

TKT4211: Timber Structures 1





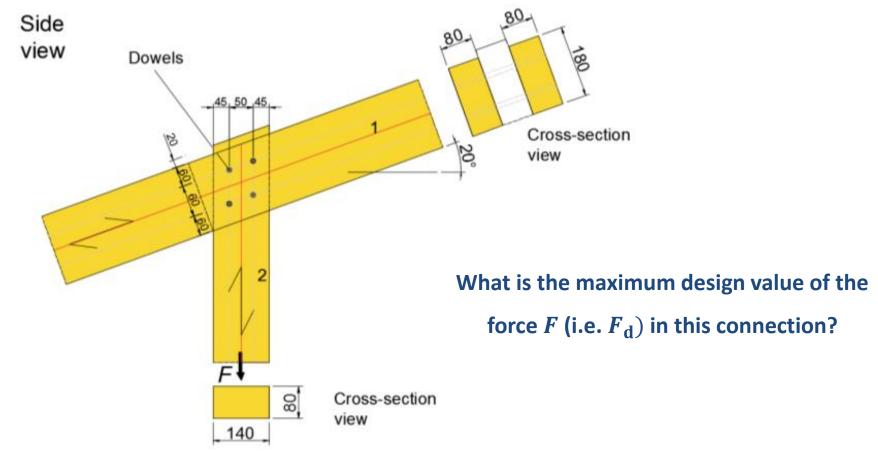


Example C2:

Timber-to-timber connection with dowels and inclined members

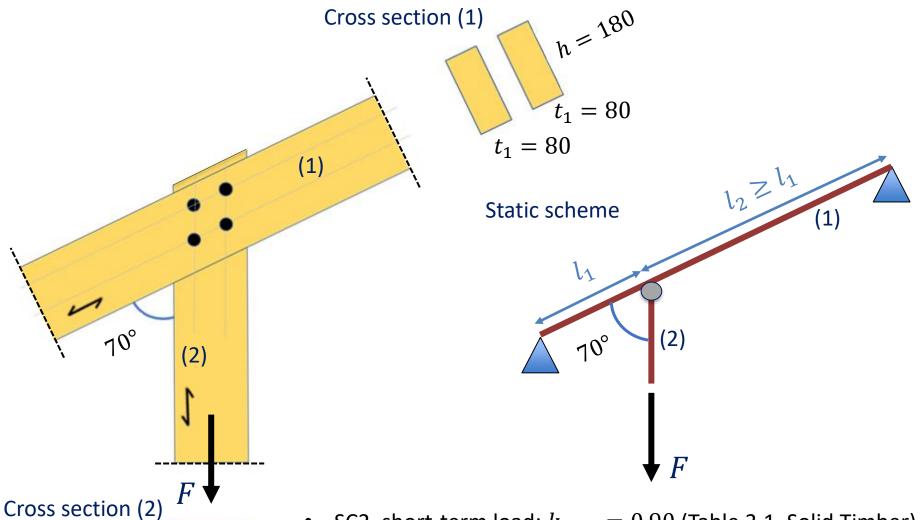
Haris Stamatopoulos

- Layout



- Timber C24
- Dowels: d = 12 mm, $f_{u,k} = 360 \text{ MPa}$
- Service class 2, short-term load

- Layout

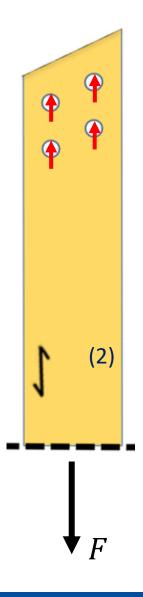


- SC2, short-term load: $k_{\rm mod} = 0.90$ (Table 3.1, Solid Timber)
- Connections: $\gamma_{\rm M}=1.30$ (Table NA.2.3)

 $t_2 = 80$

140

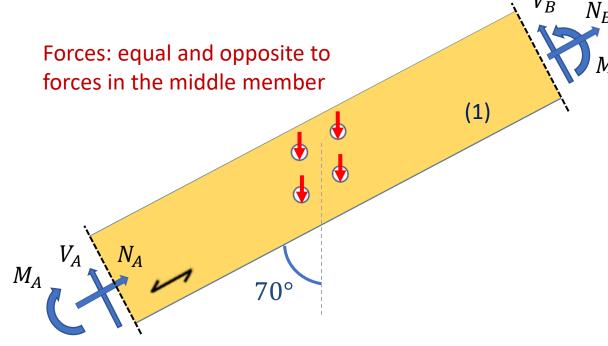
- Middle member: free body diagram



Middle member (2)

- Thickness: $t_2 = 80 \text{ mm}$ (Given by exercise)
- Timber C24 (EN338): $\rho_{\rm k,2} = 350 \ {\rm kg/m^3}$
- Eq.(8.32): $f_{h,0,k} = 0.082 \cdot (1 0.01 \cdot 12) \cdot 350 = 25.3 \text{ N/mm}^2$
- Force-to-grain angle: $\alpha_2 = 0^\circ$
- Eq.(8.31): $f_{h,2,k} = f_{h,0,k} = 25.3 \text{ N/mm}^2$

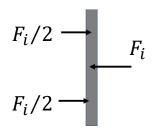
- Side members: free body diagram



Side members (1)

