

σ - sigma

resistance of the material to deformation takes place permanently.

Concept of Stress & Strain, Tensile test

We have already established that the strength of a material is the ability to withstand load without failure. A load applied to a mechanical member will induce internal forces within the member. These internal forces are called stresses within the member.

The stresses acting on the material cause deformation of the material in various manner.

The deformation of the material is called strain.

Strain = $\frac{\text{Change in length}}{\text{Original length}}$

The force of resistance per unit area offered by a body against deformation is known as stress. The load is applied on the body while the stress is induced in the material of the body. A loaded member remains in equilibrium when the resistance offered by the member against deformation and the applied load are equal.

Mathematically,

$$\text{Stress } \sigma = \frac{P}{A}$$

σ = Stress (Intensity of stress)

P = External Force (Applied load)

A = Cross-section Area

The unit of stress depends upon the unit of load and the unit of area. The S.I. unit is N/m^2 and can also be expressed as N/mm^2 .

$$1\text{KN} = 1000\text{ N}, 1\text{MN} = 10^6\text{ N}$$

$$1\text{N/mm}^2 = 10^6\text{ N/m}^2$$

$$1\text{N/mm}^2 = 1\text{ Pascal (Pa)}$$

$$1\text{KN} = 1000\text{ N}; 1\text{GPa} = 10^9\text{ N/m}^2$$

Strain

When a body is subjected to some external force, there is some change of dimension of the body. The ratio of change of dimension of the body to the original dimension is known as strain. Strain may be tensile strain, compressive strain, volumetric strain and shear strain.

If there is some increase in length of a body due to external force then the ratio of increase in length to the original length is called tensile strain but if there is decrease in length of a body due

due to an external force then the ratio of decrease in length to the original length is called compressive strain.

The ratio of change of volume to the original volume is called volumetric strain.

The strain produced by shear stress is known as shear strain.

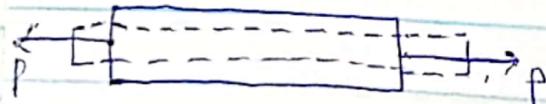
TYPES OF STRESS

-Normal stress

-Shear stress

Normal stress is the stress which acts in the direction perpendicular to the area.

It is represented by sigma (σ). The normal stress is further divided into tensile stress and compressive stress.



Tensile Stress

This is the stress induced in the body when subjected to two equal and opposite pull, as a result of which there is an increase in length. The ratio of increase in length to the original length is Tensile strain.

The tensile stress acts normal to the area and it pulls on the area.

Let P = Pull (or force) acting on the body.

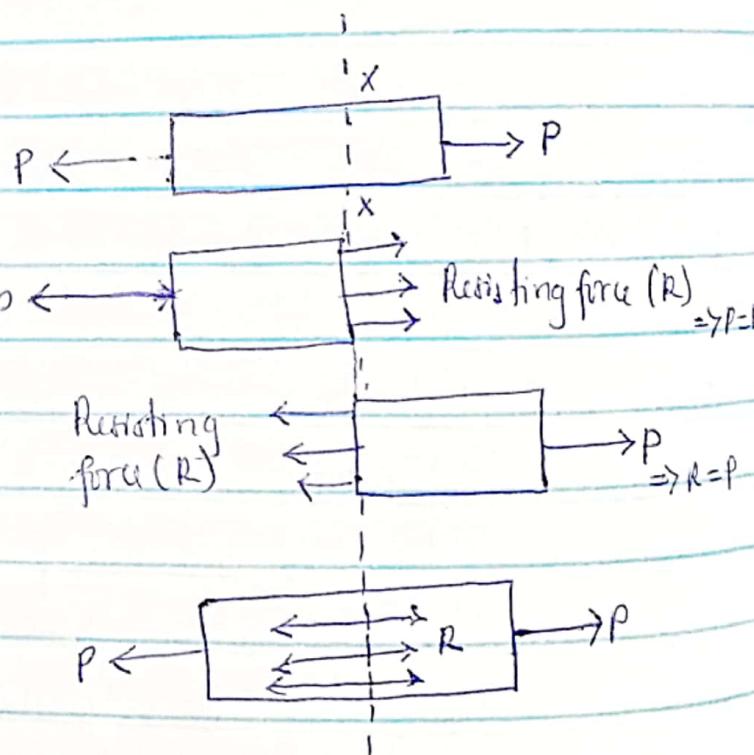
A = cross-sectional area of the body

L = original length of the body.

dL = increase in length due to pull acting on the body.

σ = stress induced in the body.

e = strain (i.e. tensile strain).



Consider a section $x-x$ which divides the bar into two parts. The part left to section $x-x$ will be in equilibrium if $P=R$.

Similarly, the part right to section $x-x$ will be in equilibrium if $R=P$.

1) known as stress or stress intensity or intensity of stress.

Tensile stress, $\sigma = \frac{\text{Resisting force } 'R'}{\text{Cross-sectional Area } 'A'}$

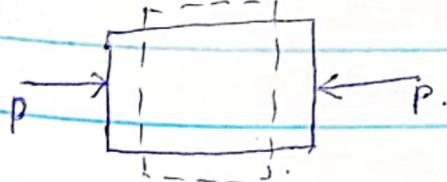
$$\sigma = \frac{\text{Tensile load } P}{A}$$

$$\sigma = \frac{P}{A}$$

$$\text{Tensile strain } e = \frac{\text{Increase in length}}{\text{Original length}}$$

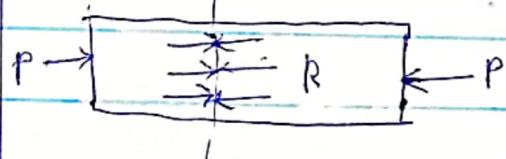
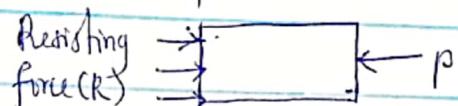
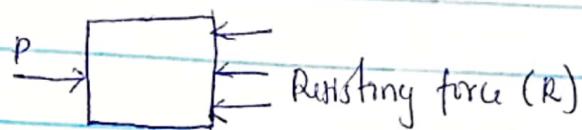
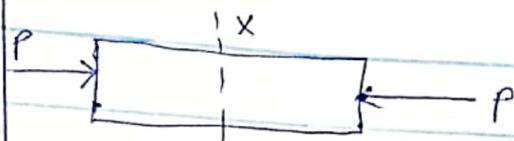
$$1 \cdot 2 \quad \ell = \underline{d} \underline{\ell}$$

Compressive stress.



