

# Rectangular PLATE ANALYSIS

## PHASE II

VIBRATIONS SPRING 1404

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THE CURRENT DOCUMENT IS A REPORT OF THE SECOND VIBRATIONS  
PROJECT REGARDING RECTANGULAR PLATE ANALYSIS



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# Phase I: Rectangular Plate Analysis

## Introduction

The current document is a report of the second Vibrations Project, prepared for the class Vibrations- ME 8106-116-0.

Knowledge of ‘**Dynamics**’, ‘**Vibrations**’ and familiarity with ‘**Abaqus**’ is required for understanding this document.

For further insights, the reader can study “Theory of Vibration with Applications” written by William T. Thomson.

The aim of this project is to familiarize students with Abaqus.

ABAQUS

## Model Design

### Plate Creation

As stated by the project descriptions, a  $0.5 \times 1 \text{ m}^2$  plate is designed in the Solid modeling environment under the “Part” section in Abaqus. The width of this plate is set to be 0.01.

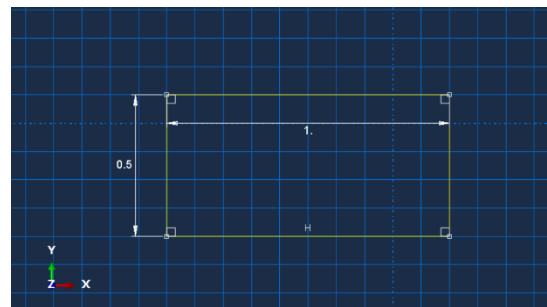


Figure 1. The 2d sketch of the plate before extrusion

### Material Selection

As suggested, the material is set as steel.

Later a surface with this material is defined and matched with the part (the rectangular plate) built in the previous section. Since we are to study the vibrations of this plate, “Frequency” step is used after the initial step.

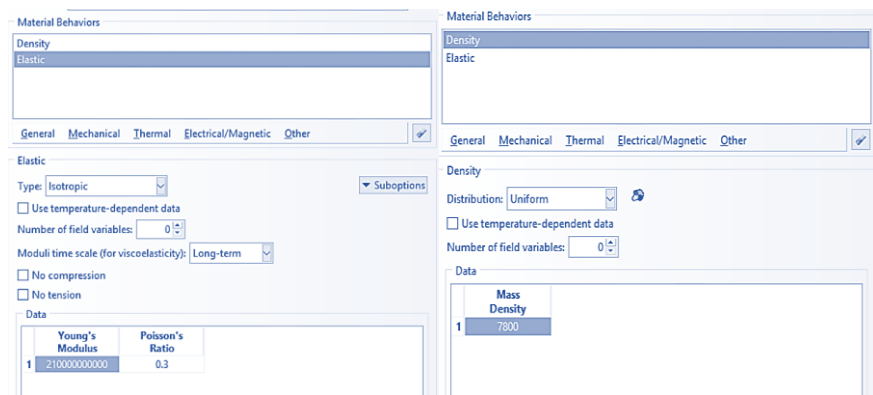


Figure 2. Material Behaviour of the rectangular plate

Notice how Abaqus doesn't provide any units settings, as it is expected that the user is consistent and careful with his/her numbers.

## Boundary Conditions

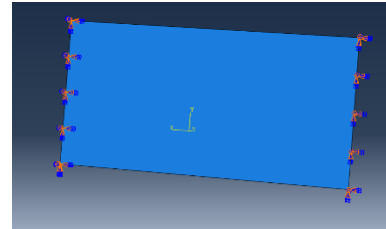
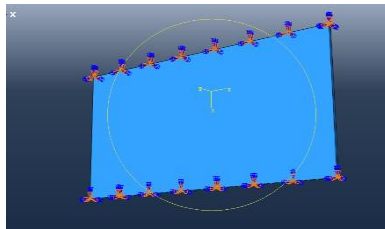
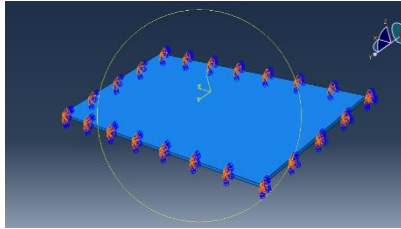
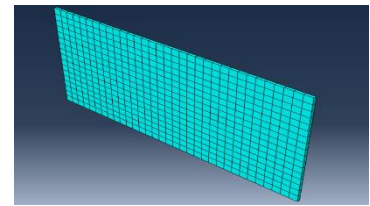


