 \text{ mm}^2 & 7129 \text{ mm}^2 & 10322 \text{ mm}^2 & 16774 \text{ mm}^2 \end{bmatrix}$$

$$c := \text{"Error: Enter values within range"}$$

$$\text{for } n \in [1.. \text{cols}(\text{STD}_{\text{Orifice}})]$$

$$\text{if } f(\text{Recommended}_{\text{Orifice}}) > \text{STD}_{\text{Orifice}}_{2n}$$

$$\quad \text{Recommended}_{\text{Orifice}} := \text{STD}_{\text{Orifice}}_{3n}$$

$$\quad \text{Orifice}_{\text{tag}} := \text{STD}_{\text{Orifice}}_{1n}$$

$$\text{else}$$

$$\quad \text{continue}$$

$$\text{Recommended}_{\text{Orifice}} = 4116 \text{ mm}^2$$

$$\text{Orifice}_{\text{tag}} = \text{"P"}$$

$$R_{\text{orificesize}}(\text{Calc}_{\text{Area}}) := \text{for } n \in [1.. \text{cols}(\text{STD}_{\text{Orifice}})]$$

$$\quad \text{if } \text{Calc}_{\text{Area}} < \text{STD}_{\text{Orifice}}_{2n}$$

$$\quad \quad x := \text{STD}_{\text{Orifice}}_{2n}$$

$$\quad \quad \text{break}$$

$$\quad \text{else}$$

$$\quad \quad \text{continue}$$

$$x$$

$$R_{\text{orificetag}}(\text{Calc}_{\text{Area}}) := \text{for } n \in [1.. \text{cols}(\text{STD}_{\text{Orifice}})]$$

$$\quad \text{if } \text{Calc}_{\text{Area}} < \text{STD}_{\text{Orifice}}_{3n}$$

$$\quad \quad x := \text{STD}_{\text{Orifice}}_{1n}$$

$$\quad \quad \text{break}$$

$$\quad \text{else}$$

$$\quad \quad \text{continue}$$

$$x$$

$$R_{\text{orificesize}}(A) = 4116.5 \text{ mm}^2$$

$$R_{\text{orificetag}}(A) = \text{"P"}$$

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Figure 11 SMath Page 5

## 4 REFERENCES

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1. API RP 521 — Guide for Pressure-Relieving and Depressuring Systems, American Petroleum Institute, 1220 L Street, NW, Washington, DC 20005.
2. API RP 520 — Recommended Practice for the Design (Part I) and Installation (Part II) of Pressure Relieving Systems in Refineries, American Petroleum Institute, 1220 L Street, NW, Washington, DC 20005.