

① Model the system

② Set performance
Metrics

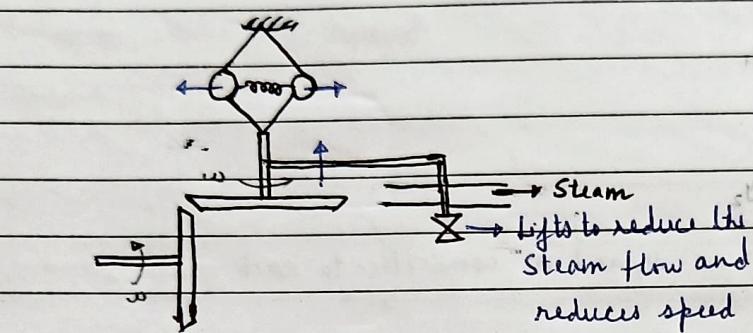
③ Check if the
System meets the
Desired performance
criteria/criteria

④ Suppose the limits;
Decide on the type of
Controller you want.

⑤ Synthesize / Design
the chosen controller

★⑥ Carry out extensive
simulation of the
designed controller
and if unsatisfactory
go to step 5

⑦ Place the controller
in actual system &
carry out experiment

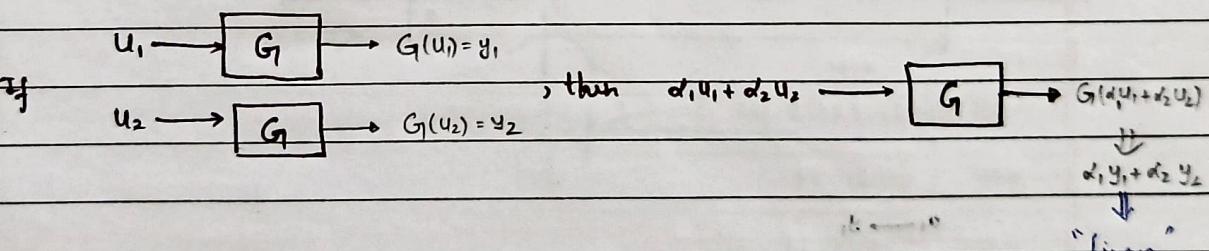


Lecture - 2

Feedback

→ Open Loop Vs Closed Loop
 ↓
 Absence of Feedback Presence of Feedback

→ Linear System Vs Non Linear System



Ex: ① $y = 3u + 5$ (Non linear)

② $y = 3u^2 + u$ (Non linear)

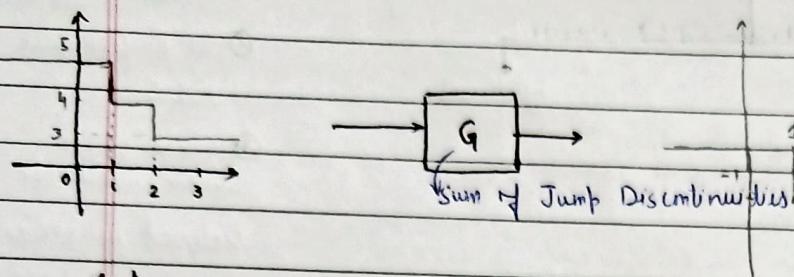
\bullet

③ $u = \begin{bmatrix} ? \\ ? \end{bmatrix}$ or $\begin{bmatrix} ? \\ ? \end{bmatrix}$ \Rightarrow Non linear \rightarrow Domain is punctured

$y = su$

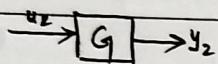
④ $y = 7u$ $u = \{0, 2, 4, 6, \dots\} \Rightarrow$ Non linear \rightarrow Domain is punctured

EE302

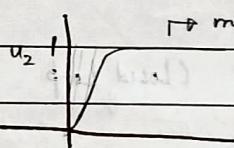
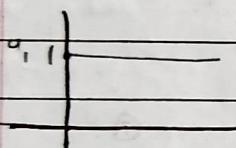


lecture - 2

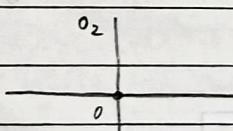
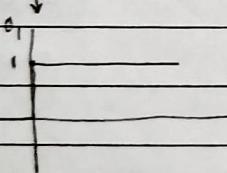
#



- A useful property is that as u_1 and u_2 come close to each other, y_1 and y_2 also approach each other.
- Mostly linear function has this property. but the above example violates this



\rightarrow making this steeper, still we don't approach 0.



$$u_1 \rightarrow y_1$$

$$u_2 \rightarrow y_2$$

$$u_k \rightarrow y_k$$

$$\hat{u} = \sum_{i=1}^{\infty} a_i u_i \rightarrow G(\hat{u}) = \sum_{i=1}^{\infty} a_i y_i$$

\downarrow and G is smooth
Only when $\hat{u} = \sum_{i=1}^{\infty} a_i u_i$ converges

- Smoothness is important; mostly later for gears etc.

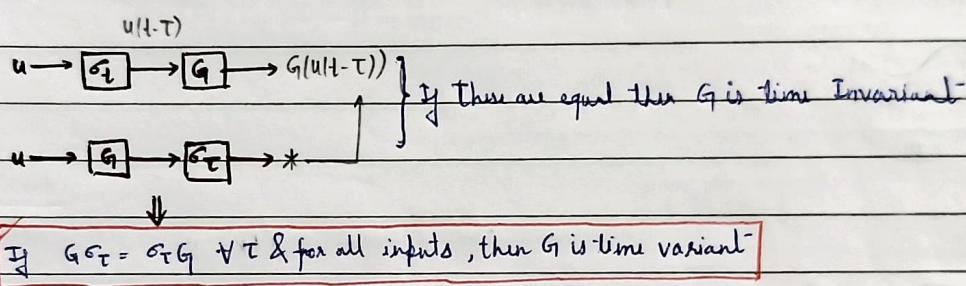
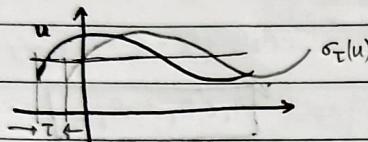
* Static Vs Dynamic → Example: $y(t) = u(\cos(t))$

- Memoryless System and Causal
- Output depends on input at that time.

Example: $V(t) = R J(t)$

* Time Invariant Vs Time Variant

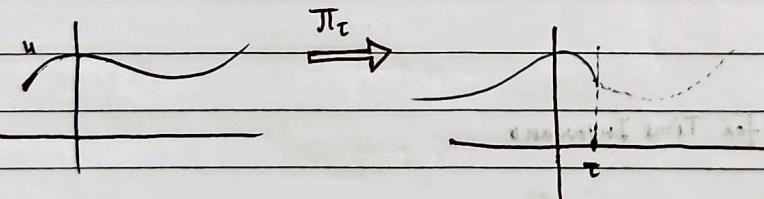
Shift Operator = σ_T



* Causal Vs Non-Causal

$$\pi_t G \pi_t = \pi_t G$$

π_t = Truncation operator



Suppose

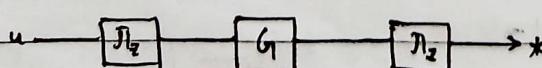
$\pi_t G \pi_t = \pi_t G \pi_t \forall t$ and for all inputs then it is called causal

π_t



⇒ Whether you truncate the input

or not, it doesn't matter



Check for Left Invertibility and Right Invertibility $\Rightarrow \sigma_c$ is right if left is invertible
 ✓ ROC and Causality.
 ↓
 If ROC has ∞ then causal

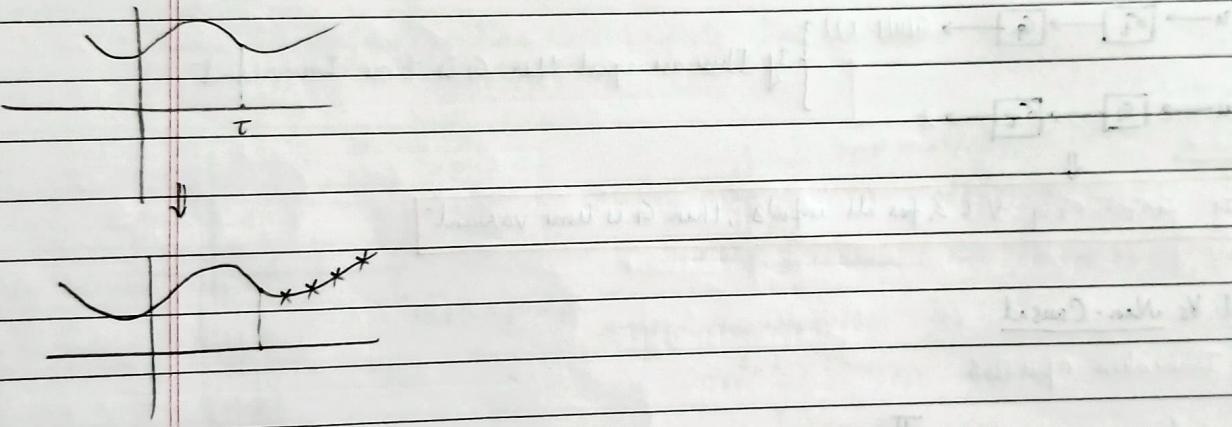
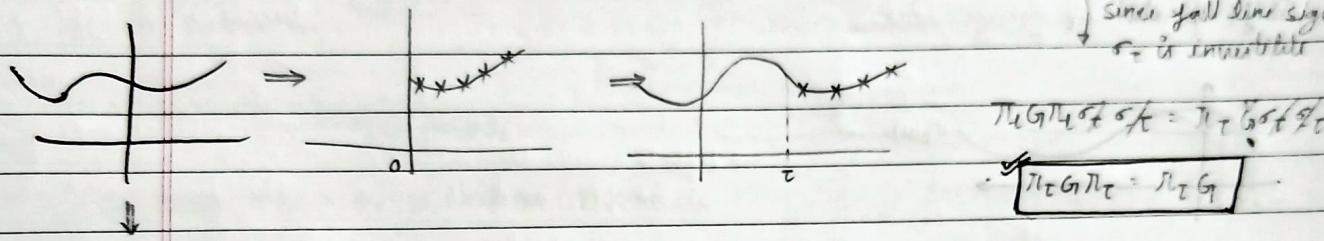
- Suppose; the system is time-invariant: We can change check for finite T , and infer about causality.



