

Riprendiamo esercizio 2 della scorsa volta. Domanda e)

Se $\kappa = \frac{mg}{8l}$, allora $\lambda = 2$, $(\theta_1, x_1) = (\frac{\pi}{2}, 0)$ stabile.

$$\underline{H}(\theta, x) = \begin{pmatrix} -mgl \sin \theta - 2\kappa l x \cos \theta & -2\kappa l \sin \theta \\ -2\kappa l \sin \theta & -\kappa \end{pmatrix} \Rightarrow \underline{H}(\theta_1, x_1) = \begin{pmatrix} -mgl & -2\kappa l \\ -2\kappa l & -\kappa \end{pmatrix}$$

Se $\kappa = \frac{mg}{8l} \Rightarrow \underline{H}(\theta_1, x_1) = \begin{pmatrix} -mgl & -\frac{mg}{4} \\ -\frac{mg}{4} & -\frac{mg}{8l} \end{pmatrix}$; $T = \frac{2}{3} m l^2 \dot{\theta}^2 + \frac{3}{4} m \dot{x}^2$

Equazione dei piccoli moti: $\underline{A}(\underline{q}_e) \ddot{\underline{\eta}} - \underline{H}(\underline{q}_e) \underline{\eta} = 0$, dove $\underline{\eta} = \underline{q} - \underline{q}_e$

$\eta_1 = \theta - \frac{\pi}{2}$, $\eta_2 = x$

$\underline{A}(\underline{q})$ matrice di massa: $T = \frac{1}{2} \underline{A}(\underline{q}) \dot{\underline{q}} \cdot \dot{\underline{q}}$

$\underline{A}(\theta, x) = \begin{pmatrix} \frac{4}{3} m l^2 & 0 \\ 0 & \frac{3}{2} m \end{pmatrix} = \underline{A}(\theta_1, x_1)$ (non dipende da θ e x)

$T = \frac{1}{2} \left(\frac{4}{3} m l^2 \dot{\theta}^2 + \frac{3}{2} m \dot{x}^2 \right)$

$$\begin{cases} \frac{4}{3} m l^2 \ddot{\eta}_1 + mgl \eta_1 + \frac{mg}{4} \eta_2 = 0 \\ \frac{3}{2} m \ddot{\eta}_2 + \frac{mg}{4} \eta_1 + \frac{mg}{8l} \eta_2 = 0 \end{cases}$$

Cerco soluzioni $\underline{\eta} = e^{\mu t} \underline{\eta}_0$: $\dot{\underline{\eta}} = \mu e^{\mu t} \underline{\eta}_0$: $\ddot{\underline{\eta}} = \mu^2 e^{\mu t} \underline{\eta}_0$

$\underline{A}(\underline{q}_e) \ddot{\underline{\eta}} - \underline{H}(\underline{q}_e) \underline{\eta} = 0 \rightarrow \mu^2 \underline{A}(\underline{q}_e) \underline{\eta}_0 - \underline{H}(\underline{q}_e) \underline{\eta}_0 = 0$

Ha soluzioni se $\det(\underline{H}(\underline{q}_e) - \mu^2 \underline{A}(\underline{q}_e)) = 0$, μ^2 autovalori di

Se $\mu^2 < 0$, $\underline{\eta} = e^{\pm i \sqrt{-\mu^2} t} \underline{\eta}_0$, $\omega = \sqrt{-\mu^2}$ $\underline{H}(\underline{q}_e)$ rispetto ad $\underline{A}(\underline{q}_e)$

Devo risolvere:

o autofrequenza.

$$\begin{vmatrix} -mgl - \mu^2 \frac{4}{3} m l^2 & -\frac{mg}{4} \\ -\frac{mg}{4} & -\frac{mg}{8l} - \mu^2 \frac{3}{2} m \end{vmatrix} = 0 \Leftrightarrow \begin{vmatrix} -gl - \mu^2 \frac{4}{3} l^2 & -\frac{g}{4} \\ -\frac{g}{4} & -\frac{g}{8l} - \mu^2 \frac{3}{2} \end{vmatrix} = 0$$

$2l^2 \mu^4 + \frac{3}{2} gl \mu^2 + \frac{1}{6} gl \mu^2 + \frac{1}{8} g^2 - \frac{1}{16} g^2 = 0 \Leftrightarrow \mu^4 + \frac{5}{6} \frac{g}{l} \mu^2 + \frac{1}{32} \frac{g^2}{l^2} = 0$

