

## ENG2009 – Modelling of Engineering Systems

### Tutorial 2

#### Linear systems and Laplace Transform

##### Linear Systems:

1) What is system order?

**Answer:** (lecture slides)

Order of a system:

- is the *highest power* of the derivative in the *differential equation*,
- or the *highest power of “s”* in the *denominator* of the *transfer function* (ratio of output/input of a system)

**System order:**

**Example :** What is the order of the following systems

$$\frac{dy(t)}{dt} + y(t) = u(t)$$

**Answer:** 1<sup>st</sup> order

-----  
**Exercises**  
-----

**Q1)** What is the order of the following systems. Explain your reasoning.

(a)  $\frac{d^2y(t)}{dt^2} + 2\frac{dy(t)}{dt} + y(t) = u(t)$

(b)  $3\frac{d^3y(t)}{dt^3} + 2\frac{d^2y(t)}{dt^2} + 2\frac{dy(t)}{dt} + y(t) = u(t)$

(c)  $G(s) = \frac{1}{2s+1}$

(d)  $G(s) = \frac{1}{2s^2+2s+1}$

(e)  $G(s) = \frac{1}{4s^3+2s^2+2s+1}$

**Answer:**

- (a) 2<sup>nd</sup> order
- (b) 3<sup>rd</sup> order
- (c) 1<sup>st</sup> order
- (d) 2<sup>nd</sup> order
- (e) 3<sup>rd</sup> order

## Linear vs nonlinear systems

**Example:** Is the following is linear or not linear:  $f(x) = 5 \cos(x)$ . Explain your reasoning.

**Answer:** nonlinear due to the cos term.

The term does not satisfy the principle of superposition (check lecture notes).

---

## Exercises

---

**Q2)** Which of the following is linear and which is not linear:

- (a)  $\frac{d^2y(t)}{dt^2} + 5 \frac{dy(t)}{dt} + 10y(t) = u(t)$
- (b)  $m \frac{d^2y(t)}{dt^2} + b \frac{dy(t)}{dt} + ky(t) = u(t)$ , where  $m, b, k$  are constants
- (c)  $m(t) \frac{d^2y(t)}{dt^2} + b \frac{dy(t)}{dt} + ky(t) = u(t)$ , where  $m(t)$  is time varying and  $b, k$  are constants
- (d)  $\frac{d^2x}{dt^2} + (1 - \cos(2t))x = 0$
- (e)  $\frac{d^2x}{dt^2} + (x^2 - 1) \frac{dx}{dt} + x = 0$
- (f)  $\frac{d^2x}{dt^2} + \frac{dx}{dt} + x + x^3 = \sin(\omega t)$

**Answer:**

- (a) LTI Linear differential eq
- (b) LTI Linear differential eq
- (c) LTV. Linear structure, parameter  $m(t)$  is time varying
- (d) Nonlinear - due to time varying parameter  $(1 - \cos(2t))$
- (e) Nonlinear – The equation contain powers  $x^2$ .
- (f) Nonlinear – The equation contain powers  $x^3$

## Linearisation

*Example:* (Lecture notes – pendulum example)

Linearise  $\sin(\theta)$  around  $\theta = 0$  (small angle)

