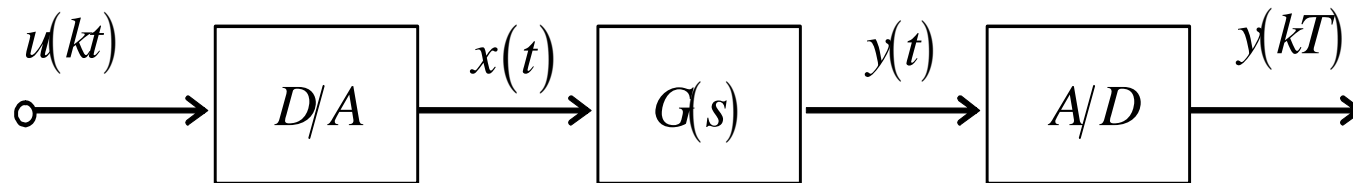


# ***Chapter 5***

## **Sampled Data Control System**

## 5.1 Discrete Models of Sampled Data Systems



The sampled-data system

**Task:** We here want to compute the discrete transfer function between the samples from digital computer to the D/A converter and samples picked up by the A/D converter. To this end, assume that the D/A is a zero-order holder (ZOH).

**Method:** Apply the unit pulse input,

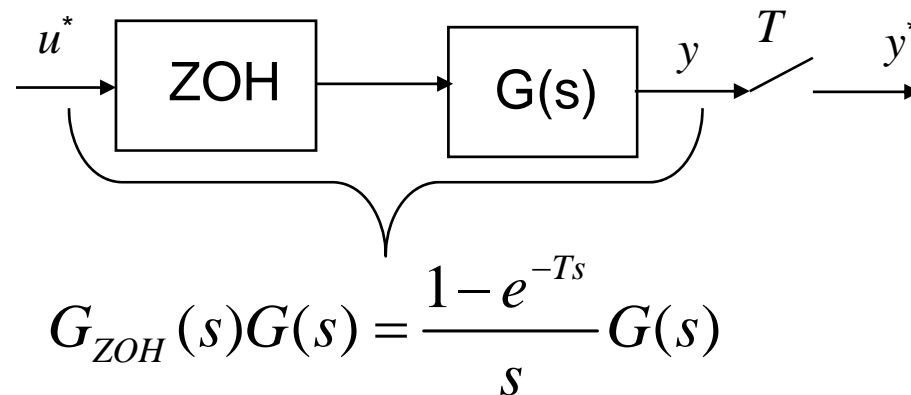
$$u(k) = \begin{cases} 1, & \text{for } k = 0, \\ 0, & \text{for } k \neq 0. \end{cases}$$

Then the corresponding ZOH output is

$$1(t) - 1(t - T).$$

The Laplace transform of the plant output  $Y(s)$  is given by

$$Y(s) = G(s)L\{1(t) - 1(t - T)\} = G(s)\frac{1 - e^{-Ts}}{s}.$$



The required discrete transfer function is

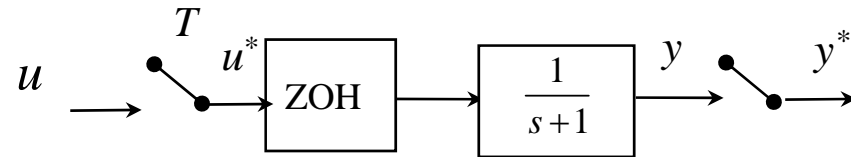
$$\begin{aligned}
 G(z) &= Z \{ y(kT) \} = Z \left\{ \left\{ L^{-1} \{ Y(s) \} \right\} \Big|_{t=Tk} \right\} \equiv Z (Y(s)). \\
 G(z) &= Z \left\{ \frac{1 - e^{-Ts}}{s} G(s) \right\} \\
 &= Z \left\{ \frac{G(s)}{s} \right\} - Z \left\{ \frac{e^{-Ts} G(s)}{s} \right\} \\
 &= Z \left\{ \frac{G(s)}{s} \right\} - z^{-1} Z \left\{ \frac{G(s)}{s} \right\}
 \end{aligned}$$

because  $e^{-Ts}$  is exactly a delay of one period.

