Online Lecture Notes

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1 Multivariate Linear Input-Output System

In this lecture we consider systems of the form

$$\dot{x}(t) = Ax(t) + Bu(t) + b \tag{1}$$

$$y(t) = Cx(t) + d (2)$$

with input (control) u(t) and output (sensor data) y(t).

1.1 Steady-State Relations

At the steady-state the equation

$$0 = Ax_{\text{ref}} + Bu_{\text{ref}} + b \tag{3}$$

$$y_{\text{ref}} = Cx_{\text{ref}} + d \tag{4}$$

If A is invertible, we have

$$x_{\rm ref} = -A^{-1}(Bu_{\rm ref} + b),$$

which can be substituted into the other equation in order to find

$$y_{\text{ref}} = Cx_{\text{ref}} + d = -CA^{-1}(Bu_{\text{ref}} + b) + d = -[CA^{-1}B]u_{\text{ref}} + [d - CA^{-1}b]$$

In general, we have $n_y > n_u$, which means that we might not be able to bring the system to any output. However, for the special case that $n_y = n_u$ and $CA^{-1}B$ invertible, we can, at least in principle, bring the system to any given set point y_{ref} by choosing the appropriate u_{ref} , given by

$$u_{\text{ref}} = [CA^{-1}B]^{-1}[d - CA^{-1}b - y_{\text{ref}}].$$

1.2 Proportional Controllers

The main idea of proportional control is to introduce a feedback law of the form

$$\forall y \in \mathbb{R}^{n_y}, \qquad \mu(y) = u_{\text{ref}} + K(y - y_{\text{ref}}) .$$

Notice that in contrast to the actual control input function u the function μ is a function of the current output data y rather than of time t. This means that we feedback the control signal

$$u(t) = u_{\text{ref}} + K(y(t) - y_{\text{ref}})$$

This means that we have the following relations:

$$0 = Ax_{\text{ref}} + Bu_{\text{ref}} + b \tag{5}$$

$$y_{\text{ref}} = Cx_{\text{ref}} + d \tag{6}$$

$$\dot{x}(t) = Ax(t) + Bu(t) + b \tag{7}$$

$$y(t) = Cx(t) + d (8)$$

$$u(t) = u_{\text{ref}} + K(y(t) - y_{\text{ref}}) \tag{9}$$

Thus, we have in total one differential equation and 4 algebraic relation that we need to substitute if we want to work out the closed-loop trajectory. Let us substitute these equations step-by-step:

$$\dot{x}(t) \stackrel{(7)}{=} Ax(t) + Bu(t) + b \tag{10}$$

$$\stackrel{(9)}{=} Ax(t) + B \left[u_{\text{ref}} + K(y(t) - y_{\text{ref}}) \right] + b \tag{11}$$

$$\stackrel{(8)}{=} Ax(t) + B \left[u_{\text{ref}} + K(Cx(t) + d - y_{\text{ref}}) \right] + b \tag{12}$$

$$\stackrel{(6)}{=} Ax(t) + B\left[u_{\text{ref}} + KC(x(t) - x_{\text{ref}})\right] + b \tag{13}$$

$$\stackrel{(5)}{=} A(x(t) - x_{\text{ref}}) + BKC(x(t) - x_{\text{ref}}) \tag{14}$$

$$= (A + BKC)(x(t) - x_{ref}) \tag{15}$$

The matrix $A_{\rm cl} = A + BKC$ is called the closed-loop system gain. The explicit solution for the closed-loop trajectory is given by

$$x(t) = e^{A_{\rm cl}t}[x(0) - x_{\rm ref}] + x_{\rm ref}$$
.

1.3 Properties of the closed-loop trajectory in dependence on K

Since the closed gain matrix $A_{\rm cl} = A + BKC$ depends on K, we can analyze the closed-loop trajectory in dependence on K. In general, this is not entirely trivial, since the explicit solution depends on the matrix exponential

$$e^{A_{\rm cl}t} = e^{(A+BKC)t}$$
.

