

Multiphase Flows – WS 2022/23

Problem Session 8 – **Solution:**

Single Droplet Heat & Mass Transfer



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Problem Session 8: Single Droplet Heat & Mass Transfer

Problem 1: D^2 Law for Droplet Evaporation

The mass balance for a single evaporating droplet can be written as $\frac{dm}{dt} = -k_m A H_m$, where k_m is the mass transport coefficient, A is the surface area of the droplet, and H_m is the driving potential for mass transfer.

- a) From this general equation, derive the ODE for the change of the droplet diameter D over time. Assume the droplet to be spherical. The Sherwood number $Sh = \frac{k_m D}{\rho_c D_{cv}}$ be known. Insert the difference of the vapor mass fractions between the droplet surface and the freestream as driving potential.

Given: $Sh, \rho_c, \rho_d, D_{cv}, Y_{v,\infty}, M_v, M_M, p, T, p_{\text{sat}}(T)$

- b) Assuming all relevant fluid properties and the temperature to be constant, derive the D^2 law for droplet evaporation by integrating the ODE.

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Problem 1a : Solution

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Given: $Sh, \rho_c, \rho_d, D_{cv}, Y_{v,\infty}, M_v, M_M, p, T, p_{sat}(T)$

Note: $Sh = \frac{\text{Mass transfer by convection}}{\text{Mass transfer by diffusion}}, Nu = \frac{\text{Heat transfer by convection}}{\text{Heat transfer by conduction}}, \rho_c \rightarrow \text{medium}, \rho_d \rightarrow \text{droplet}$

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Problem 1a : Solution

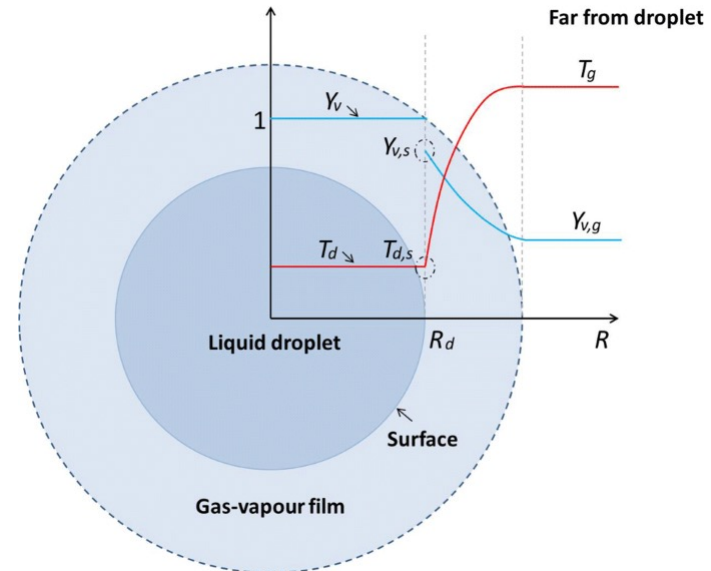
$$\frac{dm^A}{dt} = - k_m^B A^C H_m^D$$

$$A : m = \rho_d V = \rho_d \frac{\pi D^3}{6}$$

$$B : Sh = \frac{k_m D}{\rho_c D_{cv}} \Rightarrow k_m = \frac{Sh \cdot \rho_c D_{cv}}{D}$$

$$C : A = \pi D^2$$

$$D : H_m = Y_{v,s}^E - Y_{v,\infty}$$



$$E : Y_{v,s} = \frac{m_{v,s}}{m_{M,s}} = \frac{M_v n_{v,s}}{M_M n_{M,s}} = \frac{M_v p_{v,s}}{M_M p} = \frac{M_v p_{sat}(T)}{M_M p}$$

Problem I a: Solution

$$\frac{dm}{dt} = \frac{\pi}{6} \rho_d \frac{dD^3}{dt} = \frac{\pi}{2} \rho_d D^2 \frac{dD}{dt}$$

$$-k_m A H_m = -\frac{Sh \rho_c D_{cv}}{D} \pi D^2 (Y_{v,s} - Y_{v,\infty})$$

$$\Rightarrow \frac{dD}{dt} = -\frac{2 \cdot Sh \cdot \rho_c D_{cv}}{\rho_d D} (Y_{v,s} - Y_{v,\infty}) = -\frac{1}{D} \frac{\lambda}{2}$$

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Problem I b: Solution

$$\text{b)} \quad DdD = -\frac{\lambda}{2}dt \Rightarrow \int_{D_0}^D DdD = -\frac{\lambda}{2} \int_0^t dt \Rightarrow \boxed{D^2 = D_0^2 - \lambda t}$$

D^2 law

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Problem 2: Wet-Bulb Temperature

The energy equation for an evaporating water droplet can be expressed as

$$mc_p \frac{dT_d}{dt} = Nu \cdot \pi D \lambda_c (T_\infty - T_d) - Sh \cdot \pi \rho_c D_{cv} D (Y_{v,s} - Y_{v,\infty}) L,$$

where L is the latent heat of water. The cooling effect of the evaporation dilutes the heating of a droplet in warmer air. Once the droplet reaches the so-called wet-bulb temperature at saturation conditions, it does not heat further until it completely evaporates.