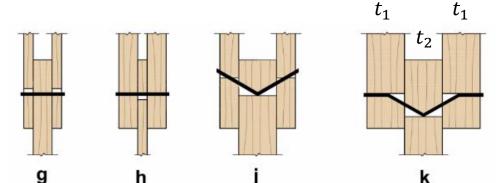
- Thickness: $t_1 = 80 \text{ mm}$ (Given)
- Timber C24 (EN338): $\rho_{k,1} = 350 \text{ kg/m}^3$
- Force-to-grain angle: $\alpha_1 = 70^{\circ}$
- Eq.(8.32): $f_{\rm h,0,k} = 0.082 \cdot (1 0.01 \cdot 12) \cdot 350 = 25.3 \,\mathrm{N/mm^2}$
- Eq.(8.33): $k_{90} = 1.35 + 0.015 \cdot 12 = 1.53$ (softwood)
- Eq.(8.31): $f_{h,\alpha,k} = f_{h,1,k} = \frac{25.3}{1.53 \cdot \sin^2(70^\circ) + \cos^2(70^\circ)} = 17.2 \text{ N/mm}^2$

- Properties of dowels



- d = 12 mm
- $f_{u,k} = 360 \text{ N/mm}^2$
- $F_{ax,Rk} = 0$ (dowels)
- Eq.(8.30): $M_{y,Rk} = 0.30 \cdot 360 \cdot 12^{2.6} = 69070 \text{ Nmm}$

- Load-carrying capacity



$$\beta = \frac{f_{\text{h,2,k}}}{f_{\text{h,1,k}}} = \frac{25.3}{17.2} = 1.47$$

EN 1995-1-1, (eq.8.8)

$$F_{\text{v,Rk}} = \min \begin{cases} f_{\text{h,1,k}} \cdot t_1 \cdot d & \text{(g)} \\ 0.5 \cdot f_{\text{h,2,k}} \cdot t_2 \cdot d & \text{(h)} \end{cases}$$

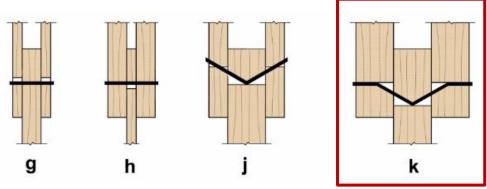
$$F_{\text{v,Rk}} = \min \begin{cases} 1.05 \frac{f_{\text{h,1,k}} \cdot t_1 \cdot d}{2 + \beta} \left[\sqrt{2\beta(1+\beta) + \frac{4\beta(2+\beta)M_{\text{y,Rk}}}{f_{\text{h,1,k}} \cdot d \cdot t_1^2}} - \beta \right] + \frac{F_{\text{ax,Rk}}}{4} & \text{(j)} \end{cases}$$

$$1.15 \sqrt{\frac{2\beta}{1+\beta}} \sqrt{2M_{\text{y,Rk}} \cdot f_{\text{h,1,k}} \cdot d} + \frac{F_{\text{ax,Rk}}}{4} & \text{(k)}$$

EN 1995-1-1, §8.2.2.(1), (eq.8.7)

Load-carrying capacity

- Force parallel to grain in side members



Timber-to-timber connections: Fasteners in double shear (eq.8.7)

$$\begin{split} F_{\text{v,Rk(g)}} &= 17.2 \cdot 80 \cdot 12 = 16512 \text{ N} \\ F_{\text{v,Rk(h)}} &= 0.5 \cdot 25.3 \cdot 80 \cdot 12 = 12144 \text{ N} \\ F_{\text{v,Rk(j)}} &= 1.05 \cdot \frac{17.2 \cdot 80 \cdot 12}{2 + 1.47} \left[\sqrt{2 \cdot 1.47 \cdot (1 + 1.47) + \frac{4 \cdot 1.47 \cdot (2 + 1.47) \cdot 69070}{17.2 \cdot 12 \cdot 80^2}} - 1.47 \right] + \frac{0}{4} = 7075 \text{ N} \\ F_{\text{v,Rk(k)}} &= 1.15 \cdot \sqrt{\frac{2 \cdot 1.47}{1 + 1.47}} \cdot \sqrt{2 \cdot 69070 \cdot 17.2 \cdot 12} + \frac{0}{4} = \mathbf{6699 N} \end{split}$$

• Load carrying capacity per shear plane per fastener:

$$F_{v,Rk} = \min(F_{v,Rk(g)}; F_{v,Rk(h)}; F_{v,Rk(j)}; F_{v,Rk(k)}) = 6699 \text{ N}$$
 [Failure mode (k)]

Load transfer

- Design check
 - Design load carrying capacity per shear plane per fastener:

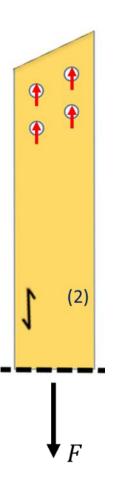
$$F_{\text{v,Rd,i}} = \frac{k_{\text{mod}}}{\gamma_{\text{M}}} \cdot F_{\text{v,Rk,i}} = \frac{0.9}{1.3} \cdot 6699 = 4638 \text{ N}$$
EN 1995-1-1, §2.4.3(1), eq.(2.17)

Load- per fastener per shear plane per fastener:

$$F_{\rm d,i} = \frac{F_{\rm d}}{n_{\rm shear \, planes} \cdot n_{\rm fasteners}} = \frac{F_{\rm d}}{2 \cdot 4} = \frac{F_{\rm d}}{8}$$