Risolvendola \rightarrow pulsazioni proprie

$\mu^2 = \frac{-\frac{5}{6} \pm \sqrt{\frac{25}{36} - \frac{1}{8}}}{2} \frac{g}{l} = \left(-\frac{5}{12} \pm \frac{1}{2} \sqrt{\frac{50-9}{72}} \right) \frac{g}{l} = \left(-\frac{5}{12} \pm \frac{1}{12} \sqrt{\frac{41}{2}} \right) \frac{g}{l}$

$\omega_1 = \sqrt{\frac{5}{12} - \frac{1}{12} \sqrt{\frac{41}{2}}} \sqrt{\frac{g}{l}}$
 $\omega_2 = \sqrt{\frac{5}{12} + \frac{1}{12} \sqrt{\frac{41}{2}}} \sqrt{\frac{g}{l}}$

Autovettori: $\underline{y}^{(1)}$ relativo a ω_1 e $\underline{y}^{(2)}$ relativo a ω_2 .

$$(H(\underline{q}_e) - \mu_1 A(\underline{q}_e)) \underline{y}^{(1)} = \underline{0} \quad (H(\underline{q}_e) - \mu_2 A(\underline{q}_e)) \underline{y}^{(2)} = \underline{0}$$

Per la prima:

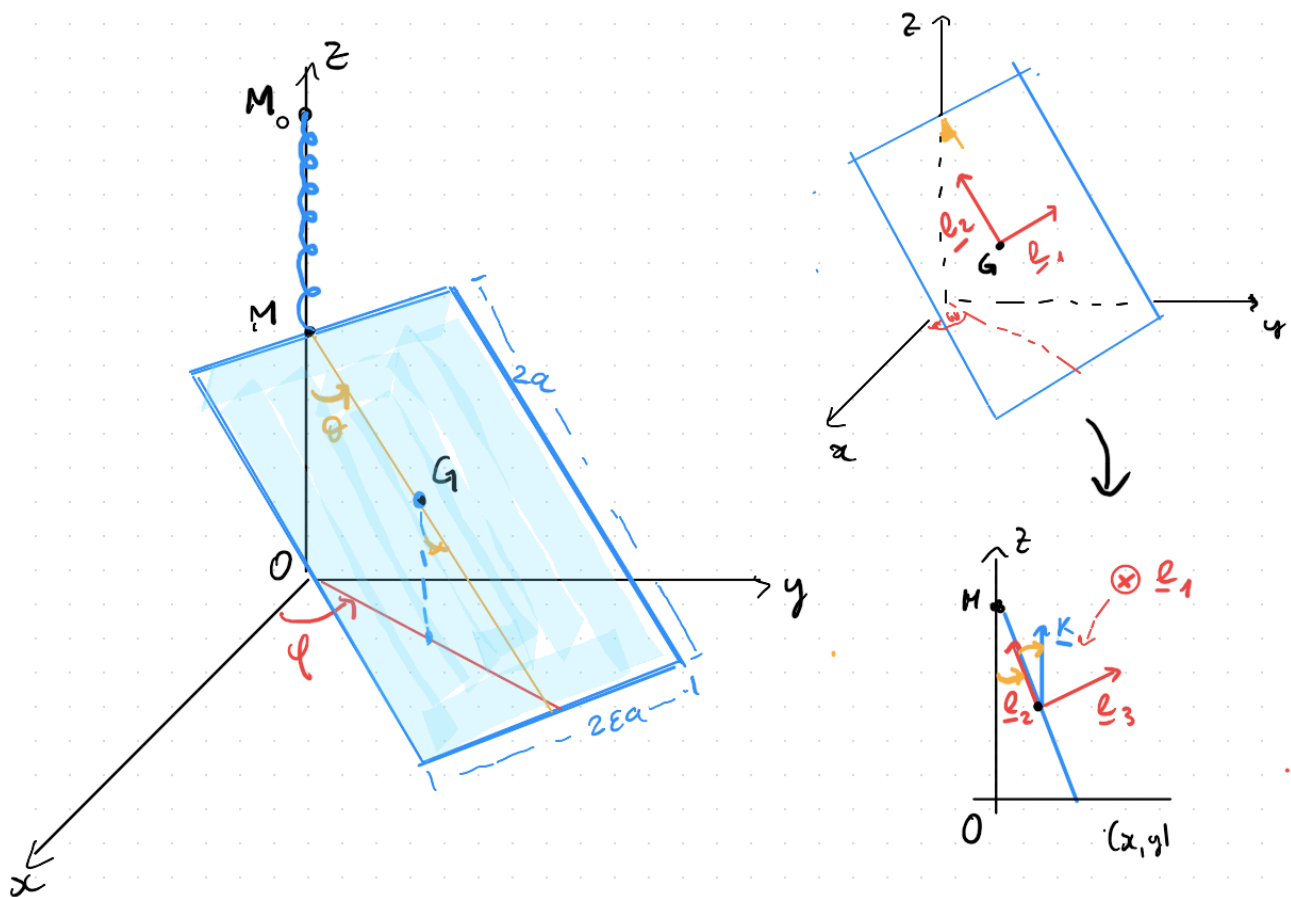
$$\begin{pmatrix} -mg\ell + \left(\frac{5}{12} - \frac{1}{12}\sqrt{\frac{41}{2}}\right) \frac{g}{\ell} \frac{4}{3} m \ell^2 & -\frac{mg}{4} \\ -\frac{mg}{4} & -\frac{mg}{8}\ell + \left(\frac{5}{12} - \frac{1}{12}\sqrt{\frac{41}{2}}\right) \frac{g}{\ell} \frac{3}{2} m \end{pmatrix} \begin{pmatrix} y_1^{(1)} \\ y_2^{(1)} \end{pmatrix} = \underline{0}$$