Stress induced in a body when subjected to two equal but opposite pushes as a result of which there is a decrease in length of the body is known as compressive stress. The ratio of decrease in length to the original length is known as compressive strain.

The compressive stress act normal to the area and pushes on the area.



Let an axial push (P) acting on the body with cross-sectional area ~~A~~ ^{original} A' due to external load let the original length of the body decrease by dl .

$$\text{Compressive stress } \sigma = \frac{R}{A} = \frac{P}{A}$$

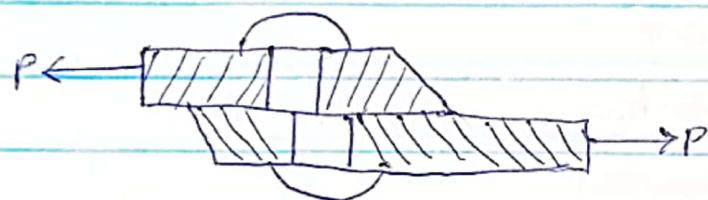
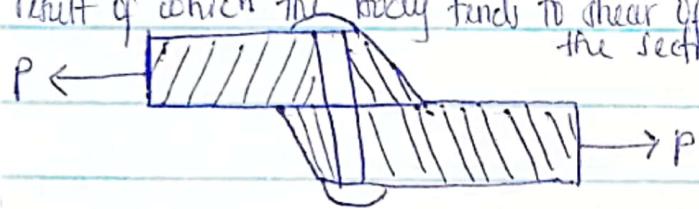
Compressive strain ϵ

$$\frac{\ell}{L} = \frac{\text{Decrease in length}}{\text{Original length}} = \frac{\delta L}{L}$$

(VE - 204 [continuation])

Shear Stress

The stress induced in a body when subjected to two equal and opposite forces which are acting tangentially across the twisting section are shown in the figure as a result of which the body tends to shear off across the section.

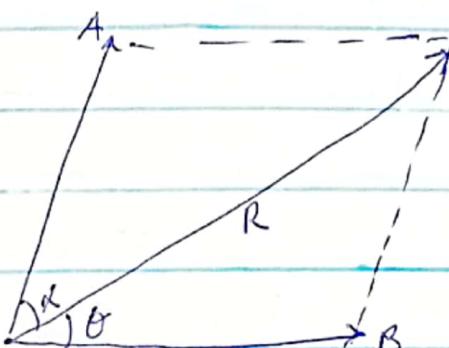
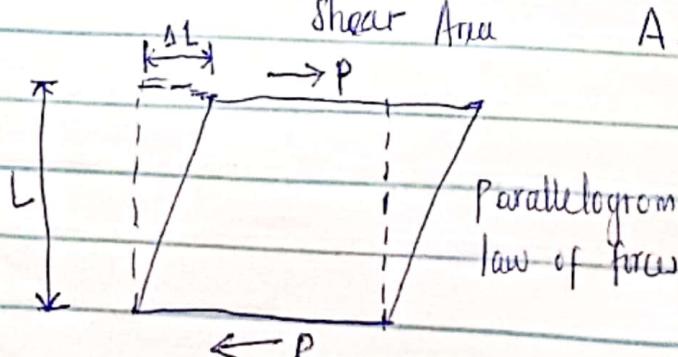


The corresponding strain is known as shear strain. The shear stress is the stress which acts tangential to the area. It is represented as τ .

τ = shear stress

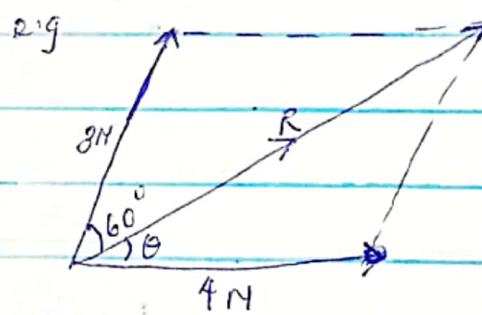
Mathematically

$$\text{Shear stress } \tau = \frac{\text{Shear Resistance}}{\text{Shear Area}} = \frac{R}{A}$$



$$R^2 = A^2 + B^2 + 2AB \cos \alpha$$

$$R = \sqrt{A^2 + B^2 + 2AB \cos \alpha}$$



$$R^2 = 3^2 + 4^2 + 2(3)(4) \cos 60^\circ$$

$$R = 6.08 \text{ N magnitude}$$

The direction of R is given by θ

$$\tan \theta = \frac{3}{4} \sin 60^\circ$$

$$B + A \cos 60^\circ$$

$$\tan \theta = \frac{3 \sin 60^\circ}{7 \cos 60^\circ} = \frac{2.59}{3.5}$$

$$\tan \theta = 0.74$$

$$\theta = \tan^{-1} 0.74$$

$$\theta = 36.5^\circ$$

10m

100mm² $\frac{10}{10 \times 10} = 10x$

Elasticity and Elastic limit

When an external force act on a body the body tends to undergo some deformation. If the external force is removed and the body comes back to its original shape and size (which means the deformation disappears completely), the body is known as elastic body. This property of which $E = 1.05 \times 10^5 \text{ N/mm}^2$.

Certain materials return back to their original position after the removal of external force is called elasticity. There is a limit value of force up to and within which the deformation entirely disappears from the removal of force.

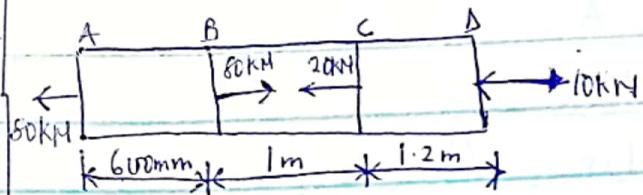
Principle of super-position

When a number of loads are acting on a body, the resulting strain according to principle of superposition will be algebraic sum of the strain caused by individual loads. While using this principle for an elastic body, which is subjected to a number of direct forces (tensile and compressive) at different section along the length of the body. First, the free body diagram of the individual section is drawn. Second, the deformation of each section is obtained. Last, the total deformation of the body will be equal to the algebraic sum of

the individual section.

