

# *Work involved in Reversible Steady Flow*

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# Previously: Entropy analysis in ideal gases

From the first  $T ds$  relation

$$ds = \frac{du}{T} + \frac{P dv}{T} \quad \begin{matrix} du = c_v dT \\ P = RT/v \end{matrix}$$

$$s_2 - s_1 = \int_1^2 c_v(T) \frac{dT}{T} + R \ln \frac{v_2}{v_1}$$

$$s_2 - s_1 = c_{v,\text{avg}} \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1}$$

$$\ln \frac{T_2}{T_1} = -\frac{R}{c_v} \ln \frac{v_2}{v_1}$$

From the second  $T ds$  relation

$$ds = \frac{dh}{T} - \frac{v dP}{T} \quad v = RT/P$$

$$s_2 - s_1 = \int_1^2 c_p(T) \frac{dT}{T} - R \ln \frac{P_2}{P_1}$$

$$s_2 - s_1 = c_{p,\text{avg}} \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1}$$

$$TV^{k-1} = \text{constant}$$

$$TP^{(1-k)/k} = \text{constant}$$

$$PV^k = \text{constant}$$

# *Work for closed system*

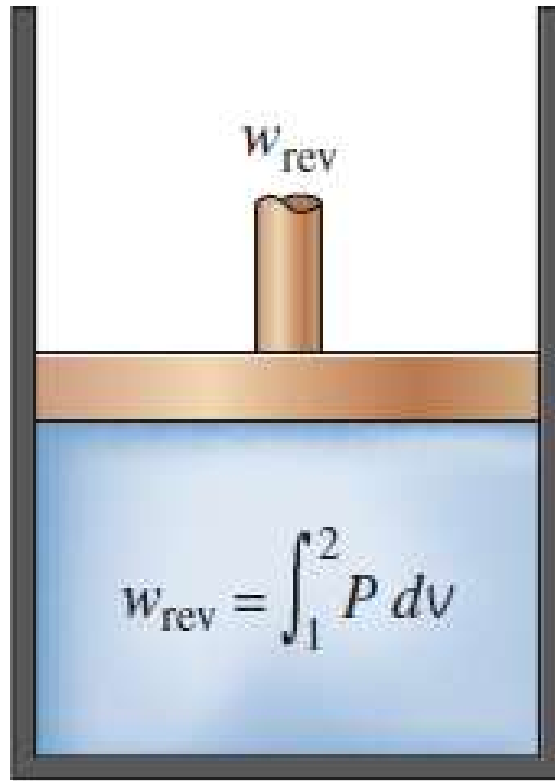


Fig-TD: Cengel & Boles

# Work for reversible steady flow device

$$\delta q_{\text{rev}} - \delta w_{\text{rev}} = dh + dke + dpe$$

From the second  $T ds$  relation

$$\delta q_{\text{rev}} = T ds$$

$$T ds = dh - v dP$$

$$-\delta w_{\text{rev}} = v dP + dke + dpe$$

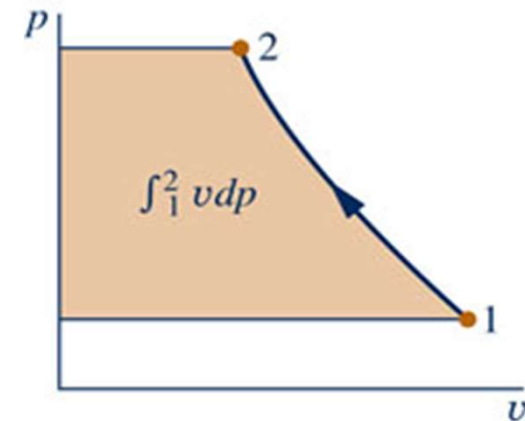
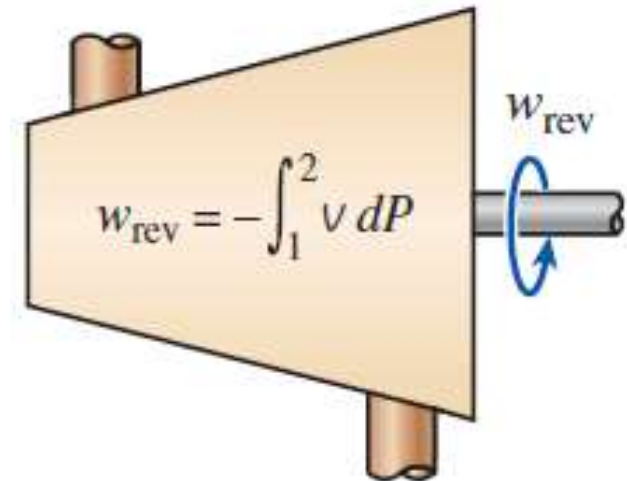
Work Output