Eliminando mg (prima riga)

$$\left(-\ell + \left(\frac{5}{12} - \frac{1}{12}\sqrt{\frac{41}{2}}\right) \frac{4}{3} \ell\right) y_1^{(1)} - \frac{1}{4} y_2^{(1)} = 0 \Rightarrow \left(-\frac{4}{3} - \frac{1}{3}\sqrt{\frac{41}{2}}\right) \ell y_1^{(1)} - \frac{1}{4} y_2^{(1)} = 0$$

$$\left(-\frac{16}{3} - \frac{4}{3}\sqrt{\frac{41}{2}}\right) \ell y_1^{(1)} - y_2^{(1)} = 0 \Rightarrow \underline{y}^{(1)} = \begin{pmatrix} 1 \\ -\frac{16}{3}\ell - \frac{4}{3}\sqrt{\frac{41}{2}}\ell \end{pmatrix}$$

Per $\underline{y}^{(2)}$ cambia segno davanti alla radice: $\Rightarrow \underline{y}^{(2)} = \begin{pmatrix} 1 \\ -\frac{16}{3}\ell + \frac{4}{3}\sqrt{\frac{41}{2}}\ell \end{pmatrix}$



ESERCIZIO "LAMINA"

$$M_0 = (0, 0, 4a) ; M = (0, 0, 2a \cos \theta) ; G = (a \sin \theta \cos \varphi, a \sin \theta \sin \varphi, a \cos \theta)$$

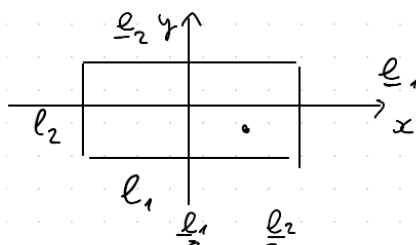
$$U = -mgz_G - \frac{1}{2}K|M_0M|^2 + \text{cost} =$$

$$-8ka^2 = \text{costante}$$

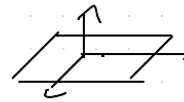
$$= -mga \cos \theta - \frac{1}{2}K(4a - 2a \cos \theta)^2 + \text{cost} = -mga \cos \theta + 8Ka^2 \cos \theta - 2Ka^2 \cos^2 \theta + \text{cost}$$

Energia cinetica:

$$T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G \underline{\omega} \cdot \underline{\omega}$$