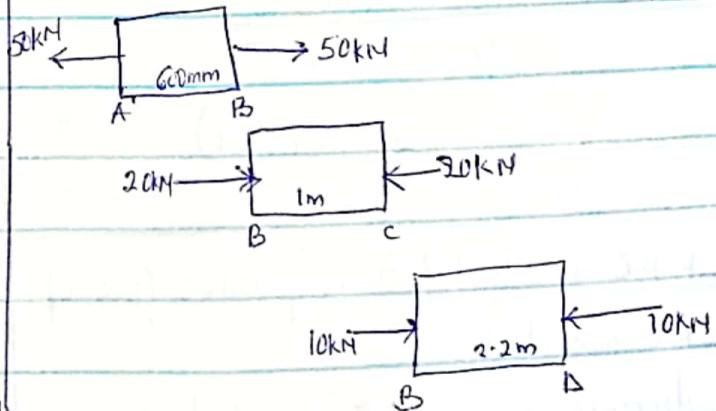
Example 1:

A brass bar having cross-sectional area of 100mm^2 is subjected to axial forces as shown in the figure below. Find the total elongation of the bar. Take $E = 1.05 \times 10^5 \text{ N/mm}^2$.



solution

(i) Draw the free body diagram



Part AB is subjected to tensile force 50kN magnitude

Increase in length $AB = \text{Elongation}$

$$L_{AB} = 600\text{mm}$$

$$P = 50\text{kN} = 50,000\text{N}$$

$$e = \left(\frac{dL_{AB}}{L_{AB}} \right)$$

$$E = \underline{\sigma}$$

e

$$\epsilon = \underline{\sigma}$$

E

$$\delta L_{AB} = \underline{\sigma}$$

$$L_{AB} E$$

$$\sigma = \frac{P}{A}$$

$$1.2 \delta L_{AB} = \frac{P}{AE}$$

$$\delta L_{AB} = \frac{PL_{AB}}{AE}$$

$$\delta L_{AB} = \frac{50,000 \times 600}{1000 \times 1.05 \times 10^5}$$
$$= 0.2857 \text{ mm (+)}$$

Part BC is subjected to compressive force of 20 kN magnitude

decrease in length BC = reduction

$$L_{AB} = 1000 \text{ mm}$$

$$\delta L_{BC} = \frac{20,000 \times 1000}{1000 \times 1.05 \times 10^5}$$
$$= 0.1905 \text{ mm (-)}$$

Part BD is subjected to compressive force of 10 kN magnitude

decrease in length BD = reduction

$$\delta L_{BD} = \frac{22,000 \times 1000}{1000 \times 1.05 \times 10^5}$$

$$= 0.2095 \text{ mm (-)}$$

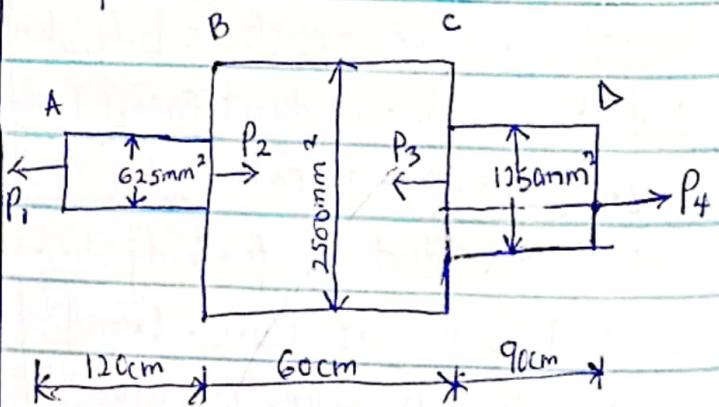
(B) Total elongation of the bar = algebraic sum of elongation or deformation as the case may be of the individual sections.

$$\delta L = \delta L_{AB} + \delta L_{BC} + \delta L_{BD}$$
$$= +0.2857 - 0.1904 - 0.2095$$

$$\delta L = -0.1142 \text{ mm}$$

The brass bar has been shortened by 0.1142 mm

Example 2:



A member A-B-C-D is subjected to point load P₁, P₂, P₃ and P₄. Calculate the force P₂ required

necessary for equilibrium if $P_1 = 45\text{ kN}$, $P_3 = 450\text{ kN}$
 $P_4 = 130\text{ kN}$, determine

the total elongation of the member assuming
 the modulus of elasticity = $2.1 \times 10^5 \text{ N/mm}^2$.

value of P_2 necessary for equilibrium

EEE 204

Types of resistors

① Carbon composition

② Deposited carbon

③ High voltage ink film

④ Metal film

⑤ Metal glaze

⑥ Wire wound

Carbon Composition

It is a combination of carbon particles and a binding resin with a different proportion of for providing desired resistance.

Deposited Carbon

This resistor consists of ceramic rods which have a carbon film deposited on them.

High Voltage Ink Film

These resistors consist of a ceramic base on which a special resistive ink is laid down in a helical band.

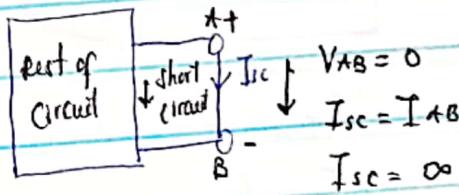
Metal Film - These are resistors made by depositing vaporized metal in vacuum on a ceramic core rod.

Metal Glaze - These resistors consist of a metal glaze mixture which is applied as a thick film to a ceramic substrate and then fired to form a film.

Short and Open Circuit

Short Circuit - When two points of a circuit are connected together by a thick metallic wire, they are said to be short-circuited. They have no resistance, no voltage can come across it and have large current.

The current is high because resistance equals to 0. The current through it is called short-circuit current is very large (theoretically infinity).

