

MA1506 Mathematics II

Chapter 3 Basic Mathematical Modelling

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3.1 Introduction

Mathematical model uses mathematical language to describe a system.

In this module, we use ODE to describe some systems.

In the last two chapters, we have studied models for falling objects, cooling, and mass-spring systems.

In this chapter, we will introduce some more models.

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3.1 Introduction

- Malthus model of population
 - Logistic growth model
 - Harvesting model
 - Plug flow reactor model
 - Cantilevered beam model
- related
1st order ODE
1st order ODE
4th order ODE

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3.1 Introduction

- The process of developing a mathematical model is called mathematical modelling.
- In constructing math. models, there is almost always a tradeoff between accuracy and simplicity.
- We should use good judgment and common sense in constructing models and in making predictions.
- Begin with simple models, understand their weakness, and then improve them with more complicated and appropriate ones.

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3.2 Malthus Model of Population

Total Population: $N(t)$

Per Capita Birth-Rate, B

babies born in $\delta t = BN \delta t$

Per Capita Death-Rate, D

deaths in $\delta t = DN \delta t$

$\delta N = \# \text{ births} - \# \text{ deaths} = (B-D)N\delta t$

$$\frac{\delta N}{\delta t} = (B-D)N$$



Thomas Malthus
1766-1834

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3.2 Malthus Model of Population

$$\frac{dN}{dt} = (B-D)N = kN$$

$$k = B - D$$

$$\int \frac{dN}{N} = \int k dt = kt + c$$

$$\ln N(t) = kt + c$$

$$N(t) = e^{kt+c} = e^c e^{kt} = N_0 e^{kt}$$

$$e^c = N_0 = N(0)$$

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$k > 0$: population explosion

as $t \rightarrow \infty$ $e^{kt} \rightarrow \infty \Rightarrow N(t) \rightarrow \infty$

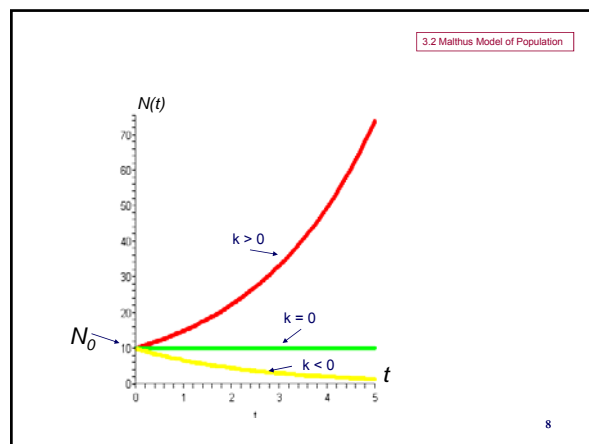
$k = 0$: stable equilibrium population

because $N(t) = N(0)$ for all t .

$k < 0$: population will extinct eventually

as $t \rightarrow \infty$ $e^{kt} \rightarrow 0 \Rightarrow N(t) \rightarrow 0$

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3.3 Logistic growth model (improved Malthus model)

In the logistic growth model, we impose one more condition that

death rate is a function of population

on Malthus model

We assume $D = sN$

which is called the logistic assumption.

The assumption says that, in a world with finite resources, large population will eventually cause starvation and disease.

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Since $D = sN$

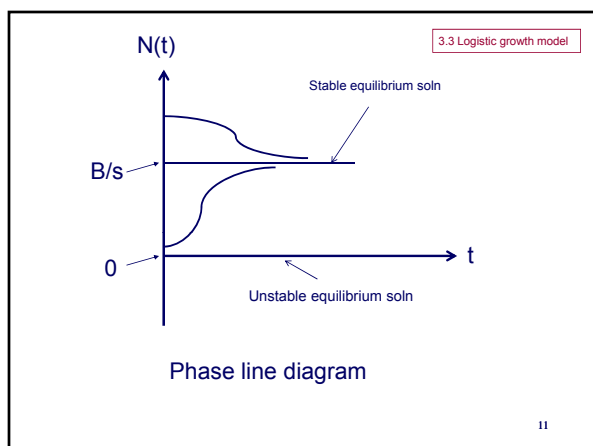
The logistic equation is

$$\begin{aligned} \frac{dN}{dt} &= BN - DN = BN - sN^2 \\ &= N(B - sN) \end{aligned}$$

Notice that the ODE has two equilibrium solutions, namely, $N(t)=0$ and $N(t)=B/s$ for all t .