$$G(z) = (1 - z^{-1}) Z \left\{ \frac{G(s)}{s} \right\}.$$

Plant discrete  
Transfer function  
with a ZOH

## An Example



$$\frac{Y(z)}{U(z)} = (1 - z^{-1}) Z \left\{ \frac{G(s)}{s} \right\}$$

$$= \left( \frac{z-1}{z} \right) Z \left\{ \frac{1}{s(s+1)} \right\}$$

$$= \frac{1 - e^{-T}}{z - e^{-T}}$$



$$\frac{z(1 - e^{-T})}{(z-1)(z - e^{-T})}$$

## 5.2 Block-Diagram Analysis

If a continuous signal is  $f(t)$ , the sampled signal is given by

$$f^*(t) = \sum_{k=-\infty}^{+\infty} f(t) \delta(t - kT)$$

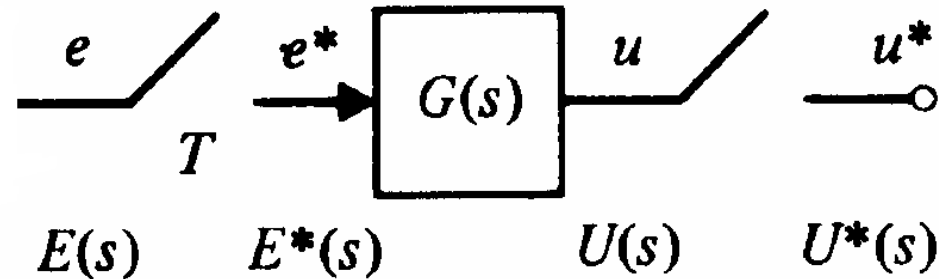
The Laplace transform of  $f^*(t)$  is

$$F^*(s) = \frac{1}{T} \sum_{n=-\infty}^{\infty} F(s - jn\omega_s)$$

where  $F(s)$  is the transform of  $f(t)$ . Note that  $F^*(s)$  is periodic but  $F(s)$  is usually not.

We assume from now on that the sampling rate is chosen and fixed.

Now consider the system.



$$U(s) = G(s)E^*(s)$$

If the transform of the signal to be sampled is a product of a transform that is already periodic (such as  $E^*(s)$ ) and one that is not, then we have the most important relation:

$$U^*(s) = \left( G(s)E^*(s) \right)^* = G^*(s)E^*(s)$$

- To show

$$\left( G(s)E^*(s) \right)^* = G^*(s)E^*(s)$$

- we see

$$E^*(s) = \frac{1}{T} \sum_{k=-\infty}^{\infty} E(s - jk\omega_s)$$

$$\therefore E^*(s - jn\omega_s) = \frac{1}{T} \sum_{k=-\infty}^{\infty} E(s - jn\omega_s - jk\omega_s)$$

$$\stackrel{n+k=l}{=} \frac{1}{T} \sum_{l=-\infty}^{\infty} E(s - jl\omega_s)$$

$$= E^*(s)$$



So,

$$\begin{aligned}\left(G(s)E^*(s)\right)^* &= \frac{1}{T} \sum_{k=-\infty}^{\infty} G(s - jk\omega_s)E^*(s - jk\omega_s) \\ &= \frac{1}{T} \sum_{k=-\infty}^{\infty} G(s - jk\omega_s)E^*(s) \\ &= \frac{1}{T} E^*(s) \sum_{k=-\infty}^{\infty} G(s - jk\omega_s) \\ &= E^*(s)G^*(s)\end{aligned}$$

The Laplace transform of a sampled signal  $Y^*(s)$  is related to the corresponding  $z$ -transform via

$$Y(z) = Y^*(s) \Big|_{z=e^{sT}}$$

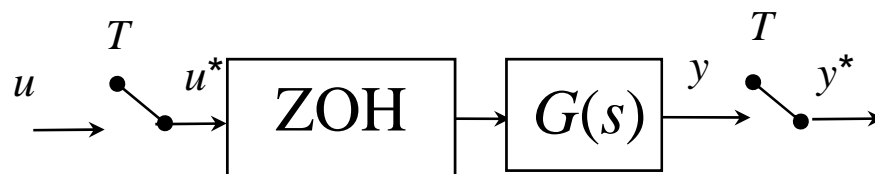
Apply this to

$$U^*(s) = G^*(s) E^*(s)$$

We get

$$U(z) = G(z) E(z)$$

## Example 1



$$Y(s) = \frac{1 - e^{-Ts}}{s} G(s) U^*(s)$$

$$Y^*(s) = \left( \frac{1 - e^{-Ts}}{s} G(s) U^*(s) \right)^* = \left( \frac{1 - e^{-Ts}}{s} G(s) \right)^* U^*(s)$$