$$w_{\text{rev}} = - \int_1^2 v dP - \Delta ke - \Delta pe$$

$$w_{\text{rev}} = - \int_1^2 v dP$$

When kinetic and potential energies are negligible

Work Input  $w_{\text{rev,in}} = \int_1^2 v dP + \Delta ke + \Delta pe$



# *Simplifications for incompressible fluids*

$$w_{\text{rev}} = - \int_1^2 v \, dP - \Delta \text{ke} - \Delta \text{pe}$$

For incompressible fluids

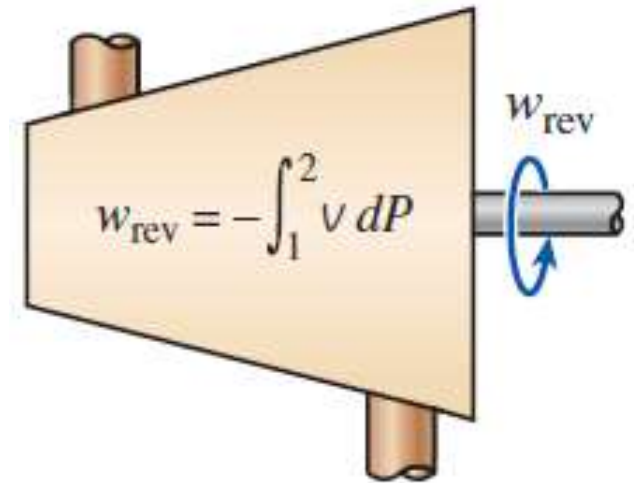
$$w_{\text{rev}} = -v(P_2 - P_1) - \Delta \text{ke} - \Delta \text{pe}$$

No work interaction: Bernoulli Equation

$$v(P_2 - P_1) + \frac{V_2^2 - V_1^2}{2} + g(z_2 - z_1) = 0$$

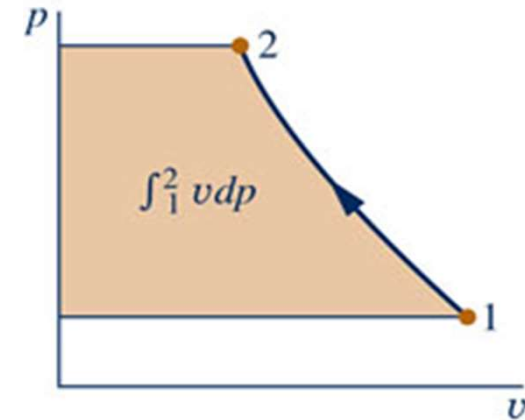
# Importance of specific volume

- Work I/P-Small  $\nu$  (Compression)
- Work O/P-Large  $\nu$  (Expansion)
- Steam power plant:  $\Delta P_{rise}^{pump} \cong \Delta P_{drop}^{turbine}$
- Pump acts on liquid and turbine via vapor; Hence, Work O/P  $\gg$  Work I/P
- Gas vs. steam power plant: Compression & Cooling