Note also that $N(t)=0$ is unstable while $N(t)=B/s$ is stable.

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As our logistic equation is simple, we can solve it by separable method.

We may assume that $N(t) \neq 0$ and $N(t) \neq B/s$

Then

$$\begin{aligned} \frac{dN}{dt} &= BN - sN^2 = N(B - sN) \\ dt &= \frac{dN}{N(B - sN)} \end{aligned}$$

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By partial fraction we have

3.3 Logistic growth model

$$\begin{aligned}\frac{1}{N(B-sN)} &= \frac{1}{BN} + \frac{s}{B(B-sN)} \\ t = \int \frac{dN}{N(B-sN)} &= \frac{1}{B} \int \frac{dN}{N} + \frac{s}{B} \int \frac{dN}{B-sN} \\ &= \frac{1}{B} \ln N - \frac{1}{B} \ln(B-sN) + c \\ &= \frac{1}{B} \ln \left(\frac{N}{B-sN} \right) + c\end{aligned}$$

Hence the solution is $\frac{N(t)}{B-sN(t)} = e^{-Bc+Bt}$

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To find the value of c, we need to know $N(0)$, i.e., the value of $N(t)$ when $t=0$.

When $t=0$, we have

$$\frac{N(0)}{B-sN(0)} = e^{-Bc}$$

Thus our solution becomes

$$\frac{N(t)}{B-sN(t)} = \frac{N(0)}{B-sN(0)} e^{Bt}$$

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Definition

3.3 Logistic growth model

Recall that B/s is the stable equilibrium solution of

$$\frac{dN}{dt} = BN - sN^2 = N(B-sN)$$

B/s is also called the carrying capacity or sustainable population.

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3.5 Harvesting

Assume the fish population $N(t)$ follows the logistic growth model

$$\frac{dN}{dt} = BN - sN^2$$

Assume further that we catch E (constant) fish per year. Then our differential equation becomes

$$\frac{dN}{dt} = BN - sN^2 - E$$

Basic Harvesting Model

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Unlike the logistic model, we are unable to solve the harvesting model by separable method.

Instead, we shall study the solutions to harvesting model using equilibrium solutions and phase lines.

To find the equilibrium solutions of

$$\frac{dN}{dt} = BN - sN^2 - E$$

we let $f(N) = -sN^2 + BN - E$

and find the roots of $f(N) = 0$

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The discriminant for $f(N) = 0$

3.5 Harvesting

$$\text{is } B^2 - 4(-s)(-E) = B^2 - 4sE$$

and we have to consider the following three cases

Case 1 $B^2 - 4sE > 0$ two real roots

Case 2 $B^2 - 4sE = 0$ one real root

Case 3 $B^2 - 4sE < 0$ no real roots

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Case 1 $B^2 - 4sE > 0$ i.e., $E < \frac{B^2}{4s}$

3.5 Harvesting

Let β_1, β_2 be the two roots of $f(N) = 0$ given by

$$\beta_1 = \frac{-B + \sqrt{B^2 - 4(-s)(-E)}}{2(-s)}$$

$$\beta_2 = \frac{-B - \sqrt{B^2 - 4(-s)(-E)}}{2(-s)}$$

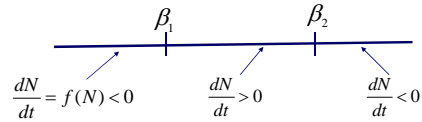
Note that $\beta_1 < \beta_2$

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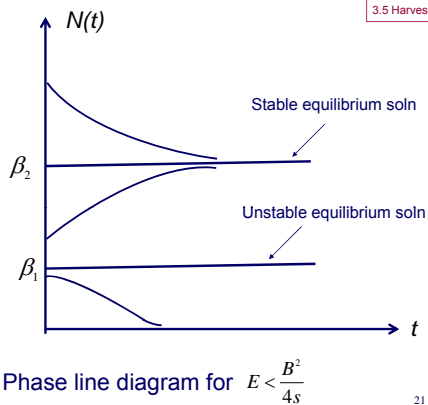
$$f(N) = -sN^2 + BN - E$$

3.5 Harvesting

$$\frac{dN}{dt} = f(N) = (-s)(N - \beta_1)(N - \beta_2)$$



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Case 2 $E = \frac{B^2}{4s}$

3.5 Harvesting

$$f(N) = -sN^2 + BN - E = 0$$

has one repeated real root given by

$$\beta = \frac{-B \pm \sqrt{B^2 - 4(-s)(-E)}}{2(-s)} = \frac{B}{2s}$$

Thus $N(t) = \frac{B}{2s}$ is an equilibrium solution.

