

# **IN-LINE SLIDER CRANK MECHANISM**

## **A MAJOR PROJECT REPORT**

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*in partial fulfilment for the award of the degree*

*Of*

**BACHELOR OF TECHNOLOGY**

**IN**

**COMPUTATIONAL ENGINEERING MECHANICS II**



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Signature

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This project report was evaluated by us on .....

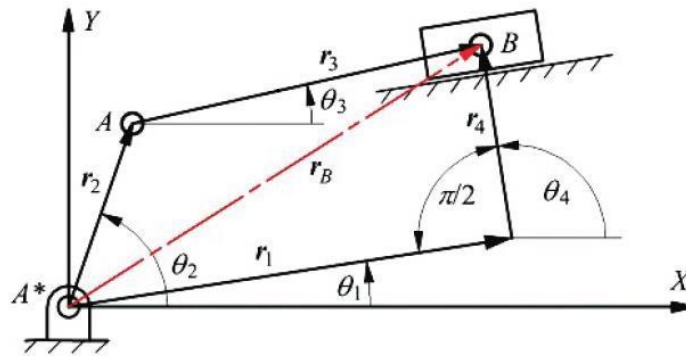
INTERNAL EXAMINER

EXTERNAL EXAMINER

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### IN-LINE SLIDER CRANK MECHANISM:



This is basically the off-set slider crank mechanism. So, to make it in-line slider crank we have to make  $\theta_1$  and  $r_4$  to zero.

### DERIVATION:

**From the above diagram we can observe that,**

- The link  $r_1$  is a fixed link making an angle  $\theta_1$  i.e. 0.
- The link  $r_2$  is the crank making an angle  $\theta_2$ .
- The link  $r_3$  is the slider making an angle  $\theta_3$ .
- The link  $r_4$  is not considered here as it is in-line slider-crank mechanism.

Considering the link lengths as  $r_1$ ,  $r_2$  and  $r_3$

**Time period for crank to rotate a full revolution:**

$$T = \frac{2\pi}{\omega_2}$$

$$\frac{d\theta_2}{dt} = \omega_2$$

$$\int d\theta_2 = \omega_2 \int dt$$

**Integrating on both sides:**

$$\theta_2 = \omega_2 \times t + \theta_0$$

### POSITIONAL ANALYSIS:

$$\vec{r_B} = \vec{r_2} + \vec{r_4} = \vec{r_2} + \vec{r_3}$$

$$r_1(\cos \theta_1 i + \sin \theta_1 j) + r_4(\cos \theta_4 i + \sin \theta_4 j) = r_2(\cos \theta_2 i + \sin \theta_2 j) + r_3(\cos \theta_3 i + \sin \theta_3 j)$$

By separating i and j terms

$$i = r_1 \cos \theta_1 + r_4 \cos \theta_4 = r_2 \cos \theta_2 + r_3 \cos \theta_3 \rightarrow 1$$

$$j = r_1 \sin \theta_1 + r_4 \sin \theta_4 = r_2 \sin \theta_2 + r_3 \sin \theta_3 \rightarrow 2$$

$$r_1, r_2, r_3, r_4 \rightarrow \text{Given} (r_4 = 0)$$

$$\theta_1, \theta_4 \rightarrow \text{constant} (\theta_1 = \theta_4 = 0)$$

$$\theta_2, \theta_3 \rightarrow \text{Variable}$$

Substituting equation 1 and 2 with the above values

$$r_3 \cos \theta_3 + r_2 \cos \theta_2 = r_1 \rightarrow 3$$

$$r_3 \sin \theta_3 = -r_2 \sin \theta_2 \rightarrow 4$$

$$(r_1 - r_2 \cos \theta_2)^2 = (r_3 \cos \theta_3)^2$$

$$(-r_2 \sin \theta_2)^2 = (-r_3 \sin \theta_3)^2$$

$$r_3^2 = (r_1 - r_2 \cos \theta_2)^2 + r_2^2 \sin^2 \theta_2$$

$$r_3^2 = r_1^2 + r_2^2 \cos^2 \theta_2 - 2r_1 r_2 \cos \theta_2 + r_2^2 \sin^2 \theta_2$$

