

Design Calculations for springs in the Suspension of speed Breaker

Assumptions:

Mass of vehicle(m)=1150kg

Acceleration due to gravity(g)=9.81m/s²

Material of Spring=Alloy Steel (Ultimate tensile strength=723.82MPa)

Factor of safety=1.5

spring index(C) as 6

Calculation of Force on each spring(p)

Total force acting on bumper=mass of vehicle X acceleration due to gravity

$$P = m \times g = 1150 \times 9.81 = 11281.5N$$

Total number of parallel connected springs =4

$$\text{Force on each spring} = \frac{P}{4} = \frac{11281.5}{4} = 2820.375N$$

Hence Force on each spring is 2820.375N

Calculation of Maximum shear stress (τ_{\max})

$$\tau_{\max} = 0.5S_{ut}$$

$$\tau_{\max} = 0.5 \times 723.82 = 361.91MPa$$

Hence Maximum Shear stress for alloy steel is 361.91Mpa

Calculation of Maximum permissible shear stress ($\tau_{allowable} = \tau$)

$$\tau_{\max} = 361.91MPa$$

F.O.S=1.5

$$\tau_{allowable} = \frac{\tau_{\max}}{F.O.S} = \frac{361.91}{1.5} = 241.27MPa$$

Hence Maximum allowed stress for our speed breaker is 241.27Mpa

Calculation of Wahl factor(kw)

Spring index(C)=6

$$k_w = \frac{4C-1}{4C-4} + \frac{0.615}{C}$$

$$k_w = \frac{(4 \times 6) - 1}{(4 \times 6) - 4} + \frac{0.615}{6} = 1.15 + 0.1025 = 1.2525$$

Hence Wahl factor is 1.2525

Calculation of Wire diameter(d)

$$\tau = \frac{k_w 8pD}{\pi d^3} = \frac{k_w 8pC}{\pi d^2}$$

$$d^2 = \frac{k_w 8 p C}{\pi \tau}$$

$$d^2 = \frac{8 \times 2820.375 \times 6 \times 1.2525}{\pi \times 241.27} = 223.70 \text{ mm}$$

$$d = 14.956 \approx 15 \text{ mm}$$

Hence the wire diameter(d) is 15 mm

Calculation of Mean coil diameter(D)

$$C = \frac{D}{d}$$

$$D = C \times d = 6 \times 15 = 90 \text{ mm}$$

Hence the Spring mean coil diameter(D) is 90 mm

Calculation of Number of turns(N)

Average height of speed breaker in India is 10cm

Individual spring deflection when vehicle passes=10cm=100mm

Shear Modulus of **Alloy Steel**(G)=79000MPa

$$\delta = \frac{8 p D^3 N}{G d^4} = \frac{8 \times 2820.375 \times (90)^3 \times N}{79000 \times (15)^4}$$

$$100 = 4.11 \times N$$

$$N = \frac{100}{4.11} = 24.33 \approx 25 \text{ Turns}$$

For Square and grounded ends, the number of inactive coils is 2

Thus, total number of coils needed is 2

Hence total number of turns of spring required is 27

Calculation of Stiffness of spring(K)

$$K = \frac{G d^4}{8 D^3 N} \text{ where,}$$

Shear Modulus of **Alloy Steel**(G)=79000MPa

wire diameter(d) is 15 mm

mean coil diameter(D) is 90 mm

Total number of turns(N)=27

$$\text{Stiffness}(K) = K = \frac{G d^4}{8 D^3 N} = \frac{79000 \times 15^4}{8 \times 90^3 \times 27} = 27.430 \text{ N / m}$$

Hence Stiffness of Spring is 27.430N/m

Calculation of Solid Length(l) and Free length(L)

Assumption: Under maximum deflection gap between consecutive coils is 3mm

Total axial Gap = (Number of coils-1) x gap between consecutive coils
= (27-1) x 3mm=78mm

Maximum deflection (δ) =100mm

Solid length of spring(l) = (N x d) =27 x 15 = 405mm

Total Spring length (Free length) = Solid length + Total axial Gap + Maximum deflection
=405mm + 78mm + 100mm=583mm

