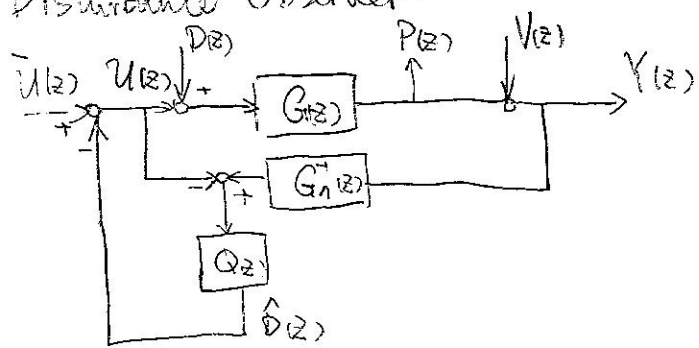


Disturbance Observer



closed loop dynamics

$$P = \frac{G(1-Q)}{1+Q(G_n^{-1}G-1)} D + \frac{G}{1+Q(G_n^{-1}G-1)} \bar{U} - \frac{GQ G_n^{-1}}{1+Q(G_n^{-1}G-1)} V$$

$$G(z) = G_n(z)(1+\Delta(z)) \quad \Delta(z) \text{ stable}$$

1. $Q(z) = G_n^{-1}(z)$ have to be causal

2. Disturbance rejection

$$Q(e^{j\omega}) \approx 1 \cdot \frac{G_n(1+\Delta)(1-Q)}{1+Q\Delta} D$$

at desired frequency range

3. Sensor noise, $|Q(e^{j\omega})|$ small for sensor noise active frequencies

4. $\|\Delta(e^{j\omega}) Q(e^{j\omega})\| \leq 1$ for all ω

With a closed loop controller, how to guarantee stability?

1. Controller design

2. minimum phase $G(z)$

If there is no mis-match?

Frequency shaped LQ.

Discrete case

cost fun

$$J = \sum_{k=0}^{\infty} \{ x^T(k) Q x(k) + u^T(k) R u(k) \}$$

1 cost fun in frequency domain

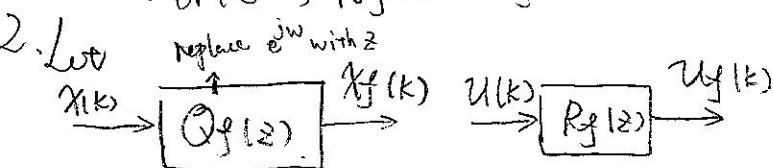
$$J = \int_{-\pi}^{\pi} X^T(e^{j\omega}) Q X(e^{j\omega}) + U^T(e^{j\omega}) R U(e^{j\omega}) d\omega$$

Instead of keeping Q, R constant, design

$$Q(e^{j\omega}) = Q_f^T(e^{j\omega}) Q_f(e^{j\omega})$$

$$R(e^{j\omega}) = R_f^T(e^{j\omega}) R_f(e^{j\omega})$$

$$J = \int_{-\pi}^{\pi} X^T(e^{j\omega}) Q_f^T(e^{j\omega}) Q_f(e^{j\omega}) X(e^{j\omega}) + U^T(e^{j\omega}) R_f^T(e^{j\omega}) R_f(e^{j\omega}) U(e^{j\omega}) d\omega$$



$$z_1(k+1) = A_1 z_1(k) + B_1 x(k)$$

$$x_f(k) = C_1 z_1(k) + D_1 x(k)$$

$$z_2(k+1) = A_2 z_2(k) + B_2 u(k)$$

$$u_f(k) = C_2 z_2(k) + D_2 u(k)$$

* Size of A_1 = Size of Q_f
 Column of B_1 = # of element of x .

$$3. J = \sum_{k=0}^{\infty} \{ x_f^T(k) x_f(k) + u_f^T(k) u_f(k) \}$$

$$\begin{bmatrix} x(k+1) \\ z_1(k+1) \\ z_2(k+1) \end{bmatrix} = \underbrace{\begin{bmatrix} A & 0 & 0 \\ B_1 & A_1 & 0 \\ 0 & 0 & A_2 \end{bmatrix}}_{A_e} \begin{bmatrix} x(k) \\ z_1(k) \\ z_2(k) \end{bmatrix} + \underbrace{\begin{bmatrix} B \\ 0 \\ B_2 \end{bmatrix}}_{B_e} u(k)$$

$$\begin{bmatrix} x_f(k) \\ u_f(k) \end{bmatrix} = \underbrace{\begin{bmatrix} D_1 & C_1 & 0 \\ 0 & 0 & C_2 \end{bmatrix}}_{C_e} \begin{bmatrix} x(k) \\ z_1(k) \\ z_2(k) \end{bmatrix} + \underbrace{\begin{bmatrix} 0 \\ D_2 \end{bmatrix}}_{D_e} u(k)$$

$$J = \sum_{k=0}^{\infty} \left\{ \begin{bmatrix} x_e(k) \\ u(k) \end{bmatrix}^T \begin{bmatrix} C_e^T \\ D_e^T \end{bmatrix} \begin{bmatrix} C_e & D_e \end{bmatrix} \begin{bmatrix} x_e(k) \\ u(k) \end{bmatrix} \right\}$$

$$u = [B_e^T P B_e + D_e^T D_e]^{-1} [B_e^T P A_e + D_e^T C_e]$$

$$P = A_e^T P A_e + C_e^T C_e - [A_e^T P B_e + C_e^T D_e] [B_e^T P B_e + D_e^T D_e]^{-1} [B_e^T P A_e + D_e^T C_e]$$