Let π_0 be the sum for $T=0$ i.e. $\pi_0 G \pi_0 = \pi_0 G$
 ↓
 (W.L.G.)

$$\sigma_c \pi_0 G \pi_0 = \sigma_c \pi_0 G = \pi_0 \sigma_c G \pi_0 = \pi_0 \sigma_c \pi_0 G \quad \forall T$$

$$\text{Claim: } \sigma_c \pi_0 = \pi_0 \sigma_c = \pi_0 G \sigma_c \pi_0 = \pi_T G \sigma_c \pi_0 = \pi_T G \pi_0 \sigma_c = \pi_T G \sigma_c \quad \forall T$$



Exercise: - $= u'(t)$

1) $y(t) = u(3t)$ Check for Time Invariance

$$y_1(t) = \int_{-\infty}^t \sqrt{u(3t)} dt$$

$$y_2(t) = \int_{-3}^{t/3} u^2(t) dt$$

① $u(t-\tau)$

$$y_1(t) = u(3t-3\tau)$$

$$y_2(t) = u(3t-\tau)$$

② $u(t-\tau)$

$$y_1(t) = \int_{-\infty}^t u(3(\alpha-\tau)) d\alpha$$

$$= \int_{t-\tau}^t u(\alpha) d\alpha$$

$$y_2(t) = y(t-\tau) = \int_0^{t-\tau} u(\alpha) d\alpha$$

Not time invariant

Lecture - 3

Discrete time Vs Continuous time# Deterministic Vs Stochastic \Rightarrow Some Randomness is there

- No uncertainty/Randomness in output
- Everytime we give some inputs we get the same output

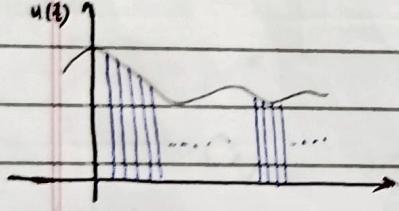
$\ddot{y} + a_1 \dot{y} + a_2 y + a_3 \dot{y} = b_0 \ddot{u} + b_1 \dot{u} + b_2 u + b_3 \dot{u} \rightarrow$ Order of derivatives of ($u \leq y$)

$$\dot{u} = \lim_{\delta \rightarrow 0} \frac{u(t+\delta) - u(t-\delta)}{\delta}$$

↓
Linear (Exercise part of)

$$\ddot{u} = \lim_{\delta \rightarrow 0} \frac{u(t) - u(t-\delta)}{\delta} = \frac{u(t-\delta) - u(t-2\delta)}{\delta} = \frac{u(t) - 2u(t-\delta) + u(t-2\delta)}{\delta^2}$$

$$\dddot{u} = \lim_{\delta \rightarrow 0} \frac{u(t) - 3u(t-\delta) + 3u(t-2\delta) - u(t-3\delta)}{\delta^2}$$

 $u(t)$ 

$$\text{Defines: } \bar{u}(t) = \sum_{k=0}^{\infty} u(t_k) \cdot \delta \alpha(t_k) \cdot \Delta$$

$$\bar{u}(t) = \sum_{k=0}^{\infty} u(t_k) \cdot (\Delta(t-t_k) - \Delta(t-t_k-\Delta))$$

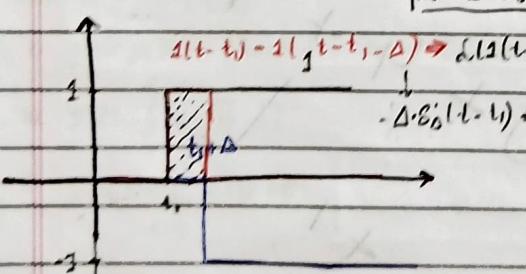
$$y(t) = g(u(t))$$

$$\hat{y}(t) = G \left(\sum_{k=0}^{\infty} u(t_k) \delta \alpha(t-t_k) \Delta \right)$$

$$= \sum_{k=0}^{\infty} G(\Delta(t-t_k)) \cdot u(t_k) \cdot \Delta$$

$$[\text{as } \Delta \rightarrow 0]: \quad = \int G(\Delta(t-\tau)) u(t) d\tau = y(t)$$

$\Delta(t-t_1) = \Delta(t-t_1-\Delta) \Rightarrow \Delta(t(t-t_1) - \Delta(t-t_1-\Delta)) = \frac{1}{5} (e^{-t} - e^{-(t+0.5)})$



$g(t-t) : g(t) = \text{Impulse response}$

$$y(t) = \int g(\Delta(t-\tau)) \cdot u(t) d\tau$$

o/p of the system to a impulse applied at t

Time Invariance is implicit

Dirac Delta

Not a function, but a generalized Distribution

$$\int f(t) \cdot \delta(t-t_1) dt = f(t_1); \text{ Shifting Property}$$

$$\delta(t-t_1) \xrightarrow{\Delta \rightarrow 0} \Delta(t-t_1)$$

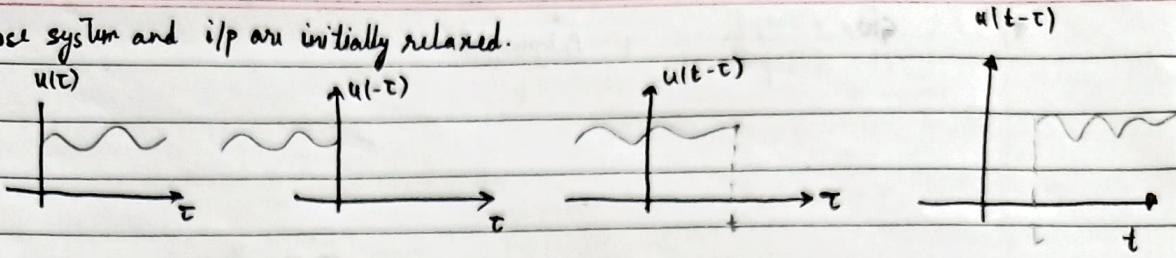
$$\# \delta(g(t)) = g(s) = \int g(t) \cdot e^{-st} dt$$

"Transform" \leftarrow

$$y(t) = \int g(t-\tau) \cdot u(\tau) d\tau$$

$$= \int u(t-\tau) g(\tau) d\tau$$

- Suppose system and i/p are initially relaxed.



$$\begin{aligned}
 Y(s) &= \mathcal{L} \left[\int_0^{\infty} g(\tau) \cdot u(t-\tau) d\tau \right] = \int_0^{\infty} \int_0^{\infty} g(\tau) \cdot u(t-\tau) \cdot e^{-st} d\tau dt \\
 &= \int_0^{\infty} g(\tau) \cdot \left[\int_0^{\infty} u(t-\tau) \cdot e^{-st} dt \right] d\tau = \int_0^{\infty} g(\tau) \cdot e^{-s\tau} \cdot \left[\int_{\tau}^{\infty} u(t-\tau) \cdot e^{-s(t-\tau)} dt \right] d\tau \\
 &= \int_0^{\infty} g(\tau) \cdot U(s) \cdot e^{-s\tau} d\tau = G(s) \cdot U(s) = \underline{\mathcal{L}(g(t)) \cdot \mathcal{L}(u(t))} \\
 &\quad \text{Transfer Function}
 \end{aligned}$$

$$\int_0^{\infty} \frac{d}{dt} (f(t)) e^{-st} dt = SF(s) - f(0^-)$$

$$\int_0^t \frac{d}{dt} (f(t)) e^{-st} dt + \int_t^{\infty} \frac{d}{dt} (f(t)) e^{-st} dt = f(t) - f(t^-) + \int_t^{\infty} \frac{d}{dt} (f(t)) e^{-st} dt = SF(s) - f(t^-)$$

$$\boxed{\lim_{s \rightarrow \infty} SF(s) = f(0^+)}$$

$$\# \dot{x}(t) + x(t) = S(t) \quad x(0^-) = 1, \text{ Obtain } x(t) \text{ for } t \geq 0$$

① Let us consider, $x(0^+)$

$$\dot{x}(t) + x(t) = 0 \Rightarrow x(t) = e^{-t} x(0^+) = 2 e^{-t}$$

$x(t)$ should produce a δ , hence it has a unit jump discontinuity

$$\downarrow$$

$$x(0^+) - x(0^-) = 1$$

$$x(0^+) = 2$$

$$\textcircled{2} \int_0^{\infty} f(t) e^{-st} dt = F(s) \quad \text{Laplace Transform}$$

$$\delta(S(t)) = 1$$

$$\delta(S(t)) = 0$$

Taking the unilateral Laplace Transform

$$S X(s) - x(0^+) + X(s) = 0$$

which we inferred from the eq in time domain; suffices the same pitfalls as in classical method.

③ Taking LT;

$$S X(s) - x(0^+) + X(s) = 1$$

$$X(s) = \frac{2}{s+1}$$

$$\delta'(X(s)) - \mathcal{L}^{-1}\left(\frac{2}{s+1}\right) = 2e^{-t} \cdot J(t)$$

Note: Soln of a linear ODE

This is different to Forced + Zero

CLASSMATE

2. Statis Soln + Z. Input Soln

Polynomial

Rational Functions: $\frac{N(s)}{D(s)}$, deg(N)=m, deg(D)=n

$n \geq m$ = proper

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$m > n$ = improper

$n > m$ = strictly proper

$$G(s) = \frac{\lambda(10upol)}{\lambda(7upol)}$$

Polynomial

$$\ddot{y} + a_2\dot{y} + a_1y + a_0y = b_1\dot{u} + b_0u, \quad \ddot{y}(0) = 0$$

$$y(0) = p$$

$$\textcircled{1} \quad G(s) = \frac{L(u)}{L(I)} \text{ with all IV set to 0}$$

} Zero State
Solutⁿ

$$y(0) = d$$

$$G(s) = \frac{b_1s + b_0}{s^3 + a_2s^2 + a_1s + a_0}$$

Both natural and forced modes included.