Figure 3. Boundry Consitions inseted on 3 models, CCCC, CFCF, FCCF respectively

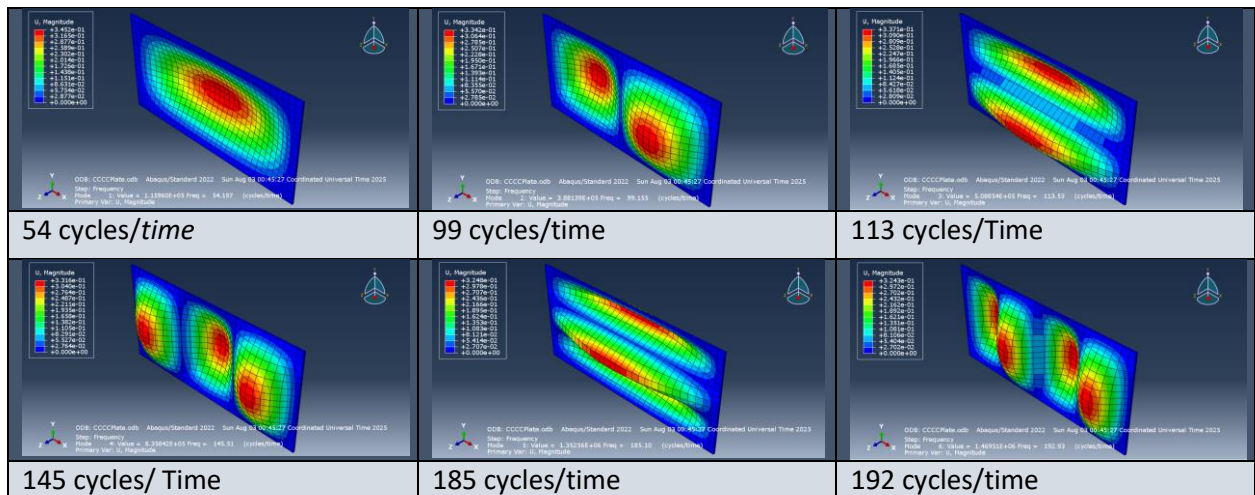
## Mesh Settings

Seed: 0.03  
Element Library: Standard  
Geometric Order: Linear

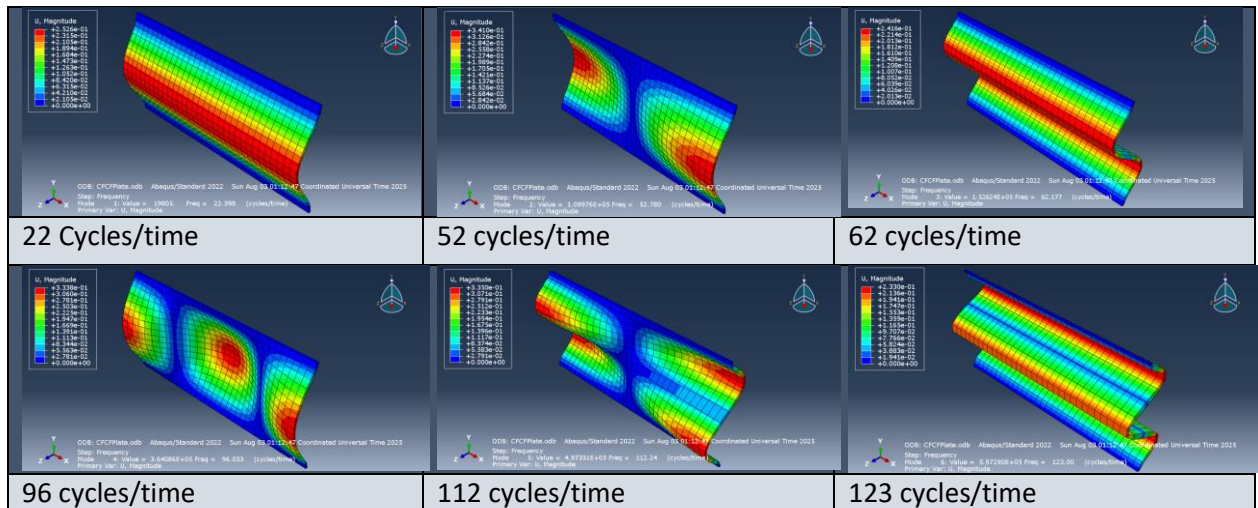


## Analysis & Results

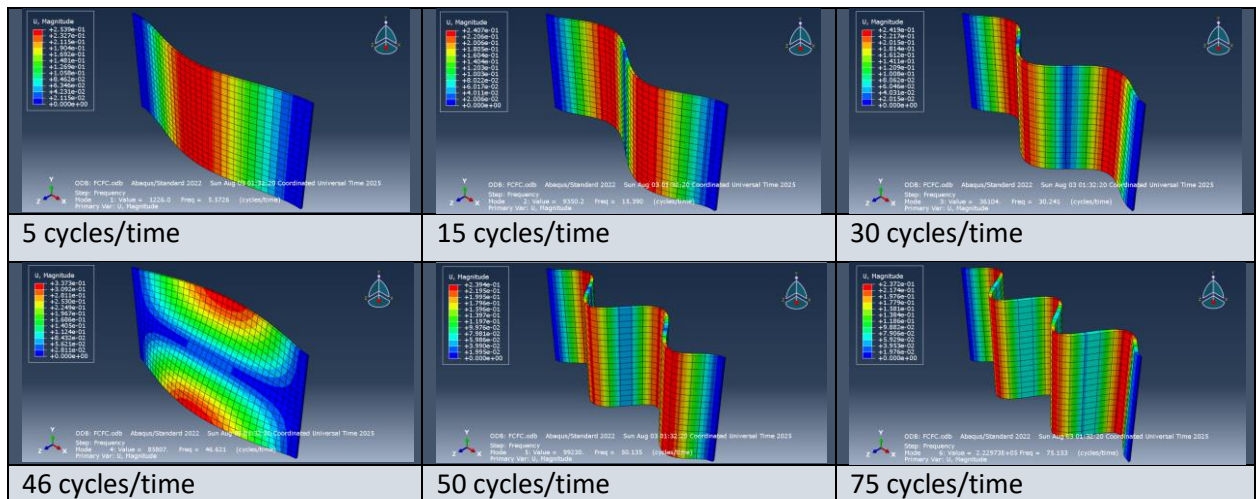
### Model 1: CCCC Plate



## Model 2: FCCF Plate



## Model 3: CFCF Plate



## Interpretation & Comparison

Notice how the natural frequencies of the first model (**CCCC Plate**) is **higher** than the other two models. A CCCC plate has all edges clamped, fully restricting translation and rotation. This creates maximum stiffness, especially against bending and lateral deformation. Because all edges are fixed, the plate cannot easily deform, especially in lower modes (like bending or twisting). So, the mode shapes that do occur correspond to **higher energy (higher frequency)**.

Similarly FCCF's natural frequencies are higher than CFCF, the same analogy can be applied here.

**Boundary conditions** heavily influence the stiffness of a structure, and natural frequency is proportional to the square root of stiffness over mass:

$$f_n = \sqrt{\frac{k}{m}}$$

### Mode Shape Comparison Table

Mode Shape Aspect	CFCF	FCCF	CCCC
Edge Constraint	asymmetric stiffness	asymmetric stiffness	Fully symmetric, maximum stiffness
First Mode Shape	shows large bending across the <b>free edges</b> , with a node line near center	shows large bending across the <b>free edges</b> , with a node line near center	Smooth, symmetric central bulge (domed), minimal edge movement
Symmetry	Typically <b>symmetric</b> about the axis connecting clamped edges	Typically <b>symmetric</b> about the axis connecting clamped edges	Highly <b>symmetric</b> in both axes
Deformation Location	Concentrated near the <b>free edges</b>	Concentrated near the <b>free edges</b>	<b>More distributed</b> , uniform shape
Torsional Modes	May occur earlier due to lower constraints	May occur earlier due to lower constraints	Occur at <b>higher modes due to high stiffness</b>
Frequency Magnitude	<b>Low</b> due to 50% free edges	<b>Low</b> due to 50% free edges (higher than CFCF)	<b>High</b> , due to all clamped edges

### Practical Implications

- **CFCF**: Often used in modeling bridges or panels with partial support. Modes highlight flexibility.
- **CCCC**: Reference model for fully supported structures like metal sheets in frames.