$$r_3^2 = r_1^2 + r_2^2 - 2r_1 r_2 \cos \theta_2$$

$$\cos \theta_2 = \frac{r_3^2 - (r_1^2 + r_2^2)}{-2r_1 r_2}$$

$$\sin \theta_3 = \frac{-r_2 \sin \theta_2}{r_3}$$

### VELOCITY ANALYSIS:

$$\dot{\vec{r}}_1 + \dot{\vec{r}}_4 = \dot{\vec{r}}_2 + \dot{\vec{r}}_3$$

$$\dot{r}_1 (\cos \theta_1 i + \sin \theta_1 j) = r_2 \dot{\theta}_2 (-\sin \theta_2 i + \cos \theta_2 j) + r_3 \dot{\theta}_3 (-\sin \theta_3 i + \cos \theta_3 j)$$

$$\dot{r}_1 \cos \theta_1 = -(r_2 \dot{\theta}_2 \sin \theta_2 + r_3 \dot{\theta}_3 \sin \theta_3)$$

$$\dot{r}_1 \sin \theta_1 = -(r_2 \dot{\theta}_2 \cos \theta_2 + r_3 \dot{\theta}_3 \cos \theta_3)$$

$$\dot{r}_1 = -(r_2 \omega_2 \sin \theta_2 + r_3 \omega_3 \sin \theta_3) \rightarrow 5$$

$$0 = r_2 \omega_2 \cos \theta_2 + r_3 \omega_3 \cos \theta_3$$

$$\omega_3 = -\frac{r_2 \omega_2 \cos \theta_2}{r_3 \cos \theta_3} \rightarrow 6$$

### ACCELERATION ANALYSIS:

$$\ddot{r}_1 (\cos \theta_1 i + \sin \theta_1 j) = r_2 \ddot{\theta}_2 (-\sin \theta_2 i + \cos \theta_2 j) + r_3 \ddot{\theta}_3 (-\sin \theta_3 i + \cos \theta_3 j)$$

$$\ddot{r}_1 (\cos \theta_1 i + \sin \theta_1 j) = r_2 \ddot{\theta}_2 (-\sin \theta_2 i + \cos \theta_2 j) - r_2 \dot{\theta}_2^2 (-\cos \theta_2 i + \sin \theta_2 j) + r_3 \ddot{\theta}_3 (-\sin \theta_3 i + \cos \theta_3 j) - r_3 \dot{\theta}_3^2 (-\cos \theta_3 i + \sin \theta_3 j)$$

$$\ddot{r}_1 \cos \theta_1 = -r_2 \ddot{\theta}_2 \sin \theta_2 - r_2 \dot{\theta}_2^2 \cos \theta_2 - r_3 \ddot{\theta}_3 \sin \theta_3 - r_3 \dot{\theta}_3^2 \cos \theta_3 \rightarrow 7$$

$$\ddot{r}_1 \sin \theta_1 = r_2 \ddot{\theta}_2 \cos \theta_2 - r_2 \dot{\theta}_2^2 \sin \theta_2 - r_3 \ddot{\theta}_3 \cos \theta_3 - r_3 \dot{\theta}_3^2 \sin \theta_3 \rightarrow 8$$

$$r_1, r_2, r_3, r_4 \rightarrow \text{Given} (r_4 = 0)$$

$$\theta_1, \theta_4 \rightarrow \text{constant} (\theta_1 = \theta_4 = 0)$$

$$\theta_2, \theta_3 \rightarrow \text{Variable}$$

$$\ddot{\theta}_2 = 0 \text{ as angular velocity is constant}$$

Substituting equation 7 and 8 with the above values

$$\ddot{r}_1 = -r_2 \dot{\theta}_2^2 \cos \theta_2 - r_3 \ddot{\theta}_3 \sin \theta_3 - r_3 \dot{\theta}_3^2 \cos \theta_3 \rightarrow 9$$

