

Detected Resonant Frequencies (0-200 Hz):

18
88
150
188

Estimated Damping Ratios:

Mode near 18.00 Hz → $\zeta = 0.2222$
Mode near 88.00 Hz → $\zeta = 0.0455$
Mode near 150.00 Hz → $\zeta = 0.0267$
Mode near 188.00 Hz → $\zeta = 0.0213$

Normalized Mode Shapes (0-200 Hz):

Mode at 18.00 Hz:

X: $0.077 \angle 10.2^\circ$
Y: $0.052 \angle 16.2^\circ$
Z: $1.000 \angle 8.6^\circ$

Mode at 88.00 Hz:

X: $0.100 \angle 90.7^\circ$
Y: $1.000 \angle -11.6^\circ$
Z: $0.887 \angle -171.6^\circ$

Mode at 150.00 Hz:

X: $0.324 \angle 24.1^\circ$
Y: $0.557 \angle -142.1^\circ$
Z: $1.000 \angle 13.2^\circ$

Mode at 188.00 Hz:

X: $0.578 \angle -6.5^\circ$
Y: $1.000 \angle 73.0^\circ$
Z: $0.683 \angle -10.1^\circ$

=== Modal Parameter Summary (0-200 Hz) ===

Freq_Hz	Damping	Phi_X	Phi_Y	Phi_Z

18	0.22222	0.076026+0.013689i	0.049614+0.014393i	0.98884+0.14895i
88	0.045455	-0.0011357+0.099792i	0.9796-0.20094i	-0.87703-0.12989i
150	0.026667	0.29521+0.13231i	-0.44007-0.3422i	0.97345+0.2289i
188	0.021277	0.57464-0.065548i	0.29236+0.95631i	0.67193-0.11986i

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Estimated Damping Ratios:

0.2222
0.0455
0.0267
0.0213

=== Modal Summary (0-200 Hz) ===

Mode 1: f=18.00 Hz, Damping=0.2222, k_dyn_X=6123318.31 N/m, k_dyn_Y=9156460.24 N/m, k_dyn_Z=473015.20 N/m

Mode Shape (X,Y,Z): 0.077∠10.2°, 0.052∠16.2°, 1.000∠8.6°

Mode 2: f=88.00 Hz, Damping=0.0455, k_dyn_X=1156981103.49 N/m, k_dyn_Y=115464859.20 N/m, k_dyn_Z=130233445.79 N/m

Mode Shape (X,Y,Z): 0.100∠90.7°, 1.000∠-11.6°, 0.887∠-171.6°

Mode 3: f=150.00 Hz, Damping=0.0267, k_dyn_X=284621559.08 N/m, k_dyn_Y=165170512.97 N/m, k_dyn_Z=92075439.94 N/m

Mode Shape (X,Y,Z): 0.324∠24.1°, 0.557∠-142.1°, 1.000∠13.2°

Mode 4: f=188.00 Hz, Damping=0.0213, k_dyn_X=364937655.67 N/m, k_dyn_Y=211068719.16 N/m, k_dyn_Z=309242849.09 N/m