\textcircled{2} Obtain U(s)

$$\textcircled{3} \quad L^{-1}(G(s) \cdot U(s)) = y_{Z. \text{ statis}}(t)$$

\textcircled{4} Set $u(t) = 0$

Find the Zero Input Solutⁿ

\textcircled{5} Solutⁿ = Z. Input Solutⁿ + Z. Statis Solutⁿ

Lecture 4

Transfer Functions

Solutⁿ divided as Z. Statis Solutⁿ + Z. Input Solutⁿ

$$G(s) = \frac{N(s)}{D(s)} ; \quad \text{Zeros} = \{z \in \mathbb{C} : N(z)=0\}$$

$$\text{Poles} = \{p \in \mathbb{C} : D(p)=0\}$$

$$\ddot{y} + 3\dot{y} + 2y = u \quad ; \quad y(0) = 0$$

$$\dot{y}(0) = -5$$

$$\Rightarrow u(t) = 10 \cdot e^{-3t} \mathbf{1}(t)$$

$$s^2 y(s) - s y(0) - \dot{y}(0) + 3(s y(s) - y(0)) + 2y(s) = \frac{10s}{s+3} \text{ initial}$$

$$y(s)(s^2 + 3s + 2) + 5 = \frac{10s}{s+3}$$

From isolated State Response

$$y(s)(s^2 + 3s + 2) = \frac{10s}{s+3} - 5 = \frac{10s}{(s+3)(s+2)(s+1)} - \frac{5}{(s+2)(s+1)}$$

State
Z. Input Solutⁿ

Input
Z. Statis Solutⁿ

From Input

$$\frac{C}{s+1} + \frac{D}{s+2} + \frac{E}{s+3} - \left(\frac{A}{s+2} + \frac{B}{s+1} \right) = \frac{-5}{s+1} + \frac{20}{s+2} - \frac{15}{s+3} + \frac{5}{s+2} - \frac{5}{s+1}$$

$$E = \frac{10x-3}{1x+1} = -15$$

$$A = 5 = \frac{25}{s+2} - \frac{10}{s+1} - \frac{15}{s+3}$$

$$f1 \times 2$$

$$C = \frac{10x+1}{2x+1} = -5$$

$$B = -5 \quad g(t) = (2.5e^{-2t} - 10e^{-t} - 15e^{-3t}) \mathbf{1}(t)$$

$$D = \frac{10x+2}{1x+1} = 20$$

Natural Response found

otherwise

$$\frac{bs + \gamma}{(s+a)^2 + b^2} = \frac{bs + \gamma}{(s+a+jb)(s+a-jb)} = \frac{k_1 + jk_2}{(s+a+jb)} + \frac{k_3 + jk_4}{(s+a-jb)}$$

$$bs + \gamma = (k_1 + jk_2)(s+a-jb) + (k_3 + jk_4)(s+a+jb)$$

$$k_2 + k_4 = 0$$

$$k_2 = -k_4, b = k_1 + k_3$$

$$\gamma =$$

$-a+jb$
 $-a-jb$

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$$y(t) = \int_0^t g(\tau) u(t-\tau) d\tau = \int_0^\infty g(t-\tau) u(\tau) d\tau \quad (\text{What if } u(\tau) = e^{x\tau})$$

$$= \int_0^\infty g(t-\tau) \cdot e^{x\tau} d\tau \text{ or } \int_0^\infty g(\tau) \cdot e^{x(t-\tau)} d\tau = e^{xt} \cdot \int_0^\infty g(\tau) \cdot e^{-x\tau} d\tau$$

$$y(t) = G(x) e^{xt}$$

$$\ddot{y} + a_2 \ddot{y} + a_1 \dot{y} + a_0 y = u + \beta u, u = e^{xt}$$

$$(x^3 + a_2 x^2 + a_1 x + a_0) G(x) = (x + \beta)$$

$G(x) = \frac{x + \beta}{x^3 + a_2 x^2 + a_1 x + a_0}$
--

$$\frac{2se^{-xs} + 3se^{-2s} + s}{(s+1)(s+2)(s+3)(s+4)} = g(s) = e^{-xs} \left[\frac{A}{s+1} + \frac{B}{s+2} + \frac{C}{s+3} + \frac{D}{s+4} \right] + e^{-2s} \left[\dots \right] + 1 \left[\dots \right]$$

$$e^{-xs} \left[\frac{A}{(s+1)(s+2)(s+3)} \right]$$

Q $\frac{d^2y}{dt^2} + 12 \frac{dy}{dt} + 32y = 32u(t)$, $y(0) = 0$ $\dot{y}(0) = 0$

$$Y(s) - s^2 + 12sY(s) - 32Y(s) = \frac{32}{s}$$

$$Y(s) = \frac{32}{s(s^2 + 12s + 32)} = \frac{32}{s(s+4)(s+8)} = \frac{A}{s} + \frac{B}{s+4} + \frac{C}{s+8}$$

$$A = 1$$

$$\frac{1}{s} - \frac{2}{s+4} + \frac{1}{s+8}$$

$$B = -2$$

$$C = 1$$

$$y(t) = u(t) - 2e^{-4t}u(t) + e^{-8t}u(t)$$

$$= u(t)(1 - 2e^{-4t} + e^{-8t})$$

Q $F(s) = \frac{10}{s(s+2)(s+3)^2} = \frac{A}{s} + \frac{B}{s+2} + \frac{C}{(s+2)^2} + \frac{D}{(s+3)} = \frac{5}{9s} - \frac{5}{s+2} + \frac{40}{9(s+3)} + \frac{10}{3(s+3)^2}$

$$\frac{10}{s^2 + 2s} = C + D(s+3) \Rightarrow C = \frac{10}{9-6} = \frac{10}{3} = \left(\frac{5}{9} - 5e^{-2t} + \frac{40}{9}e^{-3t} + \frac{10}{3}e^{-6t}\right)u(t)$$

$$\frac{-10}{(s^2 + 2s)^2} \times (2s+2) = D = \frac{-10 \times -4}{9} = \frac{40}{9}$$

$$\frac{10}{9} = A = \frac{5}{9}$$

$$B = \frac{10}{-2 \times 1} = -5$$

Lecture 5

$y(s) = G(s) \cdot u(s)$

$$y(t) = L^{-1}[G(s) \cdot u(s)]$$

If we assume $u(s)$ to be proper \Rightarrow It doesn't have impulse; bounded

① Different roots = e^{pt}

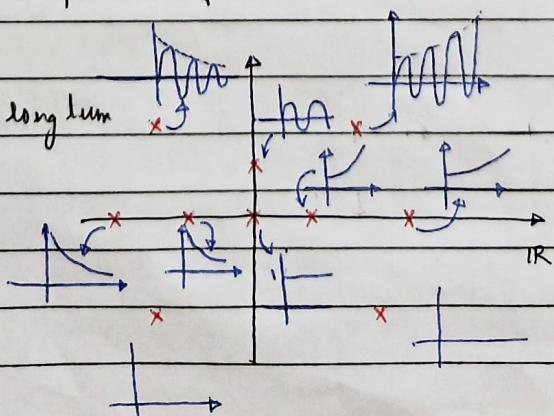
② Repeated roots = $\frac{t^{k-1}}{k!} e^{pt}$

Dependence of response upon roots of $D(s)$

• $G(s) \cdot u(s) = \frac{N(s)}{D(s)}$

• Due to repeated roots, long term response is unaffected

except when it lies on Im-axis . Then it can blow up.



Analysis of Laplace Transforms of Electrical Circuits

• Resistance

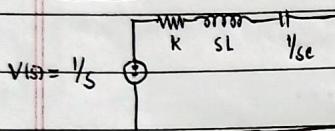
$$\text{---} \xrightarrow{R} \text{---} = V = RI \Rightarrow V(s) = I(s) \cdot R$$

• Inductor

$$\text{---} \xrightarrow{L} \text{---} = V = L \frac{di}{dt} \Rightarrow V(s) = L \cdot s I(s)$$

• Capacitor

$$\text{---} \parallel \text{---} = i = C \frac{dv}{dt} \Rightarrow I(s) = C s V(s)$$

Example:

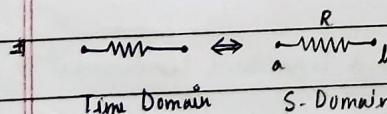
$$(R + sL + \frac{1}{sC}) I(s) = V(s) = \frac{1}{s}$$

$$I(s) = \frac{1}{RS + S^2L + \frac{1}{C}} = \frac{1/L}{s^2 + \frac{RL}{L} + \frac{1}{LC}} = \frac{1/L}{(s^2 + \frac{RL}{L}s + \frac{R^2}{4L^2}) + (\frac{1}{LC} - \frac{R^2}{4L^2})} = \frac{\sqrt{\frac{1}{LC} - \frac{R^2}{4L^2}}}{s^2 + \frac{RL}{L}s + \frac{R^2}{4L^2}}$$

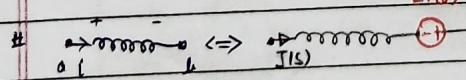
$$I(s) = \frac{\sqrt{\frac{1}{LC} - \frac{R^2}{4L^2}}}{\left[(s^2 + \frac{RL}{L}s + \frac{R^2}{4L^2}) + \left(\frac{1}{LC} - \frac{R^2}{4L^2} \right) \right] \cdot \left(\sqrt{\frac{1}{LC} - \frac{R^2}{4L^2}} \right)}$$

$$\text{Assume: } \frac{1}{LC} - \frac{R^2}{4L^2} \gg 0$$

$$I(t) = \frac{e^{-\frac{Rt}{2L}}}{(\frac{L}{C} - \frac{R^2}{4L^2})^{1/2}} \cdot \sin\left(\sqrt{\frac{1}{LC} - \frac{R^2}{4L^2}} t\right)$$

