

## Lecture 3

Transformation of stresses in 2D.



$$\sigma_n = \sigma_x \sin^2 \theta + \sigma_y \cos^2 \theta + 2\tau_{xy} \sin \theta \cos \theta$$

$$\tau_n = -\frac{\sigma_x \sin 2\theta}{2} + \frac{\sigma_y \sin 2\theta}{2} + \tau_{xy} \cos 2\theta$$

Force equilibrium:

$$\begin{aligned} \sigma_n(A) &= (\sigma_x A \cos \theta) \cos \theta + (\sigma_y A \sin \theta) \sin \theta \\ &\quad + (\tau_{xy} A \sin \theta) \cos \theta + (\tau_{xy} A \cos \theta) \sin \theta \end{aligned}$$

$$\sigma_n = \sigma_x \cos^2 \theta + \sigma_y \sin^2 \theta + 2\tau_{xy} \sin \theta \cos \theta$$

$$\begin{aligned} \tau_n(A) &= (\sigma_y A \sin \theta) \cos \theta + (\tau_{xy} A \sin \theta) \sin \theta \\ &\quad + (\sigma_x A \cos \theta) \sin \theta - (\tau_{xy} A \cos \theta) \cos \theta \end{aligned}$$

$$\sigma_{x'y'} = (\bar{\sigma}_y - \sigma_u) \sin \theta \cos \theta + \tau_{xy} (\cos^2 \theta - \sin^2 \theta)$$

$$\begin{bmatrix} \sigma_u' \\ \sigma_y' \\ \tau_{xy}' \end{bmatrix} = \begin{bmatrix} c^2 & s^2 & 2sc \\ s^2 & c^2 & -2sc \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \sigma_u \\ \sigma_y \\ \tau_{xy} \end{bmatrix}$$

$$\sigma_u' = \frac{\sigma_u + \sigma_y}{2} + \frac{\sigma_u - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_y' = \frac{\sigma_u + \sigma_y}{2} - \frac{\sigma_u - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{xy} = -\frac{\sigma_u - \sigma_y}{2} \sin 2\theta + \tau_{12} \cos 2\theta$$

Principal Stresses:

- ↳ no shear stress, maximum normal stress
- ↳ For any state of stress it is always possible to define

! . .

$$\tau_{xy} = 0$$

$$\tau_{xy} (\cos^2 \theta - \sin^2 \theta) + (\sigma_x - \sigma_y) \sin \theta \cos \theta = 0$$

$$\tan 2\theta = \frac{2\tau_{xy}}{\sigma_x - \sigma_y}$$

$\tan 2\theta = \tan(\pi + 2\theta)$  : 2 angles obtained  
90° apart.

↳ Hence two principal planes  $\theta$  &  $90 - \theta$ ,  
from original coordinates  $(x, y, z)$

$$\sigma_{max} = \sigma_1 \left\{ = \frac{\sigma_x + \sigma_y}{2} \pm \left[ \left( \frac{\sigma_x - \sigma_y}{2} \right)^2 + \tau_{xy}^2 \right]^{1/2} \right\}$$

$$\sigma_{min} = \sigma_2$$

To find plane for maximum shear stress:

$$\frac{d\tau_{xy}}{d\theta} = 0$$

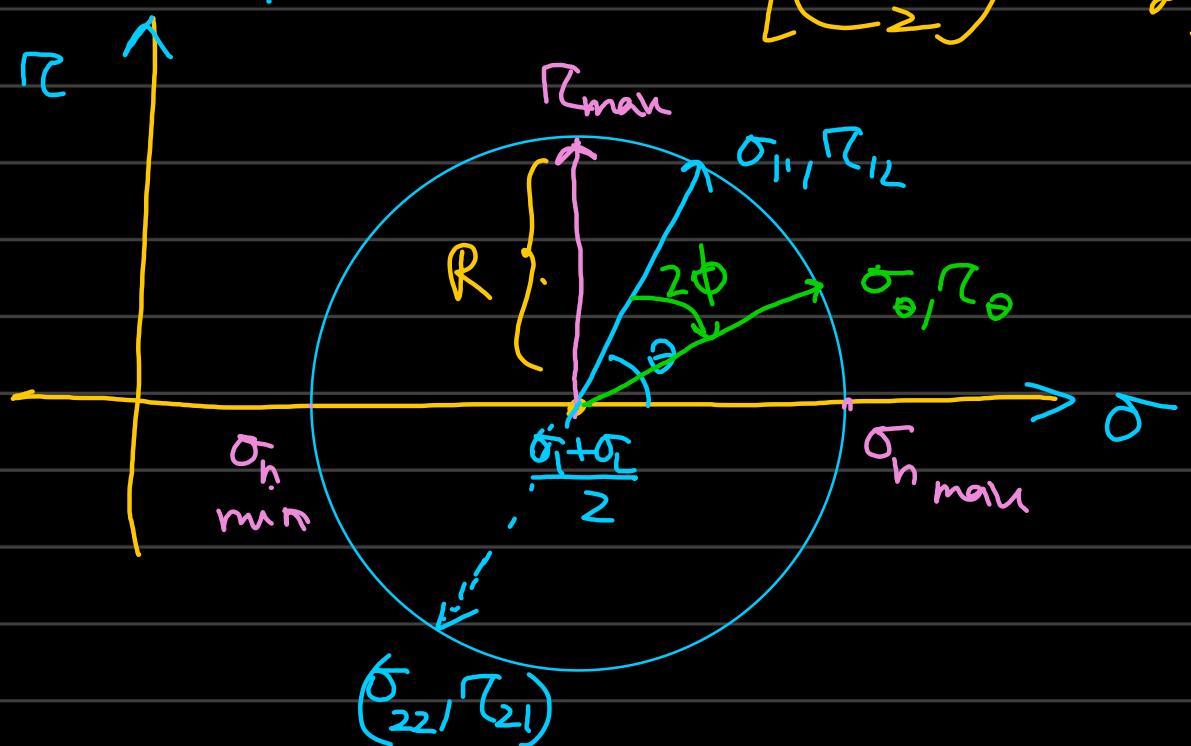
$$= (\sigma_y - \sigma_x) \cos 2\theta - 2\tau_{xy} \sin 2\theta = 0$$

$$\tan 2\theta = -\frac{(\sigma_y - \sigma_x)}{2\tau_{xy}}$$

$$\tau_{max} = \pm \left[ \left( \frac{\sigma_x - \sigma_y}{2} \right)^2 + (\tau_{xy})^2 \right]^{1/2}$$

*Mohr's Circle:*

$$R = \sqrt{(\sigma_u - \sigma_d)^2 + (\tau_{xy})^2}$$



Principal stresses in 3D.

$$\begin{vmatrix} \sigma - \sigma_x & -\tau_{yx} & -\tau_{zx} \\ -\tau_{xy} & \sigma - \sigma_y & -\tau_{zy} \\ -\tau_{xz} & -\tau_{yz} & \sigma - \sigma_z \end{vmatrix} = 0$$

$$\sigma^3 - \sigma^2 I_1 + \sigma I_2 - I_3 = 0$$

$$I_1 = \sigma_x + \sigma_y + \sigma_z$$

$$I_2 = \sigma_x \sigma_y + \sigma_y \sigma_z + \sigma_z \sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{xz}^2$$

$$\begin{aligned}
 I_3 &= \sigma_u \sigma_y \sigma_z + 2 \tau_{uy} \tau_{yz} \tau_{uz} - \sigma_u \tau_{yz}^2 - \sigma_y \tau_{uz}^2 \\
 &\quad - \sigma_z \tau_{uy}^2 \\
 &= \det(\underline{\Sigma}) \\
 &= \begin{vmatrix} \sigma_{xx} & \tau_{uj} & \tau_{uz} \\ \tau_{uy} & \sigma_{yy} & \tau_{yz} \\ \tau_{uz} & \tau_{yz} & \sigma_{zz} \end{vmatrix}
 \end{aligned}$$

$I_1, I_2, I_3$  are called the stress invariants.

Hydrostatic & Deviatoric stress

$$\sigma'_{ij} = \sigma_{ij} - \underbrace{\sigma_m \delta_{ij}}_{\text{Hydrostatic stress}} \quad \sigma_m = \frac{1}{3} \sigma_{KK}$$

The principal values of deviatoric stress:

$$(\sigma')^3 - J_1(\sigma')^2 - J_2(\sigma') - J_3 = 0$$

$J_1, J_2, J_3$  : deviatoric stress invariants