Hence free length of the spring is 583mm or 58.3cm

SPRING DESIGN IN SOLID WORKS

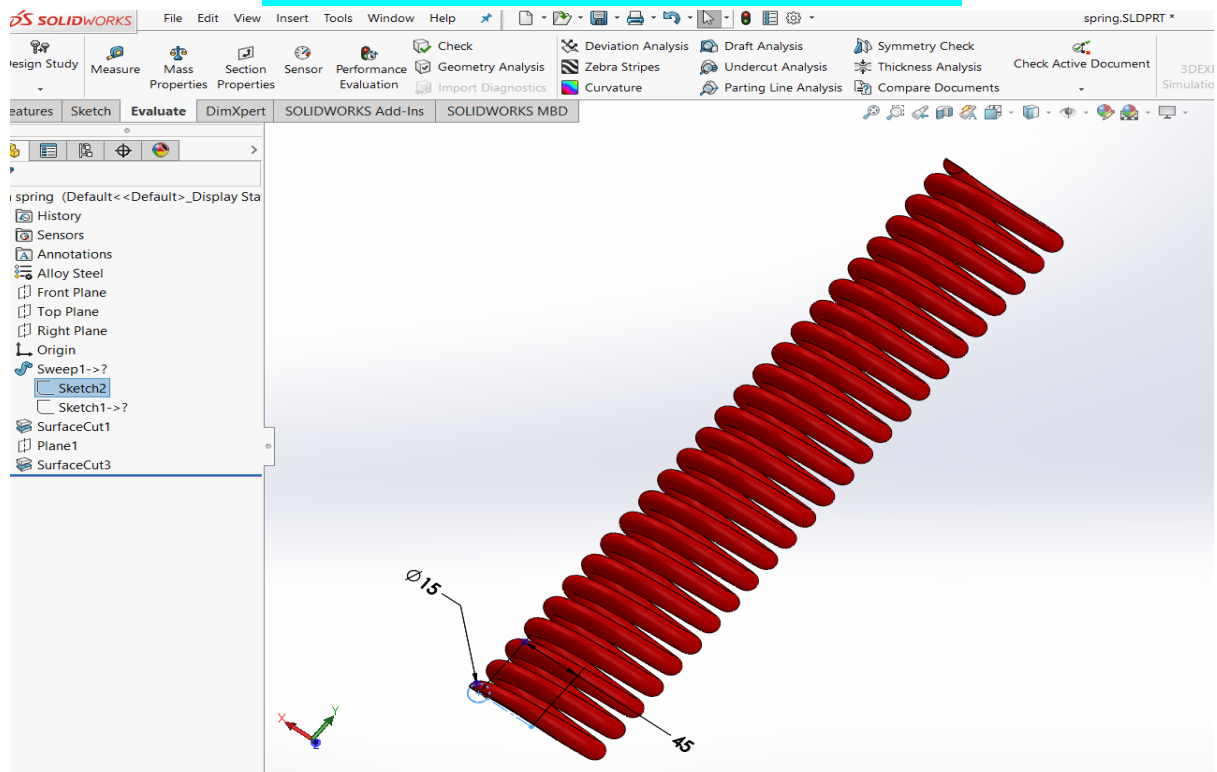