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Since $E = \frac{B^2}{4s}$

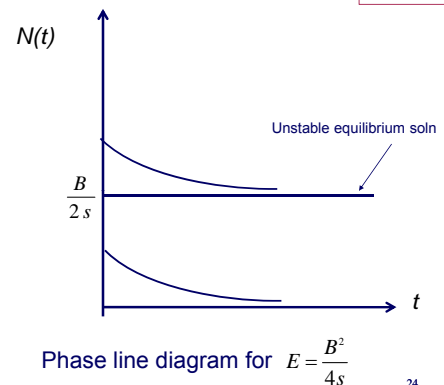
3.5 Harvesting

we have $\frac{dN}{dt} = f(N) = (-s)(N - \beta)^2 < 0$

because $s > 0$ which implies that $-s$ is negative.

Thus the fish population $N(t)$ is decreasing and all the fish will be wiped out.

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Case 3 Overfishing $B^2 - 4sE < 0$

3.5 Harvesting

Since $B^2 - 4sE < 0$

$$f(N) = -sN^2 + BN - E = 0$$

has no real roots.

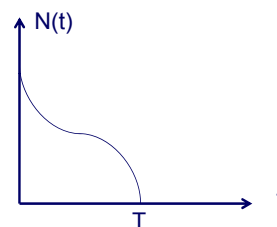
Since $s > 0$ which implies that $-s$ is negative, we have $f(N) < 0$

Thus
$$\frac{dN}{dt} = f(N) < 0$$

and fish population $N(t)$ is decreasing.

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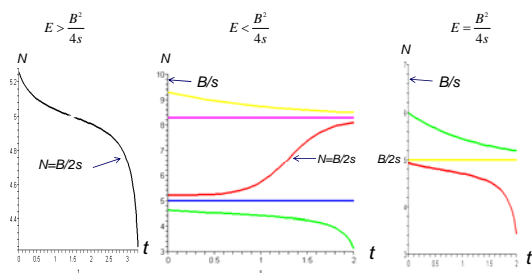
Fish population $N(t)$ will be wiped out (i.e., $N(t)=0$) at a certain time T .



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Summary: Solution curves for the three cases

3.5 Harvesting



Recall B/s = sustainable population in logistic model

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No crossing principle

We notice that the above curves do not cross each other. This is because there is one and only one solution for each initial value problem for each 1st order ODE.

Suppose we have two curves crossing each other, then we can use the intersection point as an initial point and get two curves starting from this point. This means there are two solutions with the same initial point, a contradiction.

This observation is usually called the no crossing principle.

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Remark

3.5 Harvesting

If there is no fishing for a long time, most likely the fish population is B/s .

Thus we may assume that the initial number of fish is B/s , i.e., $N(0) = B/s$.

If we start catching E fish per year, then the harvesting differentiation equation is

$$\frac{dN}{dt} = -sN^2 + BN - E$$

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Our earlier discussions show that we will be overfishing if

$$E > \frac{B^2}{4s}$$

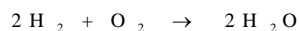
Otherwise, fish population will eventually tend to a fixed number β_2

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3.6 Plug flow reactor (PFR) model

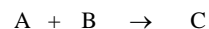
3.6.1 Chemical reactions

A chemical reaction consists of one or more starting material called reactants. They react to form new substances called products. For example,



says 2 molecules of hydrogen and 1 molecule of oxygen react to form 2 molecules of water.

Now consider the reaction



The concentration of reactant A is the number of moles per litre (1 mole = 6.022×10^{23} molecules).

3.6.2 Plug flow reactor (PFR) model

The **plug flow reactor (PFR)** model is used to describe chemical reactions in continuous, flowing systems.

PFR is like a long tube into which you push some mixture of chemicals which move through the tube while they react with each other.



