

- i) Initially, the motor runs at no-load with negligible load torque. Show that the overall transfer function of the DC motor speed control system without a controller ($G_M(s)$) is

$$G_M(s) = \frac{k_m}{(Js + b)(L_a s + R_a) + k_m}$$

- ii) The electrical resistance (R_a), inductance (L_a), and the moment of inertia of the DC motor are 1Ω , 0.5 H and 0.2 kgm^2 , respectively. The torque constant (k_m) and viscous friction constant (b) are 5 NmA^{-1} and 1 Nms . Calculate the **overshoot** of the system time response.
- iii) Calculate the required (k_D) parameter of the PD type controller to damp out the present overshoot by 40%.
- iv) If the DC motor runs a load with 3 Nm load torque, find the range of (k_p) to keep the steady state error due to the load torque less than 0.01 rads^{-1} .

[7.5 Marks]

- Q2. a) Write the general form of matrix equations so that a system is represented in state space variable form. Name the matrices in your matrix equations.

[2.5 Marks]

- b) Consider the plant of which the block diagram is shown in Figure Q2 (b).

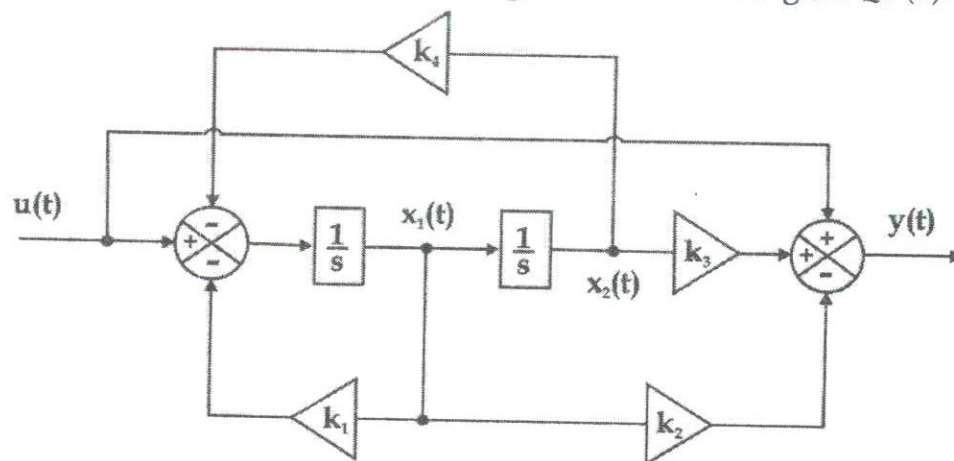


Figure Q2(b).

- i) If the state vector of the plant can be denoted as $[x_1 \ x_2 \ \dots \ x_n]$ and the block diagram has 6, 2, 1 and 8 for k_1 , k_2 , k_3 and k_4 respectively, formulate the state space model of the plant shown in Figure Q2 (b).
- ii) Hence, show that the transfer function of the plant is,

$$G(s) = \frac{s^2 + 4s + 9}{s^2 + 6s + 8}$$

[4.5 Marks]

- c) Consider the transfer function obtained in part (b). The input and output parameters of the plant are angular displacement of a load and voltage.

- i) Find the variation of angular displacement of load with time to the applied voltage $V_T(t) = 2e^{-3t}U(t) \text{ V}$.

- ii) Hence, obtain the steady state value of the system time response.
- iii) Discuss the stability of the control system.

[5.0 Marks]

- Q3. a) i) State the definition of the root locus.
- ii) Consider the closed-loop system shown in Figure Q3.a).i) where;

$$G(s) = \frac{\prod_{i=1}^m (s + z_i)}{\prod_{i=1}^n (s + p_i)}$$

Derive the mathematical expression of the root-locus for this system.

- iii) Derive the mathematical expressions for the two main properties of the root locus for the system shown in Figure Q3.a).i).

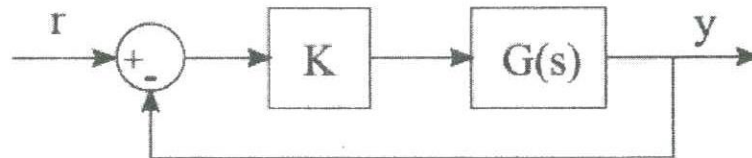


Figure Q3.a).i). Closed-loop control system.

- iv) Figure Q3.a).ii) depicts the root locus of a second-order closed-loop system. Can the percent overshoot of the closed-loop unit step response of this system be kept below 5% by adjusting the gain alone? Provide an explanation for your answer.