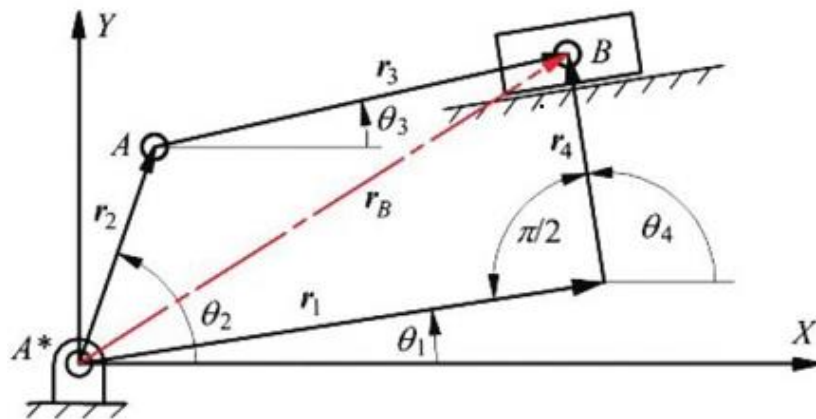
$$0 = -r_2 \dot{\theta}_2^2 \sin \theta_2 - r_3 \ddot{\theta}_3 \cos \theta_3 - r_3 \dot{\theta}_3^2 \sin \theta_3 \rightarrow 10$$

From eq 10

$$\ddot{\theta}_3 = \frac{-r_2 \dot{\theta}_2^2 \sin \theta_2 - r_3 \dot{\theta}_3^2 \sin \theta_3}{r_3 \cos \theta_3} \rightarrow 11$$

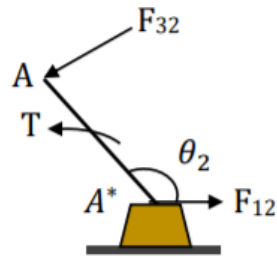
Now we can substitute the value of  $\ddot{\theta}_3$  in eq 9 to find  $\ddot{r}_1$

### STATIC FORCE ANALYSIS:

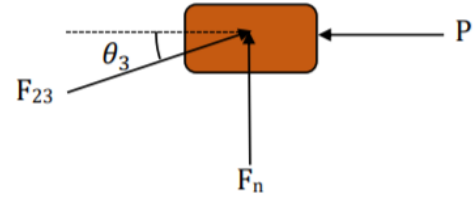


This is basically the off-set slider crank mechanism. So, to make it in-line slider crank we have to make  $\theta_1$  and  $r_4$  to zero .

Considering link A\*A:



Considering the block:



**Applying equilibrium conditions:**

$$\sum F_x = 0:$$

$$P = F_{23} \cos \theta_3 \rightarrow 12$$

$$\sum F_y = 0:$$

$$F_n = F_{23} \sin \theta_3$$

**Moment at A\*:**

$$T = -F_{32} \times l_2 (\sin(\theta_2 + \theta_3)) \rightarrow 13$$

**Substituting [1] in [2]:**

$$T = -\frac{P}{\cos \theta_3} \times l_2 (\sin(\theta_2 + \theta_3)) = -P \times l_2 [\sin \theta_2 + \cos \theta_2 \tan \theta_3]$$

EXAMPLE:

APPENDIX:

Main Frame Of GUI :