*Answer:*

From Taylor series expansion

$$f(x) \approx f(x_0) + \left. \frac{df}{dx} \right|_{x=x_0} (x - x_0)$$

Or

$$f(x) - f(x_0) \approx \dot{f}(x_0)(x - x_0)$$

Therefore, for  $\theta_0 = 0$

$$f(x) = \sin(\theta)$$

$$f(0) = \sin(0) = 0$$

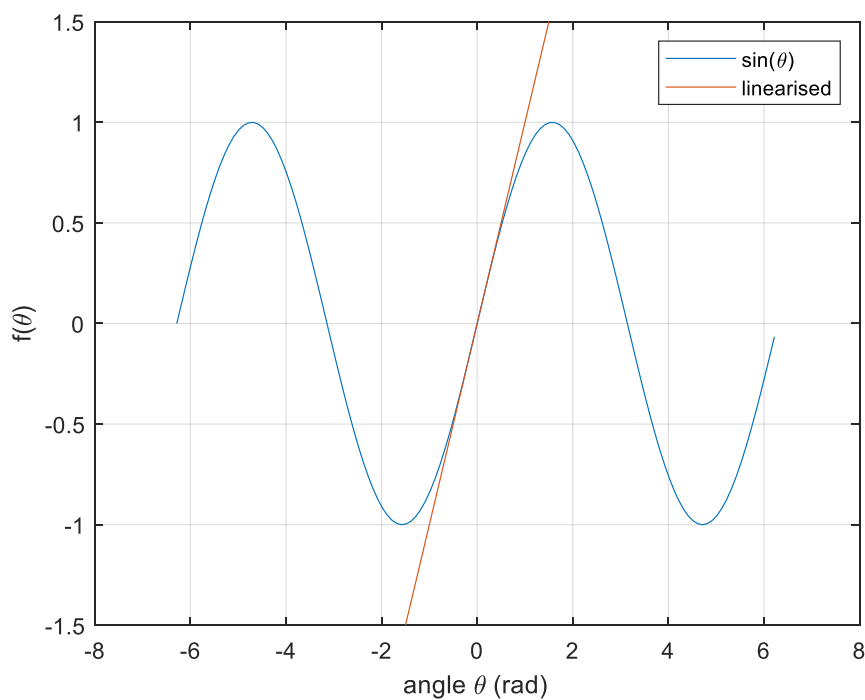
$$\dot{f}(x_0) = \left. \frac{d(\sin(\theta))}{d\theta} \right|_{\theta=0} = \cos(0) = 1$$

Therefore

$$f(x) \approx f(\theta_0) + \left. \frac{df}{d\theta} \right|_{\theta=0} (\theta - \theta_0) = 0 + (1)(\theta - 0) = \theta$$

Therefore linearisation of  $\sin(\theta)$  around small angle  $\theta = 0$  is

$$\sin(\theta) \approx \theta$$



-----  
**Exercises**  
-----

**Q3)** Linearise the following:

(i)  $f(x) = 5 \cos(x)$ , about  $x = \pi/2$

(ii)  $f(x) = 2 \ln(x)$ , about  $x = 1$

**Answer:**

(i) Solution

$$f(x) \approx f(x_0) + \left. \frac{df}{dx} \right|_{x=x_0} (x - x_0)$$

Therefore, for  $x_0 = \frac{\pi}{2}$

$$f(x) = 5 \cos(x)$$

$$f(\pi/2) = 5 \cos(\pi/2) = 0$$

$$\dot{f}(x_0) = 5 \left. \frac{d(\cos(x))}{dx} \right|_{x=\pi/2} = -5 \sin(x) \big|_{x=\pi/2} = -5$$

Therefore

$$f(x) \approx f\left(\frac{\pi}{2}\right) + \left. \frac{df}{dx} \right|_{x=\frac{\pi}{2}} \left(x - \frac{\pi}{2}\right) = 0 + (-5) \left(x - \frac{\pi}{2}\right) = -5 \left(x - \frac{\pi}{2}\right)$$

Therefore linearisation of  $5\cos(x)$  around  $x = \frac{\pi}{2}$  is

$$5 \cos(x) \approx -5 \left(x - \frac{\pi}{2}\right)$$

(ii) Solution

$$f(x) \approx f(x_0) + \left. \frac{df}{dx} \right|_{x=x_0} (x - x_0)$$

Therefore, for  $x_0 = 1$

$$f(x) = 2 \ln(x)$$

$$f(1) = 2 \ln(1) = 0$$

$$\dot{f}(x_0) = 2 \left. \frac{d(\ln(x))}{dx} \right|_{x=1} = 2 \left. \frac{1}{x} \right|_{x=1} = 2$$

Therefore

$$f(x) \approx f(1) + \left. \frac{df}{dx} \right|_{x=1} (x - x_0) = 0 + (2)(x - 1) = 2(x - 1)$$

Therefore linearisation of  $2\ln(x)$  around  $x = 1$  is

$$2 \ln(x) \approx 2(x - 1)$$

### Laplace Transform:

Find the Laplace transform for  $f(t) = 1$ . (See lecture notes).