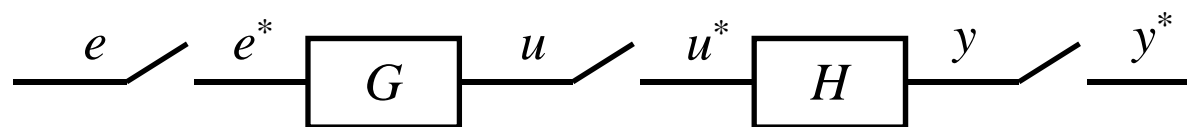
$$= (1 - e^{-Ts}) \left( \frac{G(s)}{s} \right)^* U^*(s)$$

$$Y(z) = (1 - z^{-1}) Z \left( \frac{G(s)}{s} \right) U(z) = G(z) U(z), G(z) = (1 - z^{-1}) Z \left( \frac{G(s)}{s} \right)$$

Where

$$\begin{aligned} \left(1 - e^{-Ts}\right) \Big|_{s \rightarrow s + j\omega_s} &= 1 - e^{-T(s + j\omega_s)} \\ &= 1 - e^{-Ts - jT\omega_s} = 1 - e^{-Ts} e^{-jT\omega_s} \\ &= 1 - e^{-Ts} e^{-j2\pi} = 1 - e^{-Ts}, \end{aligned}$$

is periodic.



We know

$$U(s) = E^*(s)G(s) \rightarrow U^*(s) = E^*(s)G^*(s)$$

Similarly,

$$Y(s) = H(s)U^*(s) \rightarrow Y^*(s) = H^*(s)U^*(s)$$

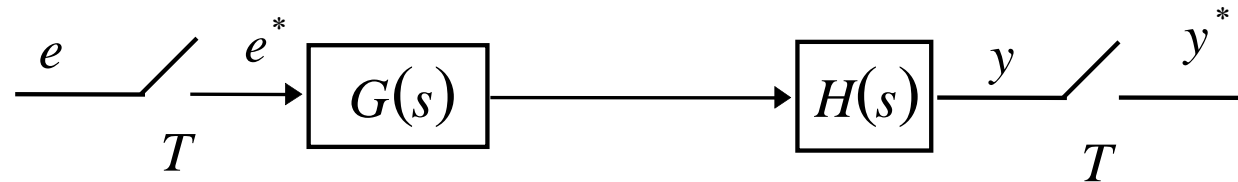
Thus,

$$Y^*(s) = H^*(s)G^*(s)E^*(s)$$

or

$$Y(z) = H(z)G(z)E(z)$$

For the system



$$Y(s) = H(s)G(s)E^*(s)$$

$$Y^*(s) = \left( H(s)G(s) \right)^* E^*(s)$$

Usually

$$\left( H(s)G(s) \right)^* \neq H^*(s)G^*(s)$$

## Example 2

$$G(s) = \frac{1}{s}, H(s) = \frac{1}{s}$$

$$G(z) = \frac{z}{z-1}, H(z) = \frac{z}{z-1}$$

$$G(z)H(z) = \frac{z^2}{(z-1)^2}$$

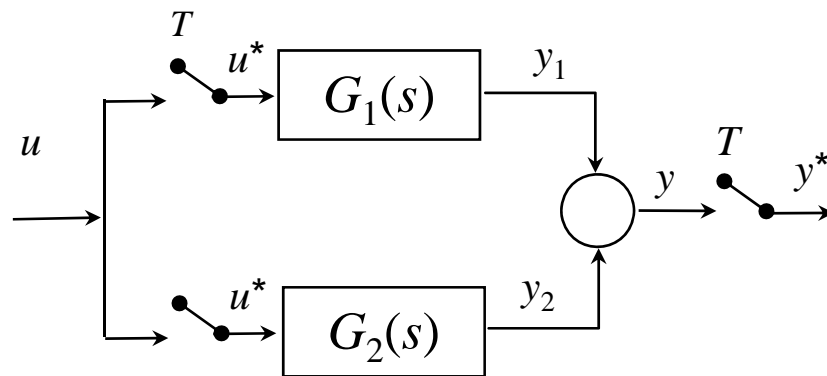
However,

$$G(s)H(s) = \frac{1}{s^2}$$

$$\mathcal{Z} \{GH\} = \frac{Tz}{(z-1)^2}$$

Conclusion: Overall transfer function of a system is dependent on position of samplers or ZOH.