Continuous case

cost fun

$$J = \int_0^{\infty} (x^T(t) Q x(t) + u^T(t) R u(t)) dt$$

1 Cost fun in frequency domain

$$J = \frac{1}{2\pi} \int_{-\infty}^{\infty} (X^T(-j\omega) Q X(j\omega) + U^T(j\omega) R U(j\omega)) d\omega$$

Instead of keeping Q & R constant, design

$$Q(j\omega) = Q_f^T(-j\omega) Q_f(j\omega)$$

$$R(j\omega) = R_f^T(-j\omega) R_f(j\omega)$$

$$J = \frac{1}{2\pi} \int_{-\infty}^{\infty} (X^T(-j\omega) Q_f^T(-j\omega) Q_f(j\omega) X(j\omega) + U^T(-j\omega) R_f^T(-j\omega) R_f(j\omega) U(j\omega)) d\omega$$



$$\dot{z}_1(t) = A_1 z_1(t) + B_1 x(t)$$

$$x_f(t) = C_1 z_1(t) + D_1 x(t)$$

$$\dot{z}_2(t) = A_2 z_2(t) + B_2 u(t)$$

$$u_f(t) = C_2 z_2(t) + D_2 u(t)$$

* Size of A_1 = Size of Q_f
 Column of B_1 = # of element of x

$$3. J = \int_0^{\infty} (x_f^T(t) x_f(t) + u_f^T(t) u_f(t)) dt$$

$$\frac{d}{dt} \begin{bmatrix} x(t) \\ z_1(t) \\ z_2(t) \end{bmatrix} = \underbrace{\begin{bmatrix} A & 0 & 0 \\ B_1 & A_1 & 0 \\ 0 & 0 & A_2 \end{bmatrix}}_{A_e} \begin{bmatrix} x(t) \\ z_1(t) \\ z_2(t) \end{bmatrix} + \underbrace{\begin{bmatrix} B \\ 0 \\ B_2 \end{bmatrix}}_{B_e} u(t)$$

$$x_f = \underbrace{[D_1 \ C_1 \ 0]}_{C_e} x_e \quad u_f = [0 \ 0 \ C_2] x_e + D_2 u$$

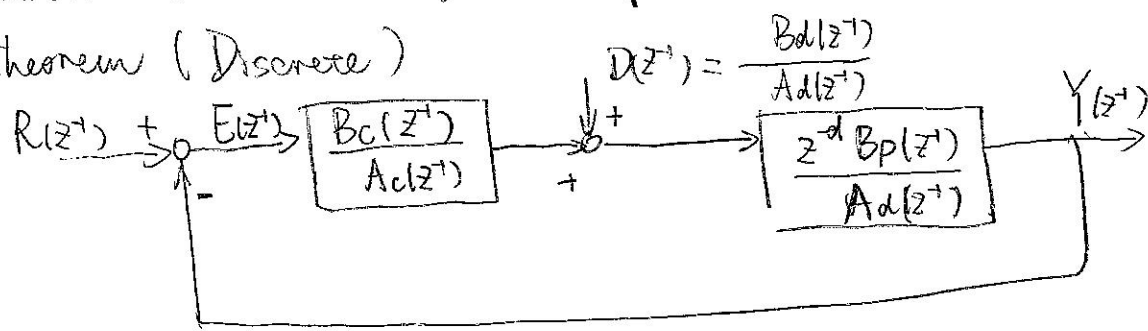
$$J = \int_0^{\infty} x_e^T \begin{bmatrix} D_1^T D_1 & D_1^T C_1 & 0 \\ C_1^T D_1 & C_1^T C_1 & 0 \\ 0 & 0 & C_2^T C_2 \end{bmatrix} x_e + 2u^T \underbrace{[0 \ 0 \ D_2^T C_2]}_{N_e} x_e + u^T \underbrace{D_2^T D_2}_{R_e} u dt$$

$$u = -R_e^{-1} (B_e^T P_e + N_e) x_e$$

$$A_e^T P_e + P_e A_e - (B_e^T P_e + N_e)^T R_e^{-1} (B_e^T P_e + N_e) + Q_e$$

Internal Model Principle & Repetitive control

Theorem (Discrete)



Assume $B_p(z^{-1}) = 0$ and $A_d(z^{-1}) = 0$ do not have common zeros. If the closed loop is asymptotically stable, and $A_d(z^{-1})$ can be factorized as $A_c(z^{-1}) = A_d(z^{-1}) A'_c(z^{-1})$ then the disturbance is asymptotically rejected.

$d(k)$	$A_d(z^{-1})$
do const	$1 - z^{-1}$
$\cos(\omega_0 k)$ & $\sin(\omega_0 k)$	$1 - 2z^{-1} \cos(\omega_0) + z^{-2}$
$d(k) = \alpha k + \beta$	$1 - 2z^{-1} + z^{-2}$
$d(k) = d(k-N)$	$1 - z^{-N}$

Procedure in design a controller to reject repetitive disturbance

1. For a disturbance, find $D(z^{-1})$
2. By internal model principle, the denominator of controller should include $A_d(z^{-1})$, i.e. controller $G_c(z^{-1}) = \frac{R(z^{-1})}{A_d(z^{-1}) S(z^{-1})}$, R & S are to be designed
3. design close loop characteristic for $A_{closed}(z^{-1})$
4. solve R & S by Diophantine eqn

$$A_d(z^{-1}) A_p(z^{-1}) S(z^{-1}) + z^{-d} B_p(z^{-1}) R(z^{-1}) = A_{closed}(z^{-1})$$