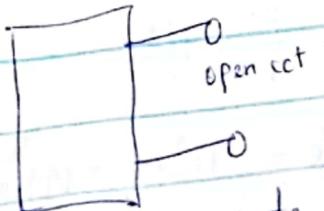


$$\textcircled{1} R = 0$$

$$\textcircled{2} V = 0 \text{ since } V = IR = I_{sc} \times 0 = 0$$

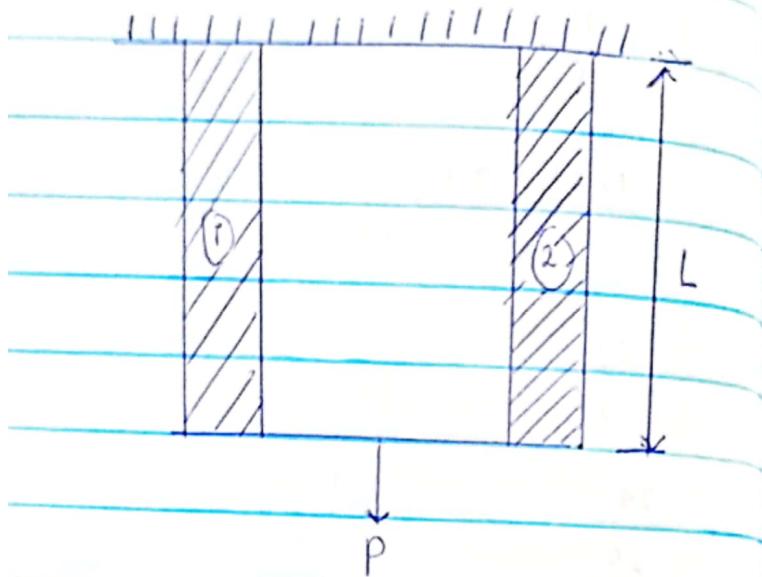
Open Circuit

Two points are said to be open circuited when there is no direct connection between them.



It has a large or infinite resistance. Since resistance is infinite Current = 0.

ANALYSIS OF BARS OF COMPOSITE SECTION



A bar made up of two or more parts of equal length of equal length but of different natural rigidly fixed with each other and behave as one unit for extension or compression when subjected to an axial tensile or compressive force is called a composite bar.

For composite bar the following four points are important.

(1) The extension or compression in each bar is equal. Hence, deformation per unit length i.e. strain in each part is equal.

(2) The total external load on the composite bar is equal to the sum of the load carried by each different material.

P = Total load on the composite.

L = length of composite bar and also length of bars of different materials.

A_1 = Area of cross-section of bar 1

A_2 = Area of x-section of bar 2

E_1 = Young Modulus of bar 1

E_2 = Young Modulus of bar 2

P_1 = Load shared by bar 1

P_2 = Load shared by bar 2

σ_1 = Stress induced by bar 1

σ_2 = Stress induced by bar 2

Now total load on a composite bar is equal to the sum of the load carried by the two bars

i.e.

$$P = P_1 + P_2 \quad \text{--- (1)}$$

Stress in bar 1 = load carried by bar 1
Area of x-section of bar 1

$$\text{i.e. } \sigma_1 = \frac{P_1}{A_1}$$

$$\text{or } P_1 = \sigma_1 A_1 \quad \text{--- (2)}$$

Similarly,

$$P_2 = \sigma_2 A_2 \quad \text{--- (3)}$$

$$\text{From eqn(1)} \Rightarrow P = P_1 + P_2$$

$$\text{i.e. } P = \sigma_1 A_1 + \sigma_2 A_2 \quad \text{--- (4)}$$

Since the ends of the two bars are rigidly connected, each bar will change in length by

the same amount.

Also, the length of each bar is same and hence the ratio of change in length to the original length (strain) would be the same for each bar, but strains.

But

Strain in bar 1 = stress in bar 1

Young modulus of bar 1

$$= \frac{\sigma_1}{E_1}$$

Strain in bar 2 = σ_2

$$E_2$$

Since strain is the same;

$$\frac{\sigma_1}{E_1} = \frac{\sigma_2}{E_2} \quad \text{--- (5)} \Rightarrow \frac{E_1}{E_2} = \frac{\sigma_1}{\sigma_2}$$

From eqn (4) and (5), the stresses (σ_1) and (σ_2) can be determine by substituting the value of σ_1 and σ_2 in eqn (2) and (3). The load carried by different materials can be computed.

The ratio $\frac{E_1}{E_2}$ is called the modular ratio of first material to the second.

Example

A reinforced concrete circular column section

of $50,000 \text{ mm}^2$ cross-sectional area carried 6 reinforcing bars whose total area

is 500 mm^2 . If the concrete is not to be stressed more than 3.5 MPa . Find the safe load the column can carry. Take modular ratio for steel and concrete as 18.

Soln

Given: Area of Column = $50,000 \text{ mm}^2$

No of reinforcing bars = 6

Total area of steel bars $A_s = 500 \text{ mm}^2$.

Maximum stress in Concrete $\sigma_c = 3.5 \text{ MPa}$

Modular ratio $\frac{E_s}{E_c} = 18$.

Area of concrete is given as A_c

Area of concrete + Area of steel = Area of column.

$A_c + A_s = \text{Area of column}$.

$$A_c = 50,000 - 500$$

$$= 49,500 \text{ mm}^2$$

Stress in steel

$$\frac{\sigma_1}{E_1} = \frac{\sigma_2}{E_2}$$

$$\frac{\sigma_s}{E_s} = \frac{\sigma_c}{E_c}$$

$$\sigma_s = \frac{E_s}{E_c} \cdot \sigma_c$$

$$\sigma_s = 18 \times 3.5$$

$$= 63 \text{ MPa.}$$

Safe load the column can carry.

$$P = P_1 + P_2$$

$$P = P_c + P_s.$$

$$P = \sigma_c A_c + \sigma_s A_s$$

$$= (3.5 \times 49500) + (63 \times 500)$$

$$= 204750 \text{ N}$$

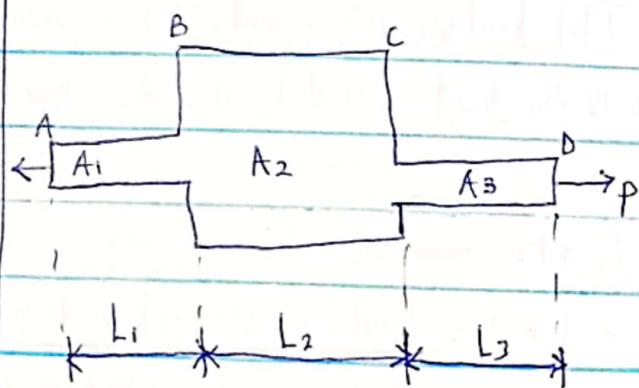
$$= \underline{204.75 \text{ kN}}$$

$$\rightarrow 1 \text{ MPa} = 1 \text{ N/mm}^2$$

Strain in bars of different sections-

Stresses in bars of Different Sections,

Elongation of bars of varying x-section



Sometimes a bar is made up of different length having different cross-sectional area as shown above. In such cases, the stresses, strain and hence, change in length for each section is worked out separately as usual.

