

论文输入输出实例

计算机科学与技术 专业

研究生 王哲

指导老师 蒋玉明

摘要

关键词：流体力学，计算机并行，平衡点

1 example

1.1 通用数据

表格 1.1 通用数据

变量	数值
管柱的长度 L	$12000in.$
空气中每单位长度管柱平均重量 w_s	$0.542lbm/in.$
内径 r_i	$1.22in$
外径 r_o	$1.438in$
外力 F	$20000lbf$
外径内径比例 R	1.178
I	$1.61in.^4$
管柱内液体密度 ρ_i	$15lbm/gal$
环空内液体密度 ρ_o	$7.31lbm/gal$
管柱内加压 p_i	$5000Psi$
环空内加压 p_o	$1000Psi$
封隔器通径 r_p	$3.25in$
r	$1.61in.$
δ	0
Δt	$-20^\circ F$

表格 1.2 单位转换表

ft	m	$1ft=0.3048m$
in	mm	$1in=25.4mm$
ft	in	$1ft=12in$
lbm^1	kg	$1lbm=0.454kg$
lbf	N	$1lbf=4.45N$
Mpa	Psi	$1Mpa=145Psi$
gal	$cu\ ft.$	$1gal=0.1336808cu\ ft.$
Psi	$lbs/sq\ in.$	$1Psi=1lbs^2/sq\ in.$

根据表 1.1 可得

$$A_o = \pi r_e^2 = 6.49sq\ in.$$

$$A_i = \pi r_e^2 = 4.68sq\ in.$$

$$A_s = A_e - A_i = 1.81sq\ in.$$

$$A_p = \pi r_p^2 = 8.30sq\ in.$$

$$P_i = p_i + \rho_i L \quad P_o = p_o + \rho_o L \quad (1.1)$$

$$W = W_s + W_i - W_o = W_s + \rho_i A_i - \rho_o A_o \quad (1.2)$$

初始密度为 $\rho_i = \rho_o = 7.31lbm/gal = 0.0317Psi/in.$ ，最终密度为 $\rho_i = 15lbm/gal =$

¹英制重量单位，一般 lb 是表示力的单位--磅，也有时表示压力、质量，通用。为了区别起见，lbm 专门表示质量，lbf 专门表示力。

² lb 的复数是 lbs

0.0649Psi/in., $\rho_o = 7.31\text{lbm/gal} = 0.0317\text{Psi/in.}$ 。所以密度变化为 $\Delta\rho_i = 0.0332\text{Psi/in.}$, $\Delta\rho_o = 0$ 。

初始压力为 $p_i = p_o = 0$, 根据公式(1.1), $P_i = P_o = 3800\text{Psi}$ 。最终压力为 $p_i = 5000\text{Psi}$, $p_o = 1000\text{Psi}$, 根据公式(1.1), $P_i = 12790\text{Psi}$, $P_o = 4800\text{Psi}$ 。所以压力变化值为 $\Delta p_i = 5000\text{Psi}$, $\Delta p_o = 1000\text{Psi}$, $\Delta P_i = 8990\text{Psi}$, $\Delta P_o = 1000\text{Psi}$ 。

根据公式(1.2)及上述初始和最终密度数值, 可得, 每单位长度的质量初始值为 $W = 0.484\text{lbm/in.}$, 最终值为 $W = 0.640\text{lbm/in.}$ 。

1.2 EXAMPLE 1-PACKER PERMITTING FREE MOTION

$$F_f = A_p(P_i - P_o) \quad (1.3)$$

$$F_a = (A_p - A_i)P_i - (A_p - A_o)P_o \quad (1.4)$$

$$\Delta L_1 = -\frac{LF}{EA_s} \quad (1.5)$$

$$\Delta L_2 = -\frac{r^2 F^2}{8EIw} \quad (1.6)$$

$$\Delta L_1 = -\frac{L}{EA_s} [(A_p - A_i)\Delta P_i - (A_p - A_o)\Delta P_o] \quad (1.7)$$