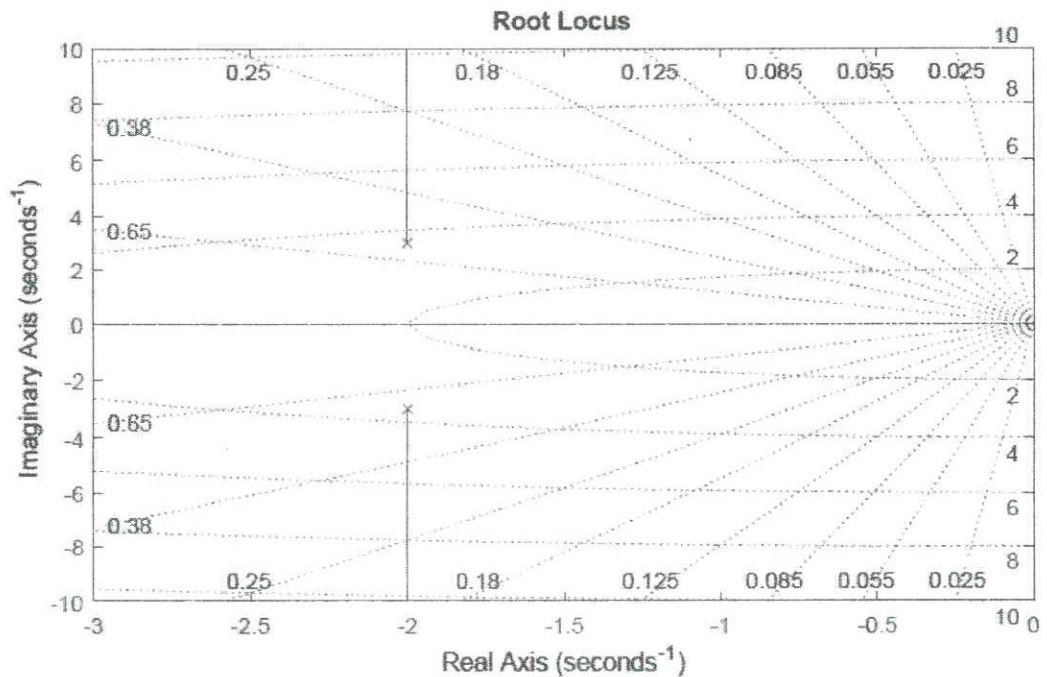


Figure Q3. a) ii).

[5.0 Marks]

- b) Consider the closed-loop control system shown in Q3.a).i). where;

$$G(s) = \frac{s + 10}{(s + 3)(s^2 + 3s + 7)}$$

Answer the following questions with respect to the root locus of this system.

- Calculate the imaginary axis crossings of the root locus.
- Find the asymptotes.
- Find the departure angles at open-loop complex poles.
- Sketch the root locus of the system.
- Determine whether the point $s = -1 + 3.8j$ is a point on the root locus.
- Calculate the poles of the closed-loop system when the gain is 15. Is the closed-loop system stable at this gain?
- Find the range of gain K where the closed-loop system is stable.

[7.0 Marks]

- Q4. a) Figure Q4.a) illustrates the response of a closed-loop system to a unit step input.

Note: The same figure is given on page 8, use that figure to answer the followings and attach that page with your answers.

- Sketch the anticipated response of the closed-loop system to a unit step input as the open-loop gain is increased.
- Sketch the anticipated response of the closed-loop system to a unit step input when a PI controller is added.
- How would you enhance the response in part ii), if you want to achieve a better settling time?