• Design check ($F_{d,i} \leq F_{v,Rd,i}$):

$$\frac{F_{\rm d}}{8}$$
 ≤ 4638 N \rightarrow $F_{\rm d}$ ≤ 37.1kN (load transfer)



Spacings, end and edge distances

- Dowels: EN1995-1-1, Table 8.5

Table 8.5 — Minimum spacings and edge and end distances for dowels

Spacing and edge/end distances (see Figure 8.7)	Angle to grain	Minimum spacings and edge/end distances
a ₁ (parallel to grain)	0° ≤ α ≤ 360°	$(3+2 \cos\alpha)d$
a ₂ (perpendicular to grain)	0° ≤ α ≤ 360°	3 d
a _{3,t} (loaded end)	- 90° ≤ α ≤ 90°	max (7 d; 80 mm)
a _{3,c} (unloaded end)	90° ≤ α ≤ 150°	$a_{3t} \sin \alpha $
	150° ≤ α ≤ 210°	max(3,5 d; 40 mm)
	210° ≤ α ≤ 270°	$a_{3t} \sin \alpha $
a _{4,t} (loaded edge)	0° ≤ α ≤ 180°	$\max((2+2\sin\alpha)d;3d)$
a _{4,c} (unloaded edge)	180° ≤ α ≤ 360°	3 d



Spacings, end and edge distances

- Middle member

• Minimum requirements (Table 8.5), $\alpha = 0^{\circ}$

$$a_1 \ge (3 + 2 \cdot |\cos 0^{\circ}|) \cdot d = 5 \cdot d = 60 \text{ mm}$$

$$a_2 \ge 3 \cdot d = 36 \text{ mm}$$

$$a_{3,t} \ge \max(7 \cdot d, 80) = 84 \text{ mm}$$

$$a_{4,c} \ge 3 \cdot d = 36 \text{ mm}$$

 $a_{3,c}$ and $a_{4,t}$ are not relevant here

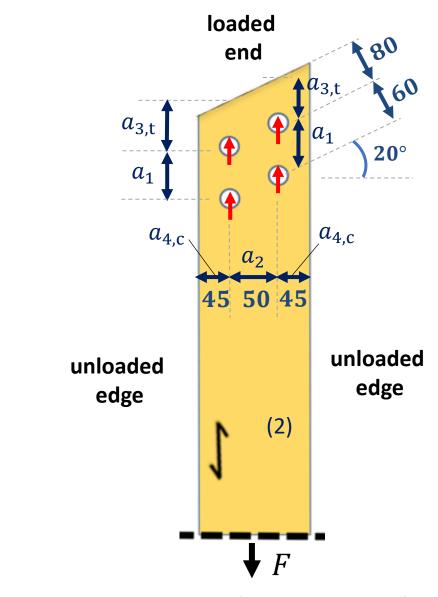
Actual distances and spacings (and check)

$$a_1 = 60/\cos 20^\circ = 64 \text{ mm} > 60 \text{ mm}$$
 (OK)

$$a_2 = 50 \text{ mm} > 36 \text{ mm}$$
 (OK)

$$a_{3,t} = 80/\cos 20^\circ = 85 \text{mm} > 84 \text{ mm}$$
 (OK)

$$a_{4,c} = 45 \text{ mm} > 36 \text{ mm}$$
 (OK)

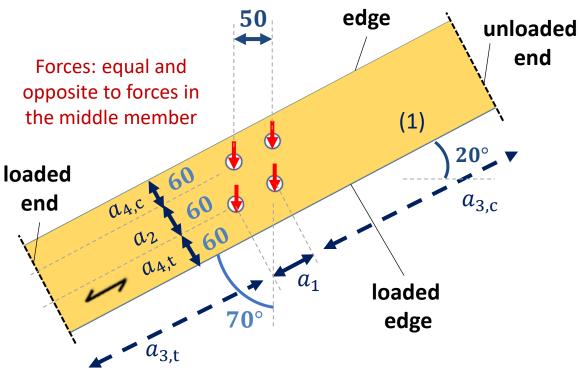


- There is not a loaded edge (both are unloaded)
- There is not an unloaded end (far away)



Spacings, end and edge distances

- Side members



 $a_{3,t}$ and $a_{3,c}$ are not relevant here (ends are far away)

• Minimum requirements (Table 8.5), $\alpha=70^\circ$

$$a_1 \ge (3 + 2 \cdot |\cos 70^{\circ}|) \cdot d = 44.2 \text{ mm}$$

$$a_2 \ge 3 \cdot d = 36 \text{ mm}$$

$$a_{4,t} \ge \max[(2 + 2 \cdot \sin 70^\circ) \cdot d, 3 \cdot d] = 46.5 \text{ mm}$$

$$a_{4,c} \ge 3 \cdot d = 36 \text{ mm}$$

Actual distances and spacings (and check)

$$a_1 = 50/\cos 20^\circ = 53 \text{ mm} > 44.2 \text{ mm}$$
 (OK)

unloaded

$$a_2 = 60 \text{ mm} > 36 \text{ mm}$$
 (OK)

$$a_{4,t} = 60 \text{ mm} > 46.5 \text{ mm}$$
 (OK)

$$a_{4,c} = 60 \text{ mm} > 36 \text{ mm}$$
 (OK)