# Minimizing Work Required by Compressor

$$w_{\text{rev,in}} = \int_1^2 v \, dP \quad \begin{array}{l} \text{When kinetic and} \\ \text{potential energies} \\ \text{are negligible} \end{array}$$



Isentropic ( $Pv^k = \text{constant}$ ):

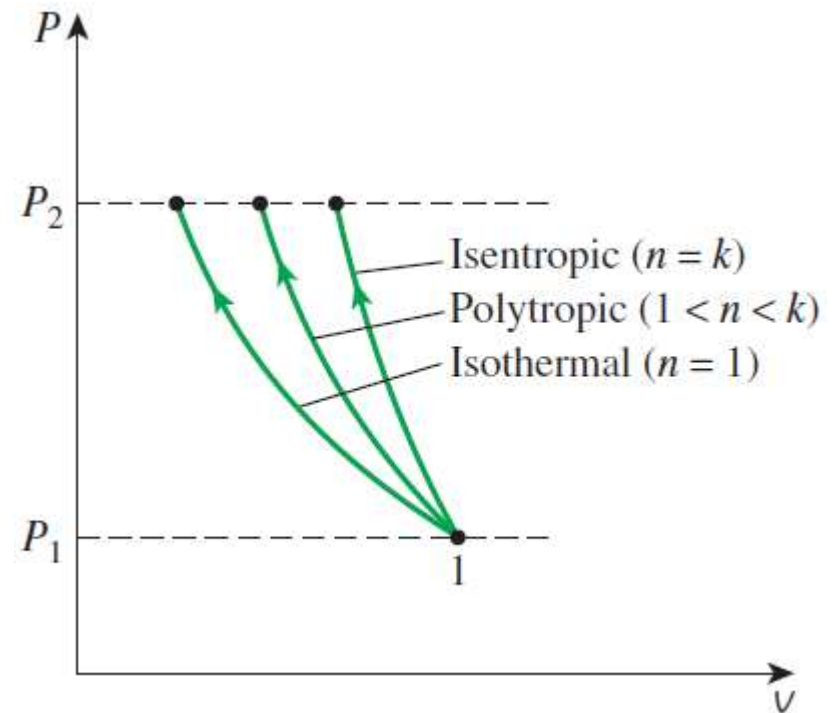
$$w_{\text{comp,in}} = \frac{kR(T_2 - T_1)}{k - 1} = \frac{kRT_1}{k - 1} \left[ \left( \frac{P_2}{P_1} \right)^{(k-1)/k} - 1 \right]$$

Polytropic ( $Pv^n = \text{constant}$ ):