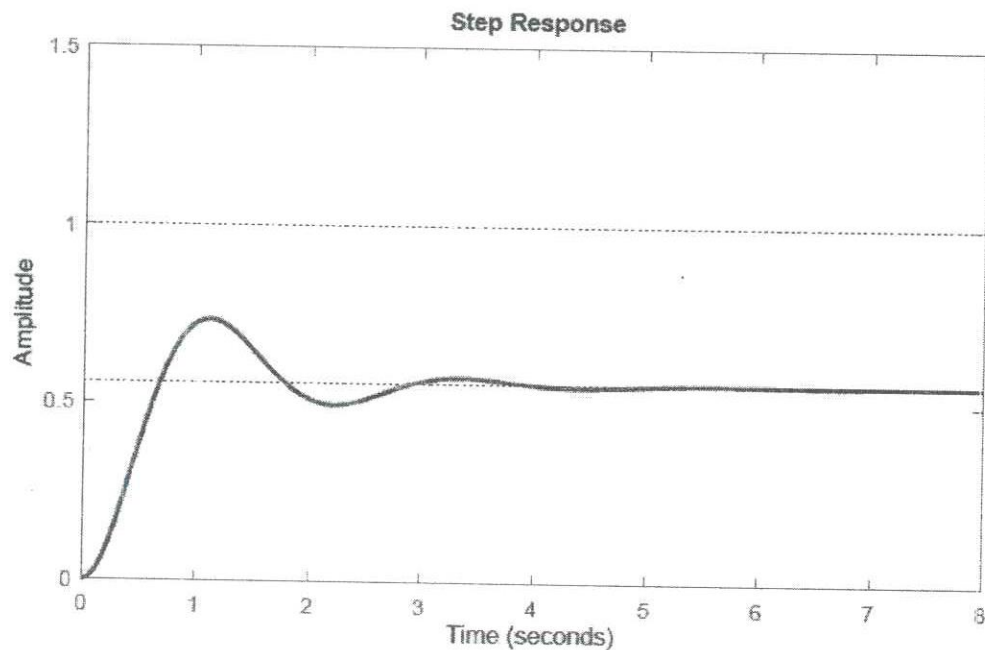


Figure Q4. a).

[3.0 Marks]

- b) Answer this question using your knowledge on root locus design technique. However, it is NOT necessary to sketch the root locus of the given system.

Figure Q4.b) shows a closed-loop control system. The transfer function of the plant is

$$G(s) = \frac{1}{(s+1)(s+2)(s+5)}$$

Design a PID compensator for the above system to meet the following specifications when subjected to a unit step input.

Percent overshoot, $M_p\% \approx 12\%$

Peak time, $t_p \approx 1$ seconds

Steady state error = 0.

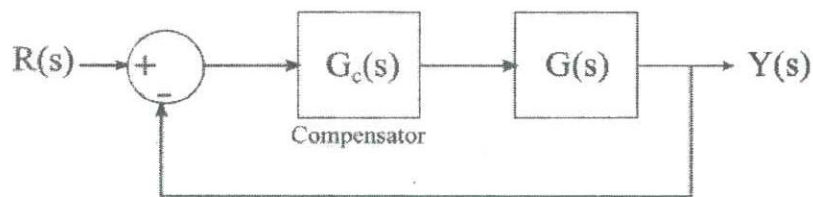


Figure Q4.b).

[9.0 Marks]

- Q5. a) i) Draw the bode diagrams for the following system using asymptotic approximations.

$$G(s) = \frac{s(s+5)}{(s+1)(s+15)}$$

- ii) Briefly explain the relationship between the open-loop frequency response and the stability of the closed-loop system.

[4.0 Marks]

- b) Answer this question using your knowledge on frequency response design technique. However, it is NOT necessary to draw the frequency response of the given system.

Figure Q4.b) shows a closed-loop control system. The transfer function of the plant is

$$G(s) = \frac{1}{(s+4)(s+11)}$$

Design a lead compensator for the above system to meet the following specifications when subjected to a unit step input.

Percent overshoot, $M_p\% \approx 10\%$

Steady state error = 1%

[8.0 Marks]

Formulas you may require:

(All notations have their usual meaning)

$$M_P = e^{-\pi\zeta/\sqrt{1-\zeta^2}}$$

$$t_P = \frac{\pi}{\omega_d}$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