**in - Line Slider Crank Mechanism Analyser**

Crank Length

Connecting rod Length

Input Range

Input Range

Dialogue Box

Kinematics

Analyse

Animate

Clear

Reset

Validate

Save Output

Off ☐ On

**in - Line Slider Crank Mechanism Analyser**

Crank Length

Connecting rod Length

Input Range

Input Range

Dialogue Box

Kinematics

Analyse

Animate

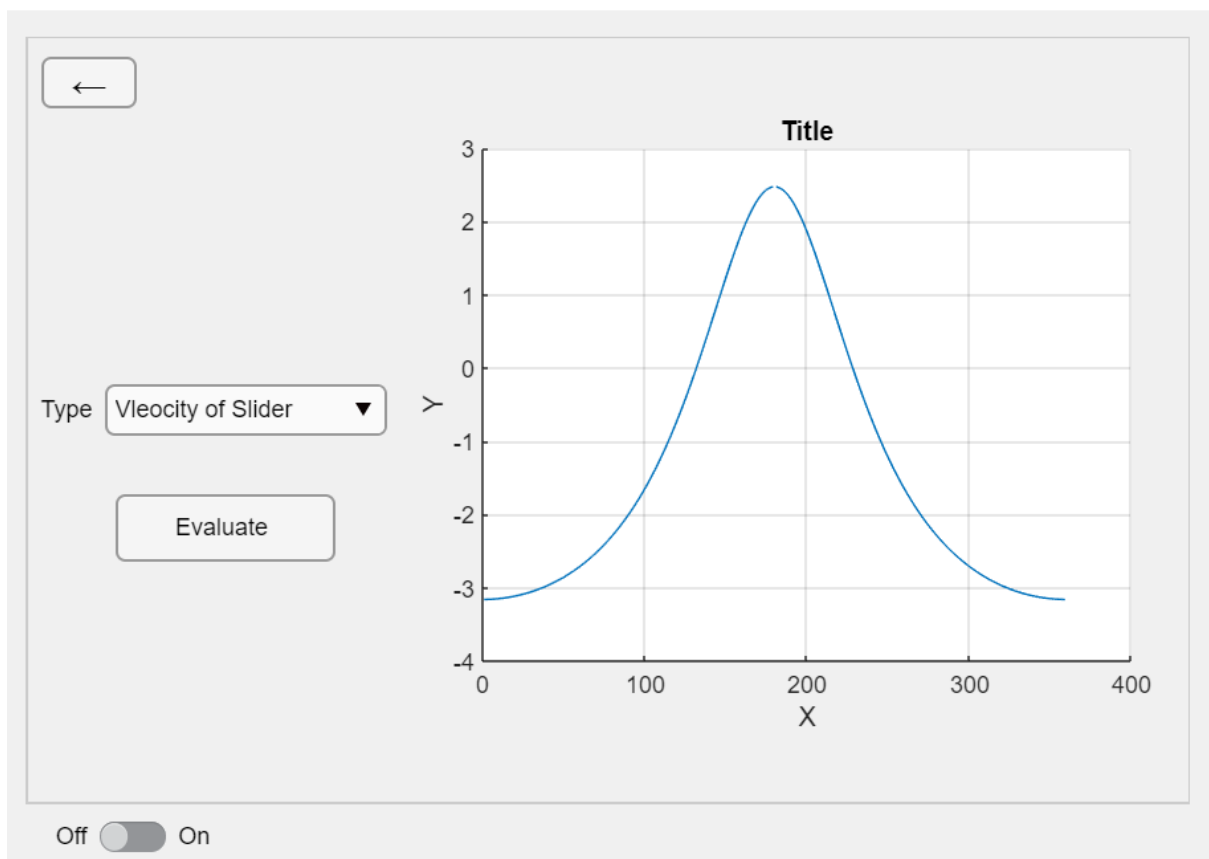
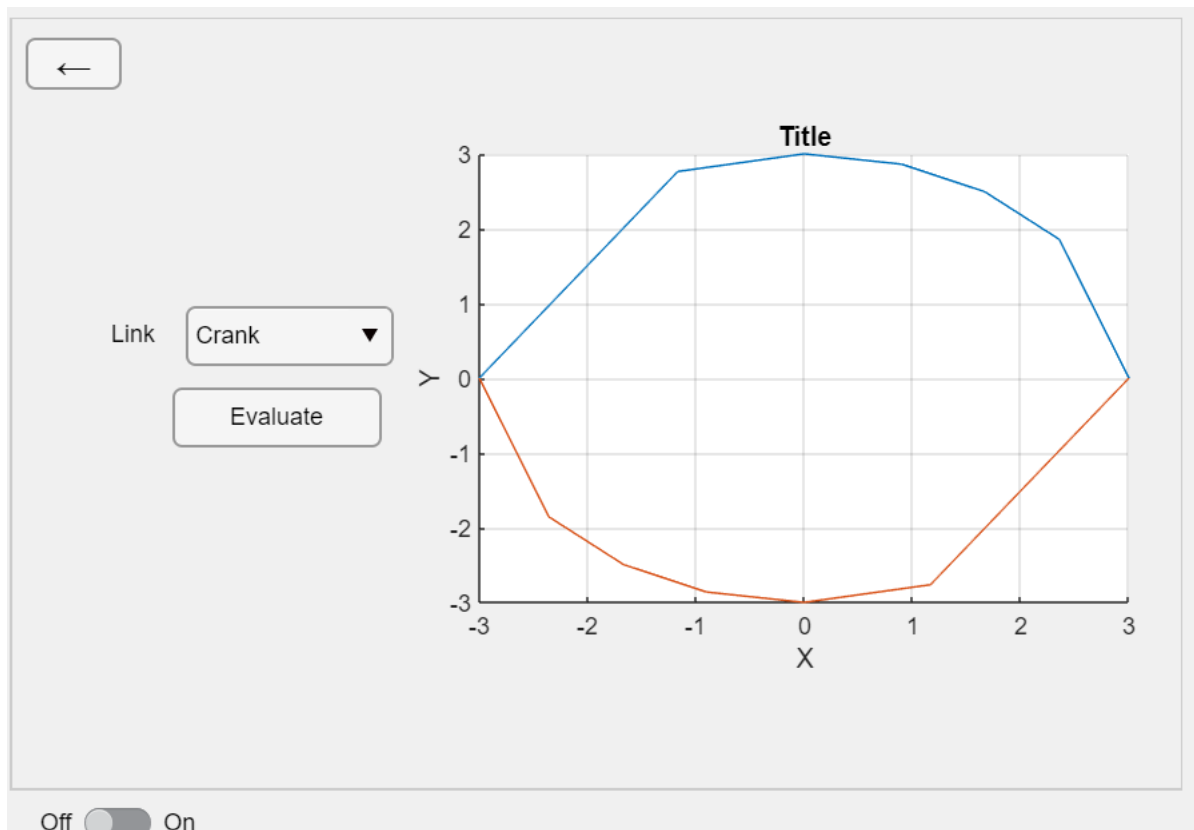
Clear