## Example 3



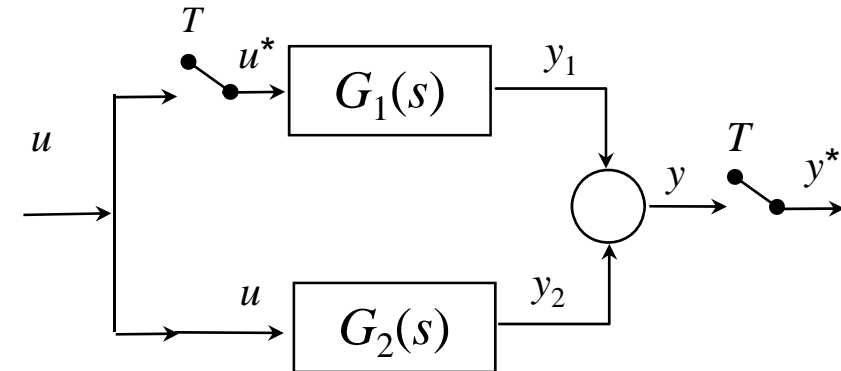
$$Y(s) = Y_2(s) + Y_1(s)$$

$$= G_2(s)U^*(s) + G_1(s)U^*(s)$$

$$Y^*(s) = G_2^*(s)U^*(s) + G_1(s)^*U^*(s)$$

$$Y(z) = G_2(z)U(z) + G_1(z)U(z)$$

$$\frac{Y(z)}{U(z)} = G_2(z) + G_1(z)$$



$$Y(s) = Y_2(s) + Y_1(s)$$

$$= G_2(s)U(s) + G_1(s)U^*(s)$$

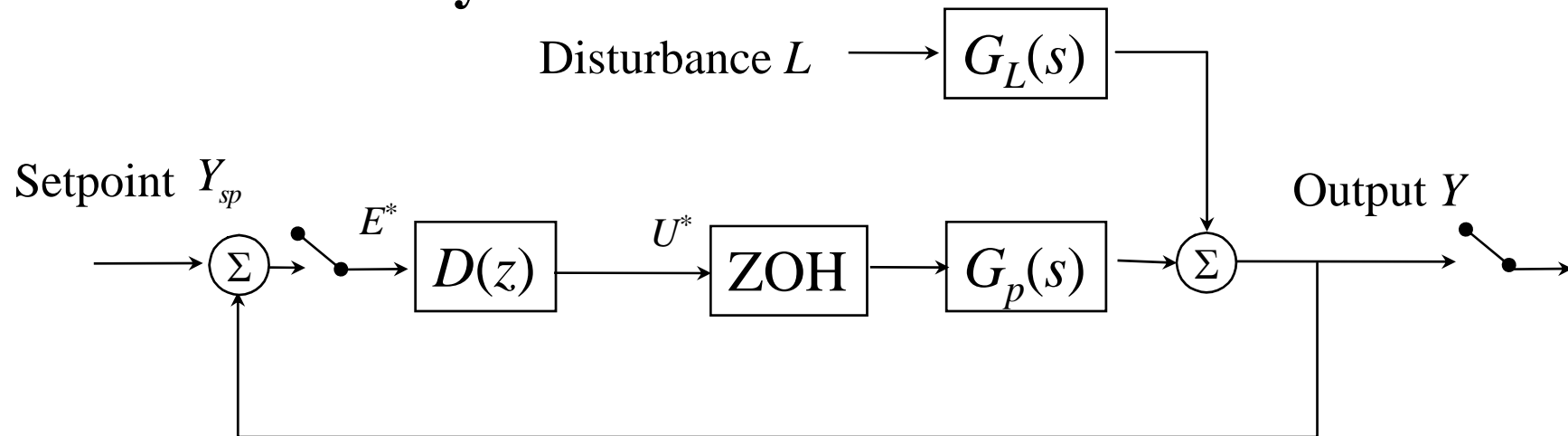
$$Y^*(s) = (G_2(s)U(s))^* + G_1(s)^*U^*(s)$$

$$Y(z) = Z(G_2(s)U(s)) + G_1(z)U(z)$$

transfer function not  
separable!



## 5.3 Control Systems



$$Y(s) = \frac{1 - e^{-Ts}}{s} G_p(s) U^*(s) + G_L(s) L(s)$$

$$U^*(s) = D^*(s) E^*(s)$$

$$E^*(s) = Y_{sp}^*(s) - Y^*(s)$$

# The Servo Problem

Consider set-point with no load disturbance ( $L(s) = 0$ )

$$Y(s) = \frac{1 - e^{-Ts}}{s} G_p(s) U^*(s), \quad U^*(s) = D^*(s) E^*(s), \quad E^*(s) = Y_{sp}^*(s) - Y^*(s)$$