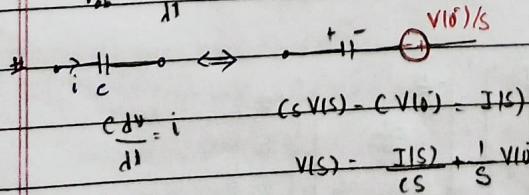
 $L(s)$

$$V(s) = t \left(L \frac{di}{dt} \right) \Rightarrow L(s) I(s) - I(0^-)$$



$$V_{ab} = L \frac{di}{dt}$$

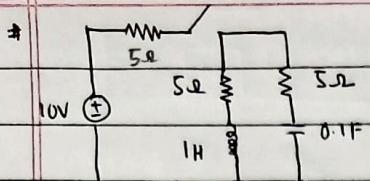
$$V_{ab}(s) = sL I(s) - L I(0^-)$$

 $V(0^-)/s$ 

$$C \frac{dv}{dt} = i$$

$$C(s)V(s) = C V(0^-) - I(0^-)$$

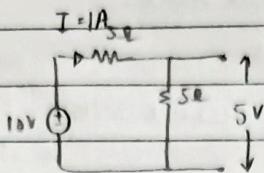
$$V(s) = \frac{I(s)}{Cs} + \frac{V(0^-)}{s}$$



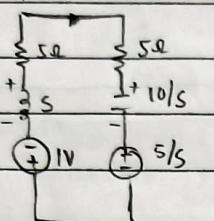
Before the current being thrown off,

$$V_r(f) = 5V$$

$$I_r(f) = 1A$$



A $I(s)$

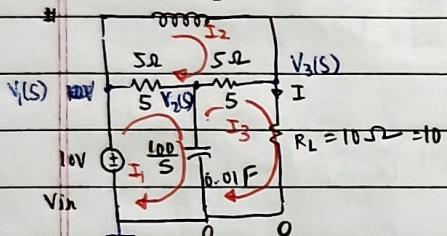


$$\frac{1}{5} - I(s) \times 5 - I(s) \times \frac{10}{s} - \frac{5}{s} - 1 - sI(s) - 5sI(s) = \frac{1}{5}$$

$$I(s) \left[10 + \frac{10}{s} + s \right] = 1 + \frac{5}{s}$$

$$I(s) = \frac{1 + \frac{5}{s}}{10 + \frac{10}{s} + s} = \frac{s+5}{s^2 + 10s + 10}$$

1H S



T. fn between $V_{in}(s)$ & $I_{RL}(s)$ ✓

By KVL

$$\begin{array}{l} \text{B} \\ \text{①} \end{array} \quad \begin{array}{c} \boxed{V_1(s)} \\ 0 \\ 0 \end{array} = \begin{bmatrix} \frac{100}{s} & -5 & -100 \\ -5 & 10+s & -s \\ -100 & -s & \frac{100}{s} \end{bmatrix} \begin{array}{c} I_1(s) \\ I_2(s) \\ I_3(s) \end{array} \quad I_3 = \text{det} \begin{bmatrix} \frac{100}{s} & 5 \\ -s & \frac{100}{s} \end{bmatrix}$$

- Diagonal elements are sum of all impedances
- Other element have - (common impedances)

↓

Transfer function is not implement.

W.W

② By KCL

$$V_1(s) = V_{in}(s) = 10$$

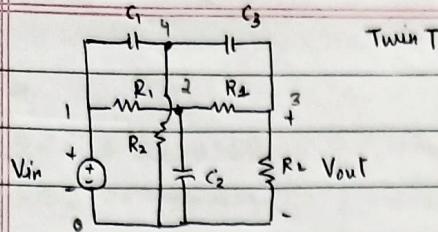
$$I_{RL} = V_3(s)/R_L$$

$$\frac{V_2(s) - 0}{5} + \frac{V_2 - V_1}{5} + \frac{V_2(s) - V_3(s)}{5} = 0$$

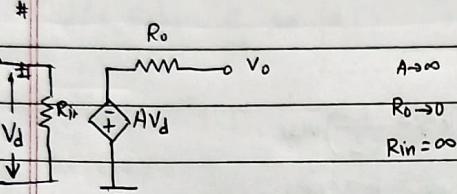
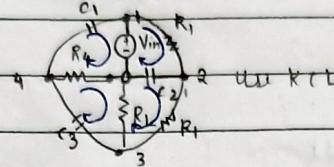
$$\begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} -\frac{1}{5} \\ \frac{1}{5} \\ -\frac{1}{5} \end{bmatrix} \begin{bmatrix} V_2 \\ V_2 \\ V_3 \end{bmatrix}$$

$$\frac{V_3(s) - V_1(s)}{s} + \frac{V_3(s) - V_2(s)}{s} = I_{RL}$$

$$\frac{1}{100} + \frac{1}{5} \times 20$$

Lecture-6

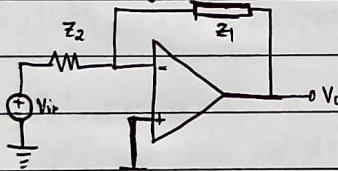
Twin T



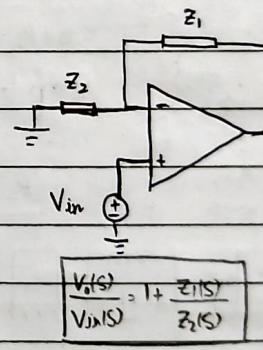
$$V_0 = A V_d = A(V_+ - V_-)$$

$$V_+ - V_- = \frac{V_0}{A} = 0 \text{ (as } A \rightarrow \infty\text{)}$$

$V_+ = V_-$ [Virtual Short]

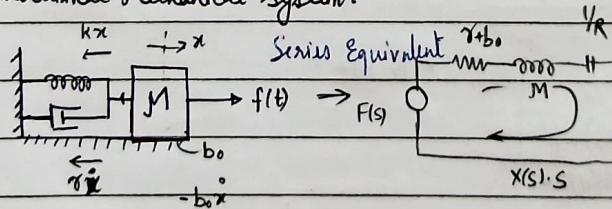
Inverting Mode

$$\frac{V_0(s)}{V_{in}(s)} = -\frac{Z_1(s)}{Z_2(s)}$$

Non Inverting Mode

$$\frac{V_0(s)}{V_{in}(s)} = 1 + \frac{Z_1(s)}{Z_2(s)}$$

Translational Mechanical System:-



$$f(t) = \gamma \ddot{x} - kx = m\ddot{x}$$

$$f(t) = m\ddot{x} + kx + \gamma \dot{x} + b_0 \dot{x} = m\ddot{x} + (b_0 + \gamma) \dot{x} + kx$$

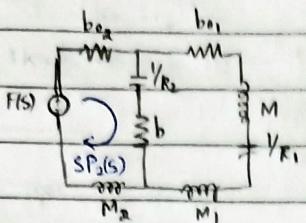
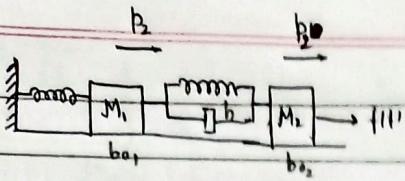
$$F(s) = (Ms^2 + (b_0 + \gamma)s + k)X(s)$$

$$F(s) = X(s) \cdot [Ms + (b_0 + \gamma) + \frac{k}{s}]$$

$$V(s) \leftrightarrow F(s)$$

$$V(s) = I(s) \cdot [sL + R + \frac{1}{Cs}] \Rightarrow$$

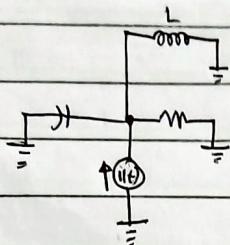
Notes: For a mech system we can always get a elec system, Consider it not true when Inductor is present



Parallel Analysis: Provides the physical insight

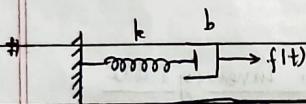
$$I(s) = V(s) \left[\frac{1}{R} + \frac{1}{sL} + sC \right]$$

$$F(s) = SP_1(s) \left(b + MS - \frac{k}{s} \right)$$



$$I(s) = \left(\frac{1}{R} + sL + \frac{1}{sM} \right) V(s)$$

$$F(s) = \left(b + MS + \frac{k}{s} \right) SP_1(s)$$

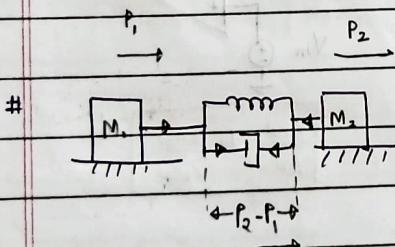


$$F(s) = bs P_2(s)$$

$$P_1 + P_2 = P(s)$$

$$F(s) = k R(s)$$

$$F(s) \left[\frac{1}{k} + \frac{1}{bs} \right]$$



$$M_2 \ddot{P}_2 = -k(P_2 - P_1) - b(\dot{P}_2 - \dot{P}_1)$$

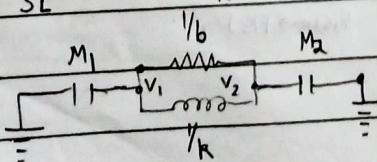
$$M_1 \ddot{P}_1 = k(P_2 - P_1) + b$$

$$s^2 M_2 P_2(s) + k(P_2(s) - P_1(s)) + bs(P_2(s) - P_1(s)) = 0$$

$$s^2 M_1 P_1(s) - k(P_2(s) - P_1(s)) - bs(P_2(s) - P_1(s)) = 0$$

$$(sP_2(s))SM_2 + \frac{k}{s}(sP_2(s) - P_1(s)) + b(s)(P_2(s) - P_1(s))$$

$$V_2(s)SC_2 + \frac{1}{sL}(V_2 - V_1) + \frac{1}{R}(V_2 - V_1)$$

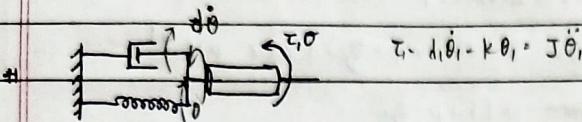


Lecture-7# Rotational Mechanical System

- Inertia  $J\ddot{\theta} = \tau$, equiv. to mass

-  $k\dot{\theta} = \tau$

-  $k\theta = \tau$



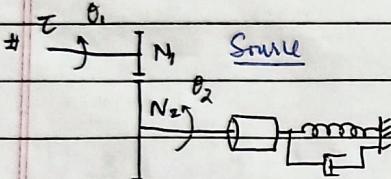
$$\tau(s) = (Js^2 + ds + k) \theta(s)$$

$$= (s^2J + s(d + \frac{k}{s})) \theta(s) = (s^2J + s(\frac{d}{s} + \frac{k}{s})) \theta(s) = \tau(s)$$