$$\phi_{PM} = \tan^{-1} \frac{2\zeta}{\sqrt{-2\zeta^2 + \sqrt{1 + 4\zeta^4}}}$$

For the lead compensator

$$G_{lead}(s) = \frac{1}{\beta} \frac{s + \frac{1}{T}}{s + \frac{1}{\beta T}}$$

$$\phi_{lead,max} = \sin^{-1} \frac{1 - \beta}{1 + \beta}$$

$$|G_{lead}(j\omega_{max})| = \frac{1}{\sqrt{\beta}}$$

$$\omega_{max} = \frac{1}{T\sqrt{\beta}}$$

Table of Laplace Transforms

| Number | $F(s)$ | $f(t), t \geq 0$ |
|--------|----------------------------------|--|
| 1 | 1 | $\delta(t)$ |
| 2 | $\frac{1}{s}$ | 1(t) |
| 3 | $\frac{1}{s^2}$ | t |
| 4 | $\frac{2!}{s^3}$ | t^2 |
| 5 | $\frac{3!}{s^4}$ | t^3 |
| 6 | $\frac{m!}{s^{m+1}}$ | t^m |
| 7 | $\frac{1}{(s+a)}$ | e^{-at} |
| 8 | $\frac{1}{(s+a)^2}$ | te^{-at} |
| 9 | $\frac{1}{(s+a)^3}$ | $\frac{1}{2!}t^2e^{-at}$ |
| 10 | $\frac{1}{(s+a)^m}$ | $\frac{1}{(m-1)!}t^{m-1}e^{-at}$ |
| 11 | $\frac{a}{s(s+a)}$ | $1 - e^{-at}$ |
| 12 | $\frac{a}{s^2(s+a)}$ | $\frac{1}{a}(at - 1 + e^{-at})$ |
| 13 | $\frac{b-a}{(s+a)(s+b)}$ | $e^{-at} - e^{-bt}$ |
| 14 | $\frac{s}{(s+a)^2}$ | $(1-at)e^{-at}$ |
| 15 | $\frac{a^2}{s(s+a)^2}$ | $1 - e^{-at}(1+at)$ |
| 16 | $\frac{(b-a)s}{(s+a)(s+b)}$ | $be^{-at} - ae^{-bt}$ |
| 17 | $\frac{a}{(s^2+a^2)}$ | $\sin at$ |
| 18 | $\frac{s}{(s^2+a^2)}$ | $\cos at$ |
| 19 | $\frac{s+a}{(s+a)^2+b^2}$ | $e^{-at} \cos bt$ |
| 20 | $\frac{b}{(s+a)^2+b^2}$ | $e^{-at} \sin bt$ |
| 21 | $\frac{a^2+b^2}{s[(s+a)^2+b^2]}$ | $1 - e^{-at} \left(\cos bt + \frac{a}{b} \sin bt \right)$ |

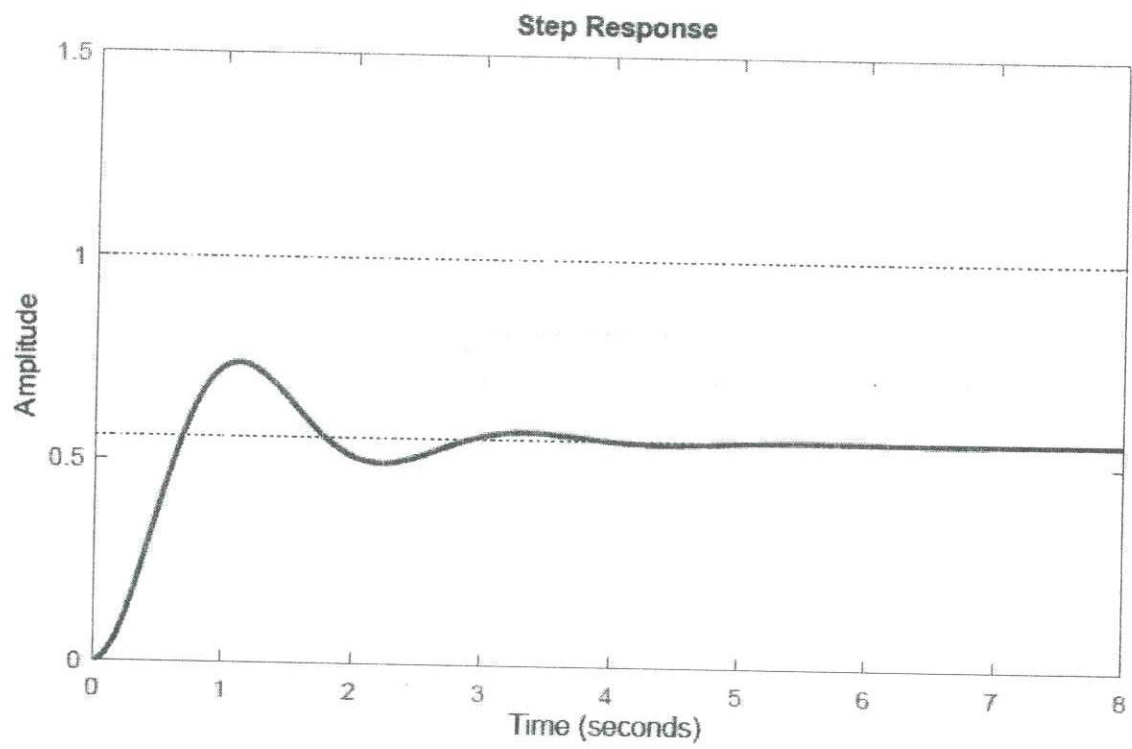


Figure Q4. a).