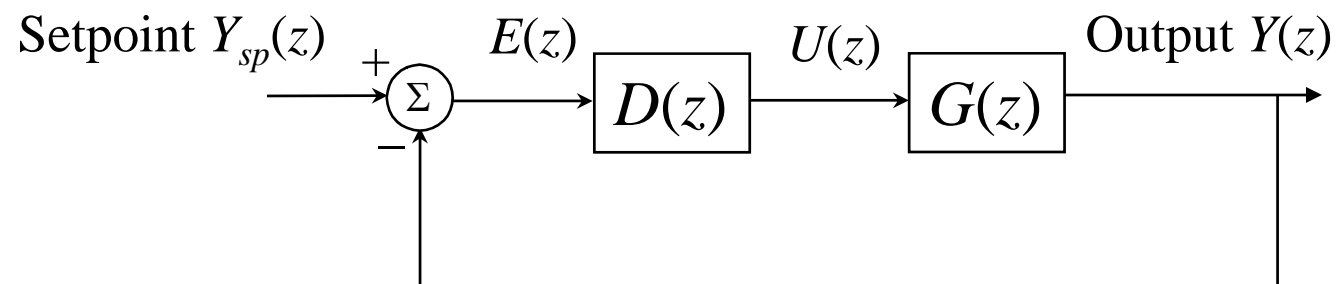
$$Y^*(s) = \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s) E^*(s) = \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s) [Y_{sp}^*(s) - Y^*(s)]$$

$$\frac{Y^*(s)}{Y_{sp}^*(s)} = \frac{\left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s)}{1 + \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s)}$$

$$\frac{Y(z)}{Y_{sp}(z)} = \frac{Z \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right] D(z)}{1 + Z \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right] D(z)}$$

$$H(z) = \frac{G(z)D(z)}{1 + G(z)D(z)}$$

## Block diagram for servo problem



block diagram manipulation exactly  
as for continuous time systems

For example, let the plant

$$G_p(s) = \frac{a}{s + a}.$$

The plant plus ZOH has the transfer function

$$\begin{aligned} G(z) &= (1 - z^{-1}) Z \left\{ \frac{G(s)}{s} \right\} \\ &= (1 - z^{-1}) Z \left\{ \frac{1}{s} - \frac{1}{s + a} \right\} \\ &= (1 - z^{-1}) \left[ \frac{1}{1 - z^{-1}} - \frac{1}{1 - e^{-aT} z^{-1}} \right] \end{aligned}$$

If  $e^{-aT} = \frac{1}{2}$ , this reduces to

$$G(z) = \frac{0.5}{z - 0.5}$$

Let the controller be a discrete integrator:

$$u(kT) = u(kT - T) + K_0 e(kT)$$

Then  $z$ -transfer function of controller is

$$D(z) = \frac{U(z)}{E(z)} = K_0 / (1 - z^{-1}) = \frac{K_0 z}{z - 1}$$

Thus, we have

$$H(z) = \frac{G(z)D(z)}{1 + G(z)D(z)} = \frac{\frac{0.5}{z - 0.5} \frac{K_0 z}{z - 1}}{1 + \frac{0.5}{z - 0.5} \frac{K_0 z}{z - 1}} = \frac{0.5 K_0 z}{z^2 + (0.5 K_0 - 1.5)z + 0.5}.$$

# The Regulator Problem

Consider load disturbance with no set-point ( $Y_{sp}=0$ )

$$Y(s) = \frac{1 - e^{-Ts}}{s} G_p(s) U^*(s) + G_L(s) L(s)$$

$$U^*(s) = D^*(s) E^*(s) = -D^*(s) Y^*(s)$$

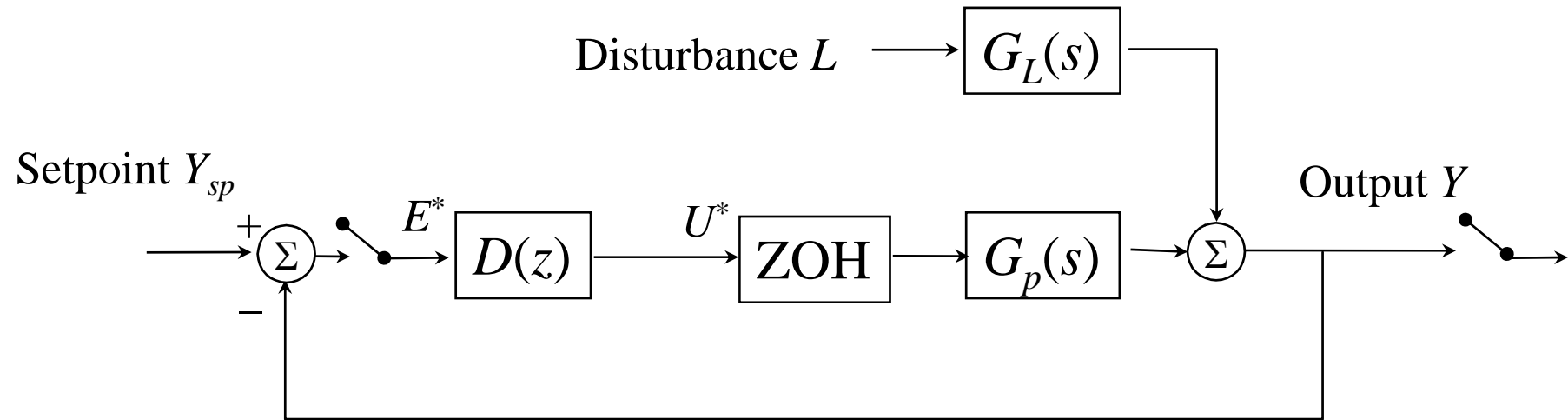
$$Y^*(s) = - \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s) Y^*(s) + [G_L(s) L(s)]^*$$