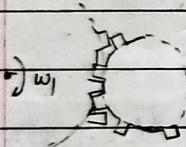
$$(sL + R + \frac{1}{Cs}) I(s) = V(s) \text{ (Series Equivalent)}$$

$$(\frac{1}{sL} + \frac{1}{R}, s) V(s) = I(s) \text{ (Parallel Equivalent)}$$

Distortion



→ This is Analogous to a transform



$$\left. \begin{aligned} N_1 &= \frac{2\pi r_1}{2c} \\ N_2 &= \frac{2\pi r_2}{2c} \end{aligned} \right\} \boxed{\frac{N_1}{N_2} = \frac{r_1}{r_2} = \frac{\omega_2}{\omega_1} = \frac{\tau_1}{\tau_2}}$$

① Tooth width = c

② $\omega_1 \tau_1 = \omega_2 \tau_2$

③ $\tau_1 \omega_1 = \tau_2 \omega_2$

$$\begin{aligned} \tau &= \frac{N_2 \tau}{N_1} \tau \\ \theta' &= \frac{N_1}{N_2} \theta \end{aligned}$$

$$\frac{N_2}{N_1} \tau_1 - k \frac{N_1}{N_2} \theta_1 - d \dot{\theta}_1 \frac{N_1}{N_2} = J_1 \ddot{\theta}_1 \frac{N_1}{N_2}$$

$$\left(\frac{N_2}{N_1} \right)^2 \tau(s) = \left(\frac{k}{s} + d + s J_1 \right) \theta(s)$$

$$\tau(s) = \left[J_1 \left(\frac{N_1}{N_2} \right)^2 s^2 + d \left(\frac{N_1}{N_2} \right)^2 s + k \left(\frac{N_1}{N_2} \right)^2 \right] \theta(s)$$

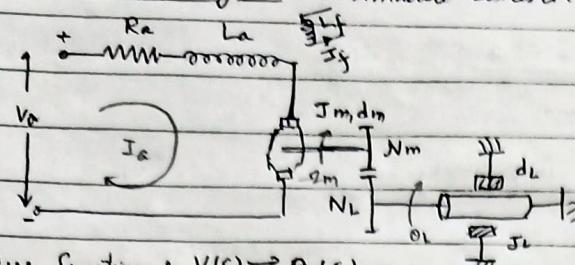
$\left(\frac{D_{\text{oth}}}{S_{\text{oth}}} \right)^2$

$$\frac{N_L}{N_m} = \frac{T_L}{T_m}$$

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Electromechanical System (Armature Control of DC Motors)



$$\frac{\theta_m}{\theta_L} = \frac{N_L}{N_m}$$

Transfer function: $V_a(s) \rightarrow \theta_L(s)$

$$V_a(s) = (R_a + sL_a) I_a(s) + V_b(s) \quad \text{Back EMF : Electrical Part} \quad V_a(s) = (R_a + sL_a) \frac{T_m(s)}{K_b \cdot s \cdot \theta_L(s) \cdot N_L} \frac{1}{J_L}$$

$$V_b(s) = K_b \cdot \theta_m(s)$$

$$J_L + J_m \left(\frac{N_L}{N_m} \right)^2 = J_{eq} V$$

$$d_L + d_m \left(\frac{N_L}{N_m} \right)^2 = J_{eq} V$$

$$\left[\left(J_L + J_m \left(\frac{N_L}{N_m} \right)^2 \right) s^2 + \left(d_L + d_m \left(\frac{N_L}{N_m} \right)^2 \right) s \right] \theta_L(s) = T_L(s) \quad \text{: Mech' Part}$$

$$T_m(s) = k_a \frac{I_a(s)}{R_a} \quad \text{Electromagnetic Part}$$

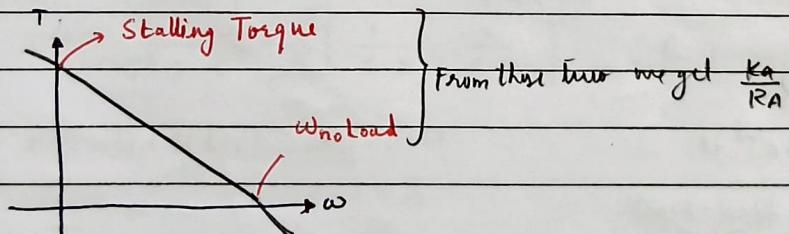
$$V_a(s) = \frac{N_m}{N_L} \frac{(R_a + sL_a)}{K_a} \left(J_{eq} s^2 + J_{eq} s \right) \theta_L(s) + K_b s \cdot \theta_L(s) \frac{N_L}{N_m}$$

$$\frac{\theta_L(s)}{V_a(s)} = \frac{1}{\left(\frac{N_m}{N_L} \frac{(R_a + sL_a)}{K_a} \left(J_{eq} s^2 + J_{eq} s \right) + \frac{K_b N_L}{N_m} s \right)} = \frac{\left(\frac{R_a N_m}{K_a N_L} \left(1 + \frac{sL_a}{R_a} \right) \left(J_{eq} s + J_{eq} \right) + \frac{K_b N_L}{N_m} s \right)}{N_m J_{eq} R_a}$$

Generally
 $R_a \gg L_a s$

$$s \left(s + \frac{J_{eq}}{J_{eq} R_a} + \frac{\frac{K_b N_L}{N_m}}{J_{eq} R_a} \right)$$

$$D(s) = K_b w(s) + R_a \frac{T_m(s)}{K_a}$$



Lecture-8

Step Response

↓
 Transient Response

- ① 1st Order System
 ② 2nd Order System

1st Order System

• $G(s) = \frac{1}{s+a}$ | $a > 0$

Step Respⁿ = $y(s) = \frac{G(s)}{s} = \frac{1}{(s+a)s} \Rightarrow y(t) = 1(t)[1 - e^{-at}]$

$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} s \cdot \frac{1}{s} \cdot \frac{1}{s+a} = 1$ $G(s)$ has dc gain equal to unity

• Time Constant (t_c)

$$t_c = \frac{1}{a}$$

Rise

• Settling Time $\geq 10\%$ to 90% of the Steady State value

$$\begin{aligned} (1 - e^{-at_1}) &= 0.1 \\ 1 - e^{-at_2} &= 0.9 \end{aligned} \quad \left. \begin{aligned} t_1 &= \frac{1}{a} \ln(0.9) \\ t_2 &= \frac{1}{a} \ln(0.1) \end{aligned} \right\} t_2 - t_1 = \frac{\ln 9}{a} = \frac{2.2}{a} = t_s$$

• Settling Time: when the system reaches 2% of the Steady State value.

$$1 - e^{-at_s} = 0.98$$

$$0.02 = e^{-at_s}$$

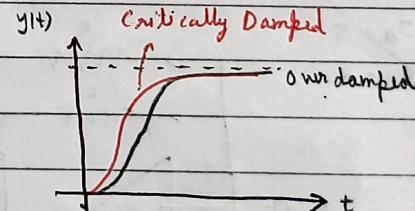
$$t_s = \frac{\ln 50}{a} \approx \frac{4}{a}$$

2nd Order System

• $G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$

ζ Damping Coeff.

Natural Frequency



① $\zeta > 1$ (Overdamped)

$$p_1, p_2 = \frac{-2\zeta\omega_n \pm \sqrt{4\zeta^2\omega_n^2 - 4\omega_n^2}}{2} = -(\zeta + \sqrt{\zeta^2 - 1})\omega_n$$

LHP

$$G(s) = \frac{p_1 p_2}{(s-p_1)(s-p_2)}$$

root closer to origin $(s-p_1)(s-p_2)$

$$\text{are dominant } \downarrow \left[\frac{1}{s} G(s) \right] = \frac{1}{s} \frac{p_1 p_2}{(s-p_1)(s-p_2)} s = \frac{1}{s} \left[\frac{p_2}{p_1 + p_2} + \frac{p_1}{s-p_2} + \frac{1}{s} \right]$$

$$\downarrow \quad y(t) = 1(t) \left[1 + \frac{1}{p_1 - p_2} [p_2 e^{p_1 t} - p_1 e^{p_2 t}] \right]$$

close the pole to the origin
 slows the response

② $S=1$ (Critically Damped)

$$d^{-1} \left[\frac{G(s)}{S} \right] = d^{-1} \left[\frac{1}{S} + \frac{-1}{S-p_1} + \frac{p}{(S-p_1)^2} \right] = 1(t) [1 - e^{\frac{p_1 t}{2}} + p_1 t e^{\frac{p_1 t}{2}}]$$

$$\frac{1}{S} \frac{p^2}{(S-p)^2} = \frac{1}{S} + \frac{B}{S-p} + \frac{C}{(S-p)^2}$$

$$\frac{p^2}{S} = S(S-p)^2 + B(S-p) + C$$

$$\delta = 1+B$$

$$B=-1$$

③ $S<1$ (Underdamped)

$$p_1, p_2 = -SW_n \pm j\omega_n \sqrt{1-\zeta^2}$$

$$|p_1| = |p_2| = \omega_n$$

$$\begin{aligned} \frac{1}{S} G(s) &= \frac{1}{S} \times \frac{\omega_n^2}{s^2 + 2SW_n s + \omega_n^2} = \frac{A}{S} + \frac{BS+C}{s^2 + 2SW_n s + \omega_n^2} = \frac{1}{S} - \frac{(s+2j\omega_n)}{(s+j\omega_n)^2 + \omega_n^2(1-\zeta^2)} \\ \boxed{A=1} \quad &= \frac{1}{S} - \frac{s+2SW_n}{(s+j\omega_n)^2 + (\omega_n\sqrt{1-\zeta^2})^2} \\ \omega_n^2 &= A(s^2 + 2SW_n s + \omega_n^2) + (BS+C)s \\ 0 &= A + B \Rightarrow B = -1 \\ 0 &= 2j\omega_n A + C \Rightarrow -2j\omega_n = C \quad \therefore (1 - e^{-j\omega_n t} \cos(\omega_n \sqrt{1-\zeta^2} t) - \frac{e^{-j\omega_n t} s}{\sqrt{1-\zeta^2}}) \end{aligned}$$

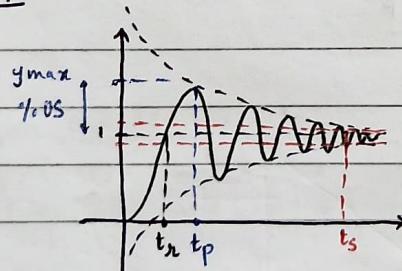
$$\tan \frac{s}{\sqrt{1-\zeta^2}} = \tan \phi$$

$$y(t) = 1(t) \left[1 - \frac{e^{-j\omega_n t}}{\sqrt{1-\zeta^2}} \cos(\omega_n t - \phi) \right]$$

$$w_d = \omega_n \sqrt{1-\zeta^2} \quad \boxed{\phi = \sin^{-1}(1/S)}$$

$$\phi > 0$$

Lecture 9



- Time Constant makes no sense here.

