Example Find displacements

3 stresses

E = 706 E = 706Pa Plane stress problem element 3 Do F problem  $B = \frac{1}{x_{13}y_{23}} - \frac{y_{23}y_{13}}{x_{23}} = \frac{y_{23}y_{31}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}y_{23}} = \frac{y_{23}y_{23}y_{23}}{x_{32}} = \frac{y_{23}y_{23}}{x_{32}} =$ 9.50 9.50 9,=0

Cancel in Brownix (instead of 16 matrix) - can do this ahead of time to reduce colculation. Redue Bratix to 3 x 3 Coleleted do this only for hand calculation, well condensation in code Force vector  $\frac{70e9}{4} = \frac{70e9}{4} = \frac{3}{100} = \frac{70e9}{4} = \frac{3}{100} = \frac{1}{100} = \frac{70e9}{4} = \frac{3}{100} = \frac{1}{100} =$ 1 2000 N - 2000 N - 2000 N Total force = Huchion x areo = 2000N localferce = | J7(000N) RIX TRAY

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3x3 3x6 6x1

How to model thermal forces AT, & given Coefficient of thermal expansion flermal ET XAT temp. raised. no short 20  $T = D(E - E^{Th})$  $= D(\mathcal{E} - \mathcal{A}T) = D(\mathcal{B}q - \mathcal{A}T)$   $= D(\mathcal{B}q - \mathcal{A}T)$   $= D(\mathcal{B}q - \mathcal{A}T)$ SETTUR = SETD(B9 - KAT) UR. = 9T((BTDB)9 = 9T(BT) (ADT) dD ne thisterm gas to the force side degin

Thermal + (BTD FXAT dar dar dar Numerical Integration With numerical integration techniques, you can directly in regrate the furctions used in the local stiffnes 3 local force  $I = \left\{ f(\xi) d\xi \right\}$ Standard guord  $= W_1 f(s_1) + W_2 f(s_2) + \dots$ fre weights 1 To mean under the curve between -1 < 5. < 1 Midpoint 5 = 0 - |

naded area One point integration point Two point integration  $\int_{S} f(s)ds$  $= \omega_1 f(\xi_1) + \omega_2 f(\xi_2)$ 2n-1 order polynomial for exactly integrated linear I(E) is exactly integrated f(s) = a765 cubic f(s) is exactly integrated 1(8) = as+bs+cs+d (= 0), weight = 2

Prove 1 I point (\$\xi = 0\$), weight = 2

Try 
$$f(\xi) = a_0 + a_1 x$$

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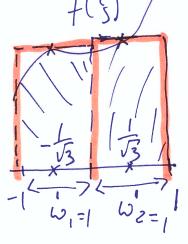
$$= a_0 x + a_1 x + a_2 x + a_3 x = 2$$

$$= a_0 x + a_1 x + a$$

$$= (\omega_{1} + \omega_{2})_{0} + (\omega_{1} + \omega_{2})_{2} = (\omega_{1} + \omega_{2})_{2} + (\omega_{1} + \omega_{2})_{2} = (\omega_{1} + \omega_{2})_{2$$

$$W_1 = W_2 = 1$$

$$S_1 = -S_2 = \frac{1}{\sqrt{3}} V$$
Foints
$$C(s)$$



7.1 in Belegandu textbook Interpolation point table Table  $W_i = 2$  $S_1 = \frac{-1}{\sqrt{3}}, S_2 = \frac{1}{\sqrt{3}}$  $I = \int (3e + x^2 + \frac{1}{x+2}) dx$ weight

J = 1 Recall Host  $B = \frac{dN}{dx} = \int \frac{dN}{ds} = \frac{2}{Le} \left( \frac{2s-1}{2}, -2s, \frac{2s+1}{2} \right)$   $\frac{dN_1}{dx} = \frac{dN_2}{dx} = \frac{2s}{dx} = \frac{2s}{dx}$ Recall Hoat  $K = \int_{C_1}^{C_2} B^T DB Adx = \int_{E}^{C_2} EA \left(\frac{2}{1e}\right)^2 \left(\frac{1}{5} + \frac{1}{2}\right)^2 - 25(5 + \frac{1}{2})^2 + 25(5 + \frac{1}{2})^2$ Since its a quadratic in S => use 2-point integration (exact up to cubicins) det J d {  $n = 2, S = \sqrt{3}, S = \sqrt{3}$ =dxweights = K = f(-1/3) + f(1/3) = BTEBAdet J BTEBAdet J/ 8=+31

$$=\frac{EA}{3L^{e}}\begin{bmatrix} 7 - 8 & 1 \\ -8 & 16 - 8 \\ 1 - 8 & 7 \end{bmatrix}$$
Code