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PFR model

Assume

- Velocity u of flow is constant
- Cross-sectional area A is constant
- Temp constant
- Flow is plug flow: see next slide

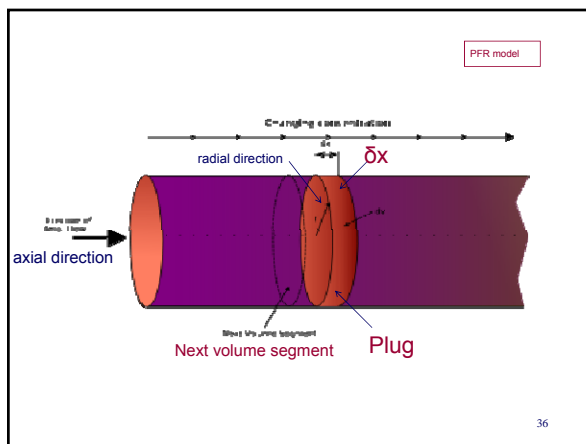
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• Plug Flow:

Fluid flowing through the reactor as a series of infinitely thin "plugs", each with a uniform composition, traveling in the axial direction of the reactor, with each plug having a different composition from the ones before and after it.

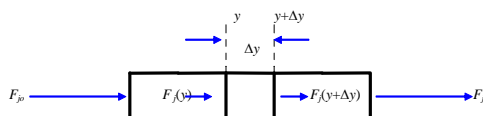
The key assumption is that the fluid is perfectly mixed in the radial direction but not in the axial direction (forwards or backwards). Each plug of small volume is considered as an independent, separate entity,

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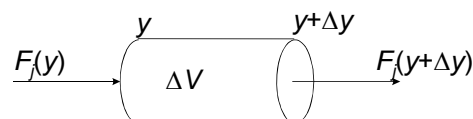
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Plug flow reactor



• Material balance $F_{j0} - F_j + \int_V r_j dV = \frac{dN_j}{dt}$

- Let us inspect a volume ΔV , where the reaction rate can be assumed constant.



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Reaction rate is constant in ΔV

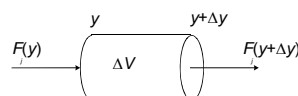
$$\int_V r_j dV = r_j \Delta V$$

Steady state assumption, i.e. no accumulation

$$\frac{dN_j}{dt} = 0$$

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For the volume ΔV we can write the material balance for component j .



$$F_j(y) - F_j(y + \Delta y) = -r_j \Delta V$$

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A = cross-sectional area

$$\Delta V = A \Delta y$$

$$F_j(y) - F_j(y + \Delta y) = -r_j A \Delta y$$

$$F_j(y) - F_j(y + \Delta y) = -r_j \Delta V$$

$$\Delta V \rightarrow 0$$

$$-\frac{dF_j}{dV} = -r_j \Leftrightarrow \frac{dF_j}{dV} = r_j$$

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Separation of variables

$$dV = F_{j0} \frac{dX}{-r_j}$$

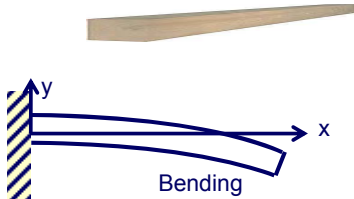
$$\int_0^V dV = F_{j0} \int_0^X \frac{dX}{-r_j}$$

$$V = F_{j0} \int_0^X \frac{dX}{-r_j}$$

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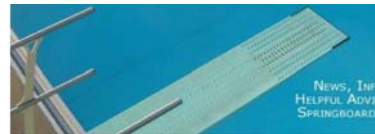
3.7 Cantilevered Beams

A cantilevered beam is a beam supported at only one end and whose length is much larger than its width and thickness.



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3.7 Cantilevered Beams



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3.7 Cantilevered Beams

In this section, we will discuss the bending of a cantilevered beam due to its weight and other forces acting on the beam called the load.

We denote the load by

$W(x)$ =force **per unit length at point x**

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3.7 Cantilevered Beams

Load=weight of
springboard



Load=weight of bamboo
stick+clothes



Load=weight of
springboard +
swimmers

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3.7 Cantilevered Beams

Recall that 2nd derivative $\frac{d^2y}{dx^2}$ gives us information about the way the graph bends.

In fact, a graph is concave up if $\frac{d^2y}{dx^2} > 0$

and concave down if $\frac{d^2y}{dx^2} < 0$.

$$\frac{d^2y}{dx^2} > 0$$

$$\frac{d^2y}{dx^2} < 0$$

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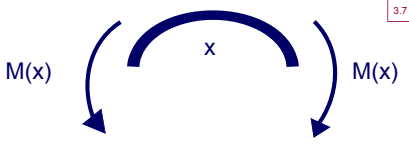
3.7 Cantilevered Beams

When the beam is supported at one end, internal forces, called the shear forces, and bending moments $M(x)$ are developed.

First we consider the bending moment $M(x)$ at x , here x is fixed.