- Settling Time: Time after which the curve remains in the 2% error band (how soon the curve settles down)

- Rise Time: 0 to 100% for the first time

how soon does the system

react after application

of input

- $\%OS(M_p) = \frac{y_{max} - y_{ss}}{y_{ss}} \times 100$

- Peak time: Time when $y = y_{max}$

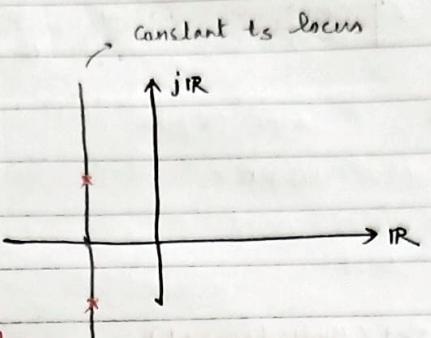
- Consider only the envelope only as an approximation
- For Settling time: is a function of real part of the poles

$$1 - \frac{e^{-\sigma w_n t_s}}{\sqrt{1-\zeta^2}} = 0.98$$

$$\frac{e^{-\sigma w_n t_s}}{\sqrt{1-\zeta^2}} = 0.02$$

$\sigma = -\zeta \omega_n$

$$t_s = \frac{1}{\zeta \omega_n} \ln \left(\frac{50}{\sqrt{1-\zeta^2}} \right) \approx \frac{4}{\zeta \omega_n}$$



For $0.1 < \zeta < 0.85$: $3.9 < t_s < 4.7$, so $\zeta \approx 0.4$

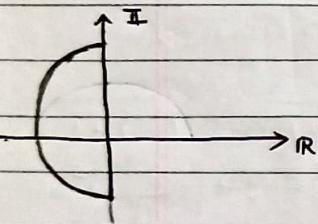
- For Rising time: related to the modulus of the poles

$$\cos(\omega_n t_r - \phi) = \cos(\pi/2)$$

$$\omega_n t_r - \phi = \pi/2$$

$$\omega_n \sqrt{1-\zeta^2} t_r = \frac{\pi}{2} + \phi$$

$$t_r = \frac{\frac{\pi}{2} + \sin^{-1}(\zeta)}{\omega_n \sqrt{1-\zeta^2}} \approx \frac{\frac{\pi}{2} + \zeta}{\omega_n \sqrt{1-\zeta^2}} \approx \frac{\pi}{2\omega_n} \approx \frac{1.6}{\omega_n}$$



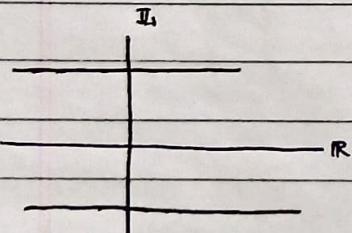
- For % of O.S (M_p):

$$y(t) = 0$$

$$d^2(G(s)) = 0$$

$$\frac{d^2}{ds^2} \left(\frac{\omega_n}{\sqrt{1-\zeta^2}} \times \frac{\omega_n \sqrt{1-\zeta^2}}{(s + j\omega_n)^2 + (\omega_n \sqrt{1-\zeta^2})^2} \right) = \frac{\omega_n}{\sqrt{1-\zeta^2}} \times e^{-j\omega_n t} \times \sin(\omega_n \sqrt{1-\zeta^2} t) = 0$$

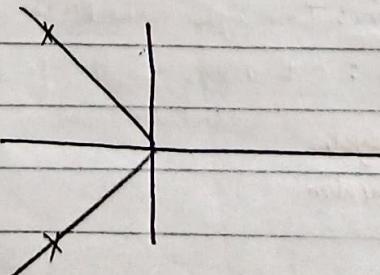
$$t_p = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}}$$



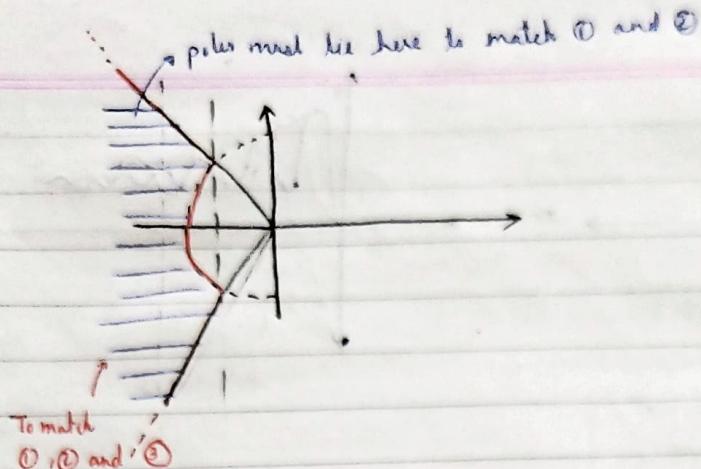
$$y_{max} = \left[1 - e^{-\frac{j\omega_n \pi}{\sqrt{1-\zeta^2}}} \times \cos(\sqrt{1-\zeta^2} \omega_n \pi - \phi) \right]$$

$$y_{max} = (1 + e^{-\frac{j\pi}{\sqrt{1-\zeta^2}}} \cos \phi)$$

$$\% \text{ O.S} = 100 e^{-\frac{j\pi}{\sqrt{1-\zeta^2}}}$$



- We have:
- ① t_0 given
 - ② γ_0
 - ③ t_s



Pade Approximation

$$P(m,n) = \frac{(s+\gamma_0)^m}{s^m} \frac{e^{-t_0}}{s^{n-m} - \gamma_0^{n-m}}$$

$$\approx e^{-\gamma_0 t}$$

$$P(1,1) = \frac{s + \gamma_0}{s + b_1} \approx e^{-\gamma_0 t}$$

$$\left(\frac{s + \gamma_0}{s + b_1}\right)\left(1 + \frac{s}{b_1}\right)^{-1} = 1 - \gamma_0 t + \frac{\gamma_0^2}{2} t^2$$

$$P(1,2) = \frac{1 - \gamma_0 t/2}{1 + \gamma_0 t/2}$$

$$P(1,0) = \frac{1}{1 + \gamma_0 t}$$

applying x RHP you have a delay effect

* Effect of Additional poles and Zeros

① $G(s) = \frac{\omega_n^2 (1 + \tau_z s)}{s^2 + 2\zeta \omega_n s + \omega_n^2}$ (Doesn't change t_0 much)
but really

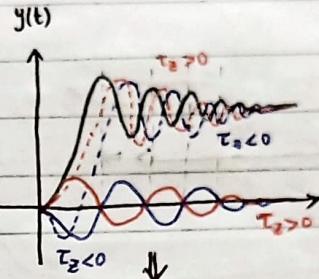
$$y(t) = y(t) + \tau_z \dot{y}(t)$$

if all
 τ_z

case I $\tau_z > 0$

move the τ_z , closer the poles and more the contribution

case II $\tau_z < 0$ we see a undershoot



② $G_2(s) = \frac{\omega_n^2}{(s^2 + 2\zeta \omega_n s + \omega_n^2)(1 + \tau_p s)}$

$$\frac{1}{s} G_2(s) = \frac{A}{s} + \frac{Bs + D}{s^2 + 2\zeta \omega_n s + \omega_n^2} + \frac{D}{s + \frac{1}{\tau_p}} \rightarrow D \cdot e^{-\frac{1}{\tau_p} t} g(t)$$