The total change in length = sum of change of all the individual length. It may be noted that each section is subjected to the same axial push or pull

let P = Force acting on the body

E = Young modulus of the body.

L_1 = length of Section 1

A_1 = y -sectional area of section 1

L_2, A_2, L_3, A_3 = corresponding values for section 2, 3... and so on.

$$\delta L = \frac{PL}{AE}$$

Section 1

$$\delta L_1 = \frac{PL_1}{A_1 E}$$

Section 2 = Ø

$$\delta L_2 = \frac{PL_2}{A_2 E}$$

Section 3

$$\delta L_3 = \frac{PL_3}{A_3 E}$$

Total deflection

$$\delta L = \delta L_1 + \delta L_2 + \delta L_3 + \dots$$

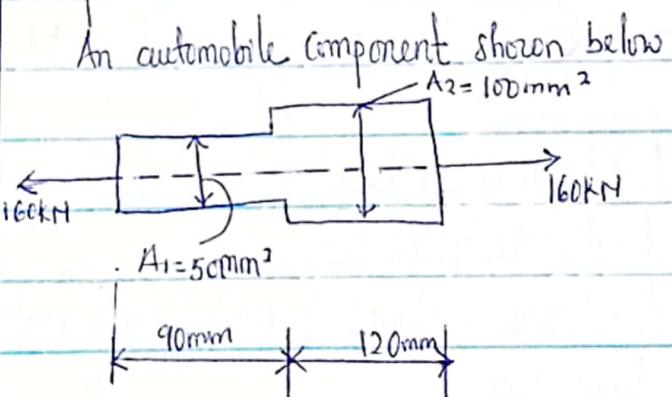
$$\delta L = \frac{PL_1}{A_1 E} + \frac{PL_2}{A_2 E} + \frac{PL_3}{A_3 E} + \dots$$

$$\delta L = \frac{P}{E} \left[\frac{L_1}{A_1} + \frac{L_2}{A_2} + \frac{L_3}{A_3} + \dots \right]$$

Sometimes modulus of elasticity is different for different sections. In such cases, deformation will be;

$$\delta L = P \left[\frac{L_1}{A_1 E_1} + \frac{L_2}{A_2 E_2} + \frac{L_3}{A_3 E_3} + \dots \right]$$

Example



is subjected to a tensile load of 160kN, determine the total elongation of the component

If $E = 200 \text{ GPa}$.

Soln

$$\delta L = \frac{P}{E} \left[\frac{L_1}{A_1} + \frac{L_2}{A_2} \right]$$

$$\delta L = \frac{160,000}{200 \times 10^9} \left[\frac{90}{50} + \frac{120}{100} \right]$$

$$= \frac{16}{20} \left[\frac{9}{5} + \frac{12}{10} \right]$$

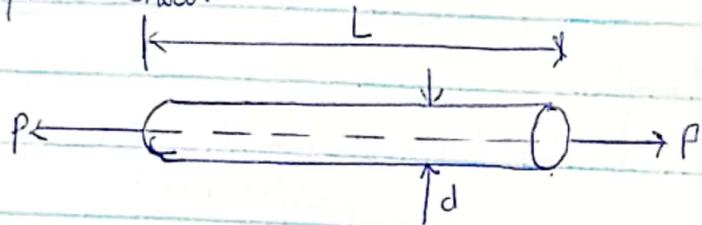
$$0.8(3)$$

$$\delta L = 2.4 \text{ mm}$$

Poisson Ratio

We have already discussed that whenever some external force acts on a body it undergoes deformation.

Consider a circular bar subjected to tensile force as shown:



L = length of the bar

d = diameter of the bar

ΔL = increase in length of the bar as a result of tensile force.

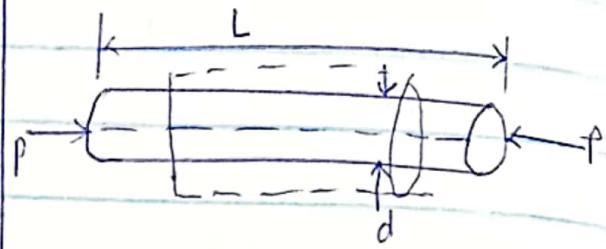
When there is tensile force (pull) on a material the length increases but cross sectional area of the material decreases (Decrease of diameter).

The total deformation of the bar per unit length in the direction of the force i.e. $\frac{\Delta L}{L}$ is known as

Primary strain or linear strain or longitudinal strain. If we actually studied the deformation of the bar, you will find that the bar has extended through length ΔL which will be followed by decrease of diameter (D) extension of the bar

$$\text{from } L \rightarrow L + \Delta L$$

diameter of the bar has been decreased from $d \rightarrow d - \Delta d$.



Similarly, if the bar is subjected to compressive force.

Compression of the bar from $L \rightarrow L - \Delta L$

diameter of the bar has been increased from $d \rightarrow d + \Delta d$.

It is obvious that every direct stress always accompanied by a strain in its direction and an opposite kind of strain in every direction at right angles to it. Such strain is known as secondary or lateral strain.