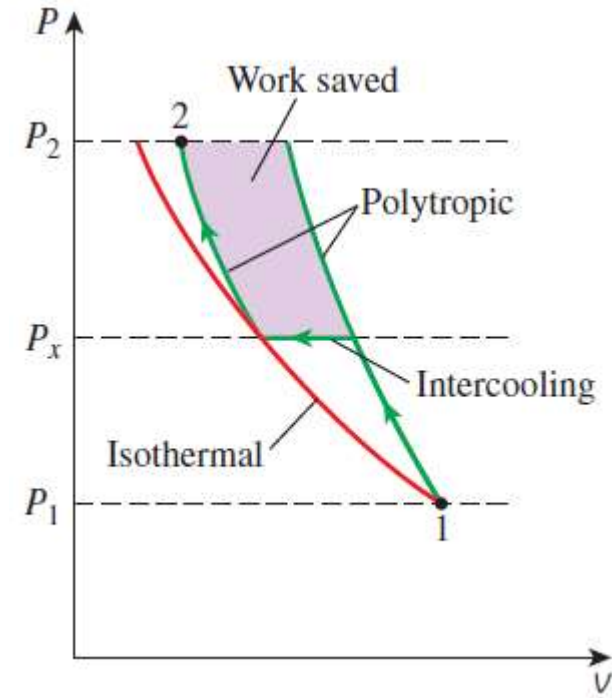
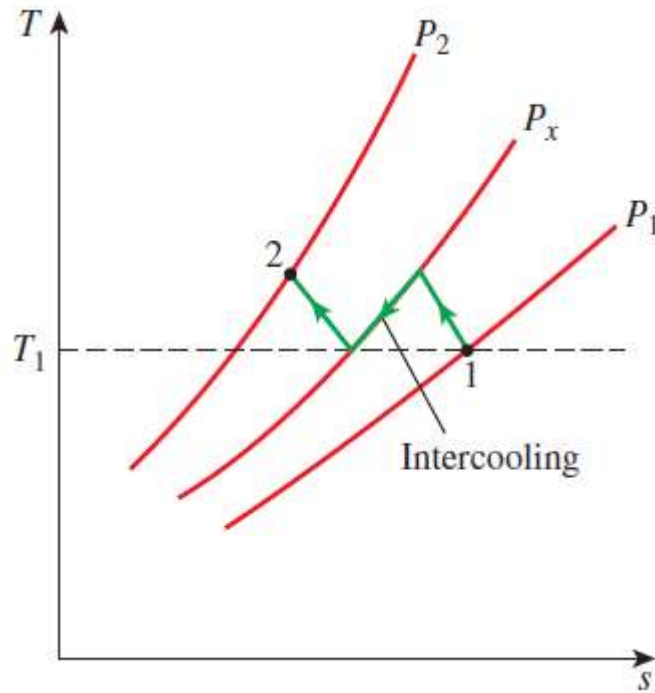
$$w_{\text{comp,in}} = \frac{nR(T_2 - T_1)}{n - 1} = \frac{nRT_1}{n - 1} \left[ \left( \frac{P_2}{P_1} \right)^{(n-1)/n} - 1 \right]$$

Isothermal ( $Pv = \text{constant}$ ):

$$w_{\text{comp,in}} = RT \ln \frac{P_2}{P_1}$$



# Work in Compressor with intercooling



$$W_{\text{comp, in}} = W_{\text{comp I, in}} + W_{\text{comp II, in}}$$

$$= \frac{nRT_1}{n-1} \left[ \left( \frac{P_x}{P_1} \right)^{(n-1)/n} - 1 \right] + \frac{nRT_1}{n-1} \left[ \left( \frac{P_2}{P_x} \right)^{(n-1)/n} - 1 \right]$$

$$P_x = (P_1 P_2)^{1/2} \quad \text{or} \quad \frac{P_x}{P_1} = \frac{P_2}{P_x}$$



*Steady-flow devices deliver (consume) most (least) work when process is reversible*

Actual

$$\delta q_{\text{act}} - \delta w_{\text{act}} = dh + dke + dpe$$

Reversible

$$\delta q_{\text{rev}} - \delta w_{\text{rev}} = dh + dke + dpe$$

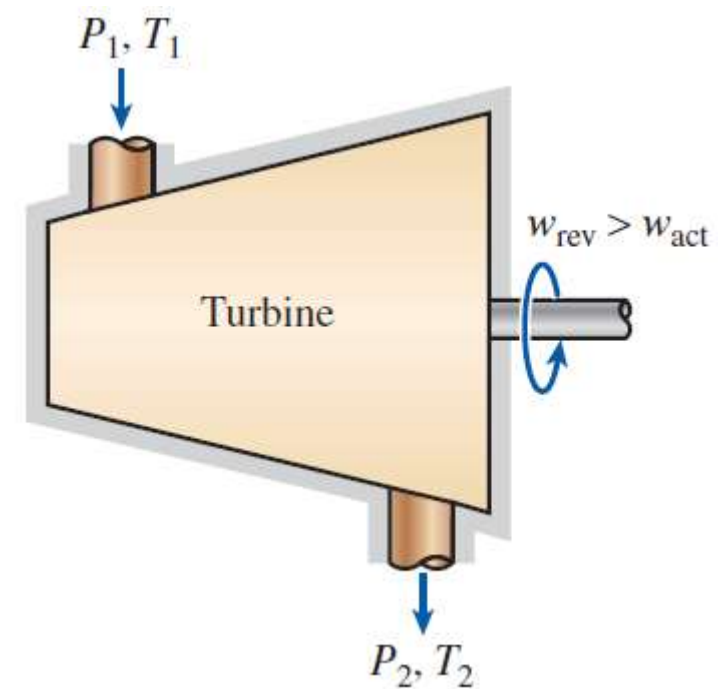
$$\delta q_{\text{act}} - \delta w_{\text{act}} = \delta q_{\text{rev}} - \delta w_{\text{rev}}$$