Splitting: force components parallel to grain

- Middle member
- Effective number of fasteners per row parallel to grain, Eq.(8.34)
- 2 fasteners in each row, i.e n=2

$$n_{\text{ef}} = \min\left(n, n^{0.90} \cdot \sqrt[4]{\frac{a_1}{13 \cdot d}}\right) = \min\left(2, 2^{0.90} \cdot \sqrt[4]{\frac{64}{13 \cdot 12}}\right) = 1.49 \text{ fasteners per row}$$

- Effective load-carrying capacity of each row, Eq.(8.1)
- Comment: for force applied parallel to grain in middle member the load-carrying capacity is $F_{v,Rk(\alpha_2=0^\circ)}=6699$ N (per fastener per shear plane). Same as before.

$$F_{\text{v,ef,Rk}} = n_{\text{ef}} \cdot F_{\text{v,Rk}(\alpha_2=0^\circ)} = 1.49 \cdot 6699 = 9982 \text{ N}$$
 per shear plane EN 1995-1-1, §8.1.2(4), eq.(8.1)

$$F_{\text{v,ef,Rd}} = n_{\text{shear planes}} \cdot k_{\text{mod}} / \gamma_{\text{M}} \cdot F_{\text{v,ef,Rk}} = 2 \cdot 0.9 / 1.3 \cdot 9982 = 13821 \text{ N}$$

EN 1995-1-1, §2.4.3(1), eq.(2.17)

Design check, §8.1.2(5)

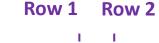
$$F_{\text{row,d}} = \frac{F_{\text{d}}}{2 \text{ rows}} \le F_{\text{v,ef,Rd}} \text{ (per row)} = 13821 \text{ N}$$

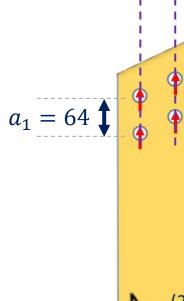
EN 1995-1-1, **§8.1.2(5)**

Comment: if members are loaded with a force parallel to grain, this failure mode will always be more critical than load transfer, because $n_{\rm ef} \leq n$

$$\rightarrow F_{\rm d} \leq 27.6 \, \rm kN$$

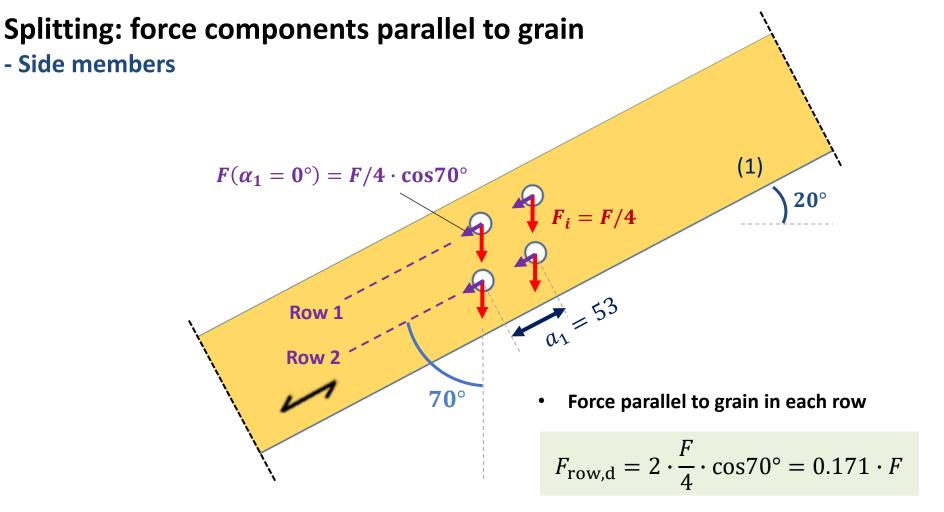
splitting for *F* parallel to grain (middle member)





per row





• We need to find the load carrying capacity for force parallel to grain in the side members $F_{\rm v,Rk}(lpha_1=0^\circ)$, i.e. for $lpha_1=0^\circ$ (and therefore $lpha_2=70^\circ$)

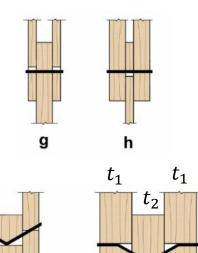
- Load-carrying capacity: force parallel to grain in side members

$$F_{\text{v,Rk}} = \min \begin{cases} f_{\text{h,1,k}} \cdot t_1 \cdot d & \text{(g)} \\ 0.5 \cdot f_{\text{h,2,k}} \cdot t_2 \cdot d & \text{(h)} \end{cases}$$

$$1.05 \frac{f_{\text{h,1,k}} \cdot t_1 \cdot d}{2 + \beta} \left[\sqrt{2\beta(1+\beta) + \frac{4\beta(2+\beta)M_{\text{y,Rk}}}{f_{\text{h,1,k}} \cdot d \cdot t_1^2}} - \beta \right] + \frac{F_{\text{ax,Rk}}}{4} & \text{(j)}$$

$$1.15 \sqrt{\frac{2\beta}{1+\beta}} \sqrt{2M_{\text{y,Rk}} \cdot f_{\text{h,1,k}} \cdot d} + \frac{F_{\text{ax,Rk}}}{4} & \text{(k)}$$

EN 1995-1-1, §8.2.2.(1), (eq.8.7)



Side members (1)