We know from Lecture 5 that this matrix exponential depends on the eigenvalues of the matrix A + BKC. In particular, we have

$$\lim_{t \to \infty} e^{(A+BKC)t} = 0$$

if all eigenvalues of A+BKC have strictly negative real part. This follows from the fact that the Jordan normal decomposition of A+BKC has the form

$$A + BKC = T(D+N)T^{-1}$$
 $\stackrel{\text{Lecture 5}}{\Longrightarrow}$ $e^{(A+BKC)t} = Te^{Dt}e^{Nt}T^{-1},$

where e^{Nt} is a polynomial in t, which is for $t \to \infty$ overpowered by the exponential diagonal function e^{Dt} as long as all the diagonal elements of D (= to the eigenvalues of A + BKC) have strictly negative real part. Thus, if we manage to

choose K in such a way that the eigenvalues of A+BKC have strictly negative real part, then we have

$$\lim_{t \to \infty} x(t) = \lim_{t \to \infty} e^{(A+BKC)t}(x(0) - x_{\text{ref}}) + x_{\text{ref}} = x_{\text{ref}};$$

that is, the closed trajectory converges to the reference point as desired.

1.4 Example

Let us consider the specific example

$$A = \begin{pmatrix} 1 & 1 \\ 1 & -2 \end{pmatrix} \quad B = \begin{pmatrix} 0 \\ 1 \end{pmatrix} \quad C = (1 \ 0) \ .$$

This corresponds to a system with 2 states, 1 control input, and 1 measurement. The corresponding closed-loop system matrix is given by

$$A_{\rm cl} = A + BKC = A = \begin{pmatrix} 1 & 1 \\ 1 & -2 \end{pmatrix} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} K \begin{pmatrix} 1 & 0 \end{pmatrix} = \begin{pmatrix} 1 & 1 \\ 1 + K & -2 \end{pmatrix} .$$

In order to analyze under which conditions the scalar proportional control gain $K \in \mathbb{R}^{1 \times 1}$ stabilizes the system, we need to work out the eigenvalues of the matrix $A_{\rm cl}$. For this aim, we first work out the roots of the characteristic polynomial

$$0 = \det(A_{\rm cl} - \lambda I) = \det\left(\begin{pmatrix} 1 - \lambda & 1\\ 1 + K & -2 - \lambda \end{pmatrix}\right)$$
 (16)

$$= (1 - \lambda)(-2 - \lambda) - (1 + K) \tag{17}$$

$$= \lambda^2 + \lambda - K - 3 \tag{18}$$

Thus, the eigenvalues are given by

$$\lambda_{1,2} = -\frac{1}{2} \pm \sqrt{\frac{1}{4} + 3 + K} \; .$$

Clearly, we have $\operatorname{Re}(\lambda_1) \geq \operatorname{Re}(\lambda_2)$. Thus, if we want to achieve that both eigenvalues have strictly negative real part, it is sufficient to ensure that $\operatorname{Re}(\lambda_1) < 0$. There are three cases

- Case 1: $K \ge -3$. In this case, we have $Re(\lambda_1) = \lambda_1 \ge 0$, which means that the closed-loop system matrix is not asymptotically stabilizing.
- Case 2: $-3 \frac{1}{4} \le K < -3$. In this case both eigenvalues are real and we have

$$\lambda_2 \le -\frac{1}{2} \le \lambda_1 < 0 \ .$$

In this case, the closed-loop system is asymptotically stable and x(t) converges to its reference without any oscillations, since both eigenvalues are real.

• Case 3: $K < -3 - \frac{1}{4}$. In this case we have

$$\operatorname{Re}(\lambda_1) = \operatorname{Re}(\lambda_2) = -\frac{1}{2}$$
 and $\omega = \operatorname{Im}(\lambda_1) = -\operatorname{Im}(\lambda_2) > 0$.

This means that the closed-loop system is also asymptotically stabilizing, but it oscillates with frequency ω before converging to the reference state.