$$\delta w_{\text{rev}} - \delta w_{\text{act}} = \delta q_{\text{rev}} - \delta q_{\text{act}}$$

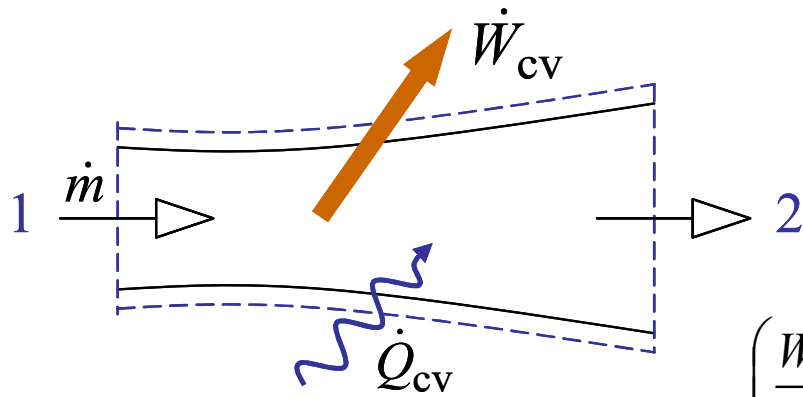
$$\delta q_{\text{rev}} = T ds \quad ds \geq \frac{\delta q_{\text{act}}}{T}$$

$$\frac{\delta w_{\text{rev}} - \delta w_{\text{act}}}{T} = ds - \frac{\delta q_{\text{act}}}{T} \geq 0$$

$$w_{\text{rev}} \geq w_{\text{act}}$$

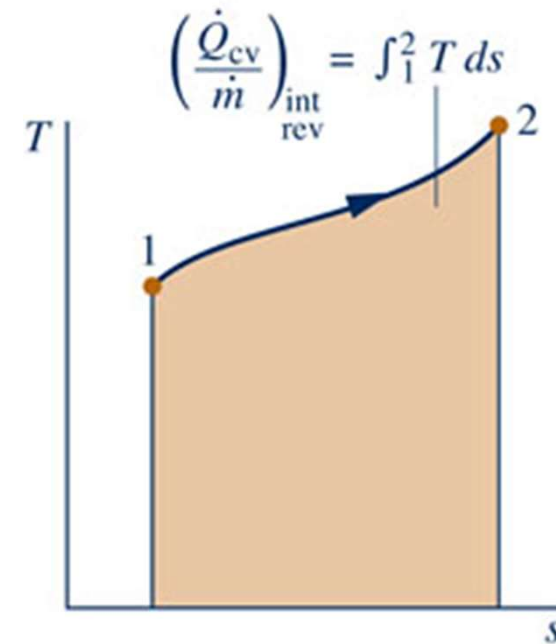


# Entropy Changes & Heat Transfer in Reversible Steady Flow process



$$\left( \frac{\dot{W}_{cv}}{\dot{m}} \right)_{\text{int rev}} = \left( \frac{\dot{Q}_{cv}}{\dot{m}} \right)_{\text{int rev}} + (h_1 - h_2) + \left( \frac{V_1^2 - V_2^2}{2} \right) + g(z_1 - z_2)$$

$$\left( \frac{\dot{Q}_{cv}}{\dot{m}} \right)_{\text{int rev}} = \int_1^2 T ds$$



## *What's next?*

- Isoentropic efficiencies of steady-flow devices