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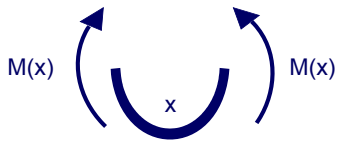
3.7 Cantilevered Beams



The beam at position x is concave down.
Thus the 2nd derivative is negative and the sign of $M(x)$ is taken to be negative.

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3.7 Cantilevered Beams



The beam at x is concave up.
Thus 2nd derivative is positive and the sign of $M(x)$ is taken to be positive.

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3.7 Cantilevered Beams

The value (always positive) of $\frac{M(x)}{\frac{d^2 y}{dx^2}}$ depends on stiffness.
From physics we have

$$\frac{M(x)}{\frac{d^2 y}{dx^2}} = EI$$

Where E is a constant called the Young's modulus and I is a constant called the moment of inertia.


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3.7 Cantilevered Beams

Next we shall prove that

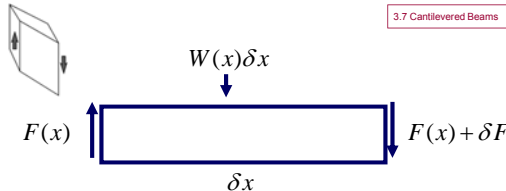
$$\frac{d^2 M}{dx^2} = -W(x)$$

To prove this formula, we need to consider shear force $F(x)$, which is parallel to the cross section and causes the internal structure of a material to slide against itself



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3.7 Cantilevered Beams

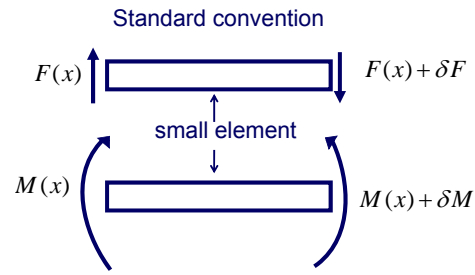


$W(x) = \text{Load} = \text{force per unit length}$
 $F(x) = F(x) + \delta F + W(x)\delta x$
 $\frac{dF}{dx} = -W(x)$

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3.7 Cantilevered Beams

Standard convention



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Moment = force x distance

3.7 Cantilevered Beams

Take moment about centre O, we get

$$M(x) + F(x) \frac{\delta x}{2} + (F(x) + \delta F) \frac{\delta x}{2} = M(x) + \delta M$$

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3.7 Cantilevered Beams

$$M(x) + F(x) \frac{\delta x}{2} + (F(x) + \delta F) \frac{\delta x}{2} = M(x) + \delta M$$

$$F(x) \delta x + (\delta F)(\delta x) / 2 = \delta M$$

$F(x) \delta x = \delta M$ Too small, ignore

$$\frac{dM}{dx} = F(x)$$

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Since

$$\frac{M(x)}{\frac{d^2 y}{dx^2}} = EI \quad \frac{dM}{dx} = F(x) \quad \frac{dF}{dx} = -W(x)$$

we have

$$M(x) = \frac{d^2 y}{dx^2} EI$$

$$\frac{d^2 M}{dx^2} = \frac{dF}{dx} = -W(x)$$

$$\frac{d^4 y}{dx^4} = \frac{1}{EI} \frac{d^2 M}{dx^2} = -\frac{W(x)}{EI}$$

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3.7 Cantilevered Beams

$$\frac{d^4 y}{dx^4} = -\frac{W(x)}{EI}$$

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Maximum deflection Δ

3.7 Cantilevered Beams

$$\frac{d^4 y}{dx^4} = -\frac{W(x)}{EI}$$

Assume: uniform mass $W(x) = \alpha$

Note that $\Delta = y(L)$

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3.7 Cantilevered Beams

To find Δ i.e., to find $y(L)$

we need to solve

$$\frac{d^4 y}{dx^4} = -\frac{W(x)}{EI} = -\frac{\alpha}{EI}$$

This is a 4th order ODE and we need FOUR initial conditions which are

$$y(0) = 0 \quad \left. \frac{dy}{dx} \right|_{x=0} = 0$$

$$M(L) = 0 \quad F(L) = 0$$

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3.7 Cantilevered Beams

$\left. \frac{dy}{dx} \right|_{x=0} = 0$
 $y(0) = 0$
 L
 $M(L) = 0$ $F(L) = 0$
 No moment at L No shear force at L