- Force-to-grain angle: $\alpha_1 = 0^{\circ}$, Eq.(8.32): $f_{h,0,k} = 25.3 \text{ N/mm}^2$
- Eq.(8.31): $f_{h,1,k} = f_{h,0,k} = 25.3 \text{ N/mm}^2$

$\beta = \frac{f_{\text{h,2,k}}}{f_{\text{h,1,k}}} = \frac{17.2}{25.3} = 0.68$

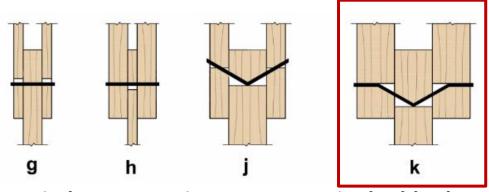
EN 1995-1-1, (eq.8.8)

Middle member (2)

- Eq.(8.32): $f_{h,0,k} = 25.3 \text{ N/mm}^2$
- Force-to-grain angle: $\alpha_2 = 70^\circ$, Eq.(8.33): $k_{90} = 1.35 + 0.015 \cdot 12 = 1.53$ (softwood)
- Eq.(8.31): $f_{h,\alpha,k} = f_{h,2,k} = \frac{25.3}{1.53 \cdot \sin^2(70^\circ) + \cos^2(70^\circ)} = 17.2 \text{ N/mm}^2$

Load-carrying capacity

- Timber-to-timber connections: Fasteners in double shear



Timber-to-timber connections: Fasteners in double shear (eq.8.7)

$$\begin{split} F_{\text{v,Rk(g)}} &= 25.3 \cdot 80 \cdot 12 = 24288 \, \text{N} \\ F_{\text{v,Rk(h)}} &= 0.5 \cdot 17.2 \cdot 80 \cdot 12 = 8256 \text{N} \\ F_{\text{v,Rk(j)}} &= 1.05 \cdot \frac{25.3 \cdot 80 \cdot 12}{2 + 0.68} \left[\sqrt{2 \cdot 0.68 \cdot (1 + 0.68) + \frac{4 \cdot 0.68 \cdot (2 + 0.68) \cdot 69070}{25.3 \cdot 12 \cdot 80^2}} - 0.68 \right] + \frac{0}{4} = 8706 \, \text{N} \\ F_{\text{v,Rk(k)}} &= 1.15 \cdot \sqrt{\frac{2 \cdot 0.68}{1 + 0.68}} \cdot \sqrt{2 \cdot 69070 \cdot 25.3 \cdot 12} + \frac{0}{4} = \mathbf{6700} \, \mathbf{N} \end{split}$$

• Load carrying capacity per shear plane per fastener:

$$F_{v,Rk} = \min(F_{v,Rk(g)}; F_{v,Rk(h)}; F_{v,Rk(j)}; F_{v,Rk(k)}) = 6700N$$
 [Failure mode (k)]

Splitting: force components parallel to grain

- Side members
 - Effective number of fasteners per row parallel to grain, Eq.(8.34)
 - 2 fasteners in each row, i.e n = 2

$$n_{\text{ef}} = \min\left(n, n^{0.90} \cdot \sqrt[4]{\frac{a_1}{13 \cdot d}}\right) = \min\left(2, 2^{0.90} \cdot \sqrt[4]{\frac{53}{13 \cdot 12}}\right) = 1.42 \text{ fasteners per row}$$

- Effective load-carrying capacity of each row, Eq.(8.1)
- Comment: for force applied parallel to grain in middle member the load-carrying capacity is $F_{\rm v,Rk}(\alpha_1=0^\circ)=6700{\rm N}$ (per fastener per shear plane).

$$F_{\text{v,ef,Rk}} = n_{\text{ef}} \cdot F_{\text{v,Rk}(\alpha_1 = 0^\circ)} = 1.42 \cdot 6700 = 9514 \text{ N}$$

per shear plane

EN 1995-1-1, §8.1.2(4), eq.(8.1)

$$F_{\text{v,ef,Rd}} = n_{\text{shear planes}} \cdot k_{\text{mod}} / \gamma_{\text{M}} \cdot F_{\text{v,ef,Rk}} = 2 \cdot 0.9 / 1.3 \cdot 9514 = 13173 \text{ N}$$
 per row

EN 1995-1-1, §2.4.3(1), eq.(2.17)

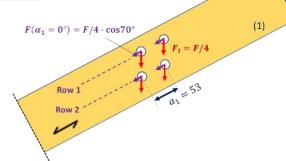
Design check, §8.1.2(5)

$$F_{\text{row,d}} = 0.171 \cdot F \le F_{\text{v,ef,Rd}} \text{ (per row)} = 13173 \text{ N}$$

