

# Algorithm for solving structural problems

## 2D bar elements

### 1 Geometry

The following terms are defined

$n_d$  : Problem dimension, i.e.  $n_d = 2$  (2D).

$n_{el}$  : Total number of bars.

$n_{nod}$  : Total number of nodes (joints).

$n_{ne}$  : Number of nodes in a bar, i.e.  $n_{ne} = 2$ .

$n_i$  : Degrees of freedom per node, i.e.  $n_i = 2$ .

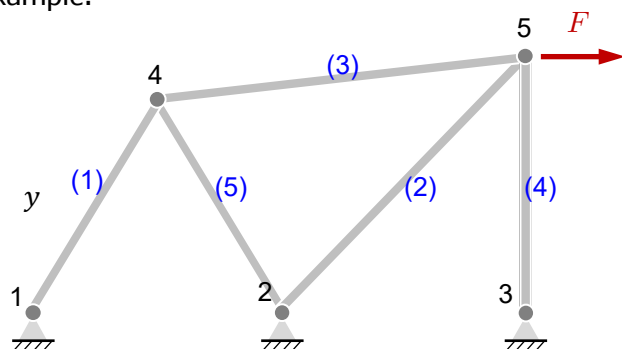
$n_{dof}$  : Total number of degrees of freedom, i.e.  $n_{dof} = n_{nod} \times n_i$ .

$\mathbf{x}$  : Nodal coordinates array ( $n_{nod} \times n_d$ ):

$$\mathbf{x} = \begin{bmatrix} x_1 & y_1 \\ \vdots & \vdots \\ x_n & y_n \end{bmatrix}.$$

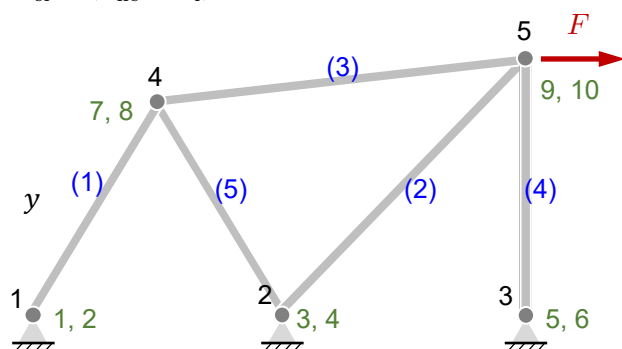
$\mathbf{T}_n$  : Nodal connectivity table ( $n_{el} \times n_{ne}$ ). Example:

$$\mathbf{T}_n = \begin{bmatrix} 1 & 4 \\ 2 & 5 \\ 4 & 5 \\ 3 & 5 \\ 2 & 4 \end{bmatrix}.$$



$\mathbf{T}_d$  : Degrees of freedom connectivity table ( $n_{el} \times (n_{ne} \times n_i)$ ). Example:

$$\mathbf{T}_d = \begin{bmatrix} 1 & 2 & 7 & 8 \\ 3 & 4 & 9 & 10 \\ 7 & 8 & 9 & 10 \\ 5 & 6 & 9 & 10 \\ 3 & 4 & 7 & 8 \end{bmatrix}.$$



## 2 Computation of the element stiffness matrices

For each bar  $e = 1 \dots n_{el}$

*a) Compute element stiffness matrix*

$$x_1^e = \mathbf{x}(\mathbf{T}_n(e, 1), 1)$$

$$y_1^e = \mathbf{x}(\mathbf{T}_n(e, 1), 2)$$

$$x_2^e = \mathbf{x}(\mathbf{T}_n(e, 2), 1)$$

$$y_2^e = \mathbf{x}(\mathbf{T}_n(e, 2), 2)$$

$$l^e = \sqrt{(x_2^e - x_1^e)^2 + (y_2^e - y_1^e)^2}$$

$$s^e = \frac{(y_2^e - y_1^e)}{l^e}$$

$$c^e = \frac{x_2^e - x_1^e}{l^e}$$

$$\mathbf{K}^e = \frac{A^e E^e}{l^e} \begin{bmatrix} (c^e)^2 & c^e s^e & -(c^e)^2 & -c^e s^e \\ c^e s^e & (s^e)^2 & -c^e s^e & -(s^e)^2 \\ -(c^e)^2 & -c^e s^e & (c^e)^2 & c^e s^e \\ -c^e s^e & -(s^e)^2 & c^e s^e & (s^e)^2 \end{bmatrix}$$