$$Y^*(s) = \frac{[G_L(s) L(s)]^*}{1 + \left[ \frac{1 - e^{-Ts}}{s} G_p(s) \right]^* D^*(s)}$$

$$Y(z) = \frac{Z \{ G_L(s) L(s) \}}{1 + G(z) D(z)}$$

We cannot separate a transfer function from  $L(z)$  to  $Y(z)$ !

## Stability of the Closed Loop



Closed-loop is said to be stable if the output sequence,  $y(kT)$ , is bounded for any bounded input sequences,  $y_{sp}(kT)$  and  $l(kT)$ .

## Conditions for Stability

- We have shown that

$$\begin{aligned} Y(z) &= \frac{G(z)D(z)}{1 + G(z)D(z)} Y_{sp}(z) + \frac{Z \{G_L(s)L(s)\}}{1 + G(z)D(z)} \\ &= Y_1(z) + Y_2(z) \end{aligned}$$

$$1 + G(z)D(z) = 0$$

CL characteristic equation should have no roots on or outside the unit circle in the  $z$  plane for stability

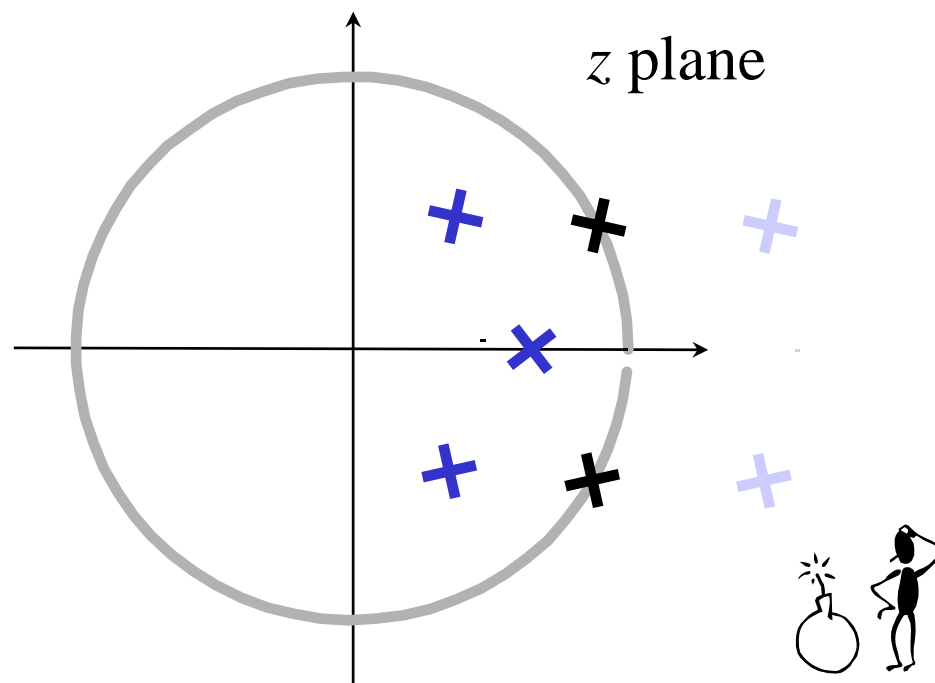


## Test for Closed-loop Stability

- Find all the roots of the CL characteristic equation

$$1 + G(z)D(z) = 0$$

- Check if all the roots lie within the unit circle, i.e., have magnitudes less than one
- Yes: stable; otherwise, unstable