Qualitatively known

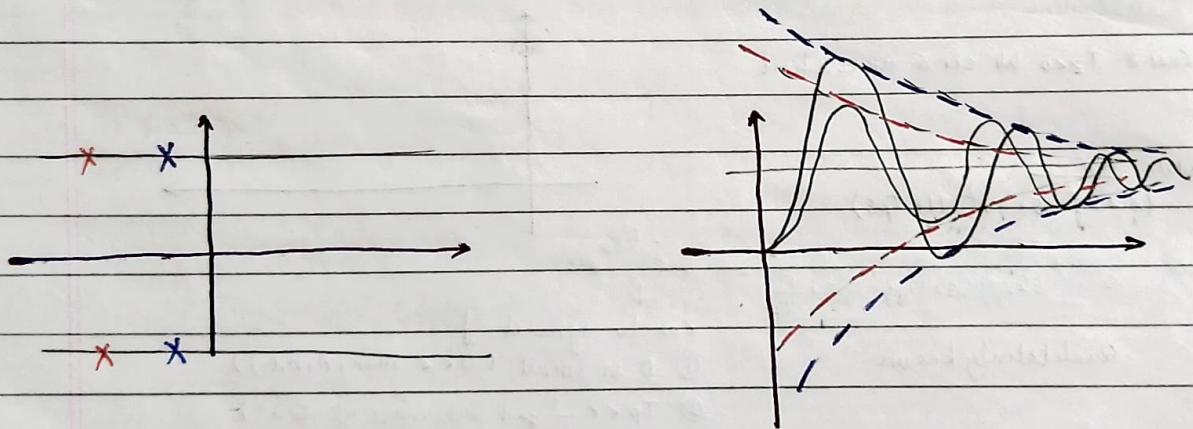
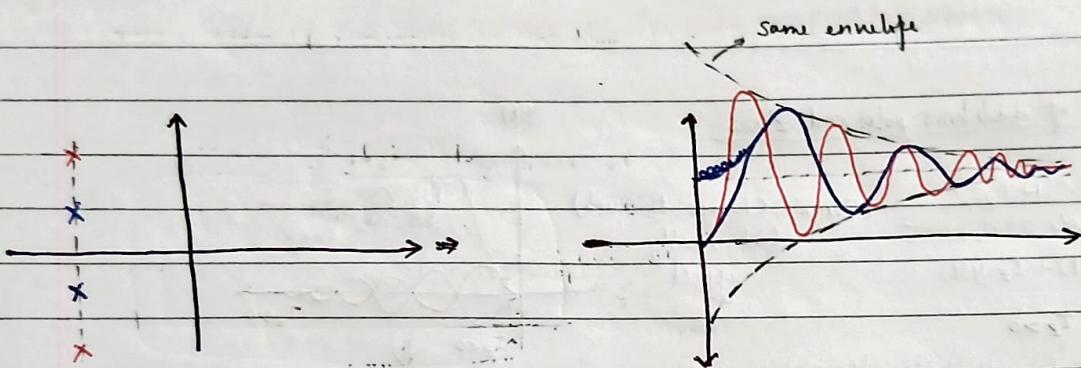
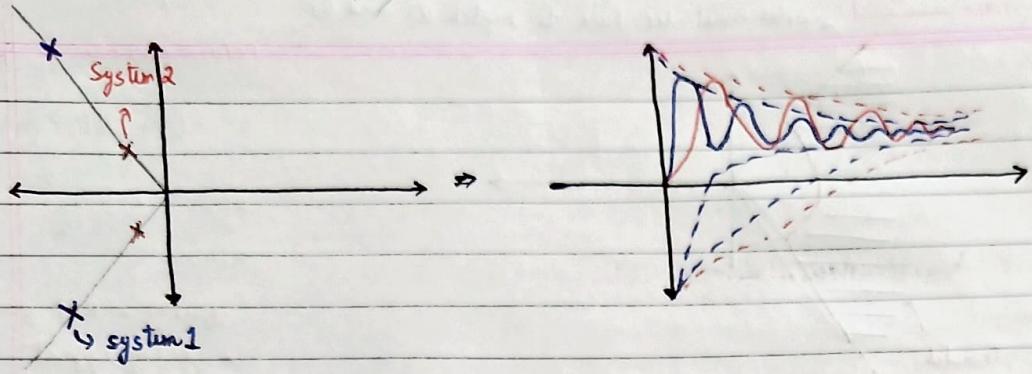
can be ignored if

① D is small ($D \ll \min(A, B, 1)$)

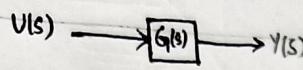
② $\tau_p \ll \frac{1}{\omega_n^2}$ (difference of 5%)

③ $G_3(s) = \frac{\omega_n^2 (1 + \tau_z s)}{(s^2 + 2\zeta \omega_n s + \omega_n^2)(1 + \tau_p s)}$

If τ_z and τ_p are close, we get a very good approximate 2nd order system

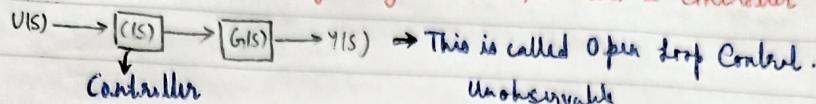


* Closed Loop Control



Plant

↓ For a badly designed Plant, we need a controller



Controller

Unobservable

$$\text{L} \ L G(s) = \frac{(s+2)}{(s+1)(s+3)(s+4)}$$

↓ don't want this pole zero
we can define $C(s)$ as $\frac{s+4}{s+2}$ to remove this.

This is a bad idea

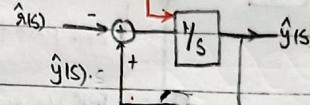
Ex: $\frac{1}{s-1} \rightarrow$ This gain input might break our system

or



uncontrollable

Zero initial input
can flow up



A zero input will blow this up.

$$\frac{\hat{u}(s) + \hat{y}(s)}{s} = \hat{y}(s)$$

$$\frac{\hat{y}(s)}{\hat{u}(s)} = \frac{1}{s-1}$$

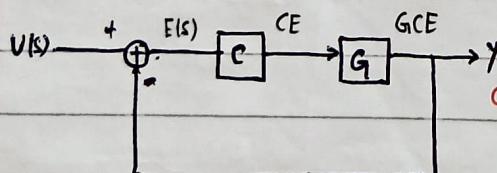
Sensitivity:

$x \Rightarrow$ Indep. Variable

$y \Rightarrow$ Dependent Variable

$$\text{Sensitivity} = \frac{\frac{\partial y}{\partial x}}{y} \approx \left(\frac{\partial y}{\partial x} \cdot \frac{x}{y} \right)$$

$$S_{\frac{y}{x} \text{ DS}} = \frac{\partial(1\% \text{ DS})}{\partial x} \cdot \frac{\frac{y}{x}}{\% \text{ DS}} = -100 \cdot e^{-\frac{P\pi}{\sqrt{1-y^2}}} \times \frac{\pi x \partial \left(\frac{y}{\sqrt{1-x^2}} \right) \times \frac{x}{100 \times e^{-\frac{P\pi}{\sqrt{1-y^2}}}}}{\sqrt{1-x^2}}$$



GC is the open-loop TF.

$$E = U - GCE$$

$$E = U - GC(U - Y)$$

$$Y + Y = Y + G(C(U - Y))$$

$$(1 + GC)Y = GCU$$

$$\frac{Y}{U} = \frac{GC}{1 + GC} \Rightarrow \boxed{\text{closed Loop Transfer Function}}$$

$$S_{G_{OL}/G_{OL}} = \frac{\partial}{\partial G_C} \left(\frac{G_C}{1 + G_C} \right) \times \frac{G_C}{G_C} (1 + G_C)$$

$$= \frac{1}{(1 + G_C)^2} = \frac{1}{1 + G_C} = S$$

$$= \frac{1}{1 + G_{OL}}$$

Is $\frac{G_{OL}}{1+G_{OL}}$ Stable?

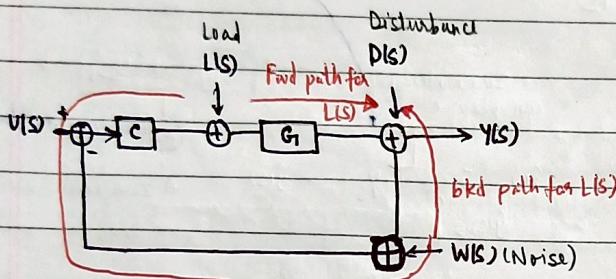
e.g.: Take G_{OL} as $k \frac{s-2}{(s+1)(s+2)}$

$$\text{Poles} \Rightarrow 1 + \frac{k(s-2)}{(s+1)(s+2)} = 0 \Rightarrow (s+2)(s+1) + k(s-2) = 0$$

can cannot be arbitrarily large

$$C(s) = \frac{s+a}{s+b} \Rightarrow |C(j\omega)| = \left| \frac{j\omega+a}{j\omega+b} \right|$$

We choose a band of frequencies, and then choose a, b for robustness



$$Y(s) = \frac{U(s) \cdot C G}{1+G_C} + \frac{G_f}{1+G_C} L(s) + \frac{1}{1+G_C} D(s) - \frac{G_f C}{1+G_C} W(s)$$

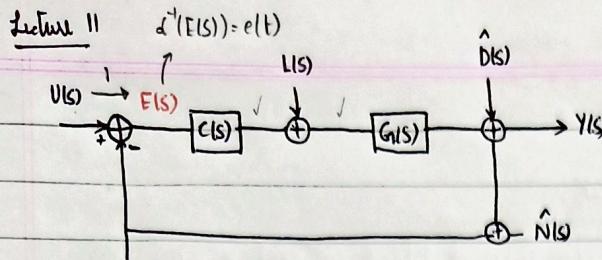
$$Y(s) = \frac{G_f w}{1+G_{fw} \cdot G_{blk}}$$

we want this to be unity

① at lower freq we don't wish this to be unity

L ↴ $C(s)$ is such ↴ At higher frequency
that it is high at lower frequencies and
low at higher frequencies

- Order: Degree of Denominator at TF
- Type of a TF: # of Poles of TF at the origin



$$G(s) = \frac{50}{(s+1)(s+2)(s+25)} = \frac{50^2}{25 \times 2 (1+s)(1+\frac{s}{2})(1+\frac{s}{25})} \approx \frac{2}{(1+s)(2+s)} \rightarrow \text{Don't do this if } G(s) \text{ is in a closed loop form}$$

Very Fast Mode; can be removed They will be much less than 25 and can be ignored

$$s = \sigma + j\omega \rightarrow \text{freq}$$

Natural frequency

$$V(s) = \underbrace{\frac{GC}{1+GC} V(s)}_{\text{DC gain}} + \underbrace{\frac{L(s) \cdot G}{1+GC}}_{j \text{ blends to 1}} + \underbrace{\frac{D(s)}{1+GC}}_{\text{Error}} - \underbrace{\frac{N(s) \cdot GC}{1+GC}}_{\text{Feedback}}$$

DC gain < 1
j blends to 1

Tracking problem

How to ensure: $\lim_{t \rightarrow \infty} u(t) - y(t) \rightarrow 0$

- If we apply a DC signal, then if we plant a integrator just in front of E(s) to get zero e(t).

$$\bullet E(s) = \frac{U(s) \cdot \frac{1}{s}}{1+GC} \quad (\text{Assuming Stable}) \quad \text{suppose } G(0) C(0) \text{ is finite}$$