*b) Store element matrix*

For each  $r = 1 \dots n_{ne} \times n_i$

For each  $s = 1 \dots n_{ne} \times n_i$

$$\mathbf{K}_{el}(r, s, e) = \mathbf{K}^e(r, s)$$

Next  $s$

Next  $r$

Next bar  $e$

## 3 Global stiffness matrix assembly

Assembly operator:

$$\mathbf{K}_G = \mathbf{A}_{e=1}^{n_{el}} \mathbf{K}^e$$

Initialization:

$$\mathbf{K}_G = \begin{bmatrix} 0 & \dots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \dots & 0 \end{bmatrix}, \quad \text{dimensions: } n_{\text{dof}} \times n_{\text{dof}}$$

For each bar  $e = 1 \dots n_{el}$

For each local degree of freedom  $i = 1 \dots n_{ne} \times n_i$  (rows)

$I = \mathbf{T}_d(e, i)$  (corresponding global degree of freedom)

For each local degree of freedom  $j = 1 \dots n_{ne} \times n_i$  (columns)

$J = \mathbf{T}_d(e, j)$  (corresponding global degree of freedom)

$\mathbf{K}_G(I, J) = \mathbf{K}_G(I, J) + \mathbf{K}_{el}(i, j, e)$

Next  $j$

Next  $i$

Next bar  $e$

## 4 Global system of equations

Global system:

$$\mathbf{K}_G \hat{\mathbf{u}} = \hat{\mathbf{F}}^{\text{ext}} + \hat{\mathbf{R}}$$

Compute external forces vector:

$\hat{\mathbf{F}}^{\text{ext}} = [\dots]$  : Array with the applied external forces.

Apply conditions:

$\hat{\mathbf{u}}_R = [\dots]$  : Array with the displacements on the imposed degrees of freedom.

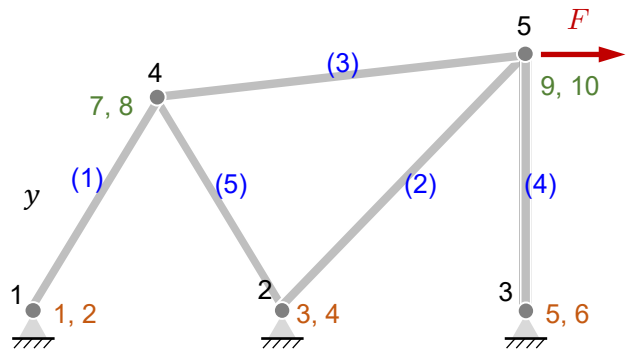
$\nu_R = [\dots]$  : Array with the imposed degrees of freedom.

$\nu_L = [\dots]$  : Array with the free degrees of freedom.

\* In the previous example:

$$\hat{\mathbf{F}}^{\text{ext}} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ F \\ 0 \end{bmatrix},$$

$$\hat{\mathbf{u}}_R = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}, \quad \nu_R = \begin{bmatrix} 1 \\ 2 \\ 3 \\ 4 \\ 5 \\ 6 \end{bmatrix}, \quad \nu_L = \begin{bmatrix} 7 \\ 8 \\ 9 \\ 10 \end{bmatrix}.$$



Partitioned system of equations:

$$\begin{aligned} \mathbf{K}_{LL} &= \mathbf{K}_G(\nu_L, \nu_L) \\ \mathbf{K}_{LR} &= \mathbf{K}_G(\nu_L, \nu_R) \\ \mathbf{K}_{RL} &= \mathbf{K}_G(\nu_R, \nu_L) \\ \mathbf{K}_{RR} &= \mathbf{K}_G(\nu_R, \nu_R) \\ \hat{\mathbf{F}}_L^{\text{ext}} &= \hat{\mathbf{F}}^{\text{ext}}(\nu_L, 1) \\ \hat{\mathbf{F}}_R^{\text{ext}} &= \hat{\mathbf{F}}^{\text{ext}}(\nu_R, 1) \end{aligned}$$