Fig: Spring Mean coil diameter and Wire diameter

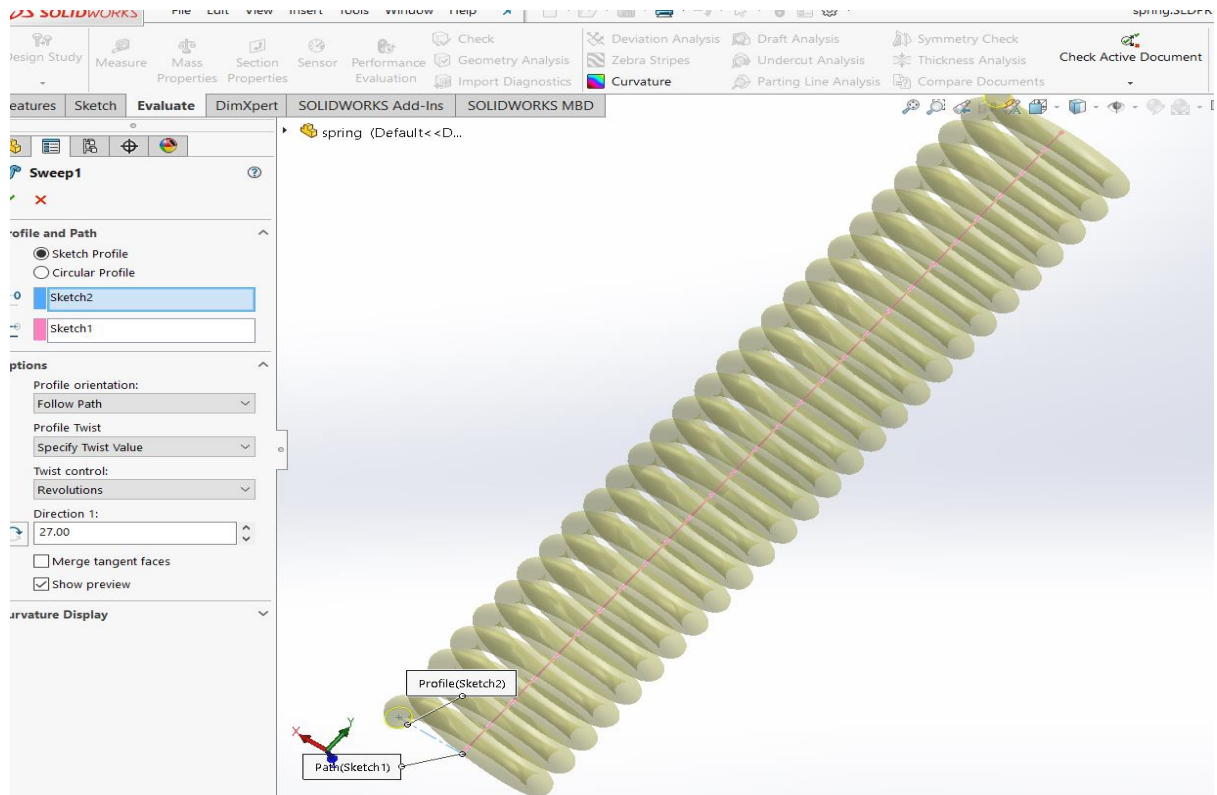


Fig: Spring Total number of coils(N)