Answer:

For a signal  $f(t)$ , the Laplace transform is defined as:

$$\mathcal{L}(f(t)) = \int_0^{\infty} f(t)e^{-st} dt$$

where  $s = \sigma + \omega j$  (complex number)

For example if  $f(t) = 1$  then

$$F(s) := \mathcal{L}(1) = \int_0^{\infty} 1e^{-st} dt = \left[ -\frac{1}{s} e^{-st} \right]_0^{\infty} = \frac{1}{s}$$

-----  
**Exercise**  
-----

**Q4)** Find the Laplace transform of the following functions:

$f(t) = Ae^{-at}$ , where  $A$  and  $a$  are constants

Answer:

$$\begin{aligned} F(s) &= \int_0^{\infty} f(t)e^{-st} dt = \int_0^{\infty} Ae^{-at}e^{-st} dt = A \int_0^{\infty} e^{-(s+a)t} dt \\ &= -\frac{A}{s+a} e^{-(s+a)t} \Big|_{t=0}^{\infty} = \frac{A}{s+a} \end{aligned}$$

**Q5)** Using the Laplace transform Table and the Laplace transform theorems table (attached at the back of this paper), derive the Laplace transforms for the following time functions:

- a)  $e^{-at} \sin(\omega t)$
- b)  $e^{-at} \cos(\omega t)$
- c)  $t^3$

**Answers**

a. Using the frequency shift theorem and the Laplace transform of  $\sin \omega t$ ,  $F(s) = \frac{\omega}{(s+a)^2 + \omega^2}$ .

b. Using the frequency shift theorem and the Laplace transform of  $\cos \omega t$ ,  $F(s) = \frac{(s+a)}{(s+a)^2 + \omega^2}$ .

c. Using the integration theorem, and successively integrating  $u(t)$  three times,  $\int dt = t$ ;  $\int t dt = \frac{t^2}{2}$  ;

$\int \frac{t^2}{2} dt = \frac{t^3}{6}$  , the Laplace transform of  $t^3 u(t)$ ,  $F(s) = \frac{6}{s^4}$  .

### Final value theorem:

#### **Example:** (Final Value Theorem)

Find the final value of the system corresponding to the following system, where the input  $u(t)$  is a unit step i.e.  $u(t) = 1$ .

$$Y(s) = U(s) \frac{6}{s+2}$$

Hint: use Final Value Theorem

#### **Solution:**

System model represented in Laplace as

$$Y(s) = U(s) \frac{6}{s+2}$$

Step 1: Simple check shows that this system is stable as the denominator has a root at left half of  $s$ -plane, i.e.  $s = -2$ . Therefore Final Value Theorem is valid.

Step 2: The input  $u(t)$  is a unit step i.e.  $u(t) = 1$ . From Laplace table

$$\mathcal{L}(u(t)) = \mathcal{L}(1) = \frac{1}{s}$$

Therefore,

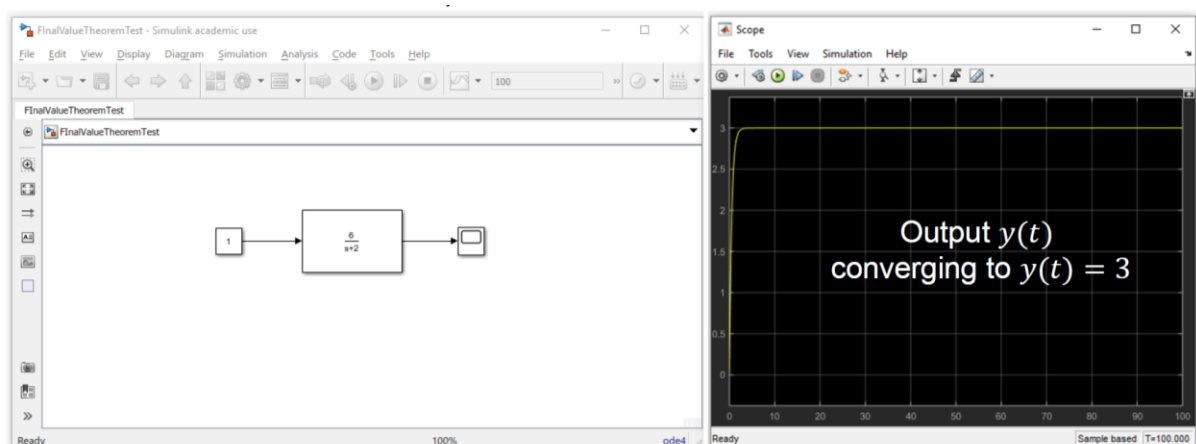
$$Y(s) = \frac{1}{s} \left( \frac{6}{s+2} \right)$$