$$\begin{bmatrix} \mathbf{K}_{RR} & \mathbf{K}_{RL} \\ \mathbf{K}_{LR} & \mathbf{K}_{LL} \end{bmatrix} \begin{bmatrix} \hat{\mathbf{u}}_R \\ \hat{\mathbf{u}}_L \end{bmatrix} = \begin{bmatrix} \hat{\mathbf{F}}_R^{\text{ext}} \\ \hat{\mathbf{F}}_L^{\text{ext}} \end{bmatrix} + \begin{bmatrix} \hat{\mathbf{R}}_R \\ \mathbf{0} \end{bmatrix}$$

Data:

$$\begin{aligned} \hat{\mathbf{u}}_R &: \text{Imposed displacement vector} \\ \hat{\mathbf{F}}^{\text{ext}} &: \text{External force vector} \end{aligned}$$

Unknowns:

$$\begin{aligned} \hat{\mathbf{u}}_L &: \text{Free displacement vector} \\ \hat{\mathbf{R}}_R &: \text{Reactions vector} \end{aligned}$$

System resolution:

$$\begin{aligned} \mathbf{K}_{LL} \hat{\mathbf{u}}_L &= \hat{\mathbf{F}}_L^{\text{ext}} - \mathbf{K}_{LR} \hat{\mathbf{u}}_R \rightarrow \hat{\mathbf{u}}_L = \mathbf{K}_{LL}^{-1} (\hat{\mathbf{F}}_L^{\text{ext}} - \mathbf{K}_{LR} \hat{\mathbf{u}}_R) \\ \hat{\mathbf{R}}_R &= \mathbf{K}_{RR} \hat{\mathbf{u}}_R + \mathbf{K}_{RL} \hat{\mathbf{u}}_L - \hat{\mathbf{F}}_R^{\text{ext}} \end{aligned}$$

Obtain generalized displacement vector:

$$\begin{aligned} \hat{\mathbf{u}}(\nu_L, 1) &= \hat{\mathbf{u}}_L \\ \hat{\mathbf{u}}(\nu_R, 1) &= \hat{\mathbf{u}}_R \end{aligned}$$

## 5 Strains and stresses

Strains and stresses for each bar can be computed by means of the following procedure:

For each bar  $e = 1 \dots n_{el}$

a) *Compute the rotation matrix*

$$x_1^e = \mathbf{x}(\mathbf{T}_n(e, 1), 1)$$

$$y_1^e = \mathbf{x}(\mathbf{T}_n(e, 1), 2)$$

$$x_2^e = \mathbf{x}(\mathbf{T}_n(e, 2), 1)$$

$$y_2^e = \mathbf{x}(\mathbf{T}_n(e, 2), 2)$$

$$l^e = \sqrt{(x_2^e - x_1^e)^2 + (y_2^e - y_1^e)^2}$$

$$s^e = \frac{(y_2^e - y_1^e)}{l^e}$$

$$c^e = \frac{(x_2^e - x_1^e)}{l^e}$$

$$\mathbf{R}^e = \begin{bmatrix} c^e & s^e & 0 & 0 \\ -s^e & c^e & 0 & 0 \\ 0 & 0 & c^e & s^e \\ 0 & 0 & -s^e & c^e \end{bmatrix}$$

*b) Obtain element displacement in global coordinates*

For each local degree of freedom  $i = 1 \dots n_{ne} \times n_i$

$$I = \mathbf{T}_d(e, i)$$

$$\hat{\mathbf{u}}^e(i, 1) = \hat{\mathbf{u}}(I)$$

Next  $i$

*c) Compute element displacement in local coordinates*

$$\hat{\mathbf{u}}^{e'} = \mathbf{R}^e \hat{\mathbf{u}}^e$$

*d) Compute element strain*

$$\hat{\varepsilon}^e = \frac{1}{l^e} [-1 \quad 0 \quad 1 \quad 0] \hat{\mathbf{u}}^{e'}$$

*e) Compute element stress*

$$\hat{\sigma}^e = E^e \hat{\varepsilon}^e$$

Next bar  $e$