3 Input:

$$\textcircled{1} \quad U(s) = \frac{1}{s}, \quad u(t) = 1(t)$$

$$\textcircled{2} \quad U(s) = \frac{1}{s^2}, \quad u(t) = t 1(t)$$

$$\textcircled{3} \quad U(s) = \frac{1}{s^3}, \quad u(t) = \frac{t^2}{2} 1(t)$$

G and C don't have a pole at origin

FVT

$$\textcircled{1} \quad \lim_{s \rightarrow 0} sE(s) = \frac{s \times \frac{1}{s}}{1+GC} = \frac{1}{1+GC}$$

$\frac{1}{1+GC}(0)$ = constant
 $K_p = \text{Position Const}$

FVT

$$\textcircled{2} \quad \lim_{s \rightarrow 0} sE(s) = \frac{\frac{1}{s} \times \frac{1}{s}}{1+G(s)(1/s)} = \frac{1}{1+G(1/s)(1/s)}$$

$$\lim_{s \rightarrow 0} sG(s)(1/s) = k_v = \text{Velocity Const}$$

FVT

$$\textcircled{3} \quad \lim_{s \rightarrow 0} sE(s) = \frac{s \times \frac{1}{s^3}}{1+G(s)(1/s)} = \frac{1}{1+G(1/s)(1/s)}$$

$$\lim_{s \rightarrow 0} s^2 G(s) C(s) = K_a = \text{accel' Const}$$

Now Consider $G(s) C(s)$ to be Type 1 :- $(s) G(s) = \frac{1}{s} \bar{C}(s) \bar{C}_n(s) \Rightarrow \bar{G}(1) \bar{C}(0) = \text{finite}$

$$\frac{1}{1 + \frac{1}{s} G \cdot C} = \boxed{0}$$

$$\lim_{s \rightarrow 0} \frac{1}{s + \bar{C}(s) G(s)} = \boxed{\frac{1}{K_V}}$$

$$\lim_{s \rightarrow 0} sE(s) = \boxed{\infty}$$

Now Consider Type 2

Step = 0

Ramp = 0

Parabola = $\frac{1}{2} K_a$

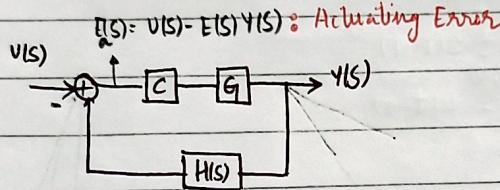
Summary:

Type	i/p	Step	Ramp	Pulse
1	$1/(1+K_p)$	∞	$K_V = 0$	$K_A = 0$
2	0, $K_p \rightarrow \infty$	$1/K_V$	∞	$K_A = 0$
3	$K_p \rightarrow \infty$	0	$K_V \rightarrow \infty$	$1/K_A$

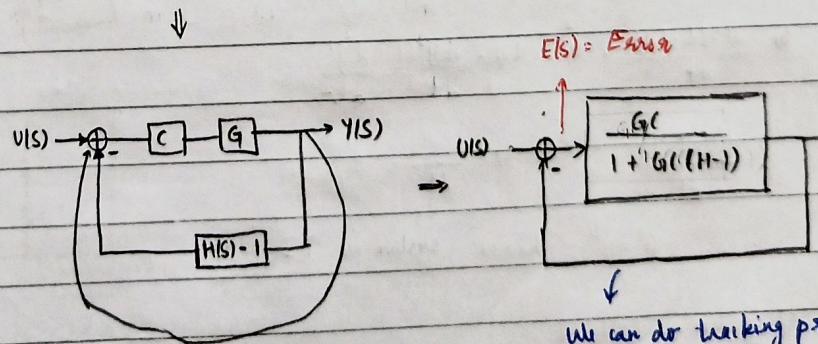
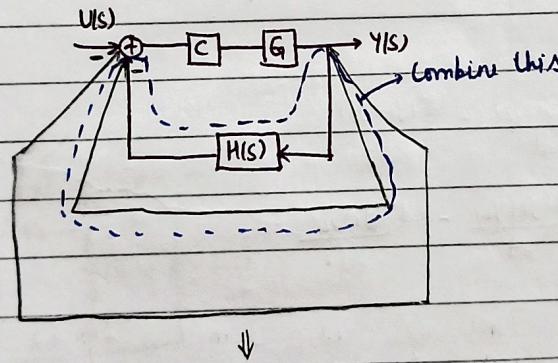
(final, non zero)

- Suppose we are told the finite value of one of the error const, what all info can be inferred?

- Closed loop System is stable
- The kind of Input that I am interested in
- The Type of Loop TF.
- The value of steady state Error



Tracking Problem; care about errors not actuating Error

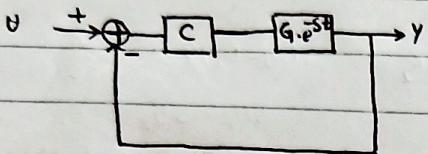
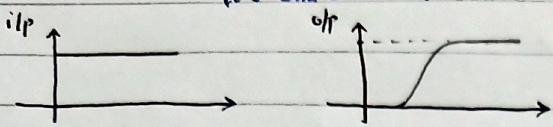


We can do tracking problem

Lecture 12

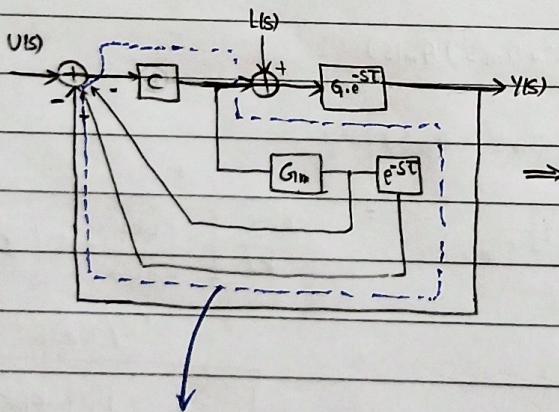
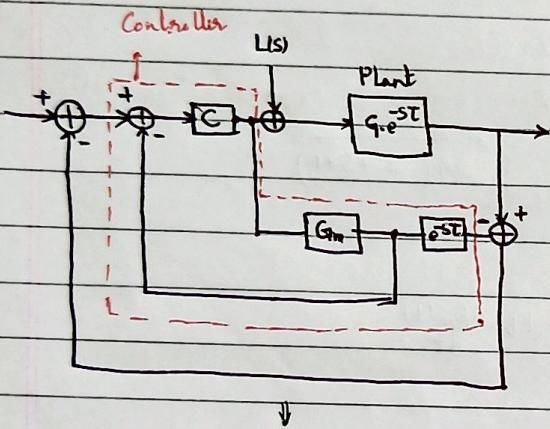
SMITH PREDICTOR

(T. J. Smith (1958))



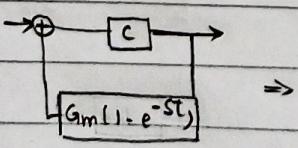
$$\frac{Y(s)}{U(s)} = \frac{G_p e^{-sT} C}{1 + G_p e^{-sT} C}$$

Transcendental: no root, hence no poles



$$Y(s) = U(s) \cdot \left[\frac{G_p e^{-sT} C}{1 + G_p e^{-sT} C} \right] + L \left[\frac{G_p e^{-sT}}{1 + e^{-sT} C} \right]$$

Let $C(s) = k_p + \frac{k_i}{s}$ (PI controller)



$$Y(s) = \frac{\frac{G_p e^{-sT} C}{1 + C G(1 - e^{-sT})}}{1 + \frac{G_p e^{-sT} C}{1 + C G(1 - e^{-sT})}} U(s) + \frac{\frac{G_p e^{-sT}}{1 + e^{-sT} C}}{1 + \frac{G_p e^{-sT}}{1 + e^{-sT} C}} t(s)$$

Assumption

$$G_m = G$$

$$\bar{T} = T$$

$$\bar{C} = \frac{C}{1 + C G(1 - e^{-sT})}$$

$$Y(s) = \frac{G_C e^{-ST}}{1 + CG} U(s) + \frac{G e^{-ST} (1 + CG(1 - e^{-ST}))}{1 + CG} L(s)$$

Assume $\rightarrow G(s)$ is type 0

G_C is type 1

} Then we have tracking property; can also reject load disturbance

- Note: If $G(s)$ is Type 1: Cannot reject Load Disturbance

$$E(s) = Y(s) - U(s)$$

$$= \frac{G_C(e^{-ST} - 1)}{1 + G_C} U(s) + \frac{G \cdot e^{-ST} (1 + CG(1 - e^{-ST}))}{1 + CG} L(s)$$

$$\lim_{s \rightarrow 0} \left[\frac{G \cdot (K_p + \frac{K_I}{s})(e^{-ST} - 1) - 1}{1 + (K_p + \frac{K_I}{s})G} \right] + \lim_{s \rightarrow 0} \frac{G \cdot e^{-ST} (1 + G(1 - e^{-ST}) \cdot (K_p + \frac{K_I}{s}))}{1 + G(K_p + \frac{K_I}{s})}$$

$$= \lim_{s \rightarrow 0} \frac{G \cdot (K_p s + K_I)(e^{-ST} - 1) - 1}{s + (K_p s + K_I)G} + \lim_{s \rightarrow 0} \frac{G e^{-ST} (1 + G(1 - e^{-ST}) \cdot (K_p s + K_I) \cdot s)}{s + G(s K_p + K_I)}$$

$$= \lim_{s \rightarrow 0} \frac{-1}{G(0) K_I}$$

ROUTH-HORWITZ CRITERION

- If a polynomial has all its roots in open LHP, then it is said to be Hurwitz

$$P(s) = a_0 + a_1 s + a_2 s^2 + \dots + a_n s^n$$

$$= a_n \prod_{i=1}^n (s - \lambda_i), \lambda_i \in \mathbb{R}$$

$$P(j\omega^*) = a_n \prod_{i=1}^n (j\omega^* - \lambda_i)$$

For a Hurwitz Polynomial

$$\sum_{i=1}^n \Delta \theta = \frac{n\pi}{2}$$

Monotonically Increasing

