82 Refrigeration and Air Conditioning

Enthalpies at 1 and 4

$$h_1 = 8.9 + 0.951 (174.2 - 8.9) = 166 \text{ kJ/kg}$$

 $h_4 = 8.9 + 0.322 (174.2 - 8.9) = 62.1 \text{ kJ/kg}$

Work of compression,
$$w_C = h_2 - h_1 = 201.5 - 166 = 35.5 \text{ kJ/kg}$$
 Work of expansion, $w_E = h_3 - h_4 = 69.5 - 62.1 = 7.4 \text{ kJ/kg}$ Refrigerating effect, $q_o = h_1 - h_4 = 166 - 62.1 = 103.9 \text{ kJ/kg}$ Heat rejected, $q_k = h_2 - h_3 = 201.5 - 69.5 = 132 \text{ kJ/kg}$ Net work, $w = w_C - w_E = 35.5 - 7.4 = 28.1 \text{ kJ/kg}$ $= q_k - q_o = 132 - 103.9 = 28.1 \text{ kJ/kg}$

COP of the cycle,
$$\mathcal{E}_{c} = \frac{q_{o}}{w} = \frac{103.9}{28.1} = 3.74$$

Alternatively, we have for Carnot COP

$$(\mathcal{E}_c)$$
 Carnot = $\frac{T_o}{T_k - T_o} = \frac{273 - 30}{35 - (-30)} = 3.74$

(b) Actual COP, $\mathcal{E}_c = 0.75 \times 3.74 = 2.8$

Power consumption per ton,
$$\dot{W} = \frac{\dot{Q}_o}{\mathcal{E}_c} = \frac{3.5167}{2.8} = 1.256 \text{ kW}$$

Heat rejected per ton, $\dot{Q}_k = \dot{Q}_0 + \dot{W} = 3.516 + 1.256 = 4.722 \text{ kW}$

2.6 GAS AS A REFRIGERANT IN REVERSED CARNOT CYCLE

Figure 2.15 shows the Carnot cycle 1-2-3-4 with gas as a refrigerant, illustrated on T-s and p-v diagrams.

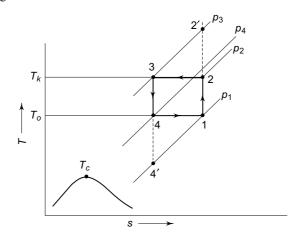


Fig. 2.15(a) Reversed Carnot cycle with gas as a refrigerant on T-s diagram

The four processes of the cycle are analysed as non-flow processes as follows in Eqs. (2.6), (2.7), (2.8) and (2.9).

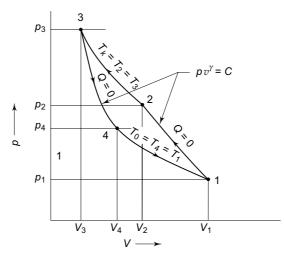


Fig. 2.15(b) Reversed Carnot cycle with gas as a refrigerant on p-v diagram

1–2 Isentropic compression: Q = 0

Pressure increases from p_1 to p_2

Specific volume reduces from v_1 to v_2

Temperature increases from $T_o = T_1$ to $T_k = T_2$

Work done,
$$|w_{1-2}| = \frac{p_2 v_2 - p_1 v_1}{\gamma - 1} = \frac{R(T_k - T_o)}{\gamma - 1}$$
 (2.6)

2–3 Isothermal compression and heat rejection: $T_2 = T_3 = T_k$

Pressure increases from p_2 to p_3

Specific volume reduces from v_2 to v_3

Work done,
$$|w_{2-3}| = p_2 v_2 \ln \frac{v_2}{v_3} = RT_k \ln \frac{v_2}{v_3}$$
 (2.7)

Heat rejected, $q_k = q_{2-3} = w_{2-3}$ (for a perfect gas in an isothermal process)

3–4 Isentropic expansion: Q = 0

Pressure falls from p_3 to p_4

Specific volume increases from v_3 to v_4

Temperature decreases from $T_k = T_3$ to $T_o = T_4$

Work done,
$$w_{3-4} = \frac{p_3 v_3 - p_4 v_4}{\gamma - 1} = \frac{R (T_k - T_o)}{\gamma - 1}$$
 (2.8)

4–1 Isothermal expansion and heat absorption: $T_4 = T_1 = T_o$

Pressure falls from p_4 to p_1

Specific volume increases from v_4 to v_1

The McGraw-Hill Companies

84 Refrigeration and Air Conditioning

Work done,
$$w_{4-1} = p_4 v_4 \ln \frac{v_1}{v_4} = RT_o \ln \frac{v_1}{v_4}$$
 (2.9)

Refrigerating effect, $q_o = q_{4-1} = w_{4-1}$ (for a perfect gas)

Net work of the cycle.

$$w = |w_{2-3}| - w_{4-1} = RT_k \ln \frac{v_2}{v_2} - RT_o \ln \frac{v_1}{v_4}$$
 (2.10)

Refrigerating effect,
$$q_o = q_{4-1} = RT_o \ln \frac{v_1}{v_4}$$
 (2.11)

Now for the isentropic processes 1-2 and 3-4

$$\frac{T_k}{T_o} = \left(\frac{p_2}{p_1}\right)^{\frac{\gamma-1}{\gamma}} = \left(\frac{p_3}{p_4}\right)^{\frac{\gamma-1}{\gamma}}$$

and

$$\frac{T_k}{T_o} = \left(\frac{v_1}{v_2}\right)^{\gamma - 1} = \left(\frac{v_4}{v_3}\right)^{\gamma - 1} = r^{\gamma - 1}$$

where r is the compression ratio for the isentropic processes.

Hence
$$\frac{p_2}{p_1} = \frac{p_3}{p_4}$$
, and $r = \frac{v_1}{v_2} = \frac{v_4}{v_3}$
or $\frac{v_2}{v_3} = \frac{v_1}{v_4} = \frac{p_3}{p_2} = \frac{p_4}{p_1}$
since $p_2v_2 = p_3v_3$ and $p_4v_4 = p_1v_1$

We have, for the COP for cooling

$$\mathcal{E}_{c} = \frac{q_{o}}{w} = \frac{RT_{o} \ln \frac{v_{1}}{v_{4}}}{RT_{k} \ln \frac{v_{2}}{v_{3}} - RT_{o} \ln \frac{v_{1}}{v_{4}}}$$

$$= \frac{T_{o}}{T_{k} - T_{o}}$$

$$= \frac{1}{\frac{T_{k}}{T_{o}} - 1} = \frac{1}{r^{\gamma - 1} - 1}$$
(2.12)

Thus COP is a function of compression ratio only.

2.7 LIMITATIONS OF REVERSED CARNOT CYCLE

It is found that serious practical difficulties are encountered in the application of Carnot cycle.