

# GATE NM-54 2022

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Q: A system with two degrees of freedom, as shown in the figure, has masses  $m_1 = 200\text{kg}$  and  $m_2 = 100\text{kg}$  and stiffness coefficients  $k_1 = k_2 = 200\text{N/m}$ . Then the lowest natural frequency of the system is \_\_\_\_\_ rad/s (rounded off to one decimal place).

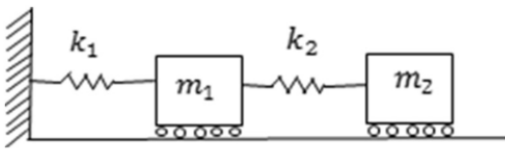


Fig. 0.

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**Solution:**

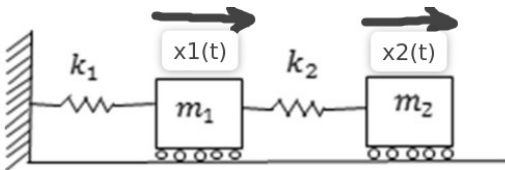


Fig. 0.

Variable	Description	Value
$m_1$	Mass of block 1	$200\text{kg}$
$m_2$	Mass of block 2	$100\text{kg}$
$k_1$	Stiffness coefficient of spring1	$200\text{N/m}$
$k_2$	Stiffness coefficient of spring2	$200\text{N/m}$
$x_i(t)$	Displacement of $i^{\text{th}}$ block	$x_i(t) = \cos(\omega t + \phi_i), x_i \geq 0$

TABLE 0  
INPUT PARAMETERS

$$m_1 \ddot{x}_1(t) - k_2 (x_2(t) - x_1(t)) + k_1 x_1(t) = 0 \quad (1)$$

$$m_2 \ddot{x}_2(t) + k_2 (x_2(t) - x_1(t)) = 0 \quad (2)$$

Writing (1) and (2) in matrix form:

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$x(t) \xleftrightarrow{\mathcal{L}} X(s) \quad (3)$$

$$x''(t) \xleftrightarrow{\mathcal{L}} s^2 X(s) - sx(0) - x'(0) \quad (4)$$

Applying Laplace transform for (1) and (2) assuming they are at their respective maximum displacement at  $t=0$

$$m_1 (s^2 X_1(s) - sx_1(0) - 0) - k_2 (X_2(s) - X_1(s)) + k_1 X_1(s) = 0 \quad (5)$$

$$m_2 (s^2 X_2(s) - sx_2(0) - 0) + k_2 (X_2(s) - X_1(s)) = 0 \quad (6)$$

Writing (5) and (6) in matrix form:

$$\begin{bmatrix} m_1 s^2 + (k_1 + k_2) & -k_2 \\ -k_2 & m_2 s^2 + k_2 \end{bmatrix} \begin{bmatrix} X_1(s) \\ X_2(s) \end{bmatrix} = \begin{bmatrix} m_1 s x_1(0) \\ m_2 s x_2(0) \end{bmatrix}$$

$$\begin{bmatrix} X_1(s) \\ X_2(s) \end{bmatrix} = \begin{bmatrix} m_1 s x_1(0) \\ m_2 s x_2(0) \end{bmatrix} \begin{bmatrix} m_2 s^2 + k_2 & k_2 \\ k_2 & m_1 s^2 + (k_1 + k_2) \end{bmatrix}$$

Finding inverse Laplace transform for this on both sides;

$$\begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} m_1 x_1(0) u(t) \\ m_2 x_2(0) u(t) \end{bmatrix} \begin{bmatrix} m_2 t u(t) + k_2 \delta(t) & k_2 \delta(t) \\ k_2 \delta(t) & m_1 t u(t) + (k_1 + k_2) \delta(t) \end{bmatrix}$$

For frequency put  $s = j\omega$ ,

$$\begin{bmatrix} -m_1 \omega^2 + (k_1 + k_2) & -k_2 \\ -k_2 & -m_2 \omega^2 + k_2 \end{bmatrix} \begin{bmatrix} X_1(j\omega) \\ X_2(j\omega) \end{bmatrix} = \begin{bmatrix} m_1 j\omega x_1(0) \\ m_2 j\omega x_2(0) \end{bmatrix}$$

For free vibration

$$\det \begin{pmatrix} -m_1 \omega^2 + (k_1 + k_2) & -k_2 \\ -k_2 & -m_2 \omega^2 + k_2 \end{pmatrix} = 0$$

$$\left(-m_1\omega^2 + (k_1 + k_2)\right)\left(-m_2\omega^2 + k_2\right) - (-k_2)(-k_2) = 0 \quad (7)$$

$$m_1m_2\omega^4 - [(k_1 + k_2)m_2 + k_2m_1]\omega^2 + k_1k_2 = 0 \quad (8)$$

$$\Rightarrow \omega^4 - \frac{[(k_1 + k_2)m_2 + k_2m_1]}{m_1m_2}\omega^2 + \frac{k_1k_2}{m_1m_2} = 0 \quad (9)$$

Substituting the values;

$$\Rightarrow \omega^4 - 4\omega^2 + 2 = 0 \quad (10)$$

$$\omega^2 = \frac{4 \pm \sqrt{16 - 8}}{2} \quad (11)$$

$$= 2 \pm \sqrt{2} \quad (12)$$

$$\omega = \pm \sqrt{2 \pm \sqrt{2}} \quad (13)$$

$$\Rightarrow \omega_{least} = 0.765rad/s \quad (14)$$