An early example has CL transfer function

$$H(z) = \frac{0.5K_0 z}{z^2 + (0.5K_0 - 1.5)z + 0.5}.$$

Its CL characteristic polynomial is

$$z^2 + (0.5K_0 - 1.5)z + 0.5$$

Let  $K_0 = 1$ . There are two roots at

$$0.5 \pm 0.5i \quad \text{stable}$$

Let  $K_0 = -1$ . Then, two roots are at

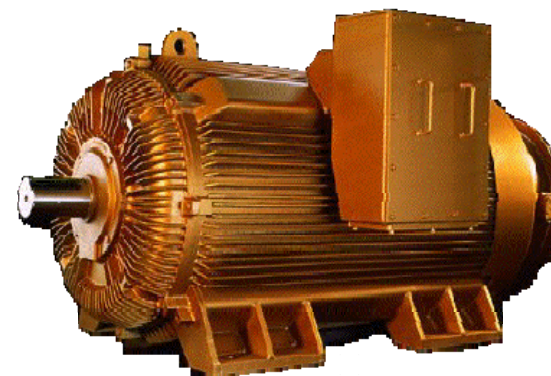
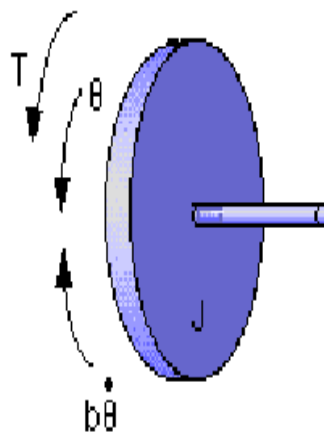
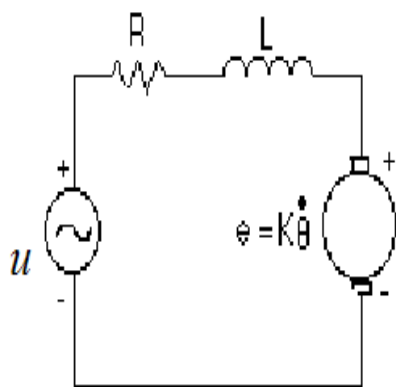
$$1 \pm \frac{\sqrt{2}}{2}i \quad \text{unstable}$$

# Design Approaches to Digital Systems

- **1<sup>st</sup> approach** Based on continuous time plant model,  $G(s)$ , obtain the controller  $D(s)$ . Then discretize  $D(s)$  to  $D(z)$  for implementation;
- **2<sup>nd</sup> approach** Discretize plant,  $G(s)$  to  $G(z)$ . Then design the controller  $D(z)$  in discrete time. Proceed to implementation.

Whichever approach is used, discretization is involved. Are there other ways of discretization that are easier or less tedious mathematically? Is exact  $z$ -transform necessary all the time?

# DC Motor Speed Control



## 1. The plant model

The model of the DC motor is

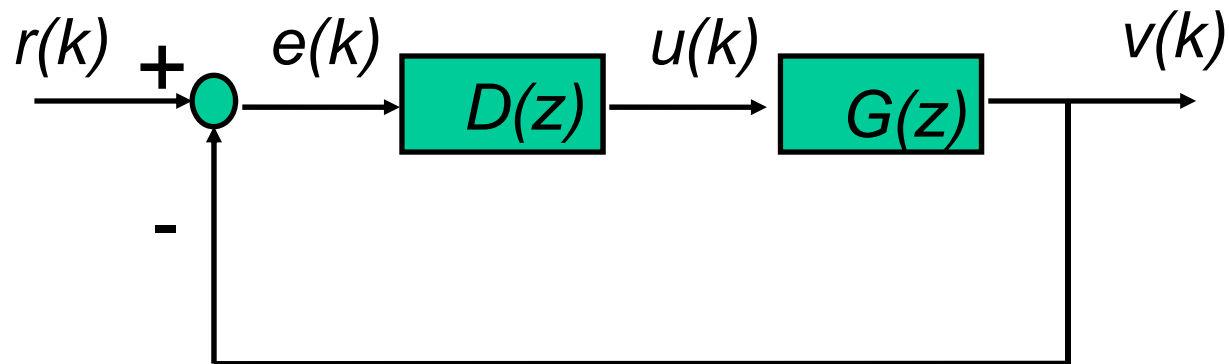
$$\frac{v(s)}{u(s)} = \frac{K}{(Js + b)(Ls + R) + K^2},$$

- \*electrical resistance ( $R$ ) = 1 ohm,
- \*electrical inductance ( $L$ ) = 0.5 H,
- \*electromotive force constant ( $K_e=K_t$ ) = 0.01 Nm/Amp,
- \*moment of inertia of the rotor ( $J$ ) = 0.01 kg\*m<sup>2</sup>/s<sup>2</sup>,
- \*damping ratio of the mechanical system ( $b$ ) = 0.1 Nms,
- \*input ( $u$ ): Source Voltage,
- \*output ( $v=\dot{\theta}$ ): Rotating speed,
- \*The rotor and shaft are assumed to be rigid.