The ratio of lateral strain to linear strain is always constant

$$\frac{\text{Lateral strain}}{\text{Linear strain}} = \text{Constant}$$

The constant is known as Poisson ratio.
Alternatively,

$$f = \frac{\text{stress}}{\text{strain}} = \frac{P}{A} \frac{L}{\delta L}$$

$$E = \frac{PL}{A \delta L}$$

$$\delta L = \frac{PL}{AE}$$

$$\frac{\text{Secondary strain}}{\text{Primary strain}} = \frac{\text{lateral strain}}{\text{Linear strain (longitudinal)}} = M$$

Example:

A steel bar 2m long, 40mm wide and 20mm thick, is subjected to an axial pull of 160kN in the direction of its length, find the changes in length, width and thickness of the bar. Take $E = 200\text{GPa}$ and Poisson ratio = 0.3.

$$(1) \delta L = \frac{PL}{AE}$$

Ans

$$\text{Strain } \epsilon = \frac{\delta L}{L} = \text{linear strain}$$

$$M = \frac{\text{lateral strain}}{\text{Linear strain}}$$

(2) Change width

$$\delta b = b \times \text{lateral strain}$$

(3) Change in thickness

$$\delta t = t \times \text{lateral strain}$$

* * * Note

$\text{Area} = b \times t$ because length has already been considered

$$\text{GPa} \rightarrow \text{N/mm}^2 = \left(\frac{200 \times 10^9}{10^6} \right) \text{N/mm}^2 = 200 \times 10^3 \text{N/mm}^2$$

Young Modulus = Modulus of elasticity

CVE 204 (Strength of Materials)

Poisson Ratio Contd.

Example 1:

A steel bar 2m long, 40mm wide and 20mm thick is subjected to an axial pull of 160kN in the direction of its length. Find the changes in length, width and thickness of the bar. Take $E = 200\text{GPa}$ ($E = \text{modulus of elasticity}$) and poisson ratio = 0.3.

Soln

$$\text{Length } L = 2\text{m} = 2000\text{mm} (2 \times 10^3 \text{mm})$$

$$\text{width } b = 40\text{mm}$$

$$\text{Thickness } t = 20\text{mm}$$

$$\text{Axial pull} = 160\text{kN} = 160 \times 10^3 \text{N}$$

$$\text{Young Modulus } E = 200\text{GPa} = 200 \times 10^9 \text{Pa}$$

$$= 200 \times 10^3 \text{N/mm}^2$$

$$(1) \text{Change in length } \delta L \cdot \delta L = \frac{PL}{AE}$$

$$= \frac{(160 \times 10^3) \times (2 \times 10^3)}{(40 \times 20) \times 200 \times 10^3} = 2\text{mm}$$

$$\delta L = 2\text{mm}$$

To find width, we need linear strain

$$\text{Linear strain } \epsilon = \frac{\delta L}{L} = \frac{2}{2 \times 10^3}$$

$$\epsilon = 0.001$$

Lateral strain = Poisson's ratio \times linear strain

$$\text{Lateral strain} = \mu \varepsilon$$

$$= 0.3 \times 0.001$$

$$= 0.0003$$

(ii) Change in width $\Delta b = b \times \text{lateral strain}$

$$= 40 \times 0.0003$$

$$= 0.012\text{mm}$$

(iii) Change in thickness $\Delta t = t \times \text{lateral strain}$

$$= 20 \times 0.003$$

$$= 0.06\text{mm}$$

Example 2: A metal bar 50mm by 50mm in section is subjected to an axial compressive load of 500kN. If the contraction of 200mm gauge length was found to be 0.5mm and the increase in thickness 0.04mm find the young modulus and the poisson ratio for the bar material.

Soh

$$\text{Width } (b) = 50\text{mm}$$

$$\text{thickness } (t) = 50\text{mm}$$

$$\text{Axial compressive load } (P) = 500\text{kN}$$

$$= 500 \times 10^3 \text{N}$$

$$\text{Length } L = 200\text{mm}$$

$$\Delta \text{length } \Delta L = 0.5\text{mm}$$

$$\Delta \text{thickness } \Delta t = 0.04\text{mm}$$

$$E = ?$$

$$\mu = ?$$

(i) Young Modulus $E = \frac{PL}{A\delta L}$

$$E = (500 \times 10^3) \times \frac{200}{(50 \times 50) \times 0.5}$$

$$E = 80 \times 10^3 \text{ N/mm}^2$$

(ii) Poisson's ratio (μ)

$$\text{linear strain } \varepsilon = \frac{\Delta L}{L}$$

$$= \frac{0.5}{200} = 0.0025$$

$$\text{lateral strain} = \mu \times \varepsilon$$

Recall that $\Delta t = t \times \text{lateral strain}$

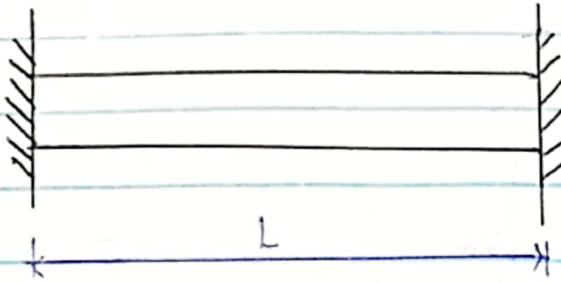
$$\therefore \text{lateral strain} = \frac{0.04}{50} = 0.0008$$

$$\mu = \frac{\text{lateral strain}}{\text{linear strain}} = \frac{0.0008}{0.0025} = 0.32$$

Temperature stresses

Materials expand or contract with rise or fall in temperature. However, if this expansion or contraction is wholly or partially resisted, stress set up in the body. Consider a bar of length L restrained (fixed) at both ends undergoing variation

in temperature.



Let L = Original length of the bar

t = Increase in temperature

α = Coefficient of linear expansion

The increase in length due to increase in temperature is given as

$$\delta L = L \cdot \alpha \cdot t$$

If the ends of the bar are fixed to rigid support so that its expansion is prevented, then compressive strain induced in the bar.

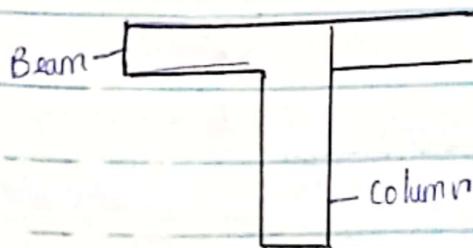
Compressive strain:

$$\epsilon = \frac{\delta L}{L} = \frac{L \cdot \alpha \cdot t}{L}$$

$$\epsilon = \alpha t$$

Strain $\sigma = E \epsilon$

$$\sigma = \alpha t E$$



Example:

A rod is 2m long at a temperature of 10°C . Find the expansion of the rod when the temperature is raised to 80°C . If the expansion is prevented, then find the stress in the material of the rod ($E = 100\text{GPa}$, $\alpha = 0.000012/\text{ }^\circ\text{C}$)

linear expansivity

solution.