$$\Delta L_2 = -\frac{r^2 A_p^2 (\Delta P_i - \Delta P_o)^2}{8EI(W_s + W_i - W_o)} \quad (1.8)$$

$$\Delta L_3 = -\frac{v}{E} \frac{\Delta p_i - R^2 \Delta p_o}{R^2 - 1} \frac{1+2v}{2v} \delta L^2 - \frac{2v}{E} \frac{\Delta p_i - R^2 \Delta p_o}{R^2 - 1} L \quad (1.9)$$

$$\Delta L_4 = L\beta\Delta t \quad (1.10)$$

$$\Delta L = \Delta L_1 + \Delta L_2 + \Delta L_3 + \Delta L_4 \quad (1.11)$$

$$n = \frac{F}{W} \quad (1.12)$$

$$P = -\sqrt{\frac{8EI}{F}} \quad (1.13)$$

根据公式(1.7)(1.8)(1.9)(1.10)可得, $\Delta L_1 = -67.92\text{in.}$, $\Delta L_2 = -46.10\text{in.}$, $\Delta L_3 = -34.69\text{in.}$, $\Delta L_4 = -16.56\text{in.}$, 所以 $\Delta L = -165.27\text{in.}$

根据公式(1.3)(1.4)可得 $F_f = 66317\text{lbf}$, $F_a = 37611.8\text{lbf}$ 。

根据公式(1.12), 及 $F_f = 66317\text{lbf}$, $W = 0.640\text{lbm/in.}$ 可得 $n = 103620\text{in.} = 8535\text{ft.}$

1.3 EXAMPLE 1-PACKER PERMITTING LIMITED MOTION

1.4 EXAMPLE 2- PACKER PERMITTING LIMITED MOTION

1.5 EXAMPLE 2-PACKER PERMITTING NO MOTION

术语:

F = externally applied force (positive if a compression), lbf (N) 施加的外力

W_s = average weight in air of the tube per unit length, lbf/in. (g/mm) 空气中的每单位长度的管平均重量

A_s = cross-sectional area of the tubing wall, sq in.(mm²) 管壁的横截面积

p_i =surface tubing pressure, Psi

p_o =surface annulus pressure, Psi

P_o = pressure outside the tube at the packer level, psi (Pa)

P_i = pressure inside the tube at the packer level, psi (Pa)

A_p = area corresponding to packer bore ID, sq in. (mm²)

A_i = area corresponding to tubing ID, sq in. (mm²)

A_o = area corresponding to tubing OD, sq in. (mm²)

F_f = fictitious force in presence of no restraint in the packer,lbf

F_a = actually existing pressure force at the lower end of the tubing subjected to no restraint in the packer ,lbf

ν =Poisson's ratio of the material (for steel , $\nu=0.3$)

β =coefficient of thermal expansion of the tubing material(for steel, $\beta = 6.9 \times 10^{-6}/1^\circ\text{F}$)

δ =drop of pressure in the tubing due to flow per unit length

Δt =change in average tubing temperature

ΔL_1 =length change of the tubing due to Hooke's law,胡克定律产生的形变由公式(1.5)定义

ΔL_2 =length change of the tubing due to helical buckling,变形弯曲产生的形变由公式(1.6)定义

ΔL_3 =length change of the tubing due to radial pressure forces and flow through the tubing,径向压力和液体流动产生的形变由公式(1.9)定义

ΔL_4 =length change of the tubing due to temperature change,温度变化产生的长度变化由公式(1.10) 定义

ΔL =Over-all tubing change due to flow and to changes of pressure, temperature and density,压力, 温度, 密度和液体流动产生的管柱总长度变化, 由公式 (1.11)定义

参考文献

[1]Lubinski A,Althouse W S,Logan J L. Helical Buckling of Tubing Sealed in Packers[J]. Journal of Petroleum Technology, 1962, 14(6):655-670

附录

中立点 (neutral point)，就是在这个点上方管柱不发生形变，在这个点下方管柱弯曲。