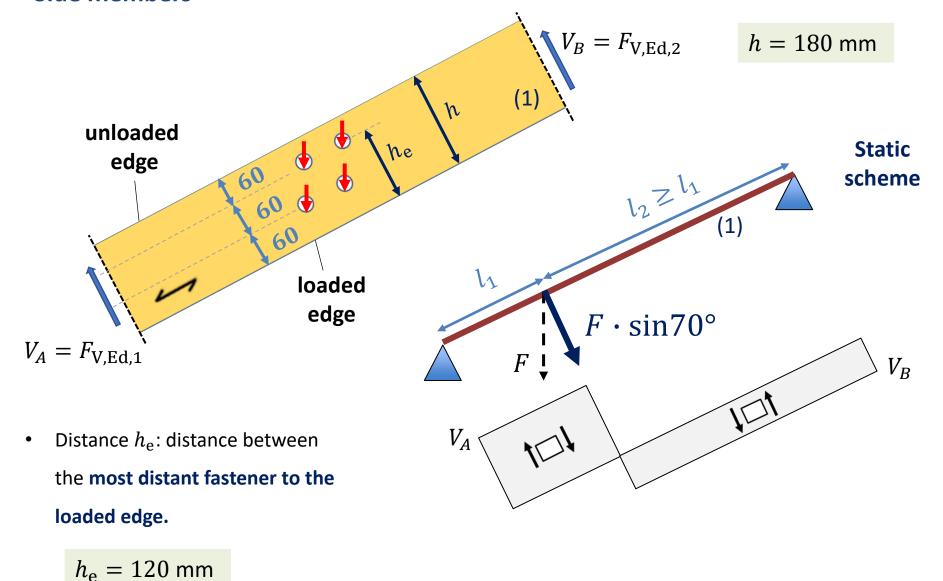
EN 1995-1-1, **§8.1.2(5)**

$$\rightarrow$$
 $F_{\rm d} \leq 77.0 \, \rm kN$

splitting for *F* parallel to grain (side members)



- Side members



- Side members
- Maximum design shear force:

$$F_{\text{v,Ed}} = \max(F_{\text{v,Ed,1}}, F_{\text{v,Ed,2}})$$
EN 1995-1-1, §8.1.4(2), (eq.8.3)

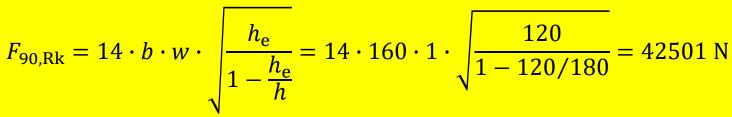
No info provided for the lengths l_1 and l_2 .

Worst case scenario:

$$l_1 = 0 \rightarrow F_{V,Ed,1} = F \cdot \sin 70^{\circ}, F_{V,Ed,2} = 0$$

$$F_{\text{v,Ed}} = \max(F \cdot \sin 70^{\circ}, 0) = F \cdot \sin 70^{\circ}$$

Characteristic splitting capacity (softwood):

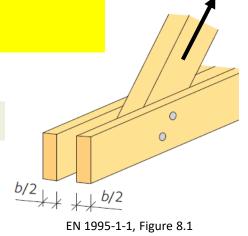


EN 1995-1-1, §8.1.4(3), (eq.8.4)

$$w = 1$$

EN 1995-1-1, §8.1.4(3), (eq.8.5)

$$b = b/2 + b/2 = 80 + 80 = 160 \text{ mm}$$



- Side members

Limtreboka mener at en kan bruke gammaM til materialet, fordi det er en materialsjekke, og ikke en forbindelsessjekk. I dette tilfelle altså 1,25.

Design splitting capacity:

$$F_{90,Rd} = k_{mod}/\gamma_{M} \cdot F_{90,Rk} = 0.9/1.3 \cdot 42501 = 29423 \text{ N}$$

EN 1995-1-1, §2.4.3(1), eq.(2.17)

Design check (worst case scenario):

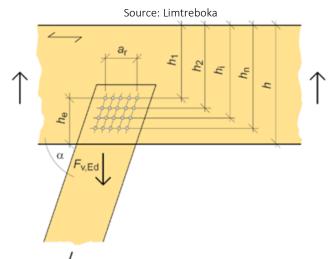
$$F_{\text{v,Ed}} = \max(F_{\text{v,Ed,1}}, F_{\text{v,Ed,2}}) = F \cdot \sin 70^{\circ} \le F_{90,\text{Rd}} = 29423 \text{ N}$$

EN 1995-1-1, §8.1.4(2), eqs.(8.2-8.3)

$$\rightarrow F_{\rm d} \leq 31.3 \, \rm kN$$

splitting for *F* perpendicular to grain (side members)

- Side members, alternative method (info)



Timber C24 (EN338):

$$f_{\rm t,90,k} = 0.4 \text{ N/mm}^2$$
 $\gamma_{\rm M} = 1.25$ $f_{\rm t,90,d} = 0.4 \cdot 0.9 / 1.25 = 0.288 \text{ N/mm}^2$

$$b = 160 \text{ mm}$$
 $a_r = 53 \text{ mm} (< h/2 = 90 \text{ OK})$

$$h = 180 \text{ mm}$$
 $t_{\text{pen}} = 80 \text{ mm}$

$$h_{\rm e} = 120 \, {\rm mm}$$
 $t_{\rm ef} = {\rm min} \{ b \; ; \; 2 \cdot t_{\rm pen}; \; 12 \cdot d \} = 144 \, {\rm mm}$

$$k_{\rm s} = \max\{1; 0.7 + 1.4 \cdot a_r/h\} = \max\{1; 0.7 + 1.4 \cdot 53/180\} = 1.11$$

$$h_1 = 60 \text{ mm}$$

$$h_2=120~\mathrm{mm}$$

$$n = 2$$
 rows

$$k_{\rm r} = \frac{n}{\sum_{i=1}^{n} (h_1/h_i)^2} = \frac{2}{(h_1/h_1)^2 + (h_1/h_2)^2} = 1.60$$