Given: Length (L) = 2m = $2 \times 10^3 \text{ mm}$

$$\text{Temp } t = 80 - 10 = 70^\circ\text{C}$$

Expansion of the rod

$$\delta L = L \cdot \alpha \cdot t$$

$$\delta L = 2 \times 10^3 \times 0.000012 \times 70 \\ = 1.68 \text{ mm}$$

$$\sigma = \alpha t E$$

$$= 0.000012 \times 70 \times 100 \times 10^3 \\ = 84 \text{ N/mm}^2$$

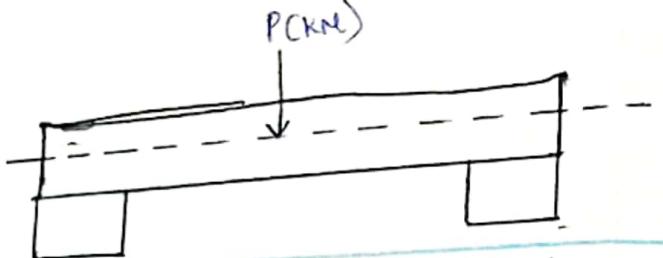
$$= 84 \text{ MPa.}$$

Shear Force and Bending Moment

General Background

Beam

Beam is a structural member loaded perpendicular to its longitudinal axis.

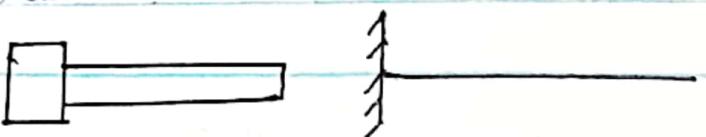


Beams transmit loads by the development of bending moment (BM) and shear force (SF) at different sections - Beams can be classified in many ways

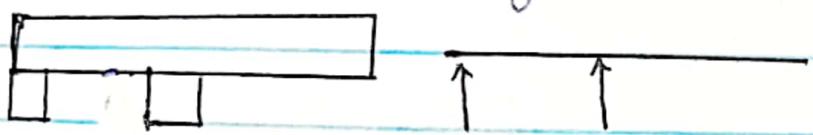
① Simply supported beam (SSB)



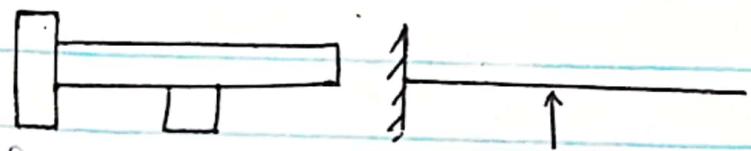
② Cantilever beam



③ Simply supported with overhangs



④ Propped cantilever

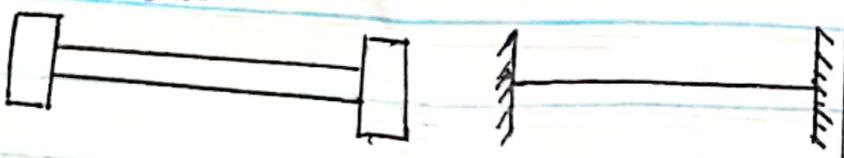


⑤ Continuous beam



Horizontal structural member is a beam.

⑥ Fixed beam

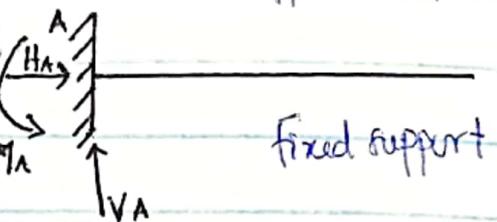


Types of support in Beam structures

Many forms of loads are dealt with in structural analysis.

The support in beams (fixed structure) supply the necessary reactive forces to maintain the structure in equilibrium as a result of applied load. There are many types of support from which a system of supports can result. Some of these supports are

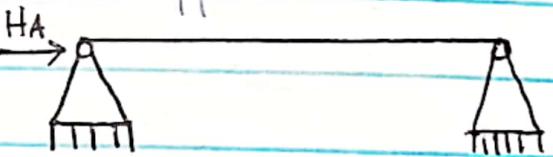
(1) Fixed and support / bulk In / Encastre



fixed support

Three supports are capable of supplying three reactive forces : Horizontal (HA), Vertical (VA), and a fixing Moment (MA). This type of support is fixed so that it cannot move or rotate under the action of superimposed load.

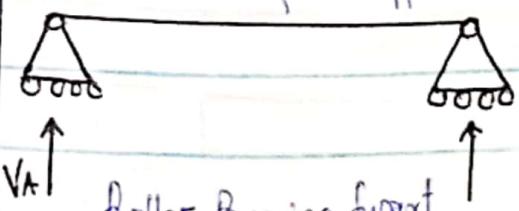
(2) Pin support.



Pinned support

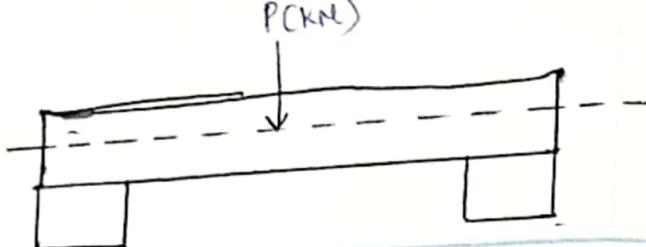
This is assumed to be free to rotate under the applied load but cannot move either vertically or horizontally. This type of support can produce two support reaction. These are Horizontal HA and Vertical VA forces.

(3) Roller / Rocker support.



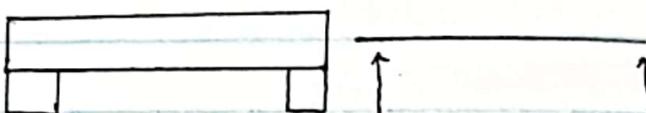
Roller Bearing Support

This can only supply one vertical reaction force. This is the case when a beam simply rests on a roller.