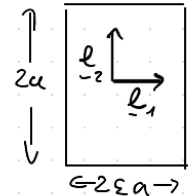


Momenti di inerzia di una lamina rettangolare, rispetto ai suoi tre assi principali di inerzia



$$I_1 = \sigma \int_{-l_1/2}^{l_1/2} dx \int_{-l_2/2}^{l_2/2} dy y^2 = \frac{m}{l_1 l_2} \int_{-l_2/2}^{l_2/2} y^2 dy = \frac{1}{12} \frac{m}{l_1} l_2^3 = \frac{1}{12} m l_2^2$$

$$I_2 = \frac{1}{12} m l_1^2 ; \quad I_3 = \frac{1}{12} m (l_1^2 + l_2^2)$$



In un'opportuna terna solidale alla lamina:

$$\underline{\underline{I}}_G = \begin{pmatrix} \frac{1}{12} m (2a)^2 & 0 & 0 \\ 0 & \frac{1}{12} m (2\epsilon a)^2 & 0 \\ 0 & 0 & \frac{1}{12} m [(2a)^2 + (2\epsilon a)^2] \end{pmatrix} = \begin{pmatrix} \frac{1}{3} m a^2 & 0 & 0 \\ 0 & \frac{1}{3} m \epsilon^2 a^2 & 0 \\ 0 & 0 & \frac{1}{3} m (1 + \epsilon^2) a^2 \end{pmatrix}$$

$$\underline{e}_1 = -\sin \varphi \underline{i} + \cos \varphi \underline{j}$$

$$\underline{e}_2 = -\sin \theta \cos \varphi \underline{i} - \sin \theta \sin \varphi \underline{j} + \cos \theta \underline{k}$$

$$\begin{aligned} \underline{e}_3 &= \underline{e}_1 \wedge \underline{e}_2 = (-\sin \varphi \underline{i} + \cos \varphi \underline{j}) \wedge (-\sin \theta \cos \varphi \underline{i} - \sin \theta \sin \varphi \underline{j} + \cos \theta \underline{k}) = \\ &= \sin \theta \sin^2 \varphi \underline{k} + \cos \theta \sin \varphi \underline{j} + \sin \theta \cos^2 \varphi \underline{k} + \cos \theta \cos \varphi \underline{i} = \\ &= \cos \theta \cos \varphi \underline{i} + \cos \theta \sin \varphi \underline{j} + \sin \theta \underline{k} \end{aligned}$$

$$\begin{cases} \underline{e}_1 = -\sin \varphi \underline{i} + \cos \varphi \underline{j} \\ \underline{e}_2 = -\sin \theta \cos \varphi \underline{i} - \sin \theta \sin \varphi \underline{j} + \cos \theta \underline{k} \\ \underline{e}_3 = \cos \theta \cos \varphi \underline{i} + \cos \theta \sin \varphi \underline{j} + \sin \theta \underline{k} \end{cases}$$

$$\text{Inoltre } \underline{k} = \underline{e}_2 \cos \theta + \underline{e}_3 \sin \theta$$

Devo ancora determinare \underline{v}_G e $\underline{\omega}$.

$$\underline{v}_G - \underline{v}_M = \underline{\omega} \wedge (G-M)$$

$$\begin{cases} \underline{v}_G = a(\dot{\theta} \cos \theta \cos \varphi - \dot{\varphi} \sin \theta \sin \varphi) \underline{i} + a(\dot{\theta} \cos \theta \sin \varphi + \dot{\varphi} \sin \theta \cos \varphi) \underline{j} - a \dot{\theta} \sin \theta \underline{k}; \\ \underline{v}_M = -2a \dot{\theta} \sin \theta \underline{k}; \quad G-M = a \sin \theta \cos \varphi \underline{i} + a \sin \theta \sin \varphi \underline{j} - a \cos \theta \underline{k} \end{cases}$$

$$\begin{aligned} & a(\dot{\theta} \cos \theta \cos \varphi - \dot{\varphi} \sin \theta \sin \varphi) \underline{i} + a(\dot{\theta} \cos \theta \sin \varphi + \dot{\varphi} \sin \theta \cos \varphi) \underline{j} + a \dot{\theta} \sin \theta \underline{k} = \\ & = (\omega_x \underline{i} + \omega_y \underline{j} + \omega_z \underline{k}) \wedge (a \sin \theta \cos \varphi \underline{i} + a \sin \theta \sin \varphi \underline{j} - a \cos \theta \underline{k}) \quad (\text{elimino } a) \end{aligned}$$

$$(\dot{\theta} \cos \theta \cos \varphi - \dot{\varphi} \sin \theta \sin \varphi) \underline{i} + (\dot{\theta} \cos \theta \sin \varphi + \dot{\varphi} \sin \theta \cos \varphi) \underline{j} + \dot{\theta} \sin \theta \underline{k} =$$

$$= \omega_x \sin \theta \sin \varphi \underline{k} + \omega_x \cos \theta \underline{j} - \omega_y \sin \theta \cos \varphi \underline{k} - \omega_y \cos \theta \underline{i} + \omega_z \sin \theta \cos \varphi \underline{j} - \omega_z \sin \theta \sin \varphi \underline{i}$$

