

Boundary Conditions

Static condensation
(harder to implement,
more general)

Penalty method
(easy to implement,
restrictive)

Until now, we have applied zero displacement B.C.'s ($u_2 = 0$) by deleting row & column 2 of global matrices.

Now let's say ~~$u_2 = 0$~~ $u_2 = \delta$ given

$$\begin{bmatrix} K_{11} & K_{12} & K_{13} & \dots & K_{1N} \\ \cancel{K_{21}} & \cancel{K_{22}} & \cancel{K_{23}} & \dots & \cancel{K_{2N}} \\ K_{31} & K_{32} & K_{33} & \dots & K_{3N} \\ \vdots & & & & \vdots \\ K_{N1} & K_{N2} & \dots & \dots & K_{NN} \end{bmatrix} \begin{bmatrix} u_1 \\ \cancel{u_2} \\ u_3 \\ \vdots \\ u_N \end{bmatrix} = \begin{bmatrix} f_1 \\ \cancel{f_2} \\ \vdots \\ f_N \end{bmatrix}$$

(Note: In the original image, the second row and column of the stiffness matrix and the second element of the displacement vector and force vector are crossed out with a red line. The displacement u_2 is boxed in red and labeled with a red δ .)

Serial implementation

$$\begin{bmatrix} K_{11} & K_{13} & K_{14} & \dots & K_{1N} \\ K_{31} & K_{33} & K_{34} & \dots & K_{3N} \\ \vdots & \vdots & & & \vdots \\ K_{N1} & K_{N3} & \dots & \dots & K_{NN} \end{bmatrix} \begin{bmatrix} u_1 \\ u_3 \\ \vdots \\ u_N \end{bmatrix} = \begin{bmatrix} f_1 - K_{12}u_2 \\ f_3 - K_{32}u_2 \\ \vdots \\ f_N - K_{N2}u_2 \end{bmatrix}$$

Penalty method (approximation method to implement B.C.)

* Modify the P.E. *

Add $\frac{1}{2} C (u_2 - \delta)^2$ term where C is a very large ~~term~~ number

$$\Pi = \Pi^{(a)} + \Pi^{(b)} + \dots + \frac{1}{2} C (u_2 - \delta)^2$$

Minimize Π

$$\frac{\partial \Pi}{\partial u_2} = \frac{\partial \Pi^{(a)}}{\partial u_2} + \frac{\partial \Pi^{(b)}}{\partial u_2} + \dots + C (u_2 - \delta)$$

Global System

$$\begin{bmatrix} \frac{\partial \Pi}{\partial u_1} \\ \frac{\partial \Pi}{\partial u_2} \\ \vdots \end{bmatrix} = \begin{bmatrix} k_{21} & k_{22} + C & k_{23} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_N \end{bmatrix} - \begin{bmatrix} f_1 \\ f_2 + C\delta \\ \vdots \\ f_N \end{bmatrix}$$

Add C to k_{22} term } add $C\delta$ to the f_2 term in global matrix/vector

General constraint

$$u_2 \cos \theta + u_3 \sin \theta = \delta$$

$$\Pi = \Pi^{(a)} + \Pi^{(b)} + \dots + \frac{1}{2} C (u_2 \cos \theta + u_3 \sin \theta - \delta)^2$$

$$\frac{\partial \Pi}{\partial u_2} = \frac{\partial \Pi}{\partial u_2} + \dots + C(u_2 \cos \theta + u_3 \sin \theta - \delta) \cos \theta$$

$$\frac{\partial \Pi}{\partial u_3} = \frac{\partial \Pi}{\partial u_3} + \dots + C(u_2 \cos \theta + u_3 \sin \theta - \delta) \sin \theta$$

$$\begin{bmatrix} \frac{\partial \Pi}{\partial u_1} \\ \frac{\partial \Pi}{\partial u_2} \\ \frac{\partial \Pi}{\partial u_3} \\ \vdots \end{bmatrix} = \begin{bmatrix} K_{22} + C \cos^2 \theta & K_{23} + C \sin^2 \theta \\ K_{32} + C \sin^2 \theta & K_{33} + C \cos^2 \theta \\ \vdots & \vdots \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ \vdots \end{bmatrix}$$

$$= \begin{bmatrix} f_1 \\ f_2 + C \delta \cos \theta \\ f_3 + C \delta \sin \theta \\ \vdots \end{bmatrix}$$