Beams transmit loads by the development of bending moment (BM) and shear force (SF) at different sections. Beams can be classified in many ways.

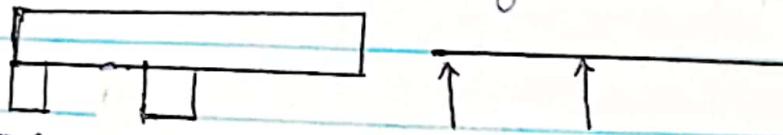
① Simply supported beam (SSB)



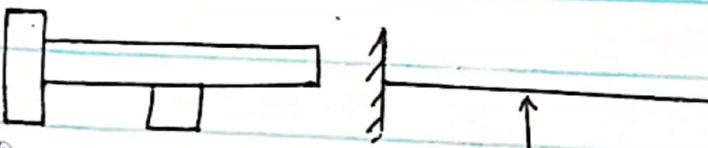
② Cantilever beam



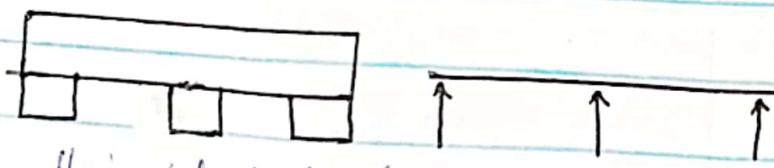
③ Simply supported with overhangs



④ Propped cantilever

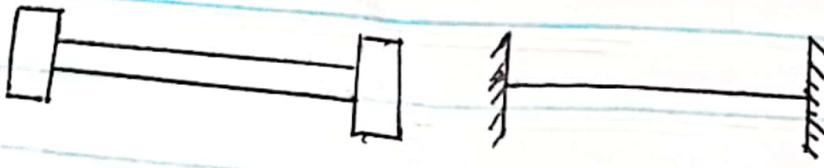


⑤ Continuous beam



Horizontal structural member is a beam.

⑥ Fixed beam

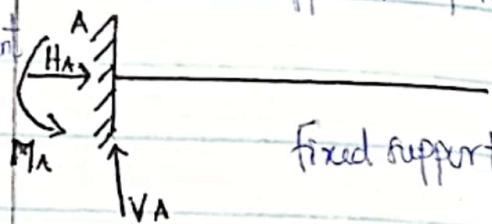


Types of support in Beam structure

Many forms of loads are dealt with in structural analysis.

The support in beams (formed structure) supply the necessary reactive forces to maintain the structure in equilibrium as a result of applied load. There are many types of support from which a system of support can result. Some of these supports are:

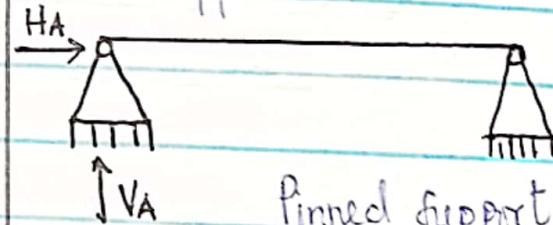
(1) Fixed and support / bulk in / Encastre



fixed support

Three supports are capable of supplying three reactive forces: Horizontal (HA), Vertical (VA), and a fixing Moment (MA). This type of support is fixed so that it cannot move or rotate under the action of superimposed load.

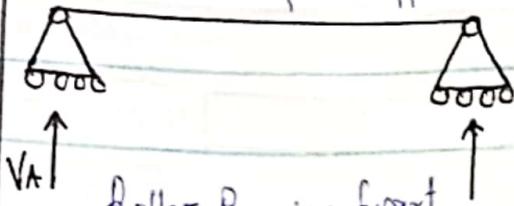
(2) Pin support.



Pinned support

This is assumed to be free to rotate under the applied load but cannot move vertically or horizontally. This type of support can produce two support reaction forces: Horizontal HA and Vertical VA forces.

(3) Roller / Rocker support.



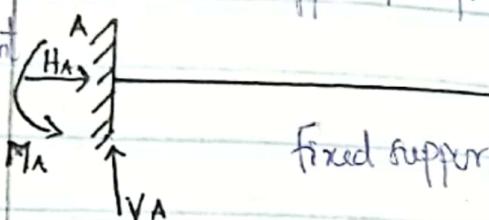
Roller Bearing support

This can only supply one vertical reaction force. This is the case when a beam simply rests on a surface.

The support in beams (formed structure) supply the necessary reactive forces to maintain the structure in equilibrium as a result of applied load. There are many types of support from which a system of supports can result. Some of these supports are

support, the beam can rotate, it can move horizontally but it is restrained in the vertical in the vertical direction only. When we sit down on a chair this is the type of support that is developed.

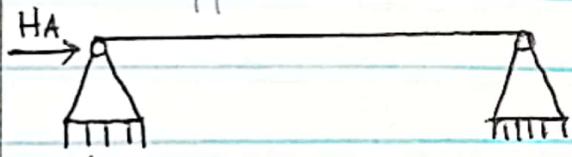
f bent (1) Fixed and support / bulk in / Encastive



fixed support

Three supports are capable of supplying three reactive forces: Horizontal (H_A), Vertical (V_A), and a fixing Moment (M_A). This type of support is fixed so that it cannot move or rotate under the action of superimposed load.

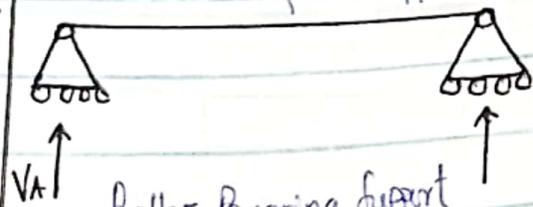
(2) Pin support.



Pinned support

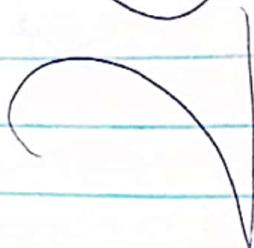
This is assumed to be free to rotate under the applied load but cannot move either vertically or horizontally. This type of support can produce two support reaction. These are Horizontal H_A and Vertical V_A forces.

(3) Roller / Rocker support.



Roller Bearing support

This can only supply one vertical reaction V_A . This is the case when a beam simply rests on a



Engineering drawing