Compute the wet-bulb temperature of a water droplet in air at 1 bar and 293 K, and at a relative humidity of 40 %. Assume $Nu = Sh$, $Pr = 0.76$, and $Sc = 0.65$.

Further: $c_{p,c} = 1.005 \frac{\text{kJ}}{\text{kgK}}$, $L = 2454 \text{ kJ/kg}$, $M_{H_2O} = 18 \text{ g/mol}$, $M_M = 29 \text{ g/mol}$.

T [K]	280	285	290	293
p_{sat} [bar]	0.009912	0.01388	0.01919	0.02239

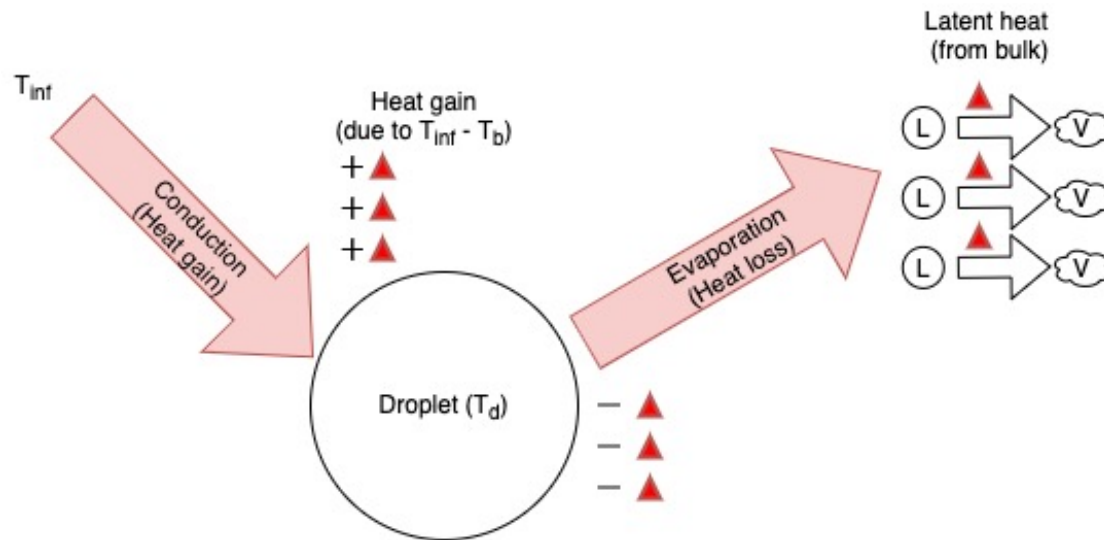
$$Pr = \frac{\mu c_{p,c}}{\lambda_c}, \quad Sc = \frac{\mu}{\rho_c D_{cv}}$$

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Problem 2: Solution

$$mc_p \frac{dT_d}{dt} = Nu \cdot \pi D \lambda_c (T_\infty - T_d) - Sh \cdot \pi \rho_c D_{cv} D (Y_{v,s} - Y_{v,\infty}) L$$

The cooling effect of the evaporation dilutes the heating of a droplet in warmer air. Once the droplet reaches the so-called wet-bulb temperature at saturation conditions, it does not heat further until it completely evaporates.



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Problem 2: Solution

Wet-bulb temperature: T_d stays constant $\frac{dT_d}{dt} = 0$

$$\Rightarrow Nu \cdot \lambda_c (T_\infty - T_d) = Sh \cdot \rho_c D_{cv} (Y_{v,s} - Y_{v,\infty}) L$$

$$\Rightarrow Nu \cdot (T_\infty - T_d) = Sh \cdot \frac{\mu c_{p,c}}{\lambda_c} \frac{\rho_c D_{cv}}{\mu c_{p,c}} \frac{L}{\mu c_{p,c}} (Y_{v,s} - Y_{v,\infty})$$

$$\Rightarrow Nu \cdot (T_\infty - T_d) = Sh \cdot \frac{Pr}{Sc} \frac{L}{c_{p,c}} (Y_{v,s} - Y_{v,\infty})$$

$$\Rightarrow T_\infty = T_d + \frac{Sh}{Nu} \frac{Pr}{Sc} \frac{L}{c_{p,c}} (Y_{v,s} - Y_{v,\infty}) \quad (I)$$

$$p_{\text{sat}}(293 \text{ K}) = 0.02239 \text{ bar}, x_{\text{rel}} = \frac{p_{H_2O}}{p_{H_2O}^*} \quad (\text{actual to equilibrium partial pressure ratio})$$

$$\Rightarrow Y_{v,\infty} = x_{\text{rel}} \frac{p_{\text{sat}}(T_\infty)}{p} \frac{M_{H_2O}}{M_M} = 0.4 \cdot \frac{0.02239 \text{ bar}}{1 \text{ bar}} \cdot \frac{18 \frac{\text{g}}{\text{mol}}}{29 \frac{\text{g}}{\text{mol}}} = 0.00556$$