Componente per componente:

$$\begin{cases} \dot{\theta} \cos \theta \cos \varphi - \dot{\varphi} \sin \theta \sin \varphi = -\omega_y \cos \theta - \omega_z \sin \theta \sin \varphi \\ \dot{\theta} \cos \theta \sin \varphi + \dot{\varphi} \sin \theta \cos \varphi = \omega_x \cos \theta + \omega_z \sin \theta \cos \varphi \\ \dot{\theta} \sin \theta = \omega_x \sin \theta \sin \varphi - \omega_y \sin \theta \cos \varphi \end{cases} \quad \begin{cases} \omega_x = \dot{\theta} \sin \varphi \\ \omega_y = -\dot{\theta} \cos \varphi \\ \omega_z = \dot{\varphi} \end{cases}$$

$$\underline{\omega} = \dot{\theta} \sin \varphi \underline{i} - \dot{\theta} \cos \varphi \underline{j} + \dot{\varphi} \underline{k}$$

$$\underline{\omega} = \dot{\theta} \sin \varphi \underline{i} - \dot{\theta} \cos \varphi \underline{j} + \dot{\varphi} \underline{k} \rightarrow \underline{\omega} = -\dot{\theta} \underline{e}_1 + \dot{\varphi} \cos \theta \underline{e}_2 + \dot{\varphi} \sin \theta \underline{e}_3$$

$$\begin{aligned} v_G^2 &= a^2 (\dot{\theta}^2 \cos^2 \theta \cos^2 \varphi - 2 \dot{\theta} \dot{\varphi} \sin \theta \cos \theta \sin \varphi \cos \varphi + \dot{\varphi}^2 \sin^2 \theta \sin^2 \varphi) + \\ &+ a^2 (\dot{\theta}^2 \cos^2 \theta \sin^2 \varphi + 2 \dot{\theta} \dot{\varphi} \sin \theta \cos \theta \sin \varphi \cos \varphi + \dot{\varphi}^2 \sin^2 \theta \cos^2 \varphi) + a^2 \dot{\theta}^2 \sin^2 \theta = \\ &= a^2 \dot{\theta}^2 \cos^2 \theta \cos^2 \varphi + a^2 \dot{\theta}^2 \cos^2 \theta \sin^2 \varphi + a^2 \dot{\varphi}^2 \sin^2 \theta \sin^2 \varphi + a^2 \dot{\varphi}^2 \sin^2 \theta \cos^2 \varphi + a^2 \dot{\theta}^2 \sin^2 \theta = \\ &= a^2 \dot{\theta}^2 \cos^2 \theta + a^2 \dot{\varphi}^2 \sin^2 \theta + a^2 \dot{\theta}^2 \sin^2 \theta = a^2 \dot{\theta}^2 + a^2 \sin^2 \theta \dot{\varphi}^2 \end{aligned}$$

Calcolo $\underline{I}_G \underline{\omega} \cdot \underline{\omega}$ nella terna mobile:

$$\begin{aligned} \underline{I}_G \underline{\omega} \cdot \underline{\omega} &= \frac{1}{3} m a^2 \begin{pmatrix} 1 & 0 & 0 \\ 0 & \varepsilon^2 & 0 \\ 0 & 0 & 1 + \varepsilon^2 \end{pmatrix} \begin{pmatrix} -\dot{\theta} \\ \dot{\varphi} \cos \theta \\ \dot{\varphi} \sin \theta \end{pmatrix} \cdot \begin{pmatrix} -\dot{\theta} \\ \dot{\varphi} \cos \theta \\ \dot{\varphi} \sin \theta \end{pmatrix} = \frac{1}{3} m a^2 \begin{pmatrix} -\dot{\theta} \\ \varepsilon^2 \dot{\varphi} \cos \theta \\ (1 + \varepsilon^2) \dot{\varphi} \sin \theta \end{pmatrix} \cdot \begin{pmatrix} -\dot{\theta} \\ \dot{\varphi} \cos \theta \\ \dot{\varphi} \sin \theta \end{pmatrix} \\ &= \frac{1}{3} m a^2 (\dot{\theta}^2 + \varepsilon^2 \cos^2 \theta \dot{\varphi}^2 + \sin^2 \theta \dot{\varphi}^2 + \varepsilon^2 \sin^2 \theta \dot{\varphi}^2) = \frac{1}{3} m a^2 (\dot{\theta}^2 + \varepsilon^2 \dot{\varphi}^2 + \sin^2 \theta \dot{\varphi}^2) \end{aligned}$$