Reset

Validate

Save Output

Off ☐ On



### Position :

```
% r2 = crank ; r3 = connecting rod
r2 = str2num(app.CrankLengthEditField.Value);
r3 = str2num(app.ConnectingrodLengthEditField.Value);
if(app.InputRangeDropDown.Value=="Position of slider , r2_1")
    r1 = str2num(app.InputRangeEditField.Value);

    for i =1: length(r1)

        theta2(i) =acosd(((r1(i)^2)+(r2^2)-(r3^2))/(2*r1(i)*r2));
        theta3(i) = asin((-r2*sind(theta2(i)))/r3);
    end

negtheta2= 180+fliplr(theta2);

if(app.LinkDropDown.Value=="Crank")
    x = r2*cosd(theta2);
    y = r2*sind(theta2);
    x_ = r2*cosd(negtheta2);
    y_ = r2*sind(negtheta2);

    plot(x,y,"Parent",app.UIAxes);
    hold(app.UIAxes,'on');
    plot(x_,y_,"Parent",app.UIAxes);
    hold(app.UIAxes,'off');
elseif(app.LinkDropDown.Value=="Slider")
    plot(theta2,r1,"Parent",app.UIAxes);

end

elseif(app.InputRangeDropDown.Value=="Angle of Crank ,  $\theta_2$ ")
    theta2=str2num(app.InputRangeEditField.Value);
    theta3 = asin((-r2*sind(theta2))/r3);

    if(app.LinkDropDown.Value=="Crank")
        x = r2*cosd(theta2);
        y= r2*sind(theta2);

        plot(x,y,"Parent",app.UIAxes);
    elseif(app.LinkDropDown.Value=="Slider")

        r1=r2.*cosd(theta2)+r3.*cosd(theta3);

        plot(theta2,r1,"Parent",app.UIAxes);

    end
```