Step 3: Therefore, the final value of  $y(t)$  is:

$$\lim_{t \rightarrow \infty} (y(t)) = \lim_{s \rightarrow 0} (sY(s)) = s \frac{1}{s} \left( \frac{6}{s+2} \right) \Big|_{s=0} = \frac{6}{2} = 3$$

Thus, after the transients have decayed to zero,  $y(t)$  will settle to a constant value of 3.

Check with simulation results:



## Exercises

**Q6)** Without solving the differential equation, find the steady state values for the following systems (if available), using Final Value Theorem. Assume that the initial conditions are zero, and the input for each systems are unit input  $u(t) = 1$ .

(a) Mechanical system (mass-spring-damper):

$$m\ddot{x}(t) + b\dot{x}(t) + kx(t) = F_e(t)$$

where the mass  $m = 2$ , damping factor  $b = 5$ , spring constant  $k = 1$ , and  $F_e(t)$  is the input, and displacement  $x(t)$  is the output

(b) Electrical system (simple RLC circuit)

$$\frac{3}{4} \frac{dv_o(t)}{dt} + \frac{1}{2} v_o(t) = \frac{1}{2} \frac{di_i(t)}{dt} + i_i(t)$$

where  $v_o(t)$  is the output and  $i_i(t)$  is the input.

**Answer:**

(a)

given  $m\ddot{x} + b\dot{x} + kx = F_e$

$$m = 2, b = 5, k = 1$$

$$\Rightarrow 2\ddot{x} + 5\dot{x} + x = F_e$$

assume zero ic,  $x(0) = 0$ ,  $\dot{x}(0) = 0$

unit step  $F_e(t) = 1$

Laplace:  $2s^2X(s) + 5sX(s) + X(s) = F_e(s)$

$$X(s) (2s^2 + 5s + 1) = F_e(s)$$

$$\text{TF} \quad \frac{X(s)}{F_e(s)} = \frac{1}{(2s^2 + 5s + 1)}$$

poles  $\Rightarrow$  roots of denominator:  $(2s^2 + 5s + 1) = 0$

$$\text{using } \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

poles @  $-2.2808, -0.2192$ , LHP  $\therefore$  stable system

FVT valid

$$X(s) = \frac{1}{(2s^2 + 5s + 1)} F_e(s)$$

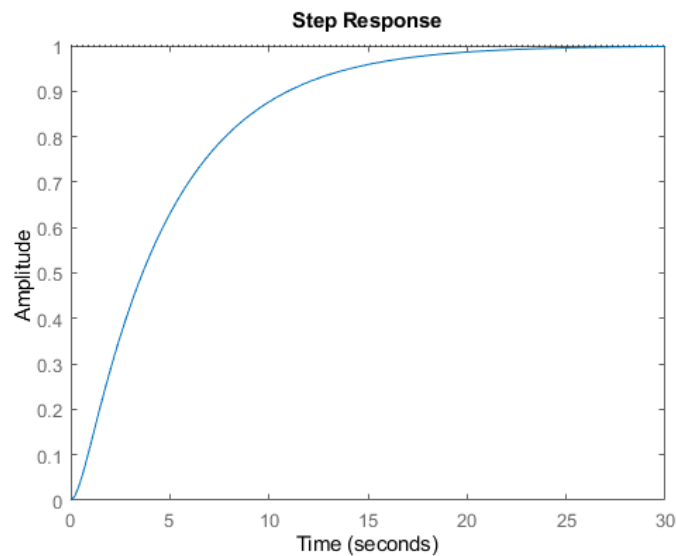
$$F_e(t) = 1 \xrightarrow{\mathcal{L}} F_e(s) = \frac{1}{s}$$