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3.7 Cantilevered Beams

Recall $\frac{d^2 y}{dx^2} = \frac{M(x)}{EI}$

So $\frac{d^3 y}{dx^3} = \frac{1}{EI} \frac{dM}{dx} = \frac{F(x)}{EI}$

Hence $\left. \frac{d^2 y}{dx^2} \right|_{x=L} = \frac{M(L)}{EI} = 0$

$\left. \frac{d^3 y}{dx^3} \right|_{x=L} = \frac{F(L)}{EI} = 0$

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3.7 Cantilevered Beams

Thus we can rewrite the initial conditions as

$$y(0) = 0 \quad \left. \frac{dy}{dx} \right|_{x=0} = 0$$

$$\left. \frac{d^2 y}{dx^2} \right|_{x=L} = 0 \quad \left. \frac{d^3 y}{dx^3} \right|_{x=L} = 0$$

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3.7 Cantilevered Beams

$$\frac{d^4 y}{dx^4} = \frac{-\alpha}{EI} \quad \text{implies} \quad \frac{d^3 y}{dx^3} = \frac{-\alpha x}{EI} + A$$

Since $\left. \frac{d^3 y}{dx^3} \right|_{x=L} = 0$

we get $A = \frac{\alpha L}{EI}$

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3.7 Cantilevered Beams

$$\frac{d^3 y}{dx^3} = \frac{-\alpha x}{EI} + \frac{\alpha L}{EI} \quad \text{implies}$$

$$\frac{d^2 y}{dx^2} = \frac{-\alpha x^2}{2EI} + \frac{\alpha Lx}{EI} + B$$

Since $\left. \frac{d^2 y}{dx^2} \right|_{x=L} = 0$

we get $B = \frac{\alpha L^2}{2EI} - \frac{\alpha L^2}{EI} = -\frac{\alpha L^2}{2EI}$

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3.7 Cantilevered Beams

$$\frac{d^2 y}{dx^2} = \frac{-\alpha x^2}{2EI} + \frac{\alpha Lx}{EI} - \frac{\alpha L^2}{2EI}$$

implies $\frac{dy}{dx} = \frac{-\alpha x^3}{6EI} + \frac{\alpha Lx^2}{2EI} - \frac{\alpha L^2 x}{2EI} + C$

Since $\left. \frac{dy}{dx} \right|_{x=0} = 0$

we get $C = 0$

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3.7 Cantilevered Beams

$$\frac{dy}{dx} = -\frac{\alpha x^3}{6EI} + \frac{\alpha Lx^2}{2EI} - \frac{\alpha L^2x}{2EI}$$

implies

$$y = -\frac{\alpha x^4}{24EI} + \frac{\alpha Lx^3}{6EI} - \frac{\alpha L^2x^2}{4EI} + D$$

Since

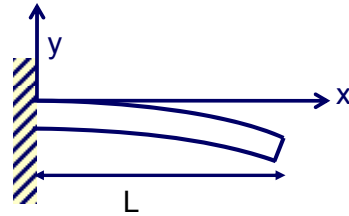
$$y(0) = 0$$

we get

$$D = 0$$

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3.7 Cantilevered Beams

Beam equation ($W(x) = \alpha$)

$$y(x) = \frac{\alpha L^4}{2EI} \left(-\frac{1}{12} \left(\frac{x}{L} \right)^4 + \frac{1}{3} \left(\frac{x}{L} \right)^3 - \frac{1}{2} \left(\frac{x}{L} \right)^2 \right)$$

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3.7 Cantilevered Beams

For $W(x) = \alpha$

$$y(x) = \frac{\alpha L^4}{2EI} \left(-\frac{1}{12} \left(\frac{x}{L} \right)^4 + \frac{1}{3} \left(\frac{x}{L} \right)^3 - \frac{1}{2} \left(\frac{x}{L} \right)^2 \right)$$

$$\Delta = y(L) = \frac{\alpha L^4}{2EI} \left(-\frac{1}{12} + \frac{1}{3} - \frac{1}{2} \right) = -\frac{\alpha L^4}{8EI}$$

Cantilever beam deflection formula

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3.7 Cantilevered Beams

Remarks on cantilever beam of length L

Differential equation

$$\frac{d^4 y}{dx^4} = \frac{-W(x)}{EI}$$

$$\frac{d^2 y}{dx^2} = \frac{M(x)}{EI}$$

Initial conditions

$$y(0) = 0$$

$$\left. \frac{dy}{dx} \right|_{x=0} = 0$$

$$\left. \frac{d^2 y}{dx^2} \right|_{x=L} = 0$$

$$\left. \frac{d^3 y}{dx^3} \right|_{x=L} = 0$$

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