### Velocity :

```
[r1,r2,r3,theta2,theta3] = func(app);
n=(r3/r2);
```

```

if(app.InputRangeDropDown.Value=="Position of slider , r _1")
for i = 1:length(theta2)
    w_crank(i) = (((r1(i)/r2)-
1)*2*n)/(sind(2*theta2(i))+(2*n*sind(theta2(i))));
    w_conn(i) = (-
r2*cosd(theta2(i))*w_crank(i))/(r3*cosd(theta3(i)));
    v_s(i)=((-r2*sind(theta2(i))*w_crank(i))+(-
r3*sind(theta3(i))*w_conn(i)));
    v_crankx=w_crank(i)*r2*cosd(theta2) ;
    v_cranky= w_crank(i)*r2*sind(theta2)';
    v_connx =w_conn(i)*r3*cosd(theta3);
    v_conny = w_conn(i)*r3*sind(theta3);

end

    if(app.TypeDropDown.Value=="Angular Velocity of Crank")
        plot(r1,w_crank,"Parent",app.UIAxes2);
        hold(app.UIAxes2,'on');

plot(r1,v_crankx+v_cranky,"Parent",app.UIAxes2,"Color",'red');
        hold(app.UIAxes2,'off');
    elseif(app.TypeDropDown.Value=="Angular Velocity of
Connectiong Rod")
        plot(r1,w_conn,"Parent",app.UIAxes2);
        hold(app.UIAxes2,'on');

plot(theta2,v_connx+v_conny,"Parent",app.UIAxes2,"Color",'red');
        hold(app.UIAxes2,'off');
    elseif(app.TypeDropDown.Value=="Vleocity of Slider")
        plot(r1,v_s,"Parent",app.UIAxes2);
    end

elseif(app.InputRangeDropDown.Value=="Angle of Crank , θ_2")
    for i = 1:length(theta2)
        w_crank(i) = (((r1(i)/r2)-
1)*2*n)/(sind(2*theta2(i))+(2*n*sind(theta2(i))));
        w_conn(i) = (-
r2*cosd(theta2(i))*w_crank(i))/(r3*cosd(theta3(i)));
        v_s(i)=((-r2*sind(theta2(i))*w_crank(i))+(-
r3*sind(theta3(i))*w_conn(i)));
        v_crankx=w_crank(i)*r2*cosd(theta2) ;
        v_cranky= w_crank(i)*r2*sind(theta2)';
        v_connx =w_conn(i)*r3*cosd(theta3);
        v_conny = w_conn(i)*r3*sind(theta3);

    end

    if(app.TypeDropDown.Value=="Angular Velocity of Crank")
        plot(theta2,w_crank,"Parent",app.UIAxes2);
        hold(app.UIAxes2,'on');

plot(theta2,v_crankx+v_cranky,"Parent",app.UIAxes2,"Color",'red');
        hold(app.UIAxes2,'off');

```

```

elseif(app.TypeDropDown.Value=="Angular Velocity of
Connectiong Rod")
    plot(theta2,w_conn,"Parent",app.UIAxes2);
    hold(app.UIAxes2,'on');

plot(theta2,v_connx+v_conny,"Parent",app.UIAxes2,"Color",'red');
    hold(app.UIAxes2,'off');
elseif(app.TypeDropDown.Value=="Vleocity of Slider")
    plot(theta2,v_s,"Parent",app.UIAxes2);
end

end

```

#### REFERENCE:

- ▶ Lecture 12 – Position, velocity and acceleration analysis of planar mechanism
- ▶ Lecture 13 – Static Force Analysis
- ▶ <https://in.mathworks.com/matlabcentral/answers/352410-in-app-designer-how-do-i-make-a-button-to-save-that-saves-my-data-from-the-different-uitables-and-e>
- ▶ [https://www.sathyabama.ac.in/sites/default/files/course-material/2020-10/SME1206-1\\_0.pdf](https://www.sathyabama.ac.in/sites/default/files/course-material/2020-10/SME1206-1_0.pdf)