$$\Rightarrow X(s) = \frac{1}{(2s^2 + 5s + 1)} \left( \frac{1}{s} \right)$$

$$\text{FVT} \quad \lim_{t \rightarrow \infty} (y(t)) = \lim_{s \rightarrow 0} (s Y(s))$$

$$= \cancel{s} \frac{1}{(2\cancel{s^2} + 5\cancel{s} + 1)} \left( \frac{1}{\cancel{s}} \right) \bigg|_{s=0} = \underline{1} \quad \begin{array}{l} \text{Final value.} \\ \text{check with sim} \end{array}$$

Simulation result



(b)

$$\text{given} \quad \frac{3}{4} \ddot{v}_o(t) + \frac{1}{2} \dot{v}_o(t) = \frac{1}{2} \ddot{i}_i(t) + \dot{i}_i(t)$$

$$\text{initial cond} \quad v_o(0) = 0, \dot{v}_o(0) = 0, \quad i_i(0) = 0, \dot{i}_i(0) = 0$$

$$\text{input} \quad i_i(t) = 1 \quad (\text{unit step})$$

$$\text{Laplace:} \quad \frac{3}{4} s V_o(s) + \frac{1}{2} V_o(s) = \frac{1}{2} s I_i(s) + I_i(s)$$

$$V_o(s) \left( \frac{3}{4} s + \frac{1}{2} \right) = I_i(s) \left( \frac{1}{2} s + 1 \right)$$



Transfer func

$$\frac{V_o(s)}{I_i(s)} = \frac{(\frac{1}{2}s + 1)}{(\frac{3}{4}s + \frac{1}{2})}$$

Poles: roots of denominator

$$\begin{aligned} \text{roots of } (\frac{3}{4}s + \frac{1}{2}) &= 0 \\ s &= -\frac{1}{\cancel{2}} \left( \frac{\cancel{4}}{3} \right) = -\frac{2}{3} = -0.6667 \end{aligned}$$

Stable pole (LHP)  $\therefore$  FVT applies

$$V_o(s) = \frac{(\frac{1}{2}s + 1)}{(\frac{3}{4}s + \frac{1}{2})} I_i(s) \quad , \quad \text{input } u_i(t) = 1$$

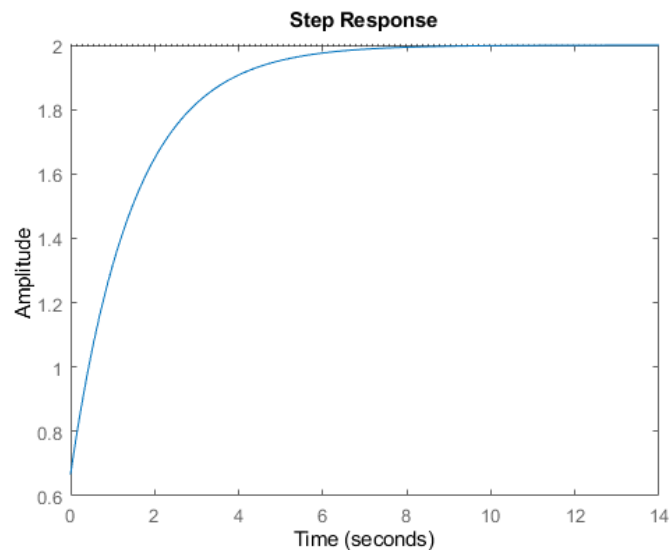
$$I_i(s) = \frac{1}{s}$$

FVT

$$\lim_{t \rightarrow \infty} (v_o(t)) = \lim_{s \rightarrow 0} (s V_o(s)) = \cancel{s} \left( \frac{\cancel{\frac{1}{2}s + 1}}{(\frac{3}{4}\cancel{s} + \frac{1}{2})} \right) \left( \frac{1}{\cancel{s}} \right) \bigg|_{s=0} = \frac{1}{(\frac{1}{2})} = 2$$