SOLIDWORKS PICTURES

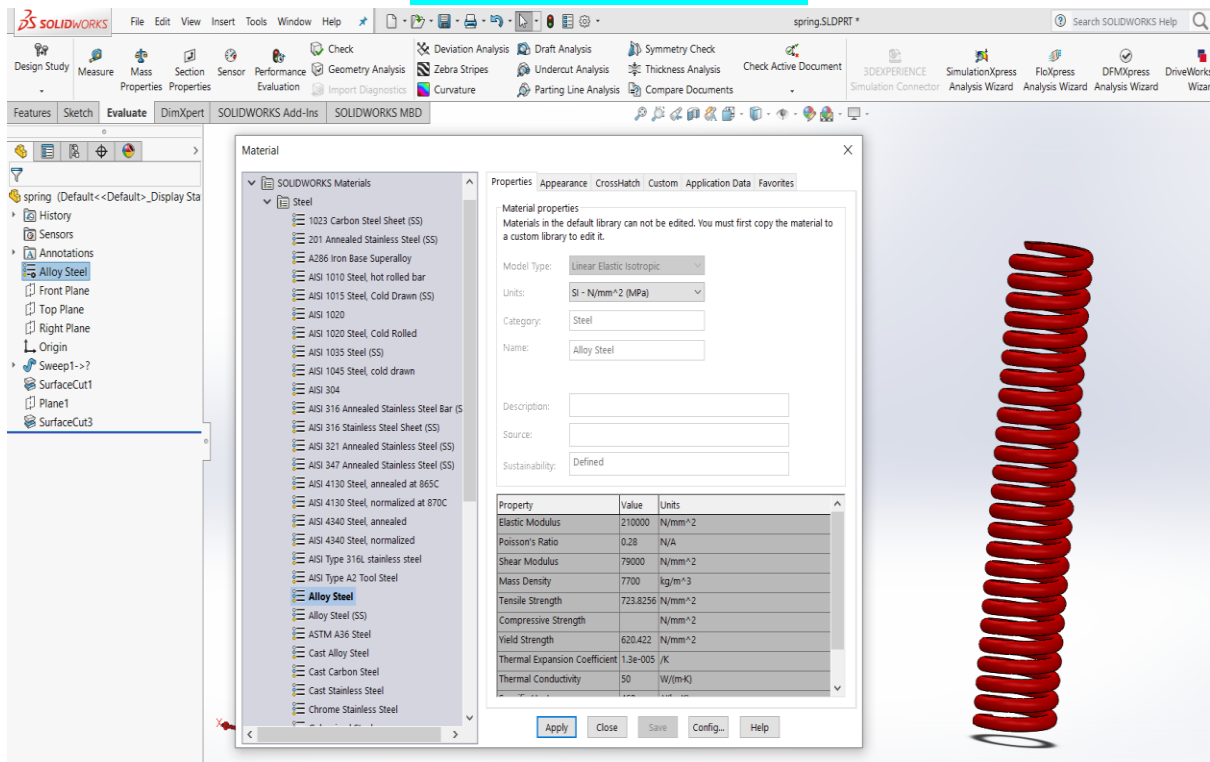


Fig: Mechanical properties of Alloy Steel

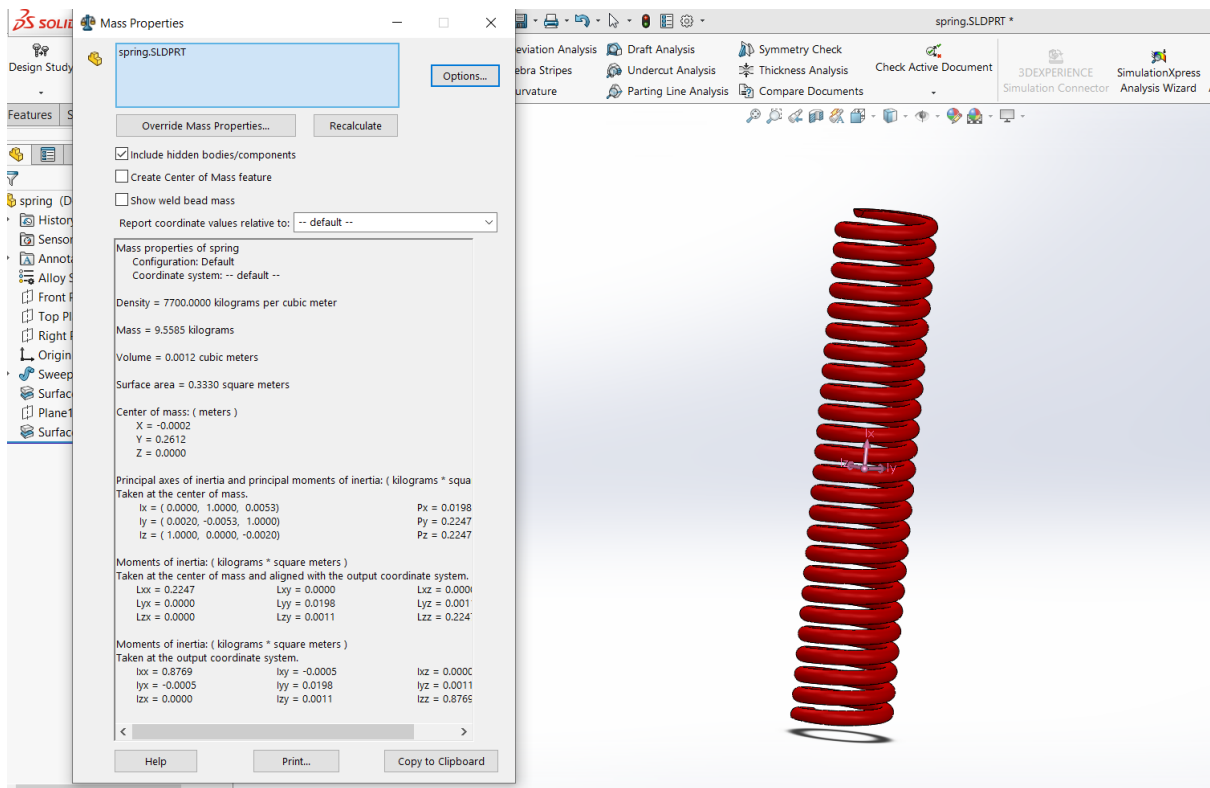


Fig: Mass of the Spring