C is a choice made by the user
- it is a guess estimate

Usually $C \approx 10^4 \times K_{\max}$

Galerkin's method in 1D

So far in Aero 510

The Galerkin method is very general \rightarrow CFD, etc.

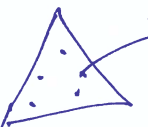
- L Direct
- L P.E. minimization (elastic)
- L Galerkin (principle of virtual work)

all these methods use the same code

These methods generate 'local' stiffness matrix, 'local' force vector.

Some definitions

Strong Form

- Differential equation
 - valid at every point on the domain
-  $\rightarrow \frac{\partial^2 u}{\partial x^2} = 0$

Weak Form

- weighted average of the strong form over an element

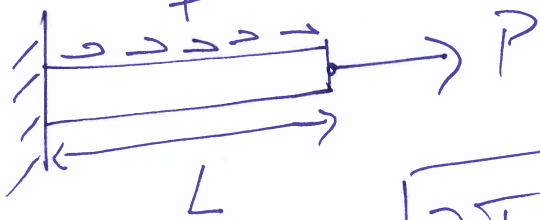
$$\int \frac{\partial^2 u}{\partial x^2} w \, dV = 0$$



weight

As you refine your grid, the solution to the weak form will converge on the strong form.

Example of derivation of 'local' stiffness matrix & 'local' force vector



Strong form: $\frac{\partial \sigma}{\partial x} + f = 0$ 1D equilibrium equation

Solve $\Rightarrow \left[E \frac{\partial^2 u}{\partial x^2} + f = 0 \right]$ at every point
 such that $u = 0 @ x = 0$
 $F(L) = P @ x = L$

Strong

Solve $\int_{x_1}^{x_2} \left(E \frac{\partial^2 u}{\partial x^2} + f \right) w dx = 0$ for each element
 such that
 $u = 0 @ x = 0$
 $F(L) = P @ x = L$

Weak

FEM $u = u_1 N_1 + u_2 N_2$
 $w = w_1 N_1 + w_2 N_2$
 for each element

In Galerkin method, the weight function is interpolated using the shape functions

Weak Form for 1 Element

$$\int_{x_1}^{x_2} (E u'' + f) w dx = 0$$

$$\int_{x_1}^{x_2} (E \underline{u'' w} + f w) dx = 0$$

Apply the u-v rule of differentiation

$$u' w' = (u' w)' - u'' w$$

$$\frac{\partial^2 u}{\partial x^2} w = \frac{\partial}{\partial x} \left(\frac{\partial u}{\partial x} w \right) - \frac{\partial u}{\partial x} \frac{\partial w}{\partial x}$$

$$= \int_{x_1}^{x_2} (E \underline{(u' w)' - u' w'} + f w) dx = 0$$

Divergence Theorem

$$\int_{x_1}^{x_2} (u' w)' dx = u' w \Big|_{x_1}^{x_2}$$

↑ integral
↑ derivative

$$= u'(x_2) w(x_2) - u'(x_1) w(x_1)$$

Now we have the reduced weak form

$$= E u' w \Big|_{x_1}^{x_2} - \int_{x_1}^{x_2} E u' w' dx + \int_{x_1}^{x_2} f w dx$$

∴ **only has 1st derivatives**

Notation

$$u'' = \frac{\partial^2 u}{\partial x^2}$$

$$= N_1'' u_1 + N_2'' u_2$$

where

$$N_1'' = \frac{\partial^2 N_1}{\partial x^2}$$