Design capacity:
$$F_{90,Rd} = k_s \cdot k_r \cdot (6.5 + 18 \cdot (h_e/h)^2) \cdot (t_{ef} \cdot h)^{0.8} \cdot f_{t,90,d}$$

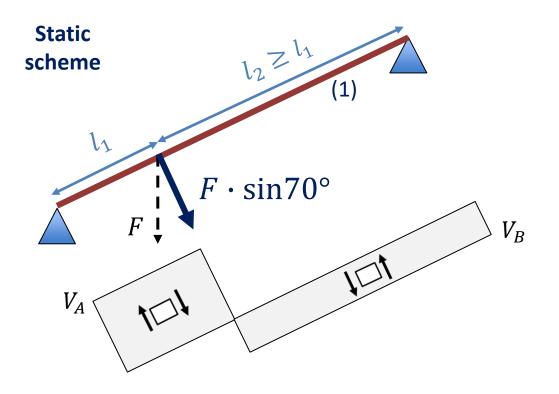
$$F_{90,Rd} = 1.11 \cdot 1.60 \cdot (6.5 + 18 \cdot (120/180)^2) \cdot (144 \cdot 180)^{0.8} \cdot 0.288 = 25200 \text{ N}$$

$$F_{\rm v,Ed} = F \cdot \sin 70^{\circ} \le F_{\rm 90,Rd} = 25.2 \text{ kN}$$
 \rightarrow $F_{\rm d} \le 26.8 \text{ kN}$

Note: here the capacity is not dependent on the value of shear (and therefore on the value of l_1)

Force components perpendicular to grain

- Shear capacity



Timber C24 (EN338):

$$f_{v,k} = 4.0 \text{ N/mm}^2$$

$$\gamma_{\rm M} = 1.25$$

$$k_{\rm cr} = 0.67$$

$$f_{\rm v,d} = 4 \cdot 0.9 / 1.25 = 2.88 \, \text{N/mm}^2$$

Shear capacity:

$$V_{\text{max,d}} = \frac{2}{3} \cdot k_{\text{cr}} \cdot b \cdot h \cdot f_{\text{v,d}} = \frac{2}{3} \cdot 0.67 \cdot (2 \cdot 80) \cdot 180 \cdot 2.88 = 37050 \text{ N}$$

EN 1995-1-1 §6.1.7.(1) eq.(6.13)

No info provided for the lengths l_1 and l_2 .

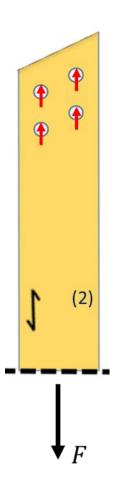
Worst case scenario:
$$l_1 = 0 \rightarrow F_{V,Ed,1} = F \cdot \sin 70^\circ \le 37050 \text{ N}$$

$$\rightarrow$$
 $F_{\rm d} \leq 39.4 \, \rm kN$

Shear (side members)

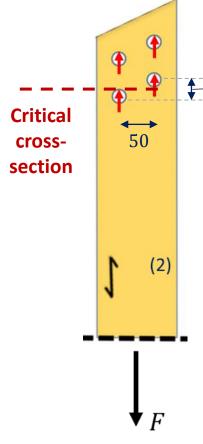
- Middle member

• Splitting perpendicular to grain is not relevant in the middle member because there are no force components perpendicular to grain (i.e. no shear: $F_{V,Ed} = 0$)



Net-section failure

- Middle member



(Table 8.5):
$$a_{1,\min} = 60 \text{ mm}$$

$$50 \cdot \tan 20^\circ = 18 \text{ mm} \left(< \frac{a_{1,\text{min}}}{2} = 30 \text{ mm} \right)$$

EN 1995-1-1 §5.2.(4), staggered

fasteners: We should consider the

fasteners in the same cross section

$$\sigma_{\text{t,0,d}} = \frac{F}{A_{\text{net}}} \le f_{\text{t,0,d}}$$

EN 1995-1-1, §6.1.2, eq.(6.1)

Assume small tolerance: $d \approx d_{\text{hole}}$

$$A_{\text{net}} = 80 \cdot (140 - 2 \cdot 12) = 9280 \text{ mm}^2$$

$$F \le f_{t,0,d} \cdot A_{\text{net}} = 10.5 \cdot 9280 = 97440 \text{ N}$$

 $F_{\rm d} \leq 97.4 \, \rm kN$

Tension parallel to grain (middle member)

Timber C24 (EN338):

$$f_{\rm t,0,k} = 14.5 \, \rm N/mm^2$$

$$\gamma_{\rm M} = 1.25$$

$$k_{\rm h} = \min((150/140)^{0.2}, 1.3) = 1.01$$

EN 1995-1-1 eq.(3.1)

$$f_{t,0,d} = 14.5 \cdot 1.01 \cdot 0.9 / 1.25 = 10.5 \text{ N/mm}^2$$

EN 1995-1-1 (eq.2.14)