## 2. Control specifications

The design requirements for 1 rad/sec step input are:

- Settling time: Less than 5 seconds ,
- Overshoot: Less than 5% ,
- Steady-state error: Less than 1%.



### 3. ZOH discrete model

Substituting the given parameters, we have

$$G(s) = \frac{v(s)}{u(s)} = \frac{0.01}{0.005s^2 + 0.06s + 0.1001}.$$

Choose the sampling time as 0.12s. The plant plus zero-order holder has the discrete transfer function:

$$G(z) = Z \left\{ \frac{1 - e^{-Ts}}{s} G(s) \right\} \bigg|_{T=0.12} = \frac{0.0092z + 0.0057}{z^2 - 1.0877z + 0.2369}.$$

## 4. Controller design

Recall that continuous system open loop needs an integrator (pole at origin) for zero steady state error for step response. The discrete case needs a pole at  $z=1$ . Note that the open loop  $G(z)$  has poles at  $0.7865$  and  $0.3012$  (stable poles). Using the stable pole-zero cancellation, we choose the controller as

$$D(z) = \frac{k(z^2 - 1.0877z + 0.2369)}{z - 1}.$$

Then the characteristic equation for the closed loop is

$$1 + \frac{k(0.0092z + 0.0057)}{z - 1} = 0.$$

For a continuous first order system with pole at  $a$ , settling time is given by

$$t_{\text{settling}} = -\frac{4}{a}.$$

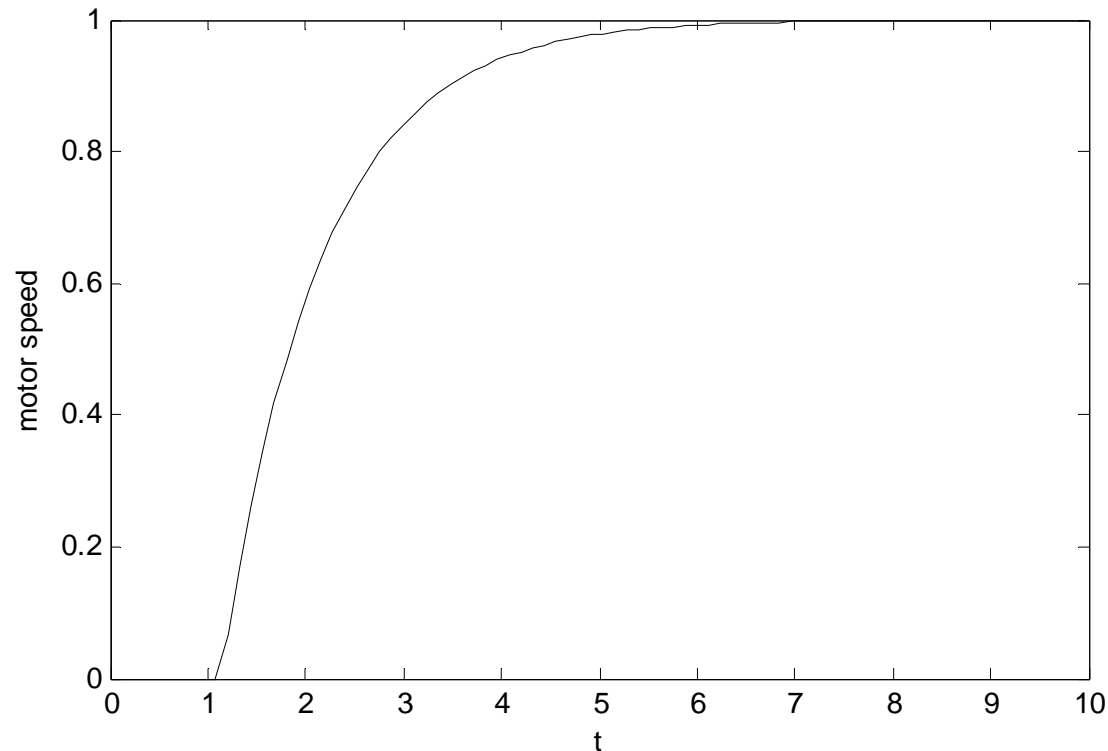
According to the specification, settling time should be less than 5 s.

This requires the pole to be located  $s < -0.8$ . Via  $z = e^{sT} = e^{(-0.8 \times 0.12)}$ , the



discrete pole should be located at  $z < 0.908$ . Set the pole at  $z = 0.9$ . Then from the characteristic equation, we have  $k = 7.153$ .

## 5. Closed-loop performance



The step response of the control system is given in the above figure. The plot shows that the settling time is less than 5 seconds and the percent overshoot is 0. In addition, the steady state error is zero. Therefore this response satisfies all of the design requirements.