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Problem 2: Solution

$$Y_{v,s}(T_d) = \frac{p_{\text{sat}}(T_d)}{p} \frac{M_{\text{H}_2\text{O}}}{M_M} = \frac{p_{\text{sat}}(T_d)}{1 \text{ bar}} \cdot \frac{18}{29}$$

T_d	280	285	290
$Y_{v,s}$	0.00615	0.00862	0.01191

From (I): $T_d - T_\infty + \frac{Sh}{Nu} \frac{Pr}{Sc} \frac{L}{c_{p,c}} (Y_{v,s} - Y_{v,\infty}) = 0$

$$\Rightarrow T_d - 293 + \frac{0.76}{0.65} \cdot \frac{2454}{1.005} (Y_{v,s} - 0.00556) = \delta$$

T_d	280	285	290
δ	-11.31	0.74	15.13



$$\Rightarrow \text{Interpolate: } T_d^{\text{wet-bulb}} = 280 \text{ K} + \frac{(285-280)\text{K} \cdot (0+11.31)}{0.74+11.31} = 284.7 \text{ K}$$

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Problem 3: Droplet Evaporation (in MATLAB)

- a) *For the conditions of Problem 2, plot the droplet diameter and the droplet mass over time for initial diameters of $1000\ \mu\text{m}$, $10\ \mu\text{m}$, and $1\ \mu\text{m}$ based on the D^2 law. Check if D^2 plot is a straight line. The Sherwood number be $Sh = 2$. Further fluid properties: $\lambda_c = 0.0257\ \frac{\text{W}}{\text{mK}}$, $\rho_{\text{H}_2\text{O}} = 1000\ \frac{\text{kg}}{\text{m}^3}$. Assumptions: $Sh = \text{const.}$, $Pr = \text{const.}$, $Sc = \text{const.}$, $\lambda_c = \text{const.}$, $\rho_d = \text{const.}$, $c_{p,c} = \text{const.}$, $x_{\text{rel}} = \text{const.}$*
- b) *Plot the droplet evaporation time $t_{\text{evap}}(p, T_\infty)$ over temperature and pressure of the surrounding air for an initial droplet diameter of $1000\ \mu\text{m}$. The air have 10 % relative humidity. Assume that the droplet always has the wet-bulb temperature at the respective conditions. The fluid properties and dimensionless numbers remain constant. Use the stiffened-gas equation of state for computing the saturation pressures and wet-bulb temperatures.*

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Problem 3 a: Solution

a) D^2 law: $D^2 = D_0^2 - \lambda t$

1. Plug in $Y_{v,s}$ and $Y_{v,inf}$ from problem 2 ($Y_{v,inf}$ found out, $Y_{v,s}$ from interpolation)
2. Calculate value of λ for D^2 equation
3. Find final evaporation time (time = 0: t_{evap})
4. Plug in expressions for diameter and mass

$$Y_{v,s} = 0.00615 + \frac{(0.00862 - 0.00615)(284.7 - 280)}{285 - 280} = 0.00847$$

$$\lambda = \frac{4 \cdot Sh \cdot Pr \cdot \lambda_c (Y_{v,s} - Y_{v,\infty})}{Sc \cdot \rho_d \cdot c_{p,c}} = 6.961 \cdot 10^{-10} \frac{\text{m}^2}{\text{s}}$$

$$t_{evap} = \frac{D_0^2}{\lambda(p, T_\infty)} \quad \text{Mass: } m_d = \rho_d \cdot \frac{\pi D^3}{6} = \rho_d \frac{\pi (D_0^2 - \lambda t)^{\frac{3}{2}}}{6}$$

Problem 3 b: Solution

b) $t_{\text{evap}} = \frac{D_0^2}{\lambda(p, T_\infty)}$

1. Plug in expressions for $Y_{v,s}$ and $Y_{v,inf}$
2. Plug in expression for λ for D^2 law
3. Plug in value of δ function for bisection method for calculating wet bulb temp.

$$Y_{v,\infty} = x_{\text{rel}} \frac{p_{\text{sat}}(T_\infty)}{p_\infty} \frac{M_{\text{H}_2\text{O}}}{M_M}$$

$$Y_{v,s} = \frac{p_{\text{sat}}(T_d)}{p_\infty} \frac{M_{\text{H}_2\text{O}}}{M_M}$$

$$\lambda = \frac{4 \cdot Sh \cdot Pr \cdot \lambda_c (Y_{v,s} - Y_{v,\infty})}{Sc \cdot \rho_d c_{p,c}}$$

$$T_d - T_{\text{inf}} + \frac{Sh}{Nu} \cdot \frac{Pr}{Sc} \frac{L}{c_p} (Y_{v,s} - Y_{v,inf}) = \delta$$



Thank you for your attention

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