Net-section failure

- Side members

Timber C24 (EN338):

$$f_{\rm m,k} = 24 \text{ N/mm}^2$$

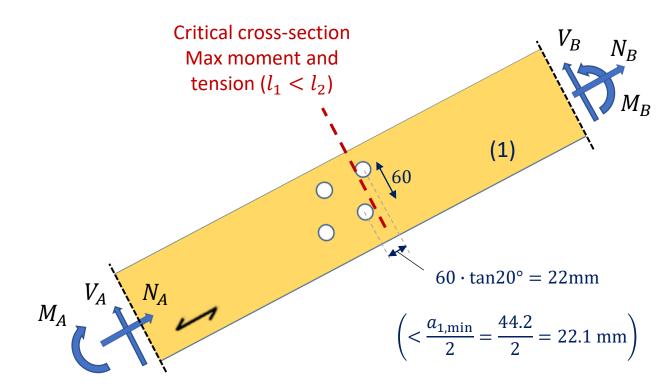
$$f_{\rm t,0,k} = 14.5 \text{ N/mm}^2$$

$$h > 150 \text{ mm} \rightarrow k_{\text{h}} = 1.0$$

EN 1995-1-1 eq.(3.1)

$$f_{c,0,k} = 21.0 \text{ N/mm}^2$$

$$\gamma_{\rm M} = 1.25$$



$$f_{\rm m,d} = 24 \cdot 0.9 / 1.25 = 17.3 \text{ N/mm}^2$$

$$f_{\rm t,0,d} = 14.5 \cdot 0.9 / 1.25 = 10.44 \text{ N/mm}^2 \left(< f_{c,0,d} \right)$$
EN 1995-1-1 (eq.2.14)

Comment: The check for tension and moment is critical (on the right) because the moment is higher $(l_1 < l_2)$ and the tensile strength is smaller than the compressive.

FIN 1995-1-1 §5.2.(4), staggered fasteners: We should consider the fasteners in the same cross section

Net-section failure

- Side members

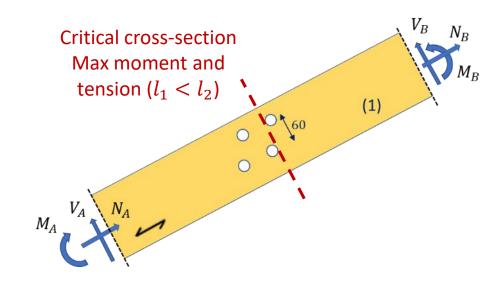
Combined side members

$$A_{\text{gross}} = 2 \cdot 80 \cdot 180 = 28800 \text{ mm}^2$$

$$A_{\text{net}} = 2 \cdot 80 \cdot (180 - 2 \cdot 12) = 24960 \text{ mm}^2$$

$$\frac{A_{\rm gross}}{A_{\rm net}} = \frac{28800}{24960} = 1.154$$

$$I_{\text{gross}} = \frac{b \cdot h^3}{12} = \frac{2 \cdot 80 \cdot 180^3}{12} = 7.78 \cdot 10^7 \text{mm}^4$$



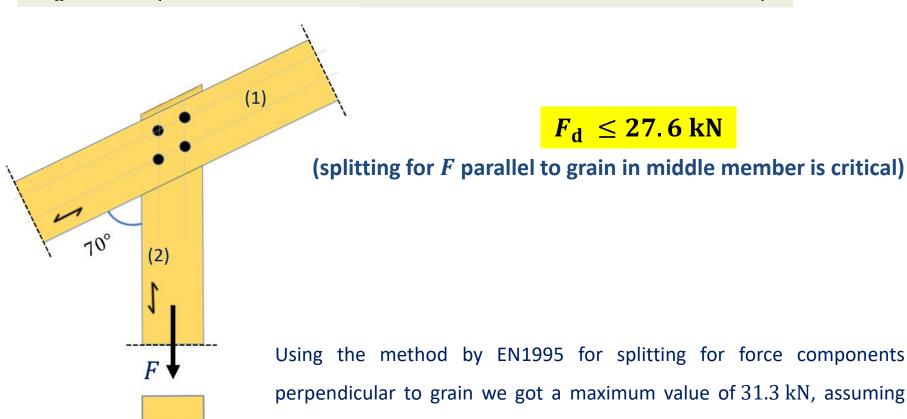
$$I_{\text{net}} = I_{\text{gross}} - 2 \cdot \left(\frac{b \cdot d^3}{12} + y_i^2 \cdot b \cdot d\right) = 7.78 \cdot 10^7 - 2 \cdot \left(\frac{2 \cdot 80 \cdot 12^3}{12} + 30^2 \cdot 2 \cdot 80 \cdot 12\right) = 7.43 \cdot 10^7 \text{ mm}^4$$

$$\frac{W_{\text{gross}}}{W_{\text{net}}} = \frac{I_{\text{gross}}/(h/2)}{I_{\text{net}}/(h/2)} = \frac{I_{\text{gross}}}{I_{\text{net}}} = \frac{7.78 \cdot 10^7}{7.43 \cdot 10^7} = 1.047$$

Comment: The lengths l_1 , l_2 are not given and we cannot determine the value of moment and perform the net-section design check here. Compared to the gross cross-section, the axial stresses and the bending stresses will be **15.4%** and **4.7%** higher respectively due to the presence of holes.

- Summary

 $F_{\rm d} \leq \min(37.1 \text{ kN}, 27.6 \text{ kN}, 77.0 \text{ kN}, 31.3 \text{ kN}^*, 39.4 \text{ kN}, 97.4 \text{ kN})$



Using the method by EN1995 for splitting for force components perpendicular to grain we got a maximum value of 31.3 kN, assuming conservatively $l_1=0.$ Using the alternative method, we got a maximum value of 26.8 kN (i.e. smaller that 27.6 kN and therefore critical)