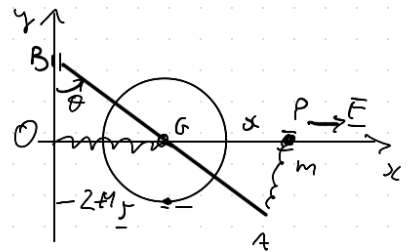
$$\begin{aligned} T &= \frac{1}{2} m v_G^2 + \frac{1}{2} \underline{I}_G \underline{\omega} \cdot \underline{\omega} = \frac{1}{2} m a^2 (\dot{\theta}^2 + \sin^2 \theta \dot{\varphi}^2) + \frac{1}{3} m a^2 (\dot{\theta}^2 + \varepsilon^2 \dot{\varphi}^2 + \sin^2 \theta \dot{\varphi}^2) = \\ &= \frac{2}{3} m a^2 \dot{\theta}^2 + \frac{2}{3} m a^2 \sin^2 \theta \dot{\varphi}^2 + \frac{1}{6} m \varepsilon^2 a^2 \dot{\varphi}^2 \end{aligned}$$

ESAME 30/06/2010

$$B = (0, l \cos \theta); G = (l \sin \theta, 0); P = (x, 0)$$

$$A = (2l \sin \theta, -l \cos \theta); C = (l \sin \theta, -R)$$

- Equilibrio e stabilità. $\vec{F}_i \cdot d\vec{x}_i$

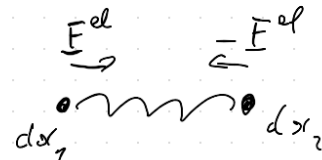


$$U = -\frac{1}{2} k |OG|^2 - \frac{1}{2} k |AP|^2 + \int \vec{F} \cdot d\vec{P} + \text{cost} = -\frac{1}{2} k |OG|^2 - \frac{1}{2} k |AP|^2 + Fx + \text{cost}$$

$$= -\frac{1}{2} k l^2 \sin^2 \theta - \frac{1}{2} k [(x - 2l \sin \theta)^2 + l^2 \cos^2 \theta] + \text{cost} =$$

$$= -\frac{1}{2} k l^2 \sin^2 \theta - \frac{1}{2} k x^2 + 2k l x \sin \theta - 2k l^2 \sin^2 \theta - \frac{1}{2} k l^2 \cos^2 \theta + \text{cost}$$

Lo svolgeremo la settimana prossima



$$U = -\frac{1}{2} k |x_1 - x_2|^2$$

Compito di Meccanica Razionale del 30 Giugno 2010 - Tema A

1) Cinematica relativa. (Max 1 pagina)

2) In un piano verticale Oxy un disco omogeneo di massa M e raggio R è vincolato a rotolare senza strisciare sull'asse $y = -R$; un'asta omogenea AB di egual massa M e lunghezza 2ℓ ha l'estremo B mobile sull'asse delle y ed il baricentro G incernierato al centro del disco. Un punto materiale P di massa m è mobile sull'asse delle x sollecitato da una forza orizzontale costante \vec{F} , con $F > 0$, e collegato con una molla ideale, di costante $k > 0$, all'estremo A dell'asta. Una forza elastica di costante $h = k$ è infine applicata al centro del disco e diretta verso l'origine O . Si suppongano i vincoli perfetti e si scelgano come coordinate lagrangiane l'angolo di rotazione θ che l'asta forma con l'asse verticale e l'ascissa x del punto P .

- Determinare le posizioni di equilibrio del sistema e studiarne la stabilità.
- Determinare la velocità angolare del disco.
- Calcolare le equazioni differenziali del moto nella forma di Lagrange.
- Linearizzare tali equazioni nell'intorno di una posizione di equilibrio stabile e determinare le pulsazioni proprie.