Final value  
To check with sim

Simulation results



### **Python code for Q6(a) mechanical system**

```
#import numpy, integrate and pyplot
import numpy as np
from scipy import integrate
from matplotlib import pyplot as plt

#sim data
m=2 #mass of the oscillator
b=5 #damper of the oscillator
k=1 #stiffness of the oscillator

#create 1000 uniformly distributed points in the interval [0,35]
t = np.linspace(0,35,1000)

#lambda returning the force
force = lambda t:t>=0.01 #unit step

#lambda with the RHS of the eq., it evaluates the force and returns a list
f = lambda X,t: [X[1],1/m*(force(t)-k*X[0]-b*X[1])]

#solve equation using odeint
Xodeint = integrate.odeint(f,[0,0],t)

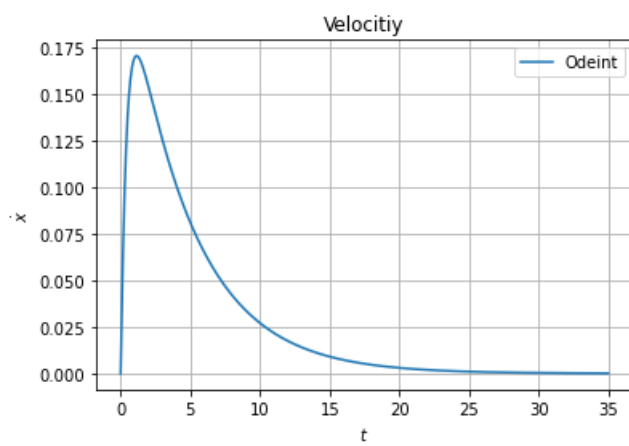
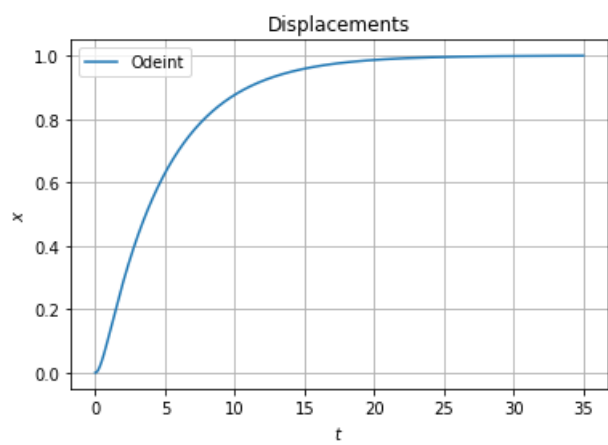
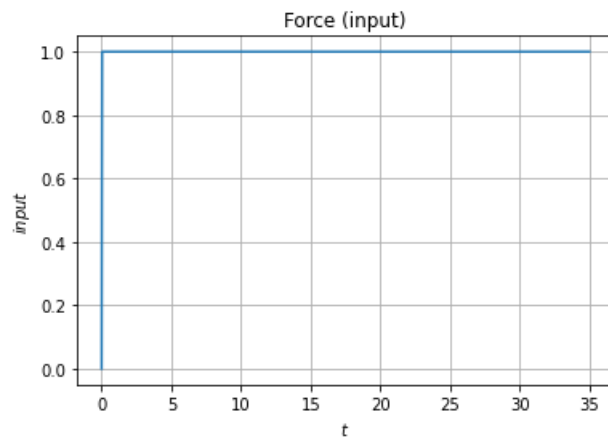
plt.plot(t,force(t))
plt.title('Force (input)')
plt.xlabel('$t$')
plt.ylabel('$input$')
plt.grid()
plt.show()

#plot displacements from both solutions (first column of each solution)
plt.plot(t,Xodeint[:,0])
plt.legend(['Odeint'])
plt.title('Displacements')
plt.xlabel('$t$')
plt.ylabel('$x$')
plt.grid()
plt.show()

#plot velocities from both solutions (first column of each solution)
plt.plot(t,Xodeint[:,1])
plt.legend(['Odeint'])
plt.title('Velocitiy')
plt.xlabel('$t$')
plt.ylabel('$\dot{x}$')
plt.grid()
plt.show()

#Save data
np.savetxt('t.txt',t)
np.savetxt('x.txt',Xodeint[:,0])
```

