

论文输入输出实例

计算机科学与技术 专业

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摘要

关键词：流体力学，计算机并行，平衡点

1 example

1.1 通用数据

表格 1.1 通用数据

| 变量 | 数值 |
|----------------------|--------------|
| 管柱的长度 L | 12000in. |
| 空气中每单位长度管柱平均重量 w_s | 0.542lbm/in. |
| 内径 r_i | 1.22in |
| 外径 r_o | 1.438in |
| 外力 F | 20000lbf |
| 外径内径比例 R | 1.178 |
| l | 1.61in. |
| 管柱内液体密度 ρ_i | 15lbm/gal |
| 环空内液体密度 ρ_o | 7.31lbm/gal |
| 管柱内加压 p_i | 5000Psi |
| 环空内加压 p_o | 1000Psi |
| 封隔器通径 r_p | 3.25in |

表格 1.2 单位转换表

| | | |
|------------------|------------|--------------------------------|
| ft | m | 1ft=0.3048m |
| in | mm | 1in=25.4mm |
| ft | in | 1ft=12in |
| lbm ¹ | kg | 1lbm=0.454kg |
| lbf | N | 1lbf=4.45N |
| Mpa | Psi | 1Mpa=145Psi |
| gal | cu ft. | 1gal=0.1336808cu ft. |
| Psi | lbs/sq in. | 1Psi=1lbs ² /sq in. |

根据表 1.1 可得

$$A_o = \pi r_e^2 = 6.49 \text{sq in.}$$

$$A_i = \pi r_e^2 = 4.68 \text{sq in.}$$

$$A_s = A_e - A_i = 1.81 \text{sq in.}$$

$$A_p = \pi r_p^2 = 8.30 \text{sq in.}$$

$$P_i = p_i + \rho_i L \quad P_o = p_o + \rho_o L \quad (1.1)$$

初始压力为 $p_i = p_o = 0$ ，根据公式(1.1)， $P_i = P_o = 3800 \text{Psi}$ 。最终压力为 $p_i = 5000 \text{Psi}$ ， $p_o = 1000 \text{Psi}$ ，根据公式(1.1)， $P_i = 12790 \text{Psi}$ ， $P_o = 4800 \text{Psi}$ 。所以压力变化值为 $\Delta p_i = 5000 \text{Psi}$ ， $\Delta p_o = 1000 \text{Psi}$ ， $\Delta P_i = 8990 \text{Psi}$ ， $\Delta P_o = 1000 \text{Psi}$ 。

初始密度为 $\rho_i = \rho_o = 7.31 \text{lbm/gal} = 0.0317 \text{Psi/in.}$ ，最终密度为 $\rho_i = 15 \text{lbm/gal} = 0.0649 \text{Psi/in.}$ ， $\rho_o = 7.31 \text{lbm/gal} = 0.0317 \text{Psi/in.}$ 。所以密度变化为 $\Delta \rho_i = 0.0332 \text{Psi/in.}$ ， $\Delta \rho_o = 0$ 。

¹英制重量单位，一般 lb 是表示力的单位--磅，也有时表示压力、质量，通用。为了区别起见，lbm 专门表示质量，lbf 专门表示力。

² lb 的复数是 lbs

根据公式(1.6)及上述初始和最终密度数值，可得，每单位长度的质量初始值为 $W = 0.484\text{lbm/in.}$ ，最终值为 $W = 0.640\text{lbm/in.}$ 。

1.2 EXAMPLE 1-PACKER PERMITTING FREE MOTION

$$n = \frac{F}{W} \quad (1.2)$$

$$P = -\sqrt{\frac{8EI}{F}} \quad (1.3)$$

$$F_f = A_p P_i \quad (1.4)$$

$$F_f = A_p (P_i - P_o) \quad (1.5)$$

$$W = W_s + W_i - W_o = W_s + \rho_i A_i - \rho_o A_o \quad (1.6)$$

$$\Delta L_1 = -\frac{LF}{EA_s} \quad (1.7)$$

$$\Delta L_2 = -\frac{r^2 F^2}{8EIw} \quad (1.8)$$

$$F_a = (A_p - A_i)P_i - (A_p - A_o)P_o \quad (1.9)$$

$$\Delta L_1 = -\frac{L}{EA_s} [(A_p - A_i)\Delta P_i - (A_p - A_o)\Delta P_o] \quad (1.10)$$

$$\Delta L_2 = -\frac{r^2 A_p^2 (\Delta P_i - \Delta P_o)^2}{8EI(W_s + W_i - W_o)} \quad (1.11)$$

$$\Delta L_3 = -\frac{v}{E} \frac{\Delta \rho_i - R^2 \Delta \rho_o - \frac{1+2v}{2v} \delta}{R^2 - 1} L^2 - \frac{2v}{E} \frac{\Delta P_i - R^2 \Delta P_o}{R^2 - 1} L \quad (1.12)$$

$$\Delta L_4 = L\beta\Delta t \quad (1.13)$$

$$\Delta L = \Delta L_1 + \Delta L_2 + \Delta L_3 + \Delta L_4 \quad (1.14)$$

1.3 EXAMPLE 1-PACKER PERMITTING LIMITED MOTION

术语：

F = externally applied force (positive if a compression), lbf (N) 施加的外力

W_s = average weight in air of the tube per unit length, lbm/in. (g/mm) 空气中的每单位长度的管平均重量

A_s = cross-sectional area of the tubing wall, sq in.(mm²) 管壁的横截面积

p_i =surface tubing pressure, Psi

p_o =surface annulus pressure, Psi

P_o = pressure outside the tube at the lower end, psi (Pa) 管柱下端外压

P_i = pressure inside the tube at the lower end, psi (Pa) 管柱下端内压

A_p = area corresponding to packer bore ID, sq in. (mm²)

A_i = area corresponding to tubing ID, sq in. (mm²)

A_o = area corresponding to tubing OD, sq in. (mm²)

F_f = fictitious force in presence of no restraint in the packer, lbf

F_a = actually existing pressure force at the lower end of the tubing subjected to no restraint in the packer, lbf

参考文献

[1]Lubinski A,Althouse W S,Logan J L. Helical Buckling of Tubing Sealed in Packers[J]. Journal of Petroleum Technology, 1962, 14(6):655-670

附录

中立点 (neutral point)，就是在这个点上方管柱不发生形变，在这个点下方管柱弯曲。