## Simulation results



**Table of Laplace Transforms**

Number	$F(s)$	$f(t), t \geq 0$
1	1	$\delta(t)$
2	$\frac{1}{s}$	$1(t)$
3	$\frac{1}{s^2}$	$t$
4	$\frac{2!}{s^3}$	$t^2$
5	$\frac{3!}{s^4}$	$t^3$
6	$\frac{m!}{s^{m+1}}$	$t^m$
7	$\frac{1}{(s+a)}$	$e^{-at}$
8	$\frac{1}{(s+a)^2}$	$te^{-at}$
9	$\frac{1}{(s+a)^3}$	$\frac{1}{2!}t^2e^{-at}$
10	$\frac{1}{(s+a)^m}$	$\frac{1}{(m-1)!}t^{m-1}e^{-at}$
11	$\frac{a}{s(s+a)}$	$1 - e^{-at}$
12	$\frac{a}{s^2(s+a)}$	$\frac{1}{a}(at - 1 + e^{-at})$
13	$\frac{b-a}{(s+a)(s+b)}$	$e^{-at} - e^{-bt}$
14	$\frac{s}{(s+a)^2}$	$(1-at)e^{-at}$
15	$\frac{a^2}{s(s+a)^2}$	$1 - e^{-at}(1+at)$
16	$\frac{(b-a)s}{(s+a)(s+b)}$	$be^{-at} - ae^{-bt}$
17	$\frac{a}{(s^2+a^2)}$	$\sin at$
18	$\frac{s}{(s^2+a^2)}$	$\cos at$
19	$\frac{s+a}{(s+a)^2+b^2}$	$e^{-at} \cos bt$
20	$\frac{b}{(s+a)^2+b^2}$	$e^{-at} \sin bt$
21	$\frac{a^2+b^2}{s[(s+a)^2+b^2]}$	$1 - e^{-at} \left( \cos bt + \frac{a}{b} \sin bt \right)$

### Properties of Laplace Transforms

Number	Laplace Transform	Time Function	Comment
—	$F(s)$	$f(t)$	Transform pair
1	$\alpha F_1(s) + \beta F_2(s)$	$\alpha f_1(t) + \beta f_2(t)$	Superposition
2	$F(s)e^{-s\lambda}$	$f(t - \lambda)$	Time delay ( $\lambda \geq 0$ )
3	$\frac{1}{ a } F\left(\frac{s}{a}\right)$	$f(at)$	Time scaling
4	$F(s + a)$	$e^{-at} f(t)$	Shift in frequency
5	$s^m F(s) - s^{m-1} f(0) - s^{m-2} \dot{f}(0) - \dots - f^{(m-1)}(0)$	$f^{(m)}(t)$	Differentiation
6	$\frac{1}{s} F(s)$	$\int_0^t f(\zeta) d\zeta$	Integration
7	$F_1(s) F_2(s)$	$f_1(t) * f_2(t)$	Convolution
8	$\lim_{s \rightarrow \infty} s F(s)$	$f(0^+)$	Initial Value Theorem
9	$\lim_{s \rightarrow 0} s F(s)$	$\lim_{t \rightarrow \infty} f(t)$	Final Value Theorem
10	$\frac{1}{2\pi j} \int_{\sigma_c - j\infty}^{\sigma_c + j\infty} F_1(\zeta) F_2(s - \zeta) d\zeta$	$f_1(t) f_2(t)$	Time product
11	$\frac{1}{2\pi} \int_{-j\infty}^{+j\infty} Y(-j\omega) U(j\omega) d\omega$	$\int_0^\infty y(t) u(t) dt$	Parseval's Theorem
12	$-\frac{d}{ds} F